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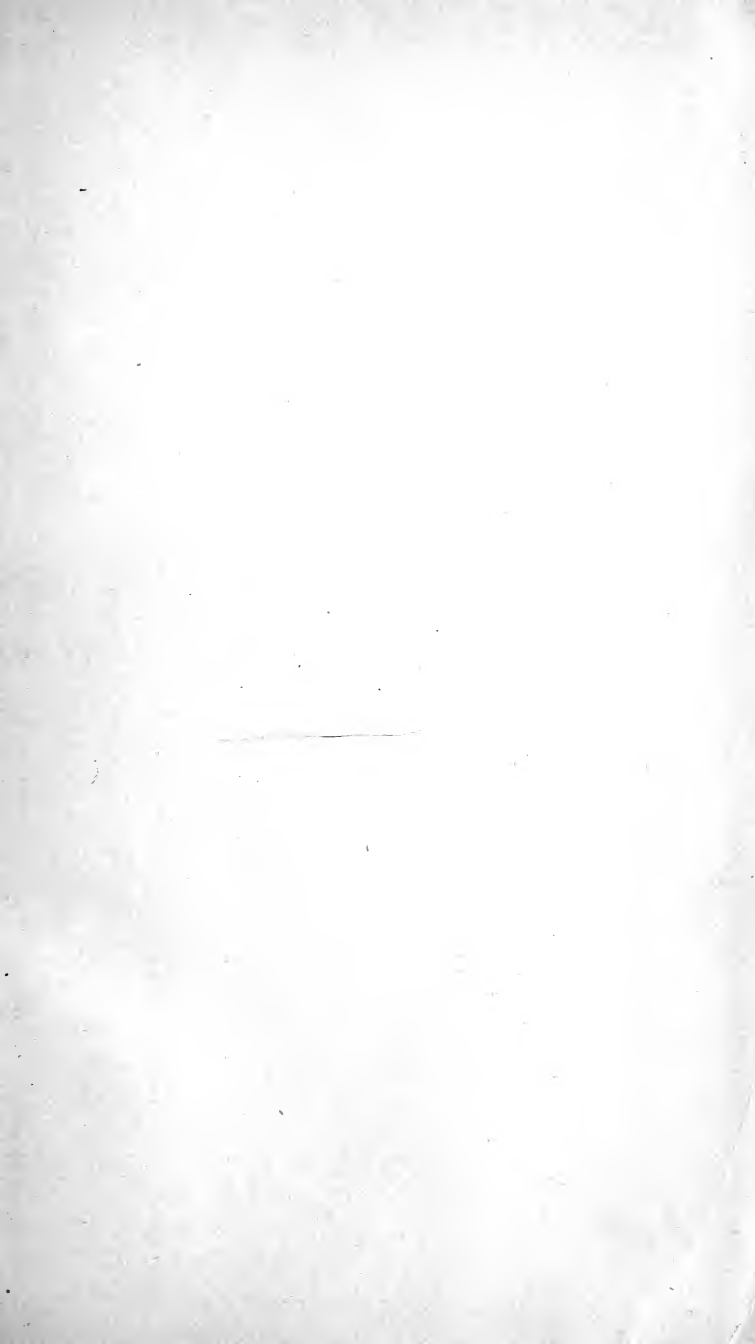


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WORKS OF
PROF. WALTER LORING WEBB

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Railroad Construction.—Theory and Practice.

A Text-book for the Use of Students in Colleges and Technical Schools. Fifth Edition. Revised and Enlarged. 16mo. xvii + 789 pages and 217 figures and 10 plates. Morocco, \$5.00.

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RAILROAD CONSTRUCTION.

THEORY AND PRACTICE.

*A TEXT-BOOK FOR THE USE OF STUDENTS
IN COLLEGES AND TECHNICAL SCHOOLS.*

BY

WALTER LORING WEBB, C.E.,

*Member American Society of Civil Engineers; Member American Railway
Engineering Association; Assistant Professor of Civil Engineer-
ing (Railroad Engineering) in the University of
Pennsylvania, 1893-1901; etc.*

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PREFACE TO FIRST EDITION.

THE preparation of this book was begun several years ago, when much of the subject-matter treated was not to be found in print, or was scattered through many books and pamphlets, and was hence unavailable for student use. Portions of the book have already been printed by the mimeograph process or have been used as lecture-notes, and hence have been subjected to the refining process of class-room use.

The author would call special attention to the following features:

a. Transition curves; the multiform-compound-curve method is used, which has been followed by many railroads in this country; the particular curves here developed have the great advantage of being exceedingly simple, and although the method is not theoretically exact, it is demonstrable that the differences are so small that they may safely be neglected.

b. A system of earthwork computations which makes the multiplication and reduction in a single operation by means of a slide-rule and which enables one to compute readily the volume of the most complicated earthwork forms with an accuracy which is only limited by the precision of the cross-sectioning.

c. The "mass curve" in earthwork; the theory and use of this very valuable process.

d. Tables I, II, III, and IV have been computed *ab novo*. Tables I and II were checked (after computation) with other tables, which are generally considered as standard, and all discrepancies were further examined. They are believed to be perfect.

e. Tables V, VI, VII, and IX have been borrowed, by permission, from "Ludlow's Mathematical Tables." It is believed that five-place tables give as accurate results as actual field

practice requires. Tables VIII and X have been compiled to conform with Ludlow's system.

The author wishes to acknowledge his indebtedness to Mr. Chas. A. Sims, civil engineer and railroad contractor, for reading and revising the portions relating to the cost of earthwork.

Since the book is written primarily for students of railroad engineering in technical institutions, the author has assumed the usual previous preparation in algebra, geometry, and trigonometry.

WALTER LORING WEBB.

UNIVERSITY OF PENNSYLVANIA,
PHILADELPHIA,
Jan. 1, 1900.

PREFACE TO SECOND EDITION.

SINCE the issue of the first edition the author has conferred with many noted educators in civil engineering, among them the late Professors E. A. Fuertes and J. B. Johnson, regarding the most desirable size of page for this book. The inconvenience of the octavo edition for field-work was found to be limiting its use. It was therefore decided to recast the whole work and reduce the page from "octavo" to "pocket-book" size. Advantage was then taken of the opportunity to revise freely and to add new matter. The original text has now been almost doubled by the addition of several chapters on structures, train resistance, rolling stock, etc., and also several chapters giving the fundamental principles of the economics of railroad location. Those who are familiar with the late Mr. Wellington's masterpiece, "The Economic Theory of Railway Location," will readily appreciate the author's indebtedness to that work. But while the same general method has been followed, the author has taken advantage of the classification of operating expenses adopted by the Interstate Commerce Commission, has used the figures

published by them (which were unavailable when Mr. Wellington wrote), and has developed the theory on an independent basis, with the exception of a few minor details. Those who deny the utility of such methods of computation are referred to §§ 367, 426, and elsewhere for a practical discussion of that subject.

The author's primary aim has been to produce a "text-book for students," and the subject-matter has therefore been cut down to that which may properly be required of students in the time usually allotted to railroad work in a civil-engineering curriculum. On this account no extended discussion has been given to the multitudinous forms of various railroad devices in the chapters on structures. The aim has been to teach the principles and to guide the students into proper methods of investigation.

January, 1903.

PREFACE TO THIRD EDITION.

IN the present edition Tables IX and X have been entirely changed, the tables having been rewritten so that the values are given for single minutes rather than for each ten minutes. There has also been added a table of squares, cubes, square roots, cube roots, and reciprocals, which are frequently of so great service in computations. Advantage has also been taken of the opportunity to make numerous typographical and verbal changes.

February, 1905.

PREFACE TO FOURTH EDITION.

IN this edition a very extensive revision has been made in the chapter on Earthwork. Table XXXIII, giving the volume of level sections, has been added to the book, with a special demonstration of the method of utilizing this table for pre-

liminary and approximate earthwork calculations. A demonstration, with table, for determining the economics of ties has also been added. In accordance with the suggestions of Prof. R. B. H. Begg, of Syracuse University, additions have been made to Table IV, which facilitate the solution of problems in transition curves. Very numerous and sometimes extensive alterations and additions, as well as mere verbal and typographical changes, have been made in various parts of the book. The chapters on Economics have been revised to make them conform to more recent estimates of cost of operation.

July, 1908.

PREFACE TO FIFTH EDITION.

THE very radical changes made, during recent years, by the Interstate Commerce Commission, in the analysis of operating expenses of railroads, have required that all of the chapters on Economics should be almost entirely rewritten. The writer has also endeavored to make all recommendations as to construction work to conform with the most recent recommendations on those points by the American Railway Engineering Association, who granted the Author special permission to make such quotations. This gives all such statements the highest possible authority. Advantage was taken of the opportunity to correct minor typographical errors and blemishes which have been discovered since issuing the fourth edition.

March, 1913.

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Figure 1. The effect of the concentration of the *Agrobacterium* suspension on the transformation efficiency of *Agrobacterium* strains.

$\Gamma = \{ \gamma_1, \gamma_2, \gamma_3, \gamma_4, \gamma_5, \gamma_6, \gamma_7, \gamma_8, \gamma_9, \gamma_{10}, \gamma_{11}, \gamma_{12}, \gamma_{13}, \gamma_{14}, \gamma_{15}, \gamma_{16}, \gamma_{17}, \gamma_{18}, \gamma_{19}, \gamma_{20}, \gamma_{21}, \gamma_{22}, \gamma_{23}, \gamma_{24}, \gamma_{25}, \gamma_{26}, \gamma_{27}, \gamma_{28}, \gamma_{29}, \gamma_{30}, \gamma_{31}, \gamma_{32}, \gamma_{33}, \gamma_{34}, \gamma_{35}, \gamma_{36}, \gamma_{37}, \gamma_{38}, \gamma_{39}, \gamma_{40}, \gamma_{41}, \gamma_{42}, \gamma_{43}, \gamma_{44}, \gamma_{45}, \gamma_{46}, \gamma_{47}, \gamma_{48}, \gamma_{49}, \gamma_{50}, \gamma_{51}, \gamma_{52}, \gamma_{53}, \gamma_{54}, \gamma_{55}, \gamma_{56}, \gamma_{57}, \gamma_{58}, \gamma_{59}, \gamma_{60}, \gamma_{61}, \gamma_{62}, \gamma_{63}, \gamma_{64}, \gamma_{65}, \gamma_{66}, \gamma_{67}, \gamma_{68}, \gamma_{69}, \gamma_{70}, \gamma_{71}, \gamma_{72}, \gamma_{73}, \gamma_{74}, \gamma_{75}, \gamma_{76}, \gamma_{77}, \gamma_{78}, \gamma_{79}, \gamma_{80}, \gamma_{81}, \gamma_{82}, \gamma_{83}, \gamma_{84}, \gamma_{85}, \gamma_{86}, \gamma_{87}, \gamma_{88}, \gamma_{89}, \gamma_{90}, \gamma_{91}, \gamma_{92}, \gamma_{93}, \gamma_{94}, \gamma_{95}, \gamma_{96}, \gamma_{97}, \gamma_{98}, \gamma_{99}, \gamma_{100} \}$

1 2 3 4

RAILROAD CONSTRUCTION.

CHAPTER I.

RAILROAD SURVEYS

THE proper conduct of railroad surveys presupposes an adequate knowledge of almost the whole subject of railroad engineering, and particularly of some of the complicated questions of Railroad Economics, which are not generally studied except at the latter part of a course in railroad engineering, if at all. This chapter will therefore be chiefly devoted to methods of instrumental work, and the problem of choosing a general route will be considered only as it is influenced by the topography or by the application of those elementary principles of Railroad Economics which are self-evident or which may be accepted by the student until he has had an opportunity of studying those principles in detail

RECONNOISSANCE SURVEYS.

1. **Character of a reconnoissance survey.** A reconnoissance survey is a very hasty examination of a belt of country to determine which of all possible or suggested routes is the most promising and best worthy of a more detailed survey. It is essentially very rough and rapid. It aims to discover those salient features which instantly stamp one route as distinctly superior to another and so narrow the choice to routes which are so nearly equal in value that a more detailed survey is necessary to decide between them.

2. **Selection of a general route.** The general question of running a railroad between two towns is usually a financial rather

than an engineering question. Financial considerations usually determine that a road *must* pass through certain more or less important towns between its termini. When a railroad runs through a thickly settled and very flat country, where, from a topographical standpoint, the road may be run by any desired route, the "right-of-way agent" sometimes has a greater influence in locating the road than the engineer. But such modifications of alignment, on account of business considerations, are foreign to the engineer's side of the subject, and it will be hereafter assumed that topography alone determines the location of the line. The consideration of those larger questions combining finance and engineering (such as passing by a town on account of the necessary introduction of heavy grades in order to reach it) will be taken up in Chap. XIX, *et seq.*

3. **Valley route.** This is perhaps the simplest problem. If the two towns to be connected lie in the same valley, it is frequently only necessary to run a line which shall have a nearly uniform grade. The reconnoissance problem consists largely in determining the difference of elevation of the two termini of this division and the approximate horizontal distance so that the proper grade may be chosen. If there is a large river running through the valley, the road will probably remain on one side or the other throughout the whole distance, and both banks should be examined by the reconnoissance party to determine which is preferable. If the river may be easily bridged, both banks may be alternately used, especially when better alignment is thereby secured. A river valley has usually a steeper slope in the upper part than in the lower part. A uniform grade throughout the valley will therefore require that the road climbs up the side slopes in the lower part of the valley. In case the "ruling grade" * for the whole road is as great as or greater than the steepest natural valley slope, more freedom may be used in adopting that alignment which has the least cost—regardless of grade. The natural slope of large rivers is almost invariably so low that grade has no influence in determining the choice of location. When bridging is necessary, the river banks should be examined for suitable locations for abutments

* The *ruling grade* may here be loosely defined as the maximum grade which is permissible. This definition is not strictly true, as may be seen later when studying Railroad Economics, but it may here serve the purpose.

and piers. If the soil is soft and treacherous, much difficulty may be experienced and the choice of route may be largely determined by the difficulty of bridging the river except at certain favorable places.

4. Cross-country route. A cross-country route always has one or more summits to be crossed. The problem becomes more complex on account of the greater number of possible solutions and the difficulty of properly weighing the advantages and disadvantages of each. The general aim should be to choose the lowest summits and the highest stream crossings, provided that by so doing the grades between these determining points shall be as low as possible and shall not be greater than the ruling grade of the road. Nearly all railroads combine cross-country and valley routes to some extent. Usually the steepest natural slopes are to be found on the cross-country routes, and also the greatest difficulty in securing a low through grade. An approximate determination of the ruling grade is usually made during the reconnoissance. If the ruling grade has been previously decided on by other considerations, the leading feature of the reconnoissance survey will be the determination of a general route along which it will be possible to survey a line whose maximum grade shall not exceed the ruling grade.

5. Mountain route. The streams of a mountainous region frequently have a slope exceeding the desired ruling grade. In such cases there is no possibility of securing the desired grade by following the streams. The penetration of such a region may only be accomplished by "development"—accompanied perhaps by tunneling. "Development" consists in deliberately increasing the length of the road between two extremes of elevation so that the rate of grade shall be as low as desired. The usual method of accomplishing this is to take advantage of some convenient formation of the ground to introduce some lateral deviation. The methods may be somewhat classified as follows:

(a) Running the line up a convenient lateral valley, turning a sharp curve and working back up the opposite slope. As shown in Fig. 1, the considerable rise between *A* and *B* was surmounted by starting off in a very different direction from the general direction of the road; then, when about one-half of the desired rise had been obtained, the line crossed the valley and continued the climb along the opposite slope. (b) *Switch-*

back. On the steep side-hill *BCD* (Fig. 1) a very considerable gain in elevation was accomplished by the switchback *CD*. The gain in elevation from *B* to *D* is very great. On the other hand, the speed must always be slow; there are two complete stoppages of the train for each run; all trains must run backward from *C* to *D*. (c) *Bridge spiral.* When a valley is so narrow at some point that a bridge or viaduct of reasonable length can span the valley at a considerable elevation above the

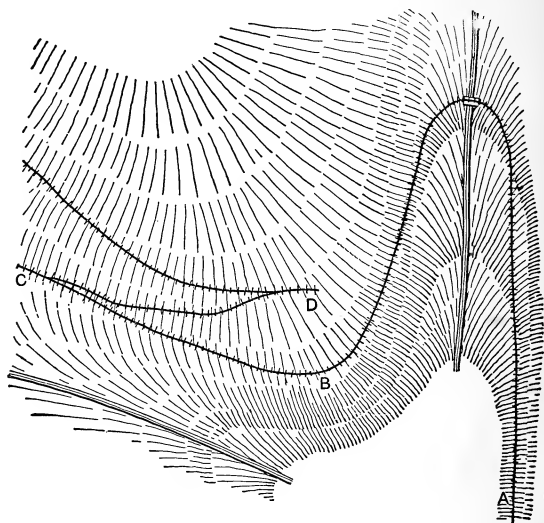
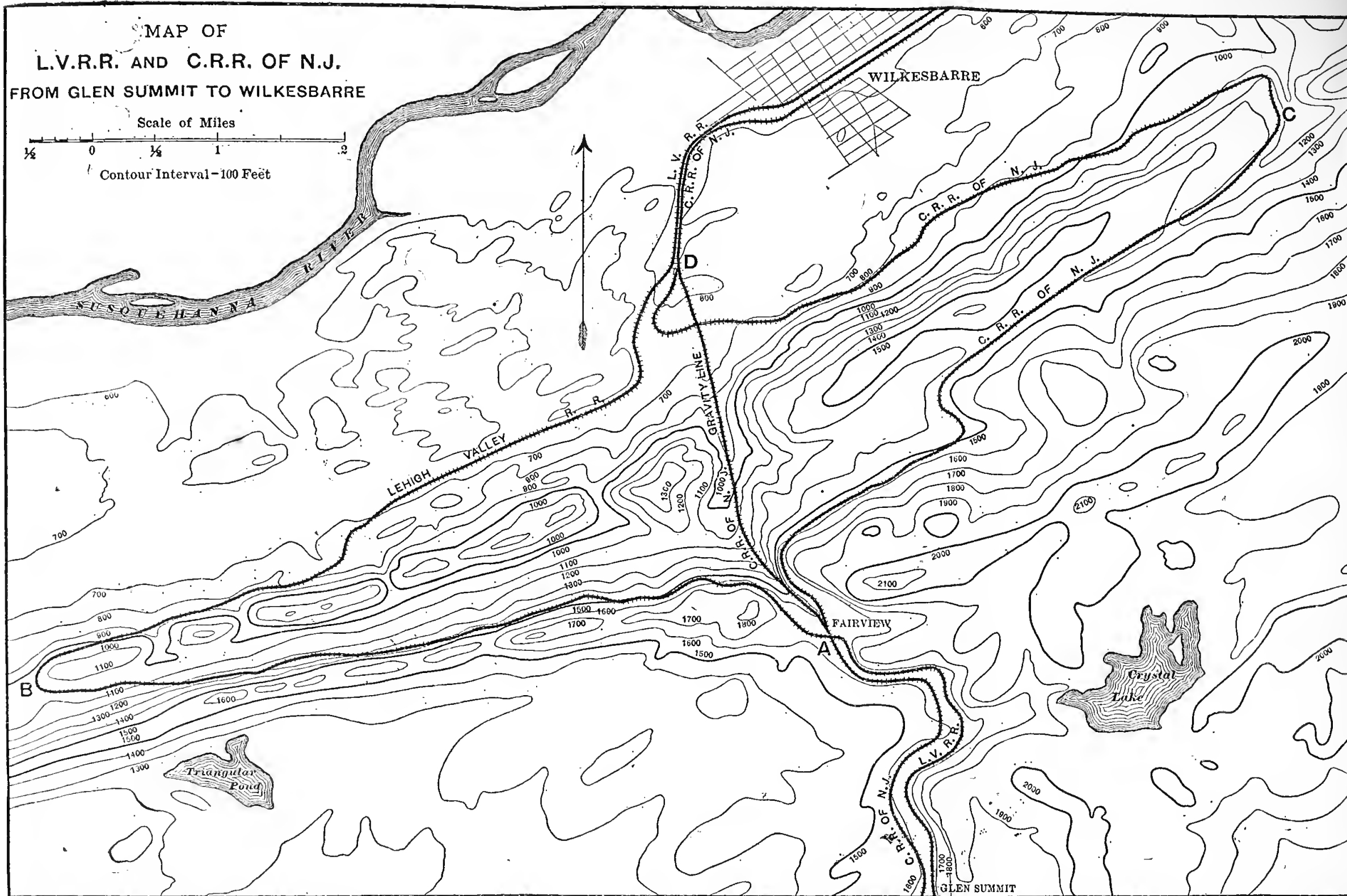
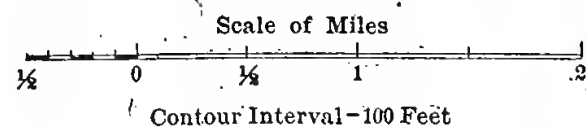


FIG. 1.

bottom of the valley, a bridge spiral may be desirable. In Fig. 2 the line ascends the stream valley past *A*, crosses the stream at *B*, works back to the narrow place at *C*, and there crosses itself, having gained perhaps 100 feet in elevation. (d) *Tunnel spiral* (Fig. 3). This is the reverse of the previous plan. It implies a thin steep ridge, so thin at some place that a tunnel through it will not be excessively long. Switchbacks and spirals are sometimes necessary in mountainous countries, but they should not be considered as normal types of construction. A region must be very difficult if these devices cannot be avoided.

On Plate I are shown three separate ways (as actually constructed) of running a railroad between two points a little over three miles apart and having a difference of elevation of nearly

MAP OF
L.V.R.R. AND C.R.R. OF N.J.
FROM GLEN SUMMIT TO WILKESBARRE



(To face page 4.)

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1100 feet. At *A* the Central R. R. of New Jersey runs *under* the Lehigh Valley R. R. and soon turns off to the northeast for about six miles, then doubles back, reaching *D*, a fall of about 1050 feet with a track distance of about 12.7 miles. The L. V. R. R. at *A* runs to the westward for six to seven miles,

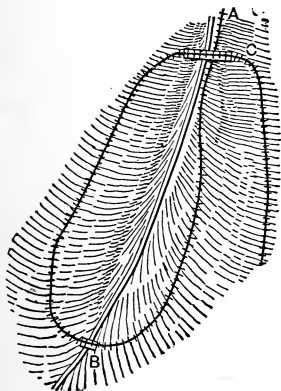


FIG. 2.

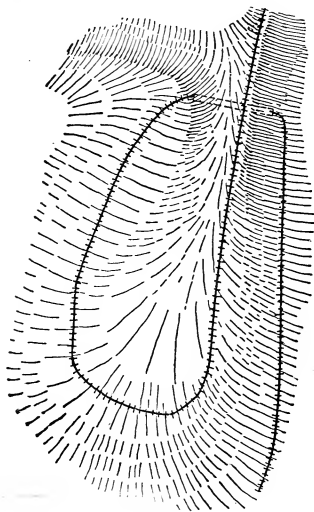


FIG. 3.

then turns back until the roads are again close together at *D*. The track distance is about 14 miles and the drop a little greater, since at *A* the L. V. R. R. crosses *over* the other, while at *D* they are at practically the same level. From *B* to *C* the distance is over eleven miles. From *A* directly down to *D* the C. R. R. of N. J. runs a "gravity" road, used exclusively for freight, on which cars alone are hauled by cable. The main-line routes are remarkable examples of sheer "development." Even as constructed the L. V. R. R. has a grade of about 95 feet per mile, and this grade has proved so excessive for freight work that the company has constructed a cut-off (not shown on the map) which leaves the main line at *A*, nearly parallels the C. R. R. to *C*, and then running in a northeasterly direction again joins the main line beyond Wilkesbarre. The grade is thereby cut down to 65° feet per mile.

Rack railways and cable roads, although types of mountain railroad construction, will not be here considered.

6. Existing maps. The maps of the U. S. Geological Survey are exceedingly valuable as far as they have been completed. So far as topographical considerations are concerned, they almost dispense with the necessity for the reconnoissance and "first preliminary" surveys. Some of the State Survey maps will give practically the same information. County and township maps can often be used for considerable information as to the relative *horizontal* position of governing points, and even some approximate data regarding elevations may be obtained by a study of the streams. Of course such information will not dispense with surveys, but will assist in so planning them as to obtain the best information with the least work. When the relative horizontal positions of points are reliably indicated on a map, the reconnoissance may be reduced to the determination of the relative elevations of the governing points of the route.

7. Determination of relative elevations. A recent description of European methods includes spirit-leveling in the reconnoissance work. This may be due to the fact that, as indicated above, previous topographical surveys have rendered unnecessary the "exploratory" survey which is required in a new country, and that their reconnoissance really corresponds more nearly to our preliminary.

The perfection to which barometrical methods have been brought has rendered it possible to determine differences of elevation with sufficient accuracy for reconnoissance purposes by the combined use of a mercurial and an aneroid barometer. The mercurial barometer should be kept at "headquarters," and readings should be taken on it at such frequent intervals that any fluctuation is noted, and throughout the period that observations with the aneroid are taken in the field. At each observation there should also be recorded the time, the reading of the attached thermometer, and the temperature of the external air. For uniformity, the mercurial readings should then be "reduced to 32° F." The form of notes for the mercurial barometer readings should be as follows:

Time.	Merc. Barom.	Attached Therm.	Reduction to 32° F.	External Therm.	Corrected reading.
7:00 A. M.	29.872	72°	— .117	73°	29.755
:15	.866	73.5	.121	75	.745
:30	.858	75	.125	76	.733
:45	.850	76	.127	77	.723

The corrections in column 4 are derived from Table XI by interpolation.

Before starting out, a reading of the aneroid should be taken at headquarters coincident with a reading of the mercurial. The difference is one value of the correction to the aneroid. As soon as the aneroid is brought back another comparison of readings should be made. Even though there has been considerable rise or fall of pressure in the interval, the *difference* in readings (the correction) should be substantially the same provided the aneroid is a good instrument. If the difference of elevation is excessive (as when climbing a high mountain) even the best aneroid will "lag" and not recover its normal reading for several hours, but this does not apply to such differences of elevation as are met with in railroad work. The best aneroids read directly to $\frac{1}{100}$ of an inch of mercury and may be estimated to $\frac{1}{1000}$ of an inch—which corresponds to about 0.9 foot difference of elevation. In the field there should be read, at each point whose elevation is desired, the aneroid, the time, and the temperature. These readings, corrected by the mean value of the correction between the aneroid and the mercurial, should then be combined with the reading of the mercurial (interpolated if necessary) for the times of the aneroid observations and the difference of elevation obtained. The field notes for the aneroid should be taken as shown in the first four columns of the tabular form. The "corrected aneroid" readings of column 5 are found by correcting the readings of column 3 by the mean difference between the mercurial and aneroid when compared at morning and night. Column 6 is a copy of the "corrected readings" from the office notes, interpolated when necessary for the proper time. Column 7 is similarly obtained. Col. 8 is obtained from cols. 4 and 5, and col. 9 from cols. 6 and 7, with the aid of Table XII. The correction for temperature (col. 11), which is generally small unless the difference of elevation is large, is obtained with the aid of Table XIII. The elevations in Table XII are elevations above an assumed datum plane, where under the given atmospheric conditions the mercurial reading would be 30". Of course the position of this assumed plane changes with varying atmospheric conditions and so the elevations are to be considered as *relative* and their difference taken. [See the author's "Problems in the Use and Adjustment of Engineering In-

(Left-hand page of Notes.)

Time.	Place.	Aneroid.	Therm.	Corr. Aner.	Corr. Merc.
7:00	Office	29.628	73°		29.755
7:10	do	29.662	72°	29.789	29.748
7:30	saddle-back	29.374	63°	29.501	29.733
7:50	river cross.	29.548	70°	29.675	29.720

struments," Prob. 22.] Important points should be observed more than once if possible. Such duplicate observations will be found to give surprisingly concordant results even when a general fluctuation of atmospheric pressure so modifies the tabulated readings that an agreement is not at first apparent. Variations of pressure produced by high winds, thunder-storms, etc., will generally vitiate possible accuracy by this method. By "headquarters" is meant any place whose elevation above any given datum is known and where the mercurial may be placed and observed while observations within a range of several miles are made with the aneroid. If necessary, the elevation of a new headquarters may be determined by the above method, but there should be if possible several independent observations whose accordance will give a fair idea of their accuracy.

The above method should be neither slighted nor used for more than it is worth. When properly used, the errors are compensating rather than cumulative. When used, for example, to determine that a pass *B* is 260 feet higher than a determined bridge crossing at *A* which is six miles distant, and that another pass *C* is 310 feet higher than *A* and is ten miles distant, the figures, even with all necessary allowances for inaccuracy, will give an engineer a good idea as to the choice of route especially as affected by ruling grade. There is no comparison between the time and labor involved in obtaining the above information by barometric and by spirit-leveling methods, and *for reconnaissance purposes* the added accuracy of the spirit-leveling method is hardly worth its cost.

8. Horizontal measurements, bearings, etc. When there is no map which may be depended on, or when only a skeleton map is obtainable, a rapid survey, sufficiently accurate for the purpose, may be made by using a pocket compass for bearings and a telemeter, odometer, or pedometer for distances. The telemeter [stadia] is more accurate, but it requires a definite clear

(Right-hand page of Notes.)

Temp. at headqu.	Approx. field read.	Approx. headq. read.	Diff.	Corr. for temp.	Diff. elev.
—	—	—	—	—	—
75°	192	230	— 38	—(+ 2)	— 40
76	457	244	+ 213	+ (+10)	+ 223
77	297	256	+ 41	+ (+ 2)	+ 43

sight from station to station, which may be difficult through a wooded country. The odometer, which records the revolutions of a wheel of known circumference, may be used even in rough and wooded country, and the results may be depended on to a small percentage. The pedometer (or pace-measurer) depends for its accuracy on the actual movement of the mechanism for each pace and on the uniformity of the pacing. Its results are necessarily rough and approximate, but it may be used to fill in some intermediate points in a large skeleton map. A hand-level is also useful in determining the relative elevation of various topographical features which may have some bearing on the proper location of the road.

9. Importance of a good reconnoissance. The foregoing instruments and methods should be considered only as aids in exercising an educated common sense, without which a proper location cannot be made. The reconnoissance survey should command the best talent and the greatest experience available. If the general route is properly chosen, a comparatively low order of engineering skill can fill in a location which will prove a paying railroad property; but if the general route is so chosen that the ruling grades are high and the business obtained is small and subject to competition, no amount of perfection in detailed alignment or roadbed construction can make the road a profitable investment.

PRELIMINARY SURVEYS.

10. Character of survey. A preliminary railroad survey is properly a topographical survey of a belt of country which has been selected during the reconnoissance and within which it is estimated that the located line will lie. The width of this belt will depend on the character of the country. When a railroad is to follow a river having very steep banks the choice of location is sometimes limited at places to a very few feet of width

and the belt to be surveyed may be correspondingly narrowed. In very flat country the desired width may be only limited by the ability to survey points with sufficient accuracy at a considerable distance from what may be called the "backbone line" of the survey.

11. Cross-section method. This is the only feasible method in a wooded country, and is employed by many for all kinds of country. The *backbone* line is surveyed either by observing magnetic bearings with a compass or by carrying forward

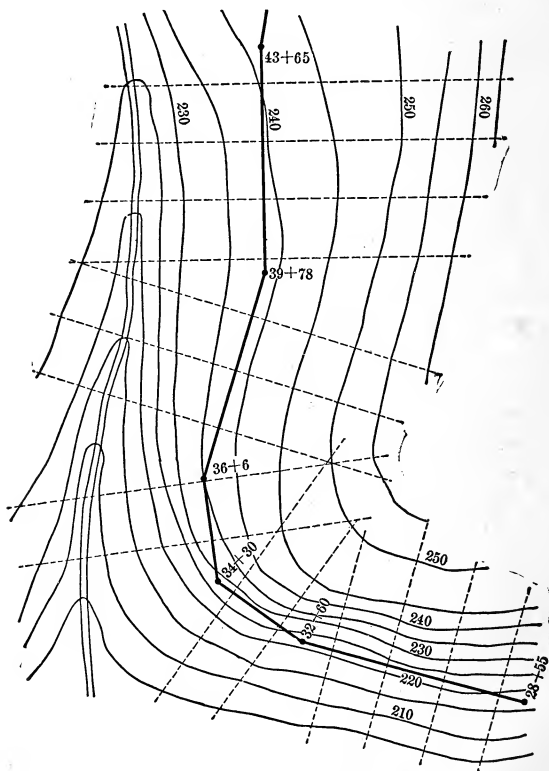


FIG. 4.

absolute azimuths with a transit. The compass method has the disadvantages of limited accuracy and the possibility of considerable local error owing to local attraction. On the other hand there are the advantages of greater simplicity, no necessity

for a back rodman, and the fact that the errors are purely local and not cumulative, and may be so limited, with care, that they will cause no vital error in the subsequent location survey. The transit method is essentially more accurate, but is liable to be more laborious and troublesome. If a large tree is encountered, either it must be cut down or a troublesome operation of offsetting must be used. If the compass is employed under these circumstances, it need only be set up on the far side of the tree and the former bearing produced. An error in reading a transit azimuth will be carried on throughout the survey. An error of only five minutes of arc will cause an offset of nearly eight feet in a mile. Large azimuth errors may, however, be avoided by immediately checking each new azimuth with a needle reading. It is advisable to obtain true azimuth at the beginning of the survey by an observation on the sun* or Polaris, and to check the azimuths every few miles by azimuth observations. Distances along the backbone line should be measured with a chain or steel tape and stakes set every 100 feet. When a course ends at a substation, as is usually the case, the remaining portion of the 100 feet should be measured along the next course. The level party should immediately obtain the elevations (to the nearest tenth of a foot) of all stations, and also of the lowest points of all streams crossed and even of dry gullies which would require culverts.

12. Cross-sectioning. It is usually desirable to obtain contours at five-foot intervals. This may readily be done by the use of a Locke level (which should be held on top of a simple five-foot stick), a tape, and a rod ten feet in length graduated to feet and tenths. The method of use may perhaps be best explained by an example. Let Fig. 5 represent a section *perpendicular* to the survey line—such a section as would be made by the dotted lines in Fig. 4. *C* represents the station point. Its elevation as determined by the level is, say, 158.3 above datum. When the Locke level on its five-foot rod is placed at *C*, the level has an elevation of 163.3. Therefore when a point is found (as at *a*) where the level will read 3.3 on the rod, that point has an elevation of 160.0 and its distance from the center gives the position of the 160-foot contour. Leaving the long rod at that point (*a*), carry the level to some point (*b*) such that the level will sight at the *top* of the rod. *b* is then on the 165-

* For detailed methods of such determinations, see the author's "Problems in the Use and Adjustment of Engineering Instruments," Problems 35 and 36.

foot contour, and the *horizontal* distance *ab* added to the horizontal distance *ac* gives the position of that contour from the center. The contours on the lower side are found similarly. The first rod reading will be 8.3, giving the 155-foot contour.

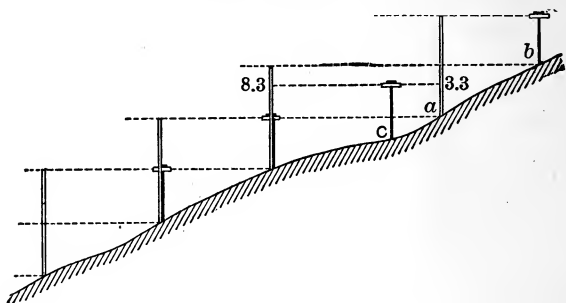


FIG. 5.

Plot the results in a note-book which is ruled in quarter-inch squares, using a scale of 100 feet per inch in both directions. Plot the work up the page; then when looking ahead along the line, the work is properly oriented. When a contour crosses

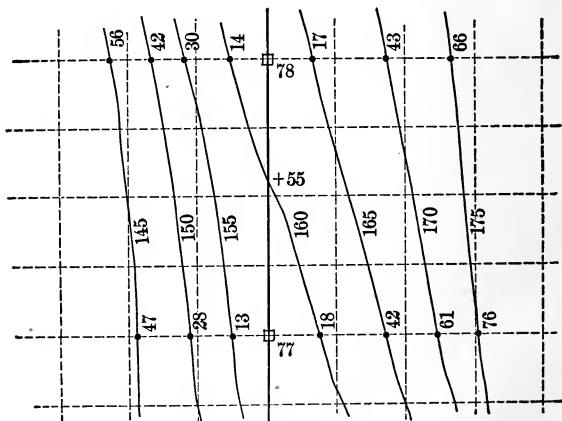


FIG. 6.

the survey line, the place of crossing may be similarly determined. If the ground flattens out so that five-foot contours are very far apart, the absolute elevations of points at even fifty-

foot distances from the center should be determined. The method is exceedingly rapid. Whatever error or inaccuracy occurs is confined in its effect to the one station where it occurs. The work being thus plotted in the field, unusually irregular topography may be plotted with greater certainty and no great error can occur without detection. It would even be possible by this method to detect a gross error that might have been made by the level party.

13. Stadia method. This method is best adapted to fairly open country where a "shot" to any desired point may be taken without clearing. The *backbone* survey line is the same as in the previous method except that each course is limited to the practicable length of a stadia sight. The distance between stations should be checked by foresight and backsight—also the vertical angle. Azimuths should be checked by the needle. Considering the vital importance of leveling on a railroad survey it might be considered desirable to run a line of levels over the stadia stations in order that the leveling may be as precise as possible; but when it is considered that a preliminary survey is a somewhat hasty survey of a route that *may* be abandoned, and that the errors of leveling by the stadia method (which are compensating) may be so minimized that no proposed route would be abandoned on account of such small error, and that the effect of such an error may be easily neutralized by a slight change in the location, it may be seen that excessive care in the leveling of the preliminary survey is hardly justifiable.

Since the students taking this work are assumed to be familiar with the methods of stadia topographical surveys, this part of the subject will not be further elaborated.

14. "First" and "Second" preliminary surveys. Some engineers advocate two preliminary surveys. When this is done, the first is a very rapid survey, made perhaps with a compass, and is only a better grade of reconnoissance. Its aim is to rapidly develop the facts which will decide for or against any proposed route, so that if a route is found to be unfavorable another more or less modified route may be adopted without having wasted considerable time in the survey of useless details. By this time the student should have grasped the fundamental idea that both the reconnoissance and preliminary surveys are not surveys of *lines* but of *areas*; that their aim is to survey only those topographical features which would have a deter-

mining influence on any railroad line which might be constructed through that particular territory, and that the value of a locating engineer is largely measured by his ability to recognize those determining influences with the least amount of work from his surveying corps. Frequently too little time is spent on the comparative study of preliminary lines. A line will be hastily decided on after very little study; it will then be surveyed with minute detail and estimates carefully worked up, and the claims of any other suggested route will then be handicapped, if not disregarded, owing to an unwillingness to discredit and throw away a large amount of detailed surveying. The cost of two or three extra preliminary surveys (*at critical sections* and not over the whole line) is utterly insignificant compared with the probable improvement in the "operating value" of a line located after such a comparative study of preliminary lines.

LOCATION SURVEYS.

15. "Paper location." When the preliminary survey has been plotted to a scale of 200 feet per inch and the contours drawn in, a study may be made for the location survey. Disregarding for the present the effect on location of transition curves, the alignment may be said to consist of straight lines (or "tangents") and circular curves. The "paper location" therefore consists in plotting on the preliminary map a succession of straight lines which are tangent to the circular curves connecting them. The determining points should first be considered. Such points are the termini of the road, the lowest practicable point over a summit, a river-crossing, etc. So far as is possible, having due regard to other considerations, the road should be a "surface" road, i.e., the cut and fill should be made as small as possible. The maximum permissible grade must also have been determined and duly considered. The method of location differs radically according as the lines joining the determining points have a very low grade or have a grade that approaches the maximum permissible. With very low natural grades it is only necessary to strike a proper balance between the requirements for easy alignment and the avoidance of excessive earthwork. When the grade between two determined points approaches the maximum, a study of the location may be begun by finding a strictly surface line which will connect those

points with a line at the given grade. For example, suppose the required grade is 1.6% and that the contours are drawn at 5-foot intervals. It will require 312 feet of 1.6% grade to rise 5 feet. Set a pair of dividers at 312 feet and step off this interval on successive contours. This line will in general be very irregular, but in an easy country it may lie fairly close to the proper location line, and even in difficult country such a surface line will assist greatly in selecting a suitable location. When the larger part of the line will evidently consist of tangents, the tangents should be first located and should then be connected by suitable curves. When the curves predominate, as they generally will in mountainous country, and particularly when the line is purposely lengthened in order to reduce the grade, the curves should be plotted first and the tangents may then be drawn connecting them. Considering the ease with which such lines may be drawn on the preliminary map, it is frequently advisable, after making such a paper location, to begin all over, draw a new line over some specially difficult section and compare results. Profiles of such lines may be readily drawn by noting their intersection with each contour crossed. Drawing on each profile the required grade line will furnish an approximate idea of the *comparative* amount of earthwork required. After deciding on the paper location, the length of each tangent, the central angle (see § 21), and the radius of each curve should be measured as accurately as possible. Since a slight error made in such measurements, taken from a map with a scale of 200 feet per inch, would by accumulation cause serious discrepancies between the plotted location and the location as afterward surveyed in the field, frequent tie lines and angles should be determined between the plotted location line and the preliminary line, and the location should be altered, as may prove necessary, by changing the length of a tangent or changing the central angle or radius of a curve, so that the agreement of the check-points will be sufficiently close. The errors of an inaccurate preliminary survey may thus be easily neutralized (see § 33). When the preliminary line has been properly run, its "backbone" line will lie very near the location line and will probably cross it at frequent intervals, thus rendering it easy to obtain short and numerous tie lines.

16. Surveying methods. A transit should be used for alignment, and only precise work is allowable. The transit stations

should be centered with tacks and should be tied to witness-stakes, which should be located outside of the range of the earth-work, so that they will neither be dug up nor covered up. All original property lines lying within the limits of the right of way should be surveyed with reference to the location line, so that the right-of-way agent may have a proper basis for settlement. When the property lines do not extend far outside of the required right of way they are frequently surveyed completely.

The leveler usually reads the target to the nearest thousandth of a foot on turning-points and bench-marks, but reads to the nearest tenth of a foot for the elevation of the ground at stations. Considering that $\frac{1}{1000}$ of a foot has an angular value of about

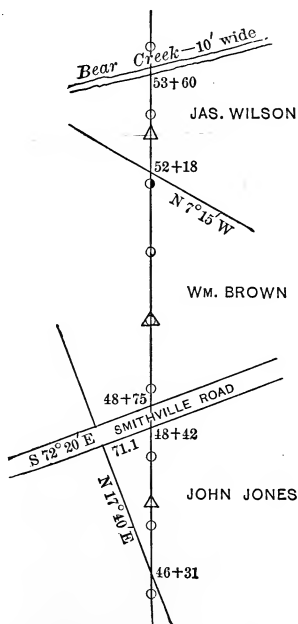
FORM OF NOTES.

[Left-hand page.]

Sta.	Align- ment.	Vernier.	Tangential Deflection.	Calculated Bearing.	Needle.
54					
⊙ 53 + 72.2	P.T.	9° 11'	18° 22'	N 54° 48' E	N 62° 15' E
52		7 57			
51		6 15			
⊙ 50	[3° 24' curve to right for 18° 22'; tang. dist., 272.5]	4 33			
49		2 51			
48		1 09			
⊙ + 32 47		0°			
46	P.C.			N 36° 26' E	N 44° 0' E

one second at a distance of 200 feet, and that one division of a level-bubble is usually about 30 seconds, it may be seen that it is a useless refinement to read to thousandths unless corresponding care is taken in the use of the level. The leveler should also locate his bench-marks outside of the range of earthwork. A knob of rock protruding from the ground affords an excellent mark. A large nail, driven in the roots of a tree, which is not to be disturbed, is also a good mark. These marks should be clearly described in the note-book. The leveler should obtain the elevation of the ground at all station-points; also at all sudden breaks in the profile line, determining also the distance of these breaks from the previous even station. This will in-

[Right-hand page.]



clude the position and elevation of all streams, and even dry gullies, which are crossed

Measurements should preferably be made with a steel tape, care being taken on steep ground to insure horizontal measurements. Stakes are set each 100 feet, and also at the beginning and end of all curves. Transit-points (sometimes called "plugs" or "hubs") should be driven flush with the ground, and a "witness-stake," having the "number" of the station, should be set three feet to the right. For example, the witness-stake might have on one side "137 + 69.92," and on the other side "P C 4° R," which would signify that the transit hub is 69.92 feet beyond station 137, or 13769.92 feet from the beginning of the line, and also that it is the "point of curve" of a "4° curve" which turns to the *right*.

Alignment. The alignment is evidently a part of the location survey, but, on account of the magnitude and importance of the subject, it will be treated in a separate chapter.

17. Form of Notes. Although the Form of Notes cannot be thoroughly understood until after curves are studied, it is here introduced as being the most convenient place. The right-hand page should have a sketch showing all roads, streams, and property lines crossed with the bearings of those lines. This should be drawn to a scale of 100 feet per inch—the quarter-inch squares which are usually ruled in note-books giving convenient 25-foot spaces. This sketch will always be more or less distorted on curves, since the center line is always shown as *straight* regardless of curves. The station points ("Sta." in first column, left-hand page) should be placed opposite to their sketched positions, which means that even stations will be recorded on every *fourth* line. This allows three intermediate lines for substations, which is ordinarily more than sufficient. The notes should read up the page, so that the sketch will be properly oriented when looking ahead along the line. The other columns on the left-hand page will be self-explanatory when the subject of curves is understood. If the "calculated bearings" are based on azimuthal observations, their agreement (or *constant* difference) with the needle readings will form a valuable check on the curve calculations and the instrumental work.

CHAPTER II.

ALIGNMENT.

IN this chapter the alignment of the *center line* only of a pair of rails is considered. When a railroad is crossing a summit in the grade line, although the horizontal projection of the alignment may be straight, the vertical projection will consist of two sloping lines joined by a curve. When a curve is on a grade, the center line is really a spiral, a curve of double curvature, although its horizontal projection is a circle. The center line therefore consists of straight lines and curves of single and double curvature. The simplest method of treating them is to consider their horizontal and vertical projections separately. In treating simple, compound, and transition curves, only the horizontal projections of those curves will be considered.

SIMPLE CURVES.

18. Designation of curves. A curve may be designated either by its radius or by the angle subtended by a chord of unit length. Such an angle is known as the "degree of curve" and is indicated by D . Since the curves that are practically used have very long radii, it is generally impracticable to make any use of the actual center, and the curve is located without reference to it. If AB in Fig. 4 represents a unit chord (C) of a curve of radius R , then by the above definition the angle AOB equals D . Then

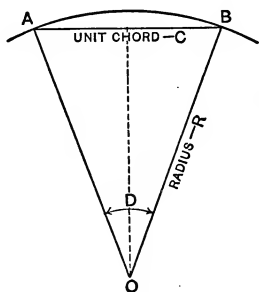


FIG. 7.

$$AO \sin \frac{1}{2}D = \frac{1}{2}AB = \frac{1}{2}C.$$

$$\therefore R = \frac{\frac{1}{2}C}{\sin \frac{1}{2}D}, \quad \dots \dots \dots (1)$$

or, by inversion,

$$\sin \frac{1}{2}D = \frac{C}{2R}. \quad (2)$$

The unit chord is variously taken throughout the world as 100 feet, 66 feet, and 20 meters. In the United States 100 feet is invariably used as the unit chord length, and throughout this work it will be so considered. Table I has been computed on this basis. It gives the radius, with its logarithm, of all curves from a $0^{\circ} 01'$ curve up to a 10° curve, varying by single minutes. The sharper curves, which are seldom used, are given with larger intervals.

An approximate value of R may be readily found from the following simple rule, which should be memorized:

$$R = \frac{5730}{D}.$$

Although such values are not mathematically correct, since R does not strictly vary inversely as D , yet the resulting value is within a tenth of one per cent for all commonly used values of R , and is sufficiently close for many purposes, as will be shown later.

19. Length of a subchord. Since it is impracticable to measure along a curved arc, curves are always measured by laying off 100-foot chord lengths.

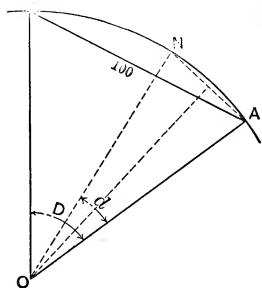


FIG. 8.

This means that the actual arc is always a little longer than the chord. It also means that a *subchord* (a chord shorter than the unit length) will be a little longer than the ratio of the angles subtended would call for. The truth of this may be seen without calculation by noting that two equal subchords, each subtending the angle $\frac{1}{2}D$, will evidently be slightly longer

than 50 feet each. If c be the length of a subchord subtending the angle d , then, as in Eq. 2,

$$\sin \frac{1}{2}d = \frac{c}{2R},$$

or, by inversion,

$$c = 2R \sin \frac{1}{2}d. \quad . \quad . \quad . \quad . \quad . \quad (3)$$

The *nominal length* of a subchord $= 100 \frac{d}{D}$ For example,

a *nominal* subchord of 40 feet will subtend an angle of $\frac{40}{100}$ of D° ; its *true length* will be slightly more than 40 feet, and may be computed by Eq. 3. The *difference* between the nominal and true lengths is maximum when the subchord is about 57 feet long, but with the low degrees of curvature ordinarily used the difference may be neglected. With a 10° curve and a nominal chord length of 60 feet, the true length is 60.049 feet. Very sharp curves should be laid off with 50-foot or even 25-foot chords (nominal length). In such cases especially the true lengths of these subchords should be computed and used instead of the nominal lengths.

20. Length of a curve. The length of a curve is always indicated by the quotient of $100L \div D$. If the quotient of $L \div D$ is a whole number, the length as thus indicated is the true length—*measured in 100-foot chord lengths*. If it is an odd number or if the curve begins and ends with a subchord (even though $L \div D$ is a whole number), theoretical accuracy requires that the *true* subchord lengths shall be used, although the difference may prove insignificant. The length of the arc (or the mean length of the two rails) is therefore always in excess of the length as given above. Ordinarily the amount of this excess is of no practical importance. It simply adds an insignificant amount to the length of rail required.

Example. Required the nominal and true lengths of a $3^\circ 45'$ curve having a central angle of $17^\circ 25'$. First reduce the degrees and minutes to decimals of a degree. $(100 \times 17^\circ 25') \div 3^\circ 45' = 1741.667 \div 3.75 = 464.444$. The curve has four 100-foot chords and a nominal chord of 64.444. The true chord should be 64.451. The actual arc is

$$17^\circ.4167 \times \frac{\pi}{180^\circ} \times R = 464.527$$

The excess is therefore $464.527 - 464.451 = 0.076$ foot.

21. Elements of a curve. Considering the line as running from *A* toward *B*, the beginning of the curve, at *A*, is called the *point of curve (PC)*. The other end of the curve, at *B*, is

called the *point of tangency* (PT). The intersection of the tangents is called the *vertex* (V).

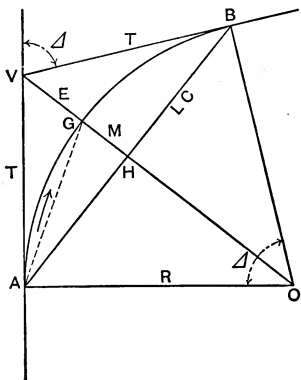


FIG. 9.

The angle made by the tangents at V, which equals the angle made by the radii to the extremities of the curve, is called the *central angle* (Δ). AV and BV, the two equal tangents from the vertex to the PC and PT, are called the *tangent distances* (T). The chord AB is called the *long chord* (LC). The intercept HG from the middle of the long chord to the middle of the arc is called the *middle ordinate* (M).

That part of the secant GV from the middle of the arc to the vertex is called the *external distance* (E). From the figure it is very easy to derive the following frequently used relations:

$$T = R \tan \frac{1}{2} \Delta \quad . \quad . \quad . \quad . \quad . \quad . \quad (4)$$

$$LC = 2R \sin \frac{1}{2} \Delta \quad . \quad . \quad . \quad . \quad . \quad . \quad (5)$$

$$M = R \text{ vers } \frac{1}{2} \Delta \quad . \quad . \quad . \quad . \quad . \quad . \quad (6)$$

$$E = R \text{ exsec } \frac{1}{2} \Delta \quad . \quad . \quad . \quad . \quad . \quad . \quad (7)$$

22. Relation between T, E, and Δ . Join A and G in Fig. 9. The angle $VAG = \frac{1}{4} \Delta$, since it is measured by one half of the arc AG between the secant and tangent. $AGO = 90^\circ - \frac{1}{4} \Delta$.

$$AV : VG :: \sin AGV : \sin VAG;$$

$$\sin AGV = \sin AGO = \cos \frac{1}{4} \Delta;$$

$$T : E :: \cos \frac{1}{4} \Delta : \sin \frac{1}{4} \Delta;$$

$$T = E \cot \frac{1}{4} \Delta. \quad . \quad . \quad . \quad . \quad . \quad . \quad (8)$$

The same relation may be obtained by dividing Eq. 4 by Eq. 7, since $\tan a \div \text{exsec } a = \cot \frac{1}{2} a$.

23. Elements of a 1° curve. From Eqs. 1 to 8 it is seen that the elements of a curve vary directly as R. It is also seen to be very nearly true that R varies inversely as D. If the elements of a 1° curve for various central angles are calculated and tabulated, the elements of a curve of D° curvature may be approximately found by dividing by D the corresponding elements of a 1° curve having the same central angle. For small

central angles and low degrees of curvature the errors involved by the approximation are insignificant, and even for larger angles the errors are so small that *for many purposes* they may be disregarded.

In Table II is given the value of the tangent distances, external distances, and long chords for a 1° curve for various central angles. The student should familiarize himself with the degree of approximation involved by solving a large number of cases under various conditions by the exact and by the approximate methods, in order that he may know when the approximate method is sufficiently exact for the intended purpose. The approximate method also gives a ready check on the exact method.

24. Exercises. (a) What is the tangent distance of a $4^\circ 20'$ curve having a central angle of $18^\circ 24'$?

(b) Given a $3^\circ 30'$ curve and a central angle of $16^\circ 20'$, how far will the curve pass from the vertex? [Use Eq. 7.]

(c) An 18° curve is to be laid off using 25-foot (nominal) chord lengths. What is the true length of the subchords?

(d) Given two tangents making a central angle of $15^\circ 24'$. It is desired to connect these tangents by a curve which shall pass 16.2 feet from their intersection. How far down the tangent will the curve begin and what will be its radius? (Use Eq. 8 and then use Eq. 4 inverted.)

25. Curve location by deflections. The angle between a secant and a tangent (or between two secants intersecting on an arc) is measured by one half of the intercepted arc. Beginning at the *PC* (*A* in Fig. 10), if the first chord is to be a full chord we may deflect an angle $V A a$ ($=\frac{1}{2}D$), and the point *a*, which is 100 feet from *A*, is a point on the curve. For the next station, *b*, deflect an *additional* angle $b A a$ ($=\frac{1}{2}D$) and, with one end of the tape at *a*, swing the other end until the 100-foot point is on the line *Ab*. The point *b* is then on the curve. If the final chord *cB* is a subchord, its *additional deflection* ($\frac{1}{2}d$) is something less than $\frac{1}{2}D$. The last deflection ($B A V$) is

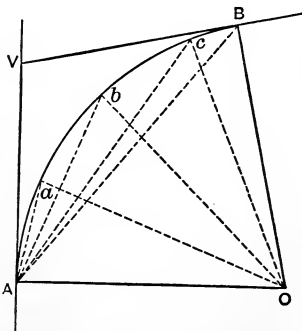


FIG. 10.

of course $\frac{1}{2}A$. It is particularly important, when a curve begins or ends with a subchord and the deflections are odd quantities, that the last additional deflection should be carefully computed and added to the previous deflection, to check the mathematical work by the agreement of this last computed deflection with $\frac{1}{2}A$.

Example. Given a $3^\circ 24'$ curve having a central angle of $18^\circ 22'$ and beginning at sta. 47+32, to compute the deflections. The nominal length of curve is $18^\circ 22' \div 3^\circ 24' = 18.367 \div 3.40 = 5.402$ stations or 540.2 feet. The curve therefore ends at sta. 52+72.2. The deflection for sta. 48 is $\frac{6.8}{100} \times \frac{1}{2}(3^\circ 24') = 0.68 \times 1^\circ.7 = 1^\circ.156 = 1^\circ 09'$ nearly. For each additional 100 feet it is $1^\circ 42'$ additional. The final additional deflection for the final subchord of 72.2 feet is

$$\frac{72.2}{100} \times \frac{1}{2}(3^\circ 24') = 1^\circ.2274 = 1^\circ 14' \text{ nearly.}$$

The deflections are

P. C . . . Sta. 47+32	0°
48	0° + 1° 09' = 1° 09'
49	1° 09' + 1° 42' = 2° 51'
50	2° 51' + 1° 42' = 4° 33'
51	4° 33' + 1° 42' = 6° 15'
52	6° 15' + 1° 42' = 7° 57'
P. T. 52+72.2	7° 57' + 1° 14' = 9° 11'

As a check $9^\circ 11' = \frac{1}{2}(18^\circ 22') = \frac{1}{2}A$. (See the Form of Notes in § 17.)

26. Instrumental work. It is generally impracticable to locate more than 500 to 600 feet of a curve from one station. Obstructions will sometimes require that the transit be moved up every 200 or 300 feet. There are two methods of setting off the angles when the transit has been moved up from the *PC*.

(a) The transit may be sighted at the previous transit station with a reading on the plates equal to the deflection angle from that station to the station occupied, but with the angle set off on the *other* side of 0° , so that when the telescope is turned to 0° it will sight along the tangent at the station occupied. Plunging the telescope, the forward stations may be set off by deflecting the proper deflections from the tangent at the station occupied

This is a very common method and, when the degree of curvature is an even number of degrees and when the transit is only set at even stations, there is but little objection to it. But the degree of curvature is sometimes an odd quantity, and the exigencies of difficult location frequently require that substations be occupied as transit stations. Method (a) will then require the recalculation of all deflections for each new station occupied. The mathematical work is largely increased and the probability of error is very greatly increased and not so easily detected. Method (b) is just as simple as method (a) even for the most simple cases, and for the more difficult cases just referred to the superiority is very great.

(b) Calculate the deflection for each station and substation throughout the curve as though the whole curve were to be located from the *PC*. The computations may thus be completed and *checked* (as above) before beginning the instrumental work. If it unexpectedly becomes necessary to introduce a substation at any point, its deflection from the *PC* may be readily interpolated. The stations actually set from the *PC* are located as usual.

RULE. When the transit is set on any forward station, backsight to ANY previous station with the plates set at the deflection angle for the station sighted at. Plunge the telescope and sight at any forward station with the deflection angle originally computed for that station. When the plates read the deflection angle for the station occupied, the telescope is sighting along the tangent at that station—which is the method of getting the forward tangent when occupying the *PT*. Even though the station occupied is an unexpected substation, when the instrument is properly oriented at that station, the angle reading for *any* station, forward or back, is that originally computed for it from the *PC*. In difficult work, where there are obstructions, a valuable check on the accuracy may be found by sighting backward at *any* visible station and noting whether

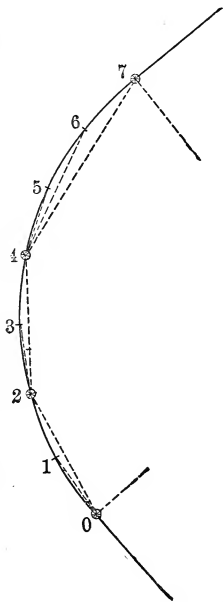


FIG. 11.

its deflection agrees with that originally computed. As a numerical illustration, assume a 4° curve, with 28° curvature, with stations 0, 2, 4, and 7 occupied. After setting stations 1 and 2, set up the transit at sta. 2 and backsight to sta. 0 with the deflection for sta. 0, which is 0° . The reading on sta. 1 is 2° ; when the reading is 4° the telescope is tangent to the curve, and when sighting at 3 and 4 the deflections will be 6° and 8° . Occupy 4; sight to 2 with a reading of 4° . When the reading is 8° the telescope is tangent to the curve and, by plunging the telescope, 5, 6, and 7 may be located with the originally computed deflections of 10° , 12° , and 14° . When occupying 7 a backsight may be taken to any visible station with the plates reading the deflection for that station; then when

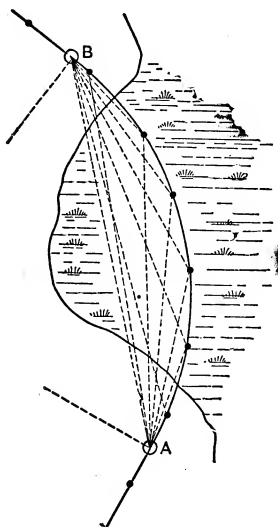


FIG. 12.

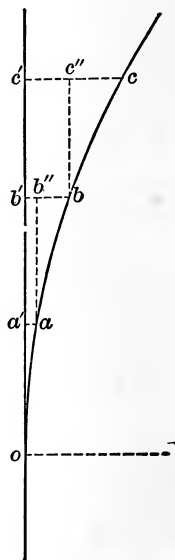


FIG. 13.

the plates read 14° the telescope will point along the forward tangent.

The location of curves by deflection angles is the normal method. A few other methods, to be described, should be considered as exceptional.

27. Curve location by two transits. A curve might be located more or less on a swamp where accurate chaining would be exceedingly difficult if not impossible. The long chord AB (Fig. 12) may be determined by triangulation or otherwise, and the elements of the curve computed, including (possibly) subchords at each end. The deflection from A and B to each point may be computed. A rodman may then be sent (by whatever means) to locate long stakes at points determined by the simultaneous sightings of the two transits.

28. Curve location by tangential offsets. When a curve is very flat and no transit is at hand the following method may be used (see Fig. 13): Produce the back tangent as far forward as necessary. Compute the ordinates Oa' , Ob' , Oc' , etc., and the abscissæ $a'a$, $b'b$, $c'c$, etc. If Oa is a full station (100 feet), then

$$\left. \begin{aligned} Oa' &= Oa' &= 100 \cos \frac{1}{2}D, \text{ also } = R \sin D; \\ Ob' &= Oa' + a'b' &= 100 \cos \frac{1}{2}D + 100 \cos \frac{3}{2}D, \\ & &\text{also } = R \sin 2D; \\ Oc' &= Oa' + a'b' + b'c' = 100(\cos \frac{1}{2}D + \cos \frac{3}{2}D + \cos \frac{5}{2}D), \\ & &\text{also } = R \sin 3D; \end{aligned} \right\} \quad (9)$$

etc.

$$\left. \begin{aligned} a'a &= &100 \sin \frac{1}{2}D, \text{ also } = R \text{ vers } D; \\ b'b &= a'a + b''b &= 100 \sin \frac{1}{2}D + 100 \sin \frac{3}{2}D, \\ & &\text{also } = R \text{ vers } 2D; \\ c'c &= b'b + c''c &= 100(\sin \frac{1}{2}D + \sin \frac{3}{2}D + \sin \frac{5}{2}D), \\ & &\text{also } = R \text{ vers } 3D; \end{aligned} \right\} \quad (10)$$

etc.

The functions $\frac{1}{2}D$, $\frac{3}{2}D$, etc., may be more conveniently used *without* logarithms, by adding the several *natural* trigonometrical functions and pointing off two decimal places. It may also be noted that Ob' (for example) is one half of the long chord for four stations; also that $b'b$ is the middle ordinate for four stations. If the engineer is provided with tables giving the long chords and middle ordinates for various degrees of curvature, these quantities may be taken (perhaps by interpolation) from such tables.

If the curve begins or ends at a substation, the angles and terms will be correspondingly altered. The modifications may

be readily deduced on the same principles as above, and should be worked out as an exercise by the student.

In Table II are given the long chords for a 1° curve for various values of Δ . Dividing the value as given by the degree of the curve, we have an approximate value which is amply close for low degrees of curvature, especially for laying out curves without a transit. For example, given a $4^\circ 30'$ curve, required the ordinate Oc' . This is evidently one half of a chord of six stations, with $\Delta = 27^\circ$. Dividing 2675.1 (which is the long chord of a 1° curve with $\Delta = 27^\circ$) by 4.5 we have 594.47; one half of this is the required ordinate, $Oc' = 297.23$. The exact value is 297.31, an excess of .08, or less than .03 of 1%. The true values are always slightly in excess of the value as computed from Table II.

Exercise. A $3^\circ 40'$ curve begins at sta. 18+70 and runs to sta. 23+60. Required the tangential offsets and their corresponding ordinates. The first ordinate $= 30 \cos \frac{1}{2}(\frac{30}{100} \times 3^\circ 40') = 30 \times .99995 = 29.9985$; the offset $= 30 \sin 0^\circ 33' = 30 \times .0096 = 0.288$. For the second full station (sta. 20) the ordinate $= \frac{1}{2}$ long chord for $\Delta = 2(1^\circ 06' + 3^\circ 40')$ with $D = 3^\circ 40'$. Dividing 476.12, from Table II, by $3\frac{2}{3}$, we have 129.85. Otherwise, by Eq. 9, the ordinate $= 30 \times \cos 0^\circ 33' + 100 \cos (1^\circ 06' + 1^\circ 50') = 30.00 + 99.87 = 129.87$. The offset for sta. 20 $= 30 \sin 0^\circ 33' + 100 \sin (1^\circ 06' + 1^\circ 50') = 0.288 + 5.12 = 5.41$. Work out similarly the ordinates and offsets for sta. 21, 22, 23, and 23+60.

29. Curve location by middle ordinates. Take first the simpler case when the curve begins at an even station. If we consider (in Fig. 14) the curve produced back to z , the chord $za = 2 \times 100 \cos \frac{1}{2}D$, $A'a = 100 \cos \frac{1}{2}D$, and $A'A = am = zn = 100 \sin \frac{1}{2}D$. Set off AA' perpendicular to the tangent and $A'a$ parallel to the tangent. $AA' = aa' = bb' = cc'$, etc. $= 100 \sin \frac{1}{2}D$. Set off aa' perpendicular to $a'A$. Produce Aa' until $a'b = A'a$, thus determining b . Succeeding points of the curve may thus be determined indefinitely.

Suppose the curve begins with a subchord. As before $ra = Am' = c' \cos \frac{1}{2}d'$, and $rA = am' = c' \sin \frac{1}{2}d'$. Also $sz = An' = c'' \cos \frac{1}{2}d''$, and $sA = zn' = c'' \sin \frac{1}{2}d''$, in which $(d' + d'') = D$. The points z and a being determined on the ground, aa' may be computed and set off as before and the curve continued in

full stations. A subchord at the end of the curve may be located by a similar process.

30. Curve location by offsets from the long chord. (Fig. 16.) Consider at once the general case in which the curve commences with a subchord (curvature, d'), continues with one or more full



FIG. 14.

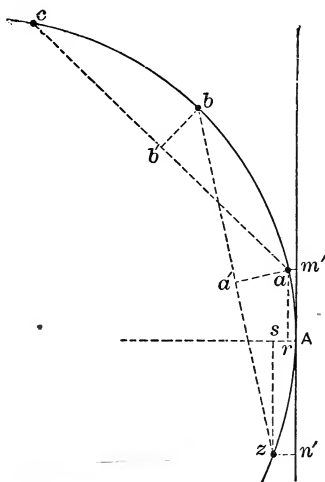


FIG. 15.

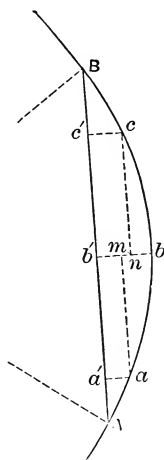


FIG. 16.

chords (curvature of each, D), and ends with a subchord with curvature d'' . The numerical work consists in computing first AB , then the various abscissæ and ordinates. $AB = 2R \sin \frac{1}{2}A$.

$$\left. \begin{aligned} Aa' &= Aa' &= c' \cos \frac{1}{2}(A - d'); \\ Ab' &= Aa' + a'b' &= c' \cos \frac{1}{2}(A - d') + 100 \cos \frac{1}{2}(A - 2d' - D); \\ Ac' &= Aa' + a'b' + b'c' &= c' \cos \frac{1}{2}(A - d') + 100 \cos \frac{1}{2}(A - 2d' - D) \\ & &+ 100 \cos \frac{1}{2}(A - 2d'' - D); \end{aligned} \right\} \quad (11)$$

also

$$= AB - Bc' = 2R \sin \frac{1}{2}A - c'' \cos \frac{1}{2}(A - d'').$$

$$\left. \begin{aligned} a'a &= a'a &= c' \sin \frac{1}{2}(A - d'); \\ b'b &= a'a + mb &= c' \sin \frac{1}{2}(A - d') + 100 \sin \frac{1}{2}(A - 2d' - D); \\ c'c &= b'b - nb &= c' \sin \frac{1}{2}(A - d') + 100 \sin \frac{1}{2}(A - 2d' - D) \\ & &- 100 \sin \frac{1}{2}(A - 2d'' - D); \end{aligned} \right\} \quad (12)$$

also

$$= c'' \sin \frac{1}{2}(A - d'').$$

The above formulæ are considerably simplified when the

curve begins and ends at even stations. When the curve is very long a regular law becomes very apparent in the formation of all terms between the first and last. There are too few terms in the above equations to show the law.

31. Use and value of the above methods. The chief value of the above methods lies in the possibility of doing the work without a transit. The same principles are sometimes employed, even when a transit is used, when obstacles prevent the use of the normal method (see § 32, c). If the terminal tangents have already been accurately determined, these methods are useful to locate points of the curve when rigid accuracy is not essential. Track foremen frequently use such methods to lay out unimportant sidings, especially when the engineer and his transit are not at hand. Location by tangential offsets (or by offsets from the long chord) is to be preferred when the curve is flat (i.e., has a small central angle Δ) and there is no obstruction along the tangent, or long chord. Location by middle ordinates may be employed regardless of the length of the curve, and in cases when both the tangents and the long chord are obstructed. The above methods are but samples of a large number of similar methods which have been devised. The choice of the particular method to be adopted must be determined by the local conditions.

32. Obstacles to location. In this section will be given only a few of the principles involved in this class of problems, with illustrations. The engineer must decide, in each case, which is the best method to use. It is frequently advisable to devise a special solution for some particular case.

a. When the vertex is inaccessible. As shown in § 26, it is not absolutely essential that the vertex of a curve should be located on the ground. But it is very evident that the angle between the terminal tangents is determined with far less probable error if it is measured by a single measurement at the vertex rather than as the result of numerous angle measurements along the curve, involving several positions of the transit and comparatively short sights. Some-

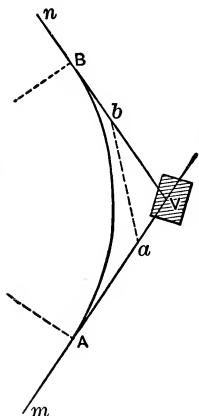


FIG. 17.

positions of the transit and comparatively short sights. Some-

times the location of the tangents is already determined on the ground (as by bn and am , Fig. 17), and it is required to join the tangents by a curve of given radius. *Method.* Measure ab and the angles Vba and baV . A is the sum of these angles. The distances bV and aV are computable from the above data. Given A and R , the tangent distances are computable, and then Bb and aA are found by subtracting bV and aV from the tangent distances. The curve may then be run from A , and the work may be checked by noting whether the curve as run ends at B —previously located from b .

Example. Assume $ab=546.82$; angle $a=15^\circ 18'$; angle $b=18^\circ 22'$; $D=3^\circ 40'$; required aA and bB .

$$A=15^\circ 18' + 18^\circ 22' = 33^\circ 40'$$

Eq. (4)	$R \quad (3^\circ 40') \dots\dots\dots$	3.19392
	$\tan \frac{1}{2}A = \tan 16^\circ 50' \dots\dots\dots$	9.48080
	$T=472.85 \dots\dots\dots$	<u>2.67472</u>
$aV=ab \frac{\sin 18^\circ 22'}{\sin 33^\circ 40'}$	$ab \dots\dots\dots$	2.73784
	$\log \sin 18^\circ 22' \dots\dots\dots$	9.49844
	$\text{co-log } \sin 33^\circ 40' \dots\dots\dots$	0.25621
	$aV=310.81 \dots\dots\dots$	<u>2.49250</u>
	$AV=472.85$	
	$aA=162.04$	
$bV=ab \frac{\sin 15^\circ 18'}{\sin 33^\circ 40'}$	$ab \dots\dots\dots$	2.73784
	$\log \sin 15^\circ 18' \dots\dots\dots$	9.42139
	$\text{co-log } \sin 33^\circ 40' \dots\dots\dots$	0.25621
	$bV=260.29 \dots\dots\dots$	<u>2.41545</u>
	$BV=472.85$	
	$bB=212.56$	

b. When the point of curve (or point of tangency) is inaccessible. At some distance (As , Fig. 18) an unobstructed line pn may be run parallel with AV . $nv=py=As=R \text{ vers } a$.

$$\therefore \text{vers } a = As \div R.$$

$$ns=ps=R \sin a.$$

At y , which is at a distance ps back from the *computed* position of A , make an offset sA to p . Run pn parallel to the tangent. A tangent to the curve at n makes an angle of α with np . From n the curve is run in as usual.

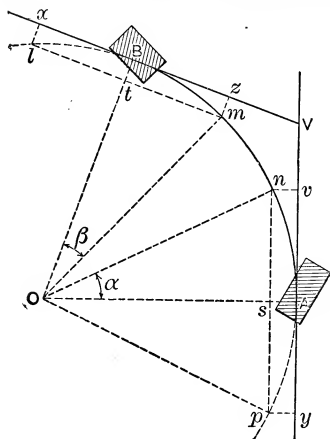


FIG. 18.

a chord may be run. α may equal *any* angle, but it is preferable that α should be a multiple of D , the degree of curve, and that the points m and n should be on even stations. $mn = 2R \sin \frac{1}{2}\alpha$. A point s may be located by an offset ks from the chord mn by a similar method to that outlined in § 30.

The device of introducing the dotted curve mn having the same radius of curvature as the other, although neither necessary nor advisable in the case shown in Fig. 19, is sometimes the best method of surveying around an obstacle. The offset from any point on the dotted curve to the corresponding point on the true curve is twice the "ordinate to the long chord," as computed in § 30.

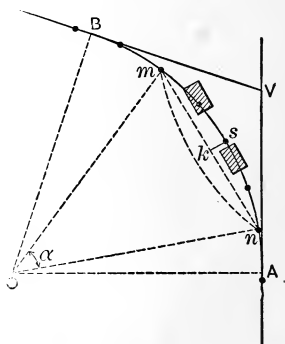


FIG. 19.

33. Modifications of location. The following methods may be used in allowing for the discrepancies between the "paper location" based on a more or less rough preliminary survey and the more accurate instrumental location. (See § 15.) They are

also frequently used in locating new parallel tracks and modifying old tracks.

a. To move the forward tangent parallel to itself a distance x , the point of curve (A) remaining fixed. (Fig. 20.)

$$V'h = B'r = x'.$$

$$VV' = \frac{V'h}{\sin hVV'} = \frac{x}{\sin A} \cdot \cdot \cdot \cdot \cdot (13)$$

$$AV' = AV + VV'.$$

The triangle BmB' is isosceles and $Bm = B'm$.

$$R' - R = O'O = mB = \frac{B'r}{\text{vers } B'mB} = \frac{x'}{\text{vers } A}.$$

$$\therefore R' = R + \frac{x'}{\text{vers } A} \cdot \cdot \cdot \cdot \cdot (14)$$

The solution is very similar in case the tangent is moved inward to $V''B''$. Note that this method necessarily changes the

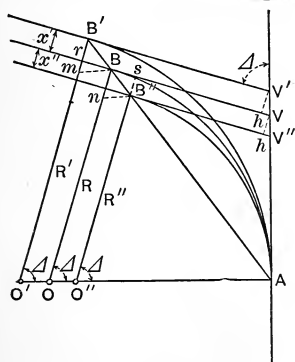


FIG. 20.

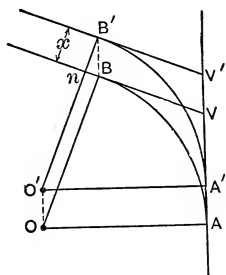


FIG. 21.

radius. If the radius is not to be changed, the point of curve must be altered as follows:

b. To move the forward tangent parallel to itself a distance x , the radius being unchanged. (Fig. 21.) In this case the whole

curve is moved bodily a distance $OO' = AA' = VV' = BB'$, and moved parallel to the first tangent AV

$$BB' = \frac{B'n}{\sin nBB'} = \frac{x}{\sin \Delta} = AA'. \quad . . . (15)$$

c. To change the direction of the forward tangent at the point of tangency. (Fig. 22.) This problem involves a change (α) in the central angle and also requires a new radius. An error in the determination of the central angle furnishes an occasion for its use.

R, Δ, α, AV , and BV are known.

$$\Delta' = \Delta - \alpha.$$

$$Bs = R \text{ vers } \Delta. \quad Bs = R' \text{ vers } \Delta'.$$

$$\therefore R' = R \frac{\text{vers } \Delta}{\text{vers } (\Delta - \alpha)}. \quad . . (16)$$

$$As = R \sin \Delta. \quad A's = R' \sin \Delta'.$$

$$\therefore AA' = A's - As = R' \sin \Delta' - R \sin \Delta. \quad . . (17)$$

The above solutions are given to illustrate a large class of problems which are constantly arising. All of the ordinary problems can be solved by the application of elementary geometry and trigonometry.

34. Limitations in location. It may be required to run a curve that shall join two given tangents and also pass through a given point. The point (P , Fig. 23) is assumed to be determined by its distance (VP) from the vertex and by the angle $AVP = \beta$.

It is required to determine the radius (R) and the tangent distance (AV). Δ is known.

$$\begin{aligned} PVG &= \frac{1}{2}(180^\circ - \Delta) - \beta \\ &= 90^\circ - (\frac{1}{2}\Delta + \beta). \end{aligned}$$

$$\begin{aligned} PP' &= 2VP \sin PVG \\ &= 2VP \cos (\frac{1}{2}\Delta + \beta). \end{aligned}$$

$$PSV = \frac{1}{2}\Delta.$$

$$\therefore SP = VP \frac{\sin \beta}{\sin \frac{1}{2}\Delta}.$$

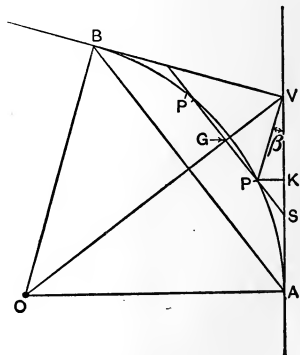


FIG. 23.

$$\begin{aligned}
 AS &= \sqrt{SP \times SP'} = \sqrt{SP(SP + PP')} \\
 &= \sqrt{VP \frac{\sin \beta}{\sin \frac{1}{2}A} \left[VP \frac{\sin \beta}{\sin \frac{1}{2}A} + 2VP \cos (\frac{1}{2}A + \beta) \right]} \\
 &= VP \sqrt{\frac{\sin^2 \beta}{\sin^2 \frac{1}{2}A} + \frac{2 \sin \beta \cos (\frac{1}{2}A + \beta)}{\sin \frac{1}{2}A}}. \\
 SV &= VP \frac{\sin (\frac{1}{2}A + \beta)}{\sin \frac{1}{2}A}. \\
 AV &= AS + SV \\
 &= \frac{VP}{\sin \frac{1}{2}A} [\sin (\frac{1}{2}A + \beta) + \sqrt{\sin^2 \beta + 2 \sin \beta \sin \frac{1}{2}A \cos (\frac{1}{2}A + \beta)}]. \quad (18) \\
 R &= AV \cot \frac{1}{2}A.
 \end{aligned}$$

In the special case in which P is on the median line OV , $\beta = 90^\circ - \frac{1}{2}A$, and $(\frac{1}{2}A + \beta) = 90^\circ$. Eq. 18 then reduces to

$$AV = \frac{VP}{\sin \frac{1}{2}A} (1 + \cos \frac{1}{2}A) = VP \cot \frac{1}{4}A,$$

as might have been immediately derived from Eq. 8.

In case the point P is given by the offset PK and by the distance VK , the triangle PKV may be readily solved, giving the distance VP and the angle β , and the remainder of the solution will be as above.

35. Determination of the curvature of existing track. (a) *Using a transit.* Set up the transit at any point in the center of the track. Measure in each direction 100 feet to points also in the center of the track. Sight on one point with the plates at 0° . Plunge the telescope and sight at the other point. The angle between the chords equals the degree of curvature.

(b) *Using a tape and string.* Stretch a string (say 50 feet long) between two points on the inside of the head of the outer rail. Measure the ordinate (x) between the *middle* of the string and the head of the rail. Then

$$R = \frac{\text{chord}^2}{8x} \text{ (very nearly). } \quad . \quad . \quad . \quad . \quad (19)$$

For, in Fig. 24, since the triangles AOE and ADC are similar,

$AO : AE :: AD : DC$ or $R = \frac{1}{2} \overline{AD}^2 \div x$. When, as is usual, the arc is very short compared with the radius, $AD = \frac{1}{2} AB$, very nearly. Making this substitution we have Eq. 19. With a chord of 50 feet and a 10° curve, the resulting difference in x is .0025 of an inch—far within the possible accuracy of such a method. The above method gives the radius of the inner head of the outer rail.

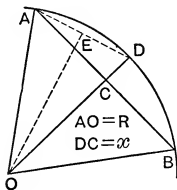


FIG. 24.

It should be diminished by $\frac{1}{2}g$ for the radius of the center of the track. With easy curvature, however, this will not affect the result by more than one or two tenths of one per cent.

The inversion of this formula gives the required middle ordinate for a rail on a given curve. For example, the middle ordinate of a 30-foot rail, bent for a 6° curve, is

$$x = 900 \div (8 \times 955) = .118 \text{ foot} = 1.4 \text{ inches.}$$

Another much used rule is to require the foreman to have a string, knotted at the center, of such length that the middle ordinate, measured in inches, equals the degree of curve. To find that length, substitute (in Eq. 19) $5730 \div D$ for R and $D \div 12$ for x . Solving for *chord*, we obtain *chord* = 61.8 feet. The rule is not theoretically exact, but, considering the uncertain stretching of the string, the error is insignificant. In fact, the distance usually given is 62 feet, which is close enough for all purposes for which such a method should be used.

36. Problems. A systematic method of setting down the solution of a problem simplifies the work. Logarithms should always be used, and *all* the work should be so set down that a revision of the work to find a supposed error may be readily done. The value of such systematic work will become more apparent as the problems become more complicated. The two solutions given below will illustrate such work.

a. Given a 3° curve beginning at Sta. 27+60 and running to Sta. 32+45. Compute the ordinates and offsets used in locating the curve by tangential offsets.

b. With the same data as above, compute the distances to locate the curve by offsets from the long chord.

c. Assume that in Fig. 17 *ab* is measured as 217.6 feet, the

angle $abV=17^{\circ} 42'$, and the angle $baV=21^{\circ} 14'$. Join the tangents by a $4^{\circ} 30'$ curve. Determine bB and aA .

d. Assume that in a case similar to Fig. 18 it was noted that a distance (As) equal to 12 feet would clear the building. Assume that $A=38^{\circ} 20'$ and that $D=4^{\circ} 40'$. Required the value of a and the position of n . *Solution:*

vers $a=As \div R$	$As=12$	$\log = 1.07918$
	R (for $4^{\circ} 40'$ curve)	$\log = 3.0892\bar{3}$
	<u>$a=8^{\circ} 01'$</u>	<u>$\log \text{ vers } a = 7.9899\bar{4}$</u>
$ns=R \sin a$		$\log \sin a = 9.1444\bar{5}$
		$\log R = 3.0892\bar{3}$
	<u>$ns=171.27$</u>	<u>$\log = 2.23369$</u>

e. Assume that the forward tangent of a $3^{\circ} 20'$ curve having a central angle of $16^{\circ} 50'$ must be moved 3.62 feet *inward*, without altering the *P.C.* Required the change in radius.

f. Given two tangents making an angle of $36^{\circ} 18'$. It is required to pass a curve through a point 93.2 feet from the vertex, the line from the vertex to the point making an angle of $42^{\circ} 21'$ with the tangent. Required the radius and tangent distance. *Solution:* Applying Eq. 18, we have

2	$\log = 0.30103$
$\beta=42^{\circ} 21'$	$\log \sin = 9.82844$
$\frac{1}{2}A=18^{\circ} 09'$	$\log \sin = 9.4934\bar{6}$
$(\frac{1}{2}A + \beta)=60^{\circ} 30'$	$\log \cos = 9.69234$
$.20667$	<u>$9.3152\bar{7}$</u>
$\log \sin^2 \beta = 9.65688 \dots .45382$	
<u>$2 \mid 9.81987 \dots .66049$</u>	
$9.9099\bar{3} \dots .81271$	
$\text{nat. sin } 60^{\circ} 30' \dots .870\bar{3}$	
$1.683\bar{0} \dots \dots \dots$	<u>$\log = 0.22610$</u>
$VP=93.2 \dots \dots \dots$	<u>$\log = 1.9694\bar{1}$</u>
	$2.1955\bar{1}$
	$\log \sin \frac{1}{2}A = 9.4934\bar{6}$
<u>$\text{Tang. dist. } AV=503.36 \dots \dots \dots$</u>	<u>$\log = 2.70205$</u>
	$\log \cot \frac{1}{2}A = 10.48437$
$R=1536.1 \dots \dots \dots$	<u>$\log = 3.18642$</u>
<u>$D=3^{\circ} 44'$</u>	

COMPOUND CURVES.

37. Nature and use. Compound curves are formed by a succession of two or more simple curves of different curvature. The curves must have a common tangent at the point of compound curvature (*P.C.C.*). In mountainous regions there is frequently a necessity for compound curves having several changes of curvature. Such curves may be located separately as a succession of simple curves, but a combination of two simple curves has special properties which are worth investigating and utilizing. In the following demonstrations R_2 always represents the *longer* radius and R_1 the *shorter*, no matter which succeeds the other. T_1 is the tangent adjacent to the curve of shorter radius (R_1), and is invariably the shorter tangent. Δ_1 is the central angle of the curve of radius R_1 , but it may be greater or less than Δ_2 .

38. Mutual relations of the parts of a compound curve having two branches. In Fig. 25, AC and CB are the two branches of

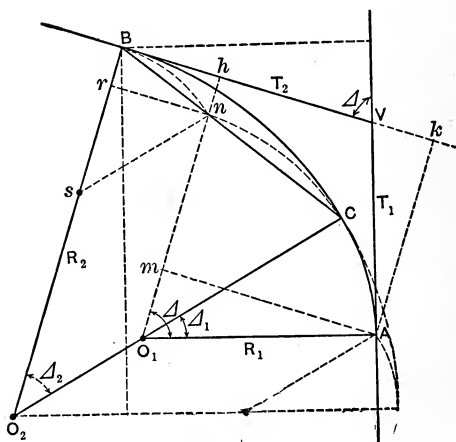


FIG. 25.

the compound curve having radii of R_1 and R_2 and central angles of Δ_1 and Δ_2 . Produce the arc AC to n so that $AO_1n = \Delta$. The chord Cn produced *must* intersect B . The line ns , parallel to CO_2 , will intersect BO_2 so that $Bs = sn = O_2O_1 = R_2 - R_1$. Draw Am perpendicular to O_1n . It will be parallel to hk .

$$Br = sn \text{ vers } Bsn = (R_2 - R_1) \text{ vers } \Delta_2;$$

$$mn = AO_1 \text{ vers } AO_1n = R_1 \text{ vers } \Delta;$$

$$Ak = AV \sin AVk = T_1 \sin \Delta;$$

$$Ak = hm = mn + nh = mn + Br.$$

$$\therefore T_1 \sin \Delta = R_1 \text{ vers } \Delta + (R_2 - R_1) \text{ vers } \Delta_2. \quad (20)$$

Similarly it may be shown that

$$T_2 \sin \Delta = R_2 \text{ vers } \Delta - (R_2 - R_1) \text{ vers } \Delta_1. \quad (21)$$

The mutual relations of the elements of compound curves may be solved by these two equations. For example, assume the tangents as fixed (Δ therefore known) and that a curve of given radius R_1 shall start from a given point at a distance T_1 from the vertex, and that the curve shall continue through a given angle Δ_1 . Required the other parts of the curve. From Eq. 20 we have

$$R_2 - R_1 = \frac{T_1 \sin \Delta - R_1 \text{ vers } \Delta}{\text{vers } \Delta_2}.$$

$$\therefore R_2 = R_1 + \frac{T_1 \sin \Delta - R_1 \text{ vers } \Delta}{\text{vers } (\Delta - \Delta_1)}. \quad (22)$$

T_2 may then be obtained from Eq. 21.

As another problem, given the location of the two tangents, with the two tangent distances (thereby locating the PC and PT), and the central angle of each curve; required the two radii. Solving Eq. 20 for R_1 , we have

$$R_1 = \frac{T_1 \sin \Delta - R_2 \text{ vers } \Delta_2}{\text{vers } \Delta - \text{vers } \Delta_2}.$$

Similarly from Eq. 21 we may derive

$$R_1 = \frac{T_2 \sin \Delta - R_2 (\text{vers } \Delta - \text{vers } \Delta_1)}{\text{vers } \Delta_1}.$$

Equating these, reducing, and solving for R_2 , we have

$$R_2 = \frac{T_1 \sin \Delta \text{ vers } \Delta_1 - T_2 \sin \Delta (\text{vers } \Delta - \text{vers } \Delta_2)}{\text{vers } \Delta_2 \text{ vers } \Delta_1 - (\text{vers } \Delta - \text{vers } \Delta_1)(\text{vers } \Delta - \text{vers } \Delta_2)}. \quad (23)$$

Although the various elements may be chosen as above with considerable freedom, there are limitations. For example, in Eq. 22, since R_2 is always greater than R_1 , the term to be added to R_1 must be essentially positive—i.e., $T_1 \sin \Delta$ must be greater than $R_1 \text{ vers } \Delta$. This means that $T_1 > R_1 \frac{\text{vers } \Delta}{\sin \Delta}$, or that

$T_1 > R_1 \tan \frac{1}{2} \Delta$, or that T_1 is greater than the corresponding tangent on a simple curve. Similarly it may be shown that T_2 is less than $R_2 \tan \frac{1}{2} \Delta$ or less than the corresponding tangent on a simple curve. Nevertheless T_2 is always greater than T_1 . In the limiting case when $R_2 = R_1$, $T_2 = T_1$, and $\Delta_2 = \Delta_1$.

39. Modifications of location. Some of these modifications may be solved by the methods used for simple curves. For example:

a. It is desired to move the tangent VB , Fig. 26, parallel to itself to $V'B'$. Run a new curve from the *P.C.C.* which shall reach the new tangent at B' , where the chord of the old curve

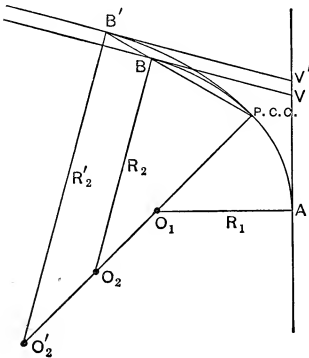


FIG. 26.

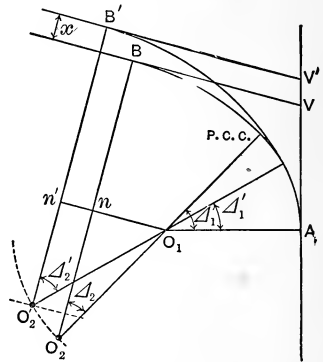


FIG. 27.

intersects the new tangent. The solution is almost identical with that in § 33, a.

b. Assume that it is desired to change the forward tangent (as above) but to retain the same radius. In Fig. 27

$$(R_2 - R_1) \cos \Delta_2 = O_2 n;$$

$$(R_2 - R_1) \cos \Delta_2' = O_2' n'.$$

$$x = O_2 n - O_2' n' = (R_2 - R_1)(\cos \Delta_2 - \cos \Delta_2').$$

$$\cos \Delta_2' = \cos \Delta_2 - \frac{x}{R_2 - R_1} \cdot \cdot \cdot \cdot \cdot (24)$$

The *P.C.C.* is moved backward along the sharper curve an angular distance of $\Delta_2' - \Delta_2 = \Delta_1 - \Delta_1'$.

In case the tangent is moved inward rather than outward, the solution will apply by transposing Δ_2 and Δ_2' . Then we shall have

$$\cos \Delta_2' = \cos \Delta_2 + \frac{x}{R_2 - R_1} \cdot \cdot \cdot \cdot \cdot (25)$$

The *P.C.C.* is then moved *forward*.

c. Assume the same case as (b) except that the larger radius comes first and that the tangent adjacent to the smaller radius is moved. In Fig. 28

$$(R_2 - R_1) \cos \Delta_1 = O_1 n;$$

$$(R_2 - R_1) \cos \Delta_1' = O_1' n'.$$

$$x = O_1' n' - O_1 n$$

$$= (R_2 - R_1)(\cos \Delta_1' - \cos \Delta_1).$$

$$\cos \Delta_1' = \cos \Delta_1 + \frac{x}{R_2 - R_1} \quad (26)$$

The *P.C.C.* is moved *forward* along the easier curve an angular distance of $\Delta_1' - \Delta_1 = \Delta_2 - \Delta_2'$.

In case the tangent is moved *inward*, transpose as before and we have

$$\cos \Delta_1' = \cos \Delta_1 - \frac{x}{R_2 - R_1} \quad (27)$$

The *P.C.C.* is moved *backward*.

d. Assume that the radius of one curve is to be altered without changing either tangent. Assume conditions as in Fig. 29.

For the diagrammatic solution assume that R_2 is to be increased by $O_2 S$. Then, since R_2' must pass through O_1 and extend beyond O_1 a distance $O_1 S$, the locus of the new center must lie on the arc drawn about O_1 as center and with $O_1 S$ as radius. The locus of O_2' is also given by a line $O_2' p$ parallel to BV and at a distance of R_2' (equal to $S \dots P.C.C.$) from it. The new center is therefore at the intersection O_2' . An arc with radius R_2' will therefore be tangent at B' and tangent to the old curve *produced* at NEW *P.C.C.* Draw $O_1 n'$ perpendicular to $O_2 B$.

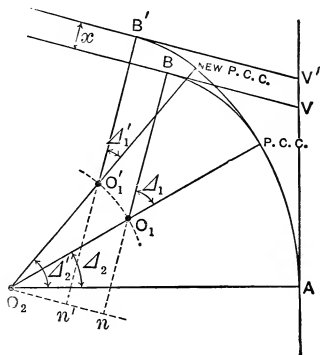


FIG. 28.

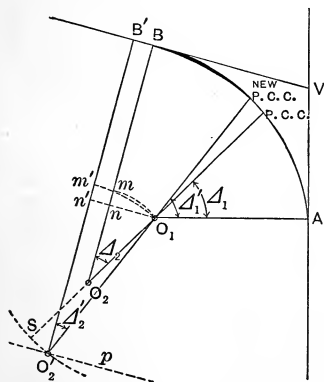


FIG. 29.

curve *produced* at NEW *P.C.C.* Draw $O_1 n'$ perpendicular to $O_2 B$.

With O_2 as center draw the arc O_1m , and with O_2' as center draw the arc O_1m' . $mB = m'B' = R_1$.

$$\therefore mn = m'n' = (R_2' - R_1) \text{ vers } \Delta_2' = (R_2 - R_1) \text{ vers } \Delta_2.$$

$$\therefore \text{vers } \Delta_2' = \frac{(R_2 - R_1)}{(R_2' - R_1)} \text{ vers } \Delta_2. \quad (28)$$

$$O_1n = (R_2 - R_1) \sin \Delta_2;$$

$$O_1n' = (R_2' - R_1) \sin \Delta_2'.$$

$$BB' = O_1n' - O_1n = (R_2' - R_1) \sin \Delta_2' - (R_2 - R_1) \sin \Delta_2. \quad (29)$$

This problem may be further modified by assuming that the radius of the curve is decreased rather than increased, or that the smaller radius follows the larger. The solution is similar and is suggested as a profitable exercise.

It might also be assumed that, instead of making a given change in the radius R_2 , a given change BB' is to be made. Δ_2' and R_2' are required. Eliminate R_2' from Eqs. 28 and 29 and solve the resulting equation for Δ_2' . Then determine R_2' by a suitable inversion of either Eq. 28 or 29.

As in §§ 32 and 33, the above problems are but a few, although perhaps the most common, of the problems the engineer may meet with in compound curves. All of the ordinary problems may be solved by these and similar methods.

40. Problems. *a.* Assume that the two tangents of a compound curve are to be 348 feet and 624 feet, and that $\Delta_1 = 22^\circ 16'$ and $\Delta_2 = 28^\circ 20'$. Required the radii.

$$[Ans. R_1 = 326.92; R_2 = 1574.85.]$$

b. A line crosses a valley by a compound curve which is first a 6° curve for $46^\circ 30'$ and then a $9^\circ 30'$ curve for $84^\circ 16'$. It is afterward decided that the last tangent should be 6 feet farther up the hill. What are the required changes? [Note. The second tangent is evidently moved *outward*. The solution corresponds to that in the first part of § 39, *c*. The *P.C.C.* is moved forward 16.39 feet. If it is desired to know how far the *P.T.* is moved in the direction of the tangent (i.e., the *projection* of BB' , Fig. 28, on $V'B'$), it may be found by observing that it is equal to $nn' = (R_2 - R_1)(\sin \Delta_1 - \sin \Delta_1')$. In this case it equals 0.65 foot, which is very small because Δ_1 is nearly 90° . The value of Δ_2 ($46^\circ 30'$) is not used, since the solution is independent of the value of Δ_2 . The student should learn to recognize

any superelevation which will fit all velocities even approximately. The above fact also shows why any over-refinement in the calculations is useless and why the above approximations, which are really small, are amply justifiable. For example, the above formula contains the approximation that $R = 5730 \div D$. In the extreme case of a 10° curve the error involved would be about 1%. A change of about $\frac{1}{2}$ of 1% in the velocity, or say from 40 to 40.2 miles per hour, would mean as much. The error in e due to the assumed constant value of sm is never more than a very small fraction of 1%. The rail-laying is not done closer than this. The following tabular form is based on Eq. (30):

SUPERELEVATION OF THE OUTER RAIL (IN FEET) FOR VARIOUS VELOCITIES AND DEGREES OF CURVATURE.

Velocity in Miles per Hour.	Degree of Curve.									
	1°	2°	3°	4°	5°	6°	7°	8°	9°	10°
30	.05	.10	.15	.20	.26	.31	.36	.41	.46	.51
40	.09	.18	.27	.37	.46	.55	.64	.73	.82	
50	.14	.29	.43	.57	.71	.86				
60	.20	.41	.62	.82						

42. Practical rules for superelevation. A much used rule for superelevation is to "elevate one half an inch for each degree of curvature." The rule is rational in that e in Eq. 30 varies directly as D . The above rule therefore agrees with Eq. 30 when V is about 27 miles per hour. However applicable the rule may have been in the days of low velocities, the elevation thus computed is too small now. The rule to elevate one inch for each degree of curvature is also used and is precisely similar in its nature to the above rule. It agrees with Eq. 30 when the velocity is about 38 miles per hour, which is more nearly the average speed of trains.

Another (and better) rule is to "elevate for the speed of the fastest trains." This rule is further justified by the fact that a four-wheeled truck, having two parallel axles, will always tend to run to the outer rail and will require considerable flange pressure to guide it along the curve. The effect of an excess of superelevation on the slower trains will only be to relieve this flange pressure somewhat. This rule is coupled with the limitation

that the elevation should never exceed a limit of six inches—sometimes eight inches. This limitation implies that locomotive engineers must reduce the speed of fast trains around sharp curves until the speed does not exceed that for which the actual superelevation used is suitable. The heavy line in the tabular form (§ 41) shows the six-inch limitation.

Some roads furnish their track foremen with a list of the superelevations to be used on each curve in their sections. This method has the advantage that each location may be separately studied, and the proper velocity, as affected by local conditions (*e.g.*, proximity to a stopping-place for all trains), may be determined and applied.

Another method is to allow the foremen to determine the superelevation for each curve by a simple measurement taken at the curve. The rule is developed as follows: By an inversion of Eq. 19 we have

$$x = \text{chord}^2 \div 8R. \quad . \quad . \quad . \quad . \quad . \quad (31)$$

Putting x equal to e in Eq. 30 and solving for “chord,” we have

$$\begin{aligned} \text{chord}^2 &= .0000572V^2D\&R \\ &= 2.621V^2. \\ \text{chord} &= 1.62V. \quad . \quad . \quad . \quad . \quad . \quad . \quad . \quad (32) \end{aligned}$$

To apply the rule, assume that 50 miles per hour is fixed as the velocity from which the superelevation is to be computed. Then $1.62V = 1.62 \times 50 = 81$ feet, which is the distance given to the trackmen. Stretch a tape (or even a string) with a length of 81 feet between two points on the concave side of the head of either the inner or the outer rail. The ordinate at the middle point then equals the superelevation. The values of this chord length for varying velocities are given in the accompanying tabular form.

Velocity in miles per hour...	20	25	30	35	40	45	50	55	60
Chord length in feet.....	32.4	40.5	48.6	56.7	64.8	72.9	81.0	89.1	97.2

The following tabular form shows the standard (at one time) on the N. Y., N. H. & H. R. R. It should be noted that the elevations do not increase proportionately with the radius, and that they are higher for descending grades than for level or

ascending grades. This is on the basis that the velocity on curves and on ascending grades will be less than on descending grades. For example, the superelevation for a $0^{\circ} 30'$ curve on a descending grade corresponds to a velocity of about 54 miles per hour, while for a 4° curve on a level or ascending grade the superelevation corresponds to a velocity of only about 38 miles per hour.

TABLE OF THE SUPERELEVATION OF THE OUTER RAIL ON CURVES.
N. Y., N. H. & H. R. R.

Degree of curve.	Level or ascending grade.	Descending grade.
	inches.	inches.
$0^{\circ} 30'$	$0\frac{1}{4}$	1
1 00	$1\frac{1}{4}$	$1\frac{1}{4}$
1 15	$1\frac{1}{4}$	2
1 30	2	$2\frac{1}{4}$
1 45	$2\frac{1}{4}$	$2\frac{1}{4}$
2 00	$2\frac{3}{8}$	$2\frac{3}{4}$
2 15	$2\frac{3}{8}$	3
2 30	$2\frac{7}{8}$	$3\frac{1}{4}$
2 45	3	$3\frac{3}{8}$
3 00	$3\frac{1}{8}$	$3\frac{3}{8}$
3 15	$3\frac{3}{8}$	$3\frac{7}{8}$
3 30	$3\frac{5}{8}$	4
3 45	$3\frac{5}{8}$	$4\frac{1}{8}$
4 00	4	$4\frac{1}{2}$

43. Transition from level to inclined track. On curves the track is inclined transversely; on tangents it is level. The transition from one condition to the other must be made gradually. If there is no transition curve, there must be either inclined track on the tangent or insufficiently inclined track on the curve or both. Sometimes the full superelevation is continued through the total length of the curve and the "run-off" (having a length of 100 to 400 feet) is located entirely on the tangents at each end. In other practice it is located partly on the tangent and partly on the curve. Whatever the method, the superelevation is correct at only one point of the run-off. At all other points it is too great or too small. This (and other causes) produces objectionable lurches and resistances when entering and leaving curves. The object of transition curves is to obviate these resistances.

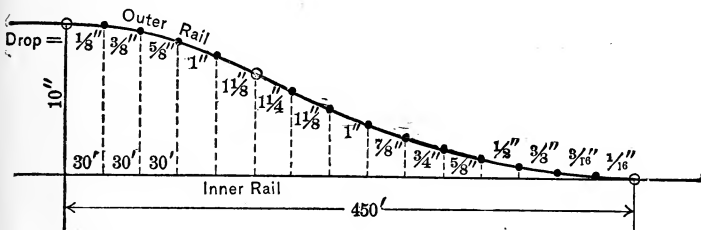
On the Lehigh Valley R. R. the run-off is made in the form of a reversed vertical curve, as shown in the accompanying figure. According to this system the length of run-off varies

from 120 feet, for a superelevation of one inch, to 450 feet, for a superelevation of ten inches. Such a superelevation as ten inches is very unusual practice, but is successfully operated on that road. The curve is concave upward for two-thirds of its length and then reverses so that it is convex upward.

TABLE FOR RUN-OFF OF ELEVATION OF OUTER RAIL OF CURVES.

Drop in inches for each 30-foot rail commencing at theoretical point of curve.

Eleva- tion.	$\frac{1}{8}"$	$\frac{1}{4}"$	$\frac{3}{8}"$	$\frac{1}{2}"$	$\frac{5}{8}"$	$\frac{3}{4}"$	$\frac{7}{8}"$	1"	$1\frac{1}{8}"$	$1\frac{1}{4}"$	$1\frac{1}{8}"$	1"	$\frac{7}{8}"$	$\frac{3}{4}"$	$\frac{5}{8}"$	$\frac{1}{2}"$	$\frac{3}{8}"$	$\frac{1}{4}"$	$\frac{3}{16}"$	$\frac{1}{8}"$	Total.
1"		30	30															30		30	120
2"		30				30														30	150
3"		30				30							30					30			180
4"		30		30			30						30					30			240
5"		30		30				30					30					30			270
6"		30		30			30		30				30					30			300
7"		30		30			30		30				30					30			330
8"		30		30			30		30				30					30			360
9"	30			30			30		30				30					30			420
10"	30		30		30			30	30				30					30			450



The figure (and also the lower line of the tabulated form) shows the drop for each thirty-foot rail length. For shorter lengths of run-off; the drop for each 30 feet is shown by the corresponding lines in the tabular form. Note in each horizontal line that the sum of the drops, under which 30 is found, equals the total superelevation as found in the first column. For example, for 4 inches superelevation, length of curve 240 feet, the successive drops are $\frac{1}{4}"$, $\frac{1}{2}"$, $\frac{7}{8}"$, $\frac{7}{8}"$, $\frac{5}{8}"$, $\frac{1}{2}"$, $\frac{1}{4}"$, and $\frac{1}{8}"$ whose sum is 4 inches. Possibly the more convenient form would be to indicate for each 30-foot point the actual superelevation of the outer rail, which would be for the above case (running from the tangent to the curve) $\frac{1}{8}"$, $\frac{3}{8}"$, $\frac{7}{8}"$, $1\frac{1}{8}"$, $2\frac{3}{8}"$, $3\frac{1}{4}"$, $3\frac{3}{4}"$, 4".

44. Fundamental principle of transition curves. If a curve

has variable curvature, beginning at the tangent with a curve of infinite radius, and the curvature gradually sharpens until it equals the curvature of the required simple curve and there becomes tangent to it, the superelevation of such a transition curve may begin at zero at the tangent, gradually increase to the required superelevation for the simple curve, and yet have at every point the superelevation required by the curvature at that point. Since in Eq. (30) e is directly proportional to D , the required curve must be one in which the degree of curve increases directly as the distance along the curve. The mathematical development of such a curve is quite complicated. It has, however, been developed, and tables have been computed for its use, by Prof. C. L. Crandall. The following method has the advantage of great simplicity, while its agreement with the true transition curve is as close as need be, as will be shown.

45. Multiform compound curves. If the transition curve commences with a very flat curve and at regular even chord lengths compounds into a curve of sharper curvature until the desired curvature is reached, the increase in curvature at each chord point being uniform, it is plain that such a curve is a close approximation to the true spiral, especially since the rails as laid will *gradually* change their curvature rather than maintain a uniform curvature throughout each chord length and then abruptly change the curvature at the chord points. Such a curve, *as actually laid*, will be a much closer approximation to the true curve than the multiform compound curve by which it is set out. There will actually be a *gradual* increase in curvature which increases directly as the length of the curve.

46. Required length of spiral. The required length of spiral evidently depends on the amount of superelevation to be gained, and also depends somewhat on the speed. If the spiral is laid off in 25-foot chord lengths, with the first chord subtending a 1° curve, the second a 2° curve, etc., the fifth chord will subtend a 5° curve, and the increase from this last chord to a 6° curve is the same as the uniform increase of curvature between the chords. The same spiral extended would run on to a 12° curve in $(12-1)25=275$ feet. The last chord of a spiral should have a smaller degree of curvature than the simple curve to which it is joined. If the curves are very sharp, such as are used in street work and even in suburban trolley work, an increase in degree of curvature of 1° per 25 feet will not be sufficiently rapid, as

such a rate would require too long curves. 2° , 10° , or even 20° increase per 25 feet may be necessary, but then the chords should be reduced to 5 feet. Such a rapid rate of increase is justified by the necessary reduction in speed. On the other hand, very high speed will make a lower rate of increase desirable, and therefore a spiral whose degree of curvature increases only $0^\circ 30'$ per 25 feet may be used. Such a spiral would require a length of 375 feet to run on to an 8° curve, which is inconveniently long, but it might be used to run on to a 4° curve, where its length would be only 175 feet. Three spirals have been developed in Table IV, each with chords of 25 feet, the rate of increase in the degree of curvature being $0^\circ 30'$, 1° and 2° per chord. One of these will be suitable for any curvature found on ordinary steam-railroads.

47. To find the ordinates of a 1° -per-25-feet spiral. Since the first chord subtends a 1° curve, its central angle is $0^\circ 15'$ and the angle aQV (Fig. 31) is $7' 30''$. The tangent at a makes an angle of $15'$ with VQ . The angle between the chord ba and the tangent at a is $\frac{1}{2}(30') = 15'$, and the angle $bab'' = \frac{1}{2}(30') + 15' = 30'$. Similarly

the angle $abc'' = \frac{1}{2}(45') + 30' + 15' = 67' 30'' = 1^\circ 07' 30''$,
and the angle $dcd'' = 2^\circ 0'$.

The ordinate $aa' = 25 \sin 7' 30''$, and

$$Qa' = 25 \cos 7' 30''.$$

$$Qb' = Qa' + a'b'$$

$$= Qa' + ab''$$

$$= 25 (\cos 7' 30'' + \cos 30').$$

$$bb' = b'b'' + bb''$$

$$= 25 (\sin 7' 30'' + \sin 30').$$

Similarly the ordinates of c , d , etc., may be obtained.

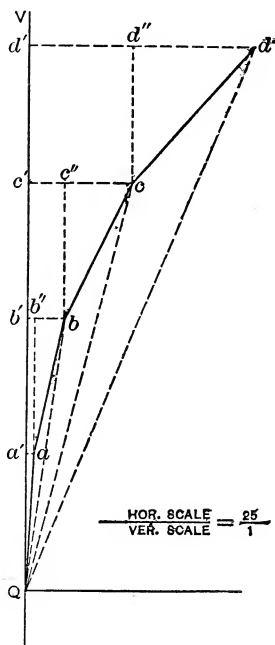


FIG. 31.

by a D° curve, having a suitable spiral at each end. If no spirals were to be used, the problem would be solved as in simple curves giving the curve AMB . Introducing the spiral has the effect of throwing the curve away from the vertex a distance MM' and reducing the central angle of the D° curve by 2ϕ . Continuing the curve beyond Z and Z' to A' and B' , we will have $AA' = BB' = MM'$. ZK = the x ordinate and is therefore known. Call $MM' = m$. $A'N = x - R \text{ vers } \phi$. Then

$$m = MM' = AA' = \frac{A'N}{\cos \frac{1}{2}A} = \frac{x - R \text{ vers } \phi}{\cos \frac{1}{2}A}. \quad (33)$$

$$NA = AA' \sin \frac{1}{2}A = (x - R \text{ vers } \phi) \tan \frac{1}{2}A.$$

$$VQ = QK - KN + NA + AV$$

$$= y - R \sin \phi + (x - R \text{ vers } \phi) \tan \frac{1}{2}A + R \tan \frac{1}{2}A \\ = y - R \sin \phi + x \tan \frac{1}{2}A + R \cos \phi \tan \frac{1}{2}A. \quad (34)$$

When $A'N$ has already been computed, it may be more convenient to write

$$VQ = y + R (\tan \frac{1}{2}A - \sin \phi) + A'N \tan \frac{1}{2}A. \quad (35)$$

$$VM' = VM + MM'$$

$$= R \text{ exsec } \frac{1}{2}A + \frac{x}{\cos \frac{1}{2}A} - \frac{R \text{ vers } \phi}{\cos \frac{1}{2}A}. \quad (36)$$

$$AQ = VQ - AV$$

$$= y - R \sin \phi + (x - R \text{ vers } \phi) \tan \frac{1}{2}A. \quad (37)$$

A method of obtaining the necessary dimensions using tables, is given in § 53a.

Example. To join two tangents making an angle of $34^\circ 20'$ by a $5^\circ 40'$ curve and suitable spirals. Use 1°-per-25-feet spirals with five chords. Then $\phi = 3^\circ 45'$, $x = 2.999$, $\frac{1}{2}A = 17^\circ 10'$, and $y = 124.942$.

[Eq. 33]

	R	3.00497
	vers ϕ	7.33063
		<u>0.33560</u>
	2.166	
	$x = 2.999$	
	$A'N = 0.833$	9.92064
		<u>9.98021</u>
	$\cos \frac{1}{2}A$	9.94043
	$m = MM' = AA' = 0.872$	<u>9.94043</u>

[Eq. 36]

	R	3.00497
	exsec $\frac{1}{2}A$	8.66863
		<u>1.67360</u>
	$VM = 47.164$	
	$m = 0.872$	
	$VM' = 48.036$	

[Eq. 35]

	$y = 124.942$	nat. $\tan \frac{1}{2}A = .30891$	
		nat. $\sin \phi = .06540$	
		<u>.24351</u>	9.38651
		R	3.00497
	246.314		<u>2.39148</u>
	[See above]	$A'N$	9.92064
		$\tan \frac{1}{2}A$	9.48984
	0.257	AN	<u>9.41048</u>
	$VQ = 371.513$		

[Eq. 37]

	R	3.00497
	$\tan \frac{1}{2}A$	9.48984
	AV	<u>2.49481</u>
	312.471	
	$AQ = 59.042$	

50. **Field-work.** When the spiral is designed during the original location, the tangent distance VQ should be computed and the point Q located. It is hardly necessary to locate all of the points of the spiral until the track is to be laid. The extremities should be located, and as there will usually be one and perhaps two full station points on the spiral, these should also be located. Z may be located by setting off $QK = y$ and $KZ = x$, or else by the tabular deflection for Z from Q and the distance ZQ , which is the long chord. Setting up the instrument at Z and sighting back at Q with the proper deflection, the tangent at Z may be found and the circular curve located as usual, its central angle being $A - 2\phi$. A similar operation will locate Q' from Z' .

To locate points on the spiral. Set up at Q , with the plates

checking the back points and locating all forward points up to Z if possible.

After the center curve has been located and Z' is reached, the other spiral must be located but *in reverse order*, i.e., the sharp curvature of the spiral is at Z' and the curvature decreases toward Q' .

51. To replace a simple curve by a curve with spirals. This may be done by the method of § 49, but it involves shifting the whole track a distance m , which in the given example equals 0.87 foot. Besides this the track is appreciably shortened, which would require rail-cutting. But the track may be kept at practically the same length and the lateral deviation from the old track may be made very small by slightly sharpening the curvature of the old track, moving the new curve so that it is wholly or partially *outside* of the old curve, the remainder of it with the spirals being *inside* of the old curve. It is found by experience that a decrease in radius of from 1% to 5% will answer the purpose. The larger the central angle the less the change. The solution is as indicated in Fig. 34.

$$O'N = R' \cos \phi + x.$$

$$O'V = O'N \sec \frac{1}{2}A$$

$$= R' \cos \phi \sec \frac{1}{2}A + x \sec \frac{1}{2}A.$$

$$m = MM' = MV - M'V$$

$$= R \operatorname{exsec} \frac{1}{2}A - (O'V - R')$$

$$= R \operatorname{exsec} \frac{1}{2}A - R' \cos \phi \sec \frac{1}{2}A - x \sec \frac{1}{2}A + R'. \quad (38)$$

$$AQ = QK - KN + NV - VA$$

$$= y - R' \sin \phi + (R' \cos \phi + x) \tan \frac{1}{2}A - R \tan \frac{1}{2}A$$

$$= y - R' \sin \phi + R' \cos \phi \tan \frac{1}{2}A - (R - x) \tan \frac{1}{2}A. \quad (39)$$

The length of the old curve from Q to $Q' = 2AQ + 100 \frac{A}{D}$.

The length of the new curve from Q to $Q' = 2L + 100 \frac{A - 2\phi}{D'}$,

in which L is the length of each spiral.

Example. Suppose the old curve is a $7^\circ 30'$ curve with a central angle of $38^\circ 40'$. As a trial, compute the relative length of a new 8° curve with spirals of seven chords. $\phi = 7^\circ 0'$; $\frac{1}{2}A = 19^\circ 20'$; R (for the $7^\circ 30'$ curve) = 764.489; R' (for the 8° curve) = 716.779; $x = 7.628$.

[Eq. 38]

	R	2.88337
	$\text{exsec } \frac{1}{2} \Delta$	8.77642
$R' = \frac{45.687}{716.779}$		<u>1.65979</u>
762.466	R'	2.85538
	$\cos \phi$	9.99675
	$\sec \frac{1}{2} \Delta$	0.02521
753.953		<u>2.87734</u>
	x	0.88241
	$\sec \frac{1}{2} \Delta$	0.02521
8.084		<u>0.90762</u>
$\frac{762.037}{762.037}$		
$m = \frac{0.429}{174.722}$		
$y = 174.722$	R'	2.85538
	$\sin \phi$	9.08589
		<u>1.94128</u>
	R'	2.85538
	$\cos \phi$	9.99675
	$\tan \frac{1}{2} \Delta$	9.54512
249.606		<u>2.39725</u>
	$R = 764.489$	
	$x = 7.628$	
	$\frac{756.861}{\tan \frac{1}{2} \Delta}$	2.87901
		9.54512
$\frac{424.328}{352.896}$		<u>2.42413</u>
352.896		
$AQ = 71.432$		

[Eq. 39]

	R'	2.85538
	$\sin \phi$	9.08589
87.353		<u>1.94128</u>
	R'	2.85538
	$\cos \phi$	9.99675
	$\tan \frac{1}{2} \Delta$	9.54512
249.606		<u>2.39725</u>
	$R = 764.489$	
	$x = 7.628$	
	$\frac{756.861}{\tan \frac{1}{2} \Delta}$	2.87901
		9.54512
$\frac{424.328}{352.896}$		<u>2.42413</u>
352.896		
$AQ = 71.432$		

The length of the old curve from Q to Q' is

$$100 \frac{\Delta}{D} = 100 \frac{38.667}{7.5} = 515.556$$

$$2AQ = 2 \times 71.432 = 142.864$$

$$658.420$$

$$\text{New curve: } 100 \frac{\Delta - 2\phi}{D'} = 100 \frac{38.667 - 14.000}{8.0} = 308.333$$

$$2L = 2 \times 175 = 350.000$$

$$658.333 \quad 658.333$$

$$\text{Difference in length} = 0.087$$

Considering that this difference may be divided among 22 joints (using 30-foot rails) no rail-cutting would be necessary. If the difference is too large, a slight variation in the value of the new radius R' will reduce the difference as much as necessary. A truer comparison of the lengths would be found by comparing the lengths of the arcs.

52. Application of transition curves to compound curves. Since compound curves are only employed when the location is limited by local conditions, the elements of the compound curve should be determined (as in §§ 38 and 39) regardless of the

transition curves, depending on the fact that the lateral shifting of the curve when transition curves are introduced is very small. If the limitations are very close, an estimated allowance may be made for them.

Methods have been devised for inserting transition curves between the branches of a compound curve, but the device is

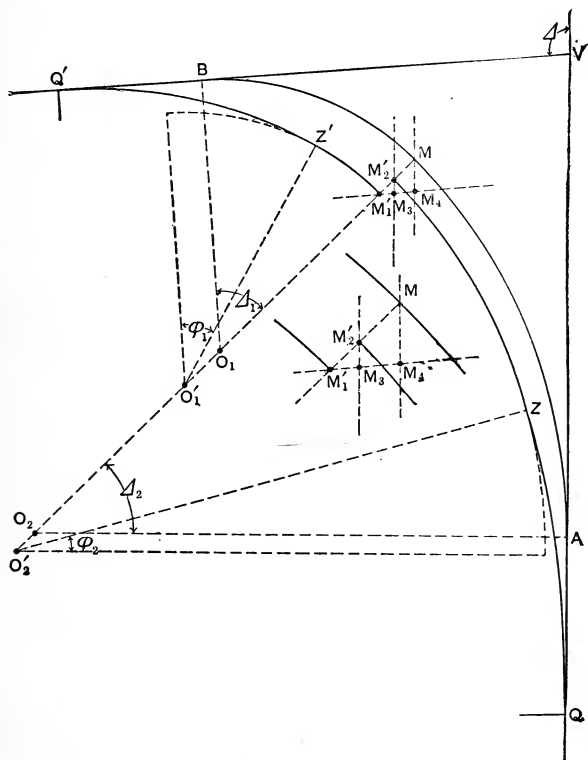


FIG. 35.

complicated and usually needless, since when the train is once on a curve the wheels press against the outer rail steadily and a change in curvature will not produce a serious jar even though the superelevation is temporarily a little more or less than it should be.

If the easier curve of the compound curve is less than 3° or 4° , there may be no need for a transition curve off from that branch. This problem then has two cases according as transition curves are used at both ends or at one end only.

a. *With transition curves at both ends.* Adopting the method of § 49, calling $\Delta_1 = \frac{1}{2}\Delta$, we may compute $m_1 = MM_1'$. Similarly, calling $\Delta_2 = \frac{1}{2}\Delta$, we may compute $m_2 = MM_2'$. But M_1' and M_2' must be made to coincide. This may be done by moving the curve $Z'M_1'$ and its transition curve parallel to $Q'V$ a distance $M_1'M_3$, and the other curve parallel to QV a distance $M_2'M_3$. In the triangle $M_1'M_3M_2'$, the angle at $M_1' = 90^\circ - \Delta_1$, the angle at $M_2' = 90^\circ - \Delta_2$, and the angle at $M_3 = \Delta$.

$$\left. \begin{aligned} \text{Then } M_1'M_3 &= M_1'M_2' \frac{\sin(90^\circ - \Delta_2)}{\sin \Delta} = (m_1 - m_2) \frac{\cos \Delta_2}{\sin \Delta} \\ \text{Similarly } M_2'M_3 &= M_1'M_2' \frac{\sin(90^\circ - \Delta_1)}{\sin \Delta} = (m_1 - m_2) \frac{\cos \Delta_1}{\sin \Delta} \end{aligned} \right\} (40)$$

b. *With a transition curve on the sharper curve only.* Compute $m_1 = MM_1'$ as before; then move the curve Z_1M_1' parallel to $Q'V$ a distance of

$$M_1'M_4 = m_1 \frac{\cos \Delta_2}{\sin \Delta} \dots \dots \dots (41)$$

The simple curve MA is moved parallel to VA a distance of

$$MM_4 = m_1 \frac{\cos \Delta_1}{\sin \Delta} \dots \dots \dots (42)$$

If Δ_1 and Δ_2 are both small, $M_1'M_4$ and MM_4 may be more than m_1 , but the lateral deviation of the new curve from the old will always be less than m_1 .

53. To replace a compound curve by a curve with spirals. The numerical illustration given below employs another method. We first solve for m_1 for the sharper branch of the curve, placing $\Delta_1 = \frac{1}{2}\Delta$ in Eq. 38. A value for R_2' may be found whose corresponding value of m_2 will equal m_1 . Solving Eq. 38 for R' , we obtain

$$R' = \frac{R \text{ vers } \frac{1}{2}\Delta - m \cos \frac{1}{2}\Delta - x}{\cos \phi - \cos \frac{1}{2}\Delta} \dots \dots (43)$$

Substituting in this equation the known value of $m_1 (=m_2)$ and calling $R'=R_2'$, $R=R_2$, and $A_2=\frac{1}{2}A$, solve for R_2' . Obtain the value of AQ for each branch of the curve separately by Eq. 39, and compare the lengths of the old and new lines.

Example. Assume a compound curve with $D_1=8^\circ$, $D_2=4^\circ$, $A_1=36^\circ$, and $A_2=32^\circ$. Use 1° -per-25-feet spirals; $\phi_1=7^\circ 0'$; $\phi_2=1^\circ 30'$. Assume that the sharper curve is sharpened from $8^\circ 0'$ to $8^\circ 12'$.

[Eq. 38]

		R_1	2.85538
		exsec 36°	9.37303
$R_1' = \frac{169.209}{699.326}$			2.22842
		R_1'	2.84468
		$\cos \phi_1$	9.99675
		$\sec A_1$	0.09204
	857.970		2.93347
		x_1	0.88241
		$\sec A_1$	0.09204
			0.97445
	9.429		
$\frac{867.399}{m_1 = 1.136}$	867.399		

[Eq. 43]

		R_2	3.15615
		vers 32°	9.18170
217.700			2.33785
		$m_1 = 1.136$	0.05538
		$\cos 32^\circ$	9.92842
	0.963		9.98380
	$x_2 = 0.763$		
	1.726		
215.974	1.726		2.33440

[Eq. 39]

		nat. cos $\phi = .99966$	
		nat. cos $A_2 = .84805$	
		.15161	9.18073
$R_2' = 1424.54$	$[4^\circ 1' 22'']$		3.15367
$y_1 = 174.722$		R_1'	2.84468
		$\sin \phi_1$	9.08589
	85.226		1.93057
		R_1'	2.84468
		$\cos \phi_1$	9.99675
		$\tan \frac{1}{2} A [A_1 = 36^\circ]$	9.86126
504.302			2.70269
		$R_1 = 716.779$	
		$x_1 = 7.628$	
		709.151	2.85074
		$\tan \frac{1}{2} A$	9.86126
	679.024		
	600.461		
$AQ_1 = 78.563$	515.235		2.71200
	600.461		

[Eq. 39]

$y_2 = 74.994$	R_2'	3.15367
	$\sin \phi_2$	8.41792
37.290		1.57159
	R_2'	3.15367
	$\cos \phi_2$	9.99985
	$\tan \frac{1}{2}A (A_2 = 32^\circ)$	9.79579
889.843		2.94931
	$R_2 = 1432.69$	
	$x_2 = 0.76$	
	1431.93	3.15592
	$\tan \frac{1}{2}A$	9.79579
894.770		2.95171
<u>964.837</u>		
<u>932.060</u>		
$AQ_2 = 32.777$		

For the length of the old track we have :

$$100 \frac{A_1}{D_1} = 100 \frac{36^\circ}{8^\circ} = 450.$$

$$100 \frac{A_2}{D_2} = 100 \frac{32^\circ}{4^\circ} = 800.$$

$$AQ_1 = 78.563$$

$$AQ_2 = 32.777$$

$$\hline 1361.340$$

For the length of the new track we have:

$$100 \frac{A_1 - \phi_1}{D_1'} = 100 \frac{29^\circ}{8^\circ.20} = 353.659$$

$$100 \frac{A_2 - \phi_2}{D_2'} = 100 \frac{30^\circ.5}{4^\circ.023} = 758.140$$

$$\text{Spiral on } 8^\circ 12' \text{ curve } 175.000$$

$$\text{" " } 4^\circ 01' 22'' \text{ " } 75.$$

$$\text{Length of new track} = 1361.799$$

$$\text{" " old " } = 1361.340$$

$$\text{Excess in length of new track} = 0.459 \text{ feet.}$$

Since the new track is slightly longer than the old, it shows that the new track runs too far *outside* the old track at the *P.C.C.* On the other hand the offset m is only 1.136. The maximum amount by which the new track comes *inside* of the old track at two points, presumably not far from Z' and Z , is very difficult to determine exactly. Since it is desirable that the maximum offsets (inside and outside) should be made as nearly equal as possible, this feature should not be sacrificed to an effort to make the two lines of precisely equal length so that the rails need not be cut. Therefore, if it is found that the offsets inside the old track are nearly equal to m (1.136), the above figures should stand. Otherwise m may be diminished (and the above excess in length of track diminished) by *increasing* R_1' very slightly and making the necessary consequent changes.

53a. Use of Table IV. Prof. R. B. H. Begg, of Syracuse University, has submitted to the author a series of tables which will materially simplify the work of solving Equations (33) to (37), and which have been added to Table IV of previous editions. Since these equations involve R and Δ (which may each have *any* values) in combination with several values of ϕ , it would require impracticably extensive tables to give precise values of the required dimensions for *any* possible combination of R , Δ , and ϕ . But the tables may be utilized by *interpolation* with all necessary accuracy within their range. Rules for the use of the tables and for the field work are as follows:

1. Find $P. C.$ (point A) as if no transition curve were to be used.
2. Lay off the distance AN (part C of table) to N ; then offset the distance $A'N$ (part B) to A' , the new $P. C.$; from N measure a distance NQ (part B) to Q .
3. Set transit on A' ; sight parallel to tangent and run in circular curve, setting Z from deflection and distance (part B); or Z can be set by measuring ZK and QK (part B) from Q .
4. Set transit on Q , sight along tangent and turn the deflection (part A) for each 25-foot station, for as many chord lengths as required; or the points may be located by measuring distances y along the tangent from Q and offsetting the corresponding distances x .

VERTICAL CURVES:

54. Necessity for their use. Whenever there is a change in the rate of grade, it is necessary to eliminate the angle that would be formed at the point of change and to connect the two grades by a curve. This is especially necessary at a sag between two grades, since the shock caused by abruptly forcing an upward motion to a rapidly moving heavy train is very severe both to the track and to the rolling stock. The necessity for vertical curves was even greater in the days when link couplers were in universal use and the "slack" in a long train was very great. Under such circumstances, when a train was moving down a heavy grade the cars would crowd ahead against the engine. Reaching the sag, the engine would begin to pull out, rapidly taking out the slack. Six inches of slack on each car would amount to several feet on a long train, and the resulting jerk on

the couplers, especially those near the rear of the train, has frequently resulted in broken couplers or even derailments. A vertical curve will practically eliminate this danger if the curve is made long enough, but the rapidly increasing adoption of close spring couplers and air-brakes, even for freight trains, is obviating the necessity for such very long curves.

55. Required length. Theoretically the length should depend on the change in the rate of grade and on the length of the longest train on the road. A sharp change in the rate of grade requires a long curve; a long train requires a long curve; but since the longest trains are found on roads with light grades and small changes of grade, the required length is thus somewhat equalized. The A.R.E.A. rule is: "On class A roads (see § 198) rates of change of 0.1 per cent per station on summits and 0.05 per cent per station in sags should not be exceeded. On minor roads 0.2 per cent per station on summits and 0.1 per cent per station in sags may be used." When changing from a down grade to an up grade (or vice versa) the change of grade equals the numerical sum of the two rates of grade. For example, if a 0.5 per cent down grade is followed by a 0.7 per cent up grade, the road being a "minor" road, then, by the above rule the length of the curve should be at least $[0.5 - (-0.7)] \div 0.1 = 12$ stations or 1200 feet. Added length increases the amount of earthwork required both in cuts and fills, but the resulting saving in operating expenses will always justify a considerable increase.

56. Form of curve. In Fig. 36 assume that *A* and *C*, equi-

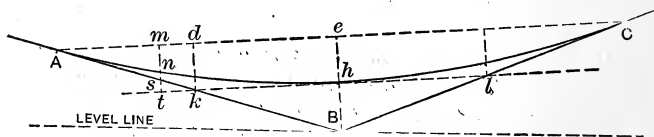


FIG. 36.

distant from *B*, are the extremities of the vertical curve. Bisect *AC* at *e*; draw *Be* and bisect it at *h*. Bisect *AB* and *BC* at *k* and *l*. The line *kl* will pass through *h*. A parabola may be drawn with its vertex at *h* which will be tangent to *AB* and *BC* at *A* and *C*. It may readily be shown * from the properties of a parabola that if an ordinate be drawn at *any* point (as at *n*) we will have

* See note at foot of p. 64.

$$sn : eh \text{ (or } hB) :: \overline{Am}^2 : \overline{Ae}^2$$

$$\text{or} \quad sn = eh \frac{\overline{Am}^2}{\overline{Ae}^2} \quad . \quad . \quad . \quad . \quad . \quad . \quad (44)$$

In Fig. 36 the grades are necessarily exaggerated enormously. With the proportions found in practice we may assume that ordinates (such as mt , eB , etc.) are perpendicular to either grade, as may suit our convenience, without any appreciable error. In the numerical case given below, the variation of these ordinates from the vertical is $0^\circ 07'$, while the effect of this variation on the calculations in this case (as in the most extreme cases) is absolutely inappreciable. It may easily be shown that the angle CAB = half the *algebraic difference* of the rates of grade. Call the difference, expressed in per cent of grade, r ; then $CAB = \frac{1}{2}r$. Let l = length (in "stations" of 100 feet) of the line AC , which is practically equal to the horizontal measurement. Since the angle CAB is one-half the total change of grade at B , it follows that $Be = \frac{1}{2}l \times \frac{1}{2}r$ Therefore

$$Bh = \frac{1}{8}lr. \quad . \quad . \quad . \quad . \quad . \quad . \quad (44a)$$

Since Bh (or eh) are constant for any one curve, the correction sn at any point (see Eq. 44) equals a constant times Am^2 .

57. Numerical example. Assume that B is located at Sta. 16+20; that the grade of AB is -0.5% , and of BC $+0.7\%$; also that the elevation of B above the datum plane is 162.6. Then the algebraic difference of the grades, r , $= 0.7 - (-0.5) = 1.2$; $l = 12$. $Bh = \frac{1}{8}lr = \frac{1}{8} \times 12 \times 1.2 = 1.8$. A is at Sta. 10+20 and its elevation is $162.6 + (6 \times 0.5) = 165.6$; C is at Sta. 22+20 and its elevation is $162.6 + (6 \times 0.7) = 166.8$. The elevation of Sta. 11 is found by adding sn to the elevation of s on the straight grade line. The constant $(eh \div \overline{Ae}^2)$ equals in this case $1.8 \div 600^2 = \frac{1}{200000}$. Therefore the curve elevations are

A, Sta. 10+20,	$162.6 + (6.00 \times 0.5)$	$= 165.60$
11	$165.6 - (0.80 \times 0.5) + \frac{1}{200000} 80^2$	$= 165.23$
12	$165.6 - (1.80 \times 0.5) + \frac{1}{200000} 180^2$	$= 164.86$
13	$165.6 - (2.80 \times 0.5) + \frac{1}{200000} 280^2$	$= 164.59$
14	$165.6 - (3.80 \times 0.5) + \frac{1}{200000} 380^2$	$= 164.42$
15	$165.6 - (4.80 \times 0.5) + \frac{1}{200000} 480^2$	$= 164.35$
16	$165.6 - (5.80 \times 0.5) + \frac{1}{200000} 580^2$	$= 164.38$

B,	16+20, 162.6 + 1.80	=164.40
	17 166.8 - (5.20 × 0.7) + $\frac{1}{200000} 520^2$	=164.51
	18 166.8 - (4.20 × 0.7) + $\frac{1}{200000} 420^2$	=164.74
	19 166.8 - (3.20 × 0.7) + $\frac{1}{200000} 320^2$	=165.07
	20 166.8 - (2.20 × 0.7) + $\frac{1}{200000} 220^2$	=165.50
	21 166.8 - (1.20 × 0.7) + $\frac{1}{200000} 120^2$	=166.03
	22 166.8 - (0.20 × 0.7) + $\frac{1}{200000} 20^2$	=166.66
C,	22+20, 162.6 + (6.00 × 0.7)	=166.80

DEMONSTRATION OF EQ. 44.

The general equation of a parabola passing through the point n (Fig. 36) may be written

$$y^2 + y_n^2 = 2p(x + x_n),$$

from which

$$x_n = \frac{y^2}{2p} + \frac{y_n^2}{2p} - x.$$

When $x = x_A$, $y = y_A$, and we have

$$x_n = \frac{y_A^2}{2p} + \frac{y_n^2}{2p} - x_A.$$

The general equation of a tangent passing through the point A may be written

$$yy_A = p(x + x_A),$$

from which

$$x = \frac{yy_A}{p} - x_A.$$

When $x = x_s$, $y = y_s [= y_n]$, and we have

$$x_s = \frac{y_n y_A}{p} - x_A.$$

$$\overline{sn} = x_n - x_s = \frac{y_A^2 + y_n^2 - 2y_n y_A}{2p}$$

$$= \frac{(y_A - y_n)^2}{2p} = \frac{\overline{Am}^2}{2p},$$

$$2p = \frac{y_A^2}{x_A} = \frac{\overline{Ae}^2}{eh}.$$

$$\therefore \overline{sn} = eh \frac{\overline{Am}^2}{\overline{Ae}^2}.$$

This proves the general proposition that if secants are drawn parallel to the axis of x , intersecting a parabola and a tangent to it, the intercepts between the tangent and the parabola are proportional to the square of the distances (measured parallel to y) from the tangent point.

CHAPTER III.

EARTHWORK.

FORM OF EXCAVATIONS AND EMBANKMENTS.

58. Usual form of cross-section in cut or fill. The normal form of cross-section in cut is as shown in Fig. 37, in which $e \dots g$ represents the natural surface of the ground, no matter

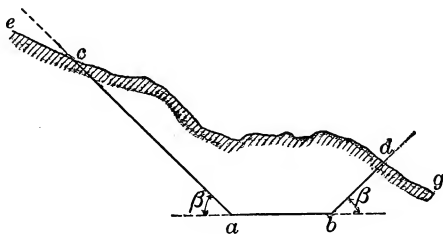


FIG. 37.

how irregular; ab represents the position and width of the required roadbed; ac and bd represent the "side slopes" which begin at a and b and which intersect the natural surface at such

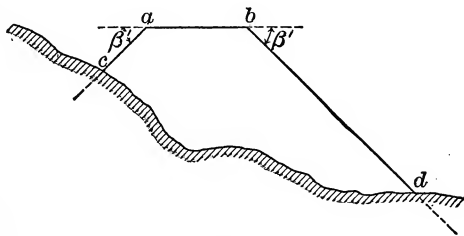


FIG. 38.

points (c and d) as will be determined by the required slope angle (β).

The normal section in fill is as shown in Fig. 38. The points c and d are likewise determined by the intersection of the re-

quired side slopes with the natural surface. In case the required roadbed (ab in Fig. 39) intersects the natural surface, both cut

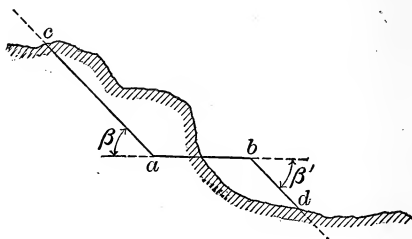


FIG. 39.

and fill are required, and the points c and d are determined as before. Note that β and β' are not necessarily equal. Their proper values will be discussed later.

59. Terminal pyramids and wedges. Fig. 40 illustrates the general form of cross-sections when there is a transition from cut to fill. $a \dots g$ represents the grade line of the road which

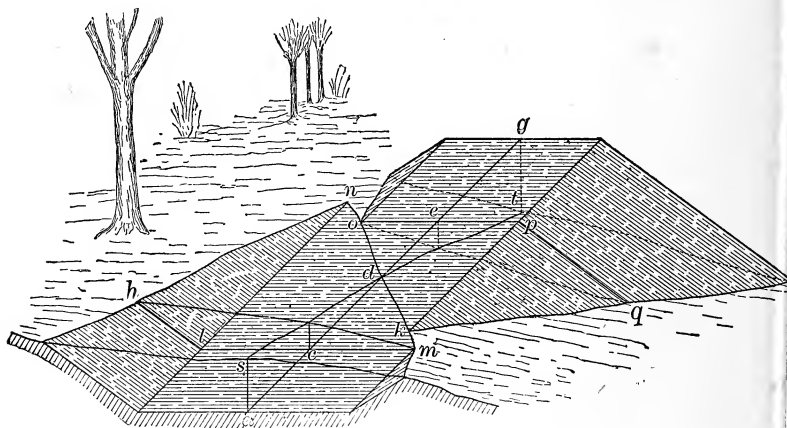


FIG. 40.

passes from cut to fill at d . sdt represents the surface profile. A cross-section taken at the point where either side of the roadbed first cuts the surface (the point m in this case) will usually be triangular if the ground is regular. A similar cross-section should be taken at o , where the other side of the roadbed cuts the surface. In general the earthwork of cut and fill terminates

in two pyramids. In Fig. 40 the pyramid vertices are at n and k , and the bases are lhm and opq . The roadbed is generally wider in cut than in fill, and therefore the section lhm and the altitude ln are generally greater than the section opq and the altitude pk . When the line of intersection of the roadbed and natural surface ($nodkm$) becomes perpendicular to the axis of the roadbed (ag) the pyramids become wedges whose bases are the nearest convenient cross-sections.

60. Slopes. a. Cuttings. The required slopes for cuttings vary from perpendicular cuts, which may be used in hard rock which will not disintegrate by exposure, to a slope of perhaps 4 horizontal to 1 vertical in a soft material like quicksand or in a clayey soil which flows easily when saturated. For earthy materials a slope of 1 : 1 is the maximum allowable, and even this should only be used for firm material not easily affected by saturation. A slope of $1\frac{1}{2}$ horizontal to 1 vertical is a safer slope for average earthwork. It is a frequent blunder that slopes in cuts are made too steep, and it results in excessive work in clearing out from the ditches the material that slides down, at a much higher cost per yard than it would have cost to take it out at first, to say nothing of the danger of accidents from possible landslides.

b. Embankments. The slopes of an embankment vary from 1 : 1 to 1.5 : 1. A rock fill will stand at 1 : 1, and if some care is taken to form the larger pieces on the outside into a rough dry wall, a much steeper slope can be allowed. This method is sometimes a necessity in steep side-hill work. Earthwork embankments generally require a slope of $1\frac{1}{2}$ to 1. If made steeper at first, it generally results in the edges giving way, requiring repairs until the ultimate slope is nearly or quite $1\frac{1}{2}$: 1. The difficulty of incorporating the added material with the old embankment and preventing its sliding off frequently makes these repairs disproportionately costly.

61. Compound sections. When the cut consists partly of earth and partly of rock, a compound cross-section must be made. If borings have been made so that the contour of the rock surface is accurately known, then the true cross-section may be determined. The rock and earth should be calculated separately, and this will require an accurate knowledge of where the rock "runs out"—a difficult matter when it must be deter-

mined by boring. During construction the center part of the earth cut would be taken out first and the cut widened until a sufficient width of rock surface had been exposed so that the rock cut would have its proper width and side slopes. Then the earth slopes could be cut down at the proper angle. A "berm" of about three feet is usually left on the edges of the rock cut as

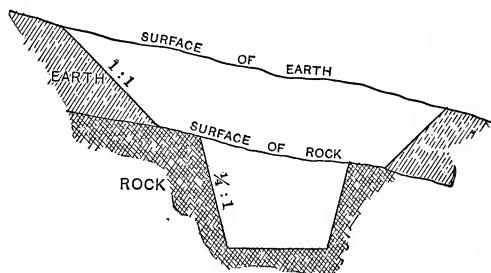


FIG. 41.

a margin of safety against a possible sliding of the earth slopes. After the work is done, the amount of excavation that has been made is readily computable, but accurate preliminary estimates are difficult. The area of the cross-section of earth in the figure must be determined by a method similar to that developed for borrow-pits (see § 89).

62. Width of roadbed. Owing to the large and often disproportionate addition to volume of cut or fill caused by the addition of even one foot to the width of roadbed, there is a natural tendency to reduce the width until embankments become unsafe and cuts are too narrow for proper drainage. The cost of maintenance of roadbed is so largely dependent on the drainage of the roadbed that there is true economy in making an ample allowance for it. The practice of some of the leading railroads of the country in this respect is given in the following table, in which are also given some data belonging more properly to the subject of superstructure.

It may be noted from the table that the average width for an *earthwork* cut, single track, is about 24.7 feet, with a minimum of 19 feet 2 inches. The widths of fills, single track, average over 18 feet, with numerous minimums of 16 feet. The widths for double track may be found by adding the distance between track centers, which is usually 13 feet.

WIDTH OF ROADBED FOR SINGLE AND DOUBLE TRACK—SLOPE RATIOS—DISTANCES BETWEEN TRACK CENTERS.

Road.	Single Track.		Double Track.		Slope Ratios.		Distance Between Track Centers.
	Cut.	Fill.	Cut.	Fill.	Cut.	Fill.	
A., T. & Santa Fé	{ 28' earth 22' rock	20			1 : 1 1 1/4 : 1	1.5 : 1	
Chicago, Burlington & Quincy	14+(2×5)*	16	28+(2×5)	30	1.5 : 1	1.5 : 1	14'
Chicago, Milwaukee & St. Paul	18+(2×6)	20 to 24	31+(2×6)	33 to 37	1.5 : 1	1.5 : 1	13'
C., C. & St. Louis	20+(2×4)	20	33+(2×4)	33	1.5 : 1	1.5 : 1	13'
Illinois Central	32.5	18			1.5 : 1	1.5 : 1	
Erie	20' 8 1/2"	20' 8 1/2"	33' 8 1/2"	33' 8 1/2"	1.5 : 1	1.5 : 1	13'
Lehigh Valley	14+(2×3.5)	16	27+(2×3.5)	30	1 : 1	1.5 : 1	13'
Lake Shore & Michigan Southern			33+(2×7.25)	32	1.5 : 1	1.5 : 1	13'
Louisville & Nashville	13+(2×4.5)	16			1 : 1	1.5 : 1	
Michigan Central			(33+2×2.5)	33	1.5 : 1	1.5 : 1	13'
N. Y., N. H. & H.	{ 21' 2" earth 16' rock	17' 2"	30	30' 2"	1.5 : 1	1.5 : 1	12'
Norfolk & Western			34' 2" earth		1.5 : 1	1.5 : 1	13'
Pennsylvania	19' 2" light traffic	19' 2"	29' rock		1 1/4 : 1 1 1/2 : 1		
Union Pacific	27' 2" heavy "	19' 2"	31' 4" + (2×4)	31' 4"	1.5 : 1	1.5 : 1	12' 2"
	14+(2×3.5)	16			1 : 1	1.5 : 1	

* (2×5) signifies two ditches each 5 feet wide; the following cases should be interpreted similarly.

63. Form of subgrade. Specifications (or the cross-section drawings) formerly required that the subgrade should have a curved form, convex upward, or that it should slope outward from a slight ridge in the center, with the evident purpose of draining to the sides all water which might percolate through the ballast. If the subsoil were hard and impenetrable by the ballast, the method might answer, but experience has shown that, with ordinary subsoils, the ballast immediately under each rail is forced a little deeper into the subsoil by the passage of each train. Periodical retamping of ballast under the ends of the ties, and little or no tamping under the center, only adds to the accumulation under each rail. A cross-section of a very old roadbed will frequently show twice as much depth of ballast under the rails as there is under the center. This method of tamping quickly obliterates the original line of demarcation between ballast and subsoil and any expected improvement in drainage due to sloping subsoil is not realized. Therefore the A.R.E.A. specifications call for *flat* subgrades.

64. Ditches. "The stability of the track depends upon the strength and permanence of the roadbed and structures upon which it rests; whatever will protect them from damage or prevent premature decay should be carefully observed. The worst enemy is WATER, and the further it can be kept away from the track, or the sooner it can be diverted from it, the better the track will be protected. Cold is damaging only by reason of the water which it freezes; therefore the first and most important provision for good track is drainage." (Rules of the Road Department, Illinois Central R. R.)

The form of ditch generally prescribed has a flat bottom 12" to 24" wide and with sides having a minimum slope, except in rock-work, of 1 : 1, more generally 1.5 : 1 and sometimes 2 : 1. Sometimes the ditches are made V-shaped, which is objectionable unless the slopes are low. The best form is evidently that which will cause the greatest flow for a given slope, and this will evidently be the form in which the ratio of area to wetted perimeter is the largest. The semicircle fulfills this condition better than any other form, but the nearly vertical sides would be difficult to maintain. (See Fig. 42.) A ditch, with a flat bottom and such



FIG. 42.

slopes as the soil requires, which approximates to the circular form will therefore be the best.

When the flow will probably be large and at times rapid it will be advisable to pave the ditches with stone, especially if the soil is easily washed away. Six-inch tile drains, placed 2' under the ditches, are prescribed on some roads. (See Fig. 43.) No better method could be devised to insure a dry subsoil. The ditches through cuts should be led off at the end of the cut so that the adjacent embankment will not be injured.

Wherever there is danger that the drainage from the land above a cut will drain down into the cut, a ditch should be made near the edge of the cut to intercept this drainage, and this ditch should be continued, and paved if necessary, to a point where the outflow will be harmless. Neglect of these simple and inexpensive precautions frequently causes the soil to be loosened on the shoulders of the slopes during the progress of a heavy rain, and results in a landslide which will cost more to repair than the ditches which would have prevented it for all time.

Ditches should be formed along the bases of embankments; they facilitate the drainage of water from the embankment, and may prevent a costly slip and disintegration of the embankment.

65. Effect of sodding the slopes, etc. Engineers are unanimously in favor of rounding off the shoulders and toes of embankments and slopes, sodding the slopes, paving the ditches, and providing tile drains for subsurface drainage, all to be put in during original construction. (See Fig. 43.) Some of the highest grade specifications call for the removal of the top layer of vegetable soil from cuts and from under proposed fills to some convenient place, from which it may be afterwards spread on the slopes, thus facilitating the formation of sod from grass-seed. But while engineers favor these measures and their economic value may be readily demonstrated, it is generally impossible to obtain the authorization of such specifications from railroad directors and promoters. The addition to the original cost of the roadbed is considerable, but is by no means as great as the capitalized value of the extra cost of maintenance resulting from the usual practice. Fig. 43 is a copy of

designs * presented at a convention of the American Society of Civil Engineers by Mr. D. J. Whittemore, Past President of the Society and Chief Engineer of the Chi., Mil. & St. Paul

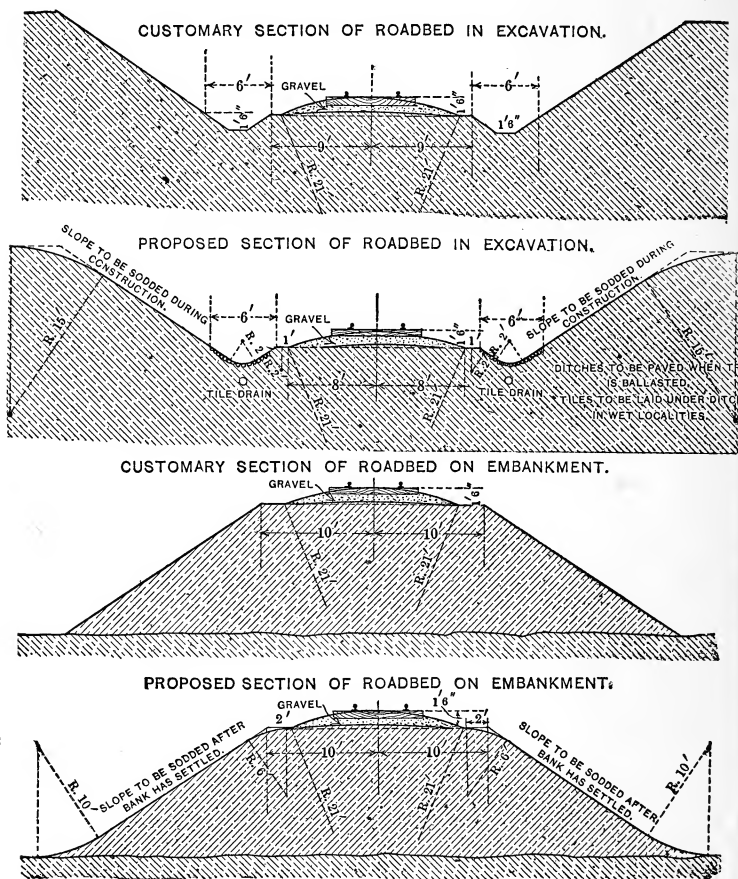


FIG. 43.—“WHITTEMORE ON RAILWAY EXCAVATION AND EMBANKMENTS”
Trans. Am. Soc. C. E., Sept. 1894.

R. R. The “customary sections” represent what is, with some variations of detail, the practice of many railroads. The “pro-

* Trans. Am. Soc. Civil Eng., Sept. 1894.

posed sections" elicited unanimous approval. They should be adopted when not prohibited by financial considerations.

EARTHWORK SURVEYS.

66. Relation of actual volume to the numerical result. It should be realized at the outset that the accuracy of the result of computations of the volume of any given mass of earthwork has but little relation to the accuracy of the mere numerical work. The process of obtaining the volume consists of two distinct parts. In the first place it is assumed that the volume of the earthwork may be represented by a more or less complicated geometrical form, and then, secondly, the volume of such a geometrical form is computed. A desire for simplicity (or a frank willingness to accept approximate results) will often cause the cross-section men to assume that the volume may be represented by a very simple geometrical form which is really only a very rough approximation to the true volume. In such a case, it is only a waste of time to compute the volume with minute numerical accuracy. One of the first lessons to be learned is that economy of time and effort requires that the accuracy of the numerical work should be kept proportional to the accuracy of the cross-sectioning work, and also that the accuracy of both should be proportional to the use to be made of the results. The subject is discussed further in § 94.

67. Prismoids. To compute the volume of earthwork, it is necessary to assume that it has some geometric form whose volume is readily determinable. The general method is to consider the volume as consisting of a series of *prismoids*, which are solids having parallel plane ends and bounded by surfaces which may be formed by lines moving continuously along the edges of the bases. These surfaces may also be considered as the surfaces generated by lines moving along the edges joining the corresponding points of the bases, these edges being the directrices, and the lines being always parallel to either base, which is a plane director. The surfaces thus developed may or may not be planes. The volume of such a prismoid is readily determinable (as explained in § 70 *et seq.*), while its definition is so very general that it may be applied to very rough ground. The "two plane ends" are sections perpendicular to the axis of the road. The roadbed and side slopes (also plane) form three of

ularity of the ground and the accuracy desired more and more "intermediate points" are taken.

The distance (d in Fig. 44) of the roadbed below (or above) the natural surface at the center is known or determined from the profile or by the computed establishment of the grade line. The distances out from the center of all "breaks" are determined with a tape. To determine the elevations for a cut, set up a level at any convenient point so that the line of sight is higher than any point of the cross-section, and take a rod reading on the center point. This rod reading added to d gives the height of the instrument (H. I.) above the roadbed. Subtracting from H. I. the rod reading at any "break" gives the height of that point above the roadbed (h_l, k_l, h_r , etc.). This is true for all cases in excavation. For fill, the rod reading at center minus d equals the H. I., which may be positive or negative. When negative, add to the "H. I." the rod readings of the intermediate points to get their depths below "grade"; when positive, subtract the "H. I." from the rod readings.

The heights or depths of these intermediate points above or below grade need only be taken to the nearest tenth of a foot, and the distances out from the center will frequently be sufficiently exact when taken to the nearest foot. The roughness of the surface of farming land or woodland generally renders useless any attempt to compute the volume with any greater accuracy than these figures would imply unless the form of the ridges and hollows is especially well defined. The position of the slope-stake points is considered in the next section. Additional discussion regarding cross-sectioning is found in § 82.

69. Position of slope-stakes. The slope-stakes are set at the intersection of the required side slopes with the natural surface, which depends on the center cut or fill (d). The distance of the slope-stake from the center for the lower side is $x = \frac{1}{2}b + s(d+y)$; for the up-hill side it is $x' = \frac{1}{2}b + s(d-y')$. s is the "slope ratio" for the side slopes, the ratio of horizontal to vertical. In the above equation both x and y are unknown. Therefore some position must be found by trial which will satisfy the equation. As a preliminary, the value of x for the point $a = \frac{1}{2}b + sd$, which is the value of x for *level* cross-sections. In the case of fills on sloping ground the value of x on the *down-hill* side is *greater* than this; on the *up-hill* side it is *less*. The difference in distance is s times the difference of elevation. Take a

out to within a few tenths at the first trial. The left-hand pages of the note-book should have the station number, surface elevation, grade elevation, center cut or fill, and rate of grade. The right-hand pages should be divided in the center and show the distances out and heights above grade of all points, as is illustrated in § 84. The notes should read *up* the page, so that when looking ahead along the line the figures are in their proper relative position. The "fractions" farthest from the center line represent the slope-stake points.

70. Setting slope-stakes by means of "automatic" slope-stake rods. The equipment consists of a specially graduated tape and a specially constructed rod. The tape may readily be prepared by marking on the *back* side of an ordinary 50-foot tape which is graduated to feet and tenths. Mark "0" at " $\frac{1}{2}b$ " from the tape-ring. Then graduate from the zero backward, at true scale, to the ring. Mark off "feet" and "tenths" on a scale proportionate to the slope ratio. For example, with the usual slope ratio of 1.5:1 each "foot" would measure 18 inches and each "tenth" in proportion.

The rod, 10 feet long, is shod at each end and has an endless tape passing within the shoes at each end and over pulleys—to reduce friction. The tape should be graduated in feet and tenths, from 0 to 20 feet—the 0 and 20 coinciding. By moving the tape so that 0 is at the bottom of the rod—or (practically) so that the 1-foot mark on the tape is one foot above the bottom of the shoe, an index mark may be placed on the back of the rod (say at 15—on the tape) and this readily indicates when the tape is "set at zero."

The method of use may best be explained from the figure and from the explicit rules as stated. The proof is given for two assumed positions of the level.

(1) Set up the level so that it is higher than the "center" and (if possible) higher than both slope-stakes, but not more than a rod-length higher. On very steep ground this may be impossible and each slope-stake must be set by separate positions of the level.

(2) Set the rod-tape at zero (i.e., so that the 15-foot mark on the *back* is at the index mark).

(3) Hold the rod at the center-stake (*B*) and note the reading (n_1 or n_2). Consider n to be always plus; consider d to be plus for cut and minus for fill.

(4) *Raise* the tape on the *face* side of the rod $(n+d)$. Applied literally (and algebraically), when the level is *below* the roadbed (only possible for fill), $(n+d) = (n_2 + (-d_f)) = n_2 - d_f$. This being numerically negative, the tape is *lowered* $(d_f - n_2)$. With level at (1), for fill, $(n+d) = (n_1 + (-d_f)) = (n_1 - d_f)$; this being positive, the tape is raised. With level at (1), for cut, the tape is raised $(n_1 + d_c)$. In every case the effect is the same as if the telescope were set at the elevation of the roadbed.

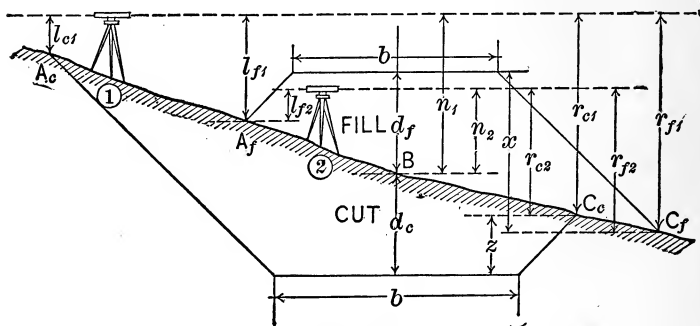


FIG. 45a.

(5) With the special distance-tape, so held that its zero is $\frac{1}{2}b$ from the center, carry the rod out until the rod reading equals the reading indicated by the tape. Since in cut the tape is raised $(n+d)$, the zero of the rod-tape is always higher than the level (unless the rod is held at or below the elevation of the roadbed—which is only possible on side-hill work), and the reading at either slope-stake is necessarily *negative*. The reading for slope-stakes in fill is always positive.

(6) Record the rod-tape reading as the numerator of a fraction and the *actual distance* out (read directly from the *other* side of the distance-tape) as the denominator of the fraction.

Proof. Fill. Level at (1). Tape is raised $(n_1 - d_f)$. When rod is held at C_f , the rod reading is $+x$, which $= r_{f1} - (n_1 - d_f)$. But the reading on the back side of the distance-tape is also x .

Fill. Level at (2). Tape is raised $(n_2 - d_f)$, i.e., it is *lowered* $(d_f - n_2)$. When rod is held at C_f , the rod reading is $+x$, which similarly $= r_{f2} - (n_2 - d_f) = r_{f2} + (d_f - n_2)$. Distance-tape as before.

Cut Level at (1). Tape is raised $(n_1 + d_c)$. When rod is held at C_c the rod reading is $-z$, which $= r_{c1} - (n_1 + d_c)$, i.e., $z = (n_1 + d_c) - r_{c1}$. The distance-tape will read z .

Side-hill work. It is easily demonstrated that the method, when followed literally, may be applied to side-hill work, although there is considerable chance for confusion and error, when, as is usual, $\frac{1}{2}b$ and the slope ratio are different for cut and for fill.

The method appears complicated at first, but it becomes mechanical and a time-saver when thoroughly learned. The advantages are especially great when the ground is fairly level transversely, but decrease when the difference of elevation of the center and the slope-stake is more than the rod length. By setting the rod-tape "at zero," the rod may always be used as an ordinary level rod and the regular method adopted, as in § 69. Many engineers who have thoroughly tested these rods are enthusiastic in their praise as a time-saver.

COMPUTATION OF VOLUME.

71. Prismoidal formula. Let Fig. 46 represent a triangular prismoid. The two triangles forming the ends lie in *parallel* planes, but since the angles of one triangle are not equal to the corresponding angles of the other triangle, at least two of the surfaces must be *warped*. If a section, parallel to the bases, is

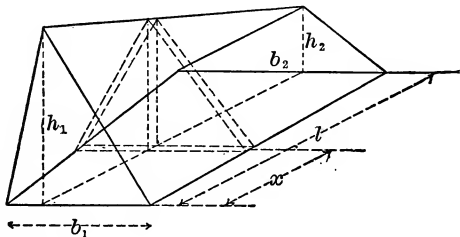


FIG. 46.

made at any point at a distance x from one end, the area of the section will evidently be

$$A_x = \frac{1}{2} b_x h_x = \frac{1}{2} \left[b_1 + (b_2 - b_1) \frac{x}{l} \right] \left[h_1 + (h_2 - h_1) \frac{x}{l} \right].$$

The volume of a section of infinitesimal length will be $A_x dx$, and the total volume of the prismoid will be *

$$\begin{aligned}
 \int_0^l A_x dx &= \frac{1}{2} \int_0^l \left[b_1 + (b_2 - b_1) \frac{x}{l} \right] \left[h_1 + (h_2 - h_1) \frac{x}{l} \right] dx \\
 &= \frac{1}{2} \left[b_1 h_1 x + (b_2 - b_1) h_1 \frac{x^2}{2l} + b_1 (h_2 - h_1) \frac{x^2}{2l} \right. \\
 &\quad \left. + (b_2 - b_1)(h_2 - h_1) \frac{x^3}{3l^2} \right]_0^l \\
 &= \frac{1}{2} \left\{ b_1 h_1 l + [(b_2 - b_1) h_1 + b_1 (h_2 - h_1)] \frac{l}{2} + (b_2 - b_1)(h_2 - h_1) \frac{l}{3} \right\} \\
 &= \frac{l}{2} \left[\frac{1}{3} b_1 h_1 + \frac{1}{6} b_1 h_2 + \frac{1}{6} b_2 h_1 + \frac{1}{3} b_2 h_2 \right] \\
 &= \frac{l}{6} \left[\frac{1}{2} b_1 h_1 + \frac{1}{2} b_1 (h_1 + h_2) + \frac{1}{2} b_2 (h_1 + h_2) + \frac{1}{2} b_2 h_2 \right] \\
 &= \frac{l}{6} \left[\frac{1}{2} b_1 h_1 + 4 \left(\frac{1}{2} \cdot \frac{b_1 + b_2}{2} \cdot \frac{h_1 + h_2}{2} \right) + \frac{1}{2} b_2 h_2 \right] \\
 &= \frac{l}{6} [A_1 + 4A_m + A_2], \quad . \quad . \quad . \quad . \quad . \quad . \quad . \quad . \quad (45)
 \end{aligned}$$

in which A_1 , A_2 , and A_m are the areas respectively of the two bases and of the middle section. Note that A_m is *not* the *mean* of A_1 and A_2 , although it does not necessarily differ very greatly from it.

The above proof is absolutely independent of the values, absolute or relative, of b_1 , b_2 , h_1 , or h_2 . For example, h_2 may be zero and the second base reduces to a line and the prismoid becomes wedge-shaped; or b_2 and h_2 may both vanish, the second base becoming a point and the prismoid reduces to a pyramid. Since every prismoid (as defined in § 67) may be reduced to a combination of triangular prismoids, wedges, and pyramids, and

* Students unfamiliar with the Integral Calculus may take for granted the fundamental formulæ that $\int dx = x$, that $\int x dx = \frac{1}{2} x^2$, and that $\int x^2 dx = \frac{1}{3} x^3$; also that in integrating between the limits of l and 0 (zero), the value of the integral may be found by simply substituting l for x after integration.

since the formula is true for any one of them individually, it is true for all collectively; therefore it may be stated that *

The volume of a prismoid equals one sixth of the perpendicular distance between the bases multiplied by the sum of the areas of the two bases plus four times the area of the middle section.

While it is always possible to compute the volume of any prismoid by the above method, it becomes an extremely complicated and tedious operation to compute the true value of the middle section if the end sections are complicated in form. It therefore becomes a simpler operation to compute volumes by approximate formulæ and apply, if necessary, a correction. The most common methods are as follows:

72. Averaging end areas. The volume of the triangular prismoid (Fig. 46), computed by averaging end areas, is $\frac{l}{2}[\frac{1}{2}b_1h_1 + \frac{1}{2}b_2h_2]$. Subtracting this from the true volume (as given in the equation above Eq. 45), we obtain the correction

$$\frac{l}{12}[(b_1 - b_2)(h_2 - h_1)]. \quad \dots \quad (46)$$

This shows that if either the h 's or b 's are equal, the correction vanishes; it also shows that if the bases are roughly similar and b varies roughly with h (which *usually* occurs, as will be seen later), the correction will be *negative*, which means that the method of averaging end areas *usually* gives *too large* results.

73. Middle areas. Sometimes the middle area is computed and the volume is assumed to be equal to the length times the middle area. This will equal $\frac{l}{2} \times \frac{b_1 + b_2}{2} \times \frac{h_1 + h_2}{2}$. Subtracting this from the true volume, we obtain the correction

$$\frac{l}{24}(b_1 - b_2)(h_1 - h_2). \quad \dots \quad (47)$$

As before, the form of the correction shows that if either the h 's or b 's are equal, the correction vanishes; also under the *usual* conditions, as before, the correction is *positive* and only one-half as large as by averaging end areas. Ordinarily the labor involved in the above method is no less than that of applying the exact prismoidal formula.

* The student should note that the derivation of equation (45) does not complete the proof, but that the statements in the following paragraph are logically necessary for a general proof.

74. Two-level ground. When *approximate* computations of earthwork are sufficiently exact the field-work may be materially reduced by observing simply the center cut (or fill) and the natural slope a , measured with a clinometer. The area of such a section (see Fig. 48) equals

$$\frac{1}{2}(a+d)(x_l+x_r) - \frac{ab}{2}.$$

But

$$x_l \tan \beta = a + d + x_l \tan a,$$

from which

$$x_l = \frac{a+d}{\tan \beta - \tan a}.$$

Similarly,

$$x_r = \frac{a+d}{\tan \beta + \tan a}.$$

Substituting,

$$\text{Area} = (a+d)^2 \frac{\tan \beta}{\tan^2 \beta - \tan^2 a} - \frac{ab}{2} \dots \dots (48)$$

The values a , $\tan \beta$, $\tan^2 \beta$ are constant for all sections, so that it requires but little work to find the area of any section.

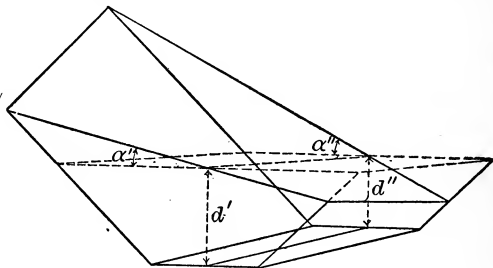


FIG. 47.

As this method of cross-sectioning implies considerable approximation, it is generally a useless refinement to attempt to com-

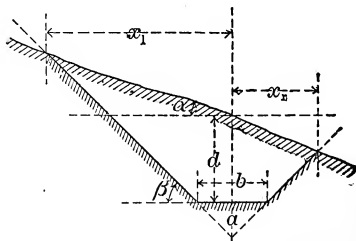


FIG. 48.

pute the volume with any greater accuracy than that obtained by averaging end areas. It may be noted that it may be easily proved that the correction to be applied is of the same form as that found in § 72 and equals

$$\frac{l}{12}[(x_l' + x_r') - (x_l'' + x_r'')][(d'' + a) - (d' + a)],$$

which reduces to

$$\text{Corr.} = \frac{l}{6} \left\{ [(a + d') \frac{\tan \beta}{\tan^2 \beta - \tan^2 \alpha'} - (a + d'') \frac{\tan \beta}{\tan^2 \beta - \tan^2 \alpha''}] [d'' - d'] \right\}. \quad (49)$$

When $d'' = d'$ the correction vanishes. This shows that when the center heights are equal there is no correction—regardless of the slope. If the slope is uniform throughout, the form of the correction is simplified and is invariably *negative*. Under the usual conditions the correction is *negative*, i.e., the method *generally* gives *too large* results.

75. Level sections. When the country is very level or when only approximate preliminary results are required, it is sometimes assumed that the cross-sections are level. The method of level sections is capable of easy and rapid computation. The area may be written as

$$(a + d)^2 s - \frac{ab}{2} \dots \dots \dots (50)$$

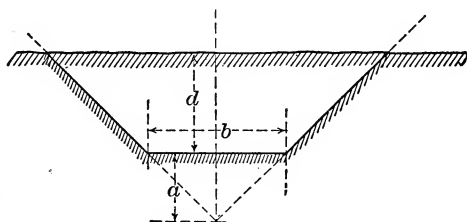


FIG. 49.

This also follows from Eq. 48 when $\alpha = 0$ and $\tan \beta = \frac{1}{s}$. s here represents the "slope ratio," i.e., the ratio of the horizontal projection of the slope to the vertical. A table is very readily formed giving the area in square feet of a section of given depth and for any given width of roadbed and ratio of side-slopes.

The area may also be readily determined (as illustrated in the following example) without the use of such a table; a table of squares will facilitate the work. Assuming the cross-sections at equal distances ($=l$) apart, the total approximate volume for any distance will be

$$\frac{l}{2}[A_0 + 2(A_1 + A_2 + \dots A_{n-1}) + A_n]. \quad (51)$$

The prismoidal correction may be directly derived from Eq. 46 as $\frac{l}{12}[2(a+d')s - 2(a+d'')s][(a+d'') - (a+d')]$, which reduces to

$$-\frac{ls}{6}(d' - d'')^2 \quad \text{or} \quad -\frac{l}{12} \frac{b}{a}(d' - d'')^2. \quad (52)$$

This may also be derived from Eq. 49, since $\alpha = 0$, $\tan \alpha = 0$, and $\tan \beta = 2a \div b$. This correction is *always* negative, showing that the method of averaging end areas, when the sections are level, always gives too large results. The prismoidal correction for any one prismoid is therefore a constant times the *square* of a difference. The squares are always positive whether the differences are positive or negative. The correction therefore becomes

$$-\frac{l}{12} \frac{b}{a} \Sigma (d' \sim d'')^2. \quad (53)$$

76. Numerical example: level sections. Given the following center heights for the same number of consecutive stations 106 feet apart; width of roadbed 18 feet; slope $1\frac{1}{2}$ to 1.

The products in the fifth column may be obtained very readily and with sufficient accuracy by the use of the slide-rule described in § 79. The products should be considered as $(a+d)(a+d) \div \frac{1}{s}$. In this problem $s = 1\frac{1}{2}$, $\frac{1}{s} = .6667$. To apply the rule to the first case above, place 6667 on scale *B* over 89 on scale *A*, then opposite 89 on scale *B* will be found 118.8 on scale *A*. The position of the decimal point will be evident from an approximate mental solution of the problem.

The true volume, according to the prismoidal formula, of a length of the road measured in this way will be

$$\frac{l}{6} \left[\frac{x_l' x_r'}{s} - \frac{ab}{2} + 4 \left(\frac{x_l' + x_l''}{2} \frac{x_r' + x_r''}{2} \frac{1}{s} - \frac{ab}{2} \right) + \frac{x_l'' x_r''}{s} - \frac{ab}{2} \right].$$

If computed by averaging end areas, the approximate volume will be

$$\frac{l}{2} \left[\frac{x_l' x_r'}{s} - \frac{ab}{2} + \frac{x_l'' x_r''}{s} - \frac{ab}{2} \right].$$

Subtracting this result from the true volume, we obtain as the correction

$$\text{Correction} = \frac{l}{6s} (x_l'' - x_l') (x_r' - x_r''). \quad . \quad . \quad (55)$$

This shows that if the side distances to either the right or left are equal at adjacent stations the correction is *zero*, and also that if the difference is small the correction is also small and very probably within the limit of accuracy obtainable by that method of cross-sectioning. In fact, as has already been shown in the latter part of § 75, it will usually be a useless

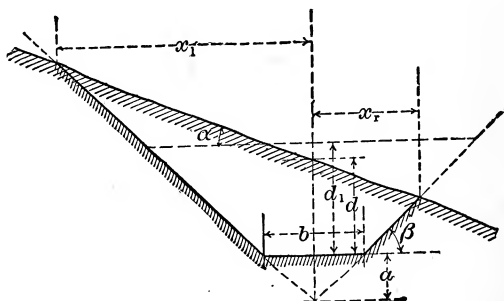


FIG. 50.

refinement to compute the prismoidal correction when the method of cross-sectioning is as rough and approximate as this method generally is.

78. Equivalent level sections. These sloping "two-level" sections are sometimes transformed into "level sections of equal

area," and the volume computed by the method of level sections (§ 75). But the true volume of a prismoid with sloping ends does not agree with that of a prismoid with equivalent bases and level ends except under special conditions, and when this method is used a correction must be applied if accuracy is desired, although, as intimated before, the assumption that the sections have uniform slopes will frequently introduce greater inaccuracies than that of this method of computation. The following demonstration is therefore given to show the scope and limitations of the errors involved in this much used method.

In Fig. 50, let d_1 be the center height which gives an equivalent level section. The area will equal $(a+d_1)^2s - \frac{ab}{2}$, which must equal the area given in § 77, $\frac{x_l x_r}{s} - \frac{ab}{2}$. $s = \frac{b}{2a}$.

$$\therefore (a+d_1)^2s = \frac{x_l x_r}{s},$$

$$\text{or } a+d_1 = \frac{\sqrt{x_l x_r}}{s}. \quad \dots \dots \dots (56)$$

To obtain d_1 directly from notes, given in terms of d and α , we may substitute the values of x_l and x_r given in § 74, which gives

$$a+d_1 = (a+d) \frac{\tan \beta}{\sqrt{\tan^2 \beta - \tan^2 \alpha}} = \frac{a+d}{\sqrt{1-s^2 \tan^2 \alpha}}. \quad \dots (57)$$

The *true* volume of the equivalent section may be represented by

$$\frac{ls}{6} \left[(a+d_1')^2 + 4 \left(\frac{a+d_1'}{2} + \frac{a+d_1''}{2} \right)^2 + (a+d_1'')^2 \right].$$

From this there should be subtracted the volume of the "grade prism" under the roadbed to obtain the volume of the cut that would be actually excavated, but in the following comparison, as well as in other similar comparisons elsewhere made, the volume of the grade prism invariably cancels out, and so for the sake of simplicity it will be disregarded. This expression for volume may be transposed to

$$\frac{ls}{6} \left[\frac{x_l' x_r'}{s^2} + 4 \left(\frac{\sqrt{x_l' x_r'}}{2s} + \frac{\sqrt{x_l'' x_r''}}{2s} \right)^2 + \frac{x_l'' x_r''}{s^2} \right].$$

The true volume of the prismoid with sloping ends is (see § 77)

$$\frac{l}{6} \left[\frac{x_l' x_r'}{s} + 4 \left(\left(\frac{x_l' + x_l''}{2} \right) \left(\frac{x_r' + x_r''}{2} \right) \frac{1}{s} \right) + \frac{x_l'' x_r''}{s} \right].$$

The difference of the two volumes

$$\begin{aligned} &= \frac{l}{6s} (x_l' x_r' + x_l'' x_r' + x_l' x_r'' + x_l'' x_r'' - x_l' x_r') \\ &\quad - 2\sqrt{x_l' x_r' x_l'' x_r''} - x_l'' x_r'' \\ &= \frac{l}{6s} (\sqrt{x_l' x_r''} - \sqrt{x_l'' x_r'})^2. \dots \dots \dots (58) \end{aligned}$$

This shows that "equivalent level sections" do *not* in general give the true volume, there being an exception when $x_l' x_r'' = x_l'' x_r'$. This condition is fulfilled when the slope is uniform, i.e., when $a' = a''$. When this is nearly so the error is evidently not large. On the other hand, if the slopes are inclined in opposite directions the error may be very considerable, particularly if the angles of slope are also large.

79. Three-level sections. The next method of cross-sectioning in the order of complexity, and therefore in the order of

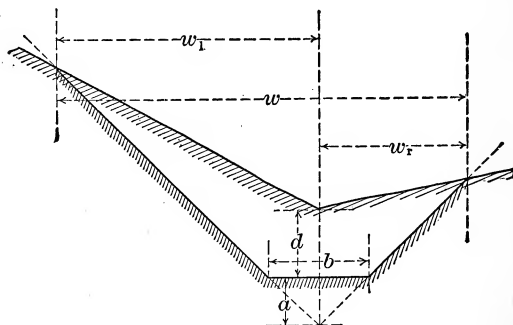


FIG. 51.

accuracy, is the method of three-level sections. The area of the section is $\frac{1}{2}(a+d)(w_r+w_l) - \frac{ab}{2}$, which may be written

$\frac{1}{2}(a+d)w - \frac{ab}{2}$, in which $w = w_r + w_l$. If the volume is computed by averaging end areas, it will equal

$$\frac{l}{4}[(a+d')w' - ab + (a+d'')w'' - ab]. \quad (59)$$

If we divide by 27 to reduce to cubic yards, we have, when $l=100$,

$$\text{Vol } (, \dots) = \frac{25}{27}(a+d')w' - \frac{25}{27}ab + \frac{25}{27}(a+d'')w'' - \frac{25}{27}ab$$

For the next section

$$\text{Vol } (, \dots) = \frac{25}{27}(a+d'')w'' - \frac{25}{27}ab + \frac{25}{27}(a+d''')w''' - \frac{25}{27}ab.$$

For a partial station length compute as usual and multiply result by $\frac{\text{length in feet}}{100}$. The prismoidal correction may be

obtained by applying Eq 46 to each side in turn. For the left side we have

$$\frac{l}{12}[(a+d') - (a+d'')](w_l'' - w_l'), \text{ which equals}$$

$$\frac{l}{12}(d' - d'')(w_l'' - w_l').$$

For the right side we have, similarly,

$$\frac{l}{12}(d' - d'')(w_r'' - w_r').$$

The total correction therefore equals

$$\begin{aligned} & \frac{l}{12}(d' - d'')[(w_l'' + w_r'') - (w_l' + w_r')] \\ &= \frac{l}{12}(d' - d'')(w'' - w'). \end{aligned}$$

Reduced to cubic yards, and with $l=100$,

$$\text{Pris. Corr.} = \frac{25}{81}(d' - d'')(w'' - w'). \quad (60)$$

When this result is compared with that given in Eq. 55 there is an apparent inconsistency. If two-level ground is considered as but a special case of three-level ground, it would seem as if

the same laws should apply. If, in Eq. 55, $x_r' = x_r''$, and x_l'' is different from x_l' , the equation reduces to zero; but in this case d' would also be different from d'' ; and since $x_l' + x_r' = w'$, and $x_l'' + x_r'' = w''$ in Eq. 60, $w'' - w'$ would not equal zero and the correction would be some finite quantity and not zero. The explanation lies in the difference in the form and volume of the prismoids, according to the method of the formation of the warped surfaces. If the surface is supposed to be generated by the locus of a line moving parallel to the ends as plane directors and along two *straight* lines lying in the side slopes, then $x_l^{\text{mid.}}$ will equal $\frac{1}{2}(x_l' + x_l'')$, and $x_r^{\text{mid.}}$ will equal $\frac{1}{2}(x_r' + x_r'')$, but the profile of the center line will *not* be straight and $d^{\text{mid.}}$ will *not* equal $\frac{1}{2}(d' + d'')$. On the other hand, if the surfaces be generated by *two* lines moving parallel to the ends as plane directors and along a *straight* center line and straight side lines lying in the slopes, a warped surface will be generated each side of the center line, which will have uniform slopes on each side of the center at the two ends and *nowhere else*. This shows that when the upper surface of earthwork is warped (as it generally is), two-level ground should not be considered as a special case of three-level ground. This discussion, however, is only valuable to explain an apparent inconsistency and error. The method of two-level ground should only be used when such refinements as are here discussed are of no importance as affecting the accuracy.

An example is given on the opposite page to illustrate the method of three-level sections.

In the first column of yards

$$\begin{aligned} 210 &= \frac{2.5}{81}(a+d)w = \frac{2.5}{81} \times 7.3 \times 31.1; \\ 507, 734, \text{ etc.}, &\text{ are found similarly;} \\ 595 &= 210 - 61 + 507 - 61; \\ 448 &= \frac{4.0}{100}(507 - 61 + 734 - 61); \\ 602 &= \frac{6.0}{100}(734 - 61 + 392 - 61); \\ 449 &= 392 - 61 + 179 - 61. \end{aligned}$$

For the prismoidal correction,

$$\begin{aligned} -20 &= \frac{2.5}{81}(d' - d'')(w'' - w') = \frac{2.5}{81}(2.6 - 8.1)(42.8 - 31.1) \\ &= \frac{2.5}{81}(-5.5)(+11.7). \end{aligned}$$

For the next line, $-3 = \frac{4.0}{100}[\frac{2.5}{81}(-2.6)(+8.7)]$, and similarly for the rest. The "*F*" in the columns of center heights, as well

Station.	Center.	Left.	Right.	$a+d$	w	Yards.	$d'-d''$	$w''-w'$	Pris. Corr.	$x_l \sim x_r$	$V(x_l \sim x_r) / 3R$	Curv. Corr.*	
17	2.6F	$\frac{10.6F}{22.9}$	$\frac{0.8F}{8.2}$	7.3	31.1	210				14.7	+1		
18	8.1F	$\frac{15.8F}{30.7}$	$\frac{3.4F}{12.1}$	12.8	42.8	507	-5.5	+11.7	-20	18.6	+3	+4	
+ 40	10.7F	$\frac{20.2F}{37.3}$	$\frac{4.8F}{14.2}$	15.4	51.5	734	-2.6	+ 8.7	- 3	23.1	+6	+4	
19	6.4F	$\frac{14.0F}{28.0}$	$\frac{2.1F}{10.1}$	11.1	38.1	392	+4.3	-13.4	-11	17.9	+2	+5	
20	3.7F	$\frac{5.8F}{15.7}$	$\frac{0.2F}{7.3}$	8.4	23.0	179	+2.7	-15.1	-13	8.4	+1	+3	
Roadbed, 14' wide in fill.										Approx. Vol. = 2094		-47	+16

$$a = \frac{b}{2s} = \frac{14}{3} = 4.7;$$

$$\frac{25}{27}ab = 61.$$

*For the derivation of the curvation correction, see § 93.

as in the columns of "right" and "left," are inserted to indicate *fill* for all those points. Cut would be indicated by "C."

80. Computation of products. The quantities $\frac{25}{27}(a+d)w$ and $\frac{25}{27}ab$ represent in each case the product of two variable terms and a constant. These products are sometimes obtained from tables which are calculated for all ordinary ranges of the variable terms as arguments. A similar table computed for $\frac{25}{81}(d'-d'')(w''-w')$ will assist similarly in computing the prismoidal correction. Prof. Charles L. Crandall, of Cornell University, is believed to be the first to prepare such a set of tables, which were first published in 1886 in "Tables for the Computation of Railway and Other Earthwork." Another easy method of obtaining these products is by the use of a slide-rule. A slide-rule has been designed by the author to accompany this volume.* It is designed particularly for this special work, although it may be utilized for many other purposes for which slide-rules are valuable. To illustrate its use, suppose $(a+d)=28.2$, and $w=62.4$; then

$$\frac{25}{27}(a+d)w = \frac{28.2 \times 62.4}{1.08}.$$

Set 108 (which, being a constant of frequent use, is specially marked) on the sliding scale (*B*) opposite 282 on the other scale (*A*), and then opposite 624 on scale *B* will be found 1629 on scale *A*, the 162 being read directly and the 9 read by estimation. Although strict rules may be followed for pointing off the final result, it only requires a very simple mental calculation to know that the result must be 1629 rather than 162.9 or 16290. For products less than 1000 cubic yards the result may be read directly from the scale; for products between 1000 and 5000 the result may be read directly to the nearest 10

* The first edition of this book was octavo, and a pasteboard slide-rule, especially marked, accompanied each volume. Cutting down the size of the pages to "pocket size" prevents the incorporation of the rule with the present edition. Any slide-rule with a logarithmic unit $22\frac{1}{2}$ inches long will do equally well provided that the 108 mark is specially distinguished for ready use in computing the volume and that the 324 mark is similarly distinguished for use in computing the prismoidal correction.

yards, and the tenths of a division estimated. Between 5000 and 10000 yards the result may be read directly to the nearest 20 yards, and the fraction estimated; but prisms of such volume will never be found as simple triangular prisms—at least, an assumption that any mass of ground was as regular as this would probably involve more error than would occur from faulty estimation of fractional parts. Facilities for reading as high as 10000 cubic yards would not have been put on the scale except for the necessity of finding such products as $\frac{2}{3}\frac{5}{7}(9.1 \times 9.5)$, for example. This product would be read off from the same part of the rule as $\frac{2}{3}\frac{5}{7}(91 \times 95)$. In the first case the product (80.0) could be read directly to the nearest .2 of a cubic yard, which is unnecessarily accurate. In the other case, the product (8004) could only be obtained by estimating $\frac{4}{10}$ of a division.

The computation for the prismoidal correction may be made similarly except that the divisor is 3.24 instead of 1.08. For example, $\frac{2}{3}\frac{5}{7}(5.5 \times 11.7) = \frac{5.5 \times 11.7}{3.24}$. Set the 324 on scale *B* (also specially marked like 108) opposite 55 on scale *A*, and proceed as before.

81. Five-level sections. Sometimes the elevations over each edge of the roadbed are observed when cross-sectioning. These are distinctively termed “five-level sections.” If the center, the slope-stakes, and *one* intermediate point on each side (*not* necessarily over the edge of the roadbed) are observed, it is termed an “irregular section.” The field-work of cross-sectioning five-level sections is no less than for irregular sections with one intermediate point; the computations, although capable of peculiar treatment on account of the location of the intermediate point, are no easier, and in some respects more laborious; the cross-sections obtained will not in general represent the actual cross-sections as truly as when there is perfect freedom in locating the intermediate point; as it is generally inadvisable or unnecessary to employ five-level sections throughout the length of a road, the change from one method to another adds a possible element of inaccuracy and loses the advantage of uniformity of method, particularly in the notes and *form* of computations. On these accounts the method will not be further developed, except to note that this case, as well as any other, may be solved by dividing the whole prismoid into triangular prismoids, computing the volume by averaging end areas, and computing

the prismoidal correction by adding the computed corrections for each elementary triangular prismoid.

82. Irregular sections. In cross-sectioning irregular sections, the distance from the center and the elevation above "grade" of every "break" in the cross-section must be observed. The area of the irregular section may be obtained by computing the area of the trapezoids (*five*, in Fig. 52 and subtracting the two external triangles. For Fig. 52 the area would be

$$\begin{aligned} \frac{h_l + k_l}{2}(x_l - y_l) + \frac{k_l + d}{2}y_l + \frac{d + j_r}{2}z_r + \frac{j_r + k_r}{2}(y_r - z_r) \\ + \frac{k_r + h_r}{2}(x_r - y_r) - \frac{h_l}{2}\left(x_l - \frac{b}{2}\right) - \frac{h_r}{2}\left(x_r - \frac{b}{2}\right). \end{aligned}$$

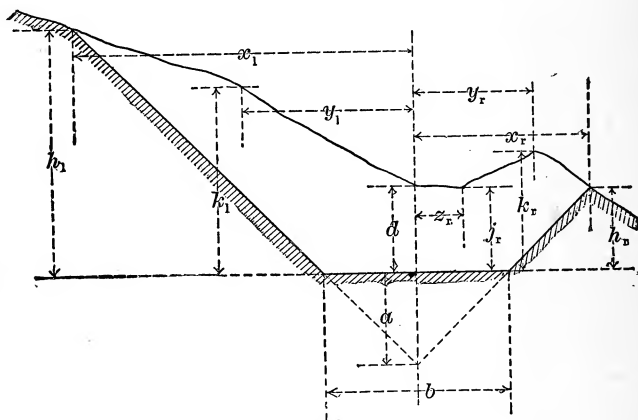


FIG. 52.

Expanding this and collecting terms, of which many will cancel, we obtain

$$\begin{aligned} \text{AREA} = \frac{1}{2} \left[x_l k_l + y_l (d - h_l) + x_r k_r + y_r (j_r - h_r) \right. \\ \left. + z_r (d - k_r) + \frac{b}{2} (h_l + h_r) \right]. \quad . \quad . \quad . \quad (61) \end{aligned}$$

An examination of this formula will show a perfect regularity in its formation which will enable one to write out a similar formula for any section, no matter how irregular or how

many points there are, without any of the preliminary work. The formula may be expressed in words as follows:

AREA equals one-half the sum of products obtained as follows :

the distance to each slope-stake times the height above grade of the point next inside the slope-stake;

the distance to each intermediate point in turn times the height of the point just inside minus the height of the point just outside;

finally, one-half the width of the roadbed times the sum of the slope-stake heights.

If one of the sides is perfectly regular from center to slope-stake, it is easy to show that the rule holds literally good. The "point next inside the slope-stake" in this case is the center; the intermediate terms for that side vanish. The *last term* must always be used. The rule holds good for three-level sections, in which case there are three terms, which may be reduced to two. Since these two terms are both variable quantities for each cross-section, the special method, given in § 78, in which one term $\left(\frac{ab}{2}\right)$ is a constant for all sections, is preferable. In the general method, each intermediate "break" adds another term.

83. Volume of an irregular prismoid. This is obtained by computing first the approximate volume by "averaging end areas" or by multiplying the length by the half sum of the end areas, as computed from Eq. (61). In other words, the Approx. volume = $\frac{100}{27} \times \frac{1}{2}$ (area' + area''). But since each area equals *one-half* the sum of products of width times height (see Eq. (61)) we may say that

$$\text{Approx. volume} = \frac{25}{27} (\text{summation of width times height}) \quad (62)$$

the terms of width times height being like those found within the bracket of Eq. (61).

As before, for partial station lengths, multiply the result by (length in feet $\div$ 100). There will be no constant subtractive term, $\frac{25}{27} ab$, as in § 79.

The correction to this approximate volume is found by considering that *for the purpose of this correction only* the end sections are considered as "three-level" sections and the correction is computed by applying Eq. (60).

84. Numerical example; irregular sections; volume with approximate prismoidal correction. Assume the earthwork notes as given below, where the roadbed is 18 feet wide in cut and the slope is $1\frac{1}{2}$ to 1. Note that the stations read up the page and that when the surveyor is looking ahead along the line the several combinations of *heights* and *distances out* have approximately the same relative position on the note book as they have on the ground. For example, beginning at the bottom line (Sta. 16) the combination $\frac{8.9c}{22.4}$ means that the extreme left-hand point of that section (the "slope stake") is 22.4 feet horizontally from the center and that it is 8.9 feet above the required roadbed. The cut (*c*) would be 8.9 feet to reach the roadbed, but of course the actual cutting is zero at the slope stake. The next point is 12.0 feet horizontally from the center and 7.6 feet above the roadbed. The cut at the center is 6.8 feet. The combinations of dimensions on the right-hand side are to be interpreted similarly.

Sta.	Center $\left\{ \begin{array}{l} \text{cut} \\ \text{or} \\ \text{fill.} \end{array} \right.$	Left.			Right.	
19	0.6c	$\frac{3.6c}{14.4}$			$\frac{0.1c}{4.2}$	$\frac{0.4c}{9.6}$
18	2.3c	$\frac{4.2c}{15.3}$	$\frac{6.8c}{8.4}$	$\frac{3.2c}{5.2}$		$\frac{1.2c}{10.8}$
17	7.6c	$\frac{8.2c}{21.3}$	$\frac{10.2c}{17.4}$	$\frac{8.0c}{6.1}$		$\frac{4.2c}{15.3}$
+42	10.2c	$\frac{12.2c}{27.3}$		$\frac{12.6c}{8.2}$	$\frac{6.2c}{7.5}$	$\frac{8.4c}{21.6}$
16	6.8c	$\frac{8.9c}{22.4}$		$\frac{7.6c}{12.0}$	$\frac{3.2c}{4.1}$	$\frac{2.6c}{12.9}$

The numerical computation is greatly facilitated by a systematic form as given below. For Sta. 16, the first term is "the distance to the left slope stake" (22.4) times "the height above grade of the point next inside" (the height being 7.6), and we place this pair of figures in the columns of "width" and "height." The "distance to the point next inside" is 12.0 and the "height of the point just inside (6.8) minus the

height of the point just outside" (8.9) equals (-2.1) and these are the next pair of widths and heights. Taking $\frac{25}{27}$ of the product of each pair of numbers we have the numbers in the first column of "yards." The sum of all these numbers in the first and second groups multiplied by $\frac{42}{100}$ (that section being only 42 feet long) equals 378 cubic yards, the volume by averaging end areas. The determination of center heights and total widths and the application of Eq. (60), to obtain the approximate prismoidal correction, is self-evident.

VOLUME OF IRREGULAR PRISMOID, WITH APPROXIMATE PRISMOIDAL CORRECTION.

Sta.	W'th	H'ght	Yards.		Cen. Height.	Total width	$d' - d''$	$w'' - w'$	Approx. pris. corr.
16	22.4	7.6	158		+6.8	35.3			
	12.0	-2.1	-23						
	12.9	3.2	40						
	4.1	4.2	16						
	9.0	11.5	96						
+42	27.3	12.6	319		+10.2	48.9	-3.4	+13.6	-14
	8.2	-2.0	-15						
	21.6	6.2	124						
	7.5	1.8	13						
	9.0	20.6	172	378					
17	21.3	10.2	201		+ 7.6	36.6	+2.6	-12.3	-10
	17.4	-0.2	- 3						
	6.1	-2.6	-14						
	15.3	7.6	107						
	9.0	12.4	103	584					
18	15.3	6.8	95		+ 2.3	26.1	+5.3	-10.5	-17
	8.4	-1.0	- 7						
	5.2	-4.5	-22						
	10.8	2.3	23						
	9.0	5.4	45	528					
19	14.4	0.6	8		+ 0.6	24.0	+1.7	-2.1	-1
	9.6	0.1	1						
	4.2	0.2	1						
	9.0	4.0	33	177					

Approx. volume = 1667

-30

Approx. pris. corr. = -30

Corrected volume = 1637 cubic yards

85. Magnitude of the probable error of this method. In previous editions of this work, methods were given for computing the mathematically exact volume of a prismoid whose ends coincide with the "irregular sections" as measured, and

whose upper surfaces are assumed to coincide with the actual surface of the ground. As in the previous methods, the "approximate volume" is computed by averaging end areas and then a correction is applied. If the end sections have the same number of intermediate points on each side, and if it can be assumed that the corresponding lines in each section are connected by plane or warped surfaces, which coincide with the surface of the ground, then the mathematically exact or "true" correction may be obtained by dividing the volume into elementary triangular prismoids, finding the correction for each and adding the results. Although such a method appears very complicated, it is readily possible to develop a law by means of which the true prismoidal correction may be written out (similarly to writing out the formula for the area, Eq. (61)) without any preliminary calculation. Such a law has a mathematical fascination, but it should be remembered that when the ground surface is so broken up that the cross-sections are "irregular" it is in general correspondingly rough and irregular between the cross-sections, especially when those sections are 100 feet apart. It is also true that the cross-sections do *not* usually have the same number of intermediate points on corresponding sides of the center. In such a case, unless the actual form of the ground between the cross-sections is observed and measured, the exact method cannot be used. An extra point in one cross-section implies an extra ridge (or hollow) which "runs out" or disappears by the time the adjoining section is reached. Theoretically a cross-section should be taken at the point where such a ridge or hollow runs out. In general this point will not be at an even 100-foot station. The attempt to compute the exact prismoidal correction usually gives merely a false appearance of extreme accuracy to the work which is not justified by the results. It should not be forgotten that it is readily possible to spend an amount of time on the surveying and computing which is worth more than the few cubic yards of earth which represents the additional accuracy of the more precise method. The accuracy of the office computation should be kept proportionate to the accuracy of the cross-sectioning in the field. The discussion of the magnitude of the prismoidal correction in §§ 72-79 shows that it is small except when the two ends of the prismoid are very dissimilar. The dissimilarity between the two ends of a prismoid would be substantially the

same whether the ends were actually "irregular" or had "three-level" sections, which for each end had the same slope stakes and center heights as the irregular sections. Experience proves that the approximate prismoidal correction, computed by considering the ground as three-level, is so nearly equal to the true prismoidal correction that the difference is perhaps no greater than the *probable* difference between the true volume of earth and the volume of the geometrical prismoid which is assumed to represent that volume. The experienced surveyor will take his cross-sections at such places and so close together that the warped surfaces joining the sections will lie very nearly in the surface or at least will so average the errors that they will substantially neutralize each other.

86. Numerical illustration of the accuracy of the approximate rule. The "true" prismoidal correction for the numerical case given in § 84 was computed by the method outlined above, and on the basis of certain figures as to the vanishing of the ridges and valleys found in one section and not found in the adjacent sections. The various quantities for the volumes between the cross-sections have been tabulated as shown.

Sections.	1 Approx. vol. by averaging end areas.	2 True pris- moidal correction.	3 True volume.	4 Approx. pris. corr. on basis of three-level ground.	5 Error; Col. 4 - col. 2.	6 Approx. vol. computed from center and side heights <i>only</i> .	7 Error; Col. 6 - col. 3.
16.....16+42	378	- 5	373	- 6	- 1	396	- 23
16+42..17	584	- 3	581	- 6	- 3	577	+ 4
17.....18	528	- 16	512	- 17	- 1	463	+ 49
18.....19	177	- 3	174	- 1	+ 2	147	+ 27
	1667	- 27	1640	- 30	- 3	1583	+ 57

There has also been shown in the last two columns the error involved if the "intermediate points" had been ignored in the cross-sectioning. From the tabular form we may learn that

1. The *differences* between the "true" and approximate corrections is so small that it is *probably* swallowed up by errors resulting from inaccurate cross-sectioning.

2. The error which would have been involved in ignoring the intermediate points is so very large in comparison with

the other corresponding errors that (although it proves nothing absolutely definite, being an individual case) the *probabilities* of the relative error from these sources are clearly indicated.

87. Cross-sectioning irregular sections. The slope stake should preferably be determined first, and then the "breaks" between the slope stake and the center. When, as is usual, the ground is not even between the cross-section just taken and the section at the next 100-foot station, a point should be selected for a cross-section such that the lines to the previous section should coincide with the actual surface of the ground as closely as the accuracy of the work demands. § 94 gives a numerical illustration of the magnitude of some of these errors. Although it is possible for a skillful surveyor to so choose his cross-sections in rough and irregular ground that the positive and negative errors will nearly balance, it requires exceptional skill. Frequently the work may be simplified by computing separately the volume of a mound or pit, the existence of which has been ignored in the regular cross-sectioning.

88. Side-hill work. When the natural slope cuts the roadbed there is a necessity for both cut and fill at the same cross-section.

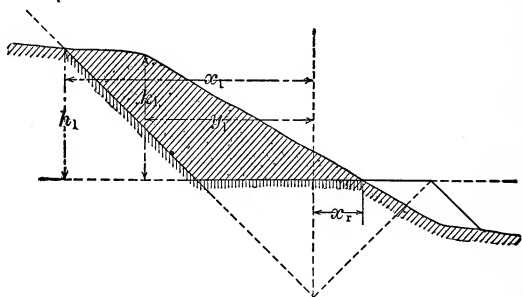


FIG. 53.

When this occurs the cross-sections of both cut and fill are often so nearly triangular that they may be considered as such without great error, and the volumes may be computed separately as triangular prismoids without adopting the more elaborate form of computation so necessary for complicated irregular sections. When the ground is too irregular for this the best plan is to

follow the uniform system. In computing the cut, as in Fig. 53, the left side would be as usual; there would be a small center cut and an ordinate of zero at a short distance to the right of the center. Then, *ignoring the fill*, and applying Eq. 61 strictly, we have two terms for the left side, one for the right, and the term involving $\frac{1}{2}b$, which will be $\frac{1}{2}bh_l$ in this case, since $h_r=0$, and the equation becomes

$$\text{Area} = \frac{1}{2}[x_l k_l + y_l(d - h_l) + x_r d + \frac{1}{2}bh_l].$$

The area for fill may also be computed by a strict application of Eq. 61, but for Fig. 54 all distances for the left side are zero and the elevation for the first point out is zero. d also must be

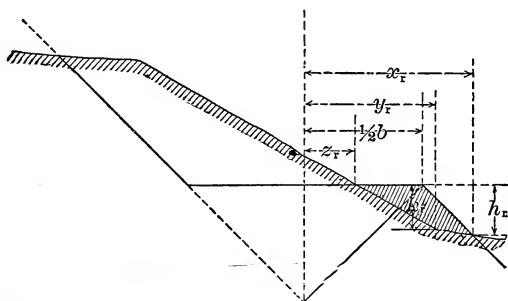


FIG. 54.

considered as zero. Following the rule, § 81, literally, the equation becomes

$$\text{Area}_{\text{(Fill)}} = \frac{1}{2}[x_r k_r + y_r(o - h_r) + z_r(o - k_r) + \frac{1}{2}b(o + h_r)],$$

which reduces to

$$\frac{1}{2}[x_r k_r - y_r h_r - z_r k_r + \frac{1}{2}bh_r].$$

(Note that x_r , h_r , etc., have different significations and values in this and in the preceding paragraphs.) The "terminal pyramids" illustrated in Fig. 40 are instances of side-hill work for very short distances. Since side-hill work always implies *both* cut and fill at the same cross-section, whenever either the cut or fill disappears and the earthwork becomes wholly cut or wholly fill, that point marks the end of the "side-hill work," and a cross-section should be taken at this point.

89. Borrow-pits. The cross-sections of borrow-pits will vary not only on account of the undulations of the surface of the

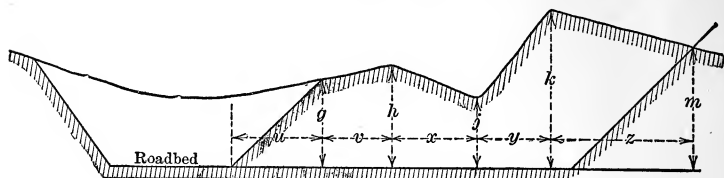


FIG. 55.

ground, but also on the sides, according to whether they are made by widening a convenient cut (as illustrated in Fig. 55) or simply by digging a pit. The sides should always be properly sloped and the cutting made cleanly, so as to avoid unsightly roughness. If the slope ratio on the right-hand side (Fig. 55) is s , the area of the triangle is $\frac{1}{2}sm^2$. The area of the section is $\frac{1}{2}[ug + (g+h)v + (h+j)x + (j+k)y + (k+m)z - sm^2]$. If all the horizontal measurements were referred to one side as an origin, a formula similar to Eq. 61 could readily be developed, but little or no advantage would be gained on account of any simplicity of computation. Since the *exact* volume of the earth borrowed is frequently necessary, the prismoidal correction should be computed; and since such a section as Fig. 55 does not even approximate to a three-level section, the method suggested in § 83 cannot be employed. It will then be necessary to employ the more exact method of dividing the volume into triangular prismoids and taking the summation of their corrections, found according to the general method of § 72.

90. Correction for curvature. The volume of a solid, generated by revolving a plane area about an axis lying in the plane but outside of the area, equals the product of the given area times the length of the path of the center of gravity of the area. If the centers of gravity of all cross-sections lie in the center of the road, where the length of the road is measured, there is absolutely no necessary correction for curvature. If all the cross-sections in any given length were exactly the same and therefore had the same eccentricity, the correction for curvature would be very readily computed according to the above principle. But when both the areas and the eccentricities vary from point to point, as is generally the case, a theoretically exact

solution is quite complex, both in its derivation and application. Suppose, for simplicity, a curved section of the road, of uniform cross-sections and with the center of gravity of every cross-section at the same distance e from the center line of the road. The length of the path of the center of gravity will be to the length of the center line as $R \pm e : R$. Therefore we have

True vol. : nominal vol. :: $R \pm e : R$. $\therefore$ True vol. = $lA \frac{R \pm e}{R}$ for

a volume of uniform area and eccentricity. For any other area

and eccentricity we have, similarly, *True vol. = $lA' \frac{R \pm e'}{R}$* . This

shows that the effect of curvature is the same as increasing (or diminishing) the area by a quantity depending on the area and eccentricity, the increased (or diminished) area being found by

multiplying the actual area by the ratio $\frac{R \pm e}{R}$. This being

independent of the value of l , it is true for infinitesimal lengths. If the eccentricity is assumed to vary uniformly between two sections, the *equivalent area* of a cross-section located midway

between the two end cross-sections would be $A_m \frac{\left(R \pm \frac{e' + e''}{2}\right)}{R}$.

Therefore the volume of a solid which, when straight, would be $\frac{l}{6}(A' + 4A_m + A'')$, would then become

$$\text{True vol.} = \frac{l}{6R} \left[A'(R \pm e') + 4A_m \left(R \pm \frac{e' + e''}{2} \right) + A''(R \pm e'') \right].$$

Subtracting the nominal volume (the true volume when the prismoid is straight), the

$$\text{Correction} = \pm \frac{l}{6R} \left[(A' + 2A_m)e' + (2A_m + A'')e'' \right]. \quad (63)$$

Another demonstration of the same result is given by Prof. C. L. Crandall in his "Tables for the Computation of Railway and other Earthwork," in which is obtained by calculus methods the summation of elementary volumes having variable areas with variable eccentricities. The exact application of Eq. 63 requires that A_m be known, which requires laborious computa-

tions, but no error worth considering is involved if the equation is written approximately

$$\text{Curv. corr.} = \frac{l}{2R}(A'e' + A''e''), \quad . \quad . \quad . \quad (64)$$

which is the equation generally used. The approximation consists in assuming that the difference between A' and A_m equals the difference between A_m and A'' but with opposite sign. The error due to the approximation is always utterly insignificant.

91. Eccentricity of the center of gravity. The determination of the true positions of the centers of gravity of a long series of irregular cross-sections would be a very laborious operation, but fortunately it is generally sufficiently accurate to consider the cross-sections as three-level ground, or, for side-hill work, to

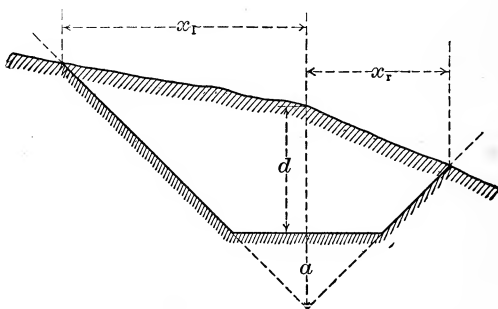


FIG. 56.

be triangular, *for the purpose of this correction*. The eccentricity of the cross-section of Fig. 56 (including the grade triangle) may be written

$$e = \frac{\frac{(a+d)x_l}{2} \frac{x_l}{3} - \frac{(a+d)x_r}{2} \frac{x_r}{3}}{\frac{(a+d)x_l}{2} + \frac{(a+d)x_r}{2}} = \frac{1}{3} \frac{x_l^2 - x_r^2}{x_l + x_r} = \frac{1}{3} (x_l - x_r). \quad . \quad (65)$$

The side toward x_l being considered positive in the above demonstration, if $x_r > x_l$, e would be negative, i.e., the center of gravity would be on the right side. Therefore, for three-level

ground, the correction for curvature (see Eq. 64) may be written

$$\text{Correction} = \frac{l}{6R} [A'(x_l' - x_r') + A''(x_l'' - x_r'')].$$

Since the approximate volume of the prismoid is

$$\frac{l}{2}(A + A') = \frac{l}{2}A' + \frac{l}{2}A'' = V' + V'',$$

in which V' and V'' represent the number of cubic yards corresponding to the area at each station, we may write

$$\text{Corr. in cub. yds.} = \frac{1}{3R} [V'(x_l' - x_r') + V''(x_l'' - x_r'')]. \quad (66)$$

It should be noted that the value of e , derived in Eq. 65, is the eccentricity of the whole area including the triangle under the roadbed. The eccentricity of the true area is greater than this and equals

$$e \times \frac{\text{true area} + \frac{1}{2}ab}{\text{true area}} = e_1.$$

The required quantity ($A'e'$ of Eq. 64) equals *true area* $\times e_1$ which equals $(\text{true area} + \frac{1}{2}ab) \times e$. Since the value of e is very simple, while the value of e_1 would, in general, be a complex quantity, it is easier to use the simple value of Eq. 65 and add $\frac{1}{2}ab$ to the area. Therefore, in the case of three-level ground the subtractive term $\frac{2}{7}ab$ (§ 79) should *not* be subtracted in computing this correction. For irregular ground, when computed by the method given in §§ 82 and 83, which does not involve the grade triangle, a term $\frac{2}{7}ab$ must be *added* at every station when computing the quantities V' and V'' for Eq. 66.

It should be noted that the factor $1 \div 3R$, which is constant for the length of the curve, may be computed with all necessary accuracy and without resorting to tables by remembering that

$$R = \frac{5730}{\text{degree of curve}}.$$

Since it is useless to attempt the computation of railroad earthwork closer than the nearest cubic yard, it will frequently

be possible to write out all curvature corrections by a simple mental process upon a mere inspection of the computation sheet. Eq. 66 shows that the correction for each station is of the form $\frac{V(x_l - x_r)}{3R}$. $3R$ is generally a large quantity—for a 6° curve it is 2865. $(x_l - x_r)$ is generally small. It may frequently be seen by inspection that the product $V(x_l - x_r)$ is roughly twice or three times $3R$, or perhaps less than half of $3R$, so that the corrective term for that station may be written 2, 3, or 0 cubic yards, the fraction being disregarded. For much larger absolute amounts the correction must be computed with a correspondingly closer percentage of accuracy.

The algebraic sign of the curvature correction is best determined by noting that the center of gravity of the cross-section is on the right or left side of the center according as x_r is greater or less than x_l , and that the correction is *positive* if the center of gravity is on the *outside* of the curve, and *negative* if on the *inside*.

It is frequently found that x_l is uniformly greater (or uniformly less) than x_r throughout the length of the curve. Then the curvature correction for each station is uniformly positive or negative. But in irregular ground the center of gravity is apt

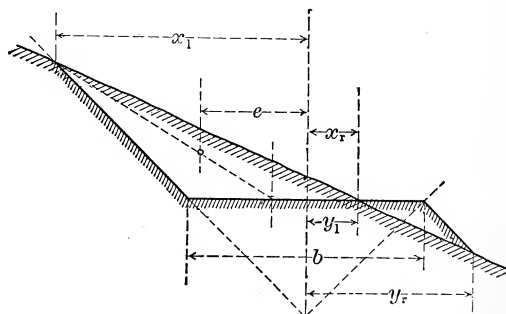


FIG. 57.

to be irregularly on the outside or on the inside of the curve, and the curvature correction will be correspondingly positive or negative. If the curve is to the *right*, the correction will be positive or negative according as $(x_l - x_r)$ is positive or negative; if the curve is to the *left*, the correction will be positive or nega-

tive according as $(x_r - x_l)$ is positive or negative. Therefore when computing curves to the *right* use the form $(x_l - x_r)$ in Eqs. 66 and 68; when computing curves to the *left* use the form $(x_r - x_l)$ in these equations; the algebraic sign of the correction will then be strictly in accordance with the results thus obtained.

92. Center of gravity of side-hill sections. In computing the correction for side-hill work the cross-section would be treated as triangular unless the error involved would evidently be too great to be disregarded. The center of gravity of the triangle lies on the line joining the vertex with the middle of the base and at $\frac{1}{3}$ of the length of this line from the base. It is therefore equal to the distance from the center to the foot of this line plus $\frac{1}{3}$ of its horizontal projection. Therefore

$$\begin{aligned} e &= \left[\frac{b}{2} - \frac{1}{2} \left(\frac{b}{2} + x_r \right) \right] + \frac{1}{3} \left[x_l - \left(\frac{b}{2} - \frac{1}{2} \left(\frac{b}{2} + x_r \right) \right) \right] \\ &= \frac{b}{4} - \frac{x_r}{2} + \frac{x_l}{3} - \frac{b}{12} + \frac{x_r}{6} \\ &= \frac{b}{6} + \frac{x_l}{3} - \frac{x_r}{3} \\ &= \frac{1}{3} \left[\frac{b}{2} + (x_l - x_r) \right] \dots \dots \dots (67) \end{aligned}$$

By the same process as that used in § 91 the correction equation may be written

$$\text{Corr. in cub. yds.} = \frac{1}{3R} \left[V' \left(\frac{b}{2} + (x_l' - x_r') \right) + V'' \left(\frac{b}{2} + (x_l'' - x_r'') \right) \right]. \quad (68)$$

It should be noted that since the grade triangle is not used in this computation the volume of the grade prism is *not* involved in computing the quantities V' and V'' .

The eccentricities of cross-sections in side-hill work are *never* zero, and are frequently quite large. The total volume is generally quite small. It follows that the correction for curvature is generally a vastly larger proportion of the total volume than in ordinary three-level or irregular sections.

If the triangle is wholly to one side of the center, Eq. 67 can still be used. For example, to compute the eccentricity of the triangle of fill, Fig. 57, denote the two distances to the slope-

stakes by y_r and $-y_l$ (note the minus sign). Applying Eq. 67 literally (noting that $\frac{b}{2}$ must here be considered as negative in order to make the notation consistent) we obtain

$$e = \frac{1}{3} \left[-\frac{b}{2} + (-y_l - y_r) \right],$$

which reduces to

$$e = -\frac{1}{3} \left[\frac{b}{2} + y_l + y_r \right]. \quad . \quad . \quad . \quad . \quad . \quad (69)$$

As the algebraic signs tend to create confusion in these formulæ, it is more simple to remember that for a triangle lying on *both* sides of the center e is always numerically equal to $\frac{1}{3} \left[\frac{b}{2} + (x_l \sim x_r) \right]$, and for a triangle entirely on one side, e is numerically equal to $\frac{1}{3} \left[\frac{b}{2} + \text{the numerical sum of the two distances out} \right]$. The algebraic sign of e is readily determinable as in § 91.

93. Example of curvature correction. Assume that the fill in § 79 occurred on a 6° curve to the *right*. $\frac{1}{3R} = \frac{1}{2865}$. The quantities 210, 507, etc., represent the quantities V' , V'' , etc., since they include in each case the 61 cubic yards due to the grade prism. Then

$$\frac{V(x_l \sim x_r)}{3R} = \frac{210(22.9 - 8.2)}{2865} = \frac{3101.7}{2865} = +1.$$

The sign is plus, since the center of gravity of the cross-section is on the left side of the center and the road curves to the right, thus making the true volume larger. For Sta. 18 the correction, computed similarly, is +3, and the correction for the whole section is $1 + 3 = 4$. For Sta. 18+40 the correction is computed as 6 yards. Therefore, for the 40 feet, the correction is $\frac{40}{100}(3 + 6) = 3.6$, which is called 4. Computing the others similarly we obtain a total correction of +16 cubic yards.

94. **Accuracy of earthwork computations.** The preceding methods give the *precise volume* (except where approximations are distinctly admitted) of the prismoids which are *supposed* to represent the volume of the earthwork. To appreciate the accuracy necessary in cross-sectioning to obtain a given accuracy in volume, consider that a fifteen-foot length of the cross-section, which is assumed to be straight, really sags 0.1 foot, so that the cross-section is in error by a triangle 15 feet wide and 0.1 foot high. This sag 0.1 foot high would hardly be detected by the eye, but in a length of 100 feet in each direction it would make an error of volume of 1.4 cubic yards in *each* of the two prismoids, assuming that the sections at the other ends were perfect. If the cross-sections at both ends of a prismoid were in error by this same amount, the volume of that prismoid would be in error by 2.8 cubic yards if the errors of area were both plus or both minus. If one were plus and one minus, the errors would neutralize each other, and it is the compensating character of these errors which permits any confidence in the results as obtained by the usual methods of cross-sectioning. It demonstrates the utter futility of attempting any closer accuracy than the nearest cubic yard. It will thus be seen that if an error really exists at *any* cross-section it involves the prismoids on *both* sides of the section, even though all the other cross-sections are perfect. As a further illustration, suppose that cross-sections were taken by the method of slope angle and center depth (§ 74), and that a cross-section, assumed as uniform, sags 0.4 foot in a width of 20 feet. Assume an equal error (of same sign) at the other end of a 100-foot section. The error of volume for that one prismoid is 38 cubic yards.

The computations further assume that the warped surface, passing through the end sections, coincides with the surface of the ground. Suppose that the cross-sectioning had been done with mathematical perfection; and, to assume a simple case, suppose a sag of 0.5 foot between the sections, which causes an error equal to the volume of a pyramid having a base of 20 feet (in each cross-section) times 100 feet (between the cross-sections) and a height of 0.5 foot. The volume of this pyramid is $\frac{1}{3}(20 \times 100) \times 0.5 = 333$ cub. ft. = 12 cub. yds. And yet this sag or hump of 6 inches would generally be utterly unnoticed, or at least disregarded.

When the ground is very rough and broken it is sometimes

practically impossible, even with frequent cross-sections, to locate warped surfaces which will closely coincide with all the sudden irregularities of the ground. In such cases the computations are necessarily more or less approximate and dependence must be placed on the compensating character of the errors.

95. Approximate computations from profiles. When a "paper location" has been laid out on a topographical map having contours, it is possible to compute approximately the amount of earthwork required by some very simple and rapid calculations. A profile may be readily drawn by noting the intersections of the proposed center line with the various contours and plotting the surface line on profile paper. Drawing the grade-line on the profile, the depth of cut or fill may be scaled off at any point. When it is only desired to obtain

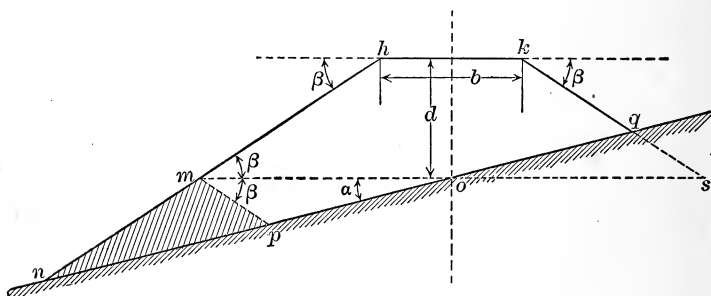


FIG. 58.

very quickly an approximate estimate of the amount of earthwork required on a suggested line, it may be done by the method described in § 75, or by the use of Table XXXIII. But the assumption that the surface of the ground at each cross-section is level invariably has the effect that the estimated volumes are not as large as those actually required. The difference between the "level section" $hkms$ and the actual slope section $hknq$ equals the difference between the triangles mon and ogs , and this difference equals the shaded area mpn . The excess volume is proportional to the area of the triangle mpn . This area may be expressed by the formula,

$$\text{Area } mnp = 2\left(\frac{1}{2}b + d \cot \beta\right)^2 \frac{\sin^2 \alpha \sin \beta \cos \beta}{\cos 2\alpha - \cos 2\beta}.$$

The percentage of this excess area to the nominal area *hkms* therefore depends on the dimensions *b* and *d* and the angles α and β . A solution of this equation for ninety different combinations of various numerical values for these four variables is included in Table XXXIII for the purpose of making corrections. A study of this correction table points conclusively to the following laws, a thorough understanding of which will enable an engineer to appreciate the degree of accuracy which is attainable by this approximate method:

(a) Increasing the *width* of the roadbed (*b*), the other three factors remaining constant, *increases* the percentage of error, but the increase is comparatively small.

(b) Increasing the *depth* of cut or fill (*d*), *decreases* the percentage of error, but the decrease is almost insignificant.

(c) Increasing the angle of the side slopes (β) *decreases* the percentage of error, the decrease being very considerable.

(d) Increasing the angle of the slope of the ground (α), *increases* the percentage of error, the percentage rapidly increasing to infinity as the value of α approaches that of β . This is another method of stating the fact that α must always be less than β and, practically, must be considerably less, so that the slope stake shall be within a reasonable distance from the center.

Since the above value for the corrective area is a function of the angle α , which is usually variable and whose value is frequently known only approximately, it is useless to attempt to apply the correction with great precision, and the following rules will usually be found amply accurate, considering the probable lack of precision in the data used.

1. For embankments or cuts, having a slope of 1.5:1, and with a surface slope of 5° (nearly 9%) the excess of true area over nominal area is *about* 2%. There is only a slight variation from this value for all ordinary depths (*d*) and widths (*b*) of roadbed. Therefore the nominal volume would be about 2% too *small*. On the other hand, the effect of the prismoidal correction is such that, even with truly level sections, the nominal volume is too *large*. See §§ 75 and 76. The amount of the prismoidal correction depends on the differences between successive center depths. In the very ordinary numerical case given in § 76, the correction was nearly 3%, which more than neutralizes the error due to surface slope. Therefore in

many cases on slightly sloping ground the error due to the surface slope will so nearly neutralize the prismoidal correction that the quantities taken directly from the tables (without correction for either cause) will equal the true volume with as close an approach to accuracy as the precision of the surveying will permit.

2. For a cut with a slope of 1:1, and with a surface slope of 5° the error is about 1%. This will be neutralized by still smaller prismoidal corrections. Therefore, for surface slopes of 5° or less, no allowance should be made for this error unless the prismoidal correction is also considered.

3. When the surface slope is 10° (nearly 18%) the error for a 1.5:1 slope is from 7% to 10% and for a 1:1 slope from 3% to 5%.

4. For a 30° surface slope and 1.5:1 side slopes the excess volume is three or four times the nominal volume. Such a steep surface slope implies the probability of "side-hill work" to which the above corrective rules are not applicable. When the surface slopes are very steep careful work must be done to avoid excessive error. For a 1:1 side slope, the errors are from 50% to 80%.

A still closer approximation, especially for the steeper surface slopes, may be obtained by using, directly or by interpolation, figures from the corrective tabular form which forms part of Table XXXIII. Unless the surface slope angle is known accurately (especially when large) no great accuracy in the final result is possible. Close accuracy would also require the determination of the prismoidal correction. But if such close accuracy is deemed essential, it can be most easily obtained by accurate cross-sectioning at each station and the adoption of other methods of computation—such as are given in §§ 83 and 84.

When the contours have been drawn in for a sufficient distance on either side to include the position of both slope stakes at every station, as will usually be the case, cross-sections may be obtained by drawing lines on the map at each station perpendicular to the center line—see Fig. 4. The intersection of these lines with the contours will furnish the distances for drawing on cross-section paper the transverse profile at each station. Drawing on the same cross-section the lines representing the roadbed and the side slopes, the cross-section of

cut (or fill) is complete and its area may be obtained by scaling from the cross-section paper. If the contours have been located on the map with sufficient accuracy, such a method will determine the cross-sectional area very closely. When cross-sections have been taken with a wye- or hand-level, as described in § 12, the cross-sections as plotted will probably be more accurate than when the contours are run in from points determined by the stadia method. In fact this semi-graphical method is frequently used, in place of the purely numerical methods described in previous sections, to make final estimates of the volume of earthwork.

As a numerical example, an assumed location line was laid out on the contours given in Fig. 4. The volume of cut, as determined by Table XXXIII for a roadbed 20 feet wide, with side slopes of 1:1, was 5746 cubic yards. The surface slope varied from 3° to 11° . Computing the corrections by a careful interpolation from the corrective table, the total correction was found to be 128 cubic yards, or an average of a little over 2%. On the other hand the negative prismoidal correction amounts to 72 cubic yards, which leaves a net correction of 56 cubic yards—about 1%. It so happens that in this case a correction for curvature would tend further to wipe out this correction. These figures merely verify numerically the general conclusions stated above, although it should not be forgotten that in individual cases the figures taken from Table XXXIII require ample correction.

The following approximate rule, for which the author is indebted to Mr. W. H. Edinger, is exceedingly useful when it is desired to rapidly determine the approximate volume of earthwork between two points along the road. Its great merit lies in the fact that it only means the memorizing of a comparatively simple rule which will make it possible to make such computations in the field, without the use of tables. The rule is based on the fact that the area of any level section equals $bd + sd^2$; and therefore,

$$\Sigma(\text{vol.}) = (b \Sigma d + s \Sigma d^2) \frac{L}{27},$$

in which L is usually 100 feet. For strict accuracy this would only be the volume provided the total length was an even number of hundred feet, and the various values of d represented

the depths which were uniform for hundred foot sections. It makes no allowance for the comparatively large prismoidal error of the pyramidal and wedge-shaped sections usually found at each end of a cut or fill, but where an approximate estimate is desired, in which this inaccuracy may be neglected, the method is very useful. The method of applying this rule without tables may best be illustrated by a simple numerical example. Assume that the levels on a stretch of fairly level ground, which is about 500 feet long, have been taken, the depths being taken at points 100 feet apart, the first and last points being about 40 or 50 feet from the ends of the cut, or fill. The depths are as given in the first column in the tabular form below; the slope is 1.5:1, and the breadth (b) is 14 feet.

d	d^2
1.6	2.56
2.8	7.84
4.5	20.25
3.1	9.61
0.9	.81
$\Sigma d = 12.9$	$\Sigma d^2 = 41.07$
14	20.53
$b \Sigma d = 180.6$	$s \Sigma d^2 = 61.60$
61.60	
242.2	
$24220 \div 27 = 897$ cubic yards.	

The 180.6 is the $b \Sigma d$ and the 61.6 is $s \Sigma d^2$; adding these and moving the decimal point two places to multiply by 100, we only have to divide by 27 to obtain the value in cubic yards. Although the above rule requires more work than the employment of earthwork tables, yet it is a very convenient method of estimating the approximate volume of a short section of earthwork when no tables are at hand.

96. Shrinkage of earthwork. The statistical data indicating the amount of shrinkage is very conflicting, a fact which is probably due to the following causes:

1. The various kinds of earthy material act very differently as respects shrinkage. There is a great lack of uniformity in

the *classification of earths* in the tests and experiments which have been made.

2. Very much depends on the *method* of forming an embankment (as will be shown later). Different reports have been based on different methods—often without mention of the method.

3. An embankment requires considerable *time* to shrink to its final volume, and therefore much depends on the time elapsed between construction and the measurement of what is supposed to be the settled volume.

4. A soft subsoil will frequently settle under the weight of a high embankment and apparently indicate a far greater shrinkage than the actual reduction in volume.

5. An embankment of very soft material will sometimes “mush” or widen at the sides, with a consequent settling of the top, due to this cause alone.

This subject has called forth much discussion in the technical press and literature. Quotations can be made of figures covering a large range of values, but space will only permit the statement of the conclusions which may be drawn from the large mass of testimony which has been presented.

1. *Volume of loose material.* When material of any character is excavated and deposited loosely in a pile, its volume is always largely in excess of the volume of the excavation. Solid rock will occupy from 60% to 80% *more* space when broken up than when solid. A soft earth will have an excess volume of about 20% to 25%.

2. *Effect of method of depositing.* When material is deposited loosely, as from a trestle, the excess of volume when the embankment is just completed is very large. The time required for final settlement is also very great. When an embankment is formed by the wheelbarrow method, the initial expansion is about as great as when the material is merely dumped from cars. When the material is deposited in small increments from wagons and each layer is subjected to compression from horses' hoofs and from wheels, the contraction during construction is far greater and the additional shrinkage is comparatively small. Wheeled scrapers and drag scrapers will produce even more initial compression.

3. *Time required for final settlement.* This depends partly on the method of formation and also on the character of the

material. When a soft loamy soil is deposited loosely, the drying out of the soil during the first long dry season will develop large cracks. Subsequent rains will close these cracks by a general contraction of the whole mass. When the embankment is loosely formed it may take two years before additional settlement becomes inappreciable, but when the method of deposition ensures compression during construction the *subsequent* shrinkage is less in time as well as amount.

4. *Classification of soils with respect to shrinkage.* Loose vegetable surface soil will expand very greatly when excavated and first deposited, but will subsequently shrink to considerably less than its original volume. Clay soils are next in order and the sandy and gravelly soils come at the other end of the list of earthy materials. Rock expands very greatly when first broken up and deposited and there is no appreciable subsequent shrinkage.

97. *Proper allowance for shrinkage.* Specifications for the Mississippi River levees require that there shall be a 10% shrinkage allowance for embankments formed by team work and 25% allowance for wheelbarrow work. It is contended

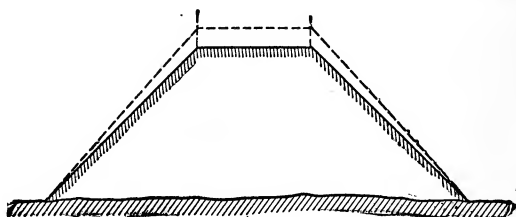


FIG. 59.

that such figures are only justified because the subsoil settles or because the embankments mush out at the sides, and that if these effects do not occur the levees are permanently higher than designed.

It is usual to require that embankments shall be constructed higher than their desired ultimate, as shown in Fig. 59. Since the base does not contract, the contraction may be said to be all vertical. Since a high embankment will unquestionably shrink a greater total amount than a low embankment (whatever the percentage), it follows that an embankment having

variable heights (as usual) should have an initial grade-line somewhat like the dotted line *adc* in Fig. 60. Although some such method is essential if there is to be no ultimate sag below the desired grade-line, the policy is sharply criticized. The grade *ad*, even though temporary, may prove objectionable from an operating standpoint. Frequently the allowance is made too great or the shrinkage is not as much as anticipated, and it becomes necessary to cut off the top of the bank. On the other hand, the expense of raising the track after the road is in operation and the inevitable loss of ballast is so great that the danger of being required to fill up a sag should be avoided if possible.

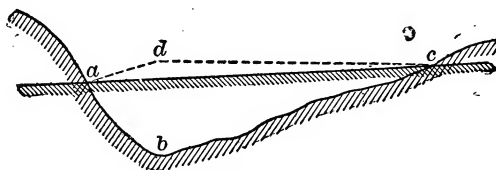


FIG. 60.

A sharp and clear distinction should be made between the *coefficient of extra height* of an embankment and the *coefficient of shrinkage* which determines how many cubic yards of settled embankment may be made from a definite volume of earth or rock measured in the excavation. The values quoted above for the Mississippi levees (from 10% to 25%) refers usually to a very soft soil and includes the effects other than actual contraction of volume. From 8% to 15% is usually quoted as the required extra height of embankments, although it is strenuously claimed by many that 3% or 2% is sufficient, or even that no allowance should be made.

The coefficients to determine the amount of settled embankment which may be made from a given volume of earth or rock measured in the excavation, are necessarily subject to variation on account of the method employed and the amount of compression and settlement which will take place during the progress of the work. The following figures have the weight of considerable authority but, if in error, the coefficients are probably high rather than low:

Gravel or sand	about	8%
Clay	"	10%
Loam	"	12%
Loose vegetable surface soil	"	15%

It may be noticed from the above table that the harder and cleaner the material the less is the contraction. Perfectly clean gravel or sand would not probably change volume appreciably. The above coefficients of shrinkage and expansion may be used to form the following convenient table:

Material.	To make 1000 cubic yards of embankment will require	1000 cubic yards measured in excavation will make
Gravel or sand	1087 cubic yards	920 cubic yards
Clay	1111 " "	900 " "
Loam	1136 " "	880 " "
Loose vegetable soil	1176 " "	850 " "
Rock, large pieces	714 " "	1400 " "
" small "	625 " "	1600 " "
	measured in excavation	of embankment.

Since writing the above the following values have been adopted by the American Railway Engineering and Maintenance of Way Association as representing standard practice:

COEFFICIENTS OF SHRINKAGE ALLOWANCE FOR DEPOSITING EARTHWORK.

	Trestle filling.	Raising under traffic.
Black dirt	15%	5%
Clay	10%	5%
Sand	6%	5%

98. Methods of forming embankments. Embankments of moderate height are sometimes formed by scraping material with drag scrapers from ditches at the sides, especially if there is little or no cutting to be done in the immediate vicinity. Over a low level swampy stretch this method has the double advantage of building an embankment which is well above the general level and also provides generous drainage ditches which keep the embankment dry. Wheeled scrapers may be used economically up to a distance of 400 feet to excavate

cuts and deposit the material on low embankments. Such methods have the advantage of compacting the embankments during construction and reducing future shrinkage.

When carts are used, an embankment of any height may be formed by "dumping over the end" and building to the full height (or even higher to allow for shrinkage) as the embankment proceeds. The method is especially applicable when the material comes from a place as high as or higher than the grade-line, so that no up-hill hauling is necessary. Only a small contractor's plant is required for all of these methods.

Trestles capable of carrying carts, or even cars and locomotives, from which excavated material may be dropped, are found to be economical in spite of the fact that their cost is a construction expense. There is the disadvantage that such embankments require a long time to settle, but there are the advantages that the earth may be hauled by the train load from a distance of perhaps several miles, dumped from the

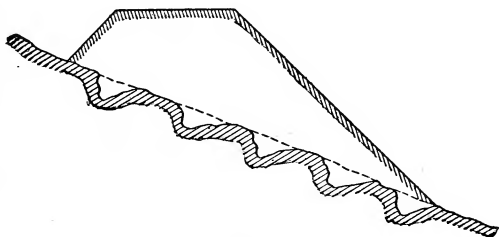


FIG. 61.

cars by train ploughs, or automatically dumped when the material is carried in patent dumping-cars, and all at a comparatively small cost per cubic yard. The disadvantages of slow settlement may be obviated, although at some additional cost, by making the trestle sufficiently strong to support regular traffic until the settlement is complete.

During recent years cableways have been utilized to fill comparatively narrow but deep ravines from material obtainable on either side of the ravine. This method obviates the construction of an excessively high trestle which might otherwise be considered necessary.

When an embankment is to be placed on a steep side hill which has a slippery clay surface, the embankment will some-

times slide down the hill; unless means are taken to prevent it. Some sort of bond between the old surface and the new material becomes necessary. This has sometimes been provided by cutting out steps somewhat as is illustrated in Fig. 61. It is possible that a deep ploughing of the surface would accomplish the result just as effectively and much cheaper. The tendency to slip is generally due not only to the nature of the soil but also to the usual accompanying characteristic that the soil is wet and springy. The sub-surface drainage of such a place with tile drains will still further prevent such slipping, which often proves very troublesome and costly.

COMPUTATION OF HAUL.

99. Nature of subject. As will be shown later when analyzing the cost of earthwork, the most variable item of cost is that depending on the distance hauled. As it is manifestly impracticable to calculate the exact distance to which every individual cartload of earth has been moved, it becomes necessary to devise a means which will give at least an equivalent of the haulage of all the earth moved. Evidently the *average* haul for any mass of earth moved is equal to the distance from the center of gravity of the excavation to the center of gravity of the embankment formed by the excavated material. As a rough approximation the center of gravity of a cut (or fill) may sometimes be considered to coincide with the center of gravity of that part of the profile representing it, but the error is frequently very large. The center of gravity may be determined by various methods, but the method of the "mass diagram" accomplishes the same ultimate purpose (the determination of the haul) with all-sufficient accuracy and also furnishes other valuable information.

100. Mass diagram. In Fig. 62 let $A'B' \dots G'$ represent a profile and grade line drawn to the usual scales. Assume A' to be a point past which no earthwork will be hauled. Such a point is determined by natural conditions, as, for example, a river crossing, or one end of a long level stretch along which no grading is to be done except the formation of a low embankment from the material excavated from ample drainage ditches on each side. Above the profile draw an indefinite horizontal line (ACn in Fig. 62), which may be called the "zero line." Above every station point in the profile draw an ordinate (above or be-

low the zero line) which will represent the algebraic sum of the cubic yards of cut and fill (calling cut + and fill -) from the point A' to the point considered. The computations of these ordinates should first be made in tabular form as shown below. In doing this shrinkage must be allowed for by considering how much embankment would actually be made by so many cubic yards of excavation of such material. For example, it will be found that 1000 cubic yards of sand or gravel, measured in place (see § 97), will make about 920 cubic yards of embankment; therefore all cuttings in sand or gravel should be discounted in about this proportion. Excavations in rock should be increased in the proper ratio. In short, all excavations should be valued according to the amount of *settled* embankment that could be made from them. Place in the first column a list of the stations; in the second column, the number of cubic yards of cut or fill between each station and the preceding station; in the third and fourth columns, the kind of material and the proper shrinkage factor; in the fifth column, a repetition of the quantities in cubic yards, except that the excavations are diminished (or increased, in the case of rock) to the number of cubic yards of settled embankment which may be made from them. In the sixth column place the *algebraic sum* of the quantities in the fifth column (calling cuts + and fills -) from the starting-point to the station considered. These algebraic sums at each station will be the ordinates, drawn to some scale, of the mass curve. The scale to be used will depend somewhat on whether

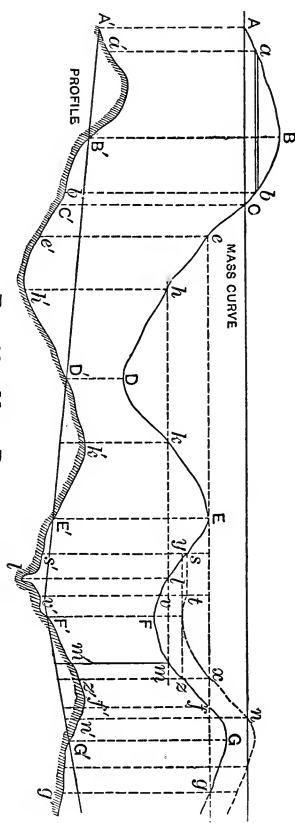


FIG. 62.—MASS DIAGRAM.

the work is heavy or light, but for ordinary cases a scale of 5000 cubic yards per inch may be used. Drawing these ordinates to scale, a curve $A, B, \dots G$ may be obtained by joining the extremities of the ordinates.

Sta.	Yards { cut + fill -	Material.	Shrinkage factor.	Yards, reduced for shrinkage.	Ordinate in mass curve.
46 + 70					0
47	+ 195	Clayey soil	- 10 per cent	+ 175	+ 175
48	+ 1792	" "	- 10 "	+ 1613	+ 1788
48 + 60	+ 614	" "	- 10 "	+ 553	+ 2341
49	- 143			- 143	+ 2198
50	- 906			- 906	+ 1292
51	- 1985			- 1985	- 693
52	- 1721			- 1721	- 2414
52 + 30	- 112			- 112	- 2526
53	+ 177	Hard rock	+ 60 per cent	+ 283	- 2243
53 + 70	+ 180	" "	+ 60 "	+ 289	- 1954
54	- 52			- 52	- 2006
54 + 42	- 71			- 71	- 2077
55	+ 276	Clayey soil	- 10 per cent	+ 249	- 1828
56	+ 1242	" "	- 10 "	+ 1118	- 710
57	+ 1302	" "	- 10 "	+ 1172	+ 462

101. Properties of the mass curve.

1. The curve will be rising while over cuts and falling while over fills.

2. A tangent to the curve will be horizontal (as at B, D, E, F , and G) when passing from cut to fill or from fill to cut.

3. When the curve is *below* the "zero line" it shows that material must be drawn *backward* (to the left); and *vice versa*, when the curve is *above* the zero line it shows that material must be drawn *forward* (to the right).

4. When the curve crosses the zero line (as at A and C) it shows (in this instance) that the cut between A' and B' will just provide the material required for the fill between B' and C' , and that no material should be hauled past C' , or, in general, past any intersection of the mass curve and the zero line.

5. If any horizontal line be drawn (as ab), it indicates that the cut and fill between a' and b' will just balance.

6. When the center of gravity of a given volume of material is to be moved a given distance, it makes no difference (at least theoretically) how far each individual load may be hauled or how any individual load may be disposed of. The summation

of the products of each load times the distance hauled will be a constant, whatever the method, and will equal the total volume times the movement of the center of gravity. The *average haul*, which is the movement of the center of gravity, will therefore equal the summation of these products divided by the total volume. If we draw two horizontal parallel lines at an infinitesimal distance dx apart, as at ab , the small increment of cut dx at a' will fill the corresponding increment of fill at b' , and this material must be hauled the distance ab . Therefore the product of ab and dx , which is the product of distance times volume, is represented by the area of the infinitesimal rectangle at ab , and the total area ABC represents the summation of volume times distance for all the earth movement between A' and C' . This summation of products divided by the total volume gives the average haul.

7. The horizontal line, tangent at E and cutting the curve at e , f , and g , shows that the cut and fill between e' and E' will just balance, and that a *possible* method of hauling (whether desirable or not) would be to "borrow" earth for the fill between C' and e' , use the material between D' and E' for the fill between e' and D' , and similarly balance cut and fill between E' and f' and also between f' and g' .

8. Similarly the horizontal line $hkml$ may be drawn cutting the curve, which will show another *possible* method of hauling. According to this plan, the fill between C' and h' would be made by borrowing; the cut and fill between h' and k' would balance; also that between k' and l' and between l' and m' . Since the area $ehDkE$ represents the measure of haul for the earth between e' and E' , and the other areas measure the corresponding hauls similarly, it is evident that the sum of the areas $ehDkE$ and $ElFmf$, which is the measure of haul of all the material between e' and f' , is largely in excess of the sum of the areas hDk , kEl , and lFm , plus the somewhat uncertain measures of haul due to borrowing material for $e'h'$ and wasting the material between m' and f' . Therefore to make the measure of haul a minimum a line should be drawn which will make the sum of the areas between it and the mass curve a minimum. Of course this is not necessarily the cheapest plan, as it implies more or less borrowing and wasting of material, which *may* cost more than the amount saved in haul. The comparison of the two methods is quite simple, however. Since the amount

of fill between e' and h' is represented by the *difference* of the ordinates at e and h , and similarly for m' and f' , it follows that the amount to be borrowed between e' and h' will exactly equal the amount wasted between m' and f' . By the first of the above methods the haul is excessive, but is definitely known from the mass diagram, and all of the material is utilized; by the second method the haul is reduced to about one-half, but there is a known quantity in cubic yards wasted at one place and the same quantity borrowed at another. The length of haul necessary for the borrowed material would need to be ascertained; also the haul necessary to waste the other material at a place where it would be unobjectionable. Frequently this is best done by widening an embankment beyond its necessary width. The computation of the relative cost of the above methods will be discussed later (§ 116).

9. Suppose that it were deemed best, after drawing the mass curve, to introduce a trestle between s' and v' , thus saving an amount in fill equal to tv . If such had been the original design, the mass curve would have been a straight horizontal line between s and t and would continue as a curve which would be at all points a distance tv above the curve $vFmzfGg$. If the line Ef is to be used as a zero line, its intersection with the new curve at x will show that the material between E' and z' will just balance if the trestle is used, and that the amount of haul will be measured by the area between the line Ex and the broken line $Estx$. The same computed result may be obtained without drawing the auxiliary curve $txn \dots$ by drawing the horizontal line zy at a distance $xz (=tv)$ below Ex . The amount of the haul can then be obtained by adding the triangular area between Es and the horizontal line Ex , the rectangle between st and Ex , and the irregular area between vFz and $y \dots z$ (which last is evidently equal to the area between tx and $E \dots x$). The disposal of the material at the right of z' would then be governed by the indications of the profile and mass diagram which would be found at the right of g' . In fact it is difficult to decide with the best of judgment as to the proper disposal of material without having a mass diagram extending to a considerable distance each side of that part of the road under immediate consideration.

102. **Area of the mass curve.** The area may be computed most readily by means of a planimeter, which is capable with reasonable care of measuring such areas with as great accuracy

as is necessary for this work. If no such instrument is obtainable, the area may be obtained by an application of "Simpson's rule." The ordinates will usually be spaced 100 feet apart. Select an *even* number of such spaces, leaving, if necessary, one or more triangles or trapezoids at the ends for separate and independent computation. Let $y_0 \dots y_n$ be the ordinates, i.e., the number of cubic yards at each station of the mass curve, or the figures of "column six" referred to in § 100. Let the uniform distance between ordinates ($=100$ feet) be called 1, i.e., one *station*. Then the units of the resulting area will be cubic yards hauled one station. Then the

$$\text{Area} = \frac{1}{3}[y_0 + 4(y_1 + y_3 + \dots y_{n-1}) + 2(y_2 + y_4 + \dots y_{n-2}) + y_n]. \quad (70)$$

When an ordinate occurs at a substation, the best plan is to ignore it at first and calculate the area as above. Then, if the difference involved is too great to be neglected, calculate the area of the triangle having the extremity of the ordinate at the substation as an apex, and the extremities of the ordinates at the adjacent stations as the ends of the base. This may be done by finding the ordinate at the substation that would be a proportional between the ordinates at the adjacent full stations. Subtract this from the real ordinate (or *vice versa*) and multiply the difference by $\frac{1}{2} \times 1$. An inspection will often show that the correction thus obtained would be too small to be worthy of consideration. If there is more than one substation between two full stations, the corrective area will consist of two triangles and one or more trapezoids which may be similarly computed, if necessary.

When the zero line (Fig. 62) is shifted to eE , the drop from AC (produced) to E is known in the same units, cubic yards. This constant may be subtracted from the numbers ("column 6," § 100) representing the ordinates, and will thus give, without any scaling from the diagram, the exact value of the modified ordinates.

103. Value of the mass diagram. The great value of the mass diagram lies in the readiness with which different plans for the disposal of material may be examined and compared. When the mass curve is once drawn, it will generally require only a shifting of the horizontal line to show the disposal of the material by any proposed method. The mass diagram also shows the

extreme length of haul that will be required by any proposed method of disposal of material. This brings into consideration the "limit of profitable haul," which will be fully discussed in § 116. For the present it may be said that with each method of carrying material there is some limit beyond which the expense of hauling will exceed the loss resulting from borrowing and wasting. With wheelbarrows and scrapers the limit of profitable haul is comparatively short, with carts and tram-cars it is much longer, while with locomotives and cars it may be several miles. If, in Fig. 62, eE or Ef exceeds the limit of profitable haul, it shows at once that some such line as $hklm$ should be drawn and the material disposed of accordingly.

104. Changing the grade line. The formation of the mass curve and the resulting plans as to the disposal of material are based on the mutual relations of the grade line and the surface profile and the amounts of cut and fill which are thereby implied. If the grade line is altered, every cross-section is altered, the amount of cut and fill is altered, and the mass curve is also changed. At the farther limit of the actual change of the grade line the revised mass curve will have (in general) a different ordinate from the previous ordinate at that point. From that point on, the revised mass curve will be parallel to its former position, and the revised curve may be treated similarly to the case previously mentioned in which a trestle was introduced. Since it involves tedious calculations to determine accurately how much the volume of earthwork is altered by a change in grade line, especially through irregular country, the effect on the mass curve of a change in the grade line cannot therefore be readily determined except in an approximate way. Raising the grade line will evidently increase the fills and diminish the cuts, and *vice versa*. Therefore if the mass curve indicated, for example, either an excessively long haul or the necessity for borrowing material (implying a fill) and wasting material farther on (implying a cut), it would be possible to diminish the fill (and hence the amount of material to be borrowed) by lowering the grade line near that place, and diminish the cut (and hence the amount of material to be wasted) by raising the grade line at or near the place farther on. Whether the advantage thus gained would compensate for the possibly injurious effect of these changes on the grade line would require patient investigation. But the method outlined shows how the mass

curve might be used to indicate a possible change in grade line which might be demonstrated to be profitable.

105. Limit of free haul. It is sometimes specified in contracts for earthwork that *all* material shall be entitled to free haul up to some specified limit, say 500 or 1000 feet, and that all material drawn farther than that shall be entitled to an allowance on the *excess* of distance. It is manifestly impracticable to measure the excess for each load, as much so as to measure the actual haul of each load. The mass diagram also solves this problem very readily. Let Fig. 63 represent a pro-

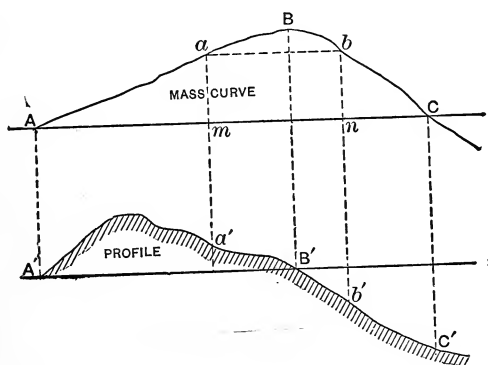


FIG. 63.

file and mass diagram of about 2000 feet of road, and suppose that 800 feet is taken as the limit of free haul. Find two points, *a* and *b*, in the mass curve which are on the same horizontal line and which are 800 feet apart. Project these points down to *a'* and *b'*. Then the cut and fill between *a'* and *b'* will just balance, and the cut between *A'* and *a'* will be needed for the fill between *b'* and *C'*. In the mass curve, the area between the horizontal line *ab* and the curve *aBb* represents the haulage of the material between *a'* and *b'*, which is all free. The rectangle *abmn* represents the haulage of the material in the cut *A'a'* across the 800 feet from *a'* to *b'*. This is also free. The sum of the two areas *Aam* and *bnC* represents the haulage entitled to an allowance, since it is the summation of the products of cubic yards times the *excess* of distance hauled.

If the amount of cut and fill was symmetrical about the point

B' , the mass curve would be a symmetrical curve about the vertical line through B , and the two limiting lines of free haul would be placed symmetrically about B and B' . In general there is no such symmetry, and frequently the difference is considerable. The area $aBbnm$ will be materially changed according as the two vertical lines am and bn , always 800 feet apart, are shifted to the right or left. It is easy to show that the area $aBbnm$ is a *maximum* when ab is horizontal. The minimum value would be obtained either when m reached A or n reached C , depending on the exact form of the curve. Since the position for the minimum value is manifestly unfair, the best definite value obtainable is the maximum, which must be obtained as above described. Since $aBbnm$ is made maximum, the remainder of the area, which is the allowance for overhaul, becomes a minimum. The areas Aam and bCn may be obtained as in § 102. If the whole area $AaBbCA$ has been previously computed, it may be more convenient to compute the area $aBbnm$ and subtract it from the total area.

Since the intersections of the mass curve and the "zero line" mark limits past which no material is drawn, it follows that there will be no allowance for overhaul except where the distance between consecutive intersections of the zero line and mass curve exceeds the limit of free haul.

Frequently all allowances for overhaul are disregarded; the profiles, estimates of quantities, and the required disposal of material are shown to bidding contractors, and they must then make their own allowances and bid accordingly. This method has the advantage of avoiding possible disputes as to the amount of the overhaul allowance, and is popular with railroad companies on this account. On the other hand the facility with which different plans for the disposal of material may be studied and compared by the mass-curve method facilitates the adoption of the most economical plan, and the elimination of uncertainty will frequently lead to a safe reduction of the bid, and so the method is valuable to both the railroad company and the contractor.

ELEMENTS OF THE COST OF EARTHWORK.

106. **Analysis of the total cost into items.** The variation in the total cost of excavating earthwork, hauling it a greater or less distance, and forming with it an embankment of definite

form or wasting it on a spoil bank, is so great that the only possible method of estimating the cost under certain assumed conditions is to separate the total cost into elementary items. Ellwood Morris was perhaps the first to develop such a method—see *Journal of the Franklin Institute*, September and October, 1841. Trautwine used the same general method with some modifications. The following analysis will follow the same general plan, will quote some of the figures given by Morris and by Trautwine, but will also include facts and figures better adapted to modern conditions. Since every item of cost (except interest on cost of plant and its depreciation) is a direct function of the current price of common labor, all calculations will be based on the simple unit of \$1 per day. Then the actual cost may be obtained by multiplying the calculated cost under the given conditions by the current price of day labor. When possible, figures will be quoted giving the cost of all items of work on a loose sandy soil which is the easiest to work and also for the cost of the heaviest soils, such as stiff clay and hard pan. These represent the extremes, excluding rock, which will be treated separately. The cost of intermediate grades may be interpolated between the extreme values according to the judgment of the engineer as to the character of the soil.

The possible division into items varies greatly according to the method adopted, but the differentiation into items given below (which is strictly applicable to the old fashioned simpler methods of work) can usually be applied to any other method by merely combining or eliminating some of the items. The items are

1. Loosening the natural soil.
2. Loading the soil into whatever carrier may be used.
3. Hauling excavated material from excavation to embankment or spoil bank.
4. Spreading or distributing the soil on the embankment.
5. Keeping roadways or tracks in good running order.
6. Trimming cuts to their proper cross-section (sometimes called "sandpapering").
7. Repairs, wear, depreciation, and interest on cost of plant.
8. Superintendence and incidentals.

107. Item 1. Loosening. (a) Ploughs. Very light sandy soils can frequently be shovelled without any previous loosening, but it is generally economical, even with very light material,

to use a plough. Morris quotes, as the results of experiments, that a three-horse plough would loosen from 250 to 800 cubic yards of earth per day, which at a valuation of \$5 per day would make the cost per yard vary from 2 cents to 0.6 cent. Trautwine estimates the cost on the basis of two men handling a two-horse plough at a total cost of \$3.87 per day, being \$1 each for the men, 75 c. for each horse, and an allowance of 37 c. for the plough, harness, etc. From 200 to 600 cubic yards is estimated as a fair day's work, which makes a cost of 1.9 c. to 0.65 c. per yard, which is substantially the same estimate as above. Extremely heavy soils have sometimes been loosened by means of special ploughs operated by traction-engines.

Gillette estimates that "a two-horse team with a driver and a man holding the plough will loosen 25 cubic yards of fairly tough clay, or 35 cubic yards of gravel and loam per hour." For ten hours per day this would be 250 to 350 cubic yards per day. These values are neither as high nor as low as the extremes above noted. It is probably very seldom that a soil will be so light that a two-horse (or three-horse) plough can loosen as much as 600 (or 800) cubic yards per day.

It is sometimes necessary to plough up a macadamized street. This may be done by using as a plough a pointed steel bar which is fastened to a very strong plough frame. A preliminary hole must be made which will start the bar under the macadam shell. Then, as the plough is drawn ahead, the shell is ripped up. Four or six horses, or even a traction-engine, are used for such work. Gillette quotes two such cases where the cost of such loosening was 2 c. and 6 c. per cubic yard, with common labor at 15 c. per hour. Two-thirds of such figures will reduce them to the \$1 per day basis. The cost for ploughing *on the \$1 per day basis* may therefore be summarized as follows:

For very loose sandy soils.....	0.6 c. per cubic yard
“ “ heavy clay “	2.0 c. “ “ “
“ hard pan and macadam, up to ...	4.0 c. “ “ “

(b) Picks. When picks are used for loosening the earth, as is frequently necessary and as is often done when ploughing would perhaps be really cheaper, an estimate * for a fair day's

* Trautwine.

work is from 14 to 60 cubic yards, the 14 yards being the estimate for stiff clay or cemented gravel, and the 60 yards the estimate for the lightest soil that would require loosening. At \$1 per day this means about 7 c. to 1.7 c. per cubic yard, which is about three times the cost of ploughing. Five feet of the face is estimated * as the least width along the face of a bank that should be allowed to enable each laborer to work with freedom and hence economically.

(c) **Blasting.** Although some of the softer shaly rocks may be loosened with a pick for about 15 to 20 c. per yard, yet rock in general, frozen earth, and sometimes even compact clay are most economically loosened by blasting. The subject of blasting will be taken up later, §§ 117-123.

(d) **Steam-shovels.** The items of loosening and loading merge together with this method, which will therefore be treated in the next section.

108. Item 2. LOADING. (a) Hand-shovelling. Much depends on proper management, so that the shovellers need not wait unduly either for material or carts. With the best of management considerable time is thus lost, and yet the intervals of rest need not be considered as entirely lost, as it enables the men to work, while actually loading, at a rate which it would be physically impossible for them to maintain for ten hours. Seven shovellers are sometimes allowed for each cart; otherwise there should be five, two on each side and one in the rear. Economy requires that the number of loads per cart per day should be made as large as possible, and it is therefore wise to employ as many shovellers as can work without mutual interference and without wasting time in waiting for material or carts. The figures obtainable for the cost of this item are unsatisfactory on account of their large disagreements. The following are quoted as the number of cubic yards that can be loaded into a cart by an average laborer in a working day of ten hours, the lower estimate referring to heavy soils, and the higher to light sandy soils: 10 to 14 cubic yards (Morris), 12 to 17 cubic yards (Haskoll), 18 to 22 cubic yards (Hurst), 17 to 24 cubic yards (Trautwine), 16 to 48 cubic yards (Ancelin). As these estimates are generally claimed to be based on actual experience, the discrepancies are probably due to differences of management. If the

* Hurst.

average of 15 to 25 cubic yards be accepted, it means, on the basis of \$1 per day, 6.7 c. to 4 c. per cubic yard. These estimates apply only to earth. *Rockwork* costs more, not only because it is harder to handle, but because a cubic yard of solid rock, measured in place, occupies about 1.8 cubic yards when broken up, while a cubic yard of earth will occupy about 1.2 cubic yards. Rockwork will therefore require about 50% more loads to haul a given volume, *measured in place*, than will the same nominal volume of earthwork. The above authorities give estimates for loading rock varying from 6.9 c. to 10 c. per cubic yard. The above estimates apply only to the loading of carts or cars with shovels or by hand (loading masses of rock). The cost of loading wheelbarrows and the cost of scraper work will be treated under the item of hauling.

(b) **Steam-shovels.*** Whenever the magnitude of the work will warrant it there is great economy in the use of steam-shovels. These have a "bucket" or "dipper" on the end of a long beam, the bucket having a capacity varying from $\frac{1}{2}$ to $2\frac{1}{2}$ cubic yards. Steam-shovels handle all kinds of material from the softest earth to shale rock, earthy material containing large boulders, tree-stumps, etc. The record of work done varies from 200 to 1000 cubic yards in 10 hours. They perform all the work of loosening and loading. Their economical working requires that the material shall be hauled away as fast as it can be loaded, which usually means that cars on a track, hauled by horses or mules, or still better by a locomotive, shall be used. The expenses for a steam-shovel, costing about \$5000, will average about \$1000 per month. Of this the engineer may get \$100; the fireman \$50; the cranesman \$90; repairs perhaps \$250 to \$300; coal, from 15 to 25 tons, cost very variable on account of expensive hauling; water, a very uncertain amount, sometimes costing \$100 per month; about five laborers and a foreman, the laborers getting \$1.25 per day and the foreman \$2.50 per day, which will amount to \$227.50 per month. This gang of laborers is employed in shifting the shovel when necessary, taking up and relaying

* For a thorough treatment of the capabilities, cost, and management of steam-shovels the reader is referred to "Steam-shovels and Steam-shovel Work," by E. A. Hermann. D. Van Nostrand Co., New York.

This book is now out of print. "Earthwork and its Cost," by H. P. Gillette, to which the student is referred for a more elaborate exposition of the subject, has used many of Hermann's cuts.

tracks for the cars, shifting loaded and unloaded cars, etc. In shovelling through a deep cut, the shovel is operated so as to undermine the upper parts of the cut, which then fall down within reach of the shovel, thus increasing the amount of material handled for each new position of the shovel. If the material is too tough to fall down by its own weight, it is sometimes found economical to employ a gang of men to loosen it or even blast it rather than shift the shovel so frequently. Non-condensing engines of 50 horse-power use so much water that the cost of water-supply becomes a serious matter if water is not readily obtainable. The lack of water facilities will often justify the construction of a pipe line from some distant source and the installation of a steam-pump. Hence the seemingly large estimate of \$100 per month for water-supply, although under favorable circumstances the cost may almost vanish. The larger steam-shovels will consume nearly a ton of coal per day of 10 hours. The expense of hauling this coal from the nearest railroad or canal to the location of the cut is often a very serious item of expense and may easily double the cost per ton. Some steam-shovels have been constructed to be operated by electricity obtained from a plant perhaps several miles away. Such a method is especially advantageous when fuel and water are difficult to obtain.

The following general requirements and specifications were recommended in 1907 by the American Railway Engineering and Maintenance of Way Association:

Three important cardinal points should be given careful attention in the selection of a steam-shovel. These are in their order

- (1) Care in the selection, inspection and acceptance of all material that enters into every part of the machine.
- (2) Design for strength.
- (3) Design for production.

GENERAL SPECIFICATIONS.

Weight of shovel: Seventy (70) tons.

Capacity of dipper: Two and one-half ($2\frac{1}{2}$) yards.

Steam pressure: One hundred and twenty (120) pounds.

Clear height above rail of shovel track at which dipper should unload: Sixteen (16) feet.

Depth below rail of shovel track at which dipper should dig
Four (4) feet.

Number of movements of dipper per minute from time of
entering bank to entering bank: Three (3).

Character of hoist: Cable.

Character of swing: Cable.

Character of housing: Permanent for all employes.

Capacity of tank: Two thousand (2000) gallons.

Capacity of coal-bunker: Four (4) tons.

Spread of jack arm: Eighteen (18) feet. A special short arm
should be provided.

Form of steam-shovel track: "T" rails on ties.

Length of rails for ordinary work: Six (6) feet.

Form of rail joint: Strap.

Manufacturers of steam-shovels will sometimes "guarantee" that certain of their shovels will excavate, say 3000 cubic yards of earth per day of ten hours. Even if it were possible for a shovel to fill a car at the rate of 5 cubic yards per minute, it is always impracticable to maintain such a speed, since a shovel must always wait for the shifting of cars and for the frequent shifting of the shovel itself. There are also delays due to adjustments and minor breakdowns. The best shovel records are made when the cars are large—other things being equal. The item of interest and depreciation of the plant is very large in steam-shovel work. This will be discussed further later. The cost of loading alone will usually come to between 3 and 4 c. per cubic yard. The cost of shifting the cars so as to place them successively under the shovel, haul them to the dumping place, dump them and haul them back, will generally be as much more. Gillette quotes five jobs on one railroad where the total cost for loading and hauling varied from 5.9 c. to 11.4 c. per cubic yard. But as these figures are based on *car* measurement, the cost per cubic yard in *place* measurement must be increased about one-fourth, or from 7.4 c. to 14.2 c.

109. Item 3. Hauling. The cost of hauling depends on the number of round trips per day that can be made by each vehicle employed. As the cost of each vehicle is practically the same whether it makes many trips or few, it becomes important that the number of trips should be made a maximum, and to that end there should be as little delay as possible in loading and

unloading. Therefore devices for facilitating the passage of the vehicles have a real money value.

(a) **Carts.** The average speed of a horse hauling a two-wheeled cart has been found to be 200 feet per minute, a little slower when hauling a load and a little faster when returning empty. This figure has been repeatedly verified. It means an allowance of one minute for each 100 feet (or "station") of "lead—the lead being the distance the earth is hauled." The time lost in loading, dumping, waiting to load, etc., has been found to average 4 minutes per load. Representing the number of stations (100 feet) of lead by s , the number of loads handled in 10 hours (600 minutes) would be $600 \div (s+4)$. The number of loads per cubic yard, measured in the bank, is differentiated by Morris into three classes, viz.:

3 loads per cubic yard in descending hauling;
 $3\frac{1}{2}$ " " " " " level hauling; and
 4 " " " " " ascending hauling.

Attempts have been made to estimate the effect of the grade of the roadway by a theoretical consideration of its rate, and of the comparative strength of a horse on a level and on various grades. While such computations are always practicable on a railway (even on a temporary construction track), the traction on a temporary earth roadway is always very large and so very variable that any refinements are useless. On railroad earthwork the hauling is generally nearly level or it is *descending*—forming embankments on low ground with material from cuts in high ground. The only common exception occurs when an embankment is formed from borrow-pits on low ground. One method of allowing for ascending grade is to add to the horizontal distance 14 times the difference of elevation for work with carts and 24 times the difference of elevation for work with wheelbarrows, and use that as the lead. For example, using carts, if the lead is 300 feet and there is a difference of elevation of 20 feet, the lead would be considered equivalent to $300 + (14 \times 20) = 580$ feet on a level.

Trautwine assumes the average load for all classes of work to be $\frac{1}{3}$ cubic yard, which figure is justified by large experience. Using one figure for all classes of work simplifies the calculations and gives the number of cubic yards carried per day of 10 hours equal to $\frac{600}{3(s+4)}$. Dividing the cost of a cart per day by the

(b) Wagons. For longer leads (i.e., from $\frac{1}{3}$ to $\frac{2}{3}$ of a mile) wagons drawn by two horses have been found most economical. The wagons have bottoms of loose thick narrow boards and are unloaded very easily and quickly by lifting the individual boards and breaking up the continuity of the bottom, thus depositing the load directly underneath the wagon. The capacity is about one cubic yard. The cost may be estimated on the same principle as that for carts.

The number of wagon trips per 10 hours will depend somewhat on the management of the shovellers. Too many shovellers per wagon is not economical, measured in yards shovelled per man, although it may reduce the time consumed in loading any one wagon. At an average figure of 20 cubic yards, measured in place, per shoveller per 10 hours, seven shovellers would load 14 cubic yards per hour or one cubic yard in 4.3 minutes. This would be the allowance for a wagon with a capacity of about $1\frac{1}{3}$ yards of loose earth. Adding time for unloading, waiting to load and other possible "lost time," there is probably a total of six minutes. This figure will vary very considerably according to the number of shovellers per wagon, the capacity of the wagon, the type of wagon (whether self-dumping) and other details in the method of management. Adopting six minutes as the time used for loading, unloading, and other "lost time," the formula becomes.

$$\text{Cost per cubic yard of hauling in wagons} = \frac{C(s+6)}{600c}, \dots \quad (71a)$$

in which C is the cost of the wagon, team and driver per day of 10 hours; s is the distance hauled in stations of 100 feet, and c is the capacity of the wagon in cubic yards, *place measurement*, which should be about three fourths of the nominal capacity of the wagon for earth and about sixty per cent when handling rock.

(c) Wheelbarrows. Gillette has computed from observations that a man will trundle a wheelbarrow at the rate of 250 feet per minute or 1.25 *stations* of lead per minute for the round trip. The time required for loading is estimated at $2\frac{1}{4}$ minutes and for unloading, adjusting wheeling planks, short rests, etc., $\frac{3}{4}$ minute, or a total of three minutes per trip for all purposes except hauling. Gillette allows for a load only $1/15$ cubic yard,

measured in place, or about 1/11 yard, 2.5 cubic feet, on the wheelbarrow. With notation as before

$$\left. \begin{array}{l} \text{Cost per cubic yard of loading and} \\ \text{hauling earth in wheelbarrows} \end{array} \right\} = \frac{C \times 15(1.25s + 3)}{600}. \quad (72)$$

In this equation C is the cost of both loading and hauling, and usually includes the allowance (Item 7) for the cost, repairs and depreciation of the wheelbarrows, whose service is very short lived. Trautwine estimates this at five cents per day or a total of \$1.05 for labor and wheelbarrow.

The number of wheelbarrow loads required for a cubic yard of rock, measured in place, is about twenty-four. The time required for loading should also be increased about one fourth; the time required for all purposes except hauling is therefore about 3.75 minutes, and the corresponding equation becomes

$$\left. \begin{array}{l} \text{Cost per cubic yard of loading and} \\ \text{hauling rock in wheelbarrows} \end{array} \right\} = \frac{C \times 24(1.25s + 3.75)}{600}. \quad (72a)$$

(d) **Scrapers.** These are made in three general ways, "buck" scrapers, "drag" scrapers and "wheeled" scrapers. The buck scraper in its original form consisted merely of a wide plank, shod with an iron strap on the lower edge and provided with a pole and a small platform on which the driver may stand to weight it down. The earth is not loaded on to any receptacle and carried, but is merely pushed over the ground. Notwithstanding the apparent inefficiency of the method, its extreme simplicity has caused its occasional adoption for the construction of canal embankments out of material from the bed of the canal. The occasions are rare when their use for railroad work would be practicable, and even then drag scrapers would probably be preferable.

A drag scraper is an immense "scoop shovel" about three feet long and three feet wide. There are usually two handles and a bail in front by which it is dragged by a team of horses. The nominal capacity varies from 7.5 cubic feet for the largest sizes, down to 3 cubic feet for the "one-horse" size, but these figures must be discounted by perhaps 40 or 50% for the actual average volume (as measured in the cut) loaded on during one scoop. The expansion of the earth during loosening is alone respons-

ible for a discount of 25%. These scrapers cost from \$10 to \$18.

A wheeled scraper is essentially an extra-large drag scraper which may be raised by a lever and carried on a pair of large wheels. Their nominal capacity ranges from 10 to 17 cubic feet, which should usually be liberally discounted when estimating output. They are loaded by dropping the scoop so that it scrapes up its load. The lever raises the scoop so that the load is carried on wheels instead of being dragged. At the dump the scoop is tipped so as to unload it. The movement of the scraper is practically continuous. They cost from \$40 to \$75. Their advantages over drag scrapers consist (1) in their greater capacity, (2) in the economy of transporting the load on wheels instead of by dragging, and (3) in the far greater length of haul over which the earth may be economically handled.

Morris estimated the speed of drag scrapers to be 140 feet per minute, or 70 feet of *lead* per minute. The "lead" should be here interpreted as the average distance from the center of the pit to the center of the dump. Gillette declares the speed to be 220 feet per minute. Some of this variation may be due to differences in the method of measuring the distance actually travelled, especially when the lead is very short, since the scraper teams must always travel a considerable extra distance at each end in order to turn around most easily. This extra distance is practically constant whether the lead is long or short. Gillette quotes an instance where the length of lead was actually about 20 feet, but the scraper teams travelled about 150 feet for each load carried. On this account Gillette adopts a minimum of 75 feet of lead no matter how short the lead actually may be. Of course the speed depends considerably on how strictly the men are kept to their work and also on the care which may be taken to obtain a full load for each scraper. As a compromise between Morris's and Gillette's estimates we may adopt the convenient rate of speed of 200 feet per minute, or 100 feet of lead per minute. There should also be allowed for the time lost in loading and unloading and for travelling the extra distance travelled by the teams in making the circuit, $1\frac{2}{3}$ minutes. Allowing the average value of seven loads per cubic yard and letting *C* represent the cost of scraper team and driver per day, we have for the cost as follows:

$$\left. \begin{array}{l} \text{Cost per cubic yard of loading and} \\ \text{hauling earth in drag scrapers} \end{array} \right\} = \frac{C \times 7(s + 1\frac{1}{3})}{600}. \quad . \quad . \quad (73)$$

In this formula C should include the cost of not only the driver, team, and scraper, but also the proper proportion of the wages of an extra man, who assists each driver in loading his scraper, and whose wages should be divided among the two (or three) scrapers to which he is assigned. Scraper work nearly always implies ploughing, the cost of which should be computed as under Item 1.

When a low embankment is formed from borrow-pits on each side of the road, it may be done with scrapers, which move from one borrow-pit to the other, taking a load alternately from each side to the center and making but one half turn for each load carried. This reduces the time lost in turning by one third of a minute and reduces the constant in the numerator in Eq. (73) from $1\frac{1}{3}$ to 1. In this case the lead will usually be not greater than 75 feet, and therefore, if we consider this as a minimum value, s will ordinarily equal .75 and the quantity in the parenthesis will equal 1.75.

When using wheeled scrapers the catalogue capacity, which varies from 9 or 10 feet for a No. 1 scraper to 16 or 17 feet for a No. 3 scraper, must be reduced to 5 loads per cubic yard (place measurement) for a No. 1 scraper and to $2\frac{1}{2}$ loads per cubic yard for a No. 3, not only on account of the expansion of the earth during loosening, but also on account of the impracticability of loading these scrapers to their maximum nominal capacity. When the haul or lead for wheeled scrapers is 300 feet or over, it will be justifiable to employ shovellers to fill up the bowl of the shovel, especially when the soil is tough and when it is impracticable to fill the shovel even approximately full by the ordinary method. A snatch team to assist in loading the scrapers is also economical, especially with the larger scrapers. The proportionate number of snatch teams to the total number of scrapers of course depends on the length of haul. The cost of these extra shovellers and extra snatch teams must be divided proportionally among the number of scrapers assisted, in determining the value C in the formula given below. The extra time to be allowed on account of turning, loading, and dumping is about $1\frac{1}{2}$ minutes. The speed is considered one station of lead per minute as before. If we call C the average

daily cost of one scraper and n the capacity of the scraper, or the number of loads per cubic yard, we may write the following formula:

$$\left. \begin{array}{l} \text{Cost per cubic yard of loading and} \\ \text{hauling earth in wheeled scrapers} \end{array} \right\} = \frac{C \times n(s + 1\frac{1}{2})}{600}. \quad \dots (73a)$$

(e) **Cars and horses.** The items of cost by this method are (a) charge for horses employed, (b) charge for men employed strictly in hauling, (c) charge for shifting rails when necessary, (d) repairs, depreciation, and interest on cost of cars and track. Part of this cost should strictly be classified under items 5 and 7, mentioned in § 106, but it is perhaps more convenient to estimate them as follows:

The traction of a car on rails is so very small that grade resistance constitutes a very large part of the total resistance if the grade is 1% or more. For all ordinary grades it is sufficiently accurate to say that the grade resistance is to the gross weight as the rise is to the distance. If the distance is supposed to be measured along the slope, the proportion is strictly true; i.e., on a 1% grade the grade resistance is 1 lb. per 100 of weight or 20 lbs. per ton. If the resistance on a level at the usual velocity is $\frac{1}{20}$, a grade of 1:120 (0.83%) will exactly double it. If the material is hauled *down* a grade of 1:120, the cars will run by gravity after being started. The work of hauling will then consist practically of hauling the empty cars up the grade. The grade resistance depends only on the rate of grade and the weight, but the tractive resistance will be *greater per ton of weight* for the unloaded than for the loaded cars. The tractive power of a horse is less on a grade than on a level, not only because the horse raises his own weight in addition to the load, but is anatomically less capable of pulling on a grade than on a level. In general it will be possible to plan the work so that loaded cars need not be hauled *up* a grade, unless an embankment is to be formed from a low borrow-pit, in which case another method would probably be advisable. These computations are chiefly utilized in designing the method of work—the proportion of horses to cars. An example may be quoted from English practice (Hurst), in which the cars had a capacity of $3\frac{1}{2}$ cubic yards, weighing 30 cwt. empty. Two horses took five “*wagons*” $\frac{3}{4}$ of a mile on a level

railroad and made 15 journeys per day of 10 hours, i.e., they handled 250 yards per day. In addition to those on the "straight road," another horse was employed to make up the train of loaded wagons. With a short lead the straight-road horses were employed for this purpose. In the above example the number of men required to handle these cars, shift the tracks, etc., is not given, and so the exact cost of the above work cannot be analyzed. It may be noticed that the two horses travelled $22\frac{1}{2}$ miles per day, drawing in one direction a load, including the weight of the cars, of about 57,300 lbs., or 28.65 net tons. Allowing $\frac{1}{120}$ as the necessary tractive force, it would require a pull of 477.5 lbs., or 239 lbs. for each horse. With a velocity of 220 feet per minute this would amount to $1\frac{1}{2}$ horse-power per horse, exerted for only a short time, however, and allowing considerable time for rest and for drawing only the empty cars. Gillette claims that the rolling resistance for such cars on a contractor's track should be considered as 40 lbs. per ton (the equivalent of a 2% grade) and quotes many figures to support the assertion. Unquestionably the resistance on tracks with very light rails, light ties with wide spacing and no tamping, would be very great and might readily amount to 40 lbs. per ton. In the above case, the resistance could not have been much if any over $\frac{1}{120}$. A resistance of 40 lbs. per ton would have required each horse to pull about 573 lbs. for nearly five hours per day, beside pulling the empty cars the rest of the time. This is far greater exertion than any ordinary horse can maintain. The cars generally used in this country have a capacity of $1\frac{1}{2}$ cubic yards and cost about \$65 apiece. Besides the shovellers and dumping-gang, several men and a foreman will be required to keep the track in order and to make the constant shifts that are necessary. Two trains are generally used, one of which is loaded while the other is run to the dump. Some passing-place is necessary, but this is generally provided by having a switch at the cut and running the trains on each track alternately. This insures a train of cars always at the cut to keep the shovellers employed. The cost of hauling per cubic yard can only be computed when the number of laborers, cars, and horses employed are known, and these will depend on the lead, on the character of the excavation, on the grade, if any, etc., and must be so proportioned that the shovellers need not wait for cars to fill, nor the dumping-gang

for material to handle, nor the horses and drivers for cars to haul. Much skill is necessary to keep a large force in smooth running order.

(f) **Cars and locomotives.** 30-lb. rails are the lightest that should be used for this work, and 35- or 40-lb. rails are better. One or two narrow-gauge locomotives (depending on the length of haul), costing about \$2500 each, will be necessary to handle two trains of about 15 cars each, the cars having a capacity of about 2 cubic yards and costing about \$100 each. Some cars can be obtained as low as \$70. A force of about five men and a foreman will be required to shift the tracks. The track-shifters, except the foreman, may be common laborers. The dumping-gang will require about seven men. Even when the material is all taken *down* grade the grades may be too steep for the safe hauling of loaded cars down the grade, or for hauling empty cars up the grade. Under such circumstances temporary trestles are necessary to reduce the grade. When these are used, the uprights and bracing are left in the embankment—only the stringers being removed. This is largely a necessity, but is partially compensated by the fact that the trestle forms a core to the embankment which prevents lateral shifting during settlement. The average speed of the trains may be taken as 10 miles per hour or 5 miles of lead per hour. The time lost in loading and unloading is estimated (Trautwine) as 9 minutes or .15 of an hour. The number of trips per day of 10 hours will equal $\frac{10}{\frac{1}{5}(\text{miles of lead}) + .15}$ or $\frac{50}{(\text{miles of lead}) + .75}$. Of course this quotient *must* be a whole number. Knowing the number of trains and their capacity, the total number of cubic yards handled is known, which, divided into the total daily cost of the trains, will give the cost of hauling per yard. The daily cost of a train will include

(a) Wages of engineer, who frequently fires his own engine;
 (b) Fuel, about $\frac{1}{4}$ to 1 ton of bituminous coal, depending on work done;

(c) Water, a very variable item, frequently costing \$3 to \$5 per day;

(d) Repairs, variable, frequently at rate of 50 to 60% per year;

(e) Interest on cost and depreciation, 16 to 40%.

To these must be added, to obtain the total cost of haul,

(f) Wages of the gang employed in shifting track.

The above calculation for the number of train loads depends on the assumption that 9 minutes is total time lost by a locomotive for each round trip. If the haul is very short it may readily happen that a steam-shovel cannot fill one train of cars before the locomotive has returned with a load of empties and is ready to haul a loaded train away. The estimation of the number of train loads is chiefly useful in planning the work so as to have every tool working at its highest efficiency. Usually the capacity of the steam-shovel or the ability to promptly "spot" the cars under the shovel is the real limiting agent which determines the daily output.

110. Choice of method of haul dependent on distance. In light side-hill work in which material need not be moved more than 12 or 15 feet, i.e., moved *laterally* across the roadbed, the earth may be moved most cheaply by mere shovelling. Beyond 12 feet scrapers are more economical. At about 100 feet drag-scrappers and wheelbarrows are equally economical. Between 100 and 200 feet wheelbarrows are generally cheaper than either carts or drag-scrappers, but wheeled scrapers are always cheaper than wheelbarrows. Beyond 500 feet two-wheeled carts become the most economical up to about 1700 feet; then four-wheeled wagons become more economical up to 3500 feet. Beyond this cars on rails, drawn by horses or by locomotives, become cheaper. The economy of cars on rails becomes evident for distances as small as 300 feet provided the volume of the excavation will justify the outlay. Locomotives will always be cheaper than horses and mules, providing the work to be done is of sufficient magnitude to justify the purchase of the necessary plant and risk the loss in selling the plant ultimately as second-hand equipment, or keeping the plant on hand and idle for an indefinite period waiting for other work. Horses will not be economical for distances much over a mile. For greater distances locomotives are more economical, but the question of "limit of profitable haul" (§ 116) must be closely studied, as the circumstances are certainly not common when it is advisable to haul material much over a mile.

111. Item 4. SPREADING. The cost of spreading varies with the method employed in dumping the load. When the earth is

tipped over the edge of an embankment there is little if any necessary work. Trautwine allows about $\frac{1}{4}$ c. per cubic yard for keeping the dumping-places clear and in order. This would represent the wages of one man at \$1 per day attending to the unloading of 1200 two-wheeled carts each carrying $\frac{1}{3}$ cubic yard. 1200 carts in 10 hours would mean an average of two per minute, which implies more rapid and efficient work than may be depended on. The allowance is probably too small. When the material is dumped in layers some levelling is required, for which Trautwine allows 50 to 100 cubic yards as a fair day's work, costing from 1 to 2 cents per cubic yard. The cost of spreading will not ordinarily exceed this and is frequently nothing—all depending on the method of unloading. It should be noted that Mr. Morris's examples and computations (Jour. Franklin Inst., Sept. 1841) disregard altogether any special charge for this item.

112. Item 5. KEEPING ROADWAYS IN ORDER. This feature is important as a measure of true economy, whatever the system of transportation, but it is often neglected. A petty saving in such matters will cost many times as much in increased labor in hauling and loss of time. With some methods of haul the cost is best combined with that of other items.

(a) **Wheelbarrows.** Wheelbarrows should generally be run on planks laid on the ground. The adjusting and shifting of these planks is done by the wheelers, and the time for it is allowed for in the " $\frac{3}{4}$ minute for short rests, adjusting the wheeling plank, etc." The actual cost of the planks must be added, but it would evidently be a very small addition per cubic yard in a large contract. When the wheelbarrows are run on planks placed on "horses" or on trestles the cost is very appreciable; but the method is frequently used with great economy. The variations in the requirements render any general estimate of such cost impracticable.

(b) **Carts and wagons.** The cost of keeping roadways in order for carts and wagons is sometimes estimated merely as so much per cubic yard, but it is evidently a function of the *lead*. The work consists in draining off puddles, filling up ruts, picking up loose stones that may have fallen off the loads, and in general doing everything that will reduce the traction as much as possible. Temporary inclines, built to avoid excessive grade

at some one point, are often measures of true economy. Trautwine suggests $\frac{1}{10}$ c. per cubic yard per 100 feet of lead for earth-work and $\frac{2}{10}$ c. for rockwork, as an estimate for this item when carts are used.

(c) **Cars.** When cars are used a shifting-gang, consisting of a foreman and several men (say five), are constantly employed in shifting the track so that the material may be loaded and unloaded where it is desired. The average cost of this item may be estimated by dividing the total daily cost of this gang by the number of cubic yards handled in one day.

112a. Item 6. TRIMMING CUTS TO THEIR PROPER CROSS-SECTION. This process, often called "sand-papering," must be treated as an expense, since the payment received for the very few cubic yards of earth excavated is wholly inadequate to pay for the work involved. Gillette quotes bids of 2 cents per *square* yard of surface trimmed, and from this argues that, for *average* excavations, it adds to the cost *four* cents per cubic yard of the total excavation. The shallower the cut the greater is the proportionate cost. Of course the actual cost to the contractor will depend largely on the accuracy of outline demanded by the engineer or inspector.

113. Item 7. REPAIRS, WEAR, DEPRECIATION, AND INTEREST ON COST OF PLANT. The amount of this item evidently depends upon the character of the soil—the harder the soil the worse the wear and depreciation. The *interest on cost* depends on the current borrowing value of money. The estimate for this item has already been included in the allowances for horses, carts, ploughs, harness, wheelbarrows, steam-shovels, etc. Trautwine estimates $\frac{1}{4}$ c. per cubic yard for picks and shovels. Depreciation is generally a large percentage of the cost of earth-working tools, the life of all being limited to a few years, and of many tools to a few months.

114. Item 8. SUPERINTENDENCE AND INCIDENTALS. The incidentals include the cost of water-boys, timekeepers, watchmen, blacksmiths, fences, and other precautions to protect the public from possible injury, cost of casualty insurance for workmen, etc. Although the cost of some of these sub-items may be definitely estimated, others are so uncertain that it is only possible to make a lump estimate and add say 5 to 7% of the sum of the previous items for this item.

115. Contractor's profit and contingencies. The word "contingencies" here refers to the abnormal expenses caused by freshets, continued wet weather, and "hard luck," as distinguished from mere incidentals which are really normal expenses. They are the expenses which literally cannot be foreseen, and on which the contractor must "take chances." They are therefore included with the expected profit. The allowance for these two elements combined is variously estimated up to 25% of the previously estimated cost of the work, according to the sharpness of the competition, the contractor's confidence in the accuracy of his estimates, and the possible uncertainty as to true cost owing to unfavorable circumstances. The contractor's real profit may vary considerably from this. He often pays clerks, boards and lodges the laborers in shanties built for the purpose, or keeps a supply-store, and has various other items both of profit and expense. His profit is largely dependent on skill in so handling the men that all can work effectively without interference or delays in waiting for others. An unusual season of bad weather will often affect the cost very seriously. It is a common occurrence to find that two contractors may be working on the same kind of material and under precisely similar conditions and at the same price, and yet one may be making money and the other losing it—all on account of difference of management.

116. Limit of profitable haul. As intimated in §§ 103 and 110, there is with every method of haul a limit of distance beyond which the expense for excessive hauling will exceed the loss resulting from borrowing and wasting. This distance is somewhat dependent on local conditions, thus requiring an independent solution for each particular case, but the general principles involved will be about as follows: Assume that it has been determined, as in Fig. 62, that the cut and fill will exactly balance between two points, as between e and x , assuming that, as indicated in § 101 (9), a trestle has been introduced between s and t , thus altering the mass curve to *Estxn* . . . Since there is a balance between A' and C' , the material for the fill between C' and e' must be obtained either by "borrowing" in the immediate neighborhood or by transportation from the excavation

between z' and n' . If cut and fill have been approximately balanced in the selection of grade line, as is ordinarily done, borrowing material for the fill $C'e'$ implies a wastage of material at the cut $z'n'$. To compare the two methods, we may place against the plan of borrowing and wasting, (a) cost, if any, of extra right of way that may be needed from which to obtain earth for the fill $C'e'$; (b) cost of loosening, loading, hauling a distance equal to that between the centers of gravity of the borrow-pit and of the fill, and the other expenses incidental to borrowing M cubic yards for the fill $C'e'$; (c) cost of loosening, loading, hauling a distance equal to that between the centers of gravity of the cut $z'n'$ and of the spoil-bank, and the other expenses incidental to wasting M cubic yards at the cut $z'n'$; (d) cost, if any, of land needed for the spoil-bank. The cost of the other plan will be the cost of loosening, loading, hauling (the hauling being represented by the trapezoidal figure $Cexn$), and the other expenses incidental to making the fill $C'e'$ with the material from the cut $z'n'$, the amount of material being M cubic yards, which is represented in the figure by the vertical ordinate from e to the line Cn . The difference between these costs will be the cost, if any, of land for borrow-pit and spoil-bank *plus* the cost of loosening, loading, etc. (except hauling and roadways) of M cubic yards, *minus* the difference in cost of the excessive haul from Ce to xn and the comparatively short hauls from borrow-pit and to spoil-bank.

As an illustration, taking some of the estimates previously given for operating with average material, the cost of all items, except hauling and roadways, would be about as follows: loosening, with plough, 1.2 c., loading 5.0 c., spreading 1.5 c., wear, depreciation, etc., .25 c., superintendence, etc., 1.5 c.; total 8.95 c. Suppose that the haul for both borrowing and wasting averages 100 feet or 1 station. Then the cost of haul per yard, using carts, would be ($\S$ 109, a) $[125 \times 3(1+4)] \div 600 = 3.125$ c. The cost of roadways would be about 0.1 c. per yard, making a total of 3.225 c. per cubic yard. Assume $M=10000$ cubic yards and the area $Cexn=180000$ yards-stations or the equivalent of 10000 yards hauled 1800 feet. This haul would cost $[125 \times 3(18+4)] \div 600 = 13.75$ c. per cubic yard. The cost of roadways will be $18 \times .1$ or 1.8 c., making a total of 15.55 c. for hauling and roadways. The difference of cost of hauling and roadways will be $15.55 - (2 \times 3.225) = 9.10$ c. per yard or \$910

for the 10000 yards. Offsetting this is the cost of loosening, etc., 10000 yards, at 8.95 c., costing \$895. These figures may be better compared as follows:

LONG HAUL.	{	Loosening, etc., 10000 yards, @ 8.95 c.	\$ 895.
		Hauling, " 10000 " @ 15.55 c.	1555.
			\$2450.
BORROWING AND WASTING.	{	Loosening, etc., 10000 yards (borrowed), @ 8.95 c.	\$895.
		" " 10000 " (wasted), @ 8.95 c.	895.
		Hauling, etc., 10000 " (borrowed), @ 3.225 c.	322.50
		" " 10000 " (wasted), @ 3.225 c.	322.50
			\$2435.00

These costs are practically balanced, but no allowance has been made for right of way. If any considerable amount had to be paid for that, it would decide this particular case in favor of the long haul. This shows that *under these conditions* 1800 feet is *about* the limit of profitable haul, the land costing nothing extra.

BLASTING.

117. Explosives. The effect of blasting is due to the extremely rapid expansion of a gas which is developed by the decomposition of a very small amount of solid matter. Blasting compounds may be divided into two general classes, (a) slow-burning and (b) detonating. Gunpowder is a type of the slow-burning compounds. These are generally ignited by heat; the ignition proceeds from grain to grain; the heat and pressure produced are comparatively low. Nitro-glycerine is a type of the detonating compounds. They are exploded by a shock which *instantaneously* explodes the whole mass. The heat and pressure developed are far in excess of that produced by the explosion of powder. Nitro-glycerine is so easily exploded that it is very dangerous to handle. It was discovered that if the nitro-glycerine was absorbed by a spongy material like infusorial earth, it was much less liable to explode, while its power when actually exploded was practically equal to that of the amount of pure nitro-glycerine contained in the dynamite, which is the name given to the mixture of nitro-glycerine and infusorial earth. Nitro-glycerine is expensive; many other explosive chemical compounds which properly belong to the slow-burning

class are comparatively cheap. It has been conclusively demonstrated that a mixture of nitro-glycerine and some of the cheaper chemicals has a greater explosive force than the sum of the strengths of the component parts when exploded separately. Whatever the reason, the fact seems established. The reason is possibly that the explosion of the nitro-glycerine is sufficiently powerful to produce a *detonation* of the other chemicals, which is impossible to produce by ordinary means, and that this explosion caused by detonation is more powerful than an ordinary explosion. The majority of the explosive compounds and "powders" on the market are of this character—a mixture of 20 to 60 per cent. of nitro-glycerine with variable proportions of one or more of a great variety of explosive chemicals.

The choice of the explosive depends on the character of the rock. A hard brittle rock is most effectively blasted by a detonating compound. The rapidity with which the full force of the explosive is developed has a shattering effect on a brittle substance. On the contrary, some of the softer tougher rocks and indurated clays are but little affected by dynamite. The result is but little more than an enlargement of the blast-hole. Quarrying must generally be done with blasting-powder, as the quicker explosives are too shattering. Although the results obtained by various experimenters are very variable, it may be said that pure nitro-glycerine is eight times as powerful as black powder, dynamite (75% nitro-glycerine) six times, and gun-cotton four to six times as powerful. For open work where time is not particularly valuable, black powder is by far the cheapest, but in tunnel-headings, whose progress determines the progress of the whole work, dynamite is so much more effective and so expedites the work that its use becomes economical.

118. Drilling. Although many very complicated forms of drill-bars have been devised, the best form (with slight modifications to suit circumstances) is as shown in Fig. 64, (a) and (b). The width should flare at the bottom (a) about 15 to 30%. For hard rock the curve of the edge should be somewhat flatter and for soft rock somewhat more curved than shown, Fig. 64, (a). Sometimes the angle of the two faces is varied from that given, Fig. 64, (b), and occasionally the edge is purposely blunted so as to give a crushing rather than a cutting effect. The drills will require sharpening for each 6 to 18 inches depth of hole, and will require a new edge to be worked every 2 to 4 days.

For drilling vertical holes the *churn-drill* is the most economical. The drill-bar is of iron, about 6 to 8 feet long, $1\frac{1}{4}$ " in diameter, weighs about 25 to 30 lbs., and is shod with a piece of steel welded on. The bar is lifted a few inches between each blow, turned partially around, and allowed to fall, the impact doing the work. From 5 to 15 feet of holes, depending on the character of the rock, is a fair day's work—10 hours. In very soft rocks even more than this may be done. This method is

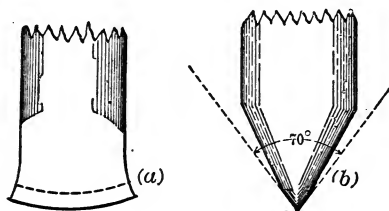


FIG. 64.

inapplicable for inclined holes or even for vertical holes in confined places, such as tunnel-headings. For such places the only practical *hand* method is to use hammers. This may be done by light drills and light hammers (one-man work), or by heavier drills held by one man and struck by one or two men with heavy hammers. The conclusion of an exhaustive investigation as to the relative economy of light or heavy hammers is that the light-hammer method is more economical for the softer rocks, the heavy-hammer method is more economical for the harder rocks, but that the light-hammer method is always more expeditious and hence to be preferred when time is important.

The subject of machine rock-drills is too vast to be treated here. The method is only practicable when the amount of work to be done is large, and especially when time is valuable. The machines are generally operated by compressed air for tunnel-work, thus doing the additional service of supplying fresh air to the tunnel-headings where it is most needed. The cost per foot of hole drilled is quite variable, but is usually somewhat less than that of hand-drilling—sometimes but a small fraction of it.

119. Position and direction of drill-holes. As the cost of drilling holes is the largest single item in the total cost of blasting, it is necessary that skill and judgment should be used in so

locating the holes that the blasts will be most effective. The greatest effect of a blast will evidently be in the direction of the "line of least resistance." In a strictly homogeneous material this will be the shortest line from the center of the explosive to the surface. The variations in homogeneity on account of laminations and seams require that each case shall be judged according to experience. In open-pit blasting it is generally easy to obtain two and sometimes three exposed faces to the

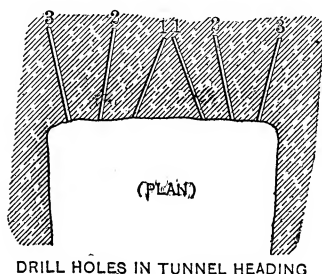


FIG. 65.

rock, making it a simple matter to drill holes so that a blast will do effective work. When a solid face of rock must be broken into, as in a tunnel-heading, the work is necessarily ineffectual and expensive. A conical or wedge-shaped mass will first be blown out by simultaneous blasts in the holes marked 1, Fig. 65; blasts in the holes marked 2 and 3 will then complete the cross-section of the heading. A great saving in cost may often be secured by skilfully taking advantage of seams, breaks, and irregularities. When the work is economically done there is but little noise or throwing of rock, a covering of old timbers and branches of trees generally sufficing to confine the smaller pieces which would otherwise fly up.

120. Amount of explosive. The amount of explosive required varies as the cube of the line of least resistance. The best results are obtained when the line of least resistance is $\frac{3}{4}$ of the depth of the hole; also when the powder fills about $\frac{1}{3}$ of the hole. For average rock the amount of powder required is as follows:

Line of least resistance.....	2 ft.	4 ft.	6 ft.	8 ft.
Weight of powder.....	$\frac{1}{4}$ lb.	2 lbs.	$6\frac{3}{4}$ lbs.	16 lbs.

Strict compliance with all of the above conditions would require that the diameter of the hole should vary for every case. While this is impracticable, there should evidently be some variation in the size of the hole, depending on the work to be done. For example, a 1" hole, drilled 2' 8" deep, with its line of least resistance 2'. and loaded with $\frac{1}{4}$ lb. of powder, would

be filled to a depth of $9\frac{1}{2}$ ", which is nearly $\frac{1}{3}$ of the depth. A 3" hole, drilled 8' deep, with its line of least resistance 6', and loaded with $6\frac{3}{4}$ lbs. of powder, would be filled to a depth of over 28", which is also nearly $\frac{1}{3}$ of the depth. One pound of blasting-powder will occupy about 28 cubic inches. Quarrying necessitates the use of numerous and sometimes repeated light charges of powder, as a heavy blast or a powerful explosive like dynamite is apt to shatter the rock. This requires more powder to the cubic yard than blasting for mere excavation, which may usually be done by the use of $\frac{1}{4}$ to $\frac{1}{3}$ lb. of powder per cubic yard of easy open blasting. On account of the great resistance offered by rock when blasted in headings in tunnels, the powder used per cubic yard will run up to 2, 4, and even 6 lbs. per cubic yard. As before stated, nitro-glycerine is about eight times (and dynamite about six times) as powerful as the same *weight* of powder.

121. Tamping. Blasting-powder and the slow-burning explosives require thorough tamping. Clay is probably the best, but sand and fine powdered rock are also used. Wooden plugs, inverted expansive cones, etc., are periodically reinvented by enthusiastic inventors, only to be discarded for the simpler methods. Owing to the extreme rapidity of the development of the force of a nitro-glycerine or dynamite explosion, tamping is not so essential with these explosives, although it unquestionably adds to their effectiveness. Blasting under water has been effectively accomplished by merely pouring nitro-glycerine into the drilled holes through a tube and then exploding the charge without any tamping except that furnished by the superincumbent water. It has been found that air-spaces about a charge make a material reduction in the effectiveness of the explosion. It is therefore necessary to carefully ram the explosive into a solid mass. Of course the liquid nitro-glycerine needs no ramming, but dynamite should be rammed with a *wooden* rammer. Iron should be carefully avoided in ramming gunpowder. A copper bar is generally used.

122. Exploding the charge. Black powder is generally exploded by means of a fuse which is essentially a cord in which there is a thin vein of gunpowder, the cord being protected by tar, extra linings of hemp, cotton, or even gutta-percha. The fuse is inserted into the middle of the charge, and the tamping carefully packed around it so that it will not be injured. To

produce the detonation required to explode nitro-glycerine and dynamite, there must be an initial explosion of some easily ignited explosive. This is generally accomplished by means of caps containing fulminating-powder which are exploded by electricity. The electricity (in one class of caps) heats a very fine platinum wire to redness, thereby igniting the sensitive powder, or (in another class) a spark is caused to jump through the powder between the ends of two wires suitably separated. Dynamite can also be exploded by using a small cartridge of gunpowder which is itself exploded by an ordinary fuse.

123. Cost. Trautwine estimates the cost of blasting (for mere excavation) as averaging 45 cents per cubic yard, falling as low as 30 cents for easy but *brittle* rock, and running up to 60 cents and even \$1 when the cutting is shallow, the rock especially tough, and the strata unfavorably placed. Soft tough rock frequently requires more powder than harder brittle rock.

124. Classification of excavated material. The classification of excavated material is a fruitful source of dispute between contractors and railroad companies, owing mainly to the fact that the variation between the softest earth and the hardest rock is so gradual that it is very difficult to describe distinctions between different classifications which are unmistakable and indisputable. The classification frequently used is (a) earth, (b) loose rock, and (c) solid rock. As blasting is frequently used to loosen "loose rock" and even "earth" (if it is frozen), the fact that blasting is employed cannot be used as a criterion, especially as this would (if allowed) lead to unnecessary blasting for the sake of classifying material as rock.

Earth. This includes clay, sand, gravel, loam, decomposed rock and slate, boulders or loose stones not greater than 1 cubic foot (3 cubic feet, P. R. R.), and sometimes even "hard-pan." In general it will signify material which *can* be loosened by a plough with two horses, or with which one picker can keep one shoveller busy.

Loose rock. This includes boulders and loose stones of more than one cubic foot and less than one cubic yard; stratified rock, not more than six inches thick, separated by a stratum of clay; also all material (not classified as earth) which may be loosened by pick or bar and which "*can* be quarried without blasting, although blasting may occasionally be resorted to."

Solid rock includes all rock found in masses of over one cubic yard which cannot be removed except by blasting.

It is generally specified that the engineer of the railroad company shall be the judge of the classification of the material, but frequently an appeal is taken from his decisions to the courts.

125. Specifications for earthwork. The following specifications, issued by the Norfolk and Western R. R., represent the average requirements. It should be remembered that very strict specifications invariably increase the cost of the work, and frequently add to the cost more than is gained by improved quality of work.

1. The grading will be estimated and paid for by the cubic yard, and will include clearing and grubbing, and all open excavations, channels, and embankments required for the formation of the roadbed, and for turnouts and sidings; cutting all ditches or drains about or contiguous to the road; digging the foundation-pits of all culverts, bridges, or walls; reconstructing turnpikes or common roads in cases where they are destroyed or interfered with; changing the course or channel of streams; and all other excavations or embankments connected with or incident to the construction of said Railroad.

2. All grading, except where otherwise specified, whether for cuts or fills, will be measured in the excavations and will be classified under the following heads, viz.: Solid Rock, Loose Rock, Hard-pan, and Earth.

SOLID ROCK shall include all rock occurring in masses which, in the judgment of the said Engineer Maintenance of Way, may be best removed by blasting.

LOOSE ROCK shall include all kinds of shale, soapstone, and other rock which, in the judgment of the said Engineer Maintenance of Way, can be removed by pick and bar, and is soft and loose enough to be removed without blasting, although blasting may be occasionally resorted to; also, detached stone of less than one (1) cubic yard and more than one (1) cubic foot.

HARD-PAN shall consist of tough indurated clay or cemented gravel, which requires blasting or other equally expensive means for its removal, or which cannot be ploughed with less than four horses and a railroad plough, or which requires two pickers to a shoveller, the said Engineer Maintenance of Way to be the judge of these conditions.

EARTH shall include all material of an earthy nature, of whatever name or character, not unquestionably loose rock or hardpan as above defined.

POWDER. The use of powder in cuts will not be considered as a reason for any other classification than earth, unless the material in the cut is clearly other than earth under the above specifications.

3. Earth, gravel, and other materials taken from the excavations, except when otherwise directed by the said Engineer Maintenance of Way or his assistant, shall be deposited in the adjacent embankment; the cost of removing and depositing which, when the distance necessary to be hauled is not more than sixteen hundred (1600) feet, shall be included in the price paid for the excavation.

4. **EXTRA HAUL** will be estimated and paid for as follows: whenever material from excavations is necessarily hauled a greater distance than sixteen hundred (1600) feet, there shall be paid in addition to the price of excavation the price of extra haul per 100 feet, or part thereof, after the first 1600 feet; the necessary haul to be determined in each case by the said Engineer Maintenance of Way or his assistant, from the profile and cross-sections, and the estimates to be in accordance therewith.

5. All embankments shall be made in layers of such thickness and carried on in such manner as the said Engineer Maintenance of Way or his assistant may prescribe, the stone and heavy materials being placed in slopes and top. And in completing the fills to the proper grade such additional heights and fulness of slope shall be given them, to provide for their settlement, as the said Engineer Maintenance of Way, or his assistant, may direct. Embankments about masonry shall be built at such times and in such manner and of such materials as the said Engineer Maintenance of Way or his assistant may direct.

6. In procuring materials for embankments from without the line of the road, and in wasting materials from cuttings, the place and manner of doing it shall in each case be indicated by the Engineer Maintenance of Way or his assistant; and care must be taken to injure or disfigure the land as little as possible. Borrow-pits and spoil-banks must be left by the Contractor in regular and sightly shape.

7. The lands of the said Railroad Company shall be cleared to the extent required by the said Engineer Maintenance of

Way, or his assistant, of all trees, brushes, logs, and other perishable materials, which shall be destroyed by burning or deposited in heaps as the said Engineer Maintenance of Way, or his assistant, may direct. Large trees must be cut not more than two and one-half ($2\frac{1}{2}$) feet from the ground, and under embankments less than four (4) feet high they shall be cut close to the ground. All small trees and bushes shall be cut close to the ground.

8. Clearing shall be estimated and paid for by the acre or fraction of an acre.

9. All stumps, roots, logs, and other obstructions shall be grubbed out, and removed from all places where embankments occur less than two (2) feet in height; also, from all places where excavations occur and from such other places as the said Engineer Maintenance of Way or his assistant may direct.

10. Grubbing shall be estimated and paid for by the acre or fraction of an acre.

11. Contractors, when directed by the said Engineer Maintenance of Way or his assistant in charge of the work, will deposit on the side of the road, or at such convenient points as may be designated, any stone, rock, or other materials that they may excavate; and all materials excavated and deposited as above, together with all timber removed from the line of the road, will be considered the property of the Railroad Company, and the Contractors upon the respective sections will be responsible for its safe-keeping until removed by said Railroad Company, or until their work is finished.

12. Contractors will be accountable for the maintenance of safe and convenient places wherever public or private roads are in any way interfered with by them during the progress of the work. They will also be responsible for fences thrown down, and for gates and bars left open, and for all damages occasioned thereby.

13. Temporary bridges and trestles, erected to facilitate the progress of the work, in case of delays at masonry structures from any cause, or for other reasons, will be at the expense of the Contractor.

14. The line of road or the gradients may be changed in any manner, and at any time, if the said Engineer Maintenance of Way or his assistant shall consider such a change necessary or expedient; but no claim for an increase in prices of excavation

or embankment on the part of the Contractor will be allowed or considered unless made in writing before the work on that part of the section where the alteration has been made shall have been commenced. The said Engineer Maintenance of Way or his assistant may also, on the conditions last recited, increase or diminish the length of any section for the purpose of more nearly equalizing or balancing the excavations and embankments, or for any other reason.

15. The roadbed will be graded as directed by the said Engineer Maintenance of Way or his assistant, and in conformity with such breadths, depths, and slopes of cutting and filling as he may prescribe from time to time, and no part of the work will be finally accepted until it is properly completed and dressed off at the required grade.

CHAPTER IV.

TRESTLES.

126. Extent of use. Trestles constitute from 1 to 3% of the length of the average railroad. It was estimated in 1889 that there was then about 2400 miles of single-track railway trestle in the United States, divided among 150,000 structures and estimated to cost about \$75,000,000. The annual charge for maintenance, estimated at $\frac{1}{8}$ of the cost, therefore amounted to about \$9,500,000 and necessitated the annual use of perhaps 300,000,000 ft. B. M. of timber. The corresponding figures at the present time must be somewhat in excess of this. The magnitude of this use, which is causing the rapid disappearance of forests, has resulted in endeavors to limit the use of timber for this purpose. Trestles may be considered as justifiable under the following conditions:

a. Permanent trestles.

1. Those of *extreme* height—then called viaducts and frequently constructed of iron or steel, as the Kinzua viaduct, 302 ft. high.

2. Those across waterways—*e.g.*, that across Lake Pontchartrain, near New Orleans, 22 miles long.

3. Those across swamps of soft deep mud, or across a river-bottom, liable to occasional overflow.

b. Temporary trestles.

1. To open the road for traffic as quickly as possible—often a reason of great financial importance.

2. To quickly replace a more elaborate structure, destroyed by accident, on a road already in operation, so that the interruption to traffic shall be a minimum.

3. To form an earth embankment with earth brought from a distant point by the train-load, when such a measure would cost less than to borrow earth in the immediate neighborhood.

4. To bridge an opening temporarily and thus allow time to learn the regimen of a stream in order to better proportion the

size of the waterway and also to facilitate bringing *suitable* stone for masonry from a distance. In a new country there is always the double danger of either building a culvert too small, requiring expensive reconstruction, perhaps after a disastrous washout, or else wasting money by constructing the culvert unnecessarily large. Much masonry has been built of a very poor quality of stone because it could be conveniently obtained and because good stone was unobtainable except at a prohibitive cost for transportation. Opening the road for traffic by the use of temporary trestles obviates both of these difficulties.

127. Trestles vs. embankments. Low embankments are very much cheaper than low trestles both in first cost and maintenance. Very high embankments are very expensive to construct, but cost comparatively little to maintain. A trestle of equal height may cost much less to construct, but will be expensive to maintain—perhaps $\frac{1}{8}$ of its cost per year. To determine the height beyond which it will be cheaper to maintain a trestle rather than build an embankment, it will be necessary to allow for the cost of maintenance. The height will also depend on the relative cost of timber, labor, and earthwork. At the present average values, it will be found that for less heights than 25 feet the *first cost* of an embankment will *generally* be less than that of a trestle; this implies that a permanent trestle should never be constructed with a height less than 25 feet except for the reasons given in § 126. The height at which a permanent trestle is certainly cheaper than earthwork is more uncertain. A high grade line joining two hills will invariably imply at least a culvert if an embankment is used. If the culvert is built of masonry, the cost of the embankment will be so increased that the height at which a trestle becomes economical will be materially reduced. The cost of an embankment increases much more rapidly than the height—with very high embankments more nearly as the square of the height—while the cost of trestles does not increase as rapidly as the height. Although local circumstances may modify the application of any set rules, it is probably seldom that it will be cheaper to build an embankment 40 or 50 feet high than to permanently maintain a wooden trestle of that height. A steel viaduct would probably be the best solution of such a case. These are frequently used for permanent structures, especially when very high. The cost of maintenance is much less than that of wood, which makes the

use of iron or steel preferable for permanent trestles unless wood is abnormally cheap. Neither the cost nor the construction of iron or steel trestles will be considered in this chapter.

128. Two principal types. There are two principal types of wooden trestles—pile trestles and framed trestles. The great objection to pile trestles is the rapid rotting of the portion of the pile which is underground, and the difficulty of renewal. The maximum height of pile trestles is about 30 feet, and even this height is seldom reached. Framed trestles have been constructed to a height of considerably over 100 feet. They are frequently built in such a manner that any injured piece may be readily taken out and renewed without interfering with traffic. Trestles consist of two parts—the supports called “bents,” and the stringers and floor system. As the stringers and floor system are the same for both pile and framed trestles, the “bents” are all that need be considered separately.

PILE TRESTLES.

129. Pile bents. A pile bent consists generally of four piles driven into the ground deep enough to afford not only sufficient vertical resistance but also lateral resistance. On top of these piles is placed a horizontal “cap.” The caps are fastened to the tops of the piles by methods illustrated in Fig. 66. The method of fastening shown in each case should not be considered as applicable only to the particular type of pile bent used to illustrate it. Fig. 66 (*a* and *d*) illustrates a mortise-joint with a hardwood pin about 1¼” in diameter. The hole for the pin should be bored separately through the cap and the mortise, and the hole through the cap should be at a slightly higher level than that through the mortise, so that the cap will be drawn down tight when the pin is driven. Occasionally an iron dowel (an iron pin about 1½” in diameter and about 6” long) is inserted partly in the cap and partly in the pile. The use of drift-bolts, shown in Fig. 66 (*b*), is cheaper in first cost, but renders repairs and renewals very troublesome and expensive. “Split caps,” shown in Fig. 66 (*c*), are formed by bolting two half-size strips on each side of a tenon on top of the pile. Repairs are very easily and cheaply made without interference with the traffic and without injuring other pieces of the bent. The smaller pieces are more easily obtainable in a sound condition; the

decay of one does not affect the other, and the first cost is but little if any greater than the method of using a single piece. For further discussion, see § 136.

For very light traffic and for a height of about 5 feet three vertical piles will suffice, as shown in Fig. 66 (a). Up to a height

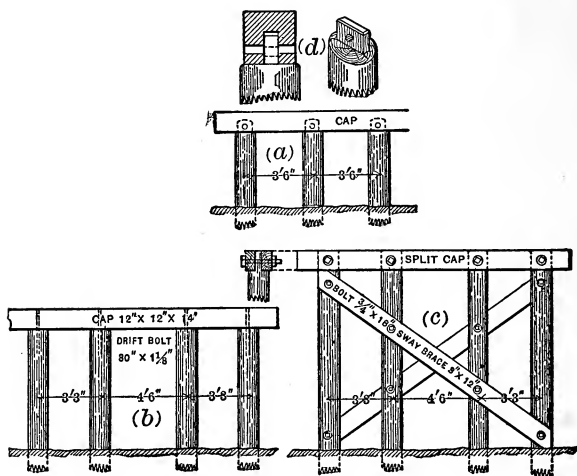


FIG. 66.

of 8 or 10 feet four piles may be used without sway-bracing, as in Fig 66 (b), if the piles have a good bearing. For heights greater than 10 feet sway-bracing is generally necessary. The outside piles are frequently driven with a batter varying from 1 : 12 to 1 : 4.

Piles are made, if possible, from timber obtained in the vicinity of the work. Durability is the great requisite rather than strength, for almost any timber is strong enough (except as noted below) and will be suitable if it will resist rapid decay. The following list is quoted as being in the order of preference on account of durability:

- | | | | |
|----------------|---------------|--------------|---------------|
| 1. Red cedar | 5. White pine | 9. White oak | 12. Black oak |
| 2. Red cypress | 6. Redwood | 10. Post-oak | 13. Hemlock |
| 3. Pitch-pine | 7. Elm | 11. Red oak | 14. Tamarac |
| 4. Yellow pine | 8. Spruce | | |

Red-cedar piles are said to have an average life of 27 years with a possible maximum of 50 years, but the timber is rather

weak, and if exposed in a river to flowing ice or driftwood is apt to be injured. Under these circumstances oak is preferable, although its life may be only 13 to 18 years.

130. Methods of driving piles. The following are the principal methods of driving piles:

a. A hammer weighing 2000 to 3000 lbs. or more, sliding in guides, is drawn up by horse-power or a portable engine, and allowed to fall *freely*.

b. The same as above except that the hammer does not fall freely, but drags the rope and revolving drum as it falls and is thus quite materially retarded. The mechanism is a little more simple, but is less effective, and is sometimes made deliberately deceptive by a contractor by retarding the blow, in order to apparently indicate the requisite resistance on the part of the pile.

The above methods have the advantage that the mechanism is cheap and can be transported into a new country with comparative ease, but the work done is somewhat ineffective and costly compared with some of the more elaborate methods given below.

c. *Gunpowder pile-drivers*, which automatically explode a cartridge every time the hammer falls. The explosion not only forces the pile down, but throws up the hammer for the next blow. For a given height of fall the effect is therefore doubled. It has been shown by experience, however, that when it is attempted to use such a pile-driver rapidly the mechanism becomes so heated that the cartridges explode prematurely, and the method has therefore been abandoned.

d. *Steam pile-drivers*, in which the hammer is operated directly by steam. The hammer falls freely a height of about 40 inches and is raised again by steam. The effectiveness is largely due to the rapidity of the blows, which does not allow time between the blows for the ground to settle around the pile and increase the resistance, which does happen when the blows are infrequent. "The hammer-cylinder weighs 5500 lbs., and with 60 to 75 lbs. of steam gives 75 to 80 blows per minute. With 41 blows a large unpainted pile was driven 35 feet into a hard clay bottom in half a minute." Such a driver would cost about \$800.

The above four methods are those usual for dry earth. In very soft wet or sandy soils, where an unlimited supply of water

is available, the *water-jet* is sometimes employed. A pipe is fastened along the side of the pile and extends to the pile-point. If water is forced through the pipe, it loosens the sand around the point and, rising along the sides, decreases the side resistance so that the pile sinks by its own weight, aided perhaps by extra weights loaded on. This loading may be accomplished by connecting the top of the pile and the pile-driver by a block and tackle so that a portion of the weight of the pile-driver is continually thrown on the pile.

Excessive driving frequently fractures the pile below the surface and thereby greatly weakens its bearing power. To prevent excessive "brooming" of the top of the pile, owing to the action of the hammer, the top should be protected by an iron ring fitted to the top of the pile. The "brooming" not only renders the driving ineffective and hence uneconomical, but vitiates the value of any test of the bearing power of the pile by noting the sinking due to a given weight falling a given distance. If the pile is so soft that brooming is unavoidable, the top should be adzed off frequently, and especially should it be done just before the final blows which are to test its bearing-power.

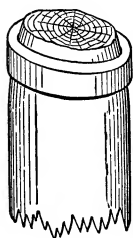


FIG. 67.

In a new country judgment and experience will be required to decide intelligently whether to employ a simple drop-hammer machine, operated by horse-power and easily transported but uneconomical in operation, or a more complicated machine working cheaply and effectively after being transported at greater expense.

131. Pile-driving formulæ. If R = the resistance of a pile, and s the set of the pile during the last blow, w the weight of the pile-hammer, and h the fall during the last blow, then we may state the approximate relation that $Rs = wh$, or $R = \frac{wh}{s}$.

This is the basic principle of all rational formulæ, but the maximum weight which a pile will sustain after it has been driven some time is by no means equal to the resistance of the pile during the last blow. There are also many other modifying elements which have been variously allowed for in the many proposed formulæ. The formulæ range from the extreme of empirical simplicity to very complicated attempts to allow

properly for all modifying causes. As the simplest rule, specifications sometimes require that the piles shall be driven until the pile will not sink more than 5 inches under five consecutive blows of a 2000-lb. hammer falling 25 feet. The "*Engineering News* formula" * gives the safe load as $\frac{2wh}{s+1}$, in which w = weight of hammer, h = fall in *feet*, s = set of pile in *inches* under the last blow. This formula is derived from the above basic formula by calling the safe load $\frac{1}{5}$ of the final resistance, and by adding (arbitrarily) 1 to the final set (s) as a compensation for the extra resistance caused by the settling of earth around the pile between each blow. This formula is used only for ordinary hammer-driving. When the piles are driven by a steam pile-driver the formula becomes safe load = $\frac{2wh}{s+0.1}$. For the "gunpowder pile-driver," since the explosion of the cartridge drives the pile in with the same force with which it throws the hammer upward, the effect is *twice* that of the fall of the hammer, and the formula becomes safe load = $\frac{4wh}{s+0.1}$. In these last two formulæ the constant in the denominator is changed from $s+1$ to $s+0.1$. The constant (1.0 or 0.1) is supposed to allow, as before stated, for the effect of the extra resistance caused by the earth settling around the pile between each blow. The more rapid the blows the less the opportunity to settle and the less the proper value of the constant.

The above formulæ have been given on account of their simplicity and their practical agreement with experience. Many other formulæ have been proposed, the majority of which are more complicated and attempt to take into account the weight of the pile, resistance of the guides, etc. While these elements, as well as many others, have their influence, their effect is so overshadowed by the indeterminable effect of other elements—as, for example, the effect of the settlement of earth around the pile between blows—that it is useless to attempt to employ anything but a purely empirical formula.

Examples. 1. A pile was driven with an ordinary hammer weighing 2500 pounds until the sinking under five consecutive blows was $15\frac{1}{2}$ inches. The fall of the hammer during the last

* *Engineering News*, Nov. 17, 1892.

blows was 24 feet. What was the safe bearing power of the pile?

$$\frac{2wh}{s+1} = \frac{2 \times 2500 \times 24}{(\frac{1}{3} \times 15.5) + 1} = \frac{120000}{4.1} = 29300 \text{ pounds.}$$

2. Piles are being driven into a firm soil with a steam pile-driver until they show a *safe* bearing power of 20 tons. The hammer weighs 5500 pounds and its fall is 40 inches. What should be the sinking under the final blow?

$$40000 = \frac{2wh}{s+0.1} = \frac{2 \times 5500 \times 3.33}{s+0.1},$$

$$s = \frac{36667}{40000} - 0.1 = .81 \text{ inch.}$$

132. Pile-points and pile-shoes. Piles are generally sharpened to a blunt point. If the pile is liable to strike boulders, sunken

logs, or other obstructions which are liable to turn the point, it is necessary to protect the point by some form of shoe. Several forms in cast iron have been used, also a wrought-iron shoe, having four "straps" radiating from the apex, the straps being nailed on to the pile, as shown in Fig. 68 (b). The cast-iron form shown in Fig. 68 (a) has a base cast around a drift-bolt. The recess on the top of the base receives the bottom of the pile and prevents a tendency to split the bottom of the pile or to force the shoe off laterally.

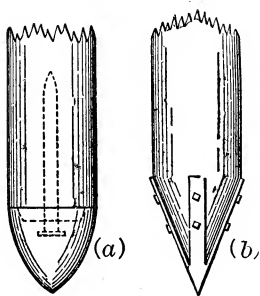


FIG. 68.

133. Details of design. No theoretical calculations of the strength of pile bents need be attempted on account of the extreme complication of the theoretical strains, the uncertainty as to the real strength of the timber used, the variability of that strength with time, and the insignificance of the economy that would be possible even if exact sizes could be computed. The piles are generally required to be not less than 10" or 12" in diameter at the large end. The P. R. R. requires that they shall

be "not less than 14 and 7 inches in diameter at butt and small end respectively, exclusive of bark, which must be removed." The removal of the bark is generally required in good work. Soft *durable* woods, such as are mentioned in § 129, are best for the piles, but the caps are generally made of oak or yellow pine. The caps are generally 14 feet long (for single track) with a cross-section 12"×12" or 12"×14". "Split caps" would consist of two pieces 6"×12". The sway-braces, never used for less heights than 6', are made of 3"×12" timber, and are spiked on with $\frac{3}{8}$ " spikes 8" long. The floor system will be the same as that described later for framed trestles.

134. Cost of pile trestles. The cost, per linear foot, of piling depends on the method of driving, the scarcity of suitable timber, the price of labor, the length of the piles, and the amount of shifting of the pile-driver required. The cost of soft-wood piles varies from 8 to 15 c. per lineal foot, and the cost of oak piles varies from 10 to 30 c. per foot according to the length, the longer piles costing more per foot. The cost of driving will average about \$2.50 per pile, or 7.5 to 10 c. per lineal foot. Since the cost of shifting the pile-driver is quite an item in the total cost, the cost of driving a long pile would be *less* per foot than for a short pile, but on the other hand the cost of the pile is *greater* per foot, which tends to make the total cost per foot constant. Specifications generally say that the piling will be paid for per lineal foot of piling *left in the work*. The wastage of the tops of piles sawed off is always something, and is frequently very large. Sometimes a small amount per foot of piling sawed off is allowed the contractor as compensation for his loss. This reduces the contractor's risk and possibly reduces his bid by an equal or greater amount than the extra amount actually paid him.

FRAMED TRESTLES.

135. Typical design. A typical design for a framed trestle bent is given in Fig. 69. This represents, with slight variations of detail, the plan according to which a large part of the framed trestle bents of the country have been built—i.e., of those less than 20 or 30 feet in height, not requiring multiple story construction.

136. Joints. (a) The mortise-and-tenon joint is illustrated in

Fig. 69 and also in Fig. 66 (a). The tenon should be about

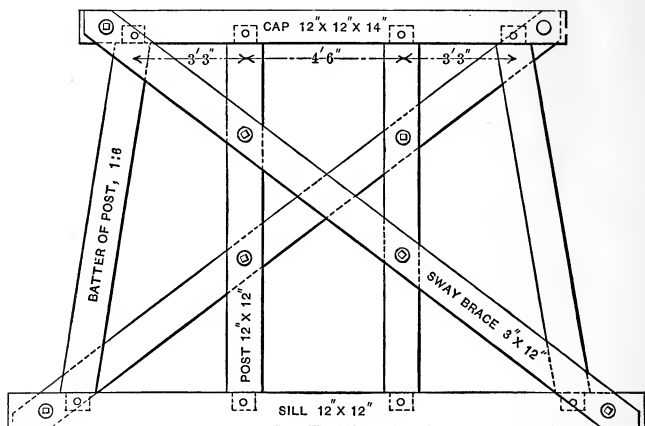


FIG. 69.

3" thick, 8" wide, and $5\frac{1}{2}$ " long. The mortise should be cut a little deeper than the tenon. "Drip-holes" from the mortise to the outside will assist in draining off water that may accumulate in the joint and thus prevent the rapid decay that would otherwise ensue. These joints are very troublesome if a single post decays and requires renewal. It is generally required that the mortise and tenon should be thoroughly daubed

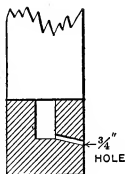


FIG. 70.

with paint before putting them together. This will tend to make the joint water-tight and prevent decay from the accumulation and retention of water in the joint.

(b) **The plaster joint.** This joint is made by bolting and spiking a $3'' \times 12''$ plank on both sides of the joint. The cap and sill should be notched to receive the posts. Repairs are greatly facilitated by the use of these joints. This method has been used by the Delaware and Hudson Canal Co. [R. R.].

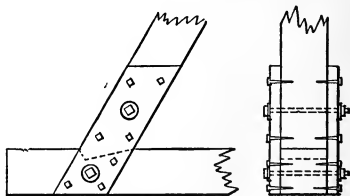


FIG. 71.

(c) **Iron plates.** An iron plate of the form shown in Fig. 72

(b) is bent and used as shown in Fig. 72 (a). Bolts passing through the bolt-holes shown secure the plates to the timbers and make a strong joint which may be readily loosened for repairs. By slight modifications in the design the method may be used for inclined posts and complicated joints.

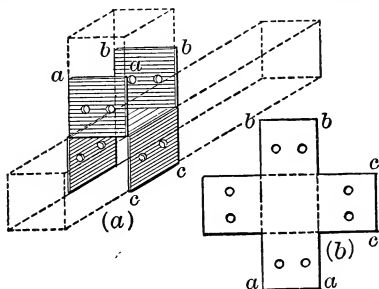


FIG. 72.

(d) **Split caps and sills.**

These are described in § 129. Their advantages apply with even greater force to framed trestles.

(e) **Dowels and drift-bolts.** These joints facilitate cheap and rapid construction, but renewals and repairs are very difficult, it being almost impossible to extract a drift-bolt, which has been driven its full length, without splitting open the pieces containing it. Notwithstanding this objection they are extensively used, especially for temporary work which is not expected to be used long enough to need repairs.

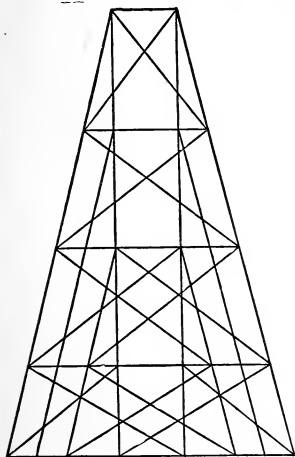


FIG. 73.

— 137. Multiple-story construction.

Single-story framed trestle bents are used for heights up to 18 or 20 feet and exceptionally up to 30 feet. For greater heights some such construction as is illustrated in a skeleton design in Fig. 73 is used. By using split sills between each story and separate vertical and batter posts in each story, any piece may readily be removed and renewed if necessary. The height of these stories varies, in different designs, from 15 to 25 and even 30 feet. In some designs the structure of each story is independent of the stories above and below. This greatly

facilitates both the original construction and subsequent repairs.

In other designs the verticals and batter-posts are made continuous through two consecutive stories. The structure is somewhat stiffer, but is much more difficult to repair.

Since the bents of any trestle are usually of variable height and those heights are not always an even multiple of the uniform height desired for the stories, it becomes necessary to make the

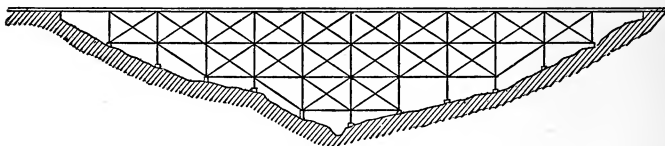


FIG. 74.

upper stories of uniform height and let the odd amount go to the lowest story, as shown in Figs. 73 and 74.

138. Span. The shorter the span the greater the number of trestle bents; the longer the span the greater the required strength of the stringers supporting the floor. Economy demands the adoption of a span that shall make the sum of these require-

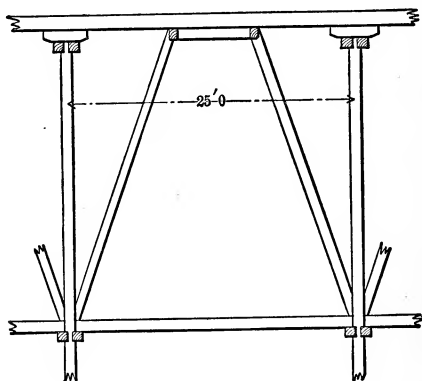


FIG. 75.

ments a minimum. The higher the trestle the greater the cost of each bent, and the greater the span that would be justifiable. Nearly all trestles have bents of variable height, but the advantage of employing uniform standard sizes is so great that many

roads use the same span and sizes of timber not only for the panels of any given trestle, but also for all trestles regardless of height. The spans generally used vary from 10 to 16 feet. The Norfolk and Western R. R. uses a span of 12' 6" for all single-story trestles, and a span of 25' for all multiple-story trestles. The stringers are the same in both cases, but when the span is 25 feet, knee-braces are run from the sill of the first story below to near the middle of each set of stringers. These knee-braces are connected at the top by a "straining-beam" on which the stringers rest, thus supporting the stringer in the center and virtually reducing the span about one-half.

139. Foundations. (a) Piles. Piles are frequently used as a foundation, as in Fig. 76, particularly in soft ground, and also for temporary structures. These foundations are cheap, quickly constructed, and are particularly valuable when it is financially necessary to open the road for traffic as soon as possible and with the least expenditure of money; but there is the disadvantage of inevitable decay within a few years unless the piles are chemically treated, as will be discussed later. Chemical treatment, however, increases the cost so that such a foundation would often cost more than a foundation of stone. A pile should be driven under each post as shown in Fig. 76.

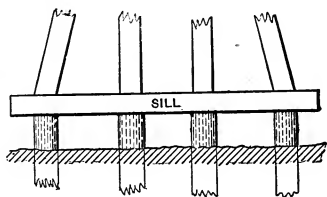


FIG. 76.

(b) Mud-sills. Fig. 77 illustrates the use of mud-sills as built by the Louisville and Nashville R. R. Eight blocks $12'' \times 12'' \times 6'$ are used under each bent. When the ground is very soft, two additional timbers ($12'' \times 12'' \times$ length of bent-sill), as shown by the dotted lines, are placed underneath. The number required evidently depends on the nature of the ground.

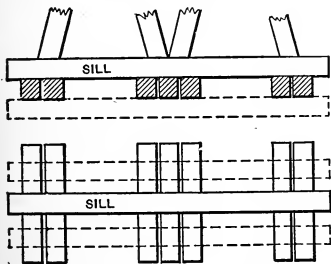


FIG. 77.

(c) Stone foundations. Stone foundations are the best and the most expensive. For very high trestles the Norfolk and

Western R. R. employs foundations as shown in Fig. 78, the walls being 4 feet thick. When the height of the trestle is 72 feet or less (the plans requiring for 72' in height a foundation-wall 39' 6" long) the foundation is made continuous. The sill



FIG. 78.

of the trestle should rest on several short lengths of 3"×12" plank, laid transverse to the sill on top of the wall.

140. Longitudinal bracing. This is required to give the structure longitudinal stiffness and also to reduce the columnar length of the posts. This bracing generally consists of horizontal "waling-strips" and diagonal braces. Sometimes the braces are placed wholly on the outside posts unless the trestle is very high. For single-story trestles the P. R. R. employs the "laced" system, i.e., a line of posts joining the cap of one bent with the sill of the next, and the sill of that bent with the cap of the next. Some plans employ braces forming an X in alternate panels. Connecting these braces in the center more than doubles their columnar strength. Diagonal braces, when bolted to posts, should be fastened to them as near the ends of the posts as possible. The sizes employed vary largely, depending on the clear length and on whether they are expected to act by tension or compression. 3"×12" planks are often used when the design would require tensile strength only, and 8"×8" posts are often used when compression may be expected.

141. Lateral bracing. Several of the more recent designs of trestles employ diagonal lateral bracing between the caps of adjacent bents. It adds greatly to the stiffness of the trestle and better maintains its alignment. 6"×6" posts, forming an X and connected at the center, will answer the purpose.

142. Abutments. When suitable stone for masonry is at hand and a suitable subsoil for a foundation is obtainable without too much excavation, a masonry abutment will be the best. Such an abutment would probably be used when masonry footings for trestle bents were employed (§ 139, c).

Another method is to construct a "crib" of 10"×12" timber,

laid horizontally, drift-bolted together, securely braced and embedded into the ground. Except for temporary construction such a method is generally objectionable on account of rapid decay.

Another method, used most commonly for pile trestles, and for framed trestles having pile foundations (§ 139, *a*), is to use a pile bent at such a place that the natural surface on the *up-hill* side is not far below the

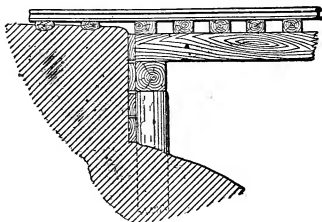


FIG. 79.

cap, and the thrust of the material, filled in to bring the surface to grade, is insignificant. 3"×12" planks are placed behind the piles, cap, and stringers to retain the filled material.

FLOOR SYSTEMS.

143. Stringers. The general practice is to use two, three, and even four stringers under each rail. Sometimes a stringer is placed under each guard-rail. Generally the stringers are made of two panel lengths and laid so that the joints alternate. A few roads use stringers of only one panel length, but this practice is strongly condemned by many engineers. The stringers should be separated to allow a circulation of air around them and prevent the decay which would occur if they were placed close together. This is sometimes done by means of 2" planks, 6' to 8' long, which are placed over each trestle bent. Several bolts, passing through all the stringers forming a group and through the separators, bind them all into one solid construction. Cast-iron "spools" or washers, varying from 4" to $\frac{3}{4}$ " in length (or thickness), are sometimes strung on each bolt so as to separate the stringers. Sometimes washers are used between the separating planks and the stringers, the object of the separating planks then being to bind the stringers, especially abutting stringers, and increase their stiffness.

The most common size for stringers is 8"×16". The Pennsylvania Railroad varies the width, depth, and number of stringers under each rail according to the clear span. It may be noticed that, assuming a uniform load per running foot, both the pressure per square inch at the ends of the stringers (the

caps having a width of 12") and also the stress due to transverse strain are kept *approximately* constant for the variable gross load on these varying spans.

Clear span.	No. of pieces under each rail.	Width.	Depth.
10 feet	2	8 inches	16 inches
12 "	2	10 "	16 "
14 "	3	10 "	16 "

144. Corbels. A corbel (in trestle-work) is a stick of timber (perhaps two placed side by side), about 3' to 6' long, placed underneath and along the stringers and resting on the cap. There are strong prejudices for and against their use, and a corresponding diversity in practice. They are bolted to the stringers and thus stiffen the joint. They certainly reduce the objectionable crushing of the fibers at each end of the stringer, but if the corbel is no wider than the stringers, as is generally the case, the area of pressure between the corbels and the cap is

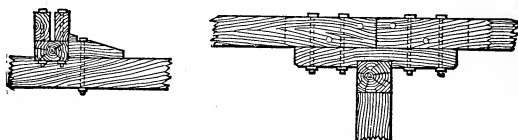


FIG. 80.

no greater and the pressure per square inch on the cap is no less than the pressure on the cap if no corbels were used. If the corbels and cap are made of hard wood, as is recommended by some, the danger of crushing is lessened, but the extra cost and the frequent scarcity of hard wood, and also the extra cost and labor of using corbels, may often neutralize the advantages obtained by their use.

145. Guard-rails. These are frequently made of 5"×8" stuff, notched 1" for each tie. The sizes vary up to 8"×8", and the depth of notch from $\frac{3}{4}$ " to $1\frac{1}{2}$ ". They are generally bolted to every third or fourth tie. It is frequently specified that they shall be made of oak, white pine, or yellow pine. The joints are made over a tie, by halving each piece, as illustrated in Fig. 81. The joints on opposite sides of the trestle should be "stag-

gered." Some roads fasten every tie to the guard-rail, using a bolt, a spike, or a lag-screw.

Guard-rails were originally used with the idea of preventing the wheels of a derailed truck from running off the ends of the ties. But it has been found that an outer guard-rail alone (without an inner guard-rail) becomes an actual element of danger, since it has frequently happened that a derailed wheel has caught on the outer guard-rail, thus causing the truck to slew around

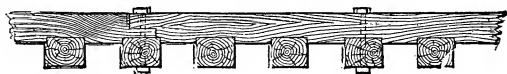


FIG. 81.

and so produce a dangerous accident. The true function of the *outside* guard-rail is thus changed to that of a tie-spacer, which keeps the ties from spreading when a derailment occurs. The inside guard-rail generally consists of an ordinary steel rail spiked about 10 inches inside of the running rail. These inner guard-rails should be bent inward to a point in the center of the track about 50 feet beyond the end of the bridge or trestle. If the inner guard-rails are placed with a clear space of 10 inches inside the running rail, the outer guard-rails should be *at least* 6' 10" apart. They are generally much farther apart than this.

146. Ties on trestles. If a car is derailed on a bridge or trestle, the heavily loaded wheels are apt to force their way between the ties by displacing them unless the ties are closely spaced and fastened. The clear space between ties is generally equal to or less than their width. Occasionally it is a little more than their width. 6"×8" ties, spaced 14" to 16" from center to center, are most frequently used. The length varies from 9' to 12' for single track. They are generally notched $\frac{1}{2}$ " deep on the under side where they rest on the stringers. Oak ties are generally required even when cheaper ties are used on the other sections of the road. Usually every third or fourth tie is bolted to the stringers. When stringers are placed underneath the guard-rails, bolts are run from the top of the guard-rail to the under side of the stringer. The guard-rails thus hold down the whole system of ties, and no direct fastening of the ties to the stringers is needed.

147. Superelevation of the outer rail on curves. The location of curves on trestles should be avoided if possible, especially when the trestle is high. Serious additional strains are intro-

duced especially when the curvature is sharp or the speed high. Since such curves are sometimes practically unavoidable, it is necessary to design the trestle accordingly. If a train is stopped on a curved trestle, the action of the train on the trestle is evidently vertical. If the train is moving with a considerable velocity, the resultant of the weight and the centrifugal action is a force somewhat inclined from the vertical. Both of these conditions may be expected to exist at times. If the *axis* of the system of posts is vertical (as illustrated in methods *a*, *b*, *c*, *d*, and *e*), any lateral force, such as would be produced by a moving train, will tend to rack the trestle bent. If the stringers are set vertically, a centrifugal force likewise tends to tip them sidewise. If the axis of the system of posts (or of the stringers) is inclined so as to coincide with the pressure of the train on the trestle when the train is moving at its normal velocity, there is no tendency to rack the trestle when the train is moving at that velocity, but there will be a tendency to rack the trestle or twist the stringers when the train is stationary. Since a moving train is usually the normal condition of affairs, as well as the condition which produces the maximum stress, an inclined axis is evidently preferable from a theoretical standpoint; but whatever design is adopted, the trestle should evidently be sufficiently cross-braced for either a moving or a stationary load, and any proposed design must be studied as to the effect of *both* of these conditions. Some of the various methods of securing the requisite superelevation may be described as follows:

(a) Framing the outer posts longer than the inner posts, so

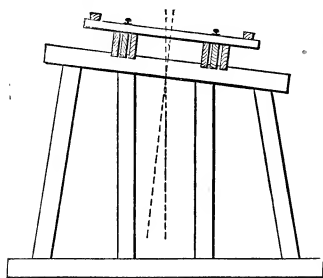


FIG. 82.

that the cap is inclined at the proper angle; axis of posts vertical. (Fig. 82.) The method requires more work in framing the trestle, but simplifies subsequent track-laying and maintenance, unless it should be found that the superelevation adopted is unsuitable, in which case it could be corrected by one of the other methods given below. The stringers tend to twist when the train is stationary.

(b) Notching the cap so that the stringers are at a different

elevation. (Fig. 83.) This weakens the cap and requires that all ties shall be notched to a bevelled surface to fit the stringers, which also weakens the ties. A centrifugal force will tend to twist the stringers and rack the trestle.

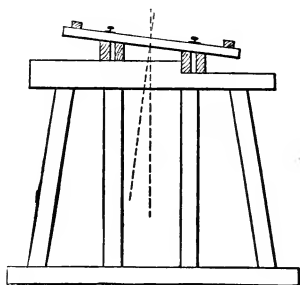


FIG. 83.

(c) **Placing wedges underneath the ties at each stringer.** These wedges are fastened with two bolts. Two or more wedges will be required for each tie. The additional number of pieces required for a long curve will be immense, and the work of inspection and keeping the nuts tight will greatly increase the cost of maintenance.

(d) **Placing a wedge under the outer rail at each tie.** This requires but one extra piece per tie. There is no need of a wedge under the inner tie in order to make the rail normal to the tread. The resulting inward inclination is substantially that produced by some forms of rail-chairs or tie-plates. The spikes (a little longer than usual) are driven through the wedge into the tie. Sometimes "lag-screws" are used instead of spikes. If experience proves that the superelevation is too much or too little, it may be changed by this method with less work than by any other.

(e) **Corbels of different heights.** When corbels are used (see § 144) the required inclination of the floor system may be obtained by varying the depth of the corbels.

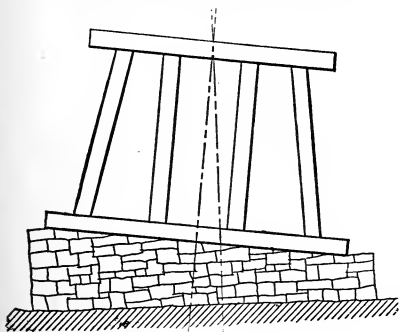


FIG. 84.

(f) **Tipping the whole trestle.** This is done by placing the trestle on an inclined foundation. If very much inclined, the trestle bent must be secured against the possibility of slipping sideways,

for the slope would be considerable with a sharp curve, and the

vibration of a moving train would reduce the coefficient of friction to a comparatively small quantity.

(g) **Framing the outer posts longer.** This case is identical with case (a) except that the axis of the system of posts is inclined, as in case (f), but the sill is horizontal.

The above-described plans will suggest a great variety of methods which are possible and which differ from the above only in minor details.

148. Protection from fire. Trestles are peculiarly subject to fire, from passing locomotives, which may not only destroy the trestle, but perhaps cause a terrible disaster. This danger is sometimes reduced by placing a strip of galvanized iron along the top of each set of stringers and also along the tops of the caps. Still greater protection was given on a long trestle on the Louisville and Nashville R. R. by making a solid flooring of timber, covered with a layer of ballast on which the ties and rails were laid as usual.

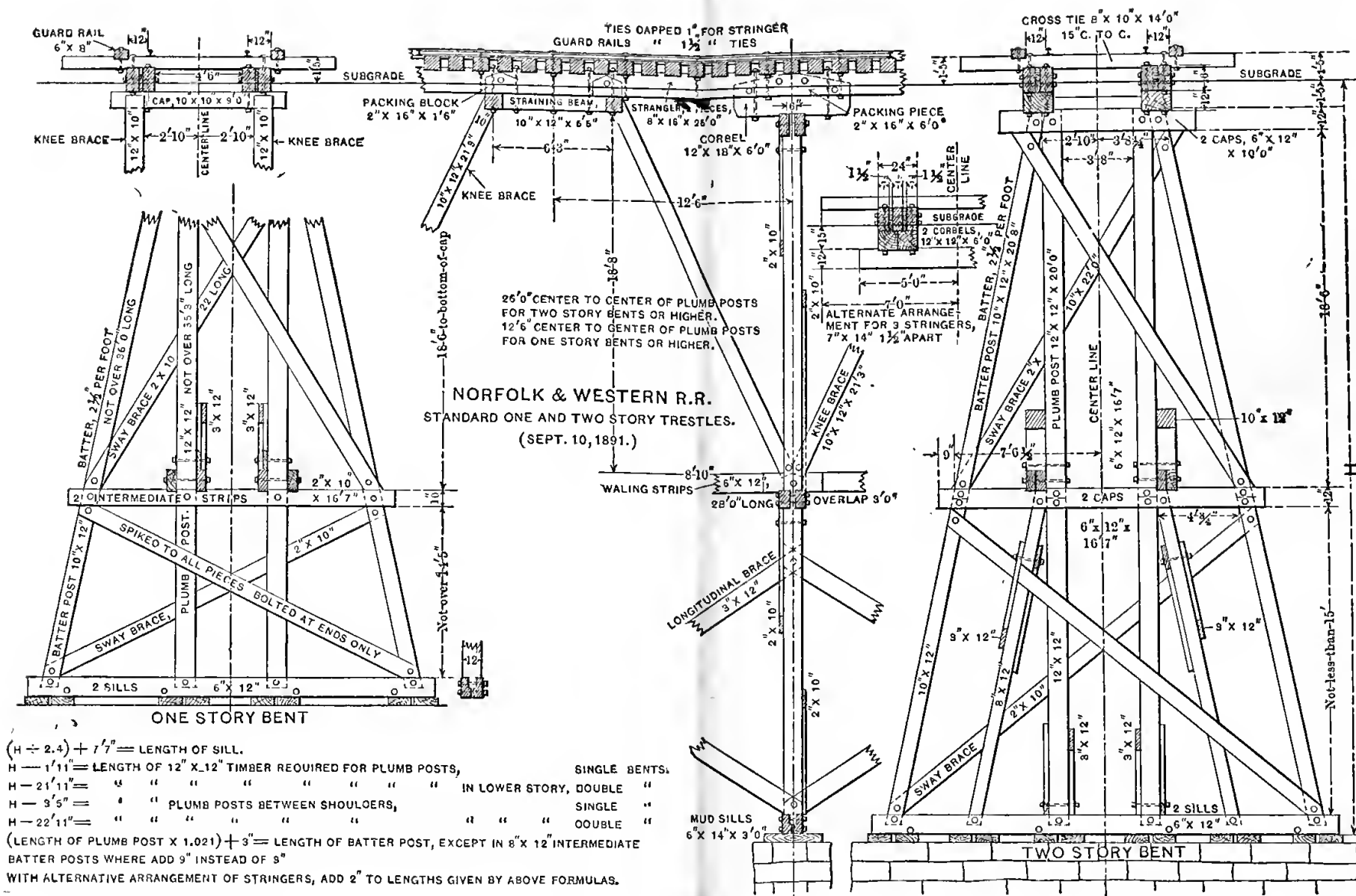
Barrels of water should be provided and kept near all trestles, and on very long trestles barrels of water should be placed every two or three hundred feet along its length. A place for the barrels may be provided by using a few ties which have an extra length of about four feet, thus forming a small platform, which should be surrounded by a railing. The track-walker should be held accountable for the maintenance of a supply of water in these barrels, renewals being frequently necessary on account of evaporation. Such platforms should also be provided as REFUGE-BAYS for track-walkers and trackmen working on the trestle. On very long trestles such a platform is sometimes provided with sufficient capacity for a hand-car.

149. Timber. Any strong durable timber may be used when the choice is limited, but oak, pine, or cypress are preferred when obtainable. When all of these are readily obtainable, the various parts of the trestle will be constructed of different kinds of wood—the stringers of long-leaf pine, the posts and braces of pine or red cypress, and the caps, sills, and corbels (if used) of white oak. The use of oak (or a similar hard wood) for caps, sills, and corbels is desirable because of its greater strength in resisting crushing across the grain, which is the critical test for these parts. There is no physiological basis to the objection, sometimes made, that different species of timber, in contact with each other, will rot quicker than if only one

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(To face page 178.)

kind of timber is used. When a very extensive trestle is to be built at a place where suitable growing timber is at hand but there is no convenient sawmill, it will pay to transport a portable sawmill and engine and cut up the timber as desired.

150. Cost of framed timber trestles. The cost varies widely on account of the great variation in the cost of timber. When a railroad is first penetrating a new and undeveloped region, the cost of timber is frequently small, and when it is obtainable from the company's right-of-way the only expense is felling and sawing. The work per M, B. M., is small, considering that a single stick $12'' \times 12'' \times 25'$ contains 300 feet, B. M., and that sometimes a few hours' work, worth less than \$1, will finish all the work required on it. Smaller pieces will of course require more work per foot, B. M. Long-leaf pine can be purchased from the mills at from \$8 to \$12 per M feet, B. M., according to the dimensions. To this must be added the freight and labor of erection. The cartage from the nearest railroad to the trestle may often be a considerable item. Wrought iron will cost about 3 c. per pound and cast iron 2 c., although the prices are often lower than these. The amount of iron used depends on the detailed design, but, as an average, will amount to \$1.50 to \$2 per 1000 feet, B. M., of timber. A large part of the trestling of the country has been built at a contract price of about \$30 per 1000 feet, B. M., erected. While the cost will frequently rise to \$40 and even \$50 when timber is scarce, it will drop to \$13 (cost quoted) when timber is cheap.

DESIGN OF WOODEN TRESTLES.

151. Common practice. A great deal of trestling has been constructed without any rational design except that custom and experience have shown that certain sizes and designs are *probably* safe. This method has resulted occasionally in failures but more frequently in a very large waste of timber. Many railroads employ a uniform size for all posts, caps, and sills, and a uniform size for stringers, all regardless of the height or span of the trestle. For repair work there are practical reasons favoring this. "To attempt to run a large lot of sizes would be more wasteful in the end than to maintain a few stock sizes only. Lumber can be bought more cheaply by giving a general order for 'the run of the mill for the season,' or 'a cargo lot,' specify

ing approximate percentages of standard stringer size, of 12×12-inch stuff, 10×10-inch stuff, etc., and a liberal proportion of 3- or 4-inch plank, all lengths thrown in. The 12×12-inch stuff, etc., is ordered all lengths, from a certain specified length up. In case of a wreck, washout, burn-out, or sudden call for a trestle to be completed in a stated time, it is much more economical and practical to order a certain number of carloads of 'trestle stuff' to the ground and there to select piece after piece as fast as needed, dependent only upon the length of stick required. When there is time to make the necessary surveys of the ground and calculations of strength, and to wait for a special bill of timber to be cut and delivered, the use of different sizes for posts in a structure would be warranted to a certain extent." * For new construction, when there is generally sufficient time to design and order the proper sizes, such wastefulness is less excusable, and under any conditions it is both safer and more economical to prepare *standard designs* which can be made applicable to varying conditions and which will at the same time utilize as much of the strength of the timber as can be depended on. In the following sections will be given the elements of the preparation of such standard designs, which will utilize uniform sizes with as little waste of timber as possible. It is *not* to be understood that special designs should be made for each individual trestle.

152. Required elements of strength. The *stringers* of trestles are subject to transverse strains, to crushing across the grain at the ends, and to shearing along the neutral axis. The strength of the timber must therefore be computed for all these kinds of stress. *Caps* and *sills* will fail, if at all, by crushing across the grain; although subject to other forms of stress, these could hardly cause failure in the sizes usually employed. There is an apparent exception to this: if piles are improperly driven and an uneven settlement subsequently occurs, it may have the effect of transferring practically all of the weight to two or three piles, while the *cap* is subjected to a severe transverse strain which may cause its failure. Since such action is caused generally by avoidable errors of construction it may be considered as abnormal, and since such a failure will generally occur by a *gradual* settlement, all danger may be avoided by reasonable

* From "Economical Designing of Timber Trestle Bridges."

care in inspection. *Posts* must be tested for their columnar strength. These parts form the bulk of the trestle and are the parts which can be definitely designed from known stresses. The stresses in the bracing are more indefinite, depending on indeterminate forces, since the inclined posts take up an unknown proportion of the lateral stresses, and the design of the bracing may be left to what experience has shown to be safe, without involving any large waste of timber.

153. Strength of timber. Until recently tests of the strength of timber have generally been made by testing small, selected, well-seasoned sticks of "clear stuff," free from knots or imperfections. Such tests would give results so much higher than the vaguely known strength of large unseasoned "commercial" timber that very large factors of safety were recommended—factors so large as to detract from any confidence in the whole theoretical design. Recently the U. S. Government has been making a thoroughly scientific test of the strength of full-size timber under various conditions as to seasoning, etc. The work has been so extensive and thorough as to render possible the economical designing of timber structures.

One important result of the investigation is the determination of the great influence of the moisture in the timber and the law of its effect on the strength. It has been also shown that timber soaked with water has substantially the same strength as green timber, even though the timber had once been thoroughly seasoned. Since trestles are exposed to the weather they should be designed on the basis of using green timber. It has been shown that the strength of green timber is very regularly about 55 to 60% of the strength of timber in which the moisture is 12% of the dry weight, 12% being the proportion of moisture usually found in timber that is protected from the weather but not heated, as, *e.g.*, the timber in a barn. Since the moduli of rupture have all been reduced to this standard of moisture (12%), if we take *one-eighth* of the rupture values, it still allows a factor of safety of about five, even on green timber. On page 182 there are quoted the values taken from the U. S. Government reports on the strength of timber, the tests probably being the most thorough and reliable that were ever made.

On page 183 are given the "working unit stresses for structural timber, expressed in pounds per square inch," as recommended by

the committee on "Wooden Bridges and Trestles," of the American Railway Engineering Association. The report was presented at their tenth annual convention, held in Chicago, in March, 1909.

Moduli of rupture for various timbers. [12% moisture.]

(Condensed from U. S. Forestry Circular, No. 15.)

No.	Species.	Weight per cubic foot.	Cross-bending.		Crushing end-wise.	Crushing across the grain.	Shearing along the grain.
			Ultimate Strength.	Modulus of Elasticity.			
1	Long-leaf pine.....	38	12 600	2 070 000	8000	1180	700
2	Cuban ".....	39	13 600	2 370 000	8700	1220	700
3	Short-leaf ".....	32	10 100	1 680 000	6500	960	700
4	Loblolly ".....	33	11 300	2 050 000	7400	1150	700
5	White ".....	24	7 900	1 390 000	5400	700	400
6	Red ".....	31	9 100	1 620 000	6700	1000	500
7	Spruce ".....	39	10 000	1 640 000	7300	1200	800
8	Bald cypress.....	29	7 900	1 290 000	6000	800	500
9	White cedar.....	23	6 300	910 000	5200	700	400
10	Douglas spruce....	32	7 900	1 680 000	5700	800	500
11	White oak.....	50	13 100	2 090 000	8500	2200	1000
12	Overcup ".....	46	11 300	1 620 000	7300	1900	1000
13	Post ".....	50	12 300	2 030 000	7100	3000	1100
14	Cow ".....	46	11 500	1 610 000	7400	1900	900
15	Red ".....	45	11 400	1 970 000	7200	2300	1100
16	Texan ".....	46	13 100	1 860 000	8100	2000	900
19	Willow ".....	45	10 400	1 750 000	7200	1600	900
20	Spanish ".....	46	12 000	1 930 000	7700	1800	900
21	Shagbark hickory..	51	16 000	2 390 000	9500	2700	1100
27	Pignut ".....	56	18 700	2 730 000	10900	3200	1200
28	White elm.....	34	10 300	1 540 000	6500	1200	800
29	Cedar ".....	46	13 500	1 700 000	8000	2100	1300
30	White ash.....	39	10 800	1 640 000	7200	1900	1100

154. Loading. As shown in § 138, the span of trestles is always small, is generally 14 feet, and is never greater than 18 feet except when supported by knee-braces. The greatest load that will ever come on any one span will be the concentrated loading of the drivers of a consolidation locomotive. With spans of 14 feet or less it is impossible for even the four pairs of drivers to be on the same span at once. The weight of the rails, ties, and guard-rails should be added to obtain the total load on the stringers, and the weight of these, plus the weight of the stringers, should be added to obtain the pressure on the caps or corbels.

WORKING UNIT STRESSES FOR STRUCTURAL TIMBER EXPRESSED IN LBS. PER SQ. IN. RECOMMENDED BY THE COMMITTEE ON WOODEN BRIDGES AND TRESTLES. AMERICAN RAILWAY ENGINEERING ASSOCIATION, 1909.

Kind of Timber.	Bending.			Shearing.			Compression.				Ratio of length of stringer to depth.			
	Extreme fiber stress.	Modulus of elasticity.	Parallel to grain.	Longitudinal shear in beams.		Perpendicular to grain.	Parallel to grain.	For 15 diams. under safe stress.	Formulas for working stress in long column over 15 diams.					
				Aver. ultimat. stress.	W'rk-ing stress.					Elastic limit.		W'rk-ing stress.		
													Aver. ultimat. stress.	W'rk-ing stress.
6100	1200	1,510,000	690	170	110	270	310	3600	1200	$1200\left(1-\frac{L}{60D}\right)$	10			
Douglas fir.	6500	1300	1,610,000	720	180	120	300	260	3800	1300	$1300\left(1-\frac{L}{60D}\right)$	10		
Long-leaf pine...	5600	1100	1,480,000	710	170	130	330	170	3400	1100	$1100\left(1-\frac{L}{60D}\right)$	10		
Short-leaf pine..	4400	900	1,130,000	400	100	70	180	150	3000	1000	$1000\left(1-\frac{L}{60D}\right)$	10		
White pine.	4800	1000	1,310,000	600	150	70	170	180	3200	1100	$1100\left(1-\frac{L}{60D}\right)$	10		
Spruce.....	4200	800	1,190,000	590	130	100	250		2600*	800	$800\left(1-\frac{L}{60D}\right)$			
Norway pine....	4600	900	1,220,000	670	170	100	260		3200*	1000	$1000\left(1-\frac{L}{60D}\right)$			
Tamarack.....	5800	1100	1,480,000	630	160	100	270*	220	3500	1200	$1200\left(1-\frac{L}{60D}\right)$			
Western hemlock	5000	900	800,000	300	80			440	3300	900	$900\left(1-\frac{L}{60D}\right)$			
Redwood.....	4800	900	1,150,000	500	120			400	3900	1100	$1100\left(1-\frac{L}{60D}\right)$			
Bald cypress....	4200	800	860,000					340	2800	900	$900\left(1-\frac{L}{60D}\right)$			
Red cedar.....	5700	1100	1,150,000	840	210	110	270	230	3500	1300	$1300\left(1-\frac{L}{60D}\right)$	12		
White oak.....								450						

Note.—These unit stresses are for a green condition of timber and are to be used without increasing the live load stresses for impact.
 * Partially air-dry.
 These working stresses are for railroad bridges and trestles. For highway bridges and trestles increase the figures by 25 per cent. For buildings, etc., when protected from weather and free from impact, increase them 50 per cent. To compute deflection, under long-continued loading, use 50 per cent of modulus of elasticity.
 L = length in inches.
 D = least side in ins.

This dead load is almost insignificant compared with the live load and may be included with it. The weight of rails, ties, etc., may be estimated at 240 pounds per foot. To obtain the weight on the caps the weight of the stringers must be added, which depends on the design and on the weight per cubic foot of the wood employed. But as the weight of the stringers is comparatively small, a considerable percentage of variation in weight will have but an insignificant effect on the result. Disregarding all refinements as to actual dimensions, the ordinary maximum loading for standard-gauge railroads may be taken as that due to four driving-axles, spaced 5' 0" apart and giving a pressure of 40000 pounds per axle. This should be increased to 54000 pounds per axle (same spacing) for the heaviest traffic. On the basis of 40000 pounds per axle or 20000 pounds per wheel the following results have been computed: This loading is assumed to allow for impact.

STRESSES ON VARIOUS SPANS DUE TO MOVING LOADS OF 20000 POUNDS, SPACED 5' 0" APART, WITH 120 POUNDS PER FOOT OF LIVE LOAD.

Span in feet.	Max. moment, ft. lbs.	Max. shear.	Max. load on one cap under one rail.
10	51 500	30 600	41 200
12	82 160	35 720	49 440
14	112 940	39 410	57 680
16	123 840	43 460	65 920
18	164 860	47 747	75 160

Although the dead load does not vary in proportion to the live load, yet, considering the very small influence of the dead load, there will be no appreciable error in assuming the corresponding values, for a load of 54000 lbs. per axle, to be $\frac{54}{40}$ of those given in the above tabulation.

155. Factors of safety. The most valuable result of the government tests is the knowledge that under given moisture conditions the strength of various species of sound timber is not the variable uncertain quantity it was once supposed to be, but that its strength can be relied on to a comparatively close percentage. This confidence in values permits the employment of lower factors of safety than have heretofore been permissible. Stresses, which when excessive would result in immediate destruction, such as cross-breaking and columnar stresses, should be allowed a higher factor of safety—say 6 or 8 for green timber. Other stresses, such as crushing across the grain and shearing along the

neutral axis, which will be apparent to inspection before it is dangerous, may be allowed lower factors—say 3 to 5.

156. Design of stringers. The strength of rectangular beams of equal width varies as the square of the depth; therefore deep beams are the strongest. On the other hand, when any cross-sectional dimension of timber much exceeds 12" the cost is much higher per M, B. M., and it is correspondingly difficult to obtain thoroughly sound sticks, free from wind-shakes, etc. Wind-shakes especially affect the shearing strength. Also, if the required transverse strength is obtained by using high narrow stringers, the area of pressure between the stringers and the cap may become so small as to induce crushing across the grain. This is a very common defect in trestle design. As already indicated in § 138, the span should vary roughly with the average height of the trestle, the longer spans being employed when the trestle bents are very high, although it is usual to employ the same span throughout any one trestle.

To illustrate, if we select a span of 14 feet, the load on one cap will be 57680 lbs. If the stringers and cap are made of long-leaf yellow pine, the allowable value, according to the table on page 183 for "compression across the grain" is 260 pounds per square inch; this will require 222 square inches of surface. If the cap is 12" wide, this will require a width of 18.5 inches, or say 2 stringers under each rail, each 9 inches wide. For rectangular beams

$$\text{Moment} = \frac{1}{6}R'bh^2.$$

Using for R' the safe value 1300 lbs. per square inch, we have

$$112940 \times 12 = \frac{1}{6} \times 1300 \times 18 \times h^2,$$

from which $h = 18''.7$. If desired, the width may be increased to 10" and the depth correspondingly reduced, which will give similarly $h = 17''.7$ or say 18". This shows that two beams, 10" $\times$ 18", under each rail will stand the transverse bending and have more than enough area for crushing.

The shear per square inch will equal

$$\frac{3 \text{ total shear}}{2 \text{ cross-section}} = \frac{3}{2} \frac{39410}{2 \times 10 \times 18} = 164 \text{ lbs. per sq. inch.}$$

This is higher than the recommended working value. The combination suggested on page 174, viz., 3 beams 10" $\times$ 16" for 14 feet span, gives a far safer value. Considering that wooden beams,

tested to destruction, usually fail by shearing, the three-beam combination is safer.

The deflection should be computed to see if it exceeds the somewhat arbitrary standard of $\frac{1}{200}$ of the span. The deflection for *uniform loading* is

$$\Delta = \frac{5Wl^3}{32bh^3E},$$

in which l = length in inches;

W = total load, assumed as uniform;

E = modulus of elasticity, given as 850,000 lbs.

per sq. in. for long-leaf pine, according to the table on p. 183. Then

$$\Delta = \frac{5 \times 57680 \times 168^3}{32 \times 20 \times 18^3 \times 850000} = 0''.431$$

$$\frac{1}{200} \times 168'' = 0''.84,$$

so that the calculated deflection is well within the limit. Of course the loading is not strictly uniform, but even with a liberal allowance the deflection is still safe.

For the heaviest practice (54000 lbs. per axle) these stringer dimensions must be correspondingly increased.

157. Design of posts. Four posts are generally used for single-track work. The inner posts are usually braced by the cross-braces, so that their columnar strength is largely increased; but as they are apt to get more than their share of work, the advantage is compensated and they should be treated as unsupported columns for the total distance between cap and sill in simple bents, or for the height of stories in multiple-story construction. The caps and sills are assumed to have a width of 12''. It facilitates the application of bracing to have the columns of the same width and vary the other dimension as required.

Unfortunately the experimental work of the U. S. Government on timber testing has not yet progressed far enough to establish unquestionably a general relation between the strength of long columns and the crushing strength of short blocks. The

post, computed as a 7"×11' post, would have a *safe* columnar strength of 706 lbs. per square inch. With an area of 77 square inches, this gives a working load of 54362 lbs. for *each post*, or 217448 lbs. for the four posts. Considering that 74200 lbs. is the maximum load on one cap (14 feet span), the great excess of strength is apparent.

Utilizing the formula given on page 183 for long-leaf pine, we have safe stress = $1300 \left(1 - \frac{L}{60D}\right)$, in which $L = 20 \text{ ft.} = 240 \text{ in.}$, and $D = 12 \text{ ins.}$ Then safe stress = $1300(1 - \frac{1}{3}) = 867 \text{ lbs. per square inch.}$ This is practically a check on the above calculations.

158. Design of caps and sills. The stresses in caps and sills are very indefinite, except as to crushing across the grain. As the stringers are placed almost directly over the inner posts, and as the sills are supported just under the posts, the transverse stresses are almost insignificant. In the above case four posts have an area of $4 \times 12'' \times 8'' = 384 \text{ sq. in.}$ The total load, 74200 lbs., will then give a pressure of 193 pounds per square inch, which is within the allowable limit. This one feature might require the use of 8"×12" posts rather than 6"×12" posts, for the smaller posts, although probably strong enough as posts, would produce an objectionably high pressure.

159. Bracing. Although some idea of the stresses in the bracing could be found from certain assumptions as to wind-pressure, etc., yet it would probably not be found wise to decrease, for the sake of economy, the dimensions which practice has shown to be sufficient for the work. The economy that would be possible would be too insignificant to justify any risk. Therefore the usual dimensions, given in §§ 139 and 140, should be employed.

CHAPTER V.

TUNNELS.

SURVEYING.

160. Surface surveys. As tunnels are always dug from each end and frequently from one or more intermediate shafts, it is necessary that an accurate surface survey should be made between the two ends. As the natural surface in a locality where a tunnel is necessary is almost invariably very steep and rough, it requires the employment of unusually refined methods of work to avoid inaccuracies. It is usual to run a line on the surface that will be at every point vertically over the center line of the tunnel. Tunnels are generally made straight unless curves are absolutely necessary, as curves add greatly to the cost. Fig. 85 represents roughly a longitudinal section of the

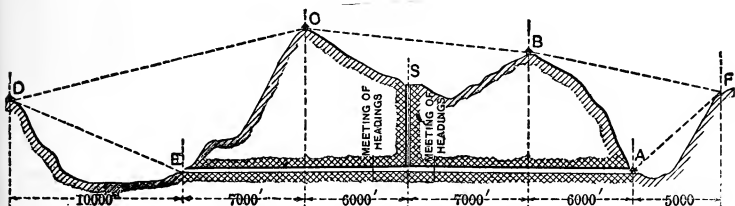


FIG. 85.—SKETCH OF SECTION OF THE HOOSAC TUNNEL.

Hoosac Tunnel. Permanent stations were located at *A*, *B*, *C*, *D*, *E*, and *F*, and stone houses were built at *A*, *B*, *C*, and *D*. These were located with ordinary field transits at first, and then all the points were placed as nearly as possible in one vertical plane by repeated trials and minute corrections, using a very large specially constructed transit. The stations *D* and *F* were necessary because *E* and *A* were invisible from *C* and *B*. The alignment at *A* and *E* having been determined with great accuracy, the true alignment was easily carried into the tunnel.

The relative elevations of *A* and *E* were determined with great accuracy. Steep slopes render necessary many settings of the level per unit of horizontal distance and require that the work be unusually accurate to obtain even fair accuracy per unit of distance. The levels are usually re-run many times until the probable error is a very small quantity.

The exact horizontal distance between the two ends of the tunnel must also be known, especially if the tunnel is on a grade. The usual steep slopes and rough topography likewise render accurate horizontal measurements very difficult. Frequently when the slope is steep the measurement is best obtained by measuring along the slope and allowing for grade. This may be very accurately done by employing two tripods (level or transit tripods serve the purpose very well), setting them up slightly less than one tape-length apart and measuring between horizontal needles set in wooden blocks inserted in the top of each tripod. The elevation of each needle is also observed. The true horizontal distance between two successive positions of the needles then equals the square root of the difference of the squares of the inclined distance and the difference of elevation. Such measurements will probably be more accurate than those made by attempting to hold the tape horizontal and plumbing down with plumb-bobs, because (1) it is practically difficult to hold both ends of the tape truly horizontal; (2) on steep slopes it is impossible to hold the down-hill end of a 100-foot tape (or even a 25-foot length) on a level with the other end, and the great increase in the number of applications of the unit of measurement very greatly increases the probable error of the whole measurement; (3) the vibrations of a plumb-bob introduce a large probability of error in transferring the measurement from the elevated end of the tape to the ground, and the increased number of such applications of the unit of measurement still further increases the probable error.

161. Surveying down a shaft. If a shaft is sunk, as at *S*, Fig. 85, and it is desired to dig out the tunnel in both directions from the foot of the shaft so as to meet the headings from the outside, it is necessary to know, when at the bottom of the shaft, the elevation, alignment, and horizontal distance from each end of the tunnel.

The *elevation* is generally carried down a shaft by means of a steel tape. This method involves the least number of appli-

cations of the unit of measurement and greatly increases the accuracy of the final result.

The *horizontal distance from each end* may be easily transferred down the shaft by means of a plumb-bob, using some of the precautions described in the next paragraph.

To transfer the *alignment* from the surface to the bottom of a shaft requires the highest skill because the shaft is always small, and to produce a line perhaps several thousand feet long in a direction given by two points 6 or 8 feet apart requires that the two points must be determined with extreme accuracy. The eminently successful method adopted in the Hoosac Tunnel will be briefly described: Two beams were securely fastened across the top of the shaft (1030 feet deep), the beams being placed transversely to the direction of the tunnel and as far apart as possible and yet allow plumb-lines, hung from the intersection of each beam with the tunnel center line, to swing freely at the bottom of the shaft. These intersections of the beams with the center line were determined by averaging the results of a large number of careful observations for alignment. Two fine parallel wires, spaced about $\frac{1}{8}$ " apart, were then stretched between the beams so that the center line of the tunnel bisected at all points the space between the wires. Plumb-bobs, weighing 15 pounds, were suspended by fine wires beside each cross-beam, the wires passing between the two parallel alignment wires and bisecting the space. The plumb-bobs were allowed to swing in pails of water at the bottom. Drafts of air up the shaft required the construction of boxes surrounding the wires. Even these precautions did not suffice to absolutely prevent vibration of the wire at the bottom through a very small arc. The mean point of these vibrations in each case was then located on a rigid cross-beam suitably placed at the bottom of the shaft and at about the level of the roof of the tunnel. Short plumb-lines were then suspended from these points whenever desired; a transit was set (by trial) so that its line of collimation passed through both plumb-lines and the line at the bottom could thus be prolonged.

Some recent experience in the "Tamarack" shaft, 4250 feet deep, shows that the accuracy of the results may be affected by air-currents to an unsuspected extent. Two 50-lb. cast-iron plumb-bobs were suspended with No. 24 piano-wire in this shaft. The carefully measured distances between the wires

at top and bottom were 16.32 and 16.43 feet respectively. After considerable experimenting to determine the cause of the variation, it was finally concluded that air-currents were alone responsible. The variation of the bobs from a true vertical plane passing through the wires at the top was of course an unknown quantity, but since the variation in *one* direction amounted to 0.11 foot, the accuracy in other directions was very questionable. This shows that a careful comparative measurement between the wires at top and bottom should always be made as a test of their parallelism.

162. Underground surveys. Survey marks are frequently placed on the timbering, but they are apt to prove unreliable on account of the shifting of the timbering due to settlement of the surrounding material. They should never be placed at the bottom of the tunnel on account of the danger of being disturbed or covered up. Frequently holes are drilled in the roof and filled with wooden plugs in which a hook is screwed exactly on line. Although this is probably the safest method, even these plugs are not always undisturbed, as the material, unless very hard, will often settle slightly as the excavation proceeds. When a tunnel is perfectly straight and not too long, alignment-points may be given as frequently as desired from

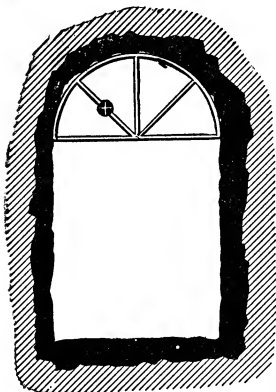


FIG. 86.

permanent stations located *outside* the tunnel where they are not liable to disturbance. This has been accomplished by running the alignment through the upper part of the cross-section, at one side of the center, where it is out of the way of the piles of masonry material, débris, etc., which are so apt to choke up the lower part of the cross-section. The position of this line relative to the cross-section being fixed, the alignment of any required point of the cross-section is readily found by means of a light

frame or template with a fixed target located where this line would intersect the frame when properly placed. A level-bubble on the frame will assist in setting the frame in its proper position.

In all tunnel surveying the cross-wires must be illuminated by a lantern, and the object sighted at must also be illuminated. A powerful dark-lantern with the opening covered with *ground glass* has been found useful. This may be used to illuminate a plumb-bob string or a very fine rod, or to place behind a brass plate having a narrow slit in it, the axis of the slit and plate being coincident with the plumb-bob string by which it is hung.

On account of the interference to the surveying caused by the work of construction and also by the smoke and dust in the air resulting from the blasting, it is generally necessary to make the surveys at times when construction is temporarily suspended.

163. Accuracy of tunnel surveying. Apart from the very natural desire to do surveying which shall check well, there is an important financial side to accurate tunnel surveying. If the survey lines do not meet as desired when the headings come together, it may be found necessary, if the error is of appreciable size, to introduce a slight curve, perhaps even a reversed curve, into the alignment, and it is even conceivable that the tunnel section would need to be enlarged somewhat to allow for these curves. The cost of these changes and the perpetual annoyance due to an enforced and undesirable alteration of the original design will justify a considerable increase in the expenses of the survey. Considering that the cost of surveys is usually but a small fraction of the total cost of the work, an increase of 10 or even 20% in the cost of the surveys will mean an insignificant addition to the total cost and frequent^{ly}, if not generally, it will result in a saving of many times the increased cost. The accuracy actually attained in two noted American tunnels is given as follows: The Musconetcong tunnel is about 5000 feet long, bored through a mountain 400 feet high. The error of alignment at the meeting of the headings was 0'.04, error of levels 0'.015, error of distance 0'.52. The Hoosac tunnel is over 25000 feet long. The heading from the east end met the heading from the central shaft at a point 11274 feet from the east end and 1563 feet from the shaft. The error in alignment was $\frac{5}{16}$ of an inch, that of levels "a few hundredths," error of distance "trifling." The alignment, corrected at the shaft, was carried on through and met the heading from the west end at a point 10138 feet from the west end and 2056 feet from

the shaft. Here the error of alignment was $\frac{9}{16}$ " and that of levels 0.134 ft.

DESIGN.

164. Cross-sections. Nearly all tunnels have cross-sections peculiar to themselves—all varying at least in the details. The *general* form of a great many tunnels is that of a rectangle surmounted by a semi-circle or semi-ellipse. In very soft material an inverted arch is necessary along the bottom. In such cases the sides will generally be arched instead of vertical. The sides are frequently battered. With very long tunnels, several forms of cross-section will often be used in the same tunnel, owing to differences in the material encountered. In solid rock, which will not disintegrate upon exposure, no lining is required, and the cross-section will be the irregular section left by the blasting, the only requirement being that no rock shall be left within the required cross-sectional figure. Farther on, in the same tunnel, when passing through some very soft treacherous material, it may be necessary to put in a full arch lining—top, sides, and bottom—which will be nearly circular in cross-section. For an illustration of this see Figs. 87 and 88.

The width of tunnels varies as greatly as the designs. Single-track tunnels generally have a width of 15 to 16 feet. Occasionally they have been built 14 feet wide, and even less, and also up to 18 feet, especially when on curves. 24 to 26 feet is the most common width for double track. Many double-track tunnels are only 22 feet wide, and some are 28 feet wide. The heights are generally 19 feet for single track and 20 to 22 feet for double track. The variations from these figures are considerable. The lower limits depend on the cross-section of the rolling stock, with an indefinite allowance for clearance and ventilation. Cross-sections which coincide too closely with what is absolutely required for clearance are objectionable, because any slight settlement of the lining which would otherwise be harmless would then become troublesome and even dangerous. Figs. 87, 88, and 89 * show some typical cross-sections.

165. Grade. A grade of at least 0.2% is needed for drainage. If the tunnel is at the summit of two grades, the tunnel grade should be practically level, with an allowance for drainage, the

* Drinker's "Tunneling."

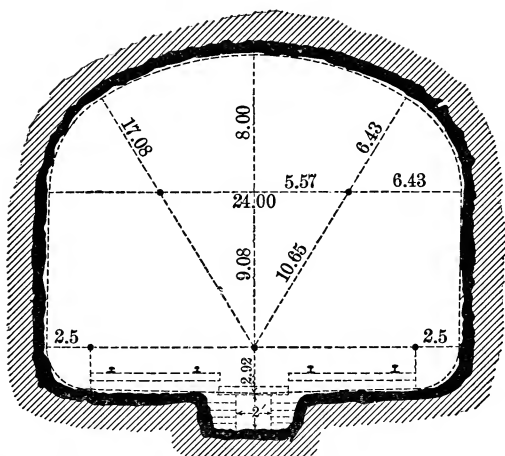


FIG. 87.—HOOSAC TUNNEL. SECTION THROUGH SOLID ROCK.

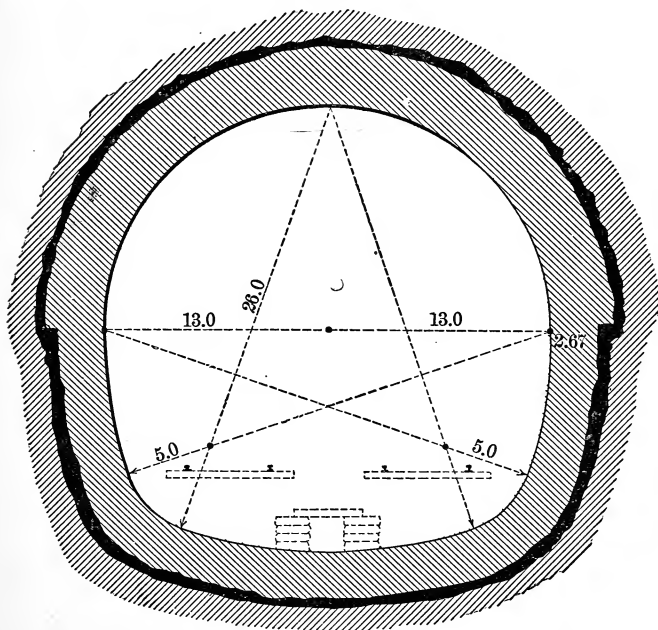


FIG. 88.—HOOSAC TUNNEL. SECTION THROUGH SOFT GROUND.

actual summit being perhaps in the center so as to drain both ways. When the tunnel forms part of a long ascending grade, it is advisable to reduce the grade through the tunnel unless the tunnel is very short. The additional atmospheric resistance and the decreased adhesion of the driver wheels on the damp rails in a tunnel will cause an engine to work very hard and still more rapidly vitiate the atmosphere until the accumulation of poisonous gases becomes a source of actual danger to the engineer and

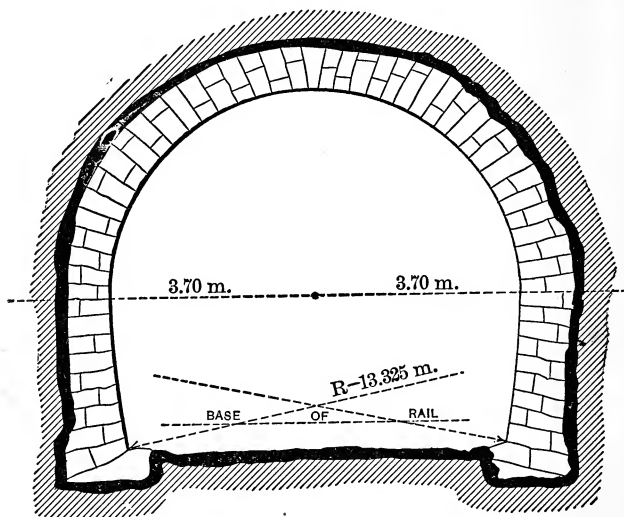


FIG. 89.—ST. CLOUD TUNNEL.

fireman of the locomotive and of extreme discomfort to the passengers. If the nominal ruling grade of the road were maintained through a tunnel, the maximum resistance would be found in the tunnel. This would probably cause trains to stall there, which would be objectionable and perhaps dangerous.

166. Lining. It is a characteristic of many kinds of rock and of all earthy material that, although they may be self-sustaining when first exposed to the atmosphere, they rapidly disintegrate and require that the top and perhaps the sides and even the bottom shall be lined to prevent caving in. In this country, when timber is cheap, it is occasionally framed as an arch and used as the *permanent* lining, but masonry is always to be preferred. Frequently the cross-section is made extra

large so that a masonry lining may subsequently be placed inside the wooden lining and thus postpone a large expense until the road is better able to pay for the work. In very soft unstable material, like quicksand, an arch of cut stone voussoirs may be necessary to withstand the pressure. A good quality of brick is occasionally used for lining, as they are easily handled and make good masonry if the pressure is not excessive. Only the best of cement mortar should be used, economy in this feature being the worst of folly. Of course the excavation must include the outside line of the lining. Any excavation which is made outside of this line (by the fall of earth or loose rock or by excessive blasting) must be refilled with stone well packed in. Occasionally it is necessary to fill these spaces with concrete. Of course it is not necessary that the lining be uniform throughout the tunnel.

167. Shafts. Shafts are variously made with square, rectangular, elliptical, and circular cross-sections. The rectangular

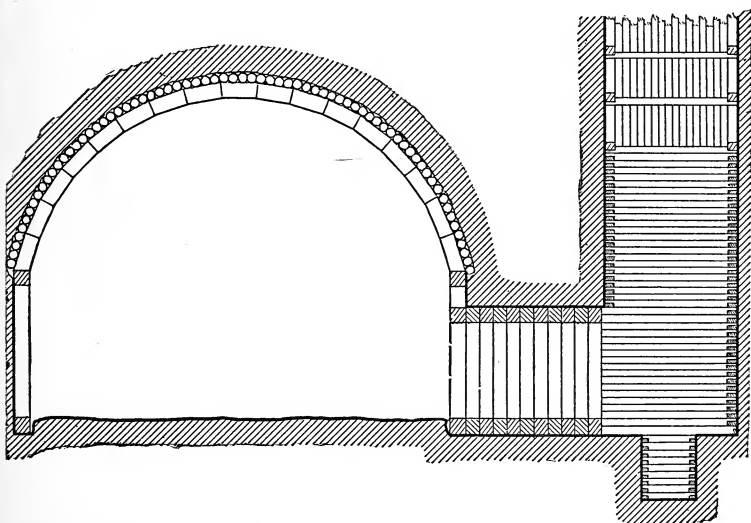


FIG. 90.—CONNECTION WITH SHAFT, CHURCH HILL TUNNEL.

cross-section, with the longer axis parallel with the tunnel, is most usually employed. Generally the shaft is directly over the center of the tunnel, but that always implies a complicated connection between the linings of the tunnel and shaft, provided

such linings are necessary. It is easier to sink a shaft near to one side of the tunnel and make an opening through the nearly vertical side of the tunnel. Such a method was employed in the Church Hill Tunnel, illustrated in Fig. 90.* Fig. 91† shows a cross-section for a large main shaft. Many shafts have been built with the idea of being left open permanently for ventilation and have therefore been elaborately lined with masonry.

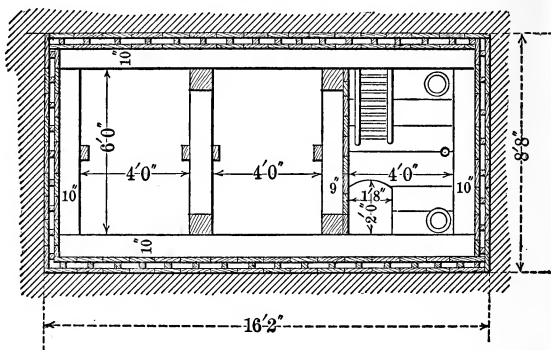


FIG. 91.—CROSS-SECTION. LARGE MAIN SHAFT.

The general consensus of opinion now appears to be that shafts are worse than useless for ventilation; that the quick passage of a train through the tunnel is the most effective ventilator; and that shafts only tend to produce cross-currents and are ineffective to clear the air. In consequence, many of these elaborately lined shafts have been permanently closed, and the more recent practice is to close up a shaft as soon as the tunnel is completed. Shafts always form drainage-wells for the material they pass through, and sometimes to such an extent that it is a serious matter to dispose of the water that collects at the bottom, requiring the construction of large and expensive drains.

168. Drains. A tunnel will almost invariably strike veins of water which will promptly begin to drain into the tunnel and not only cause considerable trouble and expense during construction, but necessitate the provision of permanent drains for its perpetual disposal. These drains must frequently be so large as

* Drinker's "Tunneling."

† Rziha, "Lehrbuch der Gesammten Tunnelbaukunst."

to appreciably increase the required cross-section of the tunnel. Generally a small open gutter on each side will suffice for this purpose, but in double-track tunnels a large covered drain is often built between the tracks. It is sometimes necessary to thoroughly grout the outside of the lining so that water will not force its way through the masonry and perhaps injure it, but may freely drain down the sides and pass through openings in the side walls near their base into the gutters.

CONSTRUCTION.

169. Headings. The methods of all tunnel excavation depend on the general principle that all earthy material, except the softest of liquid mud and quicksand, will be self-sustaining over a greater or less area and for a greater or less time after excavation is made, and the work consists in excavating some material and immediately propping up the exposed surface by timbering and poling-boards. The excavation of the cross-section begins with cutting out a "heading," which is a small horizontal drift whose breast is constantly kept 15 feet or more in advance of the full cross-sectional excavation. In solid self-sustaining rock, which will not decompose upon exposure to air, it becomes simply a matter of excavating the rock with the least possible expenditure of time and energy. In soft ground the heading must be heavily timbered, and as the heading is gradually enlarged the timbering must be gradually extended and perhaps replaced, according to some regular system, so that when the full cross-section has been excavated it is supported by such timbering as is intended for it. The heading is sometimes made on the center line near the top; with other plans, on the center line near the bottom; and sometimes two simultaneous headings are run in the two lower corners. Headings near the bottom serve the purpose of draining the material above it and facilitating the excavation. The simplest case of heading timbering is that shown in Fig. 92, in which cross-timbers are placed at intervals just under the roof, set in notches cut in the side walls and supporting poling-boards which sus-

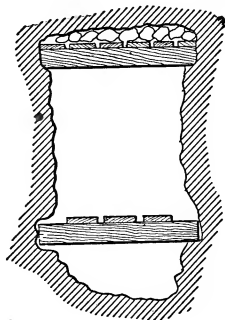


FIG. 92.

tain whatever pressure may come on them. Cross-timbers near the bottom support a flooring on which vehicles for transporting material may be run and under which the drainage may freely escape. As the necessity for timbering becomes greater, side timbers and even bottom timbers must be added, these timbers supporting poling-boards, and even the breast of the heading must be protected by boards suitably braced,

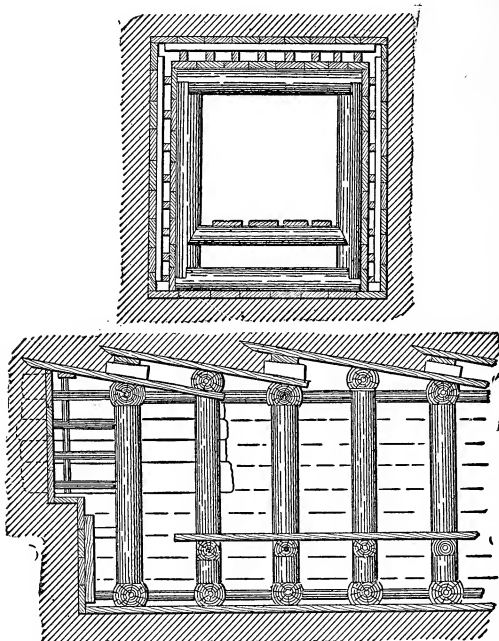


FIG. 93.—TIMBERING FOR TUNNEL HEADING.

as shown in Fig. 93. The supporting timbers are framed into collars in such a manner that added pressure only increases their rigidity.

170. Enlargement. Enlargement is accomplished by removing the poling-boards, one at a time, excavating a greater or less amount of material, and immediately supporting the exposed material with poling-boards suitably braced. (See Figs. 93 and 94.) This work being systematically done, space is thereby obtained in which the framing for the full cross-section may be gradually introduced. The framing is constructed with a cross-

section so large that the masonry lining may be constructed within it.

171. Distinctive features of various methods of construction. There are six general systems, known as the English, German, Belgian, French, Austrian, and American. They are so named

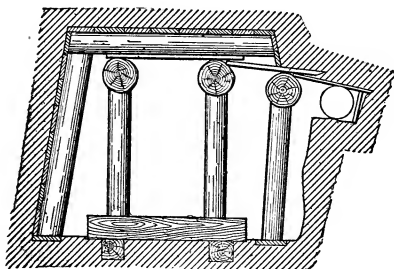
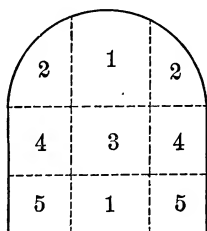


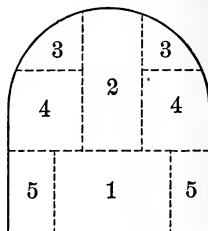
FIG. 94.

from the origin of the methods, although their use is not confined to the countries named. Fig. 95 shows by numbers (1 to 5) the order of the excavation within the cross-sections. The English, Austrian, and American systems are alike in excavating the entire cross-section before beginning the construction of the masonry lining. The German method leaves a solid core (5) until practically the whole of the lining is complete. This has the disadvantage of extremely cramped quarters for work, poor ventilation, etc. The Belgian and French methods agree in excavating the upper part of the section, building the arch at once, and supporting it temporarily until the side walls are built. The Belgian method then takes out the core (3), removes very short sections of the sides (4) immediately underpinning the arch with short sections of the side walls and thus gradually constructing the whole side wall. The French method digs out the sides (3), supporting the arch temporarily with timbers and then replacing the timbers with masonry; the core (4) is taken out last. The French method has the same disadvantage as the German—working in a cramped space. The Belgian and French systems have the disadvantage that the arch, supported temporarily on timber, is very apt to be strained and cracked by the slight settlement that so frequently occurs in soft material. The English, Austrian, and American methods differ mainly in the

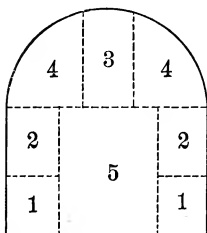
design of the timbering. The English support the roof by lines of very heavy *longitudinal* timbers which are supported at comparatively wide intervals by a heavy framework occupying the



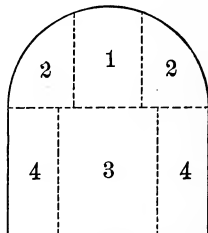
ENGLISH



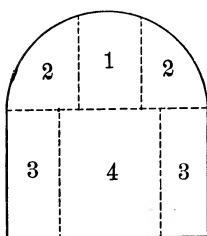
AUSTRIAN



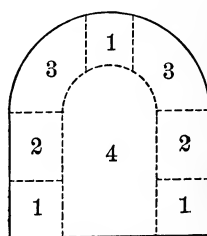
GERMAN



BELGIAN



FRENCH



AMERICAN

FIG. 95.—ORDER OF WORKING BY THE VARIOUS SYSTEMS.

whole cross-section. The Austrian system uses such frequent cross-frames of timber-work that poling-boards will suffice to support the material between the frames. The American system agrees with the Austrian in using frequent cross-frames

supporting poling-boards, but differs from it in that the "cross-frames" consist simply of arches of 3 to 15 wooden voussoirs, the voussoirs being blocks of 12"×12" timber about 2 to 8 feet long and cut with joints normal to the arch. These arches are put together on a centering which is removed as soon as the arch is keyed up and thus immediately opens up the full cross-section, so that the center core (4) may be immediately dug out and the masonry constructed in a large open space. The American system has been used successfully in very soft ground, but its advantages are greater in loose rock, when it is much cheaper than the other methods which employ more timber. Fig. 90 and Plate III illustrate the use of the American system. Fig. 90 shows the wooden arch in place. The masonry arch may be placed when convenient, since it is *possible* to lay the track and commence traffic as soon as the wooden arch is in place. The student is referred to Drinker's "Tunneling" and to Rziha's "Lehrbuch der Gesamten Tunnelbaukunst" for numerous illustrations of European methods of tunnel timbering.

172. Ventilation during construction. Tunnels of any great length must be artificially ventilated during construction. If the excavated material is rock so that blasting is necessary, the need for ventilation becomes still more imperative. The invention of compressed-air drills simultaneously solved two difficulties. It introduced a motive power which is unobjectionable in its application (as gas would be), and it also furnished at the same time a supply of just what is needed—pure air. If no blasting is done (and frequently even when there is blasting), air must be supplied by direct pumping. The cooling effect of the sudden expansion of compressed air only reduces the otherwise objectionably high temperature sometimes found in tunnels. Since pure air is being continually pumped in, the foul air is thereby forced out.

173. Excavation for the portals. Under normal conditions there is always a greater or less amount of open cut preceding and following a tunnel. Since all tunnel methods depend (to some slight degree at least) on the capacity of the exposed material to act as an arch, there is implied a considerable thickness of material above the tunnel. This thickness is reduced to nearly zero over the tunnel portals and therefore requires special treatment, particularly when the material is very soft. Fig. 96 *

* Rziha, "Lehrbuch der Gesamten Tunnelbaukunst."

illustrates one method of breaking into the ground at a portal. The loose stones are piled on the framing to give stability to the framing by their weight and also to retain the earth on the

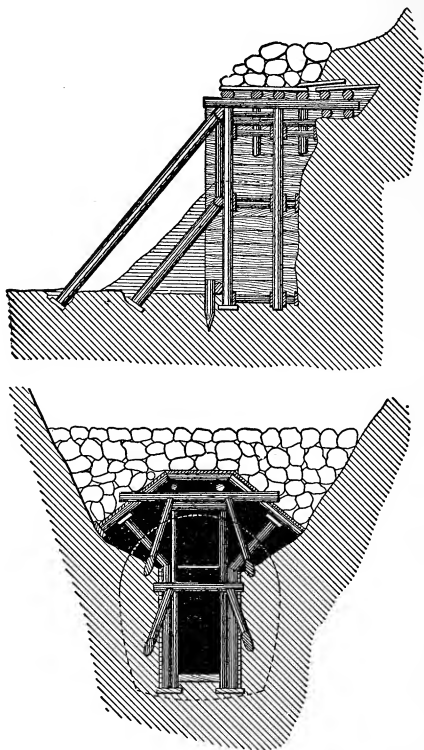
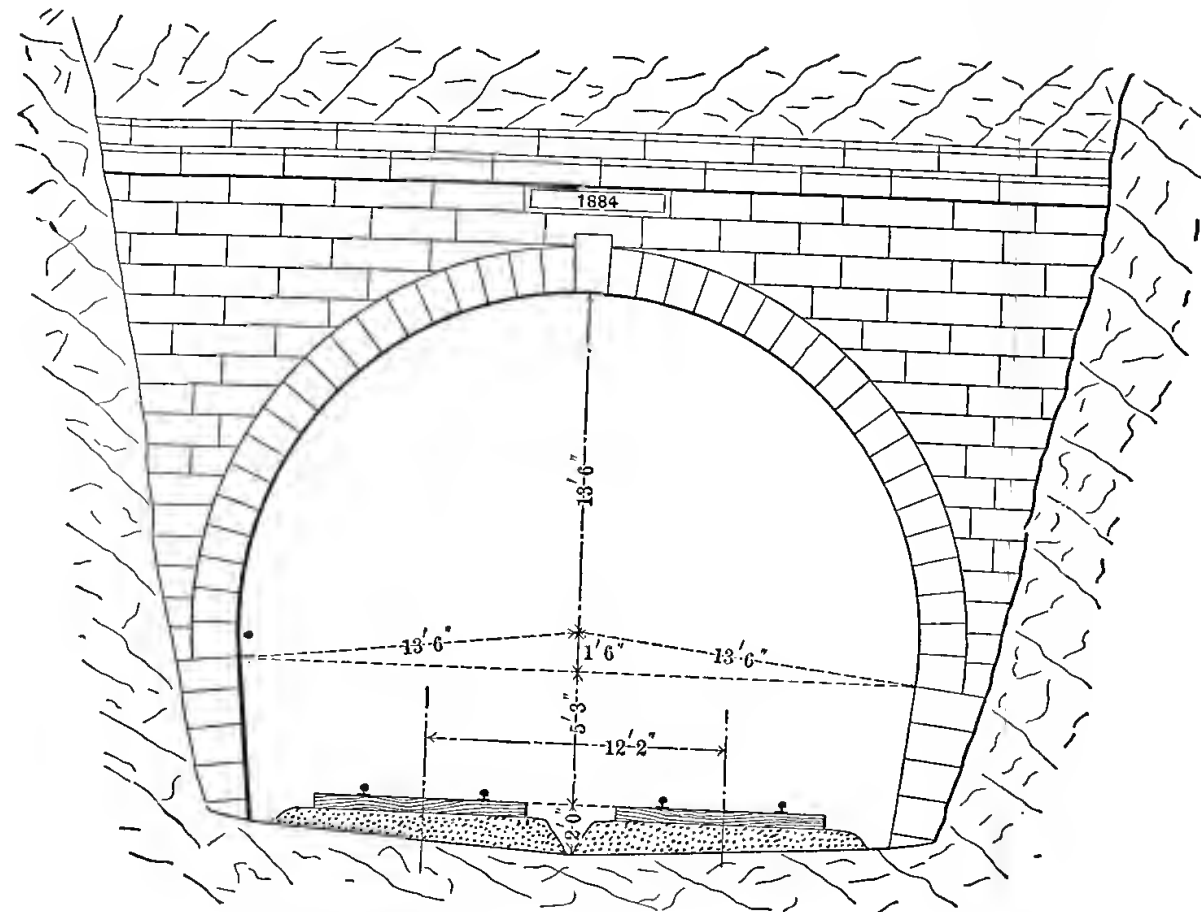
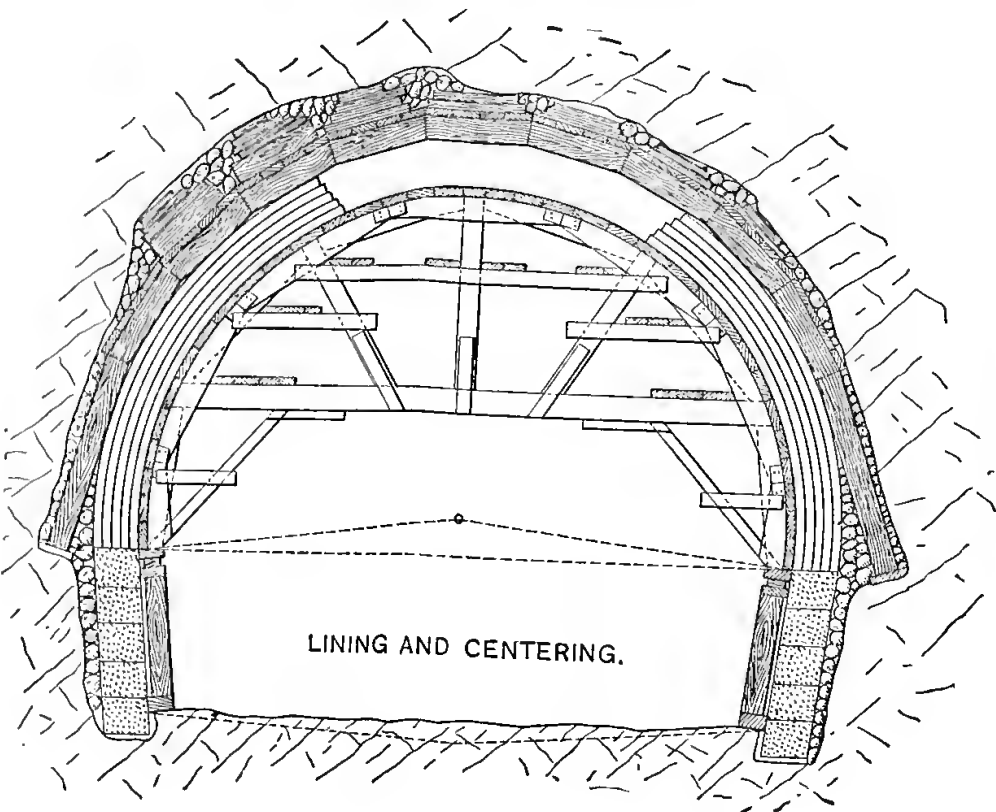
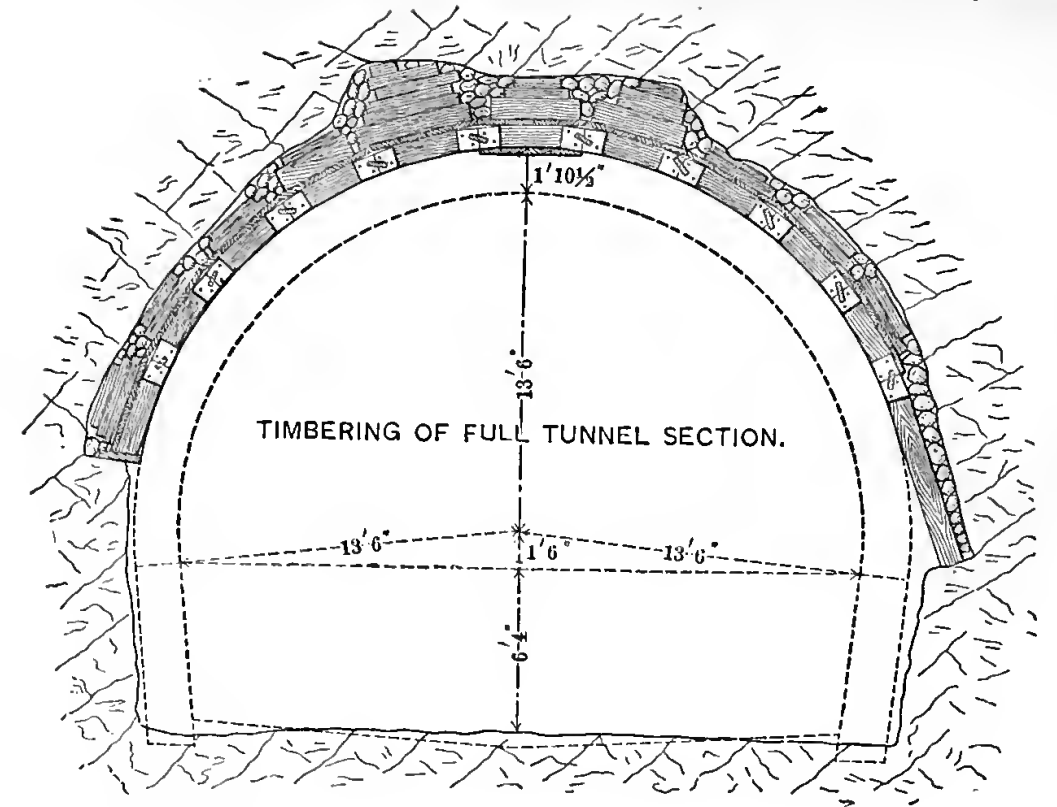
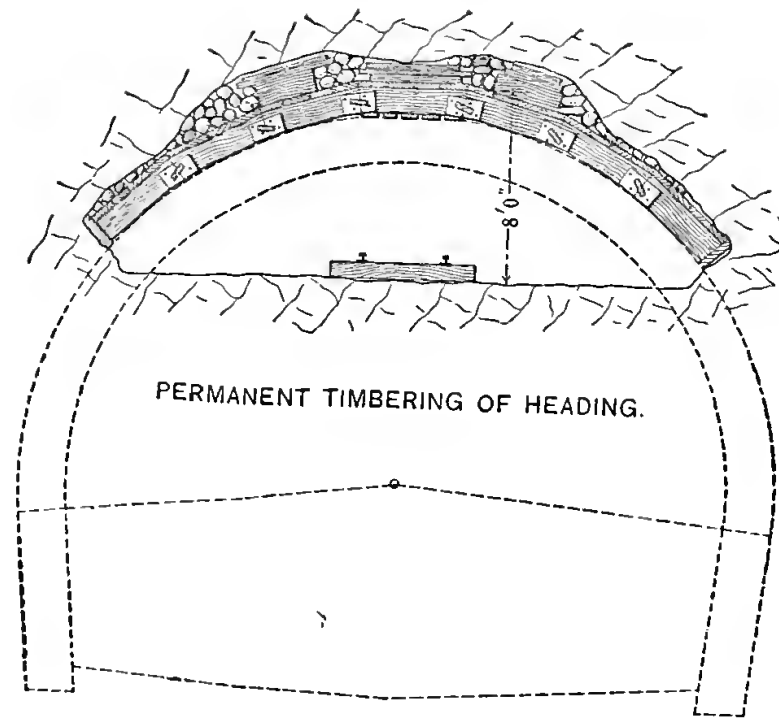
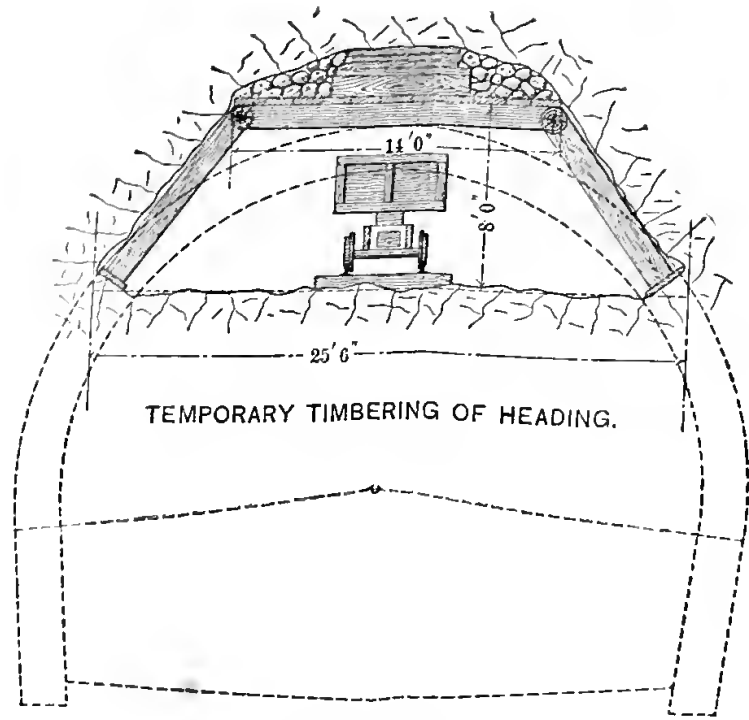


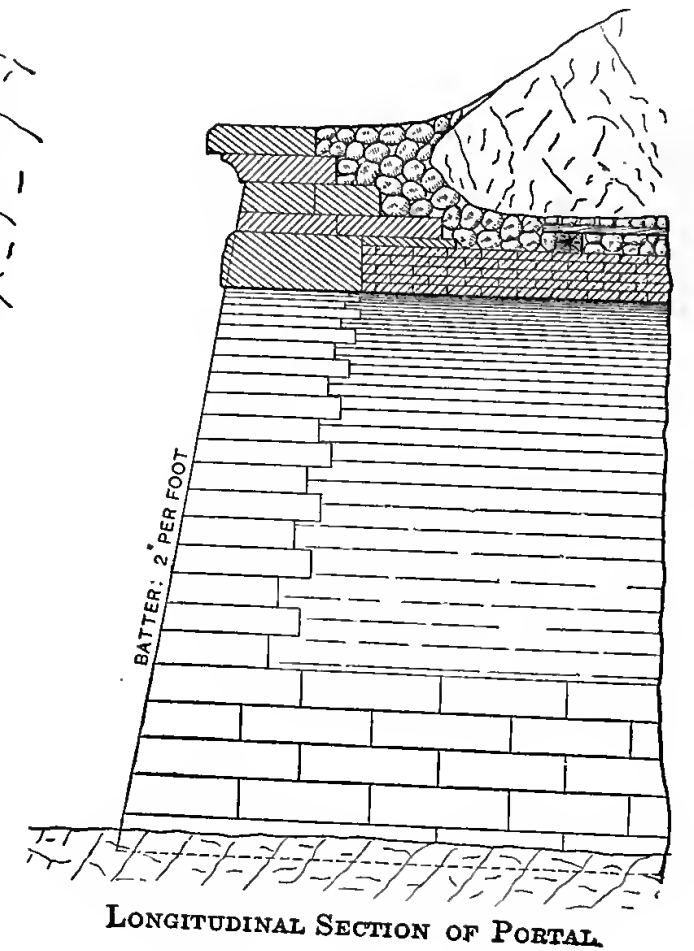
FIG. 96.—TIMBERING FOR TUNNEL PORTAL.

slope above. Another method is to sink a temporary shaft to the tunnel near the portal; immediately enlarge to the full size and build the masonry lining; then work back to the portal. This method is more costly, but is preferable in very treacherous ground, it being less liable to cause landslides of the surface material.

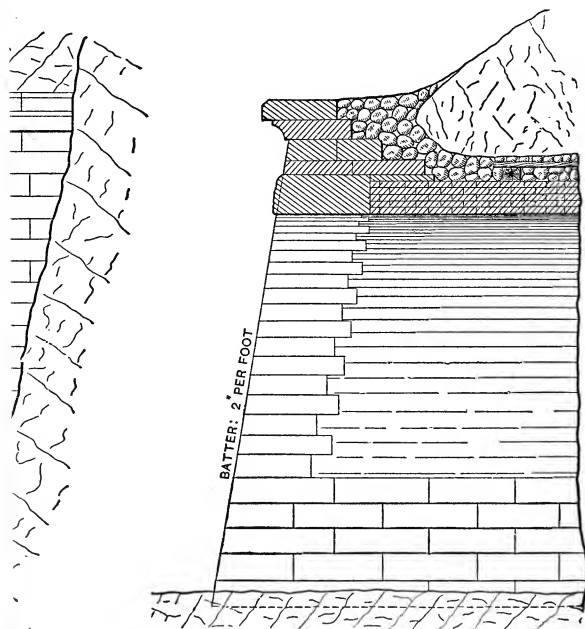
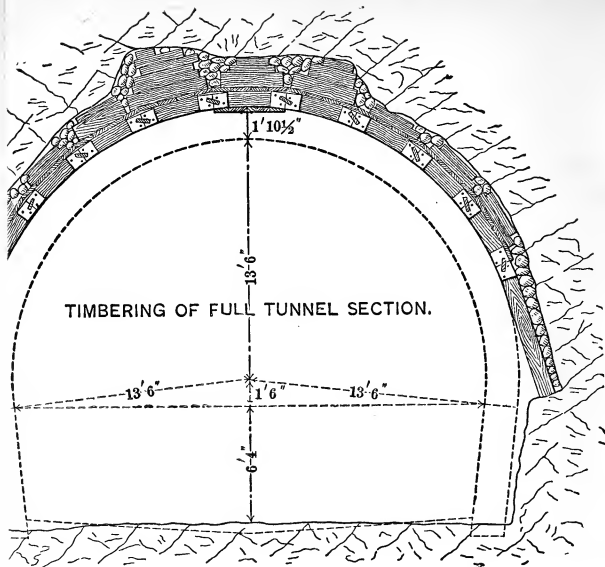
174. Tunnels vs. open cuts. In cases in which an open cut rather than a tunnel is a possibility the ultimate consideration is generally that of first cost combined with other financial con-



ELEVATION OF PORTAL.
PHOENIXVILLE TUNNEL. P. S. V. R. R.



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LONGITUDINAL SECTION OF PORTAL.

siderations and annual maintenance charges directly or indirectly connected with it. Even when an open cut may be constructed at the same cost as a tunnel (or perhaps a little cheaper) the tunnel may be preferable under the following conditions:

1. When the soil indicates that the open cut would be liable to landslides.

2. When the open cut would be subject to excessive snow-drifts or avalanches.

3. When land is especially costly or it is desired to run under existing costly or valuable buildings or monuments. When running through cities, tunnels are sometimes constructed as open cuts and then arched over.

These cases apply to tunnels *vs.* open cuts when the alignment is fixed by other considerations than the mere topography. The broader question of excavating tunnels to avoid excessive grades or to save distance or curvature, and similar problems, are hardly susceptible of general analysis except as questions of railway economics and must be treated individually.

175. Cost of tunneling. The cost of any construction which involves such uncertainties as tunneling is very variable. It depends on the material encountered, the amount and kind of timbering required, on the size of the cross-section, on the price of labor, and especially on the *reconstruction* that may be necessary on account of mishaps.

Headings generally cost \$4 to \$5 per cubic yard for excavation, while the remainder of the cross-section in the same tunnel may cost about half as much. The average cost of a large number of tunnels in this country may be seen from the following table:*

Material.	Cost per cubic yard.				Cost per lineal foot.	
	Excavation.		Masonry.		Single.	Double.
	Single.	Double.	Single.	Double.		
Hard rock.....	\$5.89	\$5.45	\$12.00	\$8.25	\$69.76	\$142.82
Loose rock.....	3.12	3.48	9.07	10.41	80.61	119.26
Soft ground....	3.62	4.64	15.00	10.50	135.31	174.42

* Figures derived from Drinker's "Tunneling."

A considerable variation from these figures may be found in individual cases, due sometimes to unusual skill (or the lack of it) in prosecuting the work, but the figures will generally be sufficiently accurate for preliminary estimates or for the comparison of two proposed routes.

CHAPTER VI.

CULVERTS AND MINOR BRIDGES.

176. Definition and object. Although a variable percentage of the rain falling on any section of country soaks into the ground and does not immediately reappear, yet a very large percentage flows over the surface, always seeking and following the lowest channels. The roadbed of a railroad is constantly intersecting these channels, which frequently are normally dry. In order to prevent injury to railroad embankments by the impounding of such rainfall, it is necessary to construct waterways through the embankment through which such rainflow may freely pass. Such waterways, called culverts, are also applicable for the bridging of very small although perennial streams, and therefore in this work the term culvert will be applied to all water-channels passing through a railroad embankment which are not of sufficient magnitude to require a special structural design, such as is necessary for a large masonry arch or a truss bridge.

177. Elements of the design. A well-designed culvert must afford such free passage to the water that it will not "back up" over the adjoining land nor cause any injury to the embankment or culvert. The ability of the culvert to discharge freely all the water that comes to it evidently depends chiefly on the area of the waterway, but also on the form, length, slope, and materials of construction of the culvert and the nature of the approach and outfall. When the embankment is very low and the amount of water to be discharged very great, it sometimes becomes necessary to allow the water to discharge "under a head," i.e., with the surface of the water above the top of the culvert. Safety then requires a much stronger construction than would otherwise be necessary to avoid injury to the culvert or embankment by washing. The necessity for such construction should be avoided if possible.

AREA OF THE WATERWAY.

178. Elements involved. The determination of the required area of the waterway involves such a multiplicity of indeterminate elements that any close determination of its value from purely theoretical considerations is a practical impossibility. The principal elements involved are:

a. Rainfall. The real test of the culvert is its capacity to discharge without injury the flow resulting from the extraordinary rainfalls and "cloud bursts" that may occur once in many years. Therefore, while a knowledge of the average annual rainfall is of very little value, a record of the maximum rainfall during heavy storms for a long term of years may give a relative idea of the maximum demand on the culvert.

b. Area of watershed. This signifies the total area of country draining into the channel considered. When the drainage area is very small it is sometimes included within the area surveyed by the preliminary survey. When larger it is frequently possible to obtain its area from other maps with a percentage of accuracy sufficient for the purpose. Sometimes a special survey for the purpose is considered justifiable.

c. Character of soil and vegetation. This has a large influence on the rapidity with which the rainflow from a given area will reach the culvert. If the soil is hard and impermeable and the vegetation scant, a heavy rain will run off suddenly, taxing the capacity of the culvert for a short time, while a spongy soil and dense vegetation will retard the flow, making it more nearly uniform and the maximum flow at any one time much less.

d. Shape and slope of watershed. If the watershed is very long and narrow (other things being equal), the water from the remoter parts will require so much longer time to reach the culvert that the flow will be comparatively uniform, especially when the slope of the whole watershed is very low. When the slope of the remoter portions is quite steep it may result in the nearly simultaneous arrival of a storm-flow from all parts of the watershed, thus taxing the capacity of the culvert.

e. Effect of design of culvert. The principles of hydraulics show that the slope of the culvert, its length, the form of the cross-section, the nature of the surface, and the form of the

approach and discharge all have a considerable influence on the area of cross-section required to discharge a given volume of water in a given time, but unfortunately the combined hydraulic effect of these various details is still a very uncertain quantity.

179. Methods of computation of area. There are three possible methods of computation.

(a) **Theoretical.** As shown above it is a practical impossibility to estimate correctly the combined effect of the great multiplicity of elements which influence the final result. The nearest approach to it is to estimate by the use of empirical formulæ the amount of water which will be presented at the upper end of the culvert in a given time and then to compute, from the principles of hydraulics, the rate of flow through a culvert of given construction, but (as shown in § 178, *e*) such methods are still very unreliable, owing to lack of experimental knowledge. This method has *apparently* greater scientific accuracy than other methods, but a little study will show that the elements of uncertainty are as great and the final result no more reliable. The method is most reliable for streams of uniform flow, but it is under these conditions that method (*c*) is most useful. The theoretical method will not therefore be considered further.

(b) **Empirical.** As illustrated in § 180, some formulæ make the area of waterway a function of the drainage area, the formula being affected by a coefficient the value of which is estimated between limits according to the judgment. Assuming that the formulæ are sound, their use only narrows the limits of error, the final determination depending on experience and judgment.

(c) **From observation.** This method, considered by far the best for permanent work, consists in observing the high-water marks on contracted channel-openings which are on the same stream and as near as possible to the proposed culvert. If the country is new and there are no such openings, the wisest plan is to bridge the opening by a temporary structure in wood which has an ample waterway (see § 126, *b*, 4) and carefully observe all high-water marks on that opening during the 6 to 10 years which is ordinarily the minimum life of such a structure. As shown later, such observations may be utilized for a close computation of the required waterway. Method (*b*) may be utilized for an approximate calculation for the required area for the tem-

porary structure, using a value which is intentionally excessive, so that a permanent structure of sufficient capacity may subsequently be constructed *within* the temporary structure.

180. Empirical formulæ. Two of the best known empirical formulæ for area of the waterway are the following:

(a) Myer's formula:

Area of waterway in square feet $= C \times \sqrt{\text{drainage area in acres}}$, where C is a coefficient varying from 1 for flat country to 4 for mountainous country and rocky ground. As an illustration, if the drainage area is 100 acres, the waterway area should be from 10 to 40 square feet, according to the value of the coefficient chosen. It should be noted that this formula does not regard the great variations in rainfall in various parts of the world nor the design of the culvert, and also that the final result depends largely on the choice of the coefficient.

(b) Talbot's formula:

Area of waterway in square feet $= C \times \sqrt[4]{(\text{drainage area in acres})^3}$. "For steep and rocky ground C varies from $\frac{2}{3}$ to 1. For rolling agricultural country subject to floods at times of melting snow, and with the length of the valley three or four times its width, C is about $\frac{1}{3}$; and if the stream is longer in proportion to the area, decrease C . In districts not affected by accumulated snow, and where the length of the valley is several times the width, $\frac{1}{3}$ or $\frac{1}{6}$, or even less, may be used. C should be increased for steep side slopes, especially if the upper part of the valley has a much greater fall than the channel at the culvert."* As an illustration, if the drainage area is 100 acres the area of waterway should be $C \times 31.6$. The area should then vary from 5 to 31 square feet, according to the character of the country. Like the previous estimate, the result depends on the choice of a coefficient and disregards local variations in rainfall, except as they may be arbitrarily allowed for in choosing the coefficient.

181. Value of empirical formulæ. The fact that these formulæ, as well as many others of similar nature that have been suggested, depend so largely upon the choice of the coefficient shows that they are valuable "more as a guide to the judgment than as a working rule," as Prof. Talbot explicitly declares in

* Prof. A. N. Talbot, "Selected Papers of the Civil Engineers' Club of the Univ. of Illinois,"

commenting on his own formula. In short, they are chiefly valuable in indicating a probable maximum and minimum between which the true result probably lies.

182. Results based on Observation. As already indicated in § 179, observation of the stream in question gives the most reliable results. If the country is new and no records of the flow of the stream during heavy storms has been taken, even the life of a temporary wooden structure may not be long enough to include one of the unusually severe storms which must be allowed for, but there will usually be some high-water mark which will indicate how much opening will be required. The following quotation illustrates this: "A tidal estuary may generally be safely narrowed considerably from the extreme water lines if stone revetments are used to protect the bank from wash. Above the true estuary, where the stream cuts through the marsh, we generally find nearly vertical banks, and we are safe if the faces of abutments are placed even with the banks. In level sections of the country, where the current is sluggish, it is usually safe to encroach somewhat on the general width of the stream, but in rapid streams among the hills the width that the stream has cut for itself through the soil should not be lessened, and in ravines carrying mountain torrents the openings must be left very much larger than the ordinary appearance of the banks of the stream would seem to make necessary." *

As an illustration of an observation of a storm-flow through a temporary trestle, the following is quoted: "Having the flood height and velocity, it is an easy matter to determine the volume of water to be taken care of. I have one ten-bent pile trestle 135 feet long and 24 feet high over a spring branch that ordinarily runs about six cubic inches per second. Last summer during one of our heavy rainstorms (four inches in less than three hours) I visited this place and found by float observations the surface velocity at the highest stage to be 1.9 feet per second. I made a high-water mark, and after the flood-water receded found the width of stream to be 12 feet and an average depth of $2\frac{3}{4}$ feet. This, with a surface velocity of 1.9 feet per second, would give approximately a discharge of 50

* J. P. Snow, Boston & Maine Railway. From Report to Association of Railway Superintendents of Bridges and Buildings. 1897.

cubic feet, or 375 gallons, per second. Having this information it is easy to determine size of opening required." *

183. Degree of accuracy required. The advantages resulting from the use of standard designs for culverts (as well as other structures) have led to the adoption of a comparatively small number of designs. The practical use made of a computation of required waterway area is to determine which one of several standard designs will most nearly fulfill the requirements. For example, if a 24-inch iron pipe, having an area of 3.14 square feet, is considered to be a little small, the next size (30-inch) would be adopted; but a 30-inch pipe has an area of 4.92 square feet, which is 56% larger. A similar result, except that the percentage of difference might not be quite so marked, will be found by comparing the areas of consecutive standard designs for stone box culverts.

The advisability of designing a culvert to withstand any storm-flow that may *ever* occur is considered doubtful. Several years ago a record-breaking storm in New England carried away a very large number of bridges, etc., hitherto supposed to be safe. It was not afterward considered that the design of those bridges was faulty, because the extra cost of constructing bridges capable of withstanding such a flood, added to interest for a long period of years, would be enormously greater than the cost of repairing the damages of such a storm once or twice in a century. Of course the element of danger has some weight, but not enough to justify a great additional expenditure, for common prudence would prompt unusual precautions during or immediately after such an extraordinary storm.

PIPE CULVERTS.

184. Advantages. Pipe culverts, made of cast iron or earthenware, are very durable, readily constructed, moderately cheap, will pass a larger volume of water in proportion to the area than many other designs on account of the smoothness of the surface, and (when using iron pipe) may be used very close to the track when a low opening of large capacity is required. Another advantage lies in the ease with which they may be inserted through a somewhat larger opening that has been

* A. J. Kelley, Kansas City Belt Railway. From Report to Association of Railway Superintendents of Bridges and Buildings. 1897.

temporarily lined with wood, without disturbing the roadbed or track.

185. Construction. Permanency requires that the foundation shall be firm and secure against being washed out. To accomplish this, the soil of the trench should be hollowed out to fit the lower half of the pipe, making suitable recesses for the bells. In very soft treacherous soil a foundation-block of concrete is sometimes placed under each joint, or even throughout the whole length. When pipes are laid through a slightly larger timber culvert great care should be taken that the pipes are properly supported, so that there will be no settling nor development of unusual strains when the timber finally decays and gives way. To prevent the washing away of material around the pipe the ends should be protected by a bulkhead. This is best constructed of masonry (see Fig. 97), although wood is sometimes used for cheap and minor constructions. The joints should be calked, especially when the culvert is liable to run full or when the outflow is impeded and the culvert is liable to be partly or wholly filled during freezing weather. The cost of a calking of clay or even hydraulic cement is insignificant compared with the value of the additional safety afforded. When the grade of the pipe is perfectly uniform, a very low rate of grade will suffice to drain a pipe culvert, but since some unevenness of grade is inevitable through uneven settlement or imperfect construction, a grade of 1 in 20 should preferably be required, although much less is often used. The length of a pipe culvert is approximately determined as follows:

$$\text{Length} = 2s \text{ (depth of embankment) } + \text{(width of roadbed),}$$

in which s is the slope ratio (horizontal to vertical) of the banks. In practice an even number of lengths should be used which will equal or exceed the length given by this formula.

186. Iron-pipe culverts. Simple cast-iron pipes are used in sizes from 12" to 48" diameter. These are usually made in lengths of 12 feet with a few lengths of 6 feet, so that any required length may be more nearly obtained. The lightest pipes made are sufficiently strong for the purpose, and even those which would be rejected because of incapacity to withstand considerable internal pressure may be utilized for this work. In Fig. 97 are shown the standard plans used on the C. C. C. & St. L. Ry., which may be considered as typical plans.

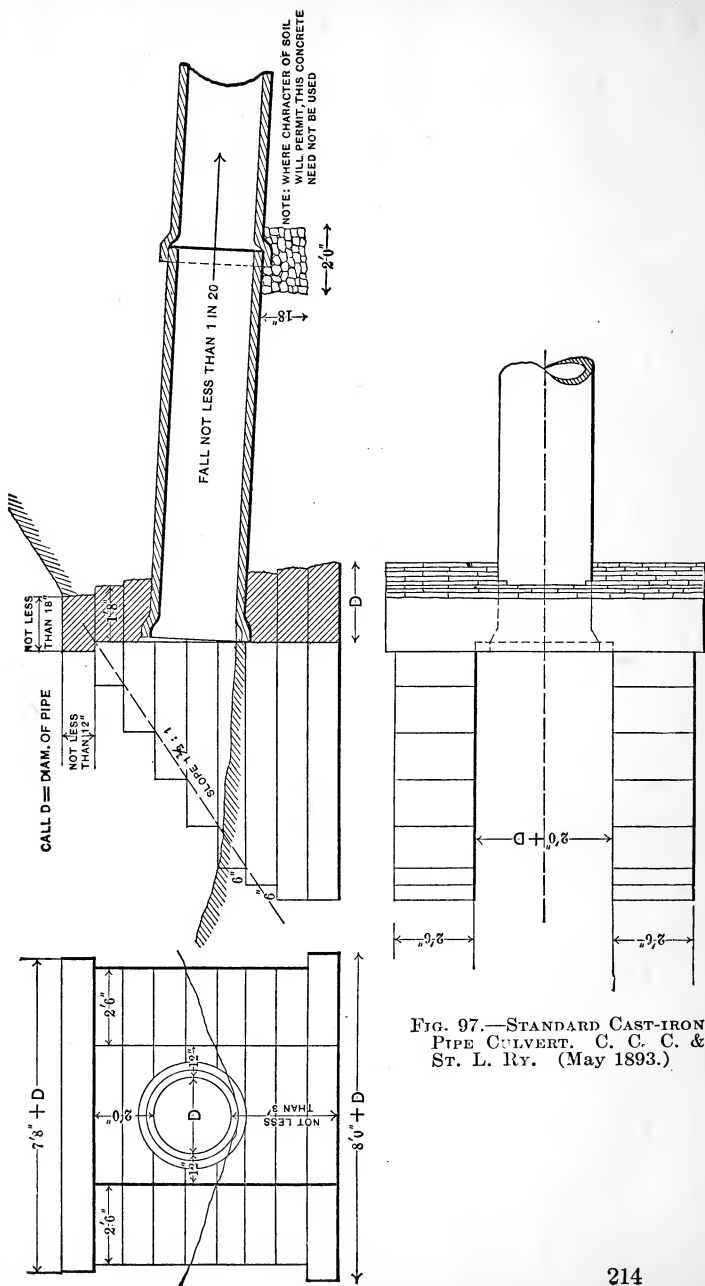


FIG. 97.—STANDARD CAST-IRON
PIPE CULVERT. C. C. C. &
ST. L. RY. (May 1893.)

Pipes formed of cast-iron segments have been used up to 12 feet diameter. The shell is then made comparatively thin, but is stiffened by ribs and flanges on the outside. The segments break joints and are bolted together through the flanges. The joints are made tight by the use of a tarred rope, together with neat cement.

187. Tile-pipe culverts. The pipes used for this purpose vary from 12" to 24" in diameter. When a larger capacity is required two or more pipes may be laid side by side, but in such a case another design might be preferable. It is frequently specified that "double-strength" or "extra-heavy" pipe shall be used, evidently with the idea that the stresses on a culvert-pipe are greater than on a sewer-pipe. But it has been conclusively demonstrated that, no matter how deep the embankment, the pressure cannot exceed a somewhat uncertain maximum, also that the greatest danger consists in placing the pipe so near the ties that shocks may be directly transferred to the pipe without the cushioning effect of the earth and ballast. When the pipes are well bedded in *clear* earth and there is a

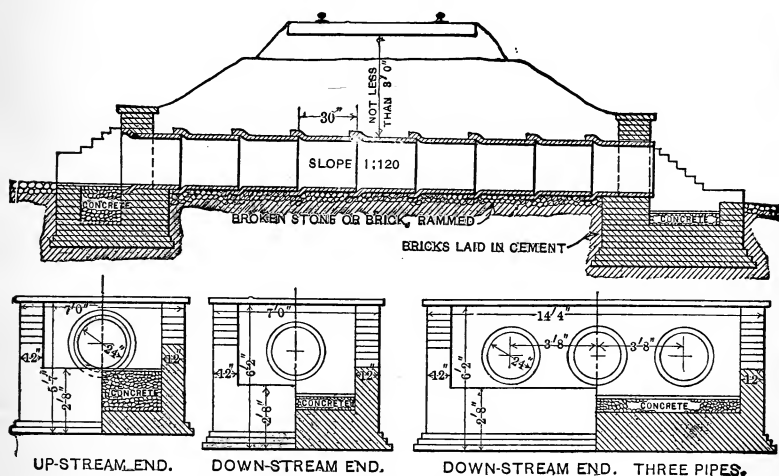


FIG. 98.—STANDARD VITRIFIED-PIPE CULVERT. PLANT SYSTEM. (1891.)

sufficient depth of earth over them to avoid direct impact (at least three feet) the ordinary sewer-pipe will be sufficiently strong. "Double-strength" pipe is frequently less perfectly burned, and the supposed extra strength is not therefore ob-

tained. In Fig. 98 are shown the standard plans for vitrified-pipe culverts as used on the "Plant system." Tile pipe is much cheaper than iron pipe, but is made in much shorter lengths and requires much more work in laying and especially to obtain a uniform grade.

BOX CULVERTS.

188. Wooden box culverts. This form serves the purpose of a cheap temporary construction which allows the use of a ballasted roadbed. As in all temporary constructions, the area should be made considerably larger than the calculated area (§§ 179–182), not only for safety but also in order that, if the smaller area is demonstrated to be sufficiently large, the permanent construction (probably pipe) may be placed inside without disturbing the embankment. All designs agree in using heavy timbers ($12'' \times 12''$, $10'' \times 12''$, or $8'' \times 12''$) for the side walls, cross-timbers for the roof, every fifth or sixth timber being notched down so as to take up the thrust of the side walls, and planks for the flooring. Fig. 99 shows some of the standard designs as used by the C., M. & St. P. Ry.

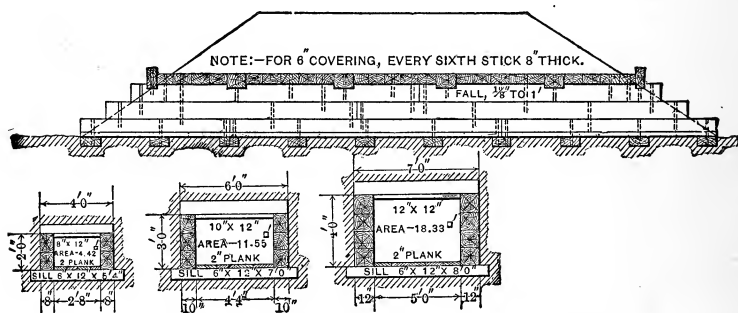


FIG. 99.—STANDARD TIMBER BOX CULVERT. C., M. & St. P. Ry. (Feb. 1889.)

189. Stone box culverts. In localities where a *good* quality of stone is cheap, stone box culverts are the cheapest form of permanent construction for culverts of medium capacity, but their use is decreasing owing to the frequent difficulty in obtaining really suitable stone within a reasonable distance of the culvert. The clear span of the cover-stones varies from 2 to 4 feet. The required thickness of the cover-stones is sometimes

calculated by the theory of transverse strains on the basis of certain assumptions of loading—as a function of the height of the embankment and the unit strength of the stone used. Such a method is simply another illustration of a class of calculations which look very precise and beautiful, but which are worse than useless (because misleading) on account of the hopeless uncer-

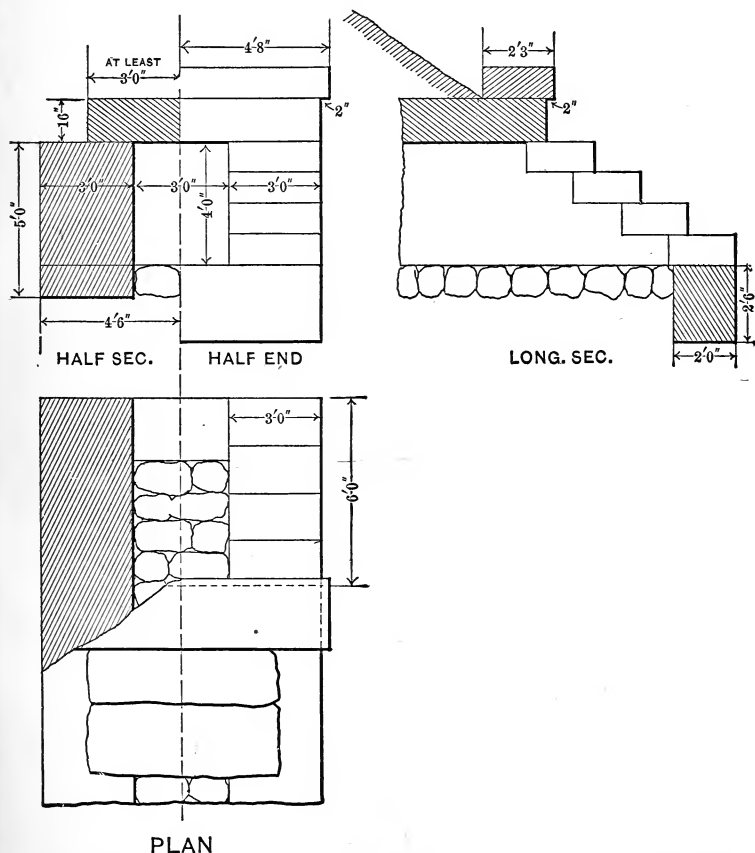


FIG. 100.—STANDARD SINGLE STONE CULVERT (3'×4'). N. & W. R.R.
(1890.)

tainty as to the true value of certain quantities which must be used in the computations. In the first place the true value of the unit tensile strength of stone is such an uncertain and variable

portionate loading, until at some uncertain height an increase in height will not increase the load on the cover-stones. The effect of frost is likewise large, but uncertain and not computable. The usual practice is therefore to make the thickness such as experience has shown to be safe with a good quality of stone, i.e., about 10 or 12 inches for 2 feet span and up to 16 or 18 inches for 4 feet span. The side walls should be carried down deep enough to prevent their being undermined by scour or heaved by frost. The use of cement mortar is also an important feature of first-class work, especially when there is a rapid scouring current or a liability that the culvert will run under a head. In Figs. 100 and 100a are shown standard plans for single and double stone box culverts as used on the Norfolk and Western R.R.

190. Old-rail culverts. It sometimes happens (although very rarely) that it is necessary to bring the grade line within 3 or 4 feet of the bottom of a stream and yet allow an area of 10 or 12 square feet. A single large pipe of sufficient area could not be used in this case. The use of several smaller pipes side by side would be both expensive and inefficient. For similar reasons neither wooden nor stone box culverts could be used. In such cases, as well as in many others where the head-room is not so limited, the plan illustrated in Fig. 101 is a very satisfactory

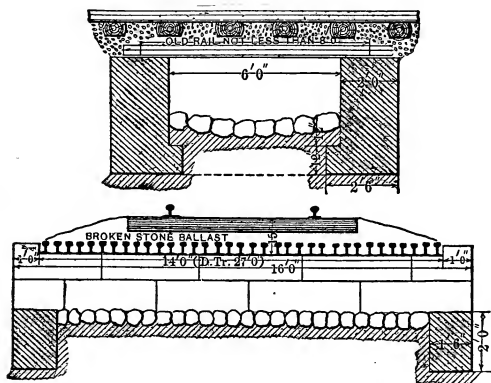


FIG. 101.—STANDARD OLD-RAIL CULVERT. N. & W. R.R. (1895.)

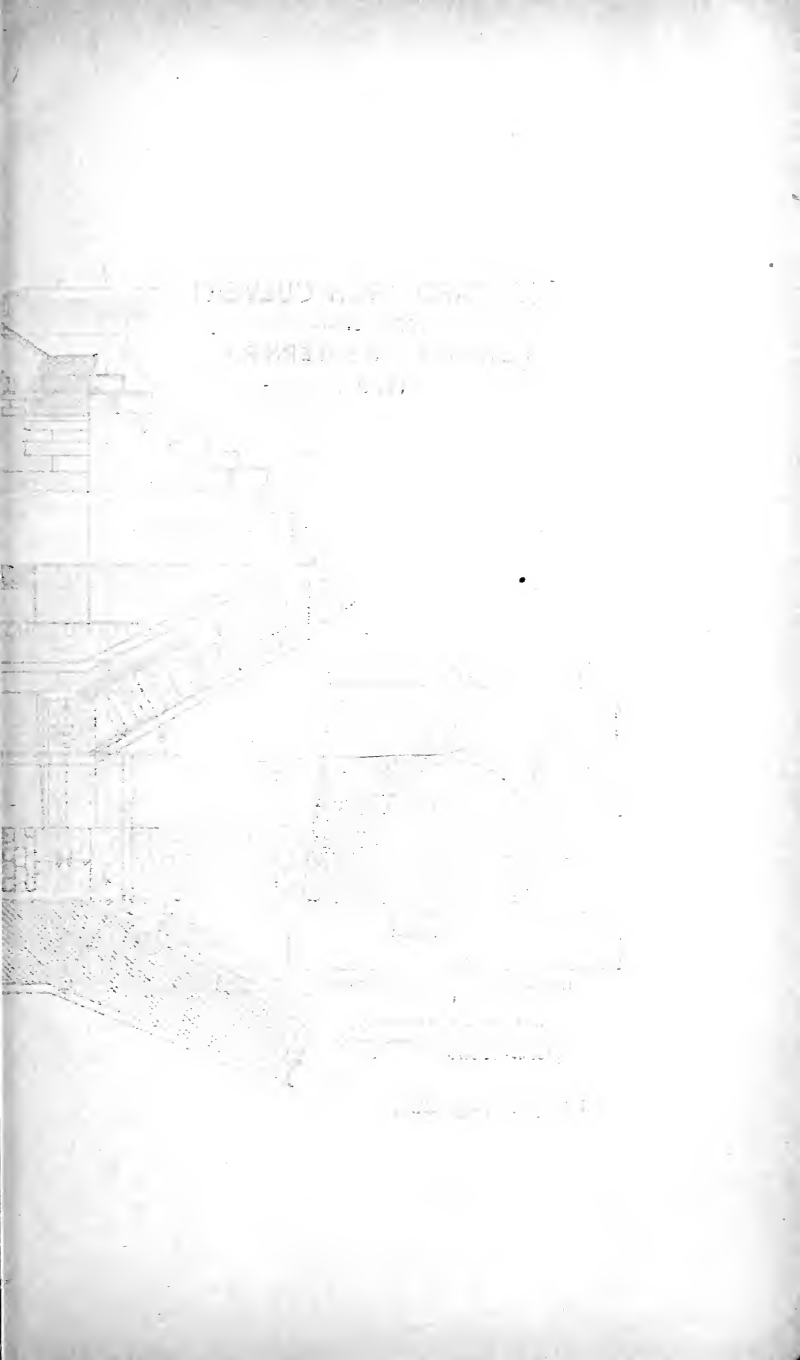
solution of the problem. The old rails, having a length of 8 or 9 feet, are laid close together across a 6-foot opening. Sometimes the rails are held together by long bolts passing through

the webs of the rails. In the plan shown the rails are confined by low end walls on each abutment. This plan requires only 15 inches between the base of the rail and the top of the culvert channel. It also gives a continuous ballasted roadbed.

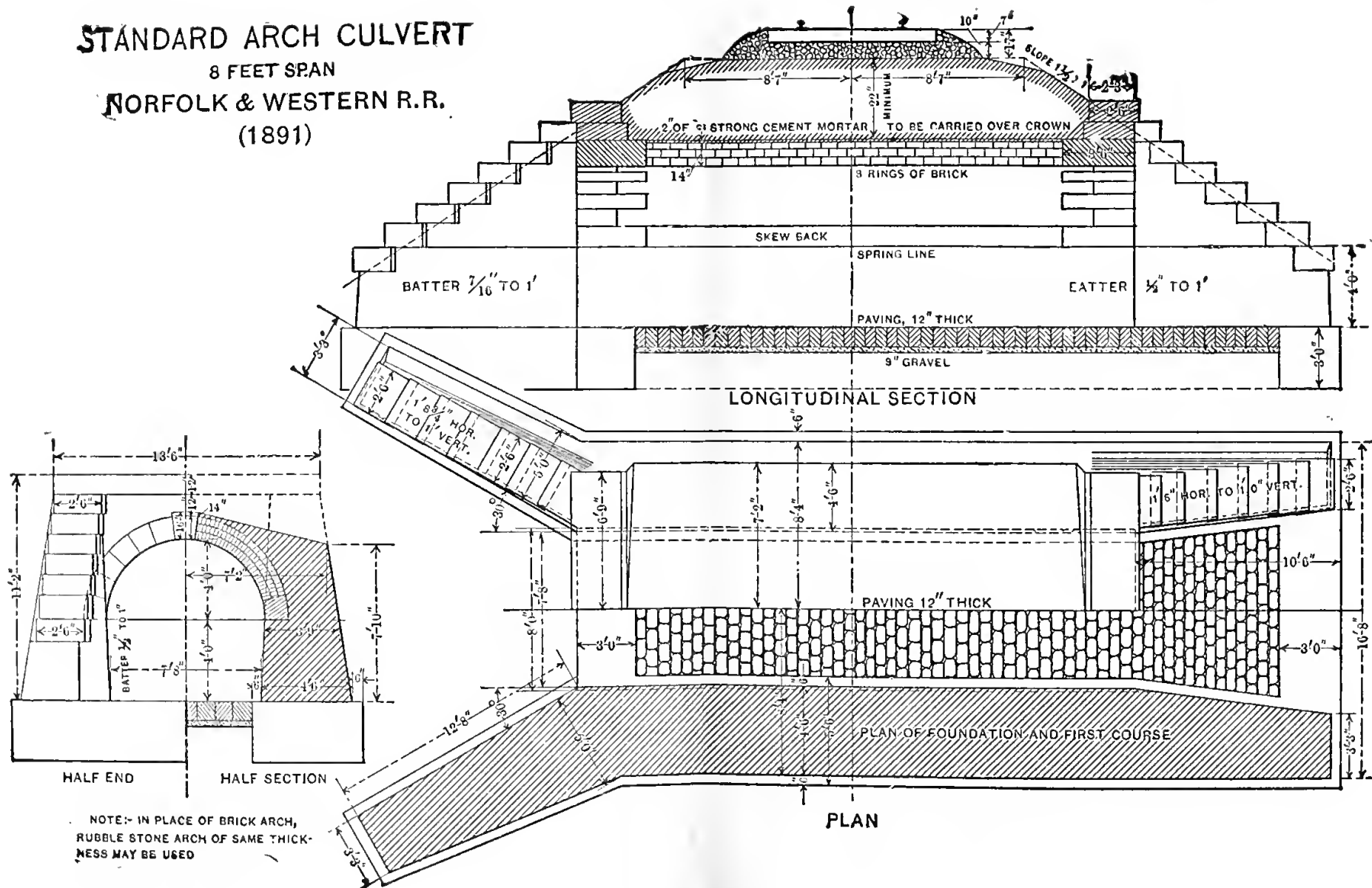
190a. Reinforced Concrete Culverts. The development of reinforced concrete as a structural material is illustrated in its extensive adoption for arches and also for culverts. One of the special types which has been adopted is that of a box culvert which has a concrete bottom. Since this bottom can be made so that it will withstand an upward transverse stress, it furnishes a broad foundation for the whole culvert, and thus entirely eliminates the necessity for extensive footing to the side walls of the culvert, such as are necessary in soft ground with an ordinary stone culvert. Another advantage is that the inside of the culvert may be made perfectly smooth and thus offer less resistance to the passage of water through it. As may be noticed from Fig. 101a, such a culvert is provided with flaring head walls, and sunken end walls, so that the water may not scour underneath the culvert, and other features common to other types. No attempt will here be made to discuss the design of reinforced concrete, except to say that all four sides of such a box culvert are designed to withstand a computed bursting pressure which tends to crush the flat sides inward. In Fig. 101a is shown one illustration of the many types of culverts which have been designed of reinforced concrete.

ARCH CULVERTS.

191. Influence of design on flow. The variations in the design of arch culverts have a very marked influence on the cost and efficiency. To combine the least cost with the greatest efficiency, due weight should be given to the following elements: (a) amount of masonry. (b) the simplicity of the constructive work, (c) the design of the wing walls, (d) the design of the junction of the wing walls with the barrel and faces of the arch, and (e) the safety and permanency of the construction. These elements are more or less antagonistic to each other, and the defects of most designs are due to a lack of proper proportion in the design of these opposing interests. The simplest construction (satisfying elements *b* and *e*) is the straight barrel arch



STANDARD ARCH CULVERT
8 FEET SPAN
NORFOLK & WESTERN R.R.
(1891)



(To face page 221.)



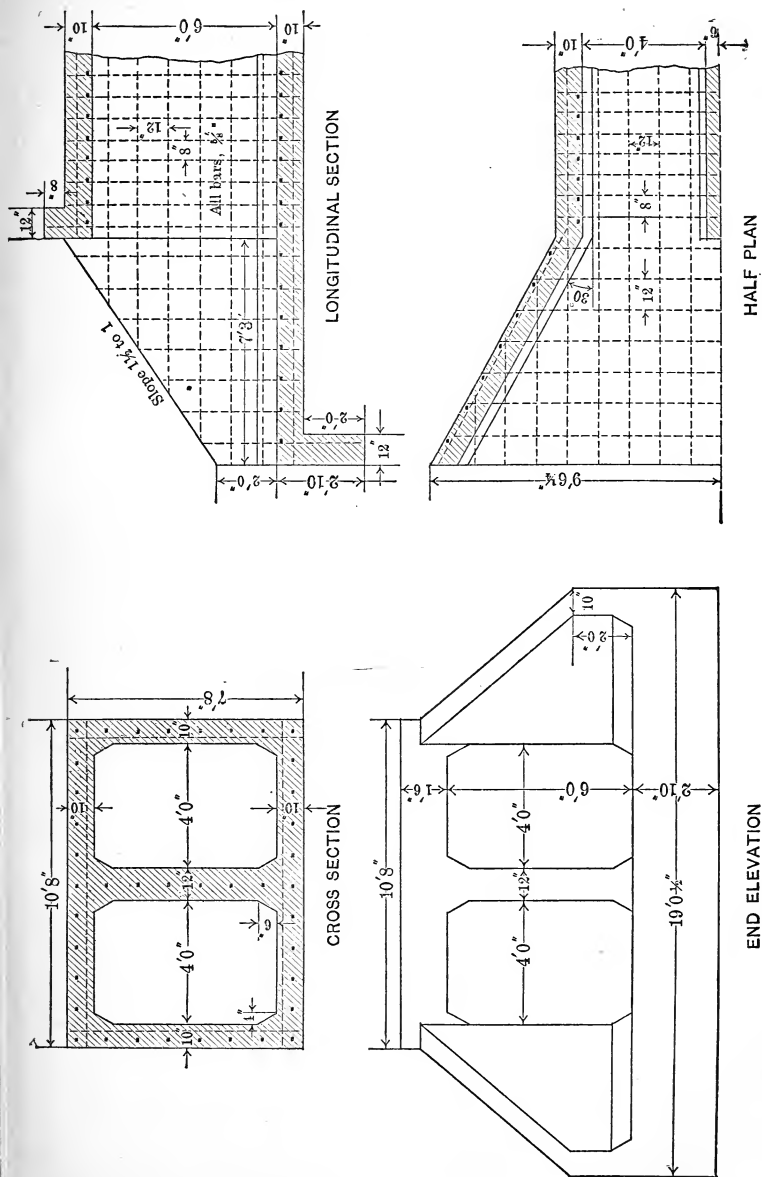


FIG. 101a.—REINFORCED CONCRETE BOX CULVERT.

between two parallel vertical head walls, as sketched in Fig. 102, *a*. From a hydraulic standpoint the design is poor, as the water eddies around the corners, causing a great resistance which decreases the flow. Fig. 102, *b*, shows a much better de-

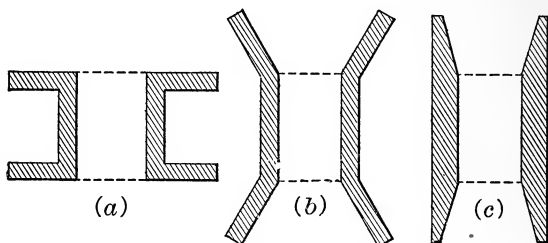


FIG. 102.—TYPES OF CULVERTS.

sign in many respects, but much depends on the details of the design as indicated in elements (*b*) and (*d*). As a general thing a good hydraulic design requires complicated and expensive masonry construction, i.e., elements (*b*) and (*d*) are opposed. Design 102, *c*, is sometimes inapplicable because the water is liable to work in behind the masonry during floods and perhaps cause scour. This design uses less masonry than (*a*) or (*b*).

192. Example of arch culvert design. In Plate IV is shown the design for an 8-foot arch culvert according to the standard of the Norfolk and Western R. R. Note that the plan uses the flaring wing walls (Fig. 102, *b*) on the up-stream side (thus protecting the abutments from scour) and straight wing walls (similar to Fig. 102, *c*) on the down-stream end. This economizes masonry and also simplifies the constructive work. Note also the simplicity of the junction of the wing walls with the barrel of the arch, there being no re-entrant angles below the springing line of the arch. The design here shown is but one of a set of designs for arches varying in span from 6' to 30'.

MINOR OPENINGS.

193. Cattle-guards. (*a*) Pit guards. Cattle-guards will be considered under the head of minor openings, since the old-fashioned plan of pit guards, which are even now defended and

preferred by some railroad men, requires a break in the continuity of the roadbed. A pit about three feet deep, five feet

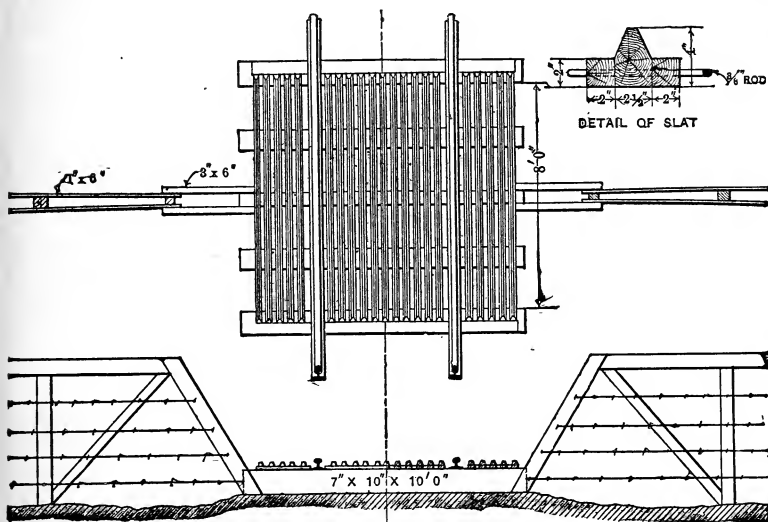


FIG. 103.—CATTLE-GUARD WITH WOODEN SLATS.

long, and as wide as the width of the roadbed, is walled up with stone (sometimes with wood), and the rails are supported on heavy timbers laid longitudinally with the rails. The break in the continuity of the roadbed produces a disturbance in the elastic wave running through the rails, the effect of which is noticeable at high velocities. The greatest objection, however, lies in the dangerous consequences of a derailment or a failure of the timbers owing to unobserved decay or destruction by fire—caused perhaps by sparks and cinders from passing locomotives. The very insignificance of the structure often leads to careless inspection. But if a single pair of wheels gets off the rails and drops into the pit, a costly wreck is inevitable.

(b) **Surface cattle-guards.** These are fastened on top of the ties; the continuity of the roadbed is absolutely unbroken and thus is avoided much of the danger of a bad wreck owing to a possible derailment. The device consists essentially of overlaying the ties (both inside and outside the rails) with a surface on

which cattle will not walk. The multitudinous designs for such a surface are variously effective in this respect. An objection,

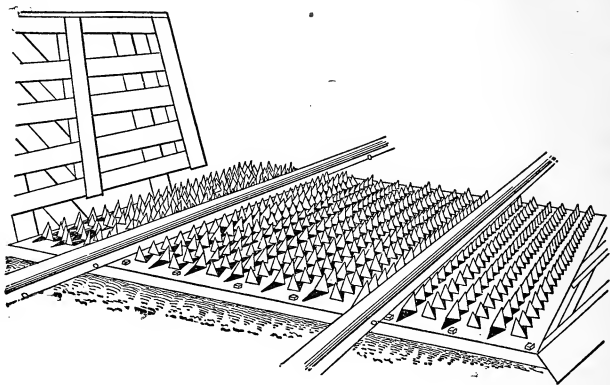


FIG. 104.—SHEFFIELD CATTLE-GUARD.

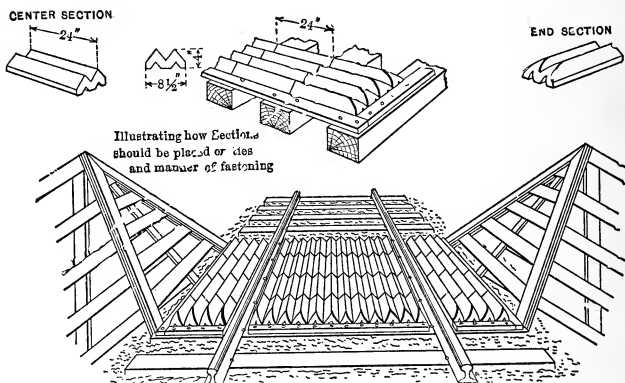


FIG. 105.—CLIMAX CATTLE-GUARD (TILE).

which is often urged indiscriminately against all such designs, is the liability that a brake-chain which may happen to be dragging *may* catch in the rough bars which are used. The bars

are sometimes "home-made," of wood, as shown in Fig. 103. Steel guards may be made as shown in Fig. 104. The general construction is the same as for the wooden bars. The metal bars have far greater durability, and it is claimed that they are more effective in discouraging cattle from attempting to cross.

194. Cattle-passes. Frequently when a railroad crosses a farm on an embankment, cutting the farm into two parts, the railroad company is obliged to agree to make a passageway through the embankment sufficient for the passage of cattle and perhaps even farm-wagons. If the embankment is high enough so that a stone arch is practicable, the initial cost is the only great objection to such a construction; but if an open wooden structure is necessary, all the objections against the old-fashioned cattle-guards apply with equal force here. The avoidance of a grade crossing which would otherwise be necessary is one of the great compensations for the expense of the construction and maintenance of these structures. The construction is sometimes made by placing two pile trestle bents about 6 to 8 feet apart, supporting the rails by stringers in the usual way, the special feature of this construction being that the embankments are filled in behind the trestle bents, and the thrust of the embankments is mutually taken up through the stringers, which are notched at the ends or otherwise constructed so that they may take up such a thrust. The designs for old-rail culverts and arch culverts are also utilized for cattle-passes when suitable and convenient, as well as the designs illustrated in the following section, and the reinforced concrete design of § 190a.

195. Standard stringer and I-beam bridges. The advantages of standard designs apply even to the covering of short spans with wooden stringers or with I beams—especially since the methods do not require much vertical space between the rails and the upper side of the clear opening, a feature which is often of prime importance. These designs are chiefly used for culverts or cattle-passes and for crossing *over* highways—providing such a narrow opening would be tolerated. The plans all imply stone abutments, or at least abutments of sufficient stability to withstand all thrust of the embankments. Some of the designs are illustrated in Plate V. The preparation of these standard designs should be attacked by the same general methods as already illustrated in § 156. When computing the required

transverse strength, due allowance should be made for lateral bracing, which should be amply provided for. Note particularly the methods of bracing illustrated in Plate V. The designs calling for iron (or steel) stringers may be classed as permanent constructions, which are cheap, safe, easily inspected and maintained, and therefore a desirable method of construction.

TABLE IV

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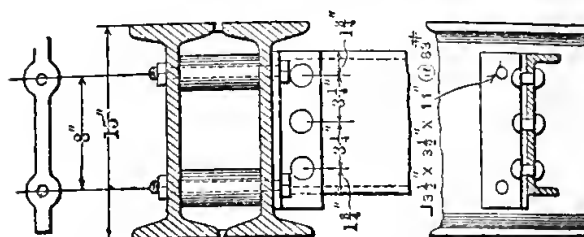
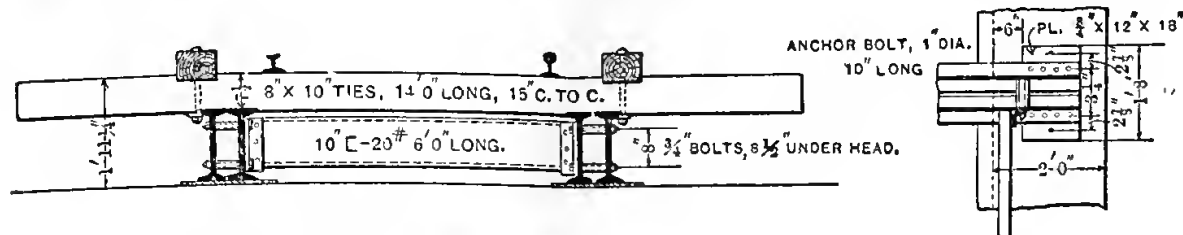
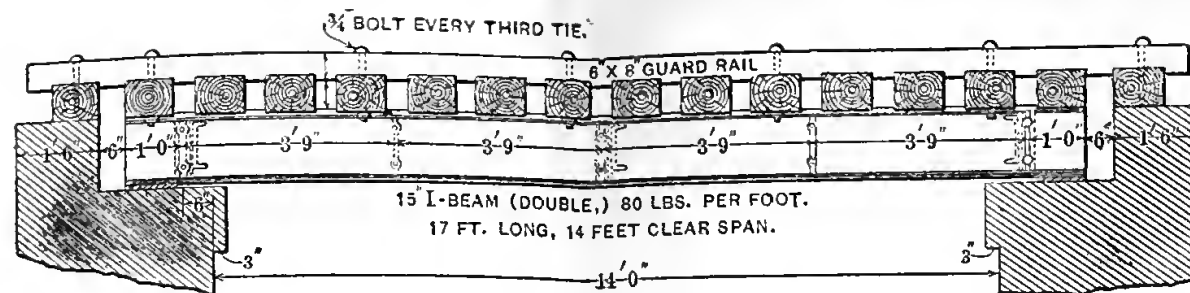
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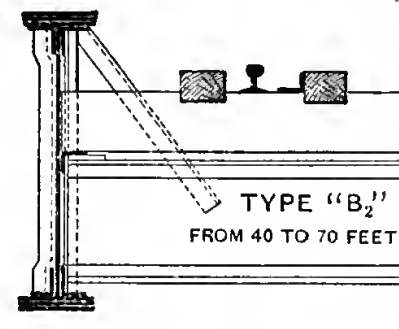
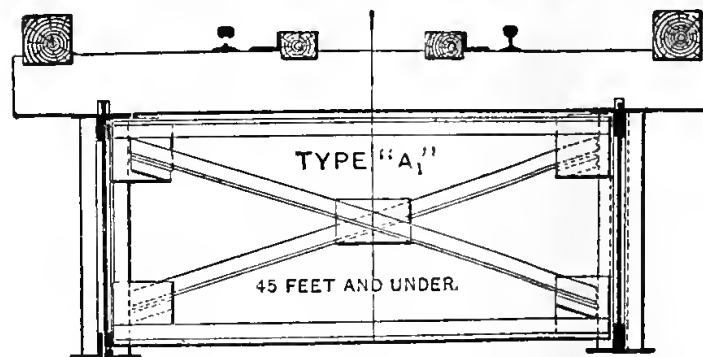
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PLATE V.

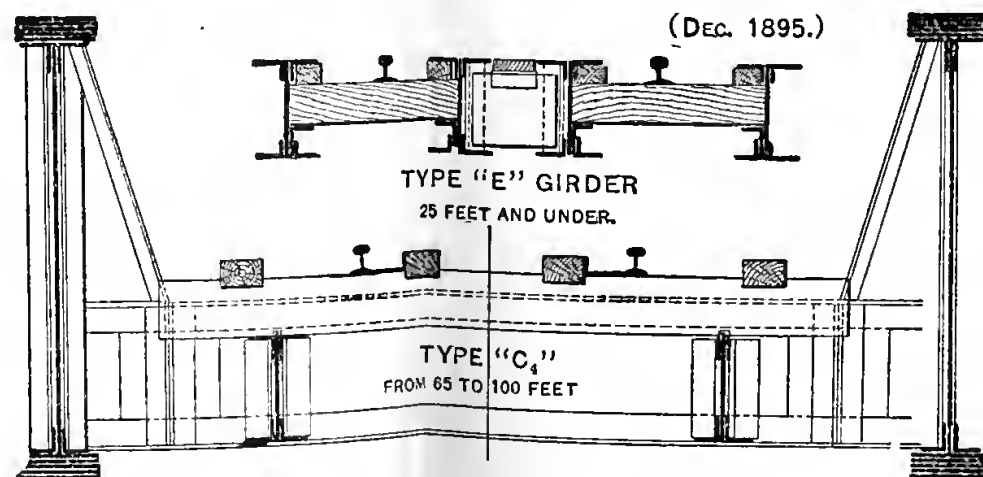


STANDARD I-BRIDGES—14-FT. SPAN.
NORFOLK AND WESTERN R.R. <
(1891.)



TYPES OF PLATE GIRDER BRIDGES.

C. M. & ST. P. RY.
(DEC. 1895.)



TYPE "C₄"

FROM 65 TO 100 FEET

(To face page 226.)



CHAPTER VII

BALLAST.

196. Purpose and requirements. "The object of the ballast is to transfer the applied load over a large surface; to hold the timber work in place horizontally; to carry off the rain-water from the superstructure and to prevent freezing up in winter; to afford means of keeping the ties truly up to the grade line; and to give elasticity to the roadbed." This extremely condensed statement is a description of an ideally perfect ballast. The value of any given kind of ballast is proportional to the extent to which it fulfills these requirements. The ideally perfect ballast is not necessarily the most economical ballast for all roads. Light traffic generally justifies something cheaper, but a very common error is to use a very cheap ballast when a small additional expenditure would procure a much better ballast, which would be much more economical in the long run.

197. Materials. The materials most commonly employed are gravel and broken stone. In many sections of the country other materials which more or less perfectly fulfill the requirements as given above, are used. The various materials including some of these special types have been defined by the American Railway Engineering and Maintenance of Way Association as follows:

DEFINITIONS.

Ballast. Selected material placed on the roadbed for the purpose of holding the track in line and surface.

Broken or crushed stone. Stone broken by artificial means into small fragments of specified sizes.

Burnt clay. A clay or gumbo which has been burned into material for ballast.

Chats. Tailings from mills in which zinc and lead ores are separated from the rocks in which they occur.

Chert. An impure flint or hornstone occurring in beds.

Cinders. The residue from the fuel used in locomotives and other furnaces.

Gravel. Small worn fragments of rock, coarser than sand, occurring in natural deposits.

Gumbo. A term commonly used for a peculiarly tenacious clay, containing no sand.

Sand. Any hard, granular, comminuted rock material, finer than gravel and coarser than dust.

Slag. The waste product, in a more or less vitrified form, of furnaces for reduction of ore. Usually the product of a blast-furnace.

There is still another classification which may or may not be considered as ballast. It is perhaps hardly correct to speak of the natural soils as ballast, yet many miles of cheap railways are "ballasted" with the natural soil, which is then called Mud ballast.

Broken or crushed stone. Rock ballast is generally specified to be that which may all be passed through a $1\frac{1}{2}$ inch (or 2 inch) ring, but which cannot pass through a $\frac{3}{4}$ -inch mesh. It is most easily handled with forks. This method also has the advantage that when it is being rehandled the fine chips which would interfere with effectual drainage will be screened out. Rock ballast is more expensive in first cost and is also more troublesome to handle, but in heavy traffic especially, the track will be kept in better surface and will require less work for maintenance after the ties have become thoroughly bedded.

Burnt clay. This material has been used in many sections of the country where broken stone or gravel are unobtainable except at a prohibitive cost, and where a suitable quality of clay is readily obtained. This clay should be of "gumbo" variety and contain no gravel. It is sometimes burnt in a kiln, or it is sometimes burnt by piling the clay in long heaps over a mass of fuel, the pile being formed in such a way that a temporary but effectual kiln is made. It is necessary that a clear, clean fuel shall be used and that the firing shall be done by a man who is experienced in maintaining such a fire until the burning is completed. Such ballast may be burned very hard and it will last from four to six years. The cost of

burning varies from 30 to 60 cents per cubic yard, according to the circumstances.

Chats. This is a form of ballast which is peculiar to Southwestern Missouri and Southeastern Kansas. When this material was first used it was obtained from the refuse piles of the mills which treated the zinc and lead ores mined in those regions. With the processes then employed the material was obtained in lumps as large as broken stone, and they were considered to be as valuable as broken stone for ballast. Improvements in the processes of treating the ores have resulted in making this by-product very much smaller grained and of less value as ballast, although it is still considered a desirable form of ballast where it may readily be obtained. It should be noted that it is classed with gravel and cinders in the forms of cross-section shown later.

Chert. This is a form of flint or hornstone which occurs in nodules of a size that is suitable for ballast, and is a very good type of ballast wherever it is found, but its occurrence is comparatively infrequent. It is classed with cemented gravel in the design of cross-sections of ballast.

Cinders. This is one of the most universal forms of ballast, since it is a by-product of every road which uses coal as fuel. The advantages consist in the fairly good drainage, the ease of handling and the cheapness—after the road is in operation. One of the greatest disadvantages is the fact that the cinders are readily reduced to dust, which in dry weather becomes very objectionable. Cinders are usually considered preferable to gravel in yards.

Gravel. This is one of the most common forms of good ballast. There are comparatively few railroads which cannot find, at some place along their line, a gravel pit which will afford a suitable supply of gravel for ballast. Sometimes it is unnecessary to screen it, but usually it is better to screen the gravel over a screen having a $\frac{1}{2}$ -inch mesh so as to screen out all the dirt and the finer stones.

Sand. Railroads which run along the coast are frequently ballasted merely with the sand obtained in the immediate neighborhood. One great advantage lies in the almost perfect drainage which is obtained.

Slag. When slag is readily obtainable it furnishes an excellent ballast which is free from dust and perfect in drainage

qualities. Slag is classified with crushed rock in the cross-sections shown below, but it should be noted that this only applies to the best qualities of slag, since its quality is quite variable.

Mud ballast. When the natural soil is gravelly so that rain will drain through it quickly, it will make a fair roadbed for light traffic, but for heavy traffic, and for the greater part of the length of most roads, the natural soil is a very poor material for ballast; for, no matter how suitable the soil might be along limited sections of the road, it would practically never happen that the soil would be uniformly good throughout the whole length of the road. Considering that a heavy rain will in one day spoil the results of weeks of patient "surfacing" with mud ballast, it is seldom economical to use "mud" if there is a gravel-bed or other source of ballast anywhere on the line of the road.

198. Cross-sections. The required depth of the cross-section to the sub-soil depends largely on the weight of the rolling stock which is to pass over the track. A careful examination of a roadbed to determine the changes which take place under the ties and also an examination of the track and ties during the passage of a heavy train shows that the heavy loads which are now common on railroad tracks force the tie into the ballast with the passage of every wheel load. The effect on the ballast is a greater or less amount of crushing of the ballast. Even the very hardest grades of broken stone are more or less crushed by grinding against each other during the passage of a train. The softer and weaker forms of ballast are ground up much more quickly. One result is the formation of a fine dust which interferes with the proper drainage of water through the ballast. A second result is the compression of the ballast immediately under the tie into the sub-soil. In a comparatively short time a hole is formed under the tie which acts virtually like a pump. With every rise and fall of the tie under each wheel load, the tie actually pumps the water from the surrounding ballast and sub-soil into these various holes. When the ballast is of such a character that the water does not drain through it easily, the water will settle in these holes long enough to seriously deteriorate the ties. When the track becomes so much out of line or level, or so loose that it needs to be tamped up, the process of tamping has practically the effect of deepen-

ing the amount of ballast immediately under the tie, while the sub-soil is forced up between the ties. A longitudinal section of the sub-soil of a track which has been frequently tamped generally has a saw-tooth appearance, and the sub-soil, instead of being a uniform line, has a high spot between each tie, while the ballast is considerably below its normal level immediately under the tie.

The variation in the traffic on railroads has caused the American Railway Engineering and Maintenance of Way Association to divide railroads into three classes with respect to the standards of construction which should be adopted for ballasting, as well as other details of construction. The three classes are as follows (quoted from the Association Manual):

"Class 'A' shall include all districts of a railway having more than one main track, or those districts of a railway having a single main track with a traffic that equals or exceeds the following:

Freight-car mileage passing over districts per year per
mile 150,000

or,

Passenger-car mileage per annum per mile of district... 10,000

with maximum speed of passenger-trains of 50 miles per hour.

"Class 'B' shall include all districts of a railway having a single main track with a traffic that equals or exceeds the following:

Freight-car mileage passing over districts per year per
mile 50,000

or,

Passenger-car mileage per annum per mile of district ... 5,000

with maximum speed of passenger-trains of 40 miles per hour.

"Class 'C' shall include all districts of a railway not meeting the traffic requirements of Classes 'A' or 'B.'"

The classification was adopted on the consideration that *quality* of traffic as well as mere tonnage should determine the classification of a railroad. For example, it is considered that a road which operates a train at a speed of 50 miles an hour should adopt the first class or Class "A" standards, even though there is but one train per day on that railroad. It

likewise means that any road whose traffic makes necessary the construction of a regular double track should adopt the first-class specifications.

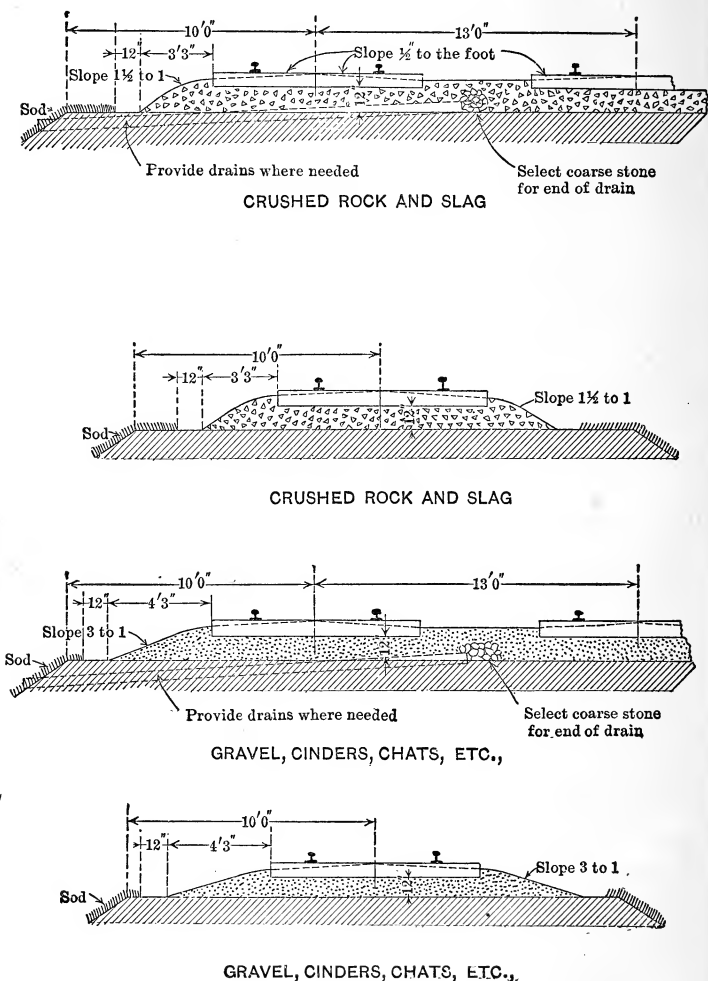


FIG. 106.—CROSS-SECTIONS OF BALLAST FOR CLASS "A" ROADS.

In Fig. 106 are shown a series of cross-sections which were recommended by that association for Class "A" traffic. It

should be noticed that in each case the cross-section of the roadbed from shoulder to shoulder of the roadbed is 20 feet plus the space between track centers for double track if any. The width of side ditches is merely added to that of the roadbed. The clear thickness of the ballast underneath the ties is made 12 inches, but even this should be considered as the *minimum* depth and is recommended for use only on the firmest, most substantial and well-drained subgrades. The slope of $\frac{1}{2}$ inch to the foot from the center of the track to the end of the tie, which is common to all the cross-sections, is designed with the idea of allowing a clear space of 1 inch underneath the rail. The ballast is then rounded off on a curve of 4 feet radius and finally reaches the subsoil on a slope which is $1\frac{1}{2}$:1 for broken stone, and 3:1 for all other materials. The flat slope adopted for gravel, etc., which adds considerably to the required width of roadbed, has been so designed in order that the considerable mass of material at the ends of the ties shall be better able to hold the track in place laterally. The sod on the embankment over the shoulder of the roadbed up to within 12 inches of the edge of the ballast is strongly recommended on account of the protection it affords to the shoulder of the roadbed. It should be noticed that the latest decision of that association regarding the form of subgrade is that the subgrade should be made level and not crowned, as suggested and discussed in § 63.

In Fig. 107 are shown a series of cross-sections for various classes of ballast for railroads that belong to Class "B." It may be noted that the thickness of the ballast under the tie is 9 inches for this class. The width of roadbed between the shoulders, recommended for Class "B" is 16 feet. As before, the width of the ditches is supposed to be added to this width. It should be noted that when using cementing gravel and chert the slope of 3:1 is made to begin at the bottom of the tie instead of at a point about 2 inches below the top of the tie. This is done in order to prevent water from accumulating around the end of the tie in a material which is less permeable than the other forms of ballast.

In Fig. 108 are shown two cross-sections for ballast for roads belonging to Class "C." On roads of this class it is assumed that crushed rock will not be used for ballast. The width of

roadbed between shoulders is 14 feet, while the depth of ballast underneath the tie is 6 inches.

It should be noticed that the above sections issued by the association do not include any cross-section which is recommended when no special ballast is used other than the natural

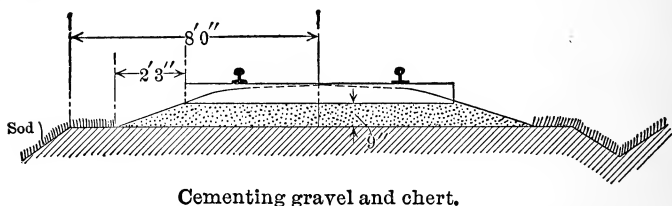
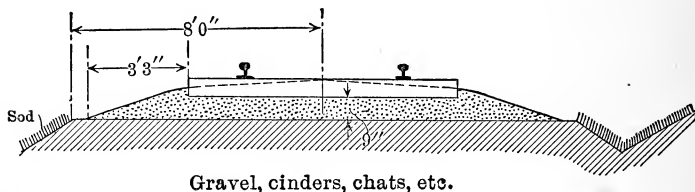
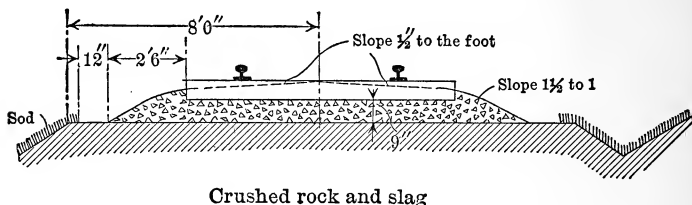


FIG. 107.—CROSS-SECTIONS OF BALLAST FOR CLASS "B" ROADS.

soil. In such a case a cross-section very similar to the sections shown for cementing gravel and chert should be used. The essential feature of such a section is that the soil, which is probably not readily permeable, should be kept away from the ends of the ties. Specifications for the placing of mud ballast, as well as other forms of ballast, have frequently specified that the ballast should be crowned about 1 inch above the level of the tops of the ties in the center of the track. This

feature of any cross-section, although proposed, was rejected by the association, in spite of the fact that when a tie is so imbedded it certainly will have a somewhat greater holding power in the ballast.

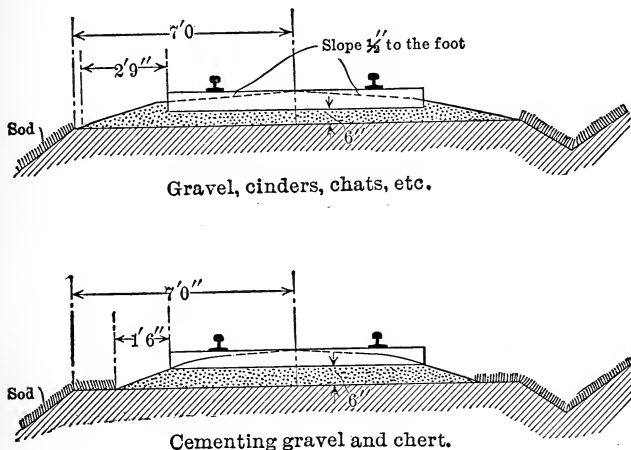


FIG. 108.—CROSS-SECTIONS OF BALLAST FOR CLASS "C" ROADS.

199. Methods of laying ballast. The cheapest method of laying ballast on new roads is to lay ties and rails directly on the prepared subgrade and run a construction train over the track to distribute the ballast. Then the track is lifted up until sufficient ballast is worked under the ties and the track is properly surfaced. This method, although cheap, is apt to injure the rails by causing bends and kinks, due to the passage of loaded construction trains when the ties are very unevenly and roughly supported, and the method is therefore condemned and prohibited in some specifications. The best method is to draw in carts (or on a contractor's temporary track) the ballast that is required under the level of the *bottom* of the ties. Spread this ballast carefully to the required surface. Then lay the ties and rails, which will then have a very fair surface and uniform support. A construction train can then be run on the rails and distribute sufficient additional ballast to pack around and between the ties and make the required cross-section.

The necessity for constructing some lines at an absolute

minimum of cost and of opening them for traffic as soon as possible has often led to the policy of starting traffic when there is little or no ballast—perhaps nothing more than a mere tamping of the natural soil under the ties. When this is done ballast may subsequently be drawn where required by the train-load on flat cars and unloaded at a minimum of cost by means of a “plough.” The plough has the same width as the cars and is guided either by a ridge along the center of each car or by short posts set up at the sides of the cars. It is drawn from one end of the train to the other by means of a cable. The cable is sometimes operated by means of a small hoisting-engine carried on a car at one end of the train. Sometimes the locomotive is detached temporarily from the train and is run ahead with the cable attached to it.

200. Cost. The cost of ballast *in the track* is quite a variable item for different roads, since it depends (a) on the first cost of the material as it comes to the road, (b) on the distance from the source of supply to the place where it is used, and (c) on the method of handling. The first cost of cinder or slag is frequently insignificant. A gravel-pit may cost nothing except the price of a little additional land beyond the usual limits of the right of way. Broken stone will usually cost \$1 or more per cubic yard. If suitable stone is obtainable on the company's land, the cost of blasting and breaking should be somewhat less than this. The cost of hauling will depend on the distance hauled, and also, to a considerable extent, on the limitations on the operation of the train due to the necessity of keeping out of the way of regular trains. There is often a needless waste in this way. The “mud train” is considered a pariah and entitled to no rights whatever, regardless of the large daily cost of such a train and of the necessary gang of men. The cost of broken-stone ballast *in the track* is estimated at \$1.25 per cubic yard. The cost of gravel ballast is estimated at 60 c. per cubic yard in the track. The cost of placing and tamping gravel ballast is estimated at 20 c. to 24 c. per cubic yard, for cinders 12 c. to 15 c. per cubic yard. The cost of loading gravel on cars, using a steam-shovel, is estimated at 6 c. to 10 c. per cubic yard.*

* Report Roadmasters' Association, 1885.

CHAPTER VIII.

TIES,

AND OTHER FORMS OF RAIL SUPPORT.

201. Various methods of supporting rails. It is necessary that the rails shall be sufficiently supported and braced, so that the gauge shall be kept constant and that the rails shall not be subjected to excessive transverse stress. It is also preferable that the rail support shall be neither rigid (as if on solid rock) nor too yielding, but shall have a *uniform* elasticity throughout. These requirements are more or less fulfilled by the following methods.

(a) **Longitudinals.** Supporting the rails throughout their entire length. This method is very seldom used in this country except occasionally on bridges and in terminals when the longitudinals are supported on cross-ties. In § 224 will be described a system of rails, used to some extent in Europe, having such broad bases that they are self-supporting on the ballast and are only connected by tie-rods to maintain the gauge.

(b) **Cast-iron "bowls" or "pots."** These are castings resembling large inverted bowls or pots, having suitable chairs on top for holding and supporting the rails, and tied together with tie-rods. They will be described more fully later (§ 223).

(c) **Cross-ties of metal or wood.** These will be discussed in the following sections.

202. Economics of ties. The true cost of ties depends on the relative total cost of maintenance for long periods of time. The first cost of the ties delivered to the road is but one item in the economics of the question. Cheap ties require frequent renewals, which cost for the *labor* of each renewal practically the same whether the tie is of oak or of hemlock. Cheap ties make a poor roadbed which will require more track labor to keep even in tolerable condition. The roadbed will require to be disturbed so frequently on account of renewals that the ties never get an opportunity to get settled and to form a smooth roadbed for any length of time. Irregularity in width, thickness, or length of ties is especially detrimental in causing the ballast to act and wear unevenly. The life of ties has thus a more or less direct influence on the life of the rails, on the wear of rolling stock, and on the speed of trains. These last items are not so readily reducible to dollars and cents, but when it can be shown that the total cost, for a long period of time, of several renewals of

cheap ties, with all the extra track labor involved, is as great as or greater than that of a few renewals of durable ties, then there is no question as to the real economy. In the following discussions of the merits of untreated ties (either cheap or costly), chemically treated ties, or metal ties, the true question is therefore of the ultimate cost of maintaining any particular kind of ties for an indefinite period, the cost including the first cost of the ties, the labor of placing them and maintaining them to surface, and the somewhat uncertain (but not therefore non-existent) effect of frequent renewals on repairs of rolling stock, on possible speed, etc.

WOODEN TIES.

203. Choice of wood. This naturally depends, for any particular section of country, on the supply of wood which is most readily available. The woods most commonly used, especially in this country, are oak and pine, oak being the most durable and generally the most expensive. Redwood is used very extensively in California and proves to be extremely durable, so far as decay is concerned, but it is very soft and is much injured by "rail-cutting." This defect is being partly remedied by the use of tie-plates, as will be explained later. Cedar, chestnut, hemlock, and tamarack are frequently used in this country. In tropical countries very durable ties are frequently obtained from the hard woods peculiar to those countries. According to a bulletin of the U. S. Department of Agriculture (Forestry service, No. 124) the number and value of the cross-ties used by the steam and street railroads of the United States during the year 1906, was as follows:

Kind of wood.	Number of ties.	Per cent.	Total value.	Aver. value
Oaks	45,357,874	44.1	\$23,278,052	\$0.51
Southern pines	18,841,210	18.3	9,567,745	.51
Douglas fir	7,248,562	7.1	3,010,392	.42
Cedar	7,083,442	6.9	3,310,116	.47
Chestnut	6,588,975	6.4	2,995,942	.49
Cypress	5,104,496	5.0	1,862,135	.36
Western pine	3,969,605	3.9	1,698,027	.43
Tamarack	2,576,859	2.5	889,561	.35
Hemlock	2,058,198	2.0	582,968	.28
Redwood	1,248,629	1.2	536,172	.43
Lodgepole pine	554,738	0.5	210,818	.38
White pine	373,387	0.3	151,052	.40
All others	1,828,067	1.8	726,144	.40
Total	102,834,042	100.0	\$48,819,124	\$0.47

The limitations of timber supply have somewhat diminished the use of oak and increased the use of the softer woods in recent years.

204. Durability. The durability of ties depends on the climate; the drainage of the ballast; the volume, weight, and speed of the traffic; the curvature, if any; the use of tie-plates; the time of year of cutting the timber; the age of the timber and the degree of its seasoning before placing in the track; the nature of the soil in which the timber is grown; and, chiefly, on the species of wood employed. The variability in these items will account for the discrepancies in the reports on the life of various woods used for ties.

White oak is credited with a life of 5 to 12 years, depending principally on the traffic. It is both hard and durable, the hardness enabling it to withstand the cutting tendency of the rail-flanges, and the durability enabling it to resist decay. *Pine* and *redwood* resist decay very well, but are so soft that they are badly cut by the rail-flanges and do not hold the spikes very well, necessitating frequent respiking. Since the spikes must be driven within certain very limited areas on the face of each tie, it does not require many spike-holes to "spike-kill" the tie. On sharp curves, especially with heavy traffic, the wheel-flange pressure produces a side pressure on the rail tending to overturn it, which tendency is resisted by the spike, aided sometimes by rail-braces. Whenever the pressure becomes too great the spike will yield somewhat and will be slightly withdrawn. The resistance is then somewhat less and the spike is soon so loose that it must be redriven in a new hole. If this occurs very often, the tie may need to be replaced long before any decay has set in. When the traffic is very light, the wood very durable, and the climate favorable, ties have been known to last 25 years.

205. Dimensions. The usual dimensions for the best roads (standard gauge) are 8' to 9' long, 6" to 7" thick, and 8" to 10" wide on top and bottom (if they are hewed) or 8" to 9" wide if they are sawed. For cheap roads and light traffic the length is shortened sometimes to 7' and the cross-section also reduced. On the other hand a very few roads use ties 9' 6" long.

Two objections are urged against sawed ties: first, that the grain is torn by the saw, leaving a woolly surface which induces decay; and secondly, that, since timber is not perfectly straight-

grained, some of the fibers are cut obliquely, exposing their ends, which are thus liable to decay. The use of a "planer-saw" obviates the first difficulty. Chemical treatment of ties obviates both of these difficulties. Sawed ties are more convenient to handle, are a necessity on bridges and trestles, and it is even claimed, although against commonly received opinion, that actual trial has demonstrated that they are more durable than hewed ties.

206. Spacing. The spacing is usually 14 to 16 ties to a 30-foot rail. This number is sometimes reduced to 12 and even 10, and on the other hand occasionally increased to 18 or 20 by employing narrower ties. There is no economy in reducing the number of ties very much, since for any required stiffness of track it is more economical to increase the number of supports than to increase the weight of the rail. The decreasing cost of rails and the increasing cost of ties have materially changed the relation between number of ties and weight of rail to produce a given stiffness at minimum cost, but many roads have found it economical to employ a large number of ties rather than increase the weight of the rail. On the other hand there is a practical limit to the number that may be employed, on account of the necessary space between the ties that is required for proper tamping. This width is ordinarily about twice the width of the tie. At this rate, with light ties 6" wide and with 12" clear space, there would be 20 ties per 30-foot rail, or 3520 per mile. The smaller ties can generally be bought much cheaper (proportionately) than the larger sizes, and hence the economy.

Track instructions to foremen generally require that the spacing of ties shall *not* be uniform along the length of any rail. Since the joint is generally the weakest part of the rail structure, the joint requires more support than the center of the rail. Therefore the ties are placed with but 8" or 10" clear space between them at the joints, this applying to 3 or 4 ties at each joint; the remaining ties, required for each rail length, are equally spaced along the remaining distance.

207. Specifications. The specifications for ties are apt to include the items of size, kind of wood, and method of construction, besides other minor directions about time of cutting, seasoning, delivery, quality of timber, etc.

(a) **Size.** The particular size or sizes required will be somewhat as indicated in § 205.

(b) **Kind of wood.** When the kind or kinds of wood are specified, the most suitable kinds that are available in that section of country are usually required.

(c) **Method of construction.** It is generally specified that the ties shall be hewed on two sides; that the two faces thus made shall be parallel planes and that the bark shall be removed. It is sometimes required that the ends shall be sawed off square; that the timber shall be cut in the winter (when the sap is down); and that the ties shall be seasoned for six months. These last specifications are not required or lived up to as much as their importance deserves. It is sometimes required that the ties shall be delivered on the right of way, neatly piled in rows, the alternate rows at right angles, piled if possible on ground not lower than the rails and at least seven feet away from them, the lower row of ties resting on two ties which are themselves supported so as to be clear of the ground.

(d) **Quality of timber.** The usual specifications for sound timber are required, except that they are not so rigid as for a better class of timber work. The ties must be sound, reasonably straight-grained, and not very crooked—one test being that a line joining the center of one end with the center of the middle shall not pass outside of the other end. Splits or shakes, especially if severe, should cause rejection.

Specifications sometimes require that the ties shall be cut from single trees, making



FIG. 109.—METHODS OF CUTTING TIES.

what is known as “pole ties” and definitely condemning those which are cut or split from larger trunks, giving two “slab ties” or four “quarter ties” for each cross-section, as is illustrated in Fig. 109. Even if pole ties are better, their exclusive use means the rapid destruction of forests of young trees.

208. Regulations for laying and renewing ties. The regulations issued by railroad companies to their track foremen will generally include the following, in addition to directions regarding dimensions, spacing, and specifications given in §§ 204–207. When hewn ties of somewhat variable size are used, as is frequently the case, the largest and best are to be selected for use as joint ties. If the upper surface of a tie is found to be warped (contrary to the usual specifications) so that one or both rails do

not get a full bearing across the whole width of the tie, it must be adzed to a true surface along its whole length and not merely notched for a rail-seat. When respiking is necessary and spikes have been pulled out, the holes should be immediately plugged with "wooden spikes," which are supplied to the foreman for that express purpose, so as to fill up the holes and prevent the decay which would otherwise take place when the hole becomes filled with rain-water. Ties should always be laid at right angles to the rails and never obliquely. Minute regulations to prevent premature rejection and renewal of ties are frequently made. It is generally required that the requisitions for renewals shall be made by the actual count of the individual ties to be renewed instead of by any wholesale estimates. It is unwise to have ties of widely variable size, hardness, or durability adjacent to each other in the track, for the uniform elasticity, so necessary for smooth riding, will be unobtainable under those circumstances.

209. Cost of ties. When railroads can obtain ties cut by farmers from woodlands in the immediate neighborhood, the price will frequently be as low as 20 c. for the smaller sizes, running up to 50 c. for the larger sizes and better qualities, especially when the timber is not very plentiful. Sometimes if a railroad cannot procure suitable ties from its immediate neighborhood, it will find that adjacent railroads control all adjacent sources of supply for their own use and that ties can only be procured from a considerable distance, with a considerable added cost for transportation. First-class oak ties cost about 75 to 80 c. and frequently much more. Hemlock ties can generally be obtained for 35 c. or less.

PRESERVATIVE PROCESSES FOR WOODEN TIES.

210. General principle. Wood has a fibrous cellular structure, the cells being filled with sap or air. The woody fiber is but little subject to decay unless the sap undergoes fermentation. Preservative processes generally aim at removing as much of the water and sap as possible and filling up the pores of the wood with an antiseptic compound. The most common methods (except one) all agree in this general process and only differ in the method employed to get rid of the sap and in the antiseptic chemical with which the fibers are filled. One valuable feature of these processes lies in the fact that the softer cheaper woods

(such as hemlock and pine) are more readily treated than are the harder woods and yet will produce practically as good a tie as a treated hard-wood tie and a very much better tie than an untreated hard-wood tie. The various processes will be briefly described, taking up first the process which is fundamentally different from the others, viz., vulcanizing.

211. Vulcanizing. The process consists in heating the timber to a temperature of 300° to 500° F. in a cylinder, the air being under a pressure of 100 to 175 lbs. per square inch. By this process the albumen in the sap is coagulated, the water evaporated, and the pores are partially closed by the coagulation of the albumen. It is claimed that the heat sterilizes the wood and produces chemical changes in the wood which give it an antiseptic character. It was once very extensively used on the elevated lines of New York City, but the process has now been abandoned as unsatisfactory.

212. Creosoting.—This process consists in impregnating the wood with *wood-creosote* or with *dead oil of coal-tar*. *Wood-creosote* is one of the products of the destructive distillation of wood—usually long-leaf pine. *Dead oil of coal-tar* is a product of the distillation of coal-tar at a temperature between 480° and 760° F. It would require about 35 to 50 pounds of creosote to completely fill the pores of a cubic foot of wood. But it would be impossible to force such an amount into the wood, nor is it necessary or desirable. About 10 pounds per cubic foot, or about 35 pounds per tie, is all that is necessary. For piling placed in salt water about 18 to 20 pounds per cubic foot is used, and the timber is then perfectly protected against the ravages of the *teredo navalis*. To do the work, long cylinders, which may be opened at the ends, are necessary. Usually the timbers are run in and out on iron carriages running on rails fastened to braces on the inside of the cylinder. When the load has been run in, the ends of the cylinder are fastened on. The water and air in the pores of the wood are first drawn out by subjecting the wood alternately to steam-pressure and to the action of a vacuum-pump. This is continued for several hours. Then, after one of the vacuum periods, the cylinder is filled with creosote oil at a temperature of about 170° F. The pumps are kept at work until the pressure is about 80 to 100 pounds per square inch, and is maintained at this pressure from one to two hours according to the size of the timber. The oil is then

withdrawn, the cylinders opened, the train pulled out and another load made up in 40 to 60 minutes. The average time required for treating a load is about 18 or 20 hours, the absorption about 10 or 11 pounds of oil per cubic foot, and the cost (1894) from \$12.50 to \$14.50 per thousand feet B. M.

213. Burnettizing (chloride-of-zinc process). This process is very similar to the creosoting process except that the chemical is chloride of zinc, and that the chemical is not heated before use. The preliminary treatment of the wood to alternate vacuum and pressure is not continued for quite so long a period as in the creosoting process. Care must be taken, in using this process, that the ties are of as uniform quality as possible, for seasoned ties will absorb much more zinc chloride than unseasoned (in the same time), and the product will lack uniformity unless the seasoning is uniform. The A., T. & S. Fé R. R. has works of its own at which ties are treated by this process at a cost of about 25 c. per tie. The Southern Pacific R. R. also has works for burnettizing ties at a cost of 9.5 to 12 c per tie. The zinc-chloride solution used in these works contains only 1.7% of zinc chloride instead of over 3% as used in the Santa Fé works, which perhaps accounts partially for the great difference in cost per tie. One great objection to burnettized ties is the fact that the chemical is somewhat easily washed out, when the wood again becomes subject to decay. Another objection, which is more forcible with respect to timber subject to great stresses, as in trestles, than to ties, is the fact that when the solution of zinc chloride is made strong (over 3%) the timber is made very brittle and its strength is reduced. The reduction in strength has been shown by tests to amount to $\frac{1}{4}$ to $\frac{1}{10}$ of the ultimate strength, and that the elastic limit has been reduced by about $\frac{1}{4}$.

214. Kyanizing (bichloride-of-mercury or corrosive-sublimate process). This is a process of "steeping." It requires a much longer time than the previously described processes, but does not require such an expensive plant. Wooden tanks of sufficient size for the timber are all that is necessary. The corrosive sublimate is first made into a concentrated solution of one part of chemical to six parts of *hot* water. When used in the tanks this solution is weakened to 1 part in 100 or 150. The wood will absorb about 5 to 6.5 pounds of the bichloride per 100 cubic feet, or about one pound for each 4 to 6 ties. The timber is allowed to soak in the tanks for several days, the general rule

being about one day for each inch of least thickness and one day over—which means seven days for six-inch ties, or thirteen (to fifteen) days for 12" timber (least dimension). The process is somewhat objectionable on account of the chemical being such a virulent poison, workmen sometimes being sickened by the fumes arising from the tanks. On the Baden railway (Germany) kyanized ties last 20 to 30 years. On this railway the wood is always air-dried for two weeks after impregnation and before being used, which is thought to have an important effect on its durability. The solubility of the chemical and the liability of the chemical washing out and leaving the wood unprotected is an element of weakness in the method.

215. Wellhouse (or zinc-tannin) process. The last two methods described (as well as some others employing similar chemicals) are open to the objection that since the wood is impregnated with an aqueous solution, it is liable to be washed out very rapidly if the wood is placed under water, and will even disappear, although more slowly, under the action of moisture and rain. Several processes have been proposed or patented to prevent this. Many of them belong to one class, of which the Wellhouse process is a sample. By these processes the timber is successively subjected to the action of two chemicals, each individually soluble in water, and hence readily impregnating the timber, but the chemicals when brought in contact form insoluble compounds which cannot be washed out of the wood-cells. By the Wellhouse process, the wood is first impregnated with a solution of chloride of zinc and glue, and is then subjected to a bath of tannin under pressure. The glue and tannin combine to form an insoluble leathery compound in the cells, which will prevent the zinc chloride from being washed out. It is being used by the A., T. & S. Fé R. R., their works being located at Las Vegas, New Mexico, and also by the Union Pacific R. R. at their works at Laramie, Wyo. In 1897 Mr. J. M. Meade, a resident engineer on the A., T. & S. Fé, exhibited to the Roadmasters Association of America a piece of a tie treated by this process which had been taken from the tracks after nearly 13 years' service. The tie was selected at random, was taken out for the sole purpose of having a specimen, and was still in sound condition and capable of serving many years longer. The cost of the treatment was then quoted as 13 c. per tie.

It was claimed that the treatment trebled the life of the tie besides adding to its spike-holding power.

In spite of this apparently favorable showing, the process was abandoned on the A. T. & S. F. R. R. in 1898 on the ground that the results did not justify the added expense.

216. Cost of Treating. The cost of treating ties by the various methods has been estimated as follows *—assuming that the plant was of sufficient capacity to do the work economically: creosoting, 25 c. per tie; vulcanizing, 25 c. per tie; burnettizing (chloride of zinc), 8.25 c. per tie; kyanizing (steeping in corrosive sublimate), 14.6 c. per tie; Wellhouse process (chloride of zinc and tannin), 11.25 c. per tie. These estimates are only for the net cost at the works and do not include the cost of hauling the ties to and from the works, which may mean 5 to 10 c. per tie. Some of these processes have been installed on cars which are transported over the road and operated where most convenient. An estimate made in 1907 by Prof. Gellert Alleman on the cost of treating ties, each assumed to have a volume of 3 cubic feet, the cost “not including royalty on patents, profit, interest, or depreciation, all of which vary widely at the various plants,” is as follows:

Zinc chloride	16 cents
“ “ and creosote	27 “
Creosote, 10 pounds to the cubic foot . . .	55 “

The very great increase in these prices, especially for creosoting, is due to the enormous increase in late years in the consumption and in the price of creosote.

217. Economics of treated ties. The fact that treated ties are not universally adopted is due to the argument that the added life of the tie is not worth the extra cost. If ties can be bought for 25 c., and cost 25 c. for treatment, and the treatment only doubles their life, there is apparently but little gained except the work of placing the extra tie in the track, which is more or less offset by the interest on 25 c. for the life of the untreated tie, and the larger initial outlay makes a stronger impression on the mind than the computed ultimate economy. But when (utilizing some statistics from the Pittsburgh, Ft. Wayne &

* Bull. No. 9, U. S. Dept. of Agric., Div. of Forestry. App. No. 1, by Henry Flad.

Chicago Railroad) it is found that white oak ties laid in rock ballast had a life of 10.17 years, and that hemlock ties treated with the zinc-tannin process and laid in the same kind of ballast lasted 10.71 years, then the economy is far more apparent. Unfortunately no figures were given for the cost of these ties nor for the cost of the treatment; but if we assume that the white oak ties cost 75 c. and the hemlock ties 35 c. plus 20 c. for treatment, there is not only a saving of 20 c. on each tie, but also the advantage of the slightly longer life of the treated tie. In the above case the total life of the two kinds of ties is so nearly the same that we may make an approximation of their relative worth by merely comparing the initial cost; but usually it is necessary to compare the value of two ties one of which may cost more than the other, but will last considerably longer. The mathematical comparison of the real value of two ties under such conditions may be developed as follows: The real cost of a tie, or any other similar item of constructive work, is measured by the cost of perpetually maintaining that item in proper condition in the structure. It will be here assumed that the annual cost of the trackwork, which is assignable to the tie, is the same for all kinds of ties, although the difference probably lies in favor of the more expensive and most durable ties. By assuming this expense as constant, the remaining expense may be considered as that due to the cost of the new ties whenever necessary, plus the cost of placing them in the track. We also may combine these two items in one, and consider that the cost of placing a tie in the track, which we will assume at the constant value of 20 c. per tie, regardless of the kind of tie, is merely an item of 20 c. in the total cost of the tie. We will assume that T_1 is the present cost of a tie, the cost including the preservative treatment if any, and the cost of placing in the track. The tie is assumed to last n years. At the end of n years another tie is placed in the track, and, for lack of more precise knowledge, we will assume that this cost T_2 equals T_1 . The "present worth" of T_2 is the sum which, placed at compound interest, would equal T_2 at the end of n years, and is expressed by the quantity $\frac{T_2}{(1+r)^n}$, in which r equals the rate of interest. Similarly at the end of $2n$ years we must expend a sum T_3 to put in the third tie, and the present worth of the cost of that third tie is ex-

pressed by the fraction $\frac{T_3}{(1+r)^{2n}}$. We may similarly express the present worths of the cost of ties for that particular spot for an indefinite period. The sum of all these present worths is given by the sum of a converging series and equals (assuming that all the T 's are equal) $\frac{T \times (1+r)^n}{(1+r)^n - 1}$. But instead of laying aside a sum of money which will maintain a tie in that particular place in perpetuity, we may compute the annual sum which must be paid at the end of each year, which would be the equivalent. We will call that annual payment A , and then the present worths of all these items are as follows:

For the first payment $\frac{A}{(1+r)^1}$

For the second payment $\frac{A}{(1+r)^2}$

For the third payment $\frac{A}{(1+r)^3}$

For the n th payment $\frac{A}{(1+r)^n}$

After the next tie is put in place we have the present worths of the annual payments on the second tie, of which the first one would be

For the $(n+1)$ payment $\frac{A}{(1+r)^{(n+1)}}$

Similarly after x ties have been put in place the last payment for the x tie would have a present worth $\frac{A}{(1+r)^{nx}}$. The sum of all these present worths is represented by the sum of a converging series and equals the very simple expression $\frac{A}{r}$.

But since the sum of the present worths of these annual payments must equal the sum of the present worths of the payments made at intervals of n years, we may place these two summations equal to each other, and say that

$$A = \frac{r \times T \times (1+r)^n}{(1+r)^n - 1}.$$

Values of A for various costs of a tie T on the basis that r equals 5% have been computed and placed in Table XXXIV. To illustrate the use of this table, assume that we are comparing the relative values of two ties, both untreated, one of them a white oak tie which will cost, say 75 c., and will last twelve years, the other a yellow pine tie which will cost, say 35 c., and will last six years. Assuming a charge for each case of 20 c. for placing the tie in the track, we have as the annual charge against the white oak tie, which costs 95 c. in the track, 10.72 c. The pine tie, costing 55 c. in the track and lasting six years, will be charged with an annual cost of 10.48 c., which shows that the costs are practically equal. It is probably true that the track work for maintaining the white oak would be less than that for the pine tie, but since the initial cost of the pine tie is less than that of the oak tie, it would probably be preferred in this case, especially if money was difficult to obtain. It may be interesting to note that if a comparison is made from a similar table which is computed on the basis of compounding the money at 4% instead of 5%, the annual charges would be 10.13 and 10.49 c. for the oak and pine ties respectively, thus showing that when money is "easier" the higher priced tie has the greater advantage.

EXAMPLE 2. Considering again the comparison previously made of a white oak untreated tie which was assumed to cost 75 c., and a hemlock treated tie, which cost 35 c. for the tie and 20 c. for the treatment, the total costs of these ties laid in the track would therefore be 95 c. and 75 c. respectively. These ties had practically the same life (10.17 and 10.71 years), but in order to use the table, we will call it ten years for each tie. The annual charge against the oak tie would therefore be 12.30 c., while that against the hemlock tie would be 9.72 c. This gives an advantage in the use of the treated tie of 2.58 c. per year, which capitalized at 5% would have a capitalized value of 51.6 c.

The Atchison, Topeka and Santa Fé R. R. has compiled a record of treated pine ties removed in 1897, '98, '99, and 1900, showing that the *average* life of the ties removed had been about 11 years. On the Chicago, Rock Island and Pacific R. R., the average life of a very large number of treated hemlock and tamarack ties was found to be 10.57 years. Of one lot of 21,850 ties, 12% still remained in the track after 15 years' exposure.

It has been demonstrated that much depends on the minor

details of the process—whatever it may be. As an illustration, an examination of a batch of ties, treated by the zinc-reosote process, showed 84% in service after 13 years' exposure; another batch, treated by another contractor by the same process (nominally), showed 50% worthless after a service of six years.

METAL TIES.

218. Extent of use. In 1894 * there were nearly 35000 miles of "metal track" in various parts of the world. Of this total, there were 3645 miles of "longitudinals" (see § 224), found exclusively in Europe, nearly all of it being in Germany. There were over 12000 miles of "bowls and plates" (see § 223), found almost entirely in British India and in the Argentine Republic. The remainder, over 18000 miles, was laid with metal cross-ties of various designs. There were over 8000 miles of metal cross-ties in Germany alone, about 1500 miles in the rest of Europe, over 6000 miles in British India, nearly 1000 miles in the rest of Asia, and about 1500 miles more in various other parts of the world. Several railroads in this country have tried various designs of these ties, but their use has never passed the experimental stage. These 35000 miles represent about 9% of the total railroad mileage of the world—nearly 400000 miles. They represent about 17.6% of the total railroad mileage, exclusive of the United States and Canada, where they are not used at all, except experimentally.† In the four years from 1890 to 1894 the use of metal track increased from less than 25000 miles to nearly 35000 miles. This increase was practically equal to the total increase in railroad mileage during that time, exclusive of the increase in the United States and Canada. This indicates a large growth in the percentage of metal track to total mileage, and therefore an increased appreciation of the advantages to be derived from their use.

219. Durability. The durability of metal ties is still far from being a settled question, due largely to the fact that the best form for such ties is not yet determined, and that a large part of the apparent failures in metal ties have been evidently due to defective design. Those in favor of them estimate the life as from 30 to 50 years. The opponents place it at not more

* Bulletin No. 9, U. S. Dept. of Agriculture, Div. of Forestry.

† See § 220 for a later development.

than 20 years, or perhaps as long as the best of wooden ties. Unlike the wooden tie, however, which deteriorates as much with time as with usage, the life of a metal tie is more largely a function of the traffic. The life of a well-designed metal tie has been estimated at 150000 to 200000 trains; for 20 trains per day, or say 6000 per year, this would mean from 25 to 33 years. 20 trains per day on a *single* track is a much larger number than will be found on the majority of railroads. Metal ties are found to be subject to rust, especially when in damp localities, such as tunnels; but on the other hand it is in such confined localities, where renewals are troublesome, that it is especially desirable to employ the best and longest-lived ties. Paint, tar, etc., have been tried as a protection against rust, but the efficacy of such protection is as yet uncertain, the conditions preventing any renewal of the protection—such as may be done by repainting a bridge, for example. Failures in metal cross-ties have been largely due to cracks which begin at a corner of one of the *square* holes which are generally *punched* through the tie, the holes being made for the bolts by which the rails are fastened to the tie. The holes are generally *punched* because it is cheaper. Reaming the holes after punching is thought to be a safeguard against this frequent cause of failure. Another method is to round the corners of the square punch with a radius of about $\frac{1}{8}$ ". If a crack has already started, the spread of the crack may be prevented by drilling a small hole at the end of it.

220. Form and dimensions of metal cross-ties. Since stability in the ballast is an essential quality for a tie, this must be accomplished either by turning down the end of the tie or by having some form of lug extending downward from one or more points of the tie. The ties are sometimes depressed in the center (see Plate VI, N. Y. C. & H. R. R. R. tie) to allow for a thick covering of ballast on top in order to increase its stability in the ballast. This form requires that the ties should be sufficiently well tamped to prevent a tendency to bend out straight, thus widening the gauge. Many designs of ties are objectionable because they cannot be placed in the track without disturbing adjacent ties. The failure of many metal cross-ties, otherwise of good design, may be ascribed to too light weight. Those weighing much less than 100 pounds have proved too light. From 100 to 130 pounds weight is being used satisfactorily on German railroads. The general outside dimensions are about

the same as for wooden ties, except as to thickness. The metal is generally from $\frac{1}{4}$ " to $\frac{3}{8}$ " thick. They are, of course, only made of wrought iron or steel, cast iron being used only for "bowls" or "plates" (see § 223). The details of construction for some of the most commonly used ties may be seen by a study of Plate VI.

The Carnegie tie is perhaps the only tie whose use on steam railroads in this country has passed the experimental stage. The Bessemer and Lake Erie R. R. in 1910 had 188 miles of track laid with these ties, and other roads are making extensive experiments. One practical difficulty, which is not of course insuperable, arises from the common practice of using the rails as parts of an electrical circuit for a block-signal system, which requires that the rails shall be insulated from each other. This requires that these metal ties shall be insulated from the rails. A method of insulation which is altogether satisfactory and inexpensive is yet to be determined. It is claimed that, on account of the better connection between the rail and the tie, there is less wear and more uniform wear to the rail. It is also claimed that there is greater lateral rigidity in the rails and ties (considered as a structure) and that this decreases the trackwork necessary to maintain alinement. These ties weigh 19.7 pounds per linear foot, or about 167 pounds for an 8 foot 6 inch tie. Even at the lowest possible price per pound the cost of the tie and its fastenings must be two or three times that of the best oak tie with spikes and even tie plates. It has been impossible to estimate the probable life of these ties. Until a reasonably close estimate of the life of steel ties can be determined, no proper comparison can be made of their economy relative to that of wooden ties. A study of Table XXXIV will show that a tie which costs, say three times as much as a cheap tie, must last *more* than three times as long in order that the annual charge against the tie shall be as low as that of the cheaper tie. For example, let us *assume* that the cost of a metal tie, laid in the track, is \$2.55 and that it will last 20 years. From Table XXXIV we may find that the annual charge against \$2.55 at 5% for 20 years = $(2 \times 8.02) + 4.41 = 20.45$ c. Compared with a tie costing 65 c., plus 20 c. for track laying, we find that the cheaper tie will only cost 19.63 c. per year even if it only lasts 5 years. Of course the *claimed* advantage of better track and less cost for track maintenance, using steel ties, will tend to offset, so far as it is true,

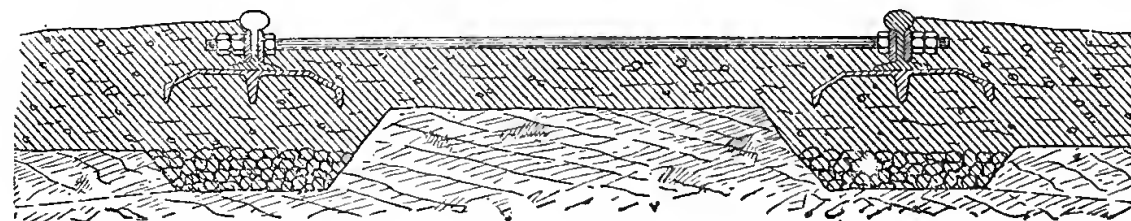
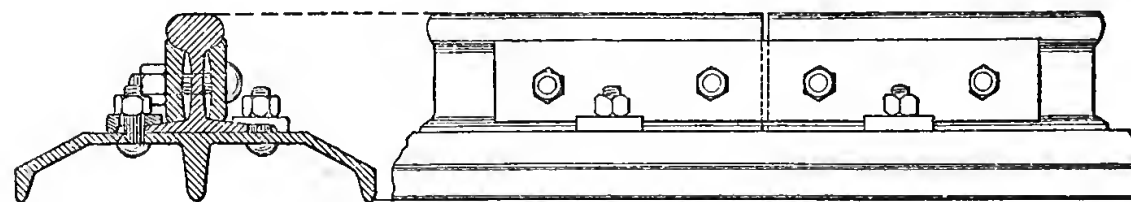
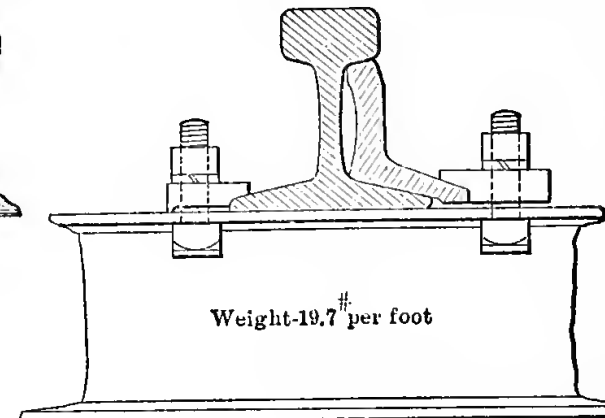
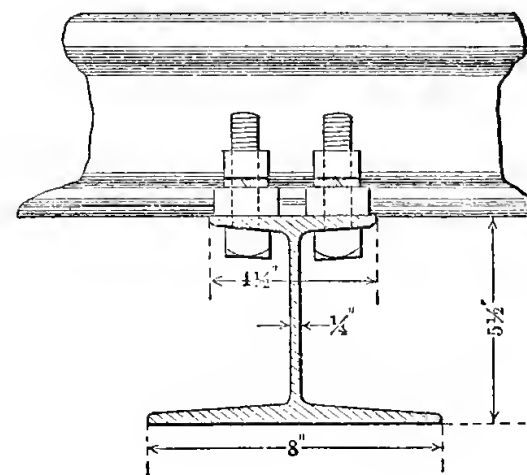
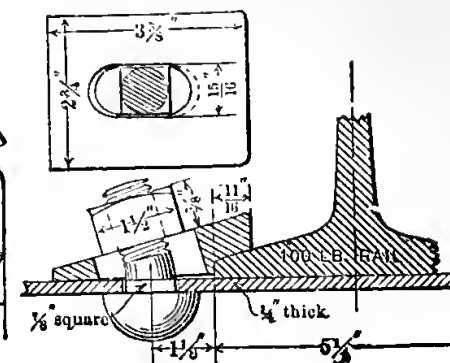
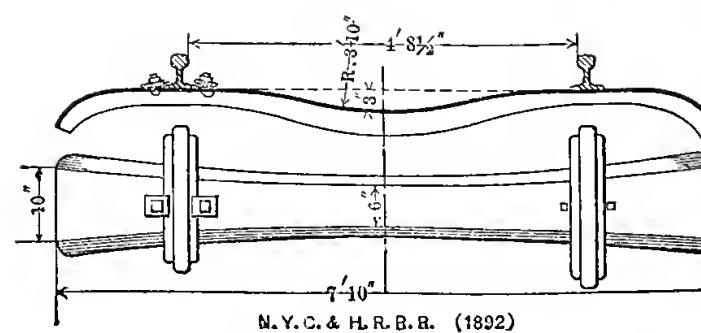
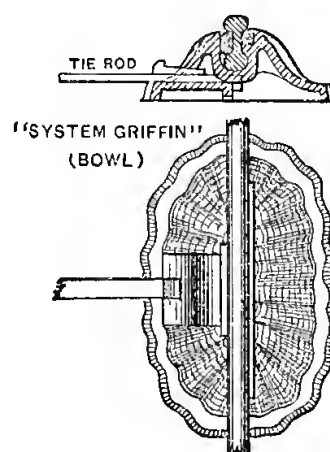
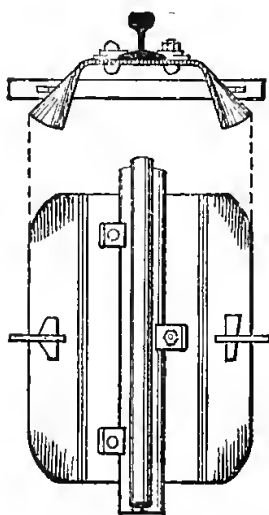
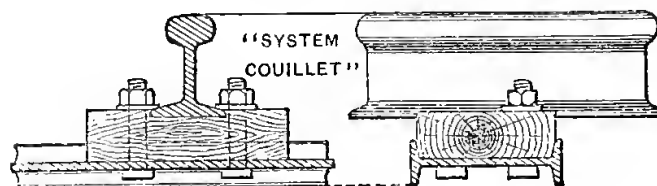
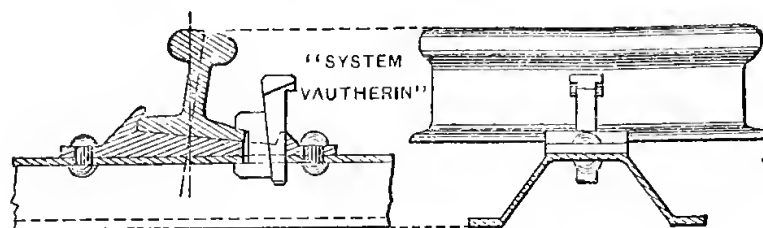
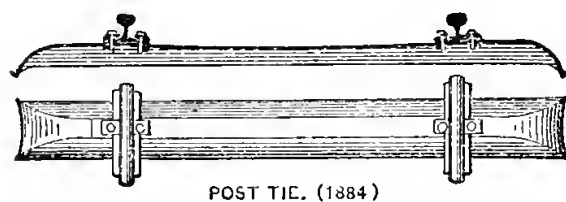
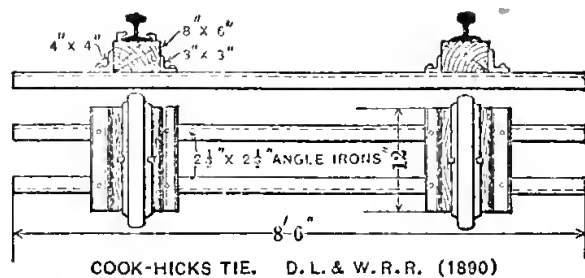


PLATE VI.—SOME FORMS OF METAL TIES.

(Between pp. 252 and 253.)

the disadvantage of the extra cost of the metal tie. Even if the extra work per tie amounts to only one-half hour for one man in a year, the cost of it, say 6 c., will utterly change the relative economics of the two ties.

221. Fastenings. The devices for fastening the rails to the ties should be such that the gauge may be widened if desired on curves, also that the gauge can be made true regardless of slight inaccuracies in the manufacture of the ties, and also that shims may be placed under the rail if necessary during cold weather when the tie is frozen into the ballast and cannot be easily disturbed. Some methods of fastening require that the base of the rail be placed against a lug which is riveted to the tie or which forms a part of it. This has the advantage of reducing the number of pieces, but is apt to have one or more of the disadvantages named above. Metal keys or wooden wedges are sometimes used, but the majority of designs employ some form of bolted clamp. The form adopted for the experimental ties used by the N. Y. C. & H. R. R. R. (see Plate VI) is especially ingenious in the method used to vary the gauge or allow for inaccuracies of manufacture. Plate VI shows some of the methods of fastening adopted on the principal types of ties.

222. Cost. The cost of metal cross-ties in Germany averages about 1.6 c. per pound or about \$1.60 for a 100-lb. tie. The ties manufactured for the N. Y. C. & H. R. R. R. in 1892 weighed about 100 lbs. and cost \$2.50 per tie, but if they had been made in larger quantities and with the present price of steel the cost would possibly have been much lower. The item of freight from the place of manufacture to the place where used is no inconsiderable item of cost with some roads. Metal cross-ties have been used by some street railroads in this country. Those used on the Terre Haute Street Railway weigh 60 pounds and cost about 66 c. for the tie, or 74 c. per tie with the fastenings.

223. Bowls or plates. As mentioned before, over 12000 miles of railway, chiefly in British India and in the Argentine Republic, are laid with this form of track. It consists essentially of large cast-iron inverted "bowls" laid at intervals under each rail and opposite each other, the opposite bowls being tied together with tie-rods. A suitable chair is riveted or bolted on to the top of each bowl so as to properly hold the rail. Being

made of cast iron, they are not so subject to corrosion as steel or wrought iron. They have the advantage that when old and worn out their scrap value is from 60% to 80% of their initial cost, while the scrap value of a steel or wrought-iron tie is practically nothing. Failure generally occurs from breakage, the failures from this cause in India being about 0.4% per annum. They weigh about 250 lbs. apiece and are therefore quite expensive in first cost and transportation charges. There are miles of them in India which have already lasted 25 years and are still in a serviceable condition. Some illustrations of this form of tie are shown in Plate VI.

224. Longitudinals.* This form, the use of which is confined almost exclusively to Germany, is being gradually replaced on many lines by metal cross-ties. The system generally consists of a compound rail of several parts, the upper bearing rail being very light and supported throughout its length by other rails, which are suitably tied together with tie-rods so as to maintain the proper gauge, and which have a sufficiently broad base to be properly supported in the ballast. One great objec-

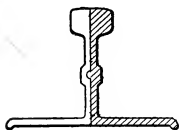


FIG. 110.

tion to this method of construction is the difficulty of obtaining proper drainage especially on grades, the drainage having a tendency to follow along the lines of the rails. The construction is much more complicated on sharp curves and at frogs and switches. Another fundamentally different form of

longitudinal is the Haarman compound "self-bearing" rail, having a base 12" wide and a height of 8", the alternate sections breaking joints so as to form a practically continuous rail.

Some of the other forms of longitudinals are illustrated in Plate VI.

For a very complete discussion of the subject of metal ties, see the "Report on the Substitution of Metal for Wood in Railroad Ties" by E. E. Russell Tratman, it being Bulletin No. 4, Forestry Division of the U. S. Dept. of Agriculture.

* Although the discussion of longitudinals might be considered to be long more properly to the subject of RAILS, yet the essential idea of all designs must necessarily be the support of a rail-head on which the rolling stock may run, and therefore this form, unused in this country, will be briefly described here.

224a. Reinforced Concrete Ties.—The wide application of reinforced concrete to various structural purposes, combined with its freedom from decay, has led to its attempted adoption for ties. At present (1908) their use is wholly in the experimental stage. In the annual Proceedings of the American Railway Engineering and Maintenance of Way Association for 1907 is a report on over a dozen different designs, the most of which were shown to be incapable of enduring traffic except on sidings. The ties are particularly subject to fracture if struck by a derailed car. A similar progress report, made in 1911, again indicated that a practicable concrete tie for general use has not yet been invented.

One of the most successful of these ties is the "Buhrer," which consists of one-fourth part of a thirty-foot scrap rail, which is inverted so that the base forms the seat of the running rails. This rail is imbedded in a mass of concrete whose form is somewhat like that of a huge "pole" tie. Several thousands of these are in use on various roads, but many of them have already required renewal and none of them have yet had time to show a service which would make them more economical than wooden ties.

CHAPTER IX.

RAILS.

225. Early forms. The first rails ever laid were wooden stringers which were used on very short tram-roads around coal-mines. As the necessity for a more durable rail increased, owing chiefly to the invention of the locomotive as a motive power, there were invented successively the cast-iron "fish-belly" rail and various forms of wrought-iron strap rails which finally developed into the T rail used in this country and the double-headed rail, supported by chairs, used so extensively in England. The cast-iron rails were cast in lengths of about 3 feet and were supported in iron chairs which were sometimes set upon stone piers. A great deal of the first railroad track of this country was laid with longitudinal stringers of wood placed upon cross-ties, the inner edge of the stringers being protected by wrought-iron straps. The "bridge" rails were first rolled in this country in 1844. The "pear" section was an approach to the present form, but was very defective on account of the difficulty of designing a good form of joint. The "Stevens" section was designed in 1830 by Col. Robert L. Stevens, Chief Engineer of the Camden and Amboy Railroad; although quite defective in its proportions, according to the present knowledge of the requirements, it is essentially the present form. In 1836, Charles Vignoles invented essentially the same form in England; this form is therefore known throughout England and Europe as the Vignoles rail.

226. Present standard forms. The larger part of modern railroad track is laid with rails which are either "T" rails or the double-headed or "bull-headed" rails which are carried in chairs. The double-headed rail was designed with a symmetrical form with the idea that after one head had been worn out by traffic the rail could be reversed, and that its life would be practically doubled. Experience has shown that the wear of the

rail in the chairs is very great; so much so that when one head has been worn out by traffic the whole rail is generally useless.

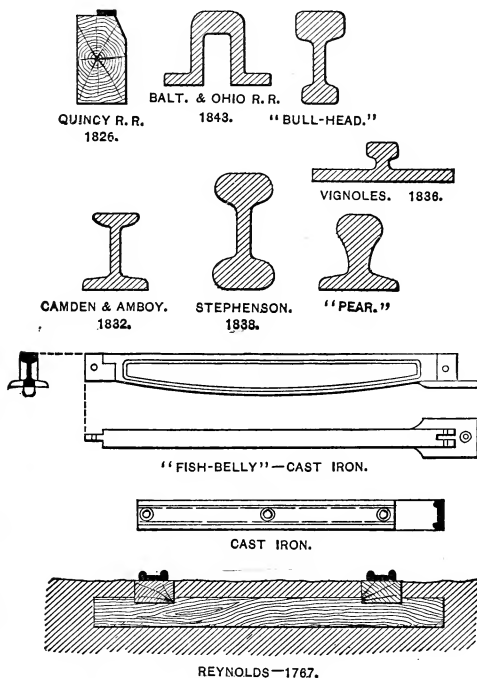


FIG. 111.—EARLY FORMS OF RAILS.

If the rail is turned over, the worn places, caused by the chairs, make a rough track and the rail appears to be more brittle and subject to fracture, possibly due to the crystallization that may have occurred during the previous usage and to the reversal of stresses in the fibers. Whatever the explanation, experience has

demonstrated the *fact*. The "bull-headed" rail has the lower head only large enough to properly hold the wooden keys with which the rail is secured to the chairs (see Fig. 112) and furnish the necessary strength. The use of these rails requires the use of two cast-iron chairs for each tie. It is claimed that such track is better for heavy and fast traffic, but it is more

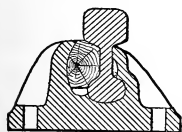


FIG. 112. — BULL-HEADED RAIL AND CHAIR.

such track is better for heavy and fast traffic, but it is more

expensive to build and maintain. It is the standard form of track in England and some parts of Europe.

Until a few years ago there was a very great multiplicity in the designs of "T" rails as used in this country, nearly every prominent railroad having its own special design, which perhaps differed from that of some other road by only a very minute and insignificant detail, but which nevertheless would require a complete new set of rolls for rolling. This certainly must have had a very appreciable effect on the cost of rails. In 1893, the American Society of Civil Engineers, after a very exhaustive investigation of the subject, extending over several years, having obtained the opinions of the best experts of the country, adopted a series of sections which have been very extensively adopted by the railroads of this country. Instead of having the rail sections for various weights to be geometrically similar figures, certain dimensions are made constant, regardless of the weight. It was decided that the metal should be distributed through the section in the proportions of—head 42%, web 21%, and flange 37%. The top of the head should have a radius of

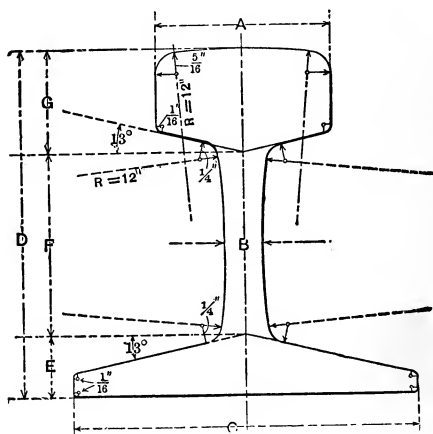


FIG. 113.—AM. SOC. C. E. STANDARD RAIL SECTION.

12"; the top corner radius of head should be $\frac{5}{16}$ "; the lower corner radius of head should be $\frac{1}{16}$ "; the corners of the flanges, $\frac{1}{16}$ " radius; side radius of web, 12"; top and bottom radii of web corners, $\frac{1}{4}$ "; and angles with the horizontal of the under side

of the head and the top of the flange, 13° . The sides of the head are vertical.

The height of the rail (D) and the width of the base (C) are always made equal to each other.

	Weight per Yard.												
	40	45	50	55	60	65	70	75	80	85	90	95	100
<i>A</i>	$1\frac{1}{2}"$	$2"$	$2\frac{1}{8}"$	$2\frac{1}{4}"$	$2\frac{3}{8}"$	$2\frac{3}{4}"$	$2\frac{1}{2}"$	$2\frac{3}{4}"$	$2\frac{1}{2}"$	$2\frac{1}{8}"$	$2\frac{1}{4}"$	$2\frac{1}{8}"$	$2\frac{1}{4}"$
<i>B</i>	$\frac{3}{4}"$	$\frac{7}{8}"$	$\frac{1}{2}"$	$\frac{3}{4}"$	$\frac{3}{4}"$	$\frac{1}{2}"$	$\frac{3}{4}"$	$\frac{3}{4}"$	$\frac{3}{4}"$	$\frac{1}{2}"$	$\frac{1}{2}"$	$\frac{1}{2}"$	$\frac{1}{2}"$
<i>C & D</i>	$3\frac{1}{2}"$	$3\frac{1}{8}"$	$3\frac{1}{4}"$	$4\frac{1}{8}"$	$4\frac{1}{2}"$	$4\frac{1}{8}"$	$4\frac{1}{2}"$	$4\frac{1}{8}"$	5	$5\frac{1}{8}"$	$5\frac{1}{4}"$	$5\frac{1}{8}"$	$5\frac{1}{4}"$
<i>E</i>	$\frac{1}{2}"$	$\frac{3}{4}"$	$\frac{1}{2}"$	$\frac{3}{4}"$	$\frac{3}{4}"$	$\frac{3}{4}"$	$\frac{1}{2}"$	$\frac{3}{4}"$	$\frac{1}{2}"$	$\frac{3}{4}"$	$\frac{3}{4}"$	$\frac{1}{2}"$	$\frac{3}{4}"$
<i>F</i>	$1\frac{5}{8}"$	$1\frac{3}{4}"$	$2\frac{1}{8}"$	$2\frac{1}{4}"$	$2\frac{1}{4}"$	$2\frac{3}{8}"$	$2\frac{3}{8}"$	$2\frac{3}{4}"$	$2\frac{1}{2}"$	$2\frac{1}{4}"$	$2\frac{5}{8}"$	$2\frac{3}{4}"$	$3\frac{1}{4}"$
<i>G</i>	$1\frac{1}{4}"$	$1\frac{1}{8}"$	$1\frac{1}{2}"$	$1\frac{3}{4}"$	$1\frac{3}{4}"$	$1\frac{3}{4}"$	$1\frac{3}{4}"$	$1\frac{3}{4}"$	$1\frac{1}{2}"$	$1\frac{3}{4}"$	$1\frac{3}{4}"$	$1\frac{3}{4}"$	$1\frac{3}{4}"$

The chief features of disagreement among railroad men relate to the radius of the upper corner of the head and the slope of the side of the head. The radius ($\frac{5}{16}"$) adopted for the upper corner (constant for all weights) is a little more than is advocated by those in favor of "sharp corners" who often use a radius of $\frac{1}{4}"$. On the other hand it is much less than is advocated by those

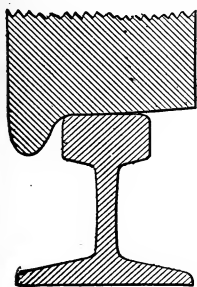


FIG. 114.—RELATION OF RAIL TO WHEEL-TREAD.

who consider that it should be nearly equal to (or even greater than) the larger radius universally adopted for the corner of the wheel-flange. The discussion turns on the relative rapidity of rail wear and the wear of the wheel-flanges as affected by the relation of the form of the wheel-tread to that of the rail. It is argued that sharp rail corners wear the wheel-flanges so as to produce sharp flanges, which are liable to cause derailment at switches and also to require that the tires of engine-drivers

must be more frequently turned down to their true form. On the other hand it is generally believed that rail wear is much less rapid while the area of contact between the rail and wheel-flange is small, and that when the rail has worn down, as it invariably does, to nearly the same form as the wheel-flange, the rail wears away very quickly.

227. Weight for various kinds of traffic. The heaviest rails in regular use weigh 100 lbs. per yard, and even these are only used on some of the heaviest traffic sections of such roads as the

N. Y. Central, the Pennsylvania, the N. Y., N. H. & H., and a few others. Probably the larger part of the mileage of the country is laid with 60- to 75-lb. rails—considering the fact that “the larger part of the mileage” consists of comparatively light-traffic roads and may exclude all the heavy trunk lines. Very light-traffic roads are sometimes laid with 56-lb. rails. Roads with fairly heavy traffic generally use 75- to 85-lb. rails, especially when grades are heavy and there is much and sharp curvature. The tendency on all roads is toward an increase in the weight, rendered necessary on account of the increase in the weight and capacity of rolling stock, and due also to the fact that the price of rails has been so reduced that it is both better and cheaper to obtain a more solid and durable track by increasing the weight of the rail rather than by attempting to support a weak rail by an excessive number of ties or by excessive track labor in tamping. It should be remembered that in buying rails the mere weight is, in one sense, of no importance. The important thing to consider is the **STRENGTH** and the **STIFFNESS**. If we assume that all weights of rails have *similar* cross-sections (which is nearly although not exactly true), then, since for beams of similar cross-sections the *strength* varies as the *cube* of the homologous dimensions and the *stiffness* as the *fourth power*, while the area (and therefore the weight per unit of length) only varies as the *square*, it follows that the stiffness varies as the square of the weight, and the strength as the $\frac{3}{2}$ power of the weight. Since for ordinary variations of weight the price per ton is the same, adding (say) 10% to the weight (and cost) adds 21% to the stiffness and over 15% to the strength. As another illustration, using an 80-lb. rail instead of a 75-lb. rail adds only $6\frac{2}{3}\%$ to the cost, but adds about 14% to the stiffness and nearly 11% to the strength. This shows why heavier rails are more economical and are being adopted even when they are not absolutely needed on account of heavier rolling stock. The stiffness, strength, and consequent durability are increased in a much greater ratio than the cost.

228. Effect of stiffness on traction. A very important but generally unconsidered feature of a stiff rail is its effect on tractive force. An extreme illustration of this principle is seen when a vehicle is drawn over a soft sandy road. The constant compression of the sand in front of the wheel has virtually the same effect on traction as drawing the wheel up a grade whose

steepness depends on the radius of the wheel and the depth of the rut. On the other hand, if a wheel, made of perfectly elastic material, is rolled over a surface which, while supported with absolute rigidity, is also perfectly elastic, there would be a forward component, caused by the expanding of the compressed metal just behind the center of contact, which would just balance the backward component. If the rail was supported throughout its length by an absolutely rigid support, the high elasticity of the wheel-tires and rails would reduce this form of resistance to an insignificant quantity, but the ballast and even the ties are comparatively inelastic. When a weak rail yields, the ballast is more or less compressed or displaced, and even though the elasticity of the rail brings it back to nearly its former place, the work done in compressing an inelastic material is wholly lost. The effect of this on the fuel account is certainly very considerable and yet is frequently entirely overlooked. It is practically impossible to compute the saving in tractive power, and therefore in cost of fuel, resulting from a given increase in the weight and stiffness of the rail, since the yielding of the rail is so dependent on the spacing of the ties, the tamping, etc. But it is not difficult to perceive in a general way that such an economy is possible and that it should not be neglected in considering the value of stiffness in rails. —

229. Length of rails. The recommended standard minimum length of rails is 33 feet. In recent years many roads have been trying 45-foot and even 60-foot rails. The argument in favor of longer rails is chiefly that of the reduction in track-joints, which are costly to construct and to maintain and are a fruitful source of accidents. Mr. Morrison of the Lehigh Valley R. R.* declares that, as a result of extensive experience with 45-foot rails on that road, he finds that they are much less expensive to handle, and that, being so long, they can be laid around sharp curves without being curved in a machine, as is necessary with the shorter rails. The great objection to longer rails lies in the difficulty in allowing for the expansion, which will require, in the coldest weather, an opening at the joint of nearly $\frac{3}{4}$ " for a 60-foot rail. The Pennsylvania R. R. and the Norfolk and Western R. R. each have a considerable mileage laid with 60-foot rails.

* Report, Roadmasters Association, 1895.

230. Expansion of rails. Steel expands at the rate of .0000065 of its length per degree Fahrenheit. The extreme range of temperature to which any rail will be subjected will be about 160° , or say from -20° F. to $+140^{\circ}$ F. With the above coefficient and a rail length of 60 feet the expansion would be 0.0624 foot, or about $\frac{3}{4}$ inch. But it is doubtful whether there would ever be such a range of motion even if there were such a range of temperature. Mr. A. Torrey, chief engineer of the Mich. Cent. R. R., experimented with a section over 500 feet long, which, although not a single rail, was made "continuous" by rigid splicing, and he found that there was no appreciable additional contraction of the rail at any temperature below $+20^{\circ}$ F. The reason is not clear, but the *fact* is undeniable.

The heavy girder rails, used by the street railroads of the country, are bonded together with perfectly tight rigid joints which do not permit expansion. If the rails are laid at a temperature of 60° F. and the temperature sinks to 0° , the rails have a *tendency* to contract .00039 of their length. If this tendency is resisted by the friction of the pavement in which the rails are buried, it only results in a tension amounting to .00039 of the modulus of elasticity, or say 10920 pounds per square inch, assuming 28 000 000 as the modulus of elasticity. This stress is not dangerous and may be permitted. If the temperature rises to 120° F., a tendency to expansion and buckling will take place, which will be resisted as before by the pavement, and a compression of 10920 pounds per square inch will be induced, which will likewise be harmless. The range of temperature of rails which are buried in pavement is much less than when they are entirely above the ground and will probably never reach the above extremes. Rails supported on ties which are only held in place by ballast must be allowed to expand and contract almost freely, as the ballast cannot be depended on to resist the distortion induced by any considerable range of temperature, especially on curves.

231. Rules for allowing for temperature. Track regulations generally require that the track foremen shall use iron (*not* wooden) shims for placing between the ends of the rails while splicing them. The thickness of these shims should vary with the temperature. Some roads use such approximate rules as the following: "The proper thickness for coldest weather is $\frac{5}{16}$ of an inch; during spring and fall use $\frac{1}{8}$ of an inch, and in the very

hottest weather $\frac{1}{16}$ of an inch should be allowed." This is on the basis of a 30-foot rail. When a more accurate adjustment than this is desired, it may be done by assuming some very high temperature (100° to 125° F.) as a maximum, when the joints should be *tight*; then compute in tabular form the spacing for each temperature, varying by 25° , allowing $0''.0643$ (very nearly $\frac{1}{16}''$) for each 25° change. Such a tabular form would be about as follows (rail length 33 feet):

Temperature . .	Over 100°	100° – 75°	75° – 50°	50° – 25°	25° – 0°	Below 0°
Rail opening . .	Close	$\frac{1}{16}''$	$\frac{1}{8}''$	$\frac{3}{16}''$	$\frac{1}{4}''$	$\frac{5}{16}'''$

One practical difficulty in the way of great refinement in this work is the determination of the real temperature of the rail when it is laid. A rail lying in the hot sun has a very much higher temperature than the air. The temperature of the rail cannot be obtained even by exposing a thermometer directly to the sun, although such a result might be the best that is easily obtainable. On a cloudy or rainy day the rail has practically the same temperature as the air; therefore on such days there need be no such trouble.

232. Chemical composition.—About 98 to 99.5% of the composition of steel rails is iron, but the value of the rail, as a rail, is almost wholly dependent upon the large number of other chemical elements which are, or may be, present in very small amounts. The manager of a steel-rail mill once declared that their aim was to produce rails having in them—

Carbon	0.32 to 0.40%
Silicon	0.04 to 0.06%
Phosphorus	0.09 to 0.105%
Manganese	1.00 to 1.50%

The analysis of 32 specimens of rails on the Chic., Mil. & St. Paul R. R. showed variations as follows:

Carbon	0.211 to 0.52%
Silicon	0.013 to 0.256%
Phosphorus	0.055 to 0.181%
Manganese	0.35 to 1.63%

These quantities have the same general relative proportions as the rail-mill standard given above, the differences lying mainly in the broadening of the limits. Increasing the percentage of carbon by even a few hundredths of one per cent makes the rail harder, but likewise more brittle. If a track is well ballasted and not subject to heaving by frost, a harder and more brittle rail may be used without excessive danger of breakage, and such a rail will wear much longer than a softer tougher rail, although the softer tougher rail may be the better rail for a road having a less perfect roadbed.

A small but objectionable percentage of sulphur is sometimes found in rails, and very delicate analysis will often show the presence, in very minute quantities, of several other chemical elements. The use of a very small quantity of nickel or aluminum has often been suggested as a means of producing a more durable rail. The added cost and the uncertainty of the amount of advantage to be gained has hitherto prevented the practical use or manufacture of such rails.

233. Proposed standard specifications for steel rails. The following specifications for steel rails are those proposed by a committee of the American Railway Engineering Association in March, 1910:

PROCESS OF MANUFACTURE.

1. The entire process of manufacture shall be in accordance with the best current state of the art.

(a) Ingots shall be kept in a vertical position until ready to be rolled, or until the metal in the interior has had time to solidify.

(b) Bled ingots shall not be used.

CHEMICAL COMPOSITION.

2. The chemical composition of the steel from which the rails are rolled shall be within the following limits:

	Bessemer.		Open-hearth.	
	80 lbs. and under.	85 to 100 lbs. inclusive.	80 lbs. and under.	85 to 100 lbs. inclusive.
Carbon	0.40 to 0.50	0.45 to 0.55	0.53 to 0.66	0.63 to 0.76
Manganese	0.80 to 1.10	0.85 to 1.15	0.75 to 1.00	0.75 to 1.00
Silicon	0.10 to 0.20	0.10 to 0.20	0.10 to 0.20	0.10 to 0.20
Phosphorus not to exceed	0.10	0.10	0.04	0.04
Sulphur not to exceed	0.075	0.075	0.06	0.06

3. When lower phosphorus can be secured in Bessemer or open-hearth steel, the carbon shall be increased at the rate of 0.035 for each 0.01 reduction in phosphorus.

The percentages of carbon, manganese, and silicon in an entire order of rails shall average as high as the mean percentages between the upper and lower limits.

SHEARING.

4. There shall be sheared from the end of the bloom formed from the top of the ingot, sufficient discard to insure sound rails. All metal from the top of the ingot, whether cut from the bloom or the rail, is the top discard.

SHRINKAGE.

5. The number and passes and speed of train shall be so regulated that, on leaving the rolls at the final pass, the temperature of the rails will not exceed that which requires a shrinkage allowance at the hot saws, for a 33-ft. rail of 100 lbs. section of $6\frac{1}{2}$ ins., and $\frac{1}{8}$ in. less for each 10 lbs. decrease of section, these allowances to be decreased at the rate of $\frac{1}{100}$ in. for each second. of time elapsed between the rail leaving the finishing rolls and being sawed. The bars shall not be held for the purpose of reducing their temperature, nor shall any artificial means of cooling them be used between the leading and finishing passes, nor after they leave the finishing pass.

SECTION.

6. The section of rail shall conform as accurately as possible to the templet furnished by the railroad company. A variation in height of $\frac{1}{64}$ in. less or $\frac{1}{32}$ in. greater than the specified height, and $\frac{1}{16}$ in. in width of flange, will be permitted; but no variations shall be allowed in the dimensions affecting the fit of splice bars.

WEIGHT.

7. The weight of the rail shall be maintained as nearly as possible, after complying with the preceding paragraph, to that specified in the contract.

A variation of one-half of one per cent from the calculated

weight of section, as applied to an entire order, will be allowed.

Rails will be accepted and paid for according to actual weight.

LENGTH.

8. The standard length of rail shall be 33 ft. Ten per cent of the entire order will be accepted in shorter lengths varying as follows: 30 ft., 28 ft., and 26 ft. A variation of $\frac{1}{4}$ in. from the specified length will be allowed.

All No. 1 rails less than 33 ft. shall be painted green on both ends.

STRAIGHTENING.

9. Care shall be taken in hot-straightening rails, and it shall result in their being left in such condition that they will not vary throughout their entire length more than four (4) ins. from a straight line in any direction when delivered to the cold-straightening presses. Those which vary beyond that amount, or have short kinks, shall be classed as second quality rails and be so marked. The distance between supports of rails in the straightening press shall not be less than forty-two (42) ins.; supports to have flat surfaces and out of wind. Rails shall be straight in line and surface and smooth on head when finished, final straightening being done while cold. They shall be sawed square at ends, variations to be not more than $\frac{1}{32}$ in., and prior to shipment shall have the burr caused by the saw cutting removed and the ends made clean.

DRILLING.

10. Circular holes for joint bolts shall be drilled in accordance with specifications of the purchaser. They shall in every respect conform accurately to drawing and dimensions furnished and shall be free from burrs.

BRANDING.

11. The name of the maker, the weight of the rail, and the month and year of manufacture shall be rolled in raised letters and figures on the side of the web. The number of the heat and a letter indicating the portion of the ingot from which the rail was made shall be plainly stamped on the web of each rail, where it will not be covered by the splice bars. Rails to be lettered

consecutively A, B, C, etc., the rail from the top of the ingot being A. In case of a top discard of twenty or more per cent the letter A will be omitted. Open-hearth rails to be branded "O. H."

DROP TESTS.

12. Drop tests shall be made on pieces of rail rolled from the top of the ingot, not less than four (4) ft. and not more than six (6) ft. long, from each heat of steel. These test pieces shall be cut from the rail bar next to either end of the top rail, as selected by the inspector.

The temperature of the test piece shall be between forty (40) and one hundred (100) degrees Fahrenheit.

The test pieces shall be placed head upward on solid supports, five (5) ins. top radius, three (3) ft. between centers, and subjected to impact tests, the tup falling free from the following heights:

60- and 70-lb. rail.....	16 ft.
80-, 85-, and 90-lb. rail.....	18 ft.
100-lb. rail.....	20 ft.

The test pieces which do not break under the first drop shall be nicked and tested to destruction.

DEFLECTION.

13. It is proposed to prescribe, under this head, the requirements in regard to deflection, fixing maximum and minimum limits, as soon as proper deflection limits have been decided on.

(a) Two pieces shall be tested from each heat of steel. If either of these test pieces breaks, a third piece shall be tested. If two of the test pieces break without showing physical defect, all rails of the heat will be rejected absolutely. If two of the test pieces do not break, all rails of the heat will be accepted as No. 1 or No. 2 classification, according as the deflection is less or more, respectively, than the prescribed limit.*

(b) If, however, any test piece broken under test "A" shows physical defect, the top rail from each ingot of that heat shall be rejected.

(c) Additional tests shall then be made of test pieces selected by the inspector from the top end of any second rails of the same

* This clause to be added when the deflection limits are specified.

heat. If two of out three of these second test pieces break, the remainder of the rails of the heat will also be rejected. If two out of three of these second test pieces do not break, the remainder of the rails of the heat will be accepted, provided they conform to the other requirements of these specifications, as No. 1 or No. 2 classification, according as the deflection is less or more, respectively, than the prescribed limit.*

(d) If any test piece, test "A," does not break, but when nicked and tested to destruction shows interior defect, the top rails from each ingot of that heat shall be rejected.

DROP-TESTING MACHINE.

14. The drop-testing machine shall be the standard of the American Railway Engineering and Maintenance of Way Association, and have a tup of 2000 lbs. weight, the striking face of which shall have a radius of five (5) ins.

The anvil block shall be adequately supported and shall weigh 20 000 lbs.

The supports shall be a part of or firmly secured to the anvil.

NO. 1 RAILS.

15. No. 1 rails shall be free from injurious defects and flaws of all kinds.

NO. 2 RAILS.

16. Rails which, by reason of surface imperfections, are not accepted as No. 1 rails, will be classed as No. 2 rails, but rails which in the judgment of the inspector contain physical defects which impair their strength, shall be rejected.

No. 2 rails to the extent of five (5) per cent of the whole order will be received. All rails accepted as No. 2 rails must have the ends painted white, and shall have two prick punch marks on the side of the web near the heat number near the end of the rail, so placed as not to be covered by the splice bars.

Rails improperly drilled or straightened, or from which the burrs have not been properly removed, shall be rejected, but may be accepted after being properly finished.

All classes of rails must be kept separate from each other and shipped in separate cars.

All rails must be loaded in the presence of the inspector.

* This clause to be added when the deflection limits are specified.

INSPECTION.

17. (a) Inspectors representing the purchaser shall have free entry to the works of the manufacturer at all times while the contract is being executed, and shall have all reasonable facilities afforded them by the manufacturer to satisfy them that the rails have been made in accordance with the terms of the specifications.

(b) For Bessemer steel the manufacturer shall, before the rails are shipped, furnish the inspector daily with carbon determinations for each heat, and two complete chemical analyses every twenty-four hours representing the average of the other elements contained in the steel, for each day and night turn. These analyses shall be made on drillings taken from small test ingots. The drillings for analyses shall be taken from the ladle test ingot at a distance of $\frac{1}{4}$ in. beneath the surface.

For open-hearth steel, the makers shall furnish the inspectors with the complete chemical analysis for each melt.

(c) On request of the inspector, the manufacturer shall furnish a portion of the test ingot for check analysis.

(d) All tests and inspections shall be made at the place of manufacture, prior to shipment, and shall be so conducted as not to unnecessarily interfere with the operation of the mill.

(e) Rails to be accepted must meet all of the requirements of the specifications.

234. Rail wear on tangents. When the wheel loads on a rail are abnormally heavy, and particularly when the rail has but little carbon and is unusually soft, the concentrated pressure on the rail is frequently greater than the elastic limit, and the metal "flows" so that the head, although greatly abraded, will spread somewhat outside of its original lines, as shown in Fig. 115. The rail wear that occurs on tangents is almost exclusively on top. Statistics show that the rate of rail wear on tangents decreases as the rails are more worn. Tests of a large number of rails on tangents have shown a rail wear averaging nearly one pound per yard per 10 000 000 tons of traffic. There is about 33 pounds of metal in one yard of the head of an 80-lb. rail. As an extreme value this may be worn down one-half, thus giving

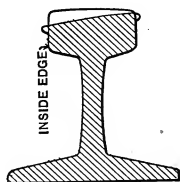


FIG. 115.

a tonnage of 165 000 000 tons for the life of the rail. Other estimates bring the tonnage down to 125 000 000 tons. Since the locomotive is considered to be responsible for one-half (and possibly more) of the damage done to the rail, it is found that the rate of wear on roads with shorter trains is more rapid in proportion to the tonnage, and it is therefore thought that the life of a rail should be expressed in terms of the number of trains. This has been estimated at 300 000 to 500 000 trains.

235. Rail wear on curves. On curves the maximum rail wear occurs on the inner side of the head of the outer rail, giving a worn form somewhat as shown in Fig. 116. The dotted line

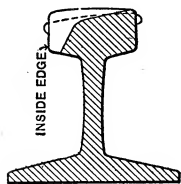


FIG. 116.

shows the nature and progress of the rail wear on the inner rail of a curve. Since the pressure on the outer rail is somewhat lateral rather than vertical, the "flow" does not take place to the same extent, if at all, on the outside, and whatever flow would take place on the inside is immediately worn off by the wheel-flange. Unlike the wear on tangents, the wear on curves is at a greater

rate as the rail becomes more worn.

The inside rail on curves wears chiefly on top, the same as on a tangent, except that the wear is much greater owing to the longitudinal slipping of the wheels on the rail, and the lateral slipping that must occur when a rigid four-wheeled truck is guided around a curve. The outside rail is subjected to a greater or less proportion of the longitudinal slipping, likewise to the lateral slipping, and, worst of all, to the grinding action of the flange of the wheel, which grinds off the side of the head.

The results of some very elaborate tests, made by Mr. A. M. Wellington, on the Atlantic and Great Western R. R., on the wear of rails, seem to show that the rail wear on curves may be expressed by the formula: "Total wear of rails on a d degree curve in pounds per yard per 10 000 000 tons duty $= 1 + 0.03d^2$."

"It is not pretended that this formula is strictly correct even in theory, but several theoretical considerations indicate that it may be nearly so." According to this formula the average rail wear on a 6° curve will be about twice the rail wear on a tangent. While this is approximately true, the various causes modifying the rate of rail wear (length of trains, age and quality of rails, etc.) will result in numerous and large variations from

the above formula, which should only be taken as indicating an approximate law.

236. Cost of rails. In 1873 the cost of steel rails was about \$120 per ton, and the cost of iron rails about \$70 per ton. Although the steel rails were at once recognized as superior to iron rails on account of more uniform wear, they were an expensive luxury. The manufacture of steel rails by the Bessemer process created a revolution in prices, and they steadily dropped in price until, several years ago, steel rails were manufactured and sold for \$22 per ton. For several years past the price has been very uniform at \$28 per ton at the mill. At such prices there is no longer any demand for iron rails, since the cost of manufacturing them is substantially the same as that of steel rails, while their durability is unquestionably inferior to that of steel rails. Rail quotations are generally on the basis of "long tons" of 2240 pounds.

The freight charge for transporting rails from the mill to the place where used is usually so large that it adds a very appreciable amount to the cost per ton. As an approximation, the freight may be estimated as 0.6 c. per ton-mile, or \$3.00 per ton for haul of 500 miles.

CHAPTER X.

RAIL-FASTENINGS.

RAIL-JOINTS.

237. Theoretical requirements for a perfect joint. A perfect rail-joint is one that has the *same strength* and *stiffness*—no more and no less—as the rails which it joins, and which will not interfere with the regular and uniform spacing of ties. It should also be reasonably cheap both in first cost and in cost of maintenance. Since the action of heavy loads on an elastic rail is to cause a wave of translation in front of each wheel, any change in the stiffness or elasticity of the rail structure will cause more or less of a shock, which must be taken up and resisted by the joint. The greater the change in stiffness the greater the shock, and the greater the destructive action of the shock. The perfect rail-joint must keep both rail-ends truly in line both laterally and vertically, so that the flange or tread of the wheel need not jump or change its direction of motion suddenly in passing from one rail to the other. A consideration of all the above requirements will show that only a perfect welding of rail-ends would produce a joint of uniform strength and stiffness which would give a uniform elastic wave ahead of each wheel. As welding is impracticable for ordinary railroad work (see § 230), some other contrivance is necessary which will approach this ideal as closely as may be.

238. Efficiency of the ordinary angle-bar. Throughout the middle portion of a rail the rail acts as a continuous girder. If we consider for simplicity that the ties are unyielding, the deflection of such a continuous girder between the ties will be but one-fourth of the deflection that would be found if the rail were cut half-way between the ties and an equal concentrated load were divided equally between the two unconnected ends. The maximum stress for the continuous girder would be but one-half of that in the cantilevers. Joining these ends with rail-joints will give the ordinary “suspended” joint. In order to main-

tain uniform strength and stiffness the angle-bars must supply the deficiency. These theoretical relations are modified to an unknown extent by the unknown and variable yielding of the ties. From some experiments made by the Association of Engineers of Maintenance of Way of the P. R. R.* the following deductions were made:

1. The capacity of a "suspended" joint is greater than that of a "supported" joint—whether supported on one or three ties. (See § 240)

2. That (with the particular patterns tested) the angle-bars alone can carry only 53 to 56% of a concentrated load placed on a joint.

3. That the capacity of the whole joint (angle-bars and rail) is only 52 4% of the strength of the unbroken rail.

4. That the ineffectiveness of the angle-bar is due chiefly to a deficiency in compressive resistance.

Although it has been universally recognized that the angle-bar is not a perfect form of joint, its simplicity, cheapness, and reliability have caused its almost universal adoption. Within a very few years other forms (to be described later) have been adopted on trial sections and have been more and more extended, until their present use is very large. These designs all agree in using metal below the base of the rail, as is shown in the several designs on Plate VII, but the general type shown in Fig. 119 is still (1912) in most common use.

239. Effect of rail gap at joints. It has been found that the jar at a joint is due almost entirely to the *deflection* of the joint and scarcely at all to the small gap required for expansion. This gap causes a drop equal to the versed sine of the arc having a chord equal to the gap and a radius equal to the radius of the wheel. Taking the extreme case (for a 30-foot rail) of a $\frac{3}{8}$ " gap and a 33" freight-car wheel, the drop is about $\frac{1}{1600}$ ". In order to test how much the jarring at a joint is due to a gap between the rails, the experiment was tried of cutting shallow notches in the top of an otherwise solid rail and running a locomotive and an inspection car over them. The resulting jarring was practically imperceptible and not comparable to the jar produced at joints. Notwithstanding this fact, many plans have

* Roadmasters Association of America—Reports for 1897.

been tried for avoiding this gap. The most of these plans consist essentially of some form of compound rail, the sections breaking joints. (Of course the design of the compound rail has also several other objects in view.) In Fig. 117 are shown a

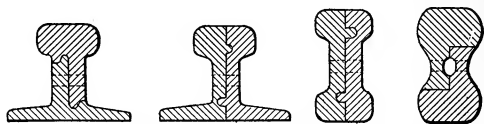


FIG. 117.—COMPOUND RAIL SECTIONS.

few of the very many designs which have been proposed. These designs have invariably been abandoned after trial. Another plan, which has been extensively tried on the Lehigh Valley R. R., is the use of mitered joints. The advantages gained by their use are as yet doubtful, while the added expense is unquestionable. The "Roadmasters Association of America" in 1895 adopted a resolution recommending mitered joints for double track, but their use has been abandoned.

240. "Supported," "suspended," and "bridge" joints. In a supported joint the ends of the rails are on a tie. If the angle-plates are short, the joint is entirely supported on one tie; if very long, it may be possible to place three ties under one angle-bar and thus the joint is virtually supported on three ties rather than one. In a suspended joint the ends of the rails are midway between two ties and the joint is supported by the two. There have always been advocates of both methods, but suspended joints are more generally used than supported joints. The opponents of three-tie joints claim that either the middle tie will be too strongly tamped, thus making it a supported joint, or that, if the middle tie is weakest, the joint becomes a very long (and therefore weak) suspended joint between the outer joint-ties, or that possibly one of the outer joint-ties gives way, thus breaking the angle-plate at the joint. Another objection which is urged is that unless the bars are very long (say 44 inches, as used on the Mich. Cent. R. R.) the ties are too close for proper tamping. The best answer to these objections is the successful use of these joints on several heavy-traffic roads.

"Bridge"-joints are similar to suspended joints in that the joint is supported on two ties, but there is the important difference that the bridge joint supports the rail from *underneath* and

there is no transverse stress in the rail, whereas the suspended joint requires the combined transverse strength of both angle-bars and rail. A serious objection to bridge-joints lies in the fact of their considerable thickness between the rail base and the tie. When joints are placed "staggered" (as is now the invariable standard practice), rather than "opposite," the ties supporting a bridge-joint must either be notched down, thus weakening the tie and promoting decay at the cut, or else the tie must be laid on a slope and the joint and the opposite rail do not get a fair bearing.

241. Failures of rail-joints. It has been observed on double-track roads that the maximum rail wear occurs a few inches beyond the rail gap at the joint in the direction of the traffic. On single-track roads the maximum rail wear is found a few inches *each* side of the joint rather than at the extreme ends of the rail, thus showing that the rail end deflects down under the wheel until (with fast trains especially) the wheel actually jumps the space and strikes the rail a few inches beyond the joint, the impact producing excessive wear. This action, which is called the "drop," is apt to cause the first tie beyond the joint to become depressed, and unless this tie is carefully watched and maintained at its proper level, the stresses in the angle-bar may actually become reversed and the bar may break at the top. The angle-bars of a suspended joint are normally in compression at the top. The mere reversal of the stresses would cause the bars

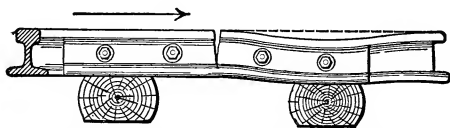


FIG. 118.—EFFECT OF "WHEEL DROP" (EXAGGERATED).

to give way with a less stress than if the stress were always the same in kind. A supported joint, and especially a three-tie joint (see § 240), is apt to be broken in the same manner.

242. Standard angle-bars. An angle-bar must be so made as to closely fit the rails. The great multiplicity in the designs of rails (referred to in Chapter IX) results in nearly as great variety in the detailed dimensions of the angle-bars. The sections here illustrated must be considered only as types of the variable forms necessary for each different shape of rail. The

absolutely essential features required for a fit are (1) the angles of the upper and lower surfaces of the bar where they fit against the rail, and (2) the height of the bar. The bolt-holes in the bar and rail must also correspond. The holes in the angle-plates are elongated or made oval, so that the track-bolts, which are

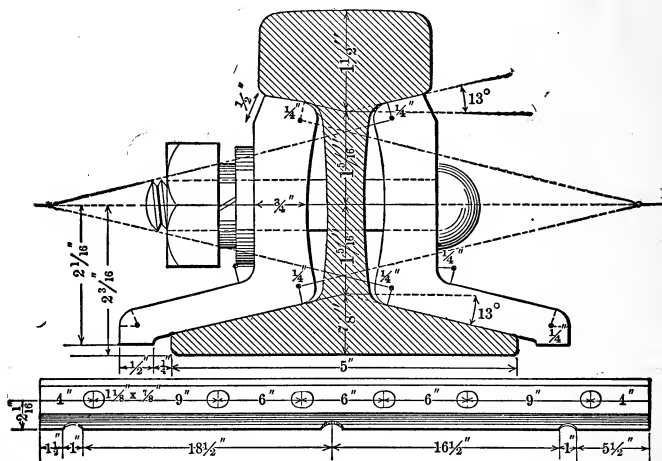


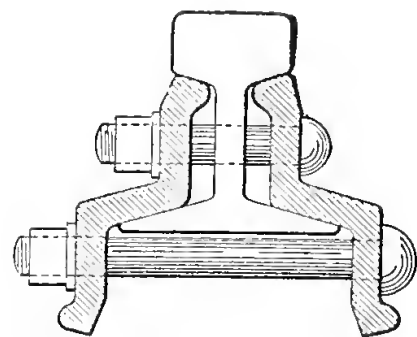
FIG. 119.—STANDARD ANGLE-BAR—80-LB. RAIL. M. C. R.R.

made of corresponding shape immediately under the head, will not be turned by jarring or vibration. The holes in the rails are made of larger diameter (by about $\frac{3}{16}$ ") than the bolts, so as to allow the rail to expand with temperature.

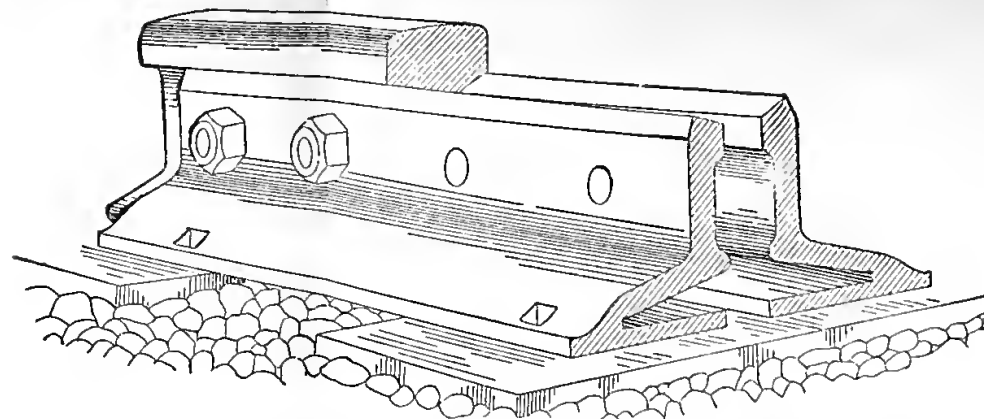
The standard drilling for bolt-holes in the end of a rail, as adopted by the A. R. E. & M. W. Assoc. in 1906, is as follows:

End of rail to center first hole	2 $\frac{11}{32}$ ins.
Center first hole to center second hole	5 "
Center second hole to center third hole (for six-hole joint)	5 "

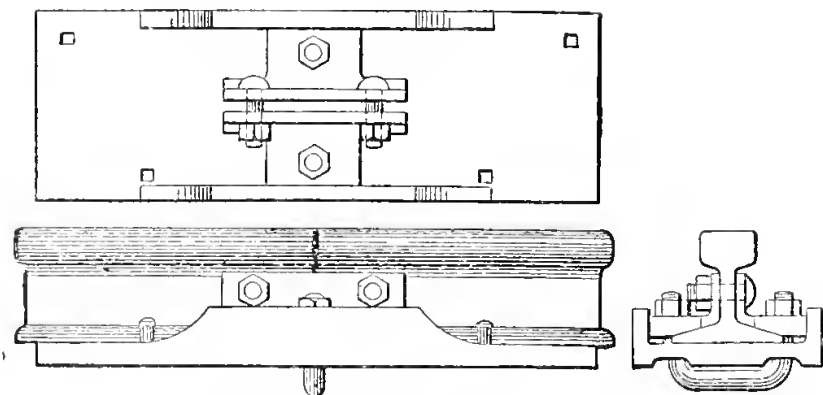
The proper length of angle-bars has not yet been standardized, but the above dimensions point very closely to the proper length. If the centers of the middle pair of holes in the angle-plate are made $4\frac{7}{8}$ inches apart, and the holes in the rails are $\frac{3}{16}$ inch larger in diameter than the bolts, it will allow for an extreme variation in the length of the rails of $\frac{3}{8}$ inch—due to expansion. Adding 4 inches at each end of the joint, from



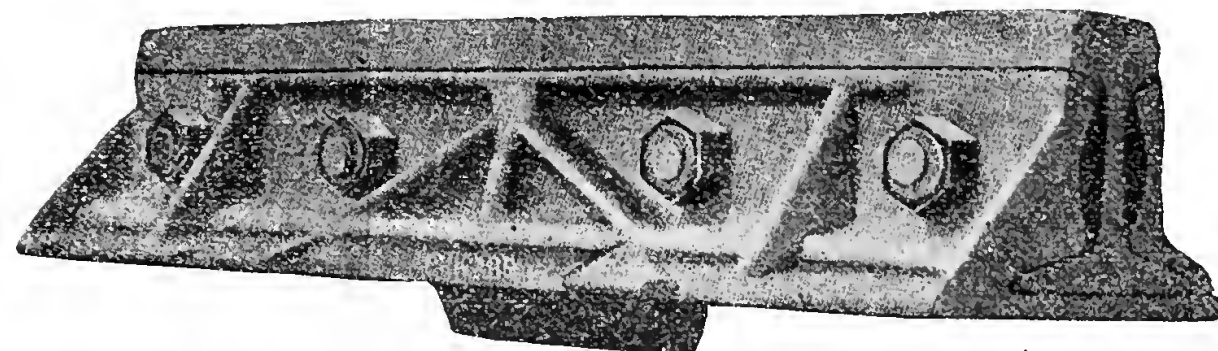
CLOUD JOINT.



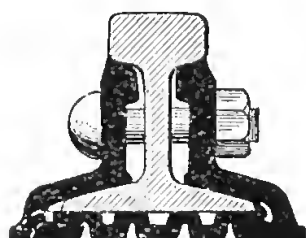
CONTINUOUS RAIL JOINT.



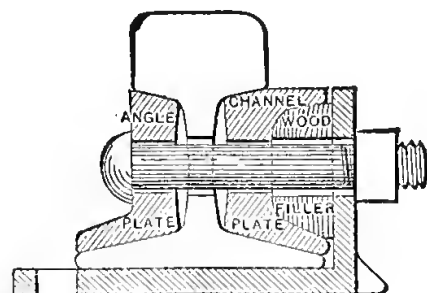
FISHER BRIDGE JOINT.



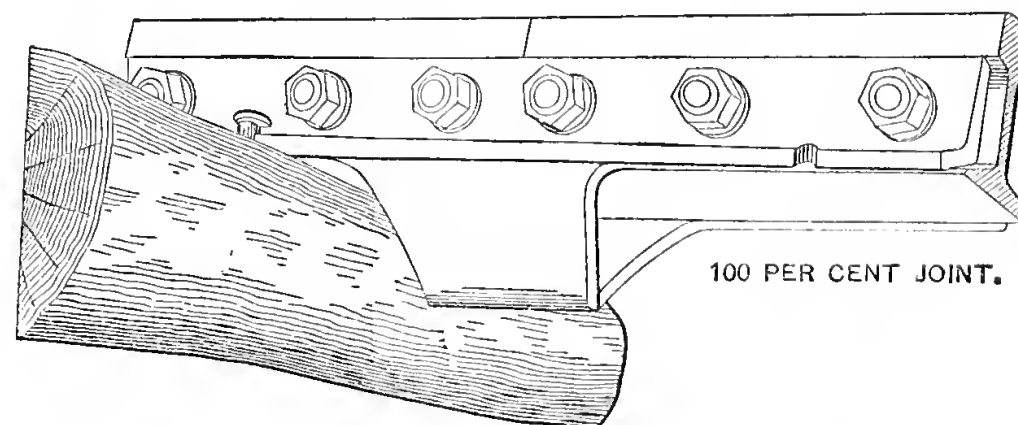
ATLAS SUSPENDED RAIL JOINT.



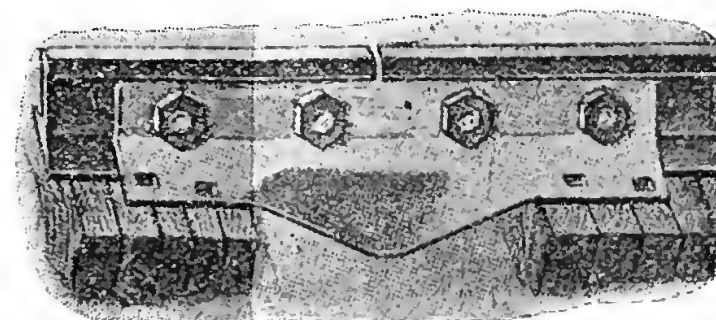
WOLHAUPTER JOINT



WEBER RAIL JOINT.



100 PER CENT JOINT.



BONZANO RAIL JOINT.

PLATE VII.—SOME FORMS OF RAIL JOINTS.
(Between pp. 274 and 275.)

the center of the last hole to the end of the angle-plate, will make a length of $22\frac{7}{8}$ inches for a four-hole and $32\frac{7}{8}$ inches for a six-hole joint. This is considerably less than the M. C. R. R. joint shown above, but this joint was purposely lengthened so that it could be used for a three-tie joint.

243. Later designs of rail-joints. In Plate VII are shown various designs which are competing for adoption. The most prominent of these (judging from the discussion in the convention of the Roadmasters Association of America in 1897) are the "Continuous" and the "Weber." Each of them has been very extensively adopted, and where used are universally preferred to angle-plates. Nearly all the later designs embody more or less directly the principle of the bridge-joint, i.e., support the rail from underneath. An experience of several years will be required to demonstrate which form of joint best satisfies the somewhat opposed requirements of minimum cost (both initial and for maintenance) and minimum wear of rails and rolling stock.

243a. Proposed specifications for steel splice-bars. The following specifications for steel splice-bars were proposed in 1900 by Committee No. 1, American Section, International Association for Testing Materials.

1. Steel for splice-bars may be made by the Bessemer or open-hearth process.

2. Steel for splice-bars shall conform to the following limits in chemical composition:

	Per cent.
Carbon shall not exceed.	0.15
Phosphorus shall not exceed.	0.10
Manganese.	0.30 to 0.60

3. Splice-bar steel shall conform to the following physical qualities:

Tensile strength, pounds per square inch.	54000 to 64000
Yield point, pounds per square inch.	32000
Elongation, per cent in eight inches shall not be less than.	25

4. (a) A test specimen cut from the head of the splice-bar shall bend 180° flat on itself without fracture on the outside of the bent portion.

(b) If preferred the bending test may be made on an unpunched splice-bar, which, if necessary, shall be first flattened and shall then be bent 180° flat on itself without fracture on the outside of the bent portion.

5. A test specimen of 8-inch gauged length, cut from the head of the splice-bar, shall be used to determine the physical properties specified in paragraph No. 3.

6. One tensile specimen shall be taken from the rolled splice-bars of each blow or melt, but in case this develops flaws, or breaks outside of the middle third of its gauged length, it may be discarded and another test specimen submitted therefor.

7. One test specimen cut from the head of the splice-bar shall be taken from a rolled bar of each blow or melt, or if preferred the bending test may be made on an unpunched splice-bar, which, if necessary, shall be flattened before testing. The bending test may be made by pressure or by blows.

8. For the purposes of this specification, the yield point shall be determined by the careful observation of the drop of the beam or halt in the gauge of the testing machine.

9. In order to determine if the material conforms to the chemical limitations prescribed in paragraph No. 2 herein, analysis shall be made of drillings taken from a small test ingot.

10. All splice-bars shall be smoothly rolled and true to templet. The bars shall be sheared accurately to length and free from fins or cracks, and shall perfectly fit the rails for which they are intended. The punching and notching shall accurately conform in every respect to the drawing and dimensions furnished.

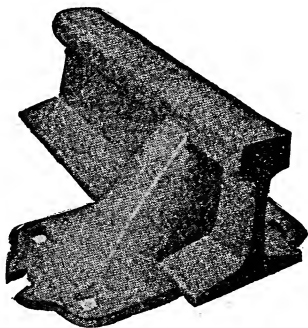
11. The name of the maker and the year of manufacture shall be rolled in raised letters on the side of the splice-bar.

12. The inspector representing the purchaser shall have all reasonable facilities afforded to him by the manufacturer, to satisfy him that the finished material is furnished in accordance with these specifications. All tests and inspections shall be made at the place of manufacture, prior to shipment.

TIE-PLATES.

244. Advantages. (a) As already indicated in § 204, the life of a soft-wood tie is very much reduced by "rail-cutting" and "spike-killing," such ties frequently requiring renewal long

before any serious decay has set in. It has been practically demonstrated that the "rail-cutting" is not due to the mere pressure of the rail on the tie, even with a maximum load on the rail, but is due to the impact resulting from vibration and to the longitudinal working of the rail. It has been proved that this rail-cutting is practically prevented by the use of tie-plates. (b) On curves there is a tendency to overturn the outer rail due to the lateral pressure on the side of the head. This produces a concentrated pressure of the outer edge of the base on the tie which produces rail-cutting and also draws the inner spikes. Formerly the only method of guarding against this was by the use of "rail-braces," one pattern of which is shown in Fig. 120. But it has been found that tie-plates serve the purpose even better, and rail-braces have been abandoned where tie-plates are used. (c) Driving spikes through holes in the plate enables the spikes on *each* side of the rail to mutually support each other, no matter in which (lateral) direction the rail may tend to move, and this probably accounts in large



[FIG. 120.—ATLAS BRACE K.

measure for the added stability obtained by the use of tie-plates. (d) The wear in spikes, called "necking," caused by the vertical vibration of the rail against them, is very greatly reduced. (e) The cost is very small compared with the value of the added life of the tie, the large reduction in the work of track maintenance, and the smoother running on the better track which is obtained. It has been estimated that by the use of tie-plates the life of hard-wood ties is increased from one to three years, and the life of soft-wood ties is increased from three to six years. From the very nature of the case, the value of tie-plates is greater when they are used to protect soft ties.

245. Elements of the design. There is still a great diversity of opinion regarding the relative advantages of tie-plates which are flat on the bottom and those which are corrugated or provided with teeth or claws which are imbedded in the tie. Those used in Europe are without exception flat on the bottom. The Pennsylvania Railroad and the Southern Pacific have also been using flat tie-plates. On the other hand, it is claimed that the pressure required to force a corrugated plate into a tie is about 20% greater than that required to imbed a flat

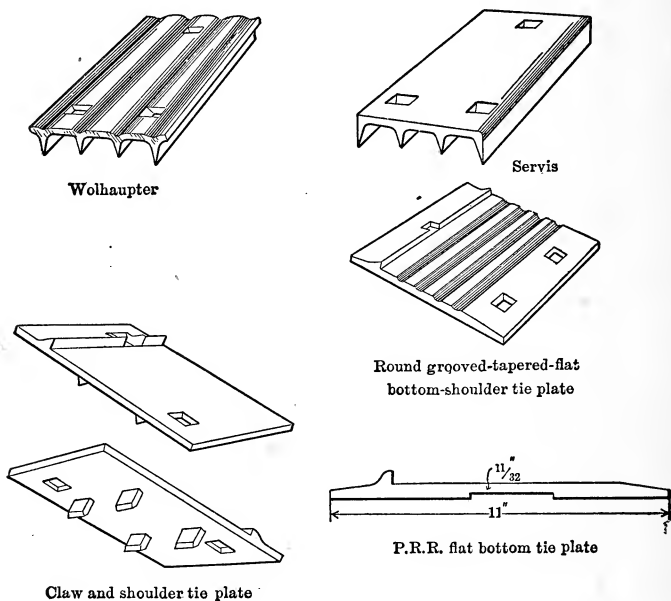


FIG. 121.—VARIOUS FORMS OF TIE-PLATES.

plate of equal thickness in the tie. It is also claimed that tests have shown that the force required to spread the rails when they are fastened with corrugated plates under the rails is from 36% to 100% greater than that required when a flat tie-plate is used. It is especially important that the plate shall be so firmly imbedded in the tie that it cannot move or "rock" with each motion of the rail over it. Instances are known where a treated tie has become unfit for service because

the tie-plate has rocked back and forth until it has worn a hole in the tie. Rain-water filling this hole has leached out the zinc chloride, and the tie has decayed at this point and become unserviceable, when the remainder of the tie showed no decay. The creeping of the rails over the ties is sometimes the cause of failure of ties which have been effectually secured against decay by the use of preservatives. This particular form of tie deterioration has been guarded against on a French railroad by using a tie-plate made of *creosoted* wood, which is 8 inches long, the same width as the width of the base of the rail, and $\frac{1}{8}$ inch thick. Such wooden plates, which will last a year and a half to two years, are made of poplar, or any other hard wood, and cost about \$2.00 per thousand. It should be observed that they are used in connection with wooden screws instead of the ordinary track spikes. When they are worn out, it is only necessary to turn the screw one or two upward turns; the new plate may then be put in endwise and the screw-spikes again fastened down.

A fault of the earlier designs of metal tie-plates was that they were made of plates which were so thin that they would buckle under the pressure of the rail. The claim made for the corrugated plates is that their transverse stress is far greater than that of a flat plate for the same amount of material; but this is not vital provided the flat plates are made sufficiently thick so that they will not buckle. The tie-plate used on the Southern Pacific Railroad has a slightly beveled surface, the plate being $\frac{3}{8}$ inch thick under the outer edge of the rail and $\frac{1}{6}\frac{3}{4}$ inch thick at the inner edge of the rail.

The holes in a tie-plate should be about $\frac{1}{16}$ inch larger than the size of the intended spike. For example, the holes are generally punched with $\frac{5}{8}'' \times \frac{3}{4}''$ holes for a $\frac{9}{16}$ inch spike, or with holes $\frac{3}{4}$ inch square for a $\frac{5}{8}$ inch spike. The length of the plate (perpendicular to the rail) should be about 3 inches more than the base of the rail; this usually means about 8 inches. For very heavy traffic, the thickness should be $\frac{5}{16}$ inch to $\frac{3}{8}$ inch; for average traffic it may be as thin as $\frac{1}{4}$ inch; some plates are made only $\frac{3}{16}$ inch thick. For flat-bottom plates the thickness may be as great as half an inch. The tie-plates under the joint ties must be somewhat longer than the intermediates, in order to allow for the extra length from out to out of the angle-plates.

246. Method of setting. A very important detail in the process of setting the tie-plates on the ties is that the plates should be rigidly attached to the ties in their intended position during the process of setting. If tie-plates with flat bottoms are used, the surface of the tie must be adzed, so that it is not only plane but level, so that there will be no danger that the plate will rock on the tie. When using tie-plates which are corrugated on the under surface, it is necessary to force them into the tie until the under side of the plate is flush with the surface of the tie. This requires a pressure of several thousand pounds. Sometimes trackmen have depended on the easy process of waiting for passing trains to force the corrugations into the tie until the plate is in its intended position. Until the plates are finally set the spikes cannot be driven home, and this apparently cheap and easy process generally results in loose spikes and rails. The best method for new work is to drive the plates into the tie before setting the tie in position. A tie-plate gauge holds both tie-plates in their proper relative position, and both plates may be driven by the use of heavy beetles. When it is necessary to place the plate under the rail and drive it in, it is somewhat difficult to drive it by striking the plate with a swage on each side of the rail alternately. When it is struck on one side, the other side flies up unless held down by a wedge driven between the plate and the rail on the other side of the rail. A straddler, which straddles the rail somewhat like an inverted U, is very useful for this purpose, since it makes it possible to strike the head of the straddler and force down both sides of the plate at once. The Southern Pacific Railroad Company has rigged up a small pile-driver on a hand-car, which is used in connection with a straddler to drive the tie-plates into position. Some western railroads have even adopted the process of rigging up a flat car with a machine which will press the tie-plates into place in the ties before the ties are placed in the track.

SPIKES.

247. Requirements. The rails must be held to the ties by a fastening which will not only give sufficient resistance, but which will retain its capacity for resistance. It must also be cheap and easily applied. The ordinary track-spike fulfills the last

requirements, but has comparatively small resisting power, compared with screws or bolts. Worse than all, the tendency to

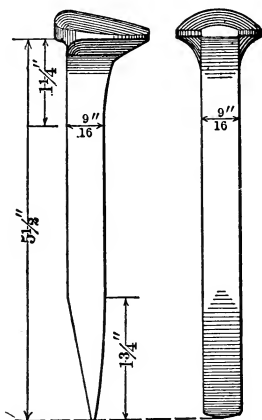


FIG. 122.

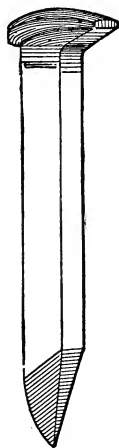


FIG. 123.

vertical vibration in the rail produces a series of upward pulls on the spike that soon loosens it. When motion has once begun the capacity for resistance is greatly reduced, and but little more vibration is required to pull the spike out so much that redriving is necessary. Driving the spike to place again in the same hole is of small value except as a very temporary expedient, as its holding power is then very small. Redriving the spikes in new holes very soon "spike-kills" the tie. Many plans have been devised to increase the holding power of spikes, such as making them jagged, twisting the spike, swelling the spike at about the center of its length, etc. But it has been easily demonstrated that the fibers of the wood are generally so crushed and torn by driving such spikes that their holding power is less than that of the plain spike.

The ordinary spike (see Fig. 122) is made with a square cross-section which is uniform through the middle of its length, the lower $13\frac{1}{4}$ " tapering down to a chisel edge, the upper part swelling out to the head. The Goldie spike (see Fig. 123) aims to improve this form by reducing to a minimum the destruction of the fibers. To this end, the sides are made smooth, the edges are clean-cut, and the point, instead of being chisel-shaped, is ground

down to a pyramidal form. Such fiber-cutting as occurs is thus accomplished without much crushing, and the fibers are thus pressed away from the spike and slightly downward. Any tendency to draw the spike will therefore cause the fibers to press still harder on the spike and thus increase the resistance.

248. Driving. The holding power of a spike depends largely on how it is driven. If the blows are eccentric and irregular in direction, the hole will be somewhat enlarged and the holding power largely decreased. The spikes on each side of the rail in any one tie should not be directly opposite, but should be staggered. Placing them directly opposite will tend to split the tie, or at least decrease the holding power of the spikes. The direction of staggering should be reversed in the two pairs of spikes in any one tie (see Fig. 124). This will tend to prevent any twisting of the tie in the ballast, which would otherwise loosen the rail from the tie.

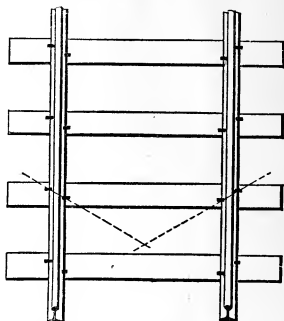


FIG. 124.—SPIKE-DRIVING.

249. Screws and bolts. The use of these abroad is very extensive, but their use in this country has not passed the experimental stage. The screws are "wood"-screws (see Fig. 125),

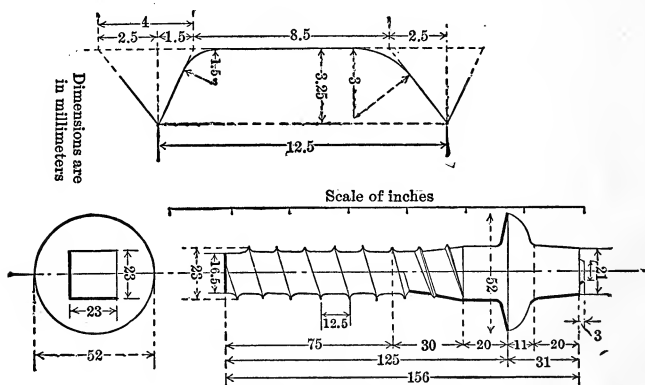


FIG. 125.—SCREW SPIKE.

having large square heads, which are screwed down with a track-wrench. Holes, having the same diameter as the *base* of the screw-heads, should be first bored into the tie, at exactly the right position and at the proper angle with the vertical. A light wooden frame is sometimes used to guide the auger at the proper angle. Sometimes the large head of the screw bears directly against the base of the rail, as with the ordinary spike. Other designs employ a plate, made to fit the rail on one side, bearing on the tie on the other side, and through which the screw passes. These screws cost much more than the spikes and require more work to put in place, but their holding power is much greater and the work of track maintenance is very much less. Screw-bolts, passing entirely through the tie, having the head at the bottom of the tie and the nut on the upper side, are also used abroad. These are quite difficult to replace, requiring that the ballast be dug out beneath the tie, but on the other hand the

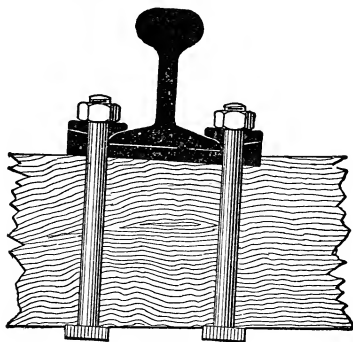


FIG. 126.

occasions for replacing such a bolt are comparatively rare, as their durability is very great. The use of screws or bolts increases the life of the tie by the avoidance of "spike-killing." It is capable of demonstration that the reduced cost of maintenance and the resulting improvement in track would much more than repay the added cost of screws and bolts, but it seems impossible to induce railroad directors to authorize a large and immediate additional expenditure to make an annual saving whose value, although unquestionably considerable, cannot be exactly computed.

250. "Wooden spikes." Among the regulations for track-laying given in § 208, mention was made of wooden "spikes,"

or plugs, which are used to fill up the holes when spikes are withdrawn. The value of the policy of filling up these holes is unquestionable, since the expense is insignificant compared with the loss due to the quick and certain decay of the tie if these holes are allowed to fill with water and remain so. But the method of making these plugs is variable. On some roads they are "hand-made" by the trackmen out of otherwise useless scraps of lumber, the work being done at odd moments. This policy, while apparently cheap, is not necessarily so, for the hand-made plugs are irregular in size and therefore more or less inefficient. It is also quite probable that if the trackmen are required to make their own plugs, they would spend time on these very cheap articles which could be more profitably employed otherwise. Since the holes made by the spikes are larger at the top than they are near the bottom, the plugs should *not* be of uniform cross-section but should be slightly wedge-shaped. The "Goldie tie-plug" (see Fig. 127) has been designed to fill these requirements. Being machine-made, they are uniform in size; they are of a shape which will best fit the hole; they can be furnished of any desired wood, and at a cost which makes it a wasteful economy to attempt to cut them by hand.



FIG. 127.

TRACK-BOLTS AND NUT-LOCKS.

251. Essential requirements. The track-bolts must have sufficient strength and must be screwed up tight enough to hold the angle-plates against the rail with sufficient force to develop the full transverse strength of the angle-bars. On the other hand the bolts should not be screwed so tight that slipping may not take place when the rail expands or contracts with temperature. It would be impossible to screw the bolts tight enough to prevent slipping during the contraction due to a considerable fall of temperature on a straight track, but when the track is curved, or when expansion takes place, it is conceivable that the resistance of the ties in the ballast to lateral motion may be less than the resistance at the joint. A test to determine this resistance was made by Mr. A. Torrey, chief engineer of the Mich. Cent. R. R., using 80-lb. rails and ordinary angle-bars, the bolts being screwed up as usual. If required a force of about 31000 to

35000 lbs. to start the joint, which would be equivalent to the stress induced by a change of temperature of about 22° . But if the central angle of any given curve is small, a comparatively small lateral component will be sufficient to resist a compression of even 35000 lbs. in the rails. Therefore there will ordinarily be no trouble about having the joints screwed too tight. The vibration caused by the passage of a train reduces the resistance to slipping. This vibration also facilitates an objectionable feature, viz., loosening of the nuts of the track-bolts. The bolt is readily prevented from turning by giving it a form which is *not* circular immediately under the head and making corresponding holes in the angle-plate. Square holes would answer the purpose, except that the square corners in the holes in the angle-plates would increase the danger of fracture of the plates. Therefore the holes (and also the bolts, under the head) are made of an oval form, or perhaps a square form with rounded corners, avoiding angles in the outline.

The nut-locks should be simple and cheap, should have a life at least as long as the bolt, should be effective, and should not lose their effectiveness with age. Many of the designs that have been tried have been failures in one or more of these particulars as will be described in detail below.

252. Design of track-bolts.—In Fig. 128 is shown a common design of track-bolt. In its general form this represents the

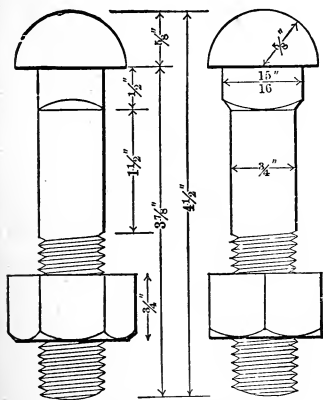


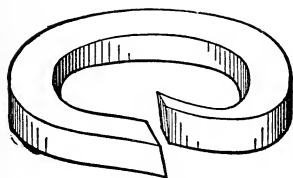
FIG. 128.—TRACK-BOLT.

bolt used on nearly all roads, being used not only with the common angle-plates, but also with many of the improved designs of rail-joints. The variations are chiefly a general increase in size to correspond with the increased weight of rails, besides variations in detail dimensions which are frequently unimportant. The diameter is usually $\frac{3}{4}$ " to $\frac{7}{8}$ "; 1" bolts are sometimes used for the heaviest sections of rails. As to length, the bolt should not extend more than $\frac{1}{2}$ " outside of the nut when

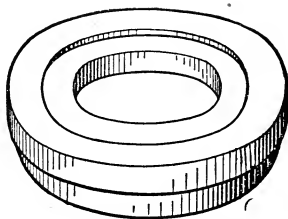
it is screwed up. If it extends farther than this it is liable to be

broken off by a possible derailment at that point. The lengths used vary from $3\frac{1}{4}$ " which may be used with 60-lb. rails, to 5", which is required with 100-lb. rails. The length required depends somewhat on the type of nut-lock used.

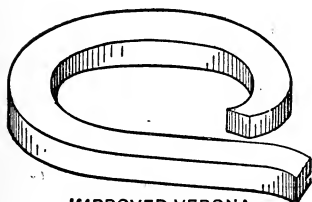
253. Design of nut-locks. The designs for nut-locks may be divided into three classes: (a) those depending entirely on an elastic washer which absorbs the vibration which might otherwise induce turning; (b) those which jam the threads of the bolt and nut so that, when screwed up, the frictional resistance is too great to be overcome by vibration; (c) the "positive" nut-locks—those which mechanically hold the nut from turning. Some of the designs combine these principles to some extent. The "vulcanized fiber" nut-lock is an example of the first class. It consists essentially of a rubber washer which is protected by an iron ring. When first placed this lock is effective, but the rubber soon hardens and loses its elasticity and it is then ineffective and worthless. Another illustration of class (a) is the use of wooden blocks, generally 1" to 2" oak, which extend the entire length of the angle-bar, a single piece forming the washer for the four or six bolts of a joint. This form is cheap, but the wood soon shrinks, loses its elasticity, or decays so that it soon becomes worthless, and it requires constant adjustment to keep it in even tolerable condition. The "Verona" nut-lock is another illustration of class (a) which also combines some of the positive elements of class (c). It is made of tempered steel and, as shown in Fig. 129, is warped and has sharp edges or points. The warped form furnishes the element of elastic pressure when the nut is screwed up. The steel being harder than the iron of the angle-bar or of the nut, it bites into them, owing to the great pressure that must exist when the washer is squeezed nearly flat, and thus prevents any *backward* movement, although forward movement (or tightening the bolt) is not interfered with. The "National" nut-lock is a type of the second class (b), in which, like the "Harvey" nut-lock, the nut and lock are combined in one piece. With six-bolt angle-bars and 30-foot rails, this means a saving of 2112 pieces on each mile of single track. The "National" nuts are open on one side. The hole is drilled and the thread is cut slightly smaller than the bolt, so that when the nut is screwed up it is forced slightly open and therefore presses on the threads of the bolt with such force that vibration cannot jar it loose. Unlike the "National" nut, the "Harvey"



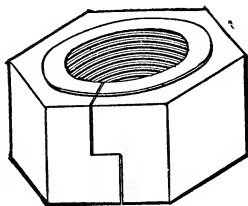
VERONA



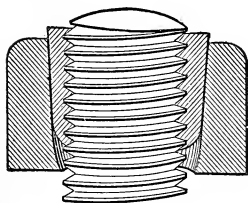
VULCANIZED FIBRE



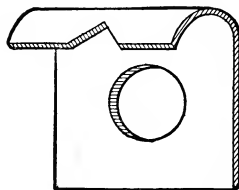
IMPROVED VERONA



NATIONAL



Columbia Nut Lock



JONES

FIG. 123.—TYPES OF NUT-LOCKS.

nut is solid, but the form of the thread is progressively varied so that the thread pinches the thread of the bolt and the frictional resistance to turning is too great to be affected by vibration.

The "Columbia" nut-lock is a two-piece nut, both parts of which must turn simultaneously. As shown in the figure, one section wedges into the other. The greater the tension in the bolt, the greater the wedging action and the greater the friction to prevent turning.

The "Jones" nut-lock, belonging to class (c), is a type of a nut-lock that does not depend on elasticity or jamming of screw-threads. It is made of a thin flexible plate, the square part of which is so large that it will not turn after being placed on the bolt. After the nut is screwed up, the thin plate is bent over so that the re-entrant angle of the plate engages the corner of the nut and thus mechanically prevents any turning. The metal is supposed to be sufficiently tough to endure without fracture as many bendings of the plate as will ever be desired. Nut-locks of class (c) are not in common use.

CHAPTER XI.

SWITCHES AND CROSSINGS.

SWITCH CONSTRUCTION.

254. Essential elements of a switch. Flanges of some sort are a necessity to prevent car-wheels from running off from the rails on which they may be moving. But the flanges, although a necessity, are also a source of complication in that they require some special mechanism which will, when desired, guide the wheels out from the controlling influence of the main-line rails. This must either be done by raising the wheels high enough so that the flanges may pass *over* the rails, or by breaking the continuity of the rails in such a way that channels or "flange spaces" are formed *through* the rails. An ordinary stub-switch breaks the continuity of the main-line rails in three places, two of them at the switch-block and one at the frog. The Wharton switch avoids two of these breaks by so placing inclined planes that the wheels, rolling on their flanges, will surmount these inclines until they are a little higher than the rails. Then the wheels on the side toward which the switch runs are guided over and across the main rail on that side. This rise being accomplished in a short distance, it becomes impracticable to operate these switches except at slow speeds, as any sudden change in the path of the center of gravity of a car causes very destructive jars both to the switch and to the rolling stock. The other general method makes a break in one main rail (or both) at the switch-block. In both methods the wheels are led to one side by means of the "lead rails," and finally one line of wheels passes *through* the main rail on that side by means of a "frog." There are some designs by which even this break in the main rail is avoided, the wheels being led *over* the main rail by means of a short *movable* rail which is on occasion placed across the main rail, but such designs have not come into general use.

255. Frogs. Frogs are provided with two channel-ways or "flange spaces" through which the flanges of the wheels move.

Each channel cuts out a parallelogram from the tread area. Since the wheel-tread is always wider than the rail, the wing rails will support the wheel not only across the space cut out by the channel, but also until the tread has passed the point of the frog and can obtain a broad area of contact on the tongue of the frog. This is the theoretical idea, but it is very imperfectly

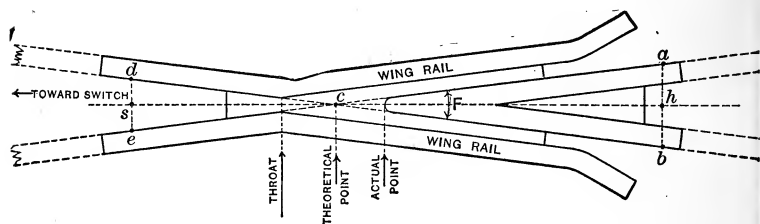


FIG. 130.—DIAGRAMMATIC DESIGN OF FROG.

realized. The wing rails are sometimes subjected to excessive wear owing to "hollow treads" on the wheels—owing also to the frog being so flexible that the point "ducks" when the wheel approaches it. On the other hand the sharp point of the frog will sometimes cause destructive wear on the tread of the wheel. Therefore the tongue of the frog is not carried out to the sharp theoretical point, but is purposely somewhat blunted. But the break which these channels make in the continuity of the tread area becomes extremely objectionable at high speeds, being mutually destructive to the rolling stock and to the frog. The jarring has been materially reduced by the device of "spring frogs"—to be described later. Frogs were originally made of cast iron—then of cast iron with wearing parts of cast steel, which were fitted into suitable notches in the cast iron. This form proved extremely heavy and devoid of that elasticity of track which is necessary for the safety of rolling stock and track at high speeds. The present universal practice is to build the frog up of pieces of rails which are cut or bent as required. These pieces of rails (at least four) are sometimes assembled by riveting them to a flat plate, but this method is now but little used, except for very light work. The usual practice is now chiefly divided between "bolted" and "keyed" frogs. In each case the space between the rails, except a sufficient flange-way, is filled with a cast-iron filler and the whole assemblage of parts

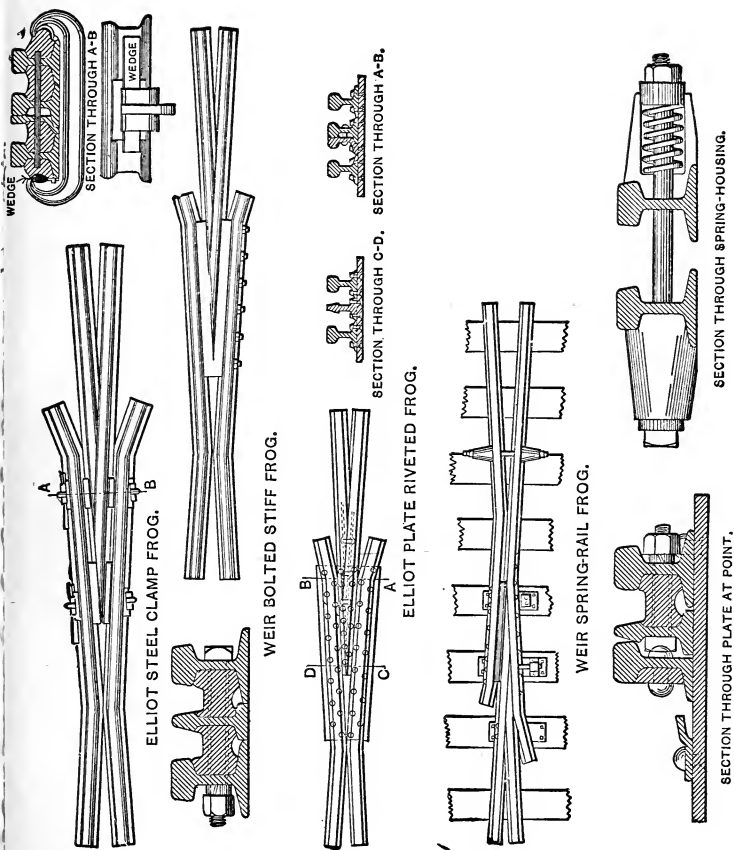


PLATE VIII.—SOME TYPES OF FROGS.

is suitably bolted or clamped together, as is illustrated in Plate VIII. The operation of a spring-rail frog is evident from the figure. Since a siding is usually operated at slow speed, while the main track may be operated at fast speed, a spring-rail frog will be so set that the tread is continuous for the main track and broken for the siding. This also means that the spring-rail will only be moved by trains moving at a (presumably) slow speed on to the siding. For the fast trains on the main line such a frog is substantially a "fixed" frog and has a tread which is practically continuous.

256. To find the frog number. The frog number (n) equals the ratio of the distance of any point on the tongue of the frog from the theoretical point of the frog divided by the width of the tongue at that point, i.e. $=hc \div ab$ (Fig. 130). This value may be directly measured by applying any convenient unit of measure (even a knife, a short pencil, etc.) to some point of the tongue where the width just equals the unit of measure, and then noting how many times the unit of measure is contained in the distance from that place to the theoretical point. But since c , the theoretical point, is not so readily determinable with exactitude, it being the imaginary intersection of the gauge lines, it may be more accurate to measure de , ab , and hs ; then n , the frog number, $=hs \div (ab + de)$. If the frog angle be called F , then

$$n = hc \div ab = hs \div (ab + de) = \frac{1}{2} \cot \frac{1}{2}F;$$

i.e.,

$$\cot \frac{1}{2}F = 2n.$$

257. Stub switches. The use of these, although once nearly universal, has been practically abandoned as **turnouts** from *main track* except for the poorest and cheapest roads. In some States their use on main track is prohibited by law. They have the sole merit of cheapness with adaptability to the circumstances of very light traffic operated at slow speed when a considerable element of danger may be tolerated for the sake of economy. The rails from A to B (see Fig. 131*) are not fastened

*The student should at once appreciate that in Fig. 131, as well as in nearly all the remaining figures in this chapter, it becomes necessary to use excessively large frog angles, short radii, and a very wide gauge in order to illustrate the desired principles with figures which are sufficiently small for the page. In fact, the proportions used in the figures are such that serious mechanical difficulties would be encountered if they were used. These difficulties are here ignored because they can be neglected in the proportions used in practice.

farther rail end, besides producing such a stress in the construction that a breakage is rendered quite likely, and such a breakage might have very serious consequences.

258. Point switches. The essential principle of a point switch is illustrated in Fig. 132. As is shown, one main rail and also one of the switch-rails is unbroken and immovable. The other main rail (from *A* to *F*) and the corresponding portion of the other lead rail are substantially the same as in a stub switch. A portion of the main rail (*AB*) and an equal length of the opposite lead rail (usually 15 to 24 feet long) are fastened together by tie-rods. The end at *A* is jointed as usual and the other end is pointed, both sides being trimmed down so that the feather edge at *B* includes the web of the rail. In order to retain in it as much strength as possible, the point-rail is raised so that it rests on the base of the stock-rail, one side of the base of the point-rail being entirely cut away. As may be seen in Fig. 133, although the influence of the point of the rail in moving the wheel-flange away from the stock-rail is really zero at that point, yet the rail has all the strength of the web and about one-half that of the base—a very fair angle-iron. The planing runs back in *straight* lines, until at about six or seven feet back from the point the full width of the head is obtained. The full width of the base will only be obtained at about 13 feet from the point. The A. R. E. A. standard switch rail is always cut on the basis that the distance between gauge lines at the heel of the switch (the distance *MN* in Fig. 140) is $6\frac{1}{4}$ inches and that the “point” is $\frac{1}{4}$ inch wide. Then, using four standard lengths, 11, $16\frac{1}{2}$, 22, and 33 feet, the angles vary from $2^{\circ} 36' 19''$ to $0^{\circ} 52' 05''$, as shown in Table III.

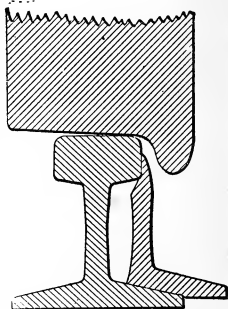


FIG. 133.

259. Switch-stands. The simplest and cheapest form is the “ground lever,” which has no target. The radius of the circle described by the connecting-rod pin is precisely one-half the throw. From the nature of the motion the device is practically

self-locking in either position, padlocks being only used to prevent malicious tampering. The numerous designs of upright stands are always combined with targets, one design of which is

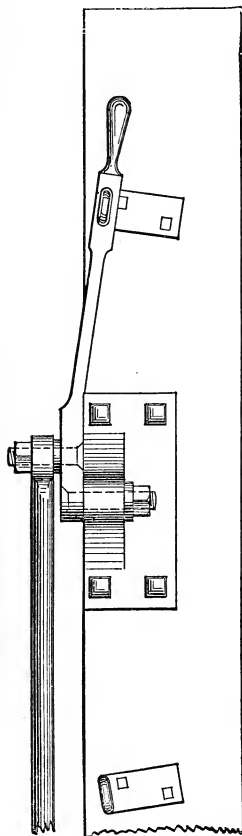


FIG. 134.—GROUND LEVER FOR THROWING A SWITCH.

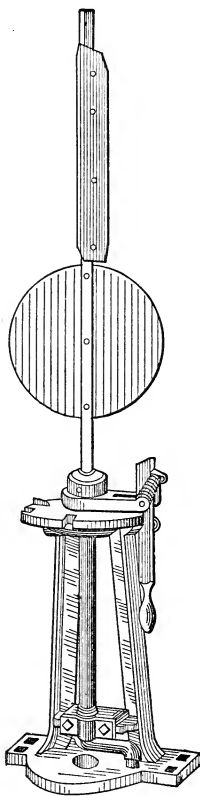


FIG. 135.

illustrated in Fig. 135. When the road is equipped with interlocking signals, the switch-throw mechanism forms a part of the design.

260. Tie-rods. These are fastened to the webs of the rails by means of lugs which are bolted on, there being usually a hinge-joint between the rod and the lug. Four such tie-rods are

generally necessary. The first rod is sometimes made without hinges, which gives additional stiffness to the comparatively weak rail-points. The old-fashioned tie-rod, having jaws fitting the base of the rail, was almost universally used in the days of stub switches. One great inconvenience in their use lies in the fact that they must be slipped on, one by one, over the *free* ends of the switch-rails.

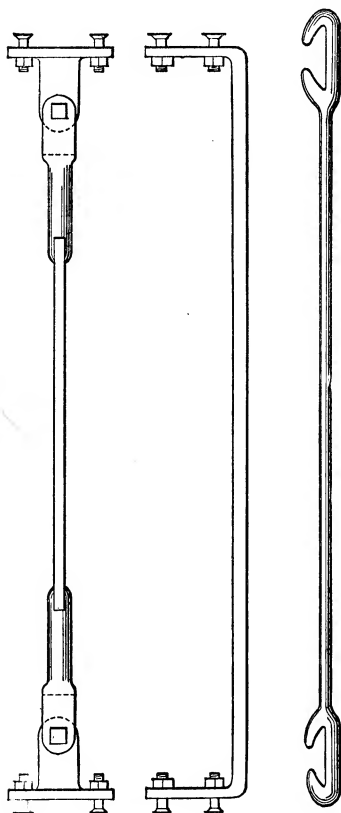


FIG. 136.—FORMS OF TIE-RODS.

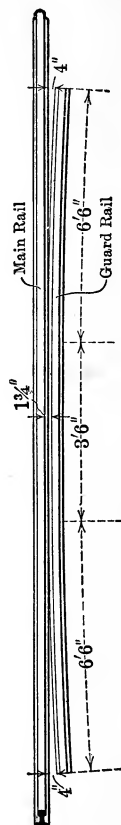


FIG. 136a.—STANDARD GUARD RAIL.

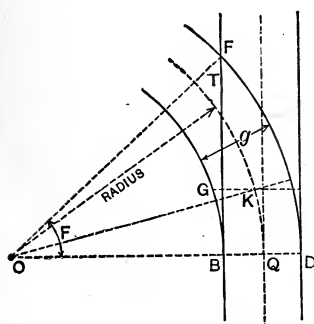
261. Guard-rails. As shown in Figs. 131 and 132, guard-rails are used on both the main and switch tracks opposite the frog-point. Their function is not only to prevent the possibility of

the wheel-flanges passing on the wrong side of the frog-point, but also to save the side of the frog-tongue from excessive wear. The flange-way space between the heads of the guard-rail and wheel-rail should equal $1\frac{3}{4}$ inches. Since this is less than the space between the heads of ordinary (say 80-pound) rails when placed base to base, to say nothing of the $\frac{3}{4}$ " required for spikes, it becomes necessary to cut the flange of the guard-rail. The length of the rail should be 16 feet 6 inches, the middle portion being straight for a length of 3 feet 6 inches, and the ends, each being 6 feet 6 inches long, curved out so that the side of the rail head at each end is 4 inches from the main rail head, when the flange-way at the center is $1\frac{3}{4}$ inches. See Fig. 136a

MATHEMATICAL DESIGN OF SWITCHES.

In all of the following demonstrations regarding switches, turnouts, and crossovers, the lines are assumed to represent the *gauge-lines*—i.e., the lines of the *inside* of the head of the rails.

262. Design with circular lead-rails. The simplest method



is to consider that the lead-rails curve out from the main track-rails by arcs of circles which are tangent to the main rails and which extend to the frog-point *F*. The simple curve from *D* to *F* is of such radius that $(r + \frac{1}{2}g)$ vers *F* = *g*, in which *F* = the frog angle, *g* = gauge, *L* = the "lead" (*BF*), and *r* = the radius of the center of the switch-rails.

FIG. 137.

$$\therefore r + \frac{1}{2}g = \frac{g}{\text{vers } F}. \quad (74)$$

Also, $BF \div BD = \cot \frac{1}{2}F$; $BD = g$; $BF = L$.

$$\therefore L = g \cot \frac{1}{2}F. \quad (75)$$

Also, $L = (r + \frac{1}{2}g) \sin F$; (76)

$$QT = 2r \sin \frac{1}{2}F. \quad (77)$$

These formulæ involve the angle *F*. As shown in Table III, the angles (*F*) are always odd quantities, and their trigonometric functions are somewhat troublesome to obtain closely with

Stub-switches are generally used with large frog angles. For small frog angles (large frog-numbers) the values of QK are so great that the length of rail left unspiked is too great for a safe track. If this were obviated by spiking down a portion of the lead the theoretical accuracy of the switch would be lost. The values of QK for various frog-numbers is given in Table III. These are based on a uniform throw (t) of $5\frac{1}{2}$ inches.

263. Effect of straight frog-rails. A portion of the ends of

the rails of a frog are free and *may* be bent to conform to the switch-rail curve, but there is a considerable portion which is fitted to the cast-iron filler, and this portion is always straight. Call the length of this straight portion back from the frog-point f ($=FH$, Fig. 138). Then we have

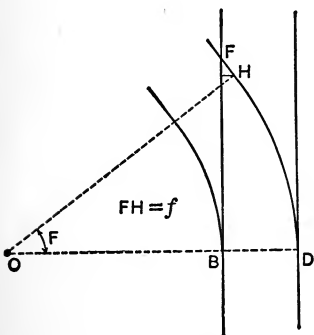


FIG. 138.

$$\begin{aligned} r + \frac{1}{2}g &= (g - f \sin F) \div \text{vers } F \\ &= \frac{g}{\text{vers } F} - f \cot \frac{1}{2}F \\ &= \frac{g}{\text{vers } F} - 2fn. \end{aligned} \quad (82)$$

$$\begin{aligned} BF = L &= (g - f \sin F) \cot \frac{1}{2}F + f \cos F \\ &= 2gn - f \sin F \cot \frac{1}{2}F + f \cos F \\ &= 2gn - f(1 + \cos F) + f \cos F \\ &= 2gn - f. \end{aligned} \quad (83)$$

Since $r - \frac{1}{2}f = (L - f \sec F) \cot F$, and

$$r + \frac{1}{2}g = (L - f \cos F) \text{cosec } F,$$

$$\begin{aligned} r &= \frac{1}{2}L(\cot F + \text{cosec } F) - \frac{1}{2}f \sec F \cot F - \frac{1}{2}f \cos F \text{cosec } F \\ &= Ln - \frac{1}{2}f \left(\frac{1 + \cos f}{\sin f} \right). \end{aligned}$$

$$\begin{aligned} r &= Ln - \frac{1}{2}f \cot \frac{1}{2}F \\ &= Ln - fn. \quad \text{Then from (83)} \\ r &= 2gn^2 - 2fn. \end{aligned} \quad (84)$$

264. Effect of straight point-rails. The "point switches," now so generally used, have *straight* switch-rails. This requires

an *angle* in the alignment rather than turning off by a tangential curve. The angle is, however, very small (between 1° and 3°), and the disadvantages of this angle are small compared with the very great advantages of the device.

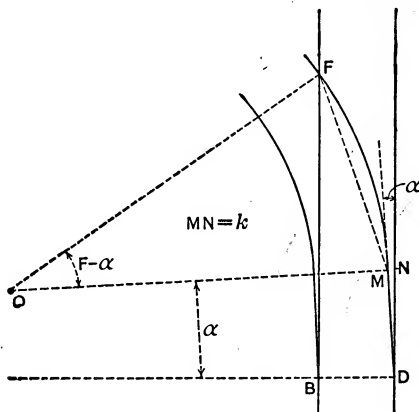


FIG. 139.

$$FM = \frac{g-k}{\sin \frac{1}{2}(F+a)};$$

$$r + \frac{1}{2}g = \frac{FM}{2 \sin \frac{1}{2}(F-a)}$$

$$= \frac{g-k}{2 \sin \frac{1}{2}(F+a) \sin \frac{1}{2}(F-a)}$$

$$= \frac{g-k}{\cos a - \cos F} \dots \dots \dots (85)$$

$$BF = L = FM \cos \frac{1}{2}(F+a) + DN$$

$$= (g-k) \cot \frac{1}{2}(F+a) + DN \dots \dots \dots (86)$$

265. Combined effect of straight frog-rails and straight point-rails. It becomes necessary in this case to find a curve which shall be tangent to both the point-rail and the frog-rail. The curve therefore begins at *M*, its tangent making an angle of *a* (varying from $0^\circ 52'$ to $2^\circ 36'$) with the main rail, and runs to *H*.

No. 9 frog, standard gauge $4' 8\frac{1}{2}''$, $f=6'.00$, $k=6\frac{1}{4}''=0'.521$, $DN=16' 6''$, and $\alpha=1^\circ 44' 11''$, we may tabulate the comparative results:

	§ 262. Simple circle. Curved frog-rail. Curved switch-rail.	§ 263. Straight frog-rail. Curved switch-rail.	§ 264. Curved frog-rail. Straight switch-rail.	§ 265. Straight frog-rail. Straight switch-rail,
Deg. of curve	762.75 7° 31'	654.74 8° 46'	733.68 7° 49'	616.63 9° 18'
$\frac{r}{L}$	84.75	78.75	75.68	72.27

This shows that the effect of using straight frog-rails and straight switch-rails is to sharpen the curve and shorten the lead in each case separately, and that the combined effect is still greater. The effect of the straight switch-rails is especially marked in reducing the length of lead, and therefore Eq. 78 to 80, although having the advantage of extreme simplicity, cannot be used for point-switches without material error. The effect of the straight frog-rail is less, and since it can be materially reduced by bending the free end of the frog-rails, the influence of this feature is frequently ignored, the frog-rails are assumed to be curved, and Eq. 85 and 86 are used. (See § 276 for a further discussion of this point.)

267. Dimensions for a turnout from the OUTER side of a curved

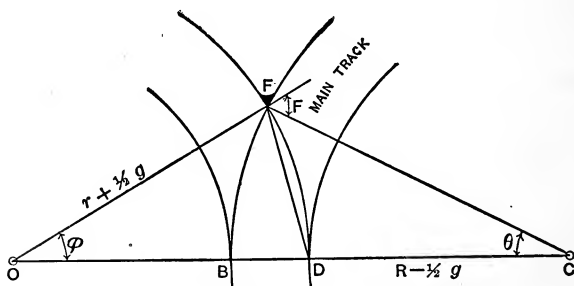


FIG. 141.

track. In this demonstration the switch-rails will be considered as uniformly circular from the switch-points to the frog-point.

In the triangle FCD (Fig. 141) we have

$$(FC + CD) : (FC - CD) :: \tan \frac{1}{2}(FDC + DFC) : \tan \frac{1}{2}(FDC - DFC);$$

but $\frac{1}{2}(FDC + DFC) = 90^\circ - \frac{1}{2}\theta$

and $\frac{1}{2}(FDC - DFC) = \frac{1}{2}F$.

Also, $FC + CD = 2R$ and $FC - CD = g$;

$$\therefore 2R : g :: \cot \frac{1}{2}\theta : \tan \frac{1}{2}F$$

$$:: \cot \frac{1}{2}F : \tan \frac{1}{2}\theta;$$

$$\therefore \tan \frac{1}{2}\theta = \frac{gn}{R} \dots \dots \dots (91)$$

Also, $OF : FC :: \sin \theta : \sin \phi$; but $\phi = (F - \theta)$;

then $r + \frac{1}{2}g = (R + \frac{1}{2}g) \frac{\sin \theta}{\sin (F - \theta)} \dots \dots \dots (92)$

$$BF = L = 2(R + \frac{1}{2}g) \sin \frac{1}{2}\theta \dots \dots \dots (93)$$

If the curvature of the main track is very sharp or the frog angle unusually small, F may be less than θ ; in which case the center O will be on the same side of the main track as C . Eq. 92 will become (by calling $r = -r$ and changing the signs)

$$(r - \frac{1}{2}g) = (R + \frac{1}{2}g) \frac{\sin \theta}{\sin (\theta - F)} \dots \dots \dots (94)$$

If we call d the degree of curve corresponding to the radius r , and D the degree of curve corresponding to the radius R , also d' the degree of curve of a turnout from a straight track (the frog angle F being the same), it may be shown that $d = d' - D$ (very nearly). To illustrate we will take three cases, a number 6 frog (very blunt), a number 9 frog (very commonly used), and a number 12 frog (unusually sharp). Suppose $D = 4^\circ 0'$; also $D = 10^\circ 0'$; $g = 4' 8\frac{1}{2}'' = 4'.708$.

A brief study of the tabular form on p. 304 will show that the error involved in the use of the approximate rule for ordinary curves (4° or less) and for the usual frogs (about No. 9) is really insignificant, and that, even for sharper curves (10° or more), or for very blunt frogs, the error would never cause damage, considering the lower probable speed. In the most unfavorable case noted above the change in radius is about 1%. On account of the closeness of the approximation the method is frequently used. The remarkable agreement of the computed values of L

Frog number.	$D = 4^\circ.$				"L" for straight track.
	d	$d' - D$	Error.	L	
6	12° 54' 20"	12° 57' 52"	0° 03' 32"	56.57	56.50
9	3 30 27	3 31 04	0 0 37	84.85	84.75
12	0 13 33	0 13 36	0 0 03	112.72	113.00

Frog number.	$D = 10^\circ.$				"L" for straight track.
	d	$d' - D$	Error.	L	
6	6° 53' 24"	6° 57' 52"	0° 04' 28"	56.66	56.50
9	2 27 54	2 28 56	0 01 02	84.86	84.75
12	5 44 26	5 46 24	0 01 58	112.91	113.00

with the corresponding values for a straight main track (the lead rails circular throughout) shows that the error is insignificant in using the more easily computed values.

268. Dimensions for a turnout from the INNER side of a curved track. (Lead rails circular throughout.) From Fig. 142 we have, from the triangle DFC ,

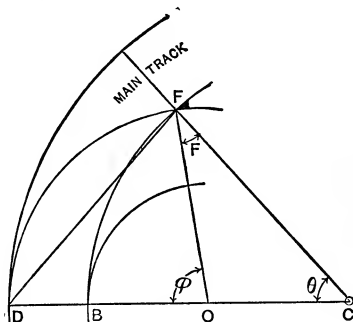


FIG. 142.

$$DC + FC : DC - FC :: \tan \frac{1}{2}(DFC + FDC) : \tan \frac{1}{2}(DFC - FDC);$$

$$\text{but} \quad \frac{1}{2}(DFC + FDC) = 90^\circ - \frac{1}{2}\theta$$

$$\text{and} \quad \frac{1}{2}(DFC - FDC) = \frac{1}{2}F;$$

$$\therefore 2R : g :: \cot \frac{1}{2}\theta : \tan \frac{1}{2}F$$

$$:: \cot \frac{1}{2}F : \tan \frac{1}{2}\theta;$$

$$\therefore \tan \frac{1}{2}\theta = \frac{gn}{R}. \quad \dots \dots \dots (95)$$

From OFC , $OF:FC::\sin \theta:\sin (F+\theta)$.

$$(r+\frac{1}{2}g)=(R-\frac{1}{2}g)\frac{\sin \theta}{\sin (F+\theta)}. \quad . \quad . \quad . \quad (96)$$

$$L=BF=2(R-\frac{1}{2}g)\sin \frac{1}{2}\theta. \quad . \quad . \quad . \quad (97)$$

As in § 267, it may be readily shown that the degree of the turnout (d) is *nearly* the *sum* of the degree of the main track (D) and the degree (d') of a turnout from a straight track when the frog angle is the same. The discrepancy in this case is somewhat greater than in the other, especially when the curvature of the main track is sharp. If the frog angle is also large, the curvature of the turnout is excessively sharp. If the frog angle is very small, the liability to derailment is great. Turnouts to the inside of a curved track should therefore be avoided, unless the curvature of the main track is small.

269. Double turnout from a straight track. In Fig. 143 the frogs F_l and F_r are generally made equal. Then, if there are

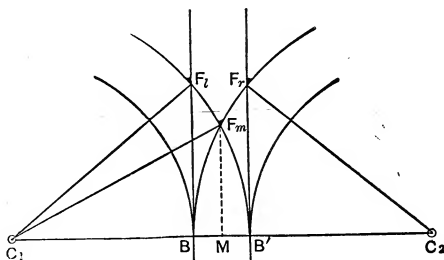


FIG. 143.

uniform curves from B' to F_l and from B to F_r , the required value of F_m is obtained from

$$\text{vers } \frac{1}{2}F_m = \frac{g}{2(r+\frac{1}{2}g)}, \quad . \quad . \quad . \quad . \quad (98)$$

r being found from Eq. 78, in which n is the frog number of F_l or F_r .

$$MF_m = r \tan \frac{1}{2}F_m;$$

but since $n_m = \frac{1}{2} \cot \frac{1}{2}F_m$,

$$MF_m = \frac{r}{2n_m}. \quad . \quad . \quad . \quad . \quad (99)$$

Since vers $F_l = \frac{g}{(r + \frac{1}{2}g)}$,

$$\text{vers } \frac{1}{2}F_m = \frac{1}{2} \text{ vers } F_l. \quad . \quad . \quad . \quad (100)$$

Also, since $(C_1 F_m)^2 = (M F_m)^2 + (C_1 M)^2$, we have

$$(r + \frac{1}{2}g)^2 = \left(\frac{r}{2n_m}\right)^2 + r^2;$$

$$r^2 + rg + \frac{1}{4}g^2 = \frac{r^2}{4n_m^2} + r^2.$$

Simplifying and substituting, $r = 2gn^2$, we have

$$2g^2n^2 + \frac{1}{4}g^2 = \frac{4g^2n^4}{4n_m^2};$$

$$n_m^2 = \frac{n^4}{2n^2 + \frac{1}{4}}.$$

Dropping the $\frac{1}{4}$, which is always insignificant in comparison with $2n^2$, we have

$$n_m = \frac{n}{\sqrt{2}} = n \times .707 (\text{approx.}). \quad . \quad . \quad . \quad (101)$$

Frogs are usually made with angles corresponding to integral values of n , or sometimes in "half" sizes, *e.g.* 6, $6\frac{1}{2}$, 7, $7\frac{1}{2}$, etc. If No. $8\frac{1}{2}$ frogs are used for F_l and F_r , the exact frog number for F_m is 6.01. This is so nearly 6 that a No. 6 frog may be used without sensible inaccuracy. Numbers 7 and 10 are a less perfect combination. If sharp frogs must be used, $8\frac{1}{2}$ and 12 form a very good combination.

If it becomes necessary to use other frogs because the right combination is unobtainable, it may be done by compounding the curve at the middle frog. F_l and F_r should be greater than $\frac{1}{2}F_m$. If equal to $\frac{1}{2}F_m$, the rails would be straight from the middle frog to the outer frogs. In Fig. 144, $\theta_1 = F_l - \frac{1}{2}F_m$.

Drawing the chord $\overline{F_l F_m}$,

$$\overline{KF_l F_m} = F_l - \frac{1}{2}\theta_1 = F_l - \frac{1}{2}F_l + \frac{1}{4}F_m = \frac{1}{2}(F_l + \frac{1}{2}F_m);$$

$$\frac{\overline{F_l F_m}}{\sin \overline{KF_l F_m}} = \frac{\overline{KF_m}}{2 \sin \frac{1}{2}(F_l + \frac{1}{2}F_m)}; \quad . \quad . \quad (102)$$

$$\overline{KF_l} = \overline{KF_m} \cot \overline{KF_l F_m} = \frac{1}{2}g \cot \frac{1}{2}(F_l + \frac{1}{2}F_m); \quad . \quad (103)$$

$$\begin{aligned} (r_1 + \frac{1}{2}g) &= \frac{\overline{F_l F_m}}{2 \sin \frac{1}{2}\theta} = \frac{g}{4 \sin \frac{1}{2}(F_l + \frac{1}{2}F_m) \sin \frac{1}{2}(F_l - \frac{1}{2}F_m)} \\ &= \frac{\frac{1}{2}g}{\cos \frac{1}{2}F_m - \cos F_l} \quad . \quad . \quad . \quad (104) \end{aligned}$$

If three frogs, all different, *must* be used, the largest may be selected as F_m ; the radius of the lead rails may be found by an inversion of Eq. 98; F_m may be located in the center of the tracks by Eq. 99; then each of the smaller frogs may be located

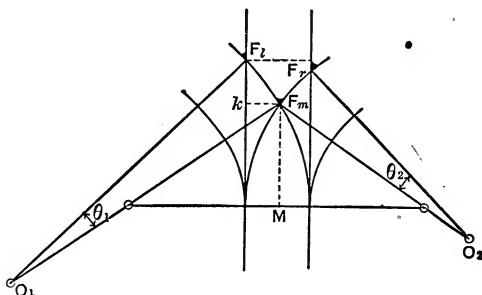


FIG. 144.

by separate applications of Eq. 102 or 103, the radius being determined by Eq. 104.

270. Two turnouts on the same side. In Fig. 145, let O_1 bisect O_2D . Then $(r_1 + \frac{1}{2}g) = \frac{1}{2}(r_2 + \frac{1}{2}g)$; also, $O_1O_2 = O_1F_l$ and $F_r = F_l$.

$$\text{vers } F_m = \frac{g}{r' + \frac{1}{2}g} = \frac{2g}{r + \frac{1}{2}g}; \quad (105)$$

$$BF_m = (r' + \frac{1}{2}g) \sin F_m. \quad (106)$$

It may readily be shown that the relative values of F_r , F_l , and F_m are almost identical with those given in § 269; as may be

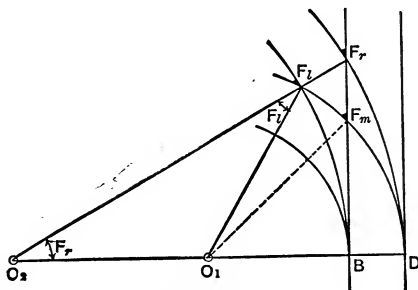


FIG. 145.

apparent when it is considered that the middle switch may be regarded simply as a curved main track, and that, as developed

in § 267, the dimensions of turnouts are nearly the same whether the main track is straight or slightly curved.

271. Connecting curve from a straight track. The "connecting curve" is the track lying between the frog and the side track where it becomes parallel to the main track (FS in Fig. 146 or 147). Call d the distance between track centers. The angle $FO_1R = F$ (see Fig. 146). Call r' the radius of the connecting curve. Then

$$(r' - \frac{1}{2}g) = \frac{d - g}{\text{vers } F}; \quad . \quad (107)$$

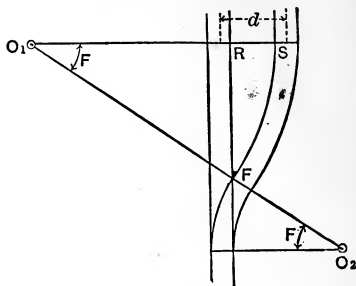


FIG. 146.

$$FR = (r' - \frac{1}{2}g) \sin F. \quad . \quad . \quad . \quad . \quad (108)$$

If it is considered that the distance FR consumes too much track room it may be shortened by the method indicated in Fig. 151.

272. Connecting curve from a curved track to the OUTSIDE. When the main track is curved, the required quantities are the

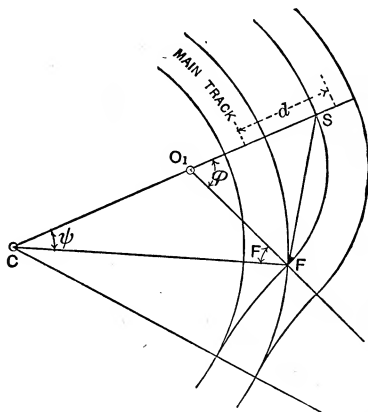


FIG. 147.

radius r of the connecting curve from F to S , Fig. 147, and its length or central angle. In the triangle CSF

$$CS + CF : CS - CF :: \tan \frac{1}{2}(CFS + CSF) : \tan \frac{1}{2}(CFS - CSF);$$

but $\frac{1}{2}(CFS + CSF) = 90 - \frac{1}{2}\psi$; and, since the triangle O_1SF is isosceles, $\frac{1}{2}(CFS - CSF) = \frac{1}{2}F$;

$$\begin{aligned} \therefore 2R + d : d - g &:: \cot \frac{1}{2}\psi : \tan \frac{1}{2}F \\ &:: \cot \frac{1}{2}F : \tan \frac{1}{2}\psi; \end{aligned}$$

$$\therefore \tan \frac{1}{2}\psi = \frac{2n(d-g)}{2R+d}. \quad \dots \dots \dots (109)$$

From the triangle CO_1F we may derive

$$r - \frac{1}{2}g : R + \frac{1}{2}g :: \sin \psi : \sin (F + \psi);$$

$$r - \frac{1}{2}g = (R + \frac{1}{2}g) \frac{\sin \psi}{\sin (F + \psi)}. \quad \dots \dots \dots (110)$$

Also $FS = 2(r - \frac{1}{2}g) \sin \frac{1}{2}(F + \psi). \quad \dots \dots \dots (111)$

273. Connecting curve from a curved track to the INSIDE.

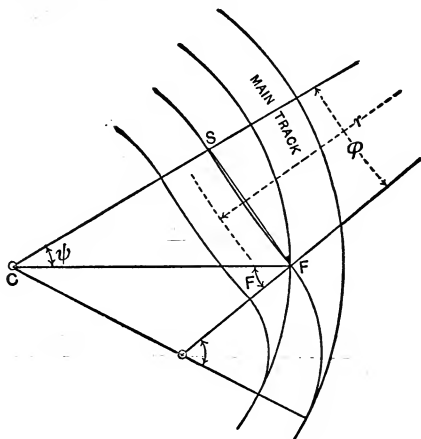


FIG. 148.

As above, it may readily be deduced from the triangle CFS (see Fig. 148) that

$$(2R - d) : (d - g) :: \cot \frac{1}{2}\psi : \tan \frac{1}{2}F,$$

and finally that

$$\tan \frac{1}{2}\psi = \frac{2n(d-g)}{2R-d}. \quad \dots \dots \dots (112)$$

Similarly we may derive (as in Eq. 110)

$$(r - \frac{1}{2}g) = (R - \frac{1}{2}g) \frac{\sin \psi}{\sin (F - \psi)}. \quad \dots \dots \dots (113)$$

Also $FS = 2(r - \frac{1}{2}g) \sin \frac{1}{2}(F - \phi) \dots \dots \dots (114)$

Two other cases are possible. (a) r may increase until it becomes infinite (see Fig. 149), then $F = \phi$. In such a case we may write, by substituting in Eq. 112,

$$2R - d = 4n^2(d - g) \dots (115)$$

This equation shows the value of R , which renders this case possible with the given values of n , d , and g . (b) ϕ may be greater than F . As before (see Fig. 150)

$$2R - d : d - g :: \cot \frac{1}{2}\psi : \tan \frac{1}{2}F;$$

$$\tan \frac{1}{2}\psi = \frac{2n(d - g)}{2R - d},$$

the same as Eq. 112, but

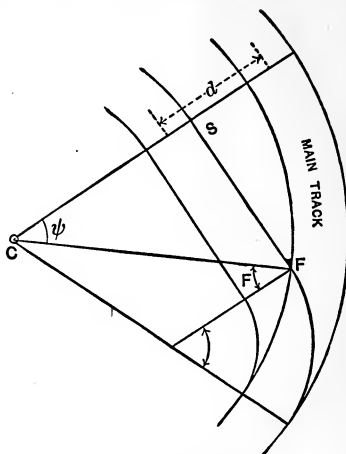


FIG. 149.

$$r + \frac{1}{2}g = (R - \frac{1}{2}g) \frac{\sin \phi}{\sin (\phi - F)} \dots \dots \dots (116)$$

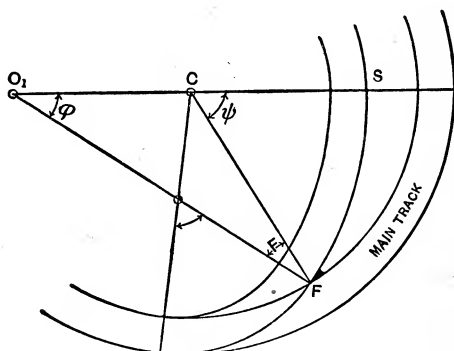


FIG. 150.

Problem. To find the dimensions of a connecting curve running to the *INSIDE* of a curved main track; number 9 frog, $4^\circ 30'$ curve, $d = 13'$, $g = 4' 8\frac{1}{2}''$.

Solution.

Eq. 112.

$$d = 13.000$$

$$g = 4.708$$

$$(d - g) = 8.292$$

$$R = 1273.6$$

$$2R = 2547.2$$

$$2R - d = 2534.2$$

$$\log (2R - d) = 3.40384$$

Eq. 116.

$$R = 1273.6$$

$$\frac{1}{2}g = 2.35$$

$$(R - \frac{1}{2}g) = 1271.25$$

$$(\Psi - F) = 1373'', \log = 3.13767$$

$$4.68557$$

$$\log \sin (\Psi - F) = 7.82324$$

Eq. 114.

$$\frac{1}{2}(\Psi - F) = 686.''5 \dots 2.83664$$

$$4.68557$$

$$\sin \frac{1}{2}(\Psi - F) = 7.52221$$

$$\log 2n = 1.25527$$

$$\log (d - g) = .91866$$

$$\text{co-log } (2R - d) = 6.59616$$

$$\log \tan \frac{1}{2}\Psi = 8.77009$$

$$\frac{1}{2}\Psi = 3^\circ 22' 14''$$

$$\Psi = 6^\circ 44' 28''$$

$$F = 6^\circ 21' 35''$$

$$(\Psi - F) = 0^\circ 22' 53''$$

$$\log (R - \frac{1}{2}g) = 3.10423$$

$$\log \sin \Psi = 9.06960$$

$$\text{co-log } \sin (\Psi - F) = 2.17676$$

$$(r + \frac{1}{2}g) = 22418.0 \dots 4.35059$$

$$r = 22415.6$$

$$d = 0^\circ 15'$$

$$2 \dots 0.30103$$

$$(r + \frac{1}{2}g) = 22418.0 \dots 4.35059$$

$$\sin \frac{1}{2}(\Psi - F) \dots 7.52221$$

$$FS = 149.22 \dots 2.17384$$

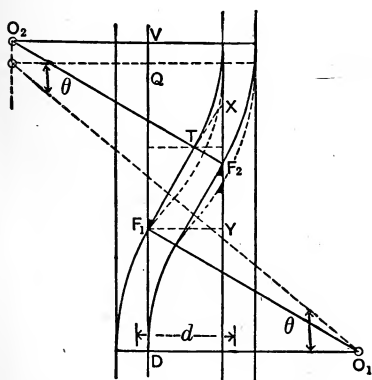
274. Crossover between two parallel straight tracks. (See

FIG. 151.

Fig. 151.) The turnouts are as usual. The crossover track may be straight, as shown by the full lines, or it may be a reversed curve, as shown by the dotted lines. The reversed curve shortens the total length of track required, but is somewhat objectionable. The first method requires that both frogs must be equal. The second method permits unequal frogs, although equal frogs

are preferable. The length of straight crossover track is F_1T .

$$F_1T \sin F_1 + g \cos F_1 = d - g;$$

$$F_1T = \frac{d - g}{\sin F_1} - g \cot F_1. \quad (117)$$

frog, and with a reversed curve between the frogs. Required the total distance between switch-points (the distance B_2N in Fig. 152).

Solution. If straight point rails and straight frog rails are used, the radii, r_1 and r_2 , taken from the middle section of Table III, are 527.91 and 681.16.

Eq. 122.		$\text{vers } \theta = \frac{d}{r_1 + r_2}$
$r_1 = 527.91$	$d = 12' 2'' = 12.16,$	$\log = 1.08517$
$r_2 = 681.16$		$\log (r_1 + r_2) = 3.08245$
$r_1 + r_2 = 1209.07$		$\log \text{vers } \theta = 8.00272$
	$\theta = 8^\circ 08' 06''$	
Eq. 122.		$\log (r_1 + r_2) = 3.08245$
		$\log \sin \theta = 9.15077$
		$\log 171.09 = 2.23322$
	$B_2N = 171.09$	

The length of the curve from $B_2 = 100(\theta \div d) = 100(8^\circ 08' 06'' \div 8^\circ 25') = 96.65$. The length of the other curve is $100(8^\circ 08' 06'' \div$

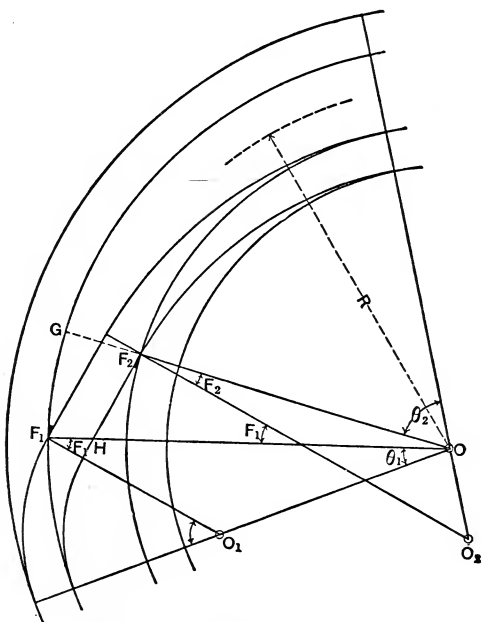


FIG. 153.

$10^\circ 52') = 74.86$. As a check, $96.65 + 74.86 = 171.51$, which is slightly in excess of 171.09, as it should be.

275. Crossover between two parallel curved tracks. (a) Using a straight connecting curve. This solution has limitations. If one frog (F_1) is chosen, F_2 becomes determined, being a function of F_1 . If F_1 is less than some limit, depending on the width (d) between the parallel tracks, this solution becomes impossible. In Fig. 153 assume F_1 as known. Then $F_1H = g \sec F_1$. In the triangle $HO F_2$ we have

$$\sin HF_2O : \sin F_2HO :: HO : F_2O;$$

$$\sin F_2HO = \cos F_1; \quad HF_2O = 90^\circ + F_2;$$

$$\therefore \sin HF_2O = \cos F_2.$$

$$HO = R + \frac{1}{2}d - \frac{1}{2}g - g \sec F_1; \quad F_2O = R - \frac{1}{2}d + \frac{1}{2}g;$$

$$\therefore \cos F_2 = \cos F_1 \frac{R + \frac{1}{2}d - \frac{1}{2}g - g \sec F_1}{R - \frac{1}{2}d + \frac{1}{2}g}. \quad \dots \dots (123)$$

Knowing F_2 , θ_2 is determinable from Eq. 91. Fig. 153 shows the case where θ_2 is greater than F_2 . Fig. 154 shows the case where it is less. The demonstration of Eq. 123 is applicable to

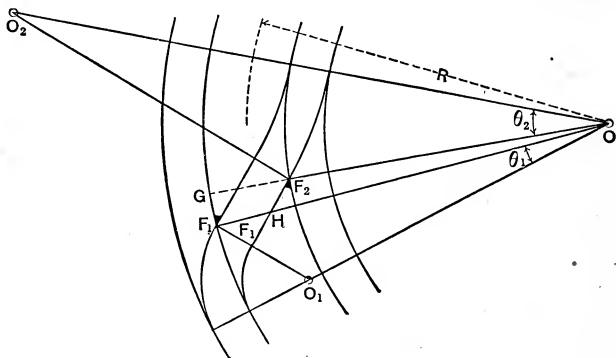


FIG. 154.

both figures. The relative position of the frogs F_1 and F_2 may be determined as follows, the solution being applicable to both Figs. 153 and 154:

$$HO F_2 = 180^\circ - (90^\circ - F_1) - (90^\circ + F_2) = F_1 - F_2.$$

Then

$$GF_1 = 2(R + \frac{1}{2}d - \frac{1}{2}g) \sin \frac{1}{2}(F_1 - F_2). \quad \dots \dots (124)$$

Since F_2 comes out *any* angle, its value will not be in general that of an even frog number, and it will therefore need to be made to order.

(b) Continuing the switch-rail curves until they meet as a reversed curve. In this case F_1 and F_2 may be chosen at pleasure (within limitations), and they will of course be of regular sizes and equal or unequal as desired. F_1 and F_2 being known, θ_1 and θ_2 are computed by Eq. 95 and 91. In the triangle OO_1O_2 (see Fig. 155)

$$\text{vers } \phi = \frac{2(S - OO_2)(S - OO_1)}{(OO_2)(OO_1)},$$

in which

$$S = \frac{1}{2}(OO_1 + OO_2 + O_1O_2);$$

but

$$OO_1 = R + \frac{1}{2}d - r_1,$$

$$OO_2 = R - \frac{1}{2}d + r_2,$$

$$O_1O_2 = r_1 + r_2;$$

$$\therefore S = \frac{1}{2}(2R + 2r_2) = R + r_2;$$

$$S - OO_2 = R + r_2 - R + \frac{1}{2}d - r_2 = \frac{1}{2}d;$$

$$S - OO_1 = R + r_2 - R - \frac{1}{2}d + r_1 = r_1 + r_2 - \frac{1}{2}d;$$

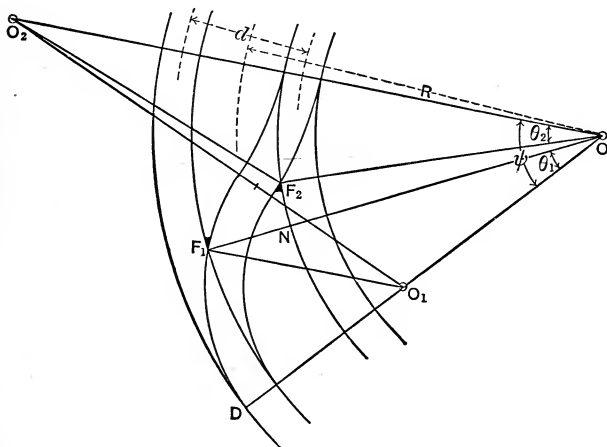


FIG. 155.

$$\text{vers } \phi = \frac{d(r_1 + r_2 - \frac{1}{2}d)}{(R - \frac{1}{2}d + r_2)(R + \frac{1}{2}d - r_1)}; \quad \dots \quad (125)$$

$$\sin OO_2O_1 = \sin \phi \frac{OO_1}{O_1O_2} = \sin \phi \frac{R + \frac{1}{2}d - r_1}{r_1 + r_2}; \quad \dots \quad (126)$$

$$O_2O_1D = \phi + O_1O_2O; \quad \dots \quad (127)$$

$$NF_2 = 2(R - \frac{1}{2}d + \frac{1}{2}g) \sin \frac{1}{2}(\phi - \theta_1 - \theta_2). \quad \dots \quad (128)$$

Although the above method introduces a reversed curve, yet it uses up less track than the first method and permits the use of ordinary frogs rather than those having some special angle which must be made to order. But the above solution implies the use of circular lead rails. We may compute dimensions and lay track between F_1 and F_2 on this basis and then change the switch rails as desired. Strictly, r_1 and r_2 should be computed by Eq. 92 and 96, but for an easy main-line curve the approximate rule is sufficiently accurate.

Problem.—Required the dimensions of a crossover on a $4^\circ 30'$ curve when the distance between track centers is 13 feet. The frog for the outer main track (F_1 in Fig. 155) is No. 9; F_2 is No. 7. Then $R=1273.6$; R_1 , for the outer main track, $=1280.1$; $D_1=4^\circ 29'$; $R_2=1267.1$; $D_2=4^\circ 31'$; r_1 =radius for $(d_1+D_1)^\circ$ curve=radius for $(7^\circ 31'+4^\circ 29')$ curve=478.34; r_2 =radius for $(d_2-D_2)^\circ$ curve=radius for $(12^\circ 26'-4^\circ 31')$ curve=724.31. (See §§ 267-268.)

Eq. 125	$d=13$	$\log=1.11394$
$r_1+r_2-\frac{1}{2}d=1196.15$		$\log=3.07778$
$R-\frac{1}{2}d+r_2=1991.31$	$\log=3.30914$	$\text{colog}=6.69086$
$R+\frac{1}{2}d-r_1=801.76$	$\log=2.90404$	$\text{colog}=7.09595$
$\phi=7^\circ 52' 26''$		$\log \text{vers } \phi=7.97854$

Eq. 126		$\log \sin \phi=9.13670$
$r_1+r_2=1202.65$	$\log=3.08013$	$\log(R+\frac{1}{2}d-r_1)=2.90404$
	$OO_2O_1=5^\circ 14' 24''$	$\text{colog}=6.91986$
		$\sin OO_2O_1=8.96061$

Eq. 127 $\underline{O_2O_1D=7^\circ 52' 26''+5^\circ 14' 24''=13^\circ 06' 50''}$

Eq. 91	$\tan \frac{1}{2}\theta_1=\frac{gn}{R}=\frac{4.708 \times 9}{1280.1}=\frac{42.372}{1280.1}$	$\log=1.62708$
		$\log=3.10724$
	(Using Table VI)	$\log \tan \frac{1}{2}\theta_1=8.51983$
	$\theta_1=3^\circ 47' 30''$	5.31426
	$\frac{1}{2}\theta_1=1^\circ 53' 45''$	$\log 6825=3.83410$

Eq. 95	$\frac{gn}{R}=\frac{4.708 \times 7}{1267.1}=\frac{32.956}{1267.1}$	$\log=1.51793$
		$\log=3.10281$
		$\tan \frac{1}{2}\theta_2=8.41512$
		5.31433
	$\theta_2=2^\circ 58' 48''$	$\log 5364=3.72945$

Eq. 128	$R-\frac{1}{2}d+\frac{1}{2}g=1269.45$	$2 \log=0.30103$
		$\log=3.10361$
	$\frac{1}{2}(\phi-\theta_1-\theta_2)=1^\circ 06' 08''$	$\log \sin=4.68555$
		3.59857
	$\underline{NF_2=48.84}$	$\log 48.84=1.68876$

Length of curve with radius $r_1 = 100 \frac{13^\circ 06' 50''}{12^\circ 0'} = 109.18$
" " " " " $r_2 = 100 \frac{5^\circ 14' 24''}{7^\circ 55'} = 66.16$

Total length of curve between switch points = 175.34

As an approximate check, the mean length subtending the angle ψ with radius R is similarly computed as 174.98. Note that the length of the curve with the radius r_2 is 66.16, which is but little more than the length of lead rails (65.92) for a No. 7 frog using circular lead rails, which means that the point of reversed curve is but little beyond the frog point. If the computations had apparently indicated the point of reversed curve coming between the frog point and the switch point, it would have shown the impracticability of the combination of No. 7 and No. 9 frogs with this particular degree of curve, gauge of track, and distance between track centers. If both frogs were made No. 9, the total length of track between switch points would be increased to over 198 feet and the point of reversed curve would be nearly at the middle point. This shows that the frog numbers should be nearly equal, but also shows that there is some choice "within limitations."

276. Practical rules for switch-laying. A consideration of the previous sections will show that the formulæ are comparatively simple when the lead rails are assumed as circular; that they become complicated, even for turnouts from a straight main track, when the effect of straight frog and point rails is allowed for, and that they become hopelessly complicated when allowing for this effect on turnouts from a curved main track. It is also shown (§ 267) that the length of the lead is practically the same whether the main track is straight or is curved with such curves as are commonly used, and that the degree of curve of the lead rails from a curved main track may be found with close approximation by mere addition or subtraction. From this it may be assumed that if the length of lead (L) and the radius of the lead rails (r) are computed from Eq. 87 and 90 for various frog angles, the same leads may be used for curved main track; also, that the degree of curve of the lead rails may be found by addition or subtraction, as indicated in § 267, and that the approximations involved will not be of practical detriment. In accordance with this plan Table III has been computed from

Eq. 87, 88, and 90. The *leads* there given may be used for all main tracks, straight or curved. The table gives the degree of curve of the lead rails for *straight* main track; for a turnout to the *inside*, add the degree of curve of the main track; for a turnout to the *outside*, subtract it.

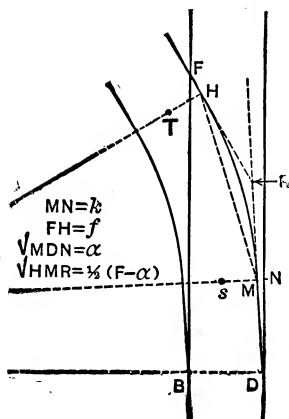


FIG. 140.

But there are complications resulting from practical and economical switch construction. A committee of the A. R. E. A. in 1910 adopted certain standards in details, which when applied to Eqs. 87 to 90 give the values for switch dimensions as quoted in the second section of Table III. They adopted four lengths of switch rails. In each case the "point" is always $\frac{1}{4}$ " thick. The gauge line at the other end is always to be placed $6\frac{1}{4}$ " from the gauge line of the main rail, and the planing is so done that when in this position the switch rail lies against the main rail. Therefore the angle α is always an angle whose sine equals 6 inches (or 0.5 foot) divided by the length of the switch rail in feet. In Fig. 140, the point *D* is not on the gauge line of the main rail but at a point $\frac{1}{4}$ " away from it, and the point *M* $6\frac{1}{4}$ " away from it. The straight rail *BF* consists of a point rail at one end, the "closure rails," and one of the wing rails of the frog at the other end. The closure rails will in general consist of one rail cut to a computed length and one or more rails from 24 to 33 feet long, the lengths being in even feet. The curved rail *DF* will also consist of a point rail, a frog wing-rail, and one or more lengths of closure rail, but the closure rails in this case are slightly longer than those for the straight rail. Since it is always practically easier to measure to the "actual point" of a frog (see Fig. 130) rather than to the theoretical point, Table III gives the distance *L'*, which is the distance *L*, = *BF*, plus the "frog bluntness," which is found by multiplying $\frac{1}{2}$ " (=0.0417 foot) by the frog number.

The curvature for a curved switch rail (for a straight track)

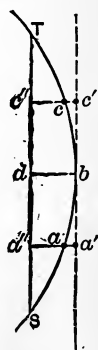


FIG. 156.

ORDINATES OF CURVED SWITCH RAIL.

Frog No.	Center.		Quarter points.	
	Ft.	Ins.	Ft.	Ins.
4	.59	7 $\frac{1}{2}$	.44	5 $\frac{3}{8}$
5	.55	6 $\frac{3}{4}$	.41	5
6	.50	6	.38	4 $\frac{1}{2}$
7	.58	7	.43	5 $\frac{1}{4}$
8	.55	6 $\frac{1}{2}$	.41	5
9	.50	6	.38	4 $\frac{1}{2}$
9 $\frac{1}{2}$	.49	5 $\frac{7}{8}$	.37	4 $\frac{3}{8}$
10	.48	5	.36	4 $\frac{3}{4}$
11	.55	6 $\frac{1}{4}$	.41	4 $\frac{7}{8}$
12	.52	6 $\frac{1}{2}$	.39	4 $\frac{3}{4}$
15	.61	7	.46	5 $\frac{1}{4}$
18	.53	6 $\frac{3}{4}$	.40	4 $\frac{3}{4}$
20	.50	6	.37	4 $\frac{1}{2}$
24	.44	5 $\frac{1}{2}$	.33	4

may be determined by measuring offsets at the quarter points from a string stretched between the points *H* and *M*. These offsets are as given in the tabular form. The distance *HM* may be computed from the equation in § 265.

If the position of the switch-block is definitely determined, then the rails must be cut accordingly; but when some freedom is allowable (which never need exceed 16.5 feet and may require but a few inches), one rail-cutting may be avoided. Mark on the rails at *B*, *F*, and *D*; measure off the length of the switch-rails *DN* and locate the point

M at the distance *k* from *N*. If the frog must be placed during the brief period between the running times of trains, it will be easier to joint up to the frog a piece of rail at one or both ends of just such a length that they may be quickly substituted for an equal length of rail taken out of the track. When the frog is thus in place the point *H* becomes located. The chord *MH* may be measured on the ground as a check. The curve between *M* and *H* is of known radius. Substituting in Eq. 31 the value of *chord* and *R*, we may compute *x*(=*db*). Locate the middle point *d* and the quarter points *a''* and *c''*. Then *a''a* and *c''c* each equal three-fourths of *db*. Theoretically this gives a parabola rather than a circle, but the difference for all practical cases is too small for measurement. See Fig. 156.

Example.—Given a main track on a 4° curve—a turnout to the outside, using a No. 9 frog; gauge 4' 8 $\frac{1}{2}$ "; *f*=6'.00; *k*=6 $\frac{1}{4}$ "; *DN*=16' 6" and *a*=1° 44' 11". Then for a *straight* track *r* would equal 616.27 [*d*=9° 18' 27"]. For this curved track *d* will be nearly 9° 18'—4°=5° 18', or *r* will be 1081.4. *L* for the *straight* track would be 72.61; but since the lead is slightly increased (see §267) when the turnout is on the outside of a curve, *L* may here be called 72.9. *H* and *M* may be located as described above. *HM* may be measured on the ground as a check on the value (in this case 49.90) computed as in § 265. Since it is slightly more for a turnout to the outside of a curve, it may be called 50.00. Then, for the outer rail, $x=db = \frac{(50.0)^2}{8 \times 1083.75} = 0.288$ feet, and *aa''* and *cc''*=0.216 foot.

276a. Slips. Track movements in crowded yards are facilitated by using "slips" (see Fig. 156a), which may be "single" or "double." The crossing of two rails is done either by operating two movable rails or by using fixed "frogs," but a comparison of the continuity of the running rails, using ordinary frogs (see Fig. 130) and these frogs, will show their radical difference. These slips can be used for frog angles from No. 6 to No. 15. The levers are so connected that the several operations necessary to set the rails for any desired train movement are accomplished by one motion.

CROSSINGS.

277. Two straight tracks. When two straight tracks cross each other, four frogs are necessary, the angles of two of them being supplementary to the angles of the other. Since such crossings are sometimes operated at high speeds, they should be

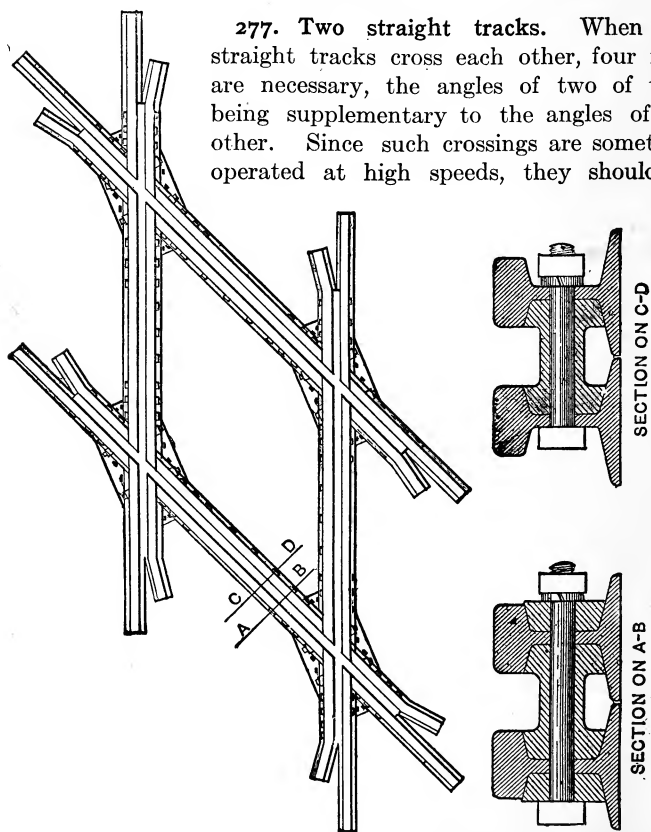


FIG. 157.—Crossing.

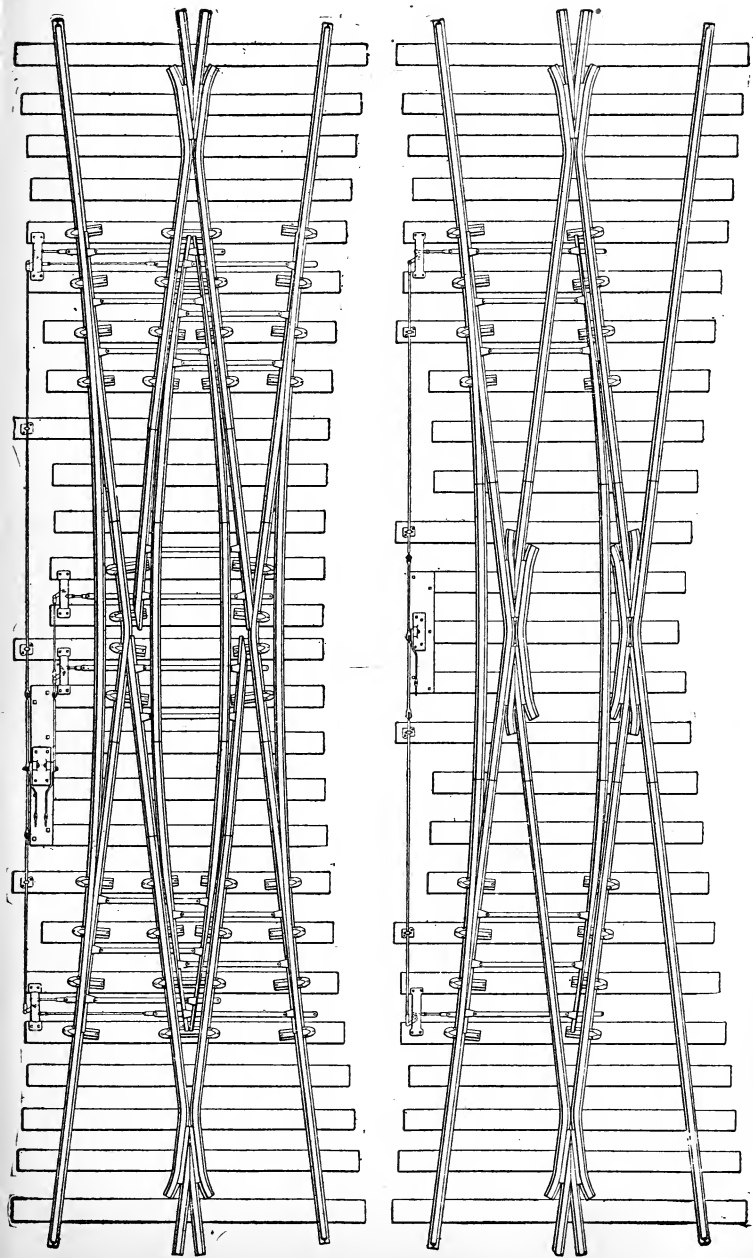


FIG. 156a.—SINGLE AND DOUBLE SLIPS.

very strongly constructed, and the angles should preferably be 90° or as near that as possible. The frogs will not in general be "stock" frogs of an even number, especially if the angles are large, but must be made to order with the required angles as measured. In Fig. 157 are shown the details of such a crossing. Note the fillers, bolts, and guard-rails.

278. One straight and one curved track. Structurally the crossing is about the same as above, but the frog angles are all unequal. In Fig. 158, R is known, and the angle M , made by the center lines of the tracks at their point of intersection, is also known.

$$M = NCM. \quad NC = R \cos M.$$

$$(R - \frac{1}{2}g) \cos F_1 = NC + \frac{1}{2}g;$$

$$\therefore \cos F_1 = \frac{R \cos M + \frac{1}{2}g}{R - \frac{1}{2}g}.$$

Similarly

$$\cos F_2 = \frac{R \cos M + \frac{1}{2}g}{R + \frac{1}{2}g}, \quad (129)$$

$$\cos F_3 = \frac{R \cos M - \frac{1}{2}g}{R + \frac{1}{2}g},$$

$$\cos F_4 = \frac{R \cos M - \frac{1}{2}g}{R - \frac{1}{2}g}.$$

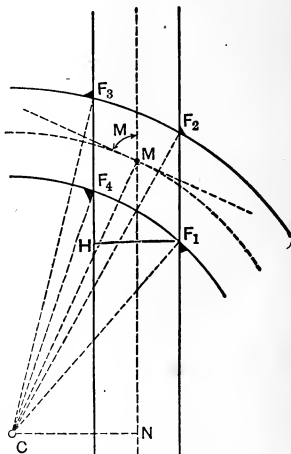


FIG. 158.

$$\left. \begin{aligned} F_3 F_4 &= (R + \frac{1}{2}g) \sin F_3 - (R - \frac{1}{2}g) \sin F_4; \\ H F_4 &= (R - \frac{1}{2}g)(\sin F_4 - \sin F_1). \end{aligned} \right\} \dots (130)$$

279. Two curved tracks. The four frogs are unequal, and the angle of each must be computed. The radii R_1 and R_2 are known; also the angle M . r_1, r_2, r_3 , and r_4 are therefore known by adding or subtracting $\frac{1}{2}g$, but the lines are so indicated for brevity. Call the angle $MC_1C_2 = C_1$, the angle $MC_2C_1 = C_2$, and the line $C_1C_2 = c$. Then

$$\frac{1}{2}(C_1 + C_2) = 90^\circ - \frac{1}{2}M$$

and

$$\tan \frac{1}{2}(C_1 - C_2) = \cot \frac{1}{2}M \frac{R_2 - R_1}{R_2 + R_1}. \quad (131)$$

C_1 and C_2 then become known and

$$c = C_1 C_2 = R_2 \frac{\sin M}{\sin C_1}. \quad (132)$$

In the triangle $F_1C_1C_2$, call $\frac{1}{2}(c+r_1+r_4)=s_1$; $s_2=\frac{1}{2}(c+r_2+r_4)$;

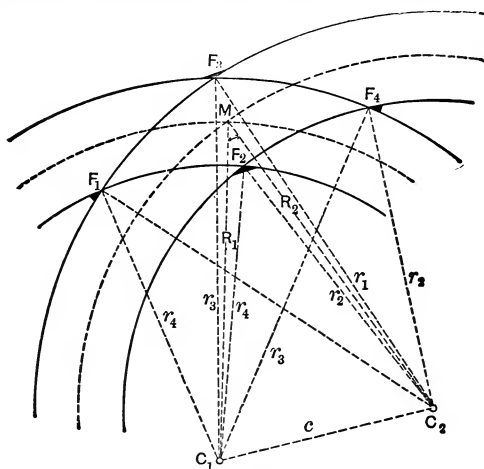


FIG. 159.

$s_3=\frac{1}{2}(c+r_1+r_3)$; and $s_4=\frac{1}{2}(c+r_2+r_3)$. Then, by formula 29, Table XXX,

$$\left. \begin{aligned} \text{vers } F_1 &= \frac{2(s_1-r_1)(s_1-r_4)}{r_1r_4} \\ \text{vers } F_2 &= \frac{2(s_2-r_2)(s_2-r_4)}{r_2r_4} \\ \text{vers } F_3 &= \frac{2(s_3-r_1)(s_3-r_3)}{r_1r_3} \\ \text{vers } F_4 &= \frac{2(s_4-r_2)(s_4-r_3)}{r_2r_3} \end{aligned} \right\} \dots \dots (133)$$

Similarly

$$\begin{aligned} \sin C_1C_2F_4 &= \sin F_4 \frac{r_3}{c}; \\ \sin C_1C_2F_2 &= \sin F_2 \frac{r_4}{c}; \\ \therefore F_2C_2F_4 &= C_1C_2F_4 - C_1C_2F_2, \dots \dots (134) \\ \sin F_1C_1C_2 &= \sin F_1 \frac{r_1}{c}; \\ \sin F_2C_1C_2 &= \sin F_2 \frac{r_2}{c}, \end{aligned}$$

$$\therefore F_1C_1F_2 = F_1C_1C_2 - F_2C_1C_2; \dots \dots (135)$$

from which the chords F_1F_2 and F_2F_4 are readily computed.

F_1F_2 and F_2F_4 are nearly equal. When the tracks are straight and the gauges equal, the quadrilateral is equilateral.

Problem. Required the frog angles and dimensions for a crossing of two curves ($D_1=4^\circ$; $D_2=3^\circ$) when the angle of their tangents at the point of intersection $=62^\circ 28'$ (the angle M in Fig. 159).

Solution

$$\begin{aligned} R_1 &= 1432.7; R_2 = 1910.1; \\ r_1 &= R_2 + \frac{1}{2}g = 1910.1 + 2.35 = 1912.45; \\ r_2 &= R_2 - \frac{1}{2}g = 1910.1 - 2.35 = 1907.75; \\ r_3 &= R_1 + \frac{1}{2}g = 1432.7 + 2.35 = 1435.05; \\ r_4 &= R_1 - \frac{1}{2}g = 1432.7 - 2.35 = 1430.35. \end{aligned}$$

Eq. 131. $\log \cot \frac{1}{2}M = 0.21723$

$$R_2 - R_1 = 477.4; \log = 2.67888$$

$$R_2 + R_1 = 3342.8; \log = 3.52411; \text{co-log} = 6.47589$$

$$\frac{1}{2}(C_1 - C_2) = 13^\circ 15' 07''; \tan 13^\circ 15' 07'' = 9.37200$$

$$\frac{1}{2}(C_1 + C_2) = 58^\circ 46' \quad \left[\frac{1}{2}(C_1 + C_2) = 90^\circ - \frac{1}{2}M \right]$$

$$C_1 = 72^\circ 01' 07''$$

$$C_2 = 45^\circ 30' 53''$$

Eq. 132. $\log R_2 = 3.28105$

$$\log \sin M = 9.94779$$

$$\log \sin C_1 = 9.97825; \text{co-log} = 0.02175$$

$$c = C_1 C_2 = 1780.7; \log C_1 C_2 = 3.25059$$

Eq. 133.

$c = 1780.7$	$c = 1780.7$	$c = 1780.7$	$c = 1780.7$
$r_1 = 1912.45$	$r_2 = 1907.75$	$r_1 = 1912.45$	$r_2 = 1907.75$
$r_4 = 1430.35$	$r_4 = 1430.35$	$r_3 = 1435.05$	$r_3 = 1435.05$
$\underline{2} \overline{5123.50}$	$\underline{2} \overline{5118.80}$	$\underline{2} \overline{5128.20}$	$\underline{2} \overline{5123.50}$
$s_1 = 2561.75$	$s_2 = 2559.40$	$s_3 = 2564.10$	$s_4 = 2561.75$
$s_1 - r_1 = 649.30$	$s_2 - r_2 = 651.65$	$s_3 - r_1 = 651.65$	$s_4 - r_2 = 654.00$
$s_1 - r_4 = 1131.40$	$-r_4 = 1129.05$	$s_3 - r_3 = 1129.05$	$s_4 - r_3 = 1126.70$

$$\log 2 = 0.30103$$

$$(s_1 - r_1); \log 649.30 = 2.81244$$

$$(s_1 - r_4); \log 1131.40 = 3.05361$$

$$\text{co-log} = 6.71841$$

$$\text{co-log} = 6.84456$$

$$\log \text{vers } 62^\circ 25' 31'' = 9.73006$$

$$\log 2 = 0.30103$$

$$(s_2 - r_2); \log 651.65 = 2.81401$$

$$(s_2 - r_4); \log 1129.05 = 3.05271$$

$$\text{co-log} = 6.71948$$

$$\text{co-log} = 6.84456$$

$$\log \text{vers } 62^\circ 33' 55'' = 9.73180$$

$$r_1 = 1912.45; \log = 3.28159;$$

$$r_4 = 1430.35; \log = 3.15544;$$

$$\underline{F_1 = 62^\circ 25' 31''};$$

$$r_2 = 1907.75; \log = 3.28052;$$

$$r_4 = 1430.35; \log = 3.15544;$$

$$\underline{F_2 = 62^\circ 33' 55''};$$

$$\begin{array}{rcl}
 r_1 = 1912.45; & \log = 3.28159; & \log 2 = 0.30103 \\
 r_3 = 1435.05; & \log = 3.15686; & (s_3 - r_1); \log 651.65 = 2.8140\bar{1} \\
 & & (s_3 - r_3); \log 1129.05 = 3.0527\bar{1} \\
 & & \text{co-log} = 6.7184\bar{1} \\
 & & \text{co-log} = 6.8431\bar{3} \\
 \underline{F_3 = 62^\circ 21' 57''}; & & \log \text{ vers } 62^\circ 21' 57'' = 9.7293\bar{0} \\
 & & \log 2 = 0.30103 \\
 r_2 = 1907.75; & \log = 3.28052; & (s_4 - r_2); \log 654.00 = 2.81558 \\
 r_3 = 1435.05; & \log = 3.15686; & (s_4 - r_3); \log 1126.70 = 3.0518\bar{1} \\
 & & \text{co-log} = 6.7194\bar{8} \\
 & & \text{co-log} = 6.8431\bar{3} \\
 \underline{F_4 = 62^\circ 30' 14''}; & & \log \text{ vers } 62^\circ 30' 14'' = 9.7310\bar{3}
 \end{array}$$

As a check, the *mean* of the frog angles = $62^\circ 27' 54''$, which is within $6''$ of the value of M .

Eq. 134.

$$\begin{array}{rcl}
 & \log c = 3.2505\bar{9}; & \log \sin F_4 = 9.9479\bar{4} \\
 & & \log r_3 = 3.1568\bar{6} \\
 C_1 C_2 F_4 = 45^\circ 37' 51''; & & \text{co-log } c = 6.7494\bar{0} \\
 & & \sin C_1 C_2 F_4 = 9.8542\bar{1} \\
 & & \log \sin F_2 = 9.9481\bar{8} \\
 & & \log r_4 = 3.1554\bar{4} \\
 & & \text{co-log } c = 6.7494\bar{0} \\
 C_1 C_2 F_2 = 45^\circ 28' 17''; & & \sin C_1 C_2 F_2 = 9.8530\bar{3} \\
 \underline{F_2 C_2 F_4 = 45^\circ 37' 51'' - 45^\circ 28' 17'' = 0^\circ 09' 34''}. & &
 \end{array}$$

$$\begin{array}{rcl}
 & \log 2 = 0.30103 \\
 & \log r_2 = 3.2805\bar{2} \\
 \frac{1}{2}(0^\circ 09' 34'') = 0^\circ 04' 47''; & \log \sin = \left(\begin{array}{l} 4.6855\bar{7} \\ 2.4578\bar{8} \end{array} \right. & \\
 \underline{F_2 F_4 = 5.309}; & & \log F_2 F_4 = 0.7250\bar{0} \\
 \text{Eq. 135.} & & \sin F_1 = 9.9476\bar{3} \\
 & & \log r_1 = 3.2815\bar{9} \\
 & & \text{co-log } c = 6.7494\bar{0}
 \end{array}$$

$$\begin{array}{rcl}
 F_1 C_1 C_2 = 72^\circ 10' 22''; & & \sin F_1 C_1 C_2 = 9.9786\bar{3} \\
 & & \sin F_2 = 9.9481\bar{8} \\
 & & \log r_2 = 3.2805\bar{2} \\
 & & \text{co-log } c = 6.7494\bar{0} \\
 F_2 C_1 C_2 = 71^\circ 57' 38''; & & \sin F_2 C_1 C_2 = 9.9781\bar{1} \\
 \underline{F_1 C_1 F_2 = 72^\circ 10' 22'' - 71^\circ 57' 38'' = 0^\circ 12' 44''}. & & \\
 & & \log 2 = 0.30103 \\
 & & \log r_4 = 3.1554\bar{4} \\
 \frac{1}{2}(0^\circ 12' 44'') = 0^\circ 06' 22''; & \log \sin = \left(\begin{array}{l} 4.6855\bar{7} \\ 2.5820\bar{6} \end{array} \right. & \\
 \underline{F_1 F_2 = 5.298}; & & \log F_1 F_2 = 0.7241\bar{1}
 \end{array}$$

As a check, $F_2 F_4$ and $F_1 F_2$ are very nearly equal, as they should be.

CHAPTER XII.

MISCELLANEOUS STRUCTURES AND BUILDINGS.

WATER-STATIONS AND WATER-SUPPLY.

280. Location. The water-tank on the tender of a locomotive has a capacity of from 3000 to 7000 gallons—sometimes less, rarely very much more. The consumption of water is very variable, and will correspond very closely with the work done by the engine. On a long down grade it is very small; on a ruling grade going up it may amount to 150 gallons per mile in exceptional cases, although 60 to 100 gallons would be a more usual figure. A passenger locomotive can run 60 miles or more on one tankful, but freight work requires a shorter interval between water-stations. On roads of the smallest traffic, 15 to 20 miles should be the maximum interval between stations; 10 miles is a more common interval on heavy traffic-roads. But these intervals are varied according to circumstances. In the early history of some of the Pacific railroads it was necessary to attach one or more tank-cars to each train in order to maintain the supply for the engine over stretches of 100 miles and over where there was no water. Since then water-stations have been obtained at great expense by boring artesian wells. The individual locations depend largely on the facility with which a sufficient supply of suitable water may be obtained. Streams intersecting the railroad are sometimes utilized, but if such a stream passes through a limestone region the water is apt to be too hard for use in the boilers. More frequently wells are dug or bored. When the local supply at some determined point is unsuitable, and yet it is necessary to locate a water-station there, it may be found justifiable to pipe the water several miles. The construction of municipal water-works at suitable places along the line has led to the frequent utilization of such supplies. In such cases the railroad is generally the largest single consumer and obtains the most favorable rates. When possible, water-stations are located at regular stopping points and at division termini.

281. Required qualities of water. Chemically pure water is unknown except as a laboratory product. The water supplied by wells, springs, etc., is always more or less charged with calcium and magnesium carbonates and sulphates, as well as other impurities. The evaporation of water in a boiler precipitates these impurities to the lower surfaces of the boiler, where they sometimes become incrustated and are difficult to remove. The protection of the iron or steel of a boiler from the fierce heat of the fire depends on the presence of water on the other side of the surface, which will absorb the heat and prevent the metal from assuming an excessively high temperature. If the water side of the metal becomes covered or incrustated with a deposit of chemicals, the conduction of heat to the water is much less free, the metal will become more heated and its deterioration or destruction will be much more rapid. An especially common effect is the production of leaks around the joints between tubes and tube-sheets and the joints in the boiler-plates. Such injury can only be prevented by the application of one (or both) of two general methods—(a) the frequent cleaning of the boilers and (b) the chemical purification of the water before its introduction into the boiler. Although “manholes” and “hand-holes” are made in boilers, it is physically impossible to clean out every corner of the inside of a boiler where deposits will form and where they are especially objectionable—on the tube-sheets. Such a cleaning is troublesome and expensive.

Chemical purification is generally accomplished by treating the water before it enters the boiler. The reagents chiefly employed are quicklime and sodium carbonate. Lime precipitates the bicarbonate of lime and magnesia. Sodium carbonate gives, by double decomposition in the presence of sulphate of lime, carbonate of lime, which precipitates, and soluble sulphate of soda, which is non-incrustant. When this is done in a purifying tank, the purified water is drawn off from the top of the tank and supplied pure to the engines. The precipitants are drawn off from the settling-basin at the bottom of the tank. This purification, which makes no pretense of being chemically perfect, may be accomplished for a few cents per 1000 gallons. It is used much more extensively in Europe than in this country, the Southern Pacific being the only railroad which has employed such methods on a large scale. Reliance is frequently placed on the employment of a “non-incrustant” which is introduced

directly into the boiler. When no incrustation takes place the accumulation of precipitant and mud in the bottom of the boiler may be largely removed by mere "blowing off" or by washing out with a hose.

On the other hand, there is the exceptional case that the water may be too pure. It is well known that distilled water has a very strong corrosive action on iron and that it is possible for the water to be so pure that corrosion of the boiler tubes will be accelerated and that the boilers will rapidly deteriorate in this way. It is therefore occasionally necessary to add a small portion of lime to a very soft water, so that a very thin scale will form over the surface of the iron, which will protect the iron from corrosion.

American practice may therefore be summarized as follows: (a) Employing as pure water as possible; (b) cleaning out boilers by "blowing off" or by washing out with a hose or by physical scraping at more or less frequent intervals or when other repairs are being made; (c) the occasional employment of non-incrustants; (d) the occasional chemical treatment of water before it enters the tender-tank.

282. Tanks. Whatever the source, the water must be led or pumped into tanks which are supported on frames so that the

bottoms of the tanks are about 12 feet above the rails. Wooden tanks having a diameter of 24 feet, 16 feet high, and with a capacity of over 50,000 gallons, are frequently employed. Iron or steel tanks are also used.

In Table XIV is shown the capacity of cylindrical water-tanks in United States standard gallons of 231 cubic inches. From this table the dimensions of a tank of any desired capacity may

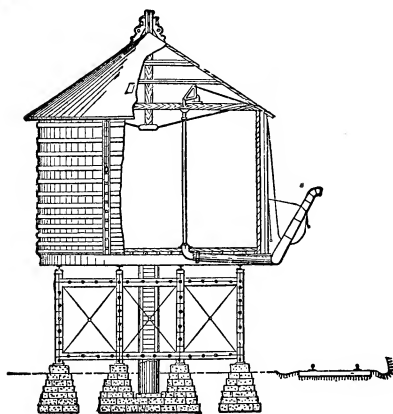


FIG. 160.—WATER-TANK.

readily be found. Two or more tanks are sometimes used rather than construct one of excessive size. The smaller sizes

TABLE XIV.—CAPACITY OF CYLINDRICAL WATER-TANKS IN UNITED STATES STANDARD GALLONS OF 231 CUBIC INCHES.

Height in feet.	Diameter of tank in feet.							
	10	12	14	16	18	20	22	24
6	3525	5076	6909	9024	11421	14101	17062	20305
7	4113	5922	8061	10528	13325	16451	19905	23689
8	4700	6768	9212	12032	15229	18801	22749	27073
9	5288	7614	10364	13536	17132	21151	25592	30457
10	5875	8460	11515	15041	19036	23501	28436	33841
11	6463	9306	12667	16545	20939	25851	31280	37225
12	7050	10152	13819	18049	22843	28201	34123	40609
13	7638	10998	14970	19553	24746	30551	36967	43994
14	8225	11844	16122	21057	26650	32901	39810	47378
15	8813	12690	17273	22561	28554	35251	42654	50762
16	9400	13536	18425	24065	30457	37601	45498	54146
17	9988	14383	19576	25569	32361	39951	48341	57530
18	10575	15229	20728	27073	34264	42301	51185	60914
19	11163	16075	21879	28577	36168	44652	54028	64298
20	11750	16921	23031	30081	38071	47002	56872	67682
21	12338	17767	24182	31585	39975	49352	59716	71067
22	12925	18613	25334	33089	41879	51702	62559	74451
23	13513	19459	26485	34593	43782	54052	65403	77835
24	14101	20305	27637	36097	45686	56402	68246	81219
25	14688	21151	28789	37601	47589	58752	71090	84603

shown in the table are of course too small for ordinary use, but that part of the table was filled out for its possible convenience otherwise. On single-track roads where all engines use one track the tank may be placed 8' 5" from the track center; this gives sufficient clearance and yet permits the use of a single swinging pipe which will reach from the bottom of the tank to the tender manhole. In Fig. 160 is illustrated one form of wooden tank. They are preferably manufactured by those who make a special business of it and who by the use of special machinery can insure tight joints. When it is inconvenient to place the tank near the track, or when there is a double track, a "stand-pipe" becomes necessary. See § 285. One of the most difficult and troublesome problems is to prevent freezing, particularly in the valves and pipes. Not only are the pipes carefully covered but fires must be maintained during cold weather. When the pumping is accomplished by means of a steam-pump, supplied from a steam-boiler in the pump-house under the tank, coils of steam-pipe may be employed to heat the water or to heat the pipes. Partial protection may be obtained by means of a double roof and double bottom, the spaces being filled with sawdust or some other non-conductor of heat.

283. Pumping. The pumping is done most reliably with steam-pumps or gas-engines, although hot-air engines, windmills, and even man-power are occasionally employed. Economy of operation requires that the water-stations shall be so located that each tank shall be used regularly and that each pump shall be regularly operated for maintaining the water-supply. On the other hand, the pump should not be *required* to regularly work at night to maintain the supply and should have an excess capacity of say 25%. When a tank is but little used, it will still require the labor of an attendant, and his time will be largely wasted unless he can be utilized for other labor about the station. In recent years gasoline has been extensively employed as a fuel for the pumping-engines. The chief advantages of its use lies in the extreme simplicity of the mechanism and the very slight attention it requires, which permits their being operated by station-agents and others, who are paid \$10 per month extra, instead of paying a regular pumper \$35 per month. "Screenings," "slack coal," etc., are used as fuel for steam-pumps and may frequently be delivered at the pump-house at a cost not exceeding 30 cents per ton, but even at that price the cost of pumping per thousand gallons, although dependent on the horizontal and vertical distances to the source of supply and to the tank, will generally run at 2 cents to 6 cents per 1000 gallons. In many cases where steam plants have been replaced by gasoline plants, the cost of pumping per 1000 gallons has been reduced to one third or even one fourth of the cost of steam pumping. Of course the cost, using windmills, is reduced to the mere maintenance of the machinery, but the unreliability of wind as a motive power and the possibility of its failure to supply water when it is imperatively needed has made this form of motive power unpopular. (See report to Ninth Annual Convention of the Association of Railway Superintendents of Bridges and Buildings, Oct. 1899.)

284. Track tanks. These are chiefly required as one of the means of avoiding delays during fast-train service. A trough, made of steel plate, is placed between the rails on a stretch of *perfectly level* track. A scoop on the end of a pipe is lowered from under the tender into the tank while the train is in motion. The rapid motion scoops up the water, which then flows into the tender tank. The following brief description of an installation on the Baltimore & Ohio Railroad between Baltimore and

Philadelphia will answer as a general description of the method. The trough is made of $\frac{3}{16}$ " steel plate, 19" wide, 6" deep, and has a length of 1200 feet. There is riveted on each side a line of $1\frac{1}{8} \times 2 \times \frac{1}{4}$ " angle bars. These angle bars rest on the ties. Ordinary track spikes hold these angle bars to the ties, but permit expansion as with rails. The tanks are firmly anchored at the center, the ends being free to expand or contract. The plates are 15 feet long and are riveted with $\frac{7}{16}$ " rivets, 20 rivets per joint. At each end is an inclined plane 13' 8" long. If the fireman should neglect to raise the scoop before the end of the tank is reached, the inclined plane will raise it automatically and a catch will hold it raised. Water is supplied to the tanks by a No. 9 Blake pump having a capacity of 260 gallons per minute. During cold weather, freezing is prevented by injecting into the side of the tanks, at intervals of 45 feet, jets of steam, which come through $\frac{1}{8}$ " holes. Two boilers of 80 and 95 H.P. are required for pumping and to keep the water from freezing. During warm weather an upright 25 H.P. boiler suffices for the pumping. The cost of installation was about \$10,000 to \$11,000, the cost of maintenance being about \$132.50 per month.

285. Stand-pipes. These are usually manufactured by those who make a specialty of such track accessories, and who can ordinarily be trusted to furnish a correctly designed article. In Fig. 161 is shown a form manufactured by the Sheffield Car Co. Attention is called to the position of the valve and to the device for holding the arm parallel to the track when not in use so that it will not be struck by a passing train. When a stand pipe is located between parallel tracks, the strict requirements of clearance demand that the tracks shall be bowed outward slightly. If the tracks were originally straight, they may be shoved over by the trackmen, the shifting gradually running out at about 100 feet each side of the stand-pipe. If the tracks were originally curved, a slight change in radius will suffice to give the necessary extra distance between the tracks.

BUILDINGS.

286. Station platforms. These are most commonly made of planks at minor stations. Concrete is used in better-class work, also paving brick. An estimate of the cost of a platform of paving brick laid at Topeka, Kan., was \$4.89 per 100 square feet when

laid flat and \$7.24 per 100 square feet when laid on edge. The curbing cost 36 cents per linear foot. Cinders, curbed by timbers

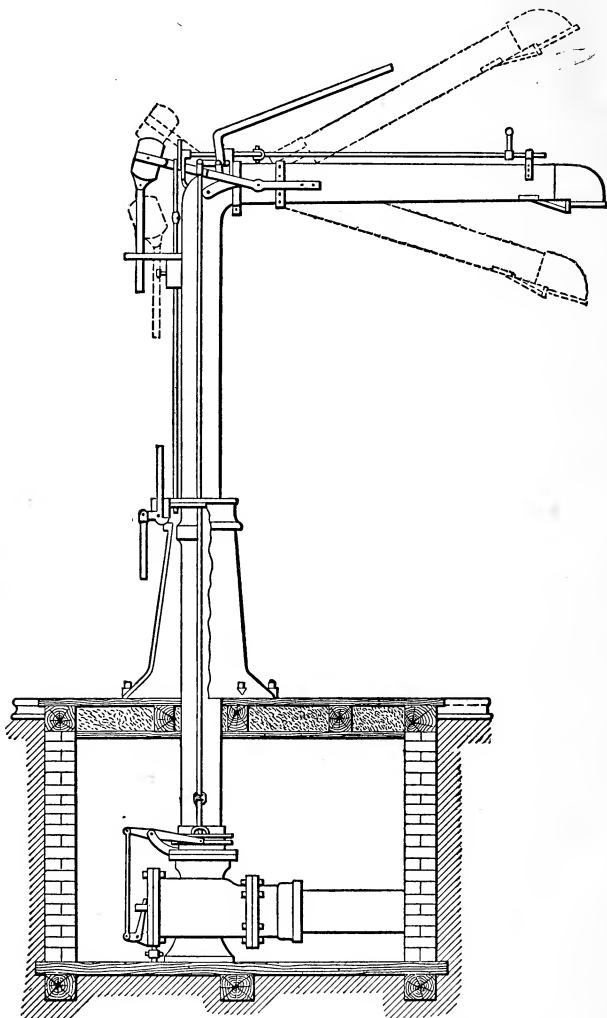


FIG. 161.—STAND-PIPE.

or stone, bound by iron rods, make a cheap and fairly durable platform, but in wet weather the cinders will be tracked into

the stations and cars. Three inches of crushed stone on a cinder foundation is considered to be still better, after it is once thoroughly packed, than a cinder surface.

Elevation.—The elevation of the platform with respect to the rail has long been a fruitful source of discussion. Some roads make the platforms on a level with the top of the rail, others 3" above, others still higher. As a matter of convenience to the passengers, the majority find it easier to enter the car from a high platform, but experience proves that accidents are more numerous with the higher platforms, unless steps are discarded altogether and the cars are entered from level platforms, as is done on elevated roads. As a railroad must generally pay damages to the stumbling passenger, they prefer to build the lower platform. Convenience requires that the rise from the platform to the lowest step should not be greater than the rise of the car steps. This rise is variable, but with the figures usually employed the application of the rule will make the platform 5" to 15" above the rail.

Position with respect to tracks.—Low platforms are generally built to the ends of the ties, or, if at the level of the top of the rail, are built to the rail head. Car steps usually extend 4' 6" from the track center and are 14" to 24" above the rail. The platform must have plenty of clearance, and when the platform is high its edge is generally required to be 5' 6" from the track center.

287. Minor stations. For a complete discussion of the design of stations of all kinds, including the details, the student is referred to "Buildings and Structures of American Railroads," by Walter G. Berg, once Chief Engineer of the Lehigh Valley Railroad. The subject is too large for adequate discussion here, but a few fundamental principles will be referred to.

Rooms required. An office and waiting-room is the minimum. A baggage-room, toilet-rooms, and express office are successively added as the business increases. In the Southern States a separate waiting-room for colored people is generally provided. It used to be common to have separate waiting-rooms for men and women. Experience proved that the men's waiting-room became a lounging place and smoking-room for loafers, and now large single waiting-rooms are more common even in the more pretentious designs, smoking being excluded. The office usually has a bay window, so that a more extended view

of the track is obtainable. The women's toilet-room is entered from the waiting-room. The men's toilet-room, although built immediately adjoining the other in order to simplify the plumbing, is entered from outdoors. Old-fashioned designs built the station as a residence for the station-agent; later designs have very generally abandoned this idea. "Combination" stations (passenger and freight) are frequently built for small local stations, but their use seems to be decreasing and there is now a tendency to handle the freight business in a separate building.

288. Section-houses. These are houses built along the right-of-way by the railroad company as residences for the trackmen. The liability of a wreck or washout at any time and at any part of the road, as well as the convenience of these houses for ordinary track labor, makes it all but essential that the trackmen should live on the right-of-way of the road, so that they may be easily called on for emergency service at any time of day or night. This is especially true when the road passes through a thinly settled section, where it would be difficult if not impossible to obtain suitable boarding-places. It is in no sense an extravagance for a railroad to build such houses. Even from the direct financial standpoint the expense is compensated by the corresponding reduction in wages, which are thus paid partly in free house rent. And the value of having men on hand for emergencies will often repay the cost in a single night. Where the country is thickly settled the need for such houses is not so great, and railroads will utilize or perhaps build any sort of suitable house, but on Southern or Western roads, where the need for such houses is greater, standard plans have been studied with great care, so as to obtain a maximum of durability, usefulness, comfort, and economy of construction. (See Berg's Buildings, etc., noted above.) On Northwestern roads, protection against cold and rain or snow is the chief characteristic; on Southern roads good ventilation and durability must be chiefly considered. Such houses may be divided into two general classes—(a) those which are intended for trackmen only and which may be built with great simplicity, the only essential requirements being a living-room and a dormitory, and (b) those which are intended for families, the houses being then distinguished as "dwelling-houses for employees.

289. Engine-houses. Small engine-houses are usually built rectangular in plan. Their minimum length should be some-

what greater than that of the longest engine on the road. They may be built to accommodate two engines on one track, but then they should be arranged to be entered at either end, so that neither engine must wait for the other. In width there may be as many tracks as desired, but if the demand for stalls is large, it will probably be preferable to build a "roundhouse." Rectangular engine-houses are usually entered by a series of parallel tracks switching off from one or more main tracks, no turn-table being necessary. If a turn-table is placed outside (because one

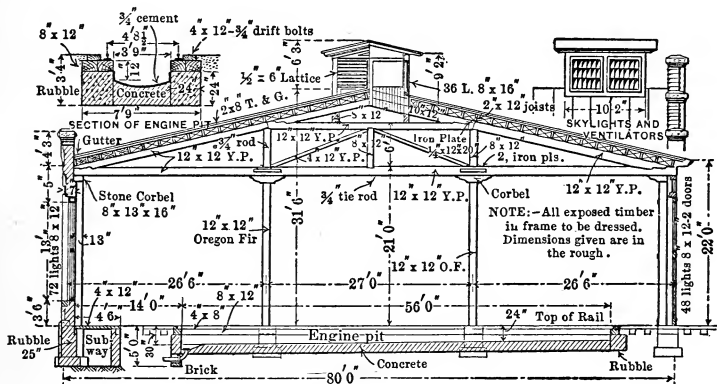


FIG. 162.—ENGINE-HOUSE.

is needed at that part of the road) enough track should be allowed between the house and the turn-table so that engines may be quickly removed from the engine-house in case of fire without depending on the turn-table to get them out of danger.

Roundhouses. The plan of these is generally polygonal rather than circular. The straight walls are easier to build; the construction is more simple, and the general purpose is equally well served. They may be built as a part of a circle or a complete circle, a passageway being allowed, so that there are two entrances instead of one. When space is very limited a roundhouse with turn-table will accommodate more engines in proportion to the space required (including the approaches) than a rectangular house. The enlarged space on the outer side of each segment of a roundhouse furnishes the extra space which is needed for the minor repairs which are usually made in a roundhouse. One disadvantage is that supervision is not quite so easy or effec-

tive as in rectangular houses. Of course such houses are used not only for storing and cleaning engines, but also for minor repairs which do not require the engine to be sent to the shops for a general overhauling.

Construction. The outer walls are usually of brick. The inner walls consist almost entirely of doors and the piers between them, although there is usually a low wall from the top of the door frames to the roof line, which usually slopes outward so as to turn rain-water away from the central space.

Roofs. Many roofs have been built of slate with iron truss framing, with the idea of maximum durability. The slate is good, but experience shows that the iron framing deteriorates very rapidly from the action of the gases of combustion of the engines which must be "fired" in the houses before starting. Roof frames are therefore preferably made of wood.

Floors. These are variously constructed of cinders, wood, brick, and concrete. Brick has been found to be the best material. Anything short of brick is a poor economy; concrete is very good if properly done but is somewhat needlessly expensive.

Ventilation. This is a troublesome more expensive. The general plan is to have "smoke-jacks" which drop down over the stack of each engine as it reaches its precise place in its stall and which will carry away all smoke and gas. Such a movable stack is most easily constructed of thin metal—say galvanized iron—but these will be corroded by the gases of combustion in two or three years. Vitrified pipe, cast iron, expanded metal and cement, and even plain wood painted with "fireproof" paint, have been variously tried, but all methods have their unsatisfactory features. (For an extended discussion of roundhouse floors and ventilation see the Proc. Assoc. of Railway Supts. of Bridges and Buildings for 1898, pp. 112–135.)

SNOW STRUCTURES.

290. Snow-fences. Snow structures are of two distinct kinds—fences and sheds. A snow-fence implies drifting snow—snow carried by wind—and aims to cause all drifting snow to be deposited away from the track. Some designs actually succeed in making the wind an agent for clearing snow from the track where it has naturally fallen. A snow-fence is placed at right angles to the prevailing direction of the wind and 50 to 100 feet away from the tracks. When the road line is at right angles to

the prevailing wind, the right-of-way fence may be built as a snow-fence—high and with tight boarding. Hedges have sometimes been planted to serve this purpose. When the prevailing wind is oblique, the snow fences must be built in sections where they will serve the best purpose. The fences act as wind breakers, suddenly lowering the velocity of the wind and causing the snow carried by the wind to be deposited along the fence. Portable fences are frequently used, which are placed (by permission of the adjoining property owners) outside of the right-of-way. If a drift forms to the height of the portable fence the fence may be replaced on the top of the drift, where it may act as before, forming a still higher drift. When the prevailing wind runs along the track line, snow-fences built in short sections on the sides will cause snow to deposit around them while it scours its way along the track line, actually clearing it. Such a method is in successful operation at some places on the White Mountain and Concord divisions of the Boston & Maine Railroad. Snow-fences, in connection with a moderate amount of shoveling and plowing, suffice to keep the tracks clear on railroads not troubled with avalanches. In such cases snow-sheds are the only alternative.

291. Snow-sheds. These are structures which will actually keep the tracks clear from snow regardless of its depth outside. Fortunately they are only necessary in the comparatively rare situations where the snowfall is excessive and where the snow is liable to slide down steep mountain slopes in avalanches. These avalanches frequently bring down with them rocks, trees, and earth, which would otherwise choke up the road-bed and render it in a moment utterly impassable for weeks to come. The sheds are usually built of 12"×12" timber framed in about the same manner as trestle timbering; the "bents" are sometimes placed as close as 5 feet, and even this has proved insufficient to withstand the force of avalanches. The sheds are therefore so designed that the avalanche will be *deflected* over them instead of spending its force against them. Although these sheds are only used in especially exposed places, yet their length is frequently very great and they are liable to destruction by fire. To confine such a fire to a limited section, "fire-breaks" are made—i.e., the shed is discontinued for a length of perhaps 100 feet. Then, to protect that section of track, a V-shaped deflector will be placed on the uphill side which will deflect all

descending material so that it passes over the sheds. Solid crib work is largely used for these structures. Fortunately suitable timber for such construction is usually plentiful and cheap where these structures are necessary. Sufficient ventilation is obtained by longitudinal openings along one side immediately under the roof. "Summer" tracks are usually built outside the sheds to avoid the discomfort of passing through these semi-tunnels in pleasant weather. The fundamental elements in the design of such structures is shown in Fig. 163, which illustrates some of the sheds used on the Canadian Pacific Railroad.

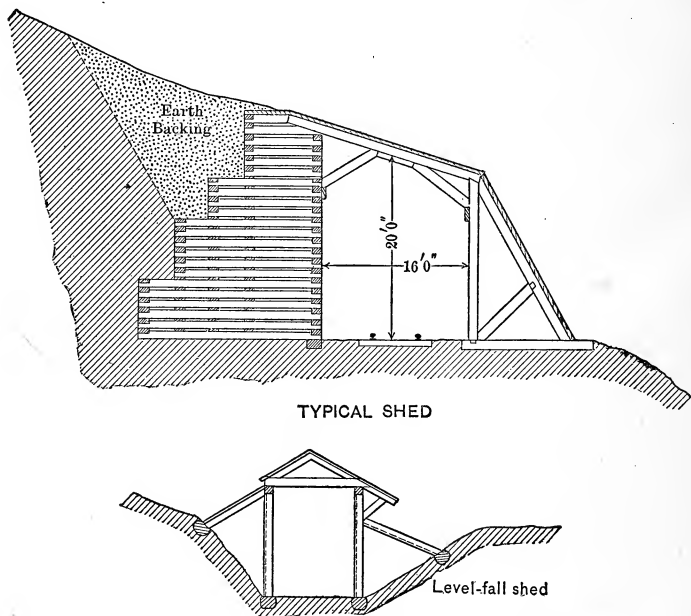


FIG. 163.—SNOW-SHEDS—CANADIAN PACIFIC RAILROAD.

292. Turn-tables. The essential feature of a turn-table is a carriage of sufficient size and strength to carry a locomotive, the carriage turning on a pivot of sufficient size to carry such a load. The carriage revolves in a circular pit whose top has the same general level as the surrounding tracks. The carriages were formerly made largely of wood; very many of those still in use are of cast iron. Structural steel is now uni-

versally employed for all modern work and since the construction of the carriage and the pivot is a special problem in structures, no further attention will here be paid to the subject, except to that part which the railroad engineer must work out.—laying out the site and preparing the foundation. The *minimum* length of such a carriage (and therefore the diameter of the pit) is evidently the length over all of the longest engine and tender in use on the road. Usually 60-foot turn-tables will suffice for an ordinary road, and for light-traffic roads employing small engines, 50 feet or even less may be sufficient. Many of the heavier freight engines of recent make have a total length of 65 to 70 feet; therefore 75-foot turn-tables are a better standard for heavy-traffic roads. A retaining-wall should be built around the pit. The stability of this wall immediately under the tracks should be especially considered. The most important feature is the stability of the foundation of the pivot, which must sustain a concentrated pressure, more or less eccentric, of perhaps 300 tons. When firm soil or rock may be easily reached, this need give no trouble, but in a soft, treacherous soil a foundation of concrete or piling may be necessary. If the soil is very porous, it may be depended on to carry away all rain-water which may fall into the pit before the foundations are affected, but when the soil is tenacious it may be necessary to drain the subsoil thoroughly and carry off immediately all surface drainage by means of subsoil pipes which have a suitable outfall.

The location of the turn-table in the yard is a part of the general subject of "Yards," and will be considered in the next chapter.

CHAPTER XIII.

YARDS AND TERMINALS.

293. Value of proper design. A large part of the total cost of handling traffic, particularly freight, is that incurred at terminals and stations. In illustration of this, consider the relative total cost of handling a car-load of coal and a car-load (of equal weight) of mixed merchandise. The coal will be loaded in bulk on the cars at the mines, where land is comparatively cheap, and the cars grouped into a train without regard to order, since they are (usually) uniform in structure, loading, and contents. When the terminal or local station is reached they are run on tracks occupying property which is usually much cheaper than the site of the terminal tracks and freight-houses; they are unloaded by gravity into pockets or machine conveyors and the empty cars are rapidly hauled by the train-load out of the way. On the other hand, the merchandise is loaded by hand on the car from a freight-house occupying a central and valuable location, the car is hauled out into a yard occupying valuable ground, is drilled over the yard tracks for a considerable aggregate mileage before starting for its destination, where the same process is repeated in inverse order. In either case the terminal expenses are evidently a large percentage of the total cost and, once loaded, it makes but little difference just how far the car is hauled to the other terminal. But the very evident increase in terminal charges for general merchandise over those for coal (large as they are) gives a better idea of the magnitude of terminal charges.

Many yards are the result of growth, adding a few tracks at a time, without much evidence of any original plan. In such cases the yard is apt to be very inefficient, requiring a much larger aggregate of drilling to accomplish desired results, requiring much more *time* and hence blocking traffic and finally adding greatly to the cost of terminal service, although the fact of its being a needless addition to cost may be unsuspected or not fully appreciated. An unwillingness or inability to spend money for

the necessary changes, and the difficulty of making the changes while the yard is being used, only prolong the bad state of affairs and an inefficient makeshift is frequently adopted. Assume that an improvement in the design of the yard will permit a saving of the use of one switching engine, or for example, that the work may be accomplished with three switching engines instead of four. Assuming a daily cost of \$25, we have in 313 working days an annual saving of \$7825, which, capitalized at 5%, gives \$156,500, enough to reconstruct any ordinary yard.*

294. Divisions of the subject. The subject naturally divides itself into three heads—(a) Yards for receiving, classifying, and distributing freight cars, called more briefly freight yards; (b) yards and conveniences for the care of engines, such as ash tracks, turn-tables, coal-chutes, sand-houses, water-tanks, or water stand-pipes, etc., and (c) passenger terminals.

FREIGHT YARDS.

295. General principles. It should be recognized at the start that at many places an ideally perfect yard is impossible, or at least impracticable, generally because ground of the required shape or area is practically unobtainable. But there are some general principles which may and should be followed in every yard and other ideals which should be approached as nearly as possible. Nevertheless every yard is an independent problem. Before taking up the design of freight yards, it is first necessary to consider the general object of such yards and the general principles by which the object is accomplished. These may be briefly stated as follows:

1. A yard is a device, a machine, by which incoming cars are sorted and classified—some sent to warehouses for unloading, some sent to connecting railroads, some made up for local distribution along the road, some sent for repairs, and, in short a device by which all cars are sent *through* and *out* of the yard as quickly as possible.

2. Except when a road's business is decreasing, or when its equipment is greater than its needs and its cars must be stored, efficiency of management is indicated by the rapidity with which the passage of cars *through* the yard is accomplished.

3. When a yard is the terminal of a "division," the freight

* Estimate of Mr. H. G. Hetzler, C., B. & Q. Ry.

trains will be pulled into a "receiving track" and the engine and caboose detached. The caboose will be run on to a "caboose track," which should be conveniently near, and the engine is run off to the engine yard. If the train is a "through" train and no change is to be made in its make-up, it will only need to wait for another engine and perhaps another caboose. If the cars are to be distributed, they will be drawn off by a switching engine to the "classification yard."

4. The design of a yard is best studied by first picking out the ladder tracks and the through tracks which lead from one division of the yard to another. These are tracks which must always be kept open for the passage of trains, in contradistinction to the tracks on which cars may be left standing, even though it is only for a few moments, while drilling is being done. Such a set of tracks, which may be called the skeleton of the yard, is shown by heavy lines in Fig. 164. Each line indicates a pair of rails. The tracks of the storage yards are shown by the lighter lines.

5. There is a distinct advantage in having all storage tracks double-ended—except "team tracks." Team tracks are those which have spaces for the accommodation of teams, so that loading or unloading may be done directly between the cars and teams. To avoid the necessity of teams passing over the tracks, these are best placed on the outskirts of the yard and consist of short stubs arranged in pairs. But storage tracks should have an outlet at each end so as to reduce the amount of drilling necessary to reach a car which may be at the extreme end of a long string of cars. This is done usually by means of two "ladder" tracks, parallel to each other, which thus make the storage tracks between them of equal length.

6. The equality of length of these storage tracks is a point insisted on by many, but on the other hand, trains are not always of uniform length even on any one division. Loaded trains and trains of empties will vary greatly in length, and the various styles and weights of freight engines employed necessitate other variations in the weights and lengths of trains hauled. With storage tracks of somewhat variable length a larger percentage of track length may be utilized, there will be less hauling over a useless length of track, and (assuming that the plot of ground available for yard purposes has equally favorable conditions for yard design) more business may be handled in a yard of given area.

7 Yards are preferably built so that the tracks have a grade of 0.5%—sometimes a little more than this—in the direction of the traffic through the yard. This grade, which will overcome a tractive resistance of 10 pounds per ton, will permit cars to be started down the ladder tracks by a mere push from the switching engine. They are then switched on to the desired storage track and run down that track by gravity until stopped at the desired place by a brakeman riding on the cars

8. Although not absolutely necessary, there is an advantage in having all frog numbers and switch dimensions uniform. No. 7 frogs are most commonly used. Sharper-angled frogs make easier riding, less resistance and less chance of derailment, but on the other hand require longer leads and more space. No. 6 and even No. 5 frogs are sometimes used on account of economy of space, but they have the disadvantages of greater tractive resistance, greater wear and tear on track and rolling stock, and greater danger of derailment.

296. Relation of yard to main tracks. Safety requires that there should be no connection between the yard tracks and the main tracks except at each end of the yard, where the switches should be amply protected by signals. Sometimes the main tracks run through the yard, making practically two yards—one for the traffic in either direction—but this either requires a double layout of tracks and houses (such as ash tracks, coal-chutes, sand-houses, etc.), or a very objectionable amount of crossing of the main-line tracks. The preferable method is to have the main line tracks entirely on the outside of the yard. A method which is in one respect still better is to spread the main tracks so that they run on each side of the yard. In this case there is never any necessity to cross one main track to pass from the yard to the other main track; a train may pass from the yard to either main track and still leave the other main track free and open. The ideal arrangement is that by which some of the tracks cross *over* or *under* all opposing tracks. By this means all connections between the yard and the main tracks may be by “trailing” switches; that is, trains will run on to the main track in the direction of motion on that main track. Of course all this applies only to *double* main track.

An important element of yard design is to have a few tracks immediately adjoining the main tracks and separate from the yard proper on which outgoing trains may await their orders to take

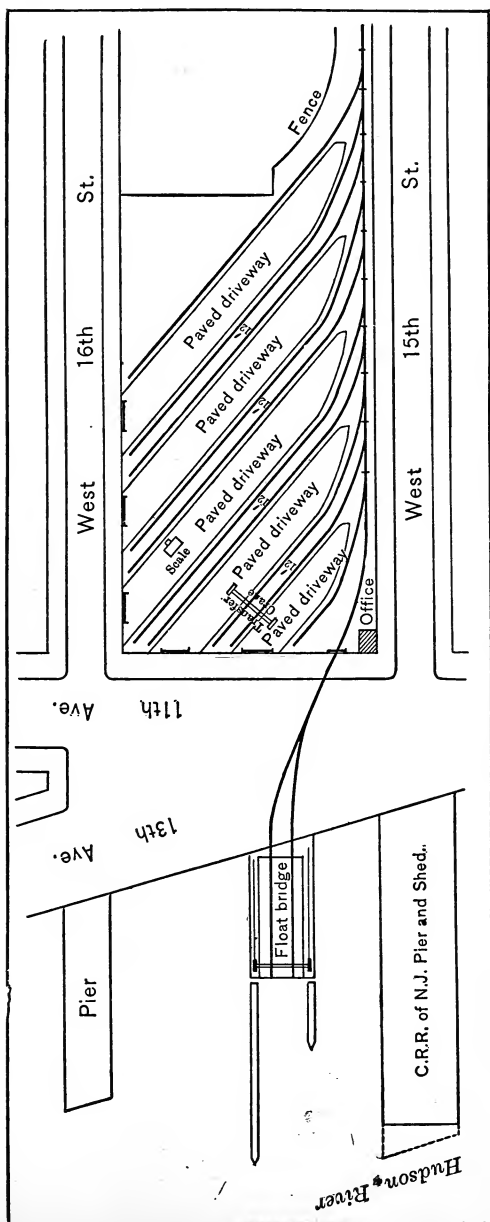


FIG. 165.—MINOR FREIGHT YARD.

the main track. When the orders come, they may start at once without any delay, without interfering with any yard operations, and they are not occupying tracks which may form part of the system needed for switching.

297. Minor freight yards. The term here refers to the substations, only found in the largest cities, to which cars will be sent to save in the amount of necessary team hauling and also to relieve a congestion of such loading and unloading at the main freight terminal. The cars are brought to these yards sometimes on floats (as is done so extensively at various points around New York Harbor), or they are run down on a long siding running perhaps through the city streets. But the essential feature of these yards is the maximum utilization of every square foot of yard space, which is always very valuable and which is frequently of such an inconvenient shape that a great ingenuity is required to obtain good results. There is generally a temptation to use excessively sharp curves. When the radii are greater than 175 feet no especial trouble is encountered. Curves with radius as short as 50 feet have been used in some yards. On such curves the long cars now generally used make a sharper angle with each other than that for which the couplers were designed and special coupler-bars become necessary. The two general methods of construction are (a) a series of parallel team tracks (as previously described and as illustrated further in Fig. 165), and (b) the "loop system," as is illustrated in Fig. 166.

298. Transfer cranes. These are almost an essential feature for yards doing a large business. The transportation of built-up girders, castings for excessively heavy machinery, etc., which weigh five to thirty tons and even more, creates a necessity for machinery which will easily transfer the loads from the car to the truck and *vice versa*. An ordinary "gin-pole" will serve the purpose for loads which do not much exceed five tons. A fixed framework, covering a span long enough for a car track and a team space, with a trolley traveling along the upper chord, is the next design in the order of cost and convenience. Increasing the span so that it covers two car tracks and two team spaces will very materially increase the capacity. Making the frame movable so that it travels on tracks which are parallel to the car tracks, giving the frame a longitudinal motion equal to two or three car lengths, and finally operating the raising and traveling mechanism by power, the facility for rapidly disposing of

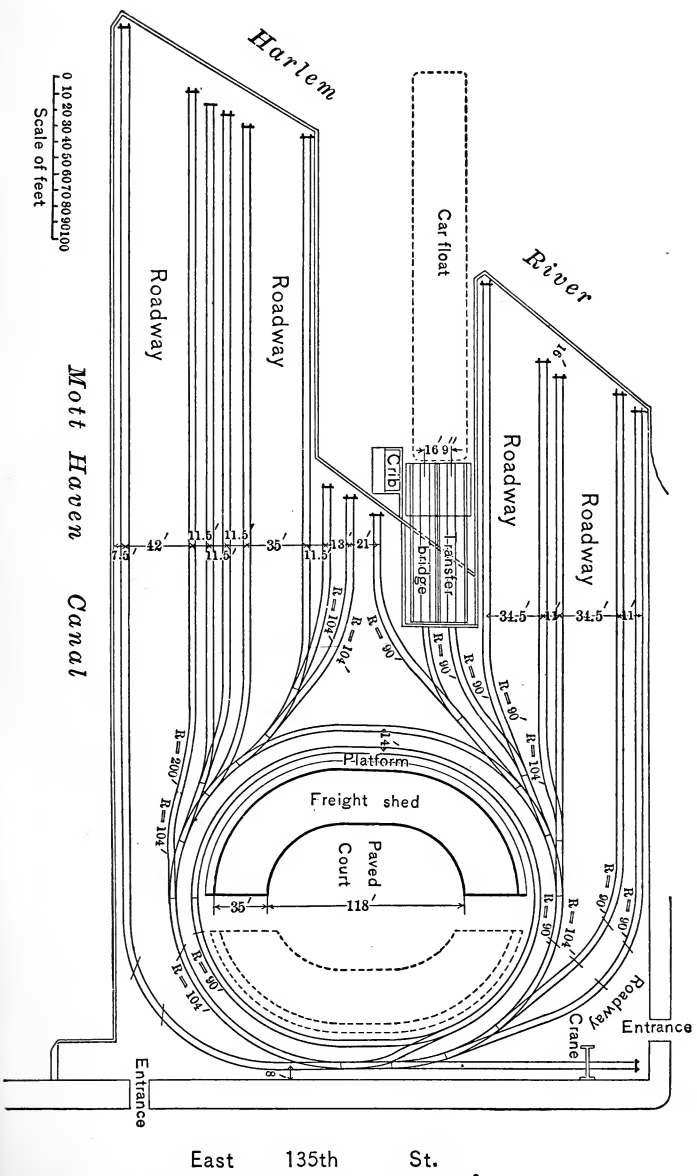


FIG. 166.—MINOR FREIGHT YARD ON A HARBOR FRONT.

heavy articles of freight is greatly increased. Of course only a very small proportion of freight requires such handling, and the business of a yard must be large or perhaps of a special character to justify and pay for the installation of such a mechanism. Figs. 165 and 166 each indicate a transfer crane, evidently of the fixed type.

299. Track scales. The location of these should be on one of the receiving tracks near the entrance to the yard, but not on the main track. It is always best to have a "dead track" over the scales—i.e., a track which has one rail on the solid side wall of the scale pit and the other supported at short intervals by posts which come up through the scale platform and yet do not touch it. These rails and the regular scale rails switch into one track by means of point rails a few feet beyond each end of the scales. The switches should be normally set so that all trains will use the dead track, unless the scales are to be operated. It has been found possible in a gravity yard to weigh a train with very little loss of time by running each car slowly by gravity over the scales and weighing them as they pass over.

ENGINE YARDS.

300. General principles. Engine yards must contain all the tracks, buildings, structures, and facilities which are necessary for the maintenance, care, and storage of locomotives and for providing them with all needed supplies. The supplies are fuel, water, sand, oil, waste, tallow, etc. Ash-pits are generally necessary for the prompt and economical disposition of ashes; engine-houses are necessary for the storage of engines and as a place where minor repairs can be quickly made. A turn-table is another all but essential requirement. The arrangement of all these facilities in an engine yard should properly depend on the form of the yard. In general they should be grouped together and should be as near as possible to the place where through engines drop the trains just brought in and where they couple on to assembled outgoing trains, so that all unnecessary running light may be avoided. In Figs. 164 and 167 are shown two designs which should be studied with reference to the relative arrangement of the yard facilities.

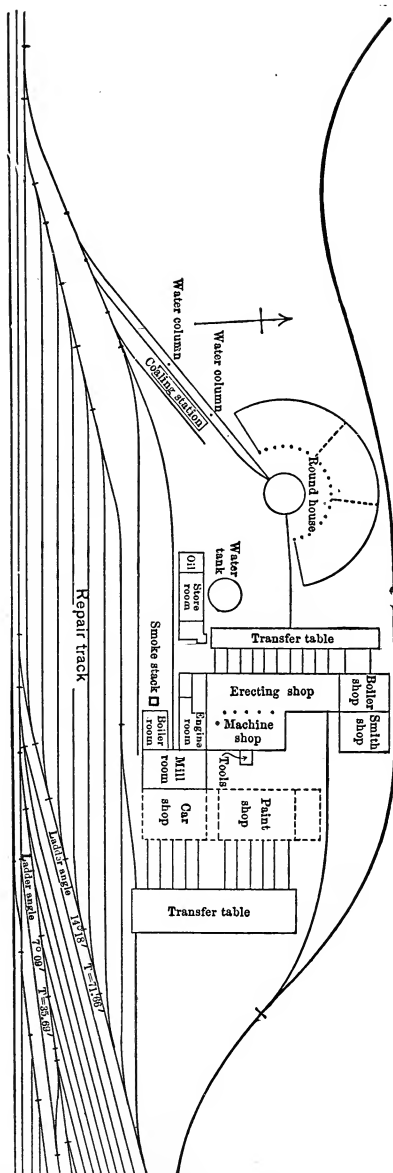
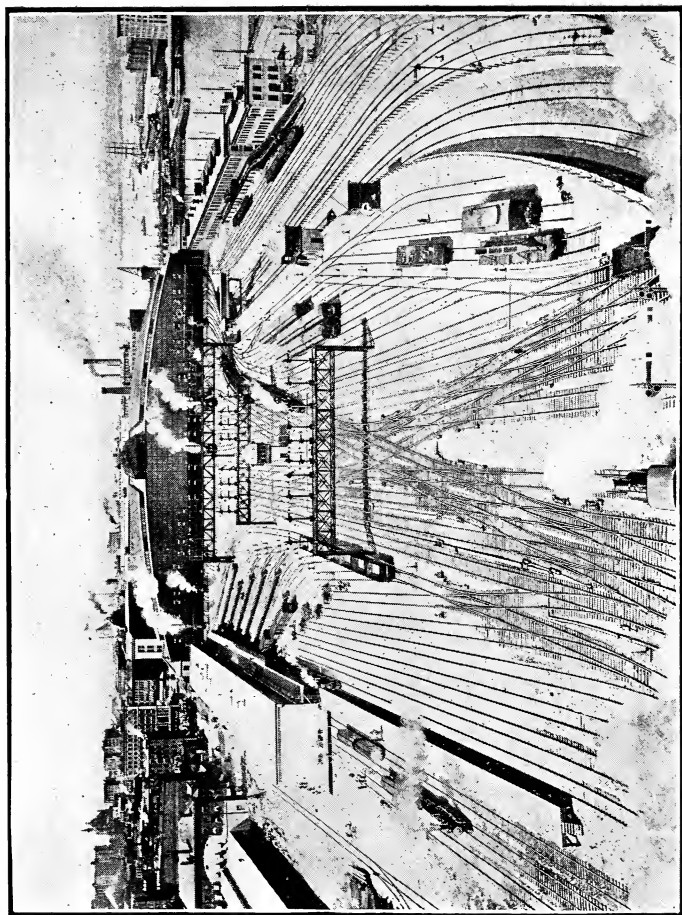


FIG. 167.—ENGINE YARD AND SHOPS, URBANA, ILL.

PASSENGER TERMINALS.

(Passenger terminals are one of the logical subdivisions of this chapter, but their construction does not concern one engineer in a thousand. The local conditions attending their construction are so varied that each case is a special problem in itself—a problem which demands in many respects the services of the architect rather than the engineer. The student who wishes to pursue this subject is referred to an admirable chapter in “Buildings and Structures of American Railroads,” by Walter G. Berg, Chief Engineer of the Lehigh Valley Railroad.)



SOUTH BOSTON TERMINAL.

To face p. 350.) (Published through courtesy of Union Switch and Signal Co.)



CHAPTER XIV.

BLOCK SIGNALING.

GENERAL PRINCIPLES.

301. Two fundamental systems. The growth of systems of block signaling has been enormous within the last few years—both in the amount of it and in the development of greater perfection of detail. The development has been along two general lines: (a) the *manual*, in which every change of signal is the result of some definite action on the part of some signalman, but in which every action is so controlled or limited or subject to the inspection of others that a mistake is nearly, if not quite, impossible; (b) the *automatic*, in which the signals are operated by mechanism, which cannot set a wrong signal as long as the mechanism is maintained in proper order. The fundamental principles of the two systems will be briefly outlined, after which the chief details of the most common systems will be pointed out.

302. Manual systems. Any railroad which has a telegraph line and an operator at all regular stations may (and frequently does) operate its trains according to the fundamental principles of the manual block system even though it makes no claim to a block-signal system. The basic idea of such a system is that after a train has passed a given telegraph- or signal-station, no other train will be permitted to follow it into that "block" until word is telegraphed from the next station ahead that the first train has passed out of that block. With a double-track road the operation is very simple; trains may be run at short intervals with long blocks; with an average speed of 30 miles per hour and blocks 5 miles long, trains *could* be run on a ten minute interval (nearly). A road with any such traffic would, of course, have much shorter blocks, and, practically, they would need to be considerably shorter.

With a single-track road the operation is much more complex, since the operator must keep himself informed of the move-

ments of the trains in *both* directions. The ratio of length of block to train interval would be only one half (and practically much less than half) what it could be with a double-track road. When such a system is adhered to rigidly, it is called an *absolute block* system. But when operating on this system, a delay of one train will necessarily delay every other train that follows closely after. A portion, if not all, of the delay to subsequent trains may be avoided, although at some loss of safety, by a system of permissive blocking. By this system an operator may give to a succeeding train a "clearance card" which permits it to pass into the next block, but at a reduced speed and with the train under such control that it may be stopped on very short notice, especially near curves. One element of the danger of this system is the *discretionary* power with which it invests the signalmen, a discretion which may be wrongfully exercised. A modification (which is a fruitful source of collisions on single-track roads) is to order two trains to enter a block approaching each other, and with instructions to pass each other at a passing siding at which there is no telegraph-station. When the instructions are properly made out and literally obeyed, there is no trouble, but every thousandth or ten thousandth time there is a mistake in the orders, or a misunderstanding or disobedience, and a collision is the result. The telegraph line, a code of rules, a corps of operators, and signals under the immediate control of the operators, are all that is absolutely needed for the simple manual system.

303. Development of the manual system. One great difficulty with the simple system just described is that each operator is practically independent of others except as he may receive general or specific orders from a train-dispatcher at the division headquarters. Such difficulties are somewhat overcome by a very rigid system of rules requiring the signalmen at each station to keep the adjacent signalmen or the train-dispatcher informed of the movements of all trains past their own stations. When these rules (which are too extensive for quotation here) are strictly observed, there is but little danger of accident, and a neglect by any one to observe any rule will generally be apparent to at least one other man. Nevertheless the safety of trains depends on *each* signalman doing his duty, and a little carelessness or forgetfulness on the part of any one man may cause an accident. The signaling between stations *may* be done by

ordinary telegraphic messages or by telephone, but is frequently done by electric bells, according to a code of signals, since these may be readily learned by men who would have more difficulty in learning the Morse code.

In order to have the signalmen mutually control each other, the "controlled manual" system has been devised. The first successful system of this kind which was brought into extensive use is the "Sykes" system, of which a brief description is as follows: Each signal is worked by a lever; the lever is locked by a latch, operated by an electro-magnet, which, with other necessary apparatus, is inclosed in a box. When a signal is set at danger, the latch falls and locks the lever, which cannot be again set free until the electro-magnet raises the latch. The magnet is energized only by a current, the circuit of which is closed by a "plunger" at the *next* station ahead; just above the plunger is an "indicator," also operated by the current, which displays the words *clear* or *blocked*. (There are variations on this detail.) When a train arrives at a block station (*A*), the signalman should have previously signaled to the station *ahead* (*B*) for permission to free the signal. The man ahead (*B*) pushes in the "plunger" on his instrument (assuming that the previous train has already passed him), which electrically opens the lock on the lever at the previous station (*A*). The signal at *A* can then be set at "safety." As soon as the train has passed *A* the signal at *A* must be set at "danger." A further development is a device by which the mere passage of the train over the track for a few feet beyond the signal will automatically throw the signal to "danger." After the signal once goes to danger, it is automatically locked and cannot be released except by the man in advance (*B*), who will not do so until the train has passed him. The "indicator" on *B*'s instrument shows "blocked" when *A*'s signal goes to danger after the train has passed *A*, and *B*'s plunger is then locked, so that he cannot release *A*'s signal while a train is in the block. As soon as the train has passed *A*, *B* should prepare to get his signals ready by signaling ahead to *C*, so that if the block between *B* and *C* is not obstructed, *B* may have his signals at "safety" so that the train may pass *B* without pausing. The student should note the great advance in safety made by the Sykes system; a signal cannot be set free except by the combined action of two men, one the man who actually operates the signal and

the other the man at the station ahead, who frees the signal electrically and who by his action certifies that the block immediately ahead of the train is clear.

A still further development makes the system still more "automatic" (as described later), and causes the signal to fall to danger or to be kept locked at danger, if even a single pair of wheels comes on the rails of a block, or if a switch leading from a main track is opened.

304. Permissive blocking. "Absolute" blocking renders accidents due to collisions almost impossible unless an engineer runs by an adverse signal. The signal mechanism is usually so designed that, if it gets out of order, it will inevitably fall to "danger," i.e., as described later, the signal-board is counterbalanced by a weight which is much heavier. If the wire breaks, the counterweight will fall and the board will assume the horizontal position, which always indicates "danger." But it sometimes happens that when a train arrives at a signal-station, the signalman is unable to set the signal at safety. This may be because the previous train has broken down somewhere in the next block, or because a switch has been left open, or a rail has become broken, or there is a defect of some kind in the electrical connections. In such cases, in order to avoid an indefinite blocking of the whole traffic of the road, the signalman may give the engineer a "caution-card" or a "clearance card," which authorizes him to proceed slowly and with his train under complete control into the block and through it if possible. If he arrives at the next station without meeting any obstruction it merely indicates a defective condition of the mechanism, which will, of course, be promptly remedied. Usually the next section will be found clear, and the train may proceed as usual. On roads where the "controlled manual" system has received its highest development, the rules for permissive blocking are so rigid that there is but little danger in the practice, unless there is an absolute disobedience of orders.

305. Automatic systems. By the very nature of the case, such systems can only be used to indicate to the engineers of trains something with reference to the passage of previous trains. The complicated shifting of switches and signals which is required in the operation of yards and terminals can only be accomplished by "manual" methods, and the only automatic features of these methods consist in the mechanical checks

(electric and otherwise), which will prevent wrong combinations of signals. But for long stretches of the road, where it is only required to separate trains by at least one block length, an automatic system is generally considered to be more reliable. As expressed forcibly by a railroad manager, "an automatic system does not go to sleep, get drunk, become insane, or tell lies when there is any trouble." The same cannot always be said of the employés of the manual system.

The basic idea of all such systems is that when a train passes a signal-station (*A*), the signal automatically assumes the "danger" position. This may be accomplished electrically, pneumatically, or even by a direct mechanism. When the train reaches the end of the block at *B* and passes into the next one, the signal at *B* will be set at danger and the signal at *A* will be set at safety. The lengths of the blocks are usually so great that the only practicable method of controlling from *B* a mechanism at *A* is by electricity, although the actual motive power at *A* may be pneumatic or mechanical. At one time the current from *A* to *B* was carried on ordinary wires. This method has the very positive advantage of reliability, definite resistance to the current, and small probability of short-circuiting or other derangement. But now all such systems use the rails for a track circuit and this makes it possible to detect the presence of a single pair of wheels on the track anywhere in the block, or an open switch, or a broken rail. Any such circumstances, as well as a defect in the mechanism, will break or short-circuit the current and will cause the signal to be set at danger. To prevent an indefinite blocking of traffic owing to a signal persistently indicating danger, most roads employing such a system have a rule substantially as follows: When a train finds a signal at danger, after waiting one minute (or more, depending on the rules), it may proceed slowly, expecting to find an obstruction of some sort; if it reaches the next block without finding any obstruction and finds the next signal clear, it may proceed as usual, but must promptly report the case to the superintendent. Further details regarding these methods will be given later. See § 310.

306. "Distant" signals. The close running of trains that is required on heavy-traffic roads, especially where several branches combine to enter a common terminal, necessitates the use of very short blocks. A heavy train running at high speed

can hardly make a "service" stop in less than 2000 feet, while the curves of a road (or other obstructions) frequently make it difficult to locate a signal so that it can be seen more than a few hundred feet away. It would therefore be impracticable to maintain the speed now used with heavy trains if the engineer had no foreknowledge of the condition in which he will find a signal until he arrives within a short distance of it. To overcome this difficulty the "distant" signal was devised. This is placed about 1800 or 2000 feet from the "home" signal, and is interlocked with it so that it gives the *same* signal. The distant signal is frequently placed on the same pole as the home signal of the previous block. When the engineer finds the distant signal "clear," it indicates that the succeeding home signal is also clear, and that he may proceed at full speed and not expect to be stopped at the next signal; for the distant signal cannot be cleared until the succeeding home signal is cleared, which cannot be done until the block succeeding that is clear. A clear distant signal therefore indicates a clear track for two succeeding blocks. When the engineer finds the distant signal blocked, he need not stop (providing the home signal is clear). It simply indicates that he must be prepared to stop at the next home signal and must reduce speed if necessary. It may happen that by the time he reaches the succeeding home signal it has already been cleared, and he may proceed without stopping. This device facilitates the rapid running of trains, with no loss of safety, and yet with but a moderate addition to the signaling plant.

307. "Advance" signals. It sometimes becomes necessary to locate a signal a few hundred feet short of a regular passenger-station. A train might be halted at such a signal because it was not cleared from the signal-station ahead—perhaps a mile or two ahead. For convenience, an "advance" signal may be erected immediately beyond the passenger-station. The train will then be permitted to enter the block as far as the advance signal and may deliver its passengers at the station. The advance signal is interlocked with the home signal back of it, and cannot be cleared until the home signal is cleared and the entire block ahead is clear. In one sense it adds another block, but the signal is entirely controlled from the signal station back of it.

MECHANICAL DETAILS.

308. Signals. The primitive signal is a mere cloth flag. A better signal is obtained when the flag is suspended in a suitable place from a fixed horizontal support, the flag weighted at the bottom, and so arranged that it may be drawn up and out of sight by a cord which is run back to the operator's office. The next step is the substitution of painted wood or sheet metal for the cloth flag, and from this it is but a step to the standard semaphore on a pole, as is illustrated in Fig. 168. The simple flag, operated for convenience with a cord, is the signal employed on thousands of miles of road, where they perhaps make no claim to a block-signal system, and yet where the trains are run according to the fundamental rules of the simple manual block method.

Semaphore boards. These are about 5 feet long, 8 inches wide at one end, and tapered to about 6 inches wide at the hinge end. The boards are fastened to a casting which has a ring to hold a red glass which may be swung over the face of a lantern, so as to indicate a red signal. "Distant" signal-boards usually have their ends notched or pointed; the "home" signal-boards are square ended. The boards are always to the *right* of the hinge when a train is approaching them. The "home" signals are generally painted red and the "distant" signals green, although these colors are not invariable. The backs of the boards are painted white. Therefore any signal-board which appears on the *left* side of its hinge will also appear *white*, and is a signal for traffic in the opposite direction, and is therefore of no concern to an engineman.

Poles and bridges. When the signals are set on poles, they are generally placed on the right-hand side of the track. When there are several tracks, four or more, a bridge is frequently built and then each signal is over its own track. When switches run off from a main track, there may be several signal-boards over one track. The upper one is the signal for the main track and the lower ones for the several switches. In Fig. 169 is shown a "bridge" with its various signal-boards controlling the several tracks and the switches running off from them.

"Banjo" signals. This name is given to a form of signal, illustrated in Fig. 170, in which the indication is taken from the

color of a round disk inclosed with glass. This is the distinctive signal of the Hall Signal Company, and is also used by the Union Switch and Signal Company. The great argument in their favor is that they may be worked by an electric current of low voltage, which is therefore easily controlled; that the mechanism is entirely inside of a case, is therefore very light, and is not exposed to the weather. The argument urged against them is that it is a signal of *color* rather than *form* or *position*, and that in foggy weather the signal cannot be seen so easily; also that unsuspected color-blindness on the part of the engineman may lead to an accident. Notwithstanding these objections, this form of signal is used on thousands of miles of line in this country.

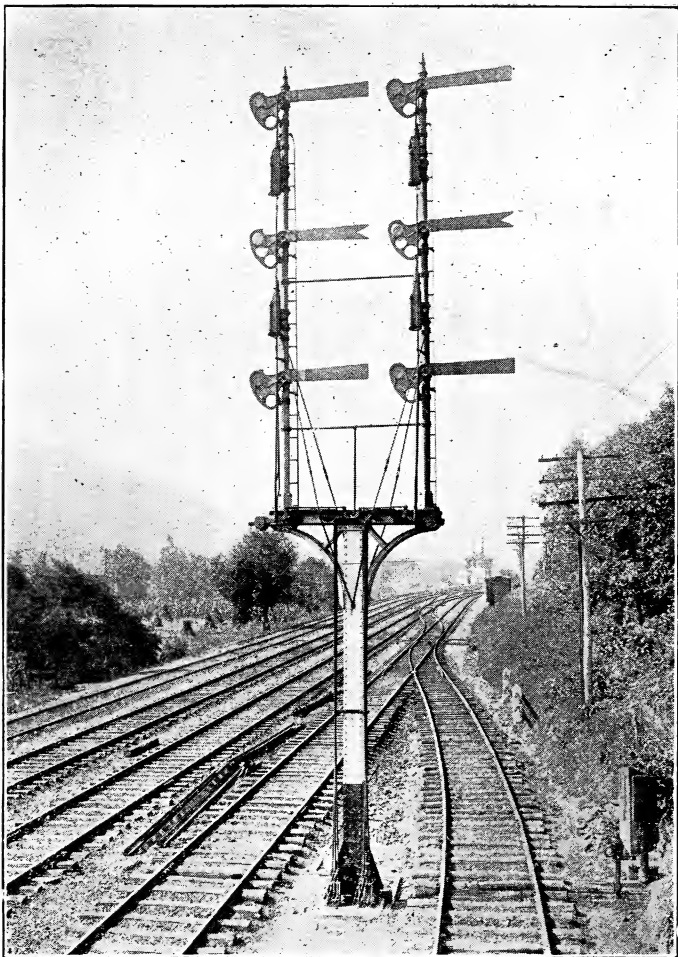
309. Wires and pipes. Signals are usually operated by levers in a signal-cabin, the levers being very similar to the reversing-lever of a locomotive. The distance from the levers to the signals is, of course, very variable, but it is sometimes 2000 feet. The connecting-link for the most distant signals is usually No. 9 wire; for nearer signals and for all switches operated from the cabin it may be 1-inch pipe. When not too long, one pipe will serve for both motions, forward and back. When wires are used, it is sometimes so designed (in the cheaper systems) that one wire serves for one motion, gravity being depended on for the other, but now all good systems require two wires for each signal.

Compensators. Variations of temperature of a material with as high a coefficient as iron will cause very appreciable difference of length in a distance of several hundred feet, and a dangerous lack of adjustment is the result. To illustrate: A fall of 60° F. will change the length of 1000 feet of wire by

$$1000 \times 60 \times .0000065 = 0.39 \text{ foot} = 4.68 \text{ inches.}$$

A much less change than this will necessitate a readjustment of length, unless automatic compensators are used. A compensator for pipes is very readily made on the principle illustrated in Fig. 171. The problem is to preserve the distance between *a* and *d* constant regardless of the temperature. Place the compensator half-way between *a* and *d*, or so that $ab = cd$. A fall of temperature contracts *ab* to *ab'*. Moving *b* to *b'* will cause *c* to move to *c'*, in which $bb' = cc'$. But *cd* has also shortened to *c'd*; therefore *d* remains fixed in position.

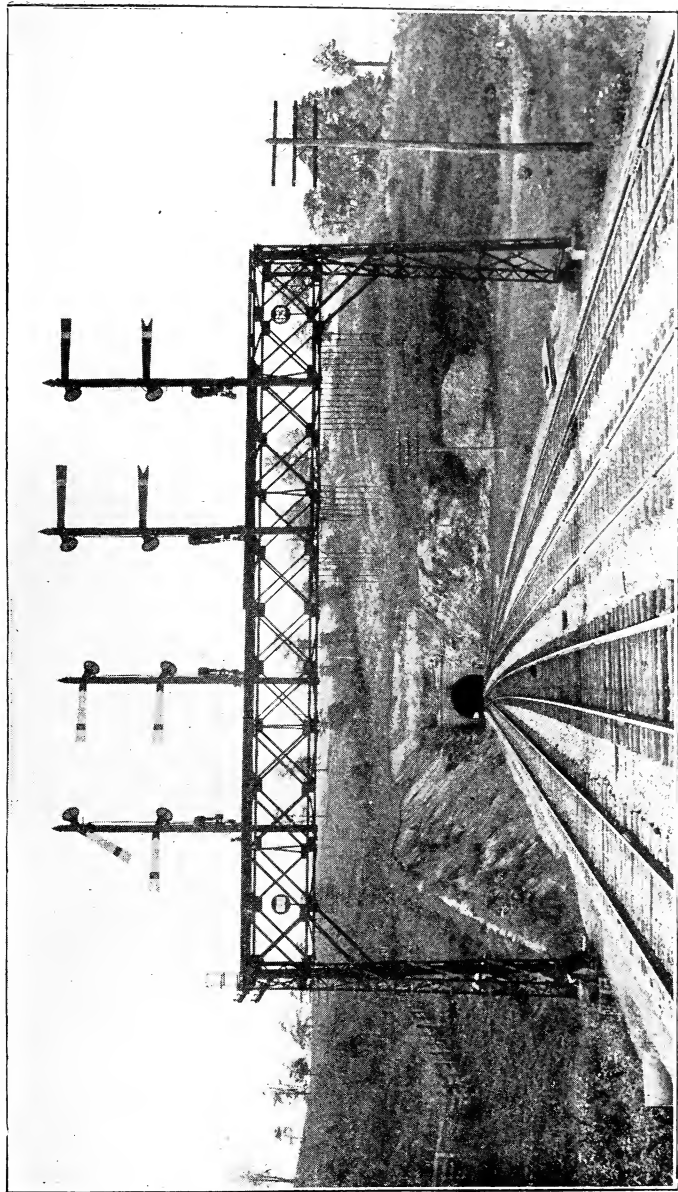
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Courtesy of the Union Switch and Signal Co.

FIG. 168.—SEMAPHORES.

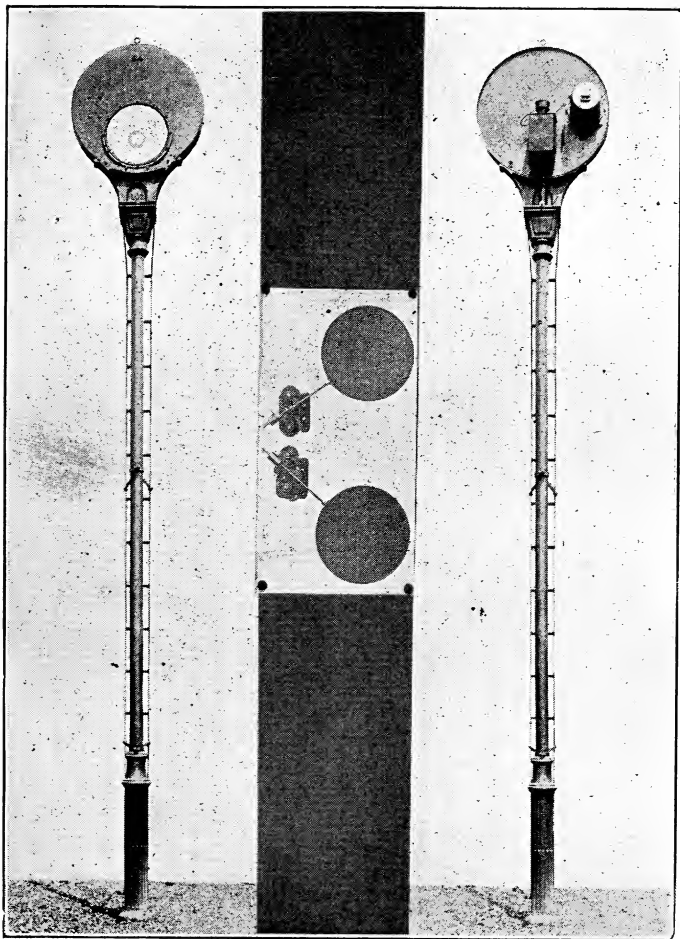




Courtesy of the Union Switch and Signal Co.

FIG. 169.—SIGNAL BRIDGE.

(To face page 358.)



Courtesy of the Union Switch and Signal Co.

FIG. 170.—“ BANJO ” SIGNALS.

The regulations of the A. R. E. & M. W. Assoc. require that "A compensator shall be provided for each pipe line over fifty (50) feet in length and under eight hundred (800) feet, with crank-arms eleven by thirteen (11×13) inch centers. From eight hundred (800) to twelve hundred (1200) feet in length, crank-arms shall be eleven by sixteen (11×16) inch centers. Pipe lines over twelve hundred (1200) feet in length shall be provided with an additional compensator.

"Compensators shall have one sixty (60) degree and one one hundred and twenty (120) degree angle-cranks and connecting

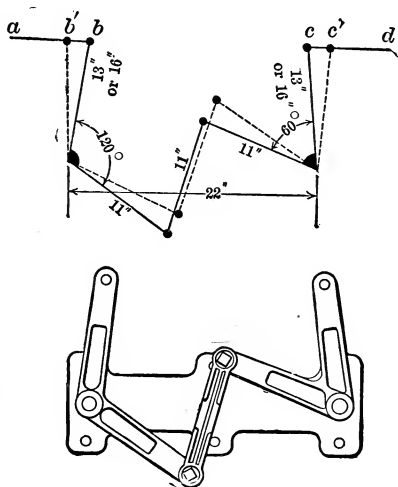


FIG. 171.—STANDARD PIPE COMPENSATOR.

link, mounted in cast iron base, having top of center pins supported. The distance between center of pin-holes shall be twenty-two (22) inches."

The compensator should be placed in the middle of the length when only one is used. When two are used they should be placed at the quarter points. Note that in operating through a compensator the *direction* of motion changes; i.e., if *a* moves to the right, *d* moves to the left, or if there is compression in *ab* there is tension in *cd*, and *vice versa*. Therefore this form of compensator can only be used with pipes which will withstand compression. It has seemed impracticable to design an equally

satisfactory compensator for wires, although there are several designs on the market.

The change of length of these bars is so great that allowance must be made for the temperature at the time of installation. On the basis of 50° as the mean temperature, the pipes are so adjusted that the distance between the points *b* and *c* of Fig. 171 is made greater or less than 22 inches, according to the temperature of installation. For example, if the temperature were 80° and the length of the piping were 900 feet, the length of the pipes should be adjusted so that *bc* is *less* than 22 inches by an amount equal to $900 \times (80^{\circ} - 50^{\circ}) \times .0000065 = 0.1755$ feet = 2.106 inches. The length should therefore be 19.9 inches instead of 22 inches. If the mean temperature was very different (say in Florida) some higher temperature should be taken as normal, so that the extreme range above and below the normal shall be approximately the same.

Guides around curves and angles. When wires are required to pass around curves of large angle, pulleys are used, and a length of chain is substituted for the wire. For pipes, when the curve is easy the pipes are slightly bent and are guided through pulleys. When the angle is sharper, "angles" are used. The operation of these details is self-evident from an inspection of Fig. 172.

310. Track circuit for automatic signaling. The several systems of automatic signaling differ in the minor details, but nearly all of them agree in the following particulars. A current of low potential is run from a battery at one end of a section through one line of rails to the other end of the section, then through a relay, and then back to the battery through the other

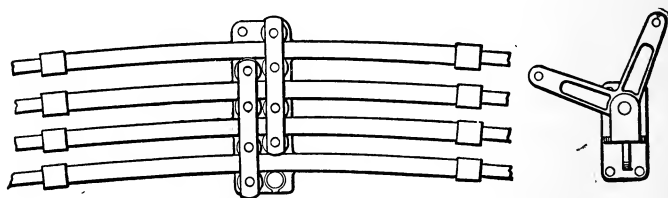


FIG. 172.—DEFLECTING-RODS.

line of rails. To avoid the excessive resistance which would occur at rail joints which may become badly rusted, a wire

suitably attached to the rails is run around each joint. In order to insulate the rails of one section from the rails at either end and yet maintain the rails structurally continuous, the ends of the rails at these dividing points are separated by an insulator and the joint pieces are either made of wood or have some insulating material placed between the rails and the ordinary metal joint. The bolts must also be insulated. When the relay is energized by a current, it closes a local circuit at the signal-station, which will set the signal there at "safety." The resistance of the relay is such that it requires nearly the whole current to work it and to keep the local circuit closed. Therefore, when there is any considerable loss of current from one rail to the other, the relay will not be sufficiently energized, the local circuit will be broken, and the signal will automatically fall to danger. This diversion of current from one rail to the other before the current reaches the relay may be caused in several ways: the presence of a pair of wheels on the rails anywhere in the section will do it; also the breakage of a rail; also the opening of a switch anywhere in the section; also the presence of a pair of wheels on a siding between the "fouling point" and the switch. (The "fouling point" of a siding is that point where the rails first commence to approach the main track.) In Fig. 173 is shown all of the above details as well as some others. At A, B, and the "fouling point" are shown the insulated joints. The batteries and signals are arranged for train motion

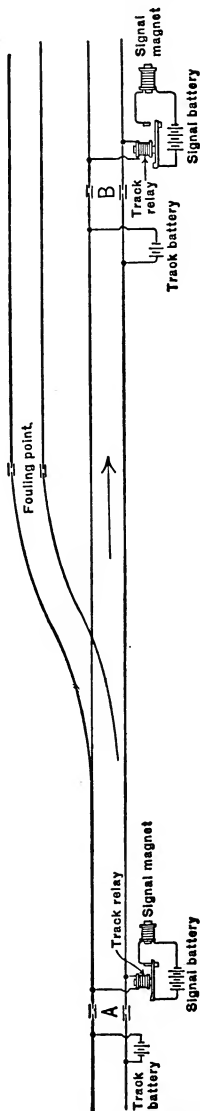


FIG. 173.

to the *right*. When a train has passed the points near *A*, where the wires leave the rails for the relay, the current from the "track battery" at *B* will pass through the wheels and axles, and although no electrical connection is broken, so much current will be shunted through the wheels and axles that the weak current still passing through the relay is not strong enough to energize it against its spring and the "signal-magnet" circuit is broken, and the signal *A* goes to "danger." At the turnout the rails between the fouling point and the switch are so connected (and insulated) that a pair of wheels on these rails will produce the same effect as a pair on the main track. This is to guard against the effect of a car standing too near the switch, even though it is not on the main track. When the train passes *B*, if there is no other interruption of the current, the track battery at *B* again energizes the relay at *A*, the signal-magnet circuit at *A* is closed, and the signal is drawn to "safety."

(The present edition has omitted several subdivisions of this general subject, notably the "staff system," used chiefly in England, and all discussions of "interlocking" which is an essential feature of the operation of large terminal yards. A future edition may supply these deficiencies, although an exhaustive treatment of the subject of Signaling would require a separate volume.)

CHAPTER XV.

ROLLING-STOCK.

(It is perhaps needless to say that the following chapter is in no sense a course in the design of locomotives and cars. Its chief idea is to give the student the elements of the construction of those vehicles which are to use the track which he may design—to point out the mutual actions and reactions of vehicle against track and to show the effect on track wear of variations in the design of rolling-stock. The most of the matter given has a direct practical bearing on track-work, and it is considered that all of it is so closely related to his work that the civil engineer may study it with profit.)

WHEELS AND RAILS.

311. Effect of rigidly attaching wheels to their axles. The wheels of railroad rolling-stock are invariably secured rigidly to the axles, which therefore revolve with the wheels. The chief reason for this is to avoid excessive wear between the axles and the wheels.

Any axle must always be somewhat loose in its journals. A sidewise force P (see Fig. 174) acting against the circumference of the wheel will produce a much greater pressure on the axle at S and S' , and if the wheel moves on the axle, the wear at S and S' will be excessive. But when the axle is fitted to the wheel with a "forced fit" and does not revolve, the mere pressure produced at S is harmless. When two wheels are fitted tight to an axle, as in Fig. 175, and the axle revolves in the journals

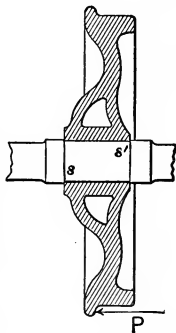


FIG. 174.

aa, a sidewise pressure of the rail against the wheel flange will only produce a slight and harmless increase of the journal pressure Q , although at Q there is sliding contact. Twist-

ing action in the journals is thus practically avoided, since a small pressure at the journal-boxes at each end of the axle suffices to keep the axle truly in line.

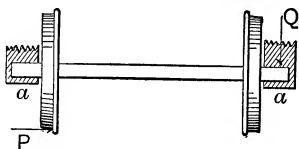


FIG. 175.

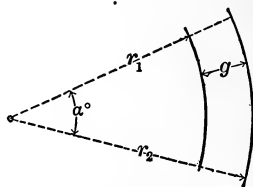


FIG. 176.

On the other hand, when the wheels are rigidly attached to their axles, both wheels must turn together, and when rounding curves, the inner rail being shorter than the outer rail, one wheel must slip by an amount equal to that difference of length. The amount of this slip is readily computable:

$$\text{Longitudinal slip} = \frac{2\pi a^\circ}{360^\circ}(r_2 - r_1) = \frac{2\pi g}{360^\circ}a^\circ = Ca^\circ, \quad (136)$$

in which C is a constant for any one gauge, and g = the track gauge $= (r_2 - r_1)$. For standard gauge (4.708) the slip is .08218 foot per degree of *central angle*. This shows that the longitudinal slipping around any curve of any given central angle will be *independent of the degree of the curve*. The constant (.08218) here given is really somewhat too small, since the true gauge that should be considered is the distance between the lines of tread on the rails. This distance is a somewhat indeterminate and variable quantity, and probably averages 4.90 feet, which would increase the constant to .086. The slipping may occur by the inner wheel slipping ahead or the outer wheel slipping back, or by both wheels slipping. The total slipping will be constant in any case. The slipping not only consumes power, but wears both the wheels and the rail. But even these disadvantages are not sufficient to offset the advantages resulting from rigid wheels and axles.

312. Effect of parallel axles. Trucks are made with two or three parallel axles (except as noted later), in order that the axles shall mutually guide each other and be kept approximately

perpendicular to the rails. If the curvature is very sharp and the wheel-base comparatively long (as is notably the case on street railways at street corners), the front and rear wheels

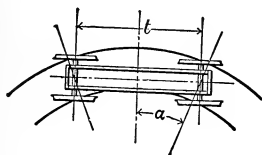


FIG. 177.

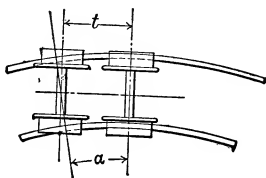


FIG. 178.

will stand at the same angle (a) with the track, as shown in Fig. 177. But it has been noticed that for ordinary degrees of curvature, the rear wheels stand radial to the curve (see Fig. 178), and for steam railroad work this is the normal case. When the two parallel axles are on a curve (as shown), the wheels tend to run in a straight line. In order that they shall run on a curve they must slip laterally. The principle is illustrated in an exaggerated form in Fig. 179. The wheel tends to roll from a toward b . Therefore in passing along the track from a to c it must actually slip laterally an amount bc which equals $ac \sin a$.

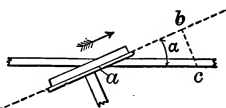


FIG. 179.

Let t = length of the wheel-base (Figs. 177 and 178); r = radius of curve; then for the first case (Fig. 177), $\sin a = t \div 2r$; for the second and usual case (Fig. 178), $\sin a = t \div r$; for $t = 5$ feet and r = radius of a 1° curve, $a = 0^\circ 03'$ for the second case. a varies (practically) as the degree of curve. The lateral slipping *per unit* of distance traveled therefore equals $\sin a$. As an illustration, given a 5-foot wheel-base on a 5° curve, $a = 0^\circ 15'$, $\sin a = .00436$, and for each 100 feet traveled along the curve the lateral slip of the front wheels would be 0.436 foot. There would be no lateral slipping of the rear wheels, assuming that the rear axle maintained itself radial.

From the above it might be inferred that the flanges of the forward wheels will have much greater wear than those of the rear wheels. Since cars are drawn in both directions about equally, no difference in flange wear due to this cause will occur, but locomotives (except switching-engines) run forward almost

exclusively, and the excess wear of the front wheels of the pilot-and tender-trucks is plainly observable.

For a given curve the angle α (and the accompanying resistance) is evidently greater the greater the distance between the axles. On the other hand, if the two axles are very close together, there will be a tendency for the truck to twist and the wheels to become jammed, especially if there is considerable play in the gauge. The flange friction would be greater and would perhaps exceed the saving in lateral slipping. A general rule is that the axles should never be closer together than the gauge.

Although the slipping per unit of length along the curve varies directly as the degree of curvature, the length of curve necessary to pass between two tangents is inversely as the degree of curve, and the total slipping between the two tangents is independent

of the degree of curve. Therefore when a train passes between two tangents, the total slipping of the wheels on the rails, longitudinal and lateral, is a quantity which depends only on the central angle and is independent of the radius or degree of curve.

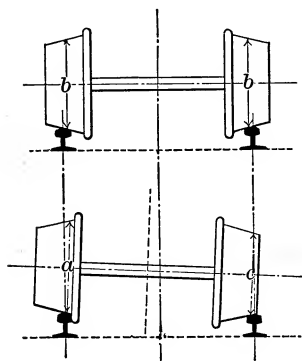


FIG. 180.

313. Effect of coning wheels.

The wheels are always set on the axle so that there is some "play" or chance for lateral motion between the wheel-flanges and the rail. The treads of the wheel are also "coned." This coning and play of gauge are shown in an exaggerated form in Fig. 180. When the

wheels are on a tangent, although there will be occasional oscillations from side to side, the normal position will be the symmetrical position in which the circles of tread bb are equal. When centrifugal force throws the wheel-flange against the rail, the circle of tread a is larger than b , and much larger than c ; therefore the wheels will tend to roll in a circle whose radius equals the slant height of a cone whose elements would pass through the unequal circles a and c . If this radius equalled the radius of the track, and if the axle were free to assume a radial position, the wheels would roll freely on the rails without any

slipping or flange pressure. Under such ideal conditions, coning would be a valuable device, but it is impracticable to have all axles radial, and the radius of curvature of the track is an extremely variable quantity. It has been demonstrated that with parallel axles the influence of coning diminishes as the distance between the axle increases, and that the effect is practically inappreciable when the axles are spaced as they are on locomotives and car-trucks. The coning actually used is very slight (see Chapter XV, § 332) and has a different object. It is so slight that even if the axles were radial it would only prevent the slipping on a very light curve—say a 1° curve.

314. Effect of flanging locomotive driving-wheels. If all the wheels of all locomotives were flanged it would be practically impossible to run some of the longer types around sharp curves. The track-gauge is always widened on curves, and especially on sharp curves, but the widening would need to be excessive to permit a consolidation locomotive to pass around an 8° or 10° curve if all the drivers were flanged. The action of the wheels on a curve is illustrated in Figs. 181, 182, and 184. All small truck-wheels are flanged. The rear drivers are always flanged and four-driver engines usually have all the drivers flanged. Consolidation engines have only the front and rear drivers flanged. Mogul and ten-wheel engines have one pair of drivers blank. On Mogul engines it is always the middle pair. On ten-wheel engines, when used on a road having sharp curves, it is preferable to flange the front and rear driving-wheels and use a "swing bolster" (see § 315); when the curvature is easy, the middle and rear drivers may be flanged and the truck made with a rigid center. The blank drivers have the same total width as the other drivers and of course a much wider tread, which enables these drivers to remain on the rail, even though the curvature is so sharp that the tread overhangs the rail considerably.

315. Action of a locomotive pilot-truck. The purpose of the pilot-truck is to guide the front end of a locomotive around a curve and to relieve the otherwise excessive flange pressure that would be exerted against the driver-flanges. There are two classes of pilot-trucks—(a) those having fixed centers and (b) those having shifting centers. This second class is again subdivided into two classes, which are radically different in their action—(b₁) four-wheeled trucks having two parallel axles

and (b_2) two-wheeled trucks which are guided by a "radius-bar." The action of the four-wheeled fixed-centered truck (a) is shown in Fig. 181. Since the center of the truck is forced

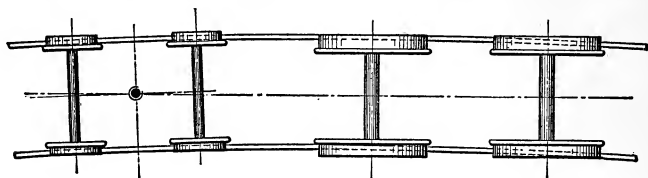


FIG. 181.—FIXED CENTER PILOT-TRUCK.

to be in the center of the track, the front drivers are drawn away from the outer rail. The rear outer driver tends to roll *away* from the outer rail rather than toward it, and so the effect

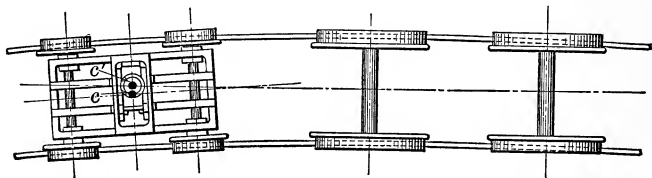


FIG. 182.—FOUR-WHEELED TRUCK—SHIFTING CENTER.

of the truck is to relieve the driver-flanges of any excessive pressure due to curvature. The only exception to this is the case where the curvature is sharp. Then the front inner driver may be pressed against the *inner* rail, as indicated in Fig. 181.

This limits the use of this type of wheel-base on the sharper curves.

The next type—(b_1) four-wheeled trucks with shifting centers—is much more flexible on sharp curvature; it likewise draws the front drivers away from the outer rail. The relative position of the wheels is shown in Fig. 182, in which c' represents the position of center-pin and c the displaced truck center. The structure and action of the truck is shown in Fig. 183. The "center-pin" (1) is

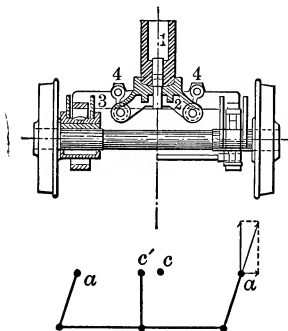


FIG. 183.—ACTION OF SHIFTING CENTER.

supported on the "truck-bolster" (2), which is hung by the "links" (4) from the "cross-ties" (3). The links are therefore

in tension and when the wheels are forced to one side by the rails the *links* are inclined and the front of the engine is drawn inward by a force equal to the weight on the bolster times the tangent of the angle of inclination of the links. This assumes that all links are vertical when the truck is in the center. Frequently the opposite links are normally inclined to each other, which somewhat complicates the above simple relation of the forces, although the general principle remains identical.

The two-wheeled pilot-truck with shifting center is illustrated in Fig. 184. The figure shows the facility with which

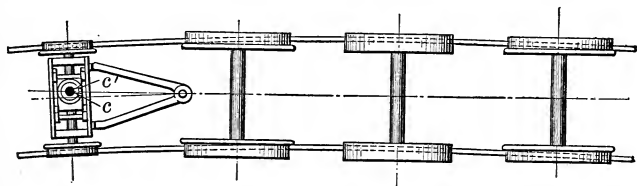


FIG. 184.—TWO-WHEELED TRUCK—SHIFTING CENTER.

an engine with long wheel-base may be made to pass around a comparatively sharp curve by omitting the flanges from the middle drivers and using this form of pilot-truck. As in the previous case, the eccentricity of the center of the truck relative to the center-pin induces a centripetal force which draws the front of the engine inward. But the swing-truck is not the only source of such a force. If the

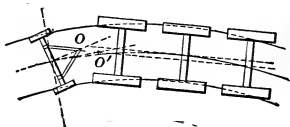


FIG. 185.—ACTION OF TWO-WHEELED TRUCK.

“radius-bar pin” were placed at O' (see Fig. 185), the truck-axle would be radial. But the radius-bar is always made somewhat shorter than this, and the pin is placed at O , a considerable distance ahead of O' , thus creating a tendency for the truck to run toward the inner rail and draw the front of the locomotive in that direction. This tendency will be objectionably great if the radius-bar is made too short, as has been practically demonstrated in cases when the radius-bar has been subsequently lengthened with a resulting improvement in the running of the engine. This type of pilot truck is used on both Mogul and Consolidation locomotives and explains why these long engines can so easily operate on sharp curves.

LOCOMOTIVES.

GENERAL STRUCTURE.

316. Frame. The frame or skeleton of a locomotive consists chiefly of a collection of forged wrought-iron bars, as shown in Figs. 186 and 187. These bars are connected at the

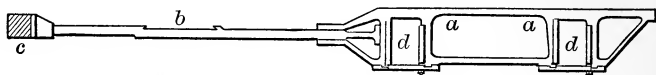


FIG. 186.—ENGINE-FRAME.

front end by the “bumper” (c), which is usually made of wood. A little further back they are rigidly connected at *bb* by the cylinders and boiler-saddle. The boilers rest on the frames at *aaaa* by means of “pads,” which are bolted to the fire-box, but which permit a free expansion of the boiler along the frame. This expansion is sometimes as much as $\frac{5}{16}$ ". On a “consolidation” engine (frame shown in Fig. 187) it is frequently

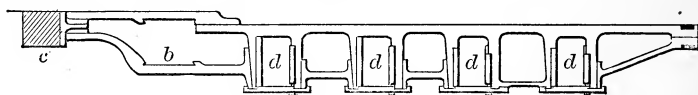


FIG. 187.—ENGINE-FRAME—CONSOLIDATION TYPE.

necessary to use vertical swing-levers about 12" long instead of “pads.” The swinging of the levers permit all necessary expansion. At the back the frames are rigidly connected by the iron “foot-plate.” The driving-axles pass through the “jaws” *dddd*, which hold the axle-boxes. The frame-bars have a width (in plan) of 3" to 4". The depth (at *a*) is about the same. Fig. 186 shows a frame for an “American” type of locomotive; Fig. 187 shows a frame for a “Consolidation” type (see § 323).

317. Boiler. A boiler is a mechanism for transferring the latent heat of fuel to water, so that the water is transformed from cold water into high-pressure steam, which by its expansion will perform work. The efficiency of the boiler depends largely on its ability to do its work rapidly and to reduce to a minimum the waste of heat through radiation. The boiler contains a fire-box (see Fig. 188), in which the fuel is burned. The gases of consumption pass from the fire-box through the numerous boiler-tubes into the “smoke-box” *S* and out through the smoke-stack. The fire-box consists of an inner and outer

shell separated by a layer of water about 3" thick. The exposure of water-surface to the influence of the fire is thus very complete. The efficiency of this transference of heat is somewhat indicated by the fact that, although the temperature of the gases in the fire-box is probably from 3000° to 4000° F., the temperature in the smoke-box is generally reduced to 500° to

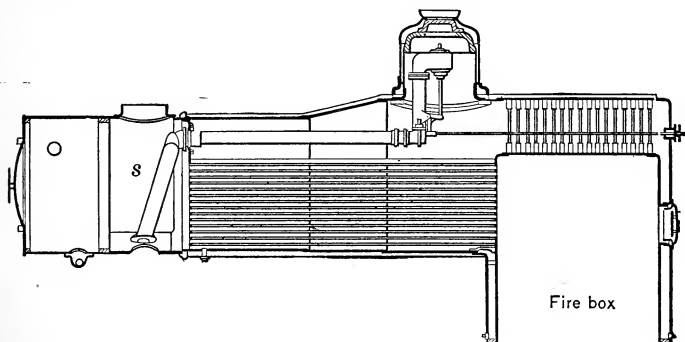


FIG. 188.—LOCOMOTIVE-BOILER.

600° F. If the steam pressure is 180 lbs., the temperature of the water is about 380° F., and, considering that heat will not pass from the gas to the water unless the gas is hotter than the water, the water evidently absorbs a large part of the theoretical maximum. Nevertheless gases at a temperature of 600° F. pass out of the smoke-stack and such heat is utterly wasted.

The tubes vary from $1\frac{3}{4}$ " to 2", inside diameter, with a thickness of about 0".10 to 0".12. The aggregate cross-sectional area of the tubes should be about one eighth of the grate area. The number will vary from 140 to 250. They are made as long as possible, but the length is virtually determined by the type and length of engine.

318. Fire-box. The fire-box is surrounded by water on the four sides and the top, but since the water is subjected to the boiler pressure, the plates, which are about $\frac{5}{16}$ " thick, must be stayed to prevent the fire-box from collapsing. This is easily accomplished over the larger part of the fire-box surface by having the outside boiler-plates parallel to the fire-box plates and separated from them by a space of about 3". The plates are then mutually held by "stay-bolts." See Fig. 189. These are about $\frac{7}{8}$ " in diameter and spaced 4" to $4\frac{1}{2}$ ". The $\frac{3}{16}$ " hole,

drilled $1\frac{1}{4}$ " deep, indicated in the figure, will allow the escape of steam if the bolt breaks just behind the plate, and thus calls attention to the break. The stay-bolts are turned down to a diameter equal to that at the root of the screw-threads. This method of supporting the fire-box sheets is used for the two sides, the entire rear, and for the front of the fire-box up to the boiler-barrel. The "furnace tube-sheet"—the upper part of the front of the fire-box—is stayed by the tubes. But the top of the fire-box is troublesome. It must always be covered with water so that it will not be "burned" by the intense heat. It must therefore be nearly, if not quite, flat. There are three general methods of accomplishing this.

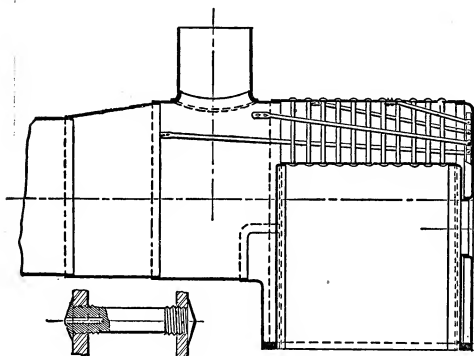


FIG. 189.

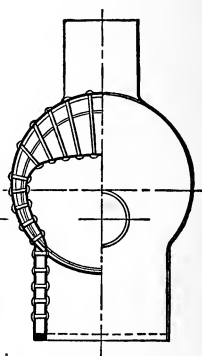


FIG. 190.

(a) **Radial stays.** This construction is indicated in Fig. 190. Incidentally there is also shown the diagonal braces for resisting the pressure on the back end of the boiler above the fire-box. It may be seen that the stays are not perpendicular to either the crown-sheet or the boiler-plate. This is objectionable and is obviated by the other methods.

(b) **Crown-bars.** These bars are in pairs, rest on the side furnace-plates, and are further supported by stays. See Fig. 191.

(c) **Belpaire fire-box.** The boiler above the fire-box is rectangular, with rounded corners. The stays therefore are perpendicular to the plates. See Fig. 192.

Fire-brick arches. These are used, as shown in Fig. 193, to force all the gases to circulate through the upper part of the fire-

box. Perfect combustion requires that all the carbon shall be turned into carbon dioxide, and this is facilitated by the forced circulation.

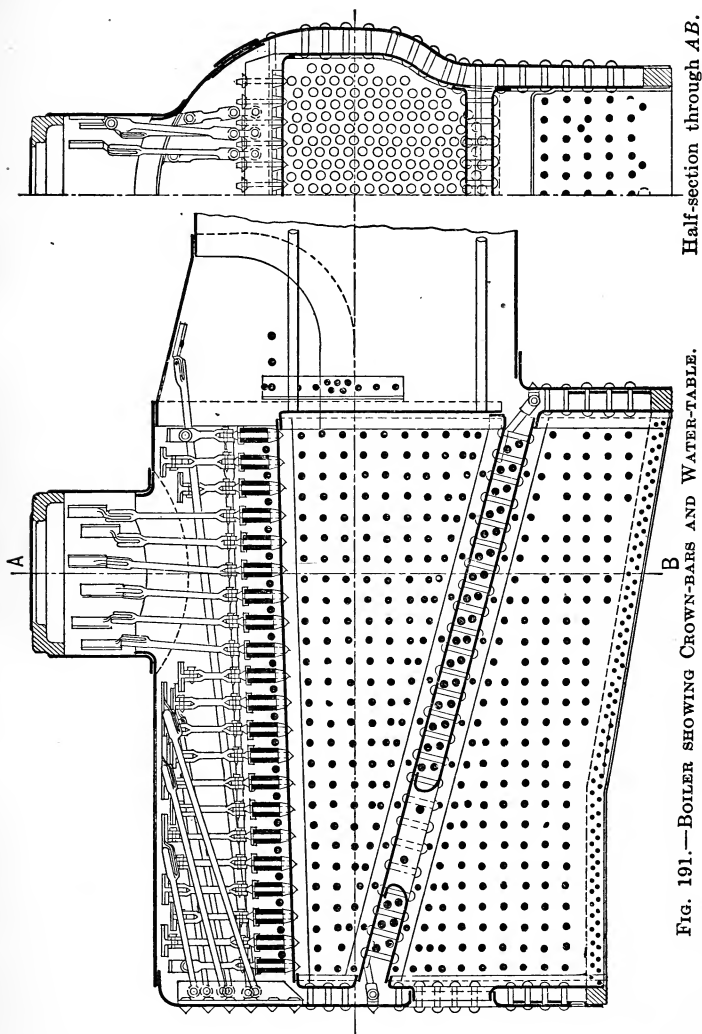


FIG. 191.—BOILER SHOWING CROWN-BARS AND WATER-TABLE.

Water-tables. The same object is attained by using a water-table instead of a brick arch—as shown in Fig. 191. But it has

the further advantages of giving additional heating-surface and avoiding the continual expense of maintaining the bricks. One

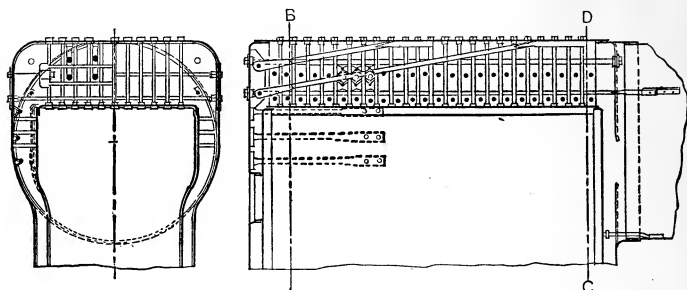


FIG. 192.—"BELPAIRE" FIRE-BOX.

Half-section through AB.

Half-section through CD.

feature of the design is the use of a number of steam-jets which force air into the fire-box and assist the combustion.

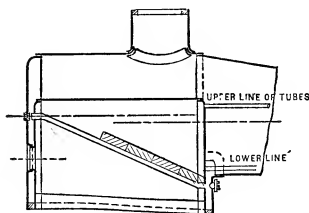


FIG. 193.—FIRE-BRICK ARCH.

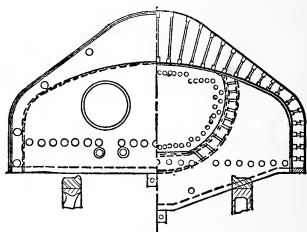


FIG. 194.—WOOTTEN FIRE-BOX.

Area. Fire-boxes are usually limited in width to the practicable width between the wheels—thus giving a net inside width of about 3 feet and a maximum length of 10 to 11 feet—this being about the maximum distance over which the firemen can properly control the fire. About 37 square feet is the maximum area obtainable except when the "Wootten" fire-box is used—illustrated in Fig. 194. Here the grate is raised above the driving-wheels and has (in the case shown) a width of 8' 0 $\frac{3}{8}$ ". The fire-box area is over 76 square feet. Note that *two* furnace-doors are used.

319. Coal consumption. No form of steam-boiler (except a boiler for a steam fire-engine) requires as rapid production of steam, considering the size of the boiler and fire-box, as a

locomotive. The combustion of coal per square foot of grate per hour for stationary boilers averages about 15 to 25 lbs. and seldom exceeds that amount. An ordinary maximum for a locomotive is 125 lbs. of coal per square foot of grate-area per hour, and in some recent practice 220 lbs. have been used. Of course such excessive amounts are wasteful of coal, because a considerable percentage of the coal will be blown out of the smoke-stack unconsumed, the draft necessary for such rapid consumption being very great. The only justification of such rapid and wasteful coal consumption is the necessity for rapid production of steam. The best quality of coal is capable of evaporating about 14 lbs. of water per pound of coal, i.e., change it from water at 212° to steam at 212° ; the heat required to change water at ordinary temperatures to steam at ordinary working pressure is (roughly) about 20% more. From 6 to 9 lbs. of water per pound of coal is the average performance of ordinary locomotives, the efficiency being less with the higher rates of combustion. Some careful tests of locomotive coal consumption gave the following figures: when the consumption of coal was 50 lbs. per square foot of grate-area per hour, the rate of evaporation was 8 lbs. of water per pound of coal. When the rate of coal consumption was raised to 180, the evaporation dropped to 5 lbs. of water per pound of coal. It has been demonstrated that the efficiency of the boiler is largely increased by an increased length of boiler-tubes. The actual consumption of coal per mile is of course an exceedingly variable quantity, depending on the size and type of the engine and also on the work it is doing—whether climbing a heavy grade with its maximum train-load or running easily over a level or down grade. A test of a 50-ton engine, running without any train at about 20 to 25 miles per hour, showed an average consumption of 21 lbs. of coal per mile. Statistics of the Pennsylvania Railroad show a large increase (as might be expected, considering the growth in size of engines and weight of trains) in the average number of pounds of coal burned per *train-mile*—some of the figures being 55 lbs. in 1863, 72 lbs. in 1872, and nearly 84 lbs. in 1883. Figures are published showing an average consumption of about 10 lbs. of coal per passenger-car mile, and 4 to 5 lbs. per freight-car mile. But these figures are always obtained by dividing the total consumption per train-mile by the number of cars, the coal due to the weight of the engine

being thrown in Wellington developed a rule, based on the actual performance of a very large number of passenger-trains, that the number of pounds of coal per mile = $21.1 + 6.74$ times the number of passenger-cars. The amount of coal assigned to the engine agrees remarkably with the test noted above. For freight-trains the amount assigned to the engine should be much greater (since the engine is much heavier), and that assigned to the individual cars much less, although the great increase in freight-car weights in recent years has caused an increase in the coal required per car.

320. Heating-surface. The rapid production of steam requires that the hot gases shall have a large heating-surface to which they can impart their heat. From 50 to 75 square feet of heating-surface is usually designed for each square foot of grate-area. A more recently used rule is that there should be from 60 to 70 square feet of *tube* heating-surface per square foot of grate-area for bituminous coal. 40 or 50 to 1 is more desirable for anthracite coal. Almost the whole surface of the fire-box has water behind it, and hence constitutes heating-surface. Although this surface forms but a small part of the total (nominally), it is really the most effective portion, since the difference of temperature of the gases of combustion and the water is here a maximum, and the flow of heat is therefore the most rapid. The heating-surface of the tubes varies from 85 to 93% of the total, or about 7 to 15 times the heating-surface in the fire-box. Sometimes the heating-surface is as much as 2300 square feet, but usually it is less than 2000, even for engines which must produce steam rapidly.

A "Mikado" type locomotive, wheel plan, 2-8-2 (see § 323), constructed in 1912, has cylinders 28" diameter, 32" stroke; drivers, 64" diameter; steam pressure, 180 lbs.; grate area, 78 sq.ft.; heating surface, 4627 sq.ft.; besides a superheating surface of 961 sq. ft.; total weight of engine and tender, 480,000 lbs.; weight of engine alone, 303,400 lbs.; weight on drivers, 231,000 lbs.; tractive power 60,000. There is about 71 sq.ft. of heating surface per square foot of grate area. The adhesion ratio is 1: 3.85 or about 26%.

Another rule used by designers is that the engine should have 1 sq. ft. of heating-surface for each 50 or 60 lbs. of weight, efficiency being indicated by a low weight. For the above engine the ratio is 54.

321. **Loss of efficiency in steam pressure.** The effective work done by the piston is never equal to the theoretical energy contained in the steam withdrawn from the boiler. This is due chiefly to the following causes:

(a) The steam is "**wire-drawn**," i.e., the pressure in the cylinder is seldom more than 85 to 90% of the boiler pressure. This is due largely to the fact that the steam-ports are so small that the steam cannot get into the cylinder fast enough to exert its full pressure. It is often purposely wire-drawn by partially closing the throttle, so that the steam may be used less rapidly.

(b) **Entrained water.** Steam is always drawn from a dome placed over the boiler so that the steam shall be as far above the water-surface as possible, and shall be as dry as possible. In spite of this the steam is not perfectly dry and carries with it water at a temperature of, say, 361° , and pressure of 140 lbs. per square inch. When the pressure falls during the expansion and exhaust, this hot water turns into steam and absorbs the necessary heat from the hot cylinder-walls. This heat is then carried out by the exhaust and wasted.

(c) The **back pressure** of the exhaust-steam, which depends on the form of the exhaust-passages, etc. This amounts to from 2 to 20% of the power developed.

(d) **Clearance-spaces.** When cutting off at full stroke this waste is considerable (7 to 9%), but when the steam is used expansively the steam in these clearance-spaces expands and so its power is not wholly lost.

(e) **Radiation.** In spite of all possible care in jacketing the cylinders, some heat is lost by radiation.

(f) **Radiation into the exhaust-steam.** This is somewhat analogous to (b). Steam enters the cylinder at a temperature of, say, 361° ; the walls of the cylinder are much cooler, say 250° ; some heat is used in raising the temperature of the cylinder-walls; some steam is vaporized in so doing; when the exhaust is opened the temperature and pressure fall; the heat temporarily absorbed by the cylinder-walls is reabsorbed by the exhaust-steam, re-evaporating the vapor previously formed, and thus a certain portion of heat-energy goes **through** the cylinder without doing any useful work. With an early cut-off the loss due to this cause is very great.

The sum of all these losses is exceedingly variable. They are usually less at lower speeds. The loss in *initial pressure*

(the difference between boiler pressure and the cylinder pressure at the beginning of the stroke) is frequently over 20%, but this is not all a net loss. With an early cut-off the average cylinder pressure for the whole stroke is but a small part of the boiler pressure, yet the horse-power developed may be as great as, or greater than, that developed at a lower speed, later cut-off, and higher average pressure.

322. Tractive power The work done by the two cylinders during a complete revolution of the drivers evidently = area of pistons $\times$ average steam pressure $\times$ stroke $\times 2 \times 2$. The resistance overcome evidently = tractive force at circumference of drivers times distance traveled by drivers (which is the circumference of the drivers). Therefore

$$\text{Tractive force} = \left\{ \frac{\text{area pistons} \times \text{average steam pressure} \times \text{stroke} \times 2 \times 2}{\text{circumference of drivers}} \right\}.$$

Dividing numerator and denominator by π (3.1415), we have

$$\text{Tractive force} = \left\{ \frac{(\text{diam piston})^2 \times \text{average steam pressure} \times \text{stroke}}{\text{diameter of driver}} \right\}, \quad (137)$$

which is the usual rule. Although the rule is generally stated in this form, there are several deductions. In the first place the net effective area of the piston is less than the nominal on account of the area of the piston-rod. The ratio of the areas of the piston-rod and piston varies, but the effect of this reduction is usually from 1.3 to 1.7%. No allowance has been made for friction—of the piston, piston-rod, cross-head, and the various bearings. This would make a still further reduction of several per cent. Nevertheless the above simple rule is used, because, as will be shown, no great accuracy can be utilized.

The tractive force is limited by the adhesion between the drivers and the rails, and this is a function of the weight on the drivers. Under the most favorable conditions this has been tested to amount to one-third the weight on the drivers, but such a ratio cannot be depended on. Wellington used the ratio one-fourth. The Baldwin Locomotive Works in their "Locomotive Data" give tables and diagrams based on $\frac{1}{4}$, $\frac{2}{3}$,

and $\frac{1}{5}$ adhesion. As low a value as $\frac{1}{6}$ or even $\frac{1}{7}$ is occasionally used, but such a low rate of adhesion would only be found when the rails were abnormally slippery. In a well-designed locomotive the tractive force, as computed above, and the tractive adhesion should be made about equal. The uncertainty in the coefficient of adhesion shows the futility of any refinement in the computation of the tractive force.

It is only at very slow speeds that an engine can utilize all of its tractive force. When running at a high speed, the utmost horse-power that the engine can develop will only produce a draw-bar pull, which is but a small part of the possible tractive force. Power is the product of force times velocity. If the power is constant and the velocity increases, the force must decrease. This fact is well shown in the figures of some tests of a locomotive. The dimensions were as follows: cylinders, 18"×24"; drivers, 68"; weight on drivers, 60000 lbs.; heating-surface, 1458 sq. ft.; grate-area, 17 sq. ft. During one test the average cylinder pressure was 83.3 lbs. (boiler pressure, 145; 14-inch cut-off and throttle $\frac{3}{4}$ open). By the above formula (137),

$$\text{Tractive force} = \frac{18^2 \times 83.3 \times 24}{68} = 9525 \text{ lbs.}$$

At $\frac{1}{4}$ adhesion the tractive force was 15000 lbs; even at $\frac{1}{5}$ adhesion, it would be 12000 lbs. This shows that at the speed of this test (26.3 in. per hour) scarcely more than $\frac{2}{3}$ of the tractive power was utilized. A still more marked case, shown by another test with the same engine, taken when the speed was 53.4 miles per hour, indicated an average cylinder pressure of 37.2 lbs., the throttle being $\frac{1}{2}$ open and the valves cutting off at 8". In this case the tractive power, computed as before, equals 4254 lbs., about $\frac{1}{4}$ of the weight on the drivers and about $\frac{1}{3}$ of the tractive force which is possible at slow speeds. In the first case, the tractive power (9525) times the speed in feet per second (38.57) divided by 550 gives the indicated horse-power, 668. In the second case, although the tractive force developed was so much less, the speed was much greater and the horse-power was about the same, 606.

The above figures illustrate some of the foregoing statements regarding loss of efficiency. In both cases the steam was wire-drawn. The boiler pressure was 145 lbs.; but when the throttle

was only $\frac{3}{4}$ open and the steam was cut-off at 14" (24" stroke) the average steam pressure in the cylinder was reduced to 83.3 lbs. With the throttle but $\frac{1}{2}$ open and the valves cutting off at 8" ($\frac{1}{3}$ of the stroke), the average pressure was cut down to 37.2 lbs.—about $\frac{1}{4}$ of the boiler pressure. Note that the heating-surface per square foot of grate-area ($1458 \div 17 = 86$) is very large (see § 320). Note also that the horse-power developed divided by the grate-area (17) gives 39 and 36 H.P. per square foot of grate-area. This is exceptionally large—25 or 30 being a more common figure.

The maximum tractive power is required when a train is starting, and fortunately it is at low velocities that the maximum tractive force can be developed. The motion of the piston is so slow that there is but little reduction of steam pressure, and the valves are generally placed to cut off at full stroke. For the above engine, with 145 lbs. boiler pressure, the absolute maximum of tractive force is $\frac{18^2 \times 145 \times 24}{68} =$

16581 lbs. Of course, this maximum would never be reached unless the boiler pressure were increased. A common rule is to consider that the average effective cylinder pressure for slow speed and full stroke will be 80% of the boiler pressure. This would reduce the tractive force to the (nominal) value of 13265 lbs., and the corresponding cylinder pressure would be 116 lbs. per square inch. With an effective cylinder pressure of about 131 lbs. the tractive power is 15000 lbs., which is $\frac{1}{4}$ of the total weight on the drivers. This illustrates the general rule, stated above, that the cylinders, drivers, and boiler pressure should be so proportioned that the maximum tractive force should about equal the maximum adhesion which could be obtained.

As another numerical example, the dimensions of a recently constructed heavy consolidation engine are quoted. The cylinders are 24" $\times$ 32"; diameter of drivers, 54"; total weight of engine and tender, 391400 lbs.; weight of engine, 250300 lbs.; weight on drivers, 225200 lbs.; capacity of tender, 7500 gallons; the boiler has 406 tubes, 24" in diameter and 15' long; fire-box, 132" $\times$ 40 $\frac{1}{4}$ "; heating-surface of tubes, 3564 sq. ft.; of fire-box, 241 sq. ft.—total, 3805 sq. ft.; boiler pressure, 220 lbs. per square inch. Applying Eq. 132, we may compute 75093 lbs. as the absolute maximum of tractive power. In fact this is an unattainable limit, for reasons before stated. The trac-

tive force is given as 63000, which corresponds to an effective cylinder pressure of about 185 lbs., about 84% of the boiler pressure. This tractive force is 28% of the weight on the drivers, a tractive ratio of 1:3.6.

RUNNING GEAR.

323. Types of running gear. (a) "American." This was once the almost universal type for both passenger and freight service. It is still very commonly used for passenger service, but it is not the best form for heavy freight work.

(b) "Columbia." Four drivers, one pair of pilot-truck wheels and one pair of trailing wheels behind the drivers. The low trailing wheels permit a desirable enlargement of the fire-box. This is a recent type, used exclusively for passenger service.

(c) "Atlantic." Similar to b except that the pilot-truck has four wheels instead of two.

(d) "Mogul." These are used for both passenger and freight service, but are not well adapted for either high speed or great tractive power.

(e) "Ten-wheel." Similar to d except that the pilot-truck has four wheels instead of two. The use is similar to that of d.

(f) "Consolidation." The present standard for freight service. It permits great tractive power without excessive concentrated loads on the track.

(g) Switching-engines. These have four or six (and exceptionally even eight or ten) drivers and no truck-wheels. They are only adapted for slow speed when a maximum of tractive power is needed. Sometimes the water-tank and even a small fuel-box is loaded on. Since fuel is always near at hand for a yard-engine, the fuel-box need not be large.

(h) "Double-enders." As explained in § 315, truck-wheels are needed in front of the drivers to guide them around curves. If an ordinary engine is run backward, the flanges of the rear

drivers will become badly worn, and if the speed is high, the danger of derailment is considerable. In suburban service, when the runs are short, it is preferable to run the engines forward and backward, rather than turn them at each end of the route. Therefore a pilot-truck is placed at each end.

(i) **"Miscellaneous types."** Almost every conceivable combination of drivers and truck-wheels has been used. The "Mastodon" is similar to the "Consolidation" except that the pilot-truck has four wheels instead of two. The "Decapod" has ten driving-wheels. The "Forney" (named after the inventor) has been very extensively used on elevated roads. The weight of the boiler and machinery is carried on four driving-wheels; the engine-frame is extended so as to include a small tank and fuel-box, the weight of which is chiefly supported by a truck of two or four wheels. They run best when running "backward," i.e., tender first.

The great variation in types of running gear which has been developed in recent years, has started a convenient and unmistakable method of indicating the running gear. Commencing with the front of the engine (or pilot) always at the **LEFT** (instead of at the right, as in the illustrations above) the number of wheels of the pilot truck on *both* rails is indicated by 0, 2 or 4, according as there is *no* pilot truck, a two-wheeled or a four-wheeled pilot truck. Then the number of drivers on both rails is indicated by the next number and the number of trailing wheels by the third number. The running gear of the tender is not indicated. This method may be illustrated by applying it to the types indicated above:

American.....	4-4-0
Columbia.....	2-4-2
Atlantic.....	4-4-2
Mogul.....	2-6-0
Ten-wheel.....	4-6-0
Consolidation.....	2-8-0
Six-wheel switcher.....	0-6-0
Mastodon.....	4-8-0

The running gear of any new type may thus be unmistakably indicated by three figures.

324. Equalizing-levers. The ideal condition of track, from the standpoint of smooth running of the rolling stock, is that the rails should always lie in a plane surface. While this condition is theoretically possible on tangents, it is unobtainable on curves, and especially on the approaches to curves when the outer rail is being raised. Even on tangents it is impossible to *maintain* a perfect surface, no matter how perfectly the track may have been laid. In consequence of this, the points of contact of the wheels of a locomotive, or even of a four-wheeled truck, will not ordinarily lie in one plane. The rougher and more defective the track, the worse the condition in this respect. Since the frame of a locomotive is practically rigid, if the frame rests on the driver-axles through the medium of springs at each axle-bearing, the compression of the springs (and hence the pressure of the drivers on the rail) will be variable if the bearing-points of the drivers are not in one plane. This variable pressure affects the tractive power and severely strains the frame. Applying the principle that a tripod will stand on an uneven surface, a mechanism is employed which *virtually* supports the locomotive on three points, of which one is usually the center-bearing of the forward truck. On each side the pressure is so distributed among the drivers that even if a driver rises or falls with reference to the others, the load carried by each driver is unaltered, and that side of the engine rises or falls by one n th of the rise or fall of the single driver, where n represents the number of wheels. The principle involved is shown in an exaggerated form in Fig. 195. In the diagram, MN represents the normal position of the frame when the wheels are on line. The frame is supported by the hangers at a , c , f , and h . ab , de , and gh are horizontal levers vibrating about the points H , K , and L , which are supported by the axles. While it is *possible* with such a system of levers to make MN assume a position not parallel with its natural position, yet, by an extension of the principle that a beam balance loaded with equal weights will always be horizontal, the effect of raising or lowering a wheel will be to move MN parallel to itself. It only remains to determine *how much* is the motion of MN relative to the rise or drop of the wheel.

The dotted lines represent the positions of the wheels and levers when one wheel drops into a depression. The wheel center drops from p to q , a distance m . L drops to L' , a distance m (see Fig. 195, b); M drops to M' , an unknown dis-

tance x ; therefore $aa' = x$; $bb' = x$; $cc' = x$; $dd' = 3x = ee'$; $ff' = x$; $\therefore gg' = 5x$; $hh' = x$; $LL' = \frac{1}{2}(gg' + hh') = \frac{1}{2}(6x) = m$; $\therefore x = \frac{1}{3}m$; i.e., MN drops, parallel to itself, $1/n$ as much as the wheel drops, where n is the number of wheels. The resultant effect caused by the simultaneous motion of two wheels with refer-

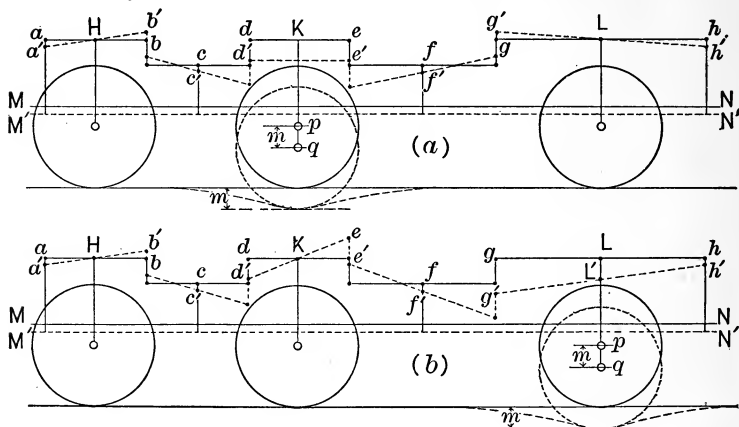


FIG. 195.—ACTION OF EQUALIZING-LEVERS.

ence to the third is evidently the algebraic sum of the effects of each wheel taken separately.

The practical benefits of this device are therefore as follows:

(a) When any driver reaches a rough place in the track, a high place or a low place, the stress in all the various hangers and levers is unchanged.

(b) The motion of the frame (represented by the bar MN in Fig. 195) is but $1/n$ of the motion of the wheel, and the jar and vibration caused by a roughness in the track is correspondingly reduced.

The details of applying these principles are varied, but in general it is done as follows;

(a) **American and ten-wheeled types.** Drivers on each side form a system. The center-bearing pilot-truck is the third point of support. The method is illustrated in Fig. 196.

(b) **Mogul and consolidation types.** The front pair of drivers is connected with the two-wheeled pilot-truck (as illustrated in Fig. 197) to form one system. The remaining drivers on each side are each formed into a system

The device of equalizers is an American invention. Until recently it has not been used on foreign locomotives. The necessity for its use becomes less as the track is maintained

with greater perfection and is more free from sharp curves. A locomotive not equipped with this device would deteriorate

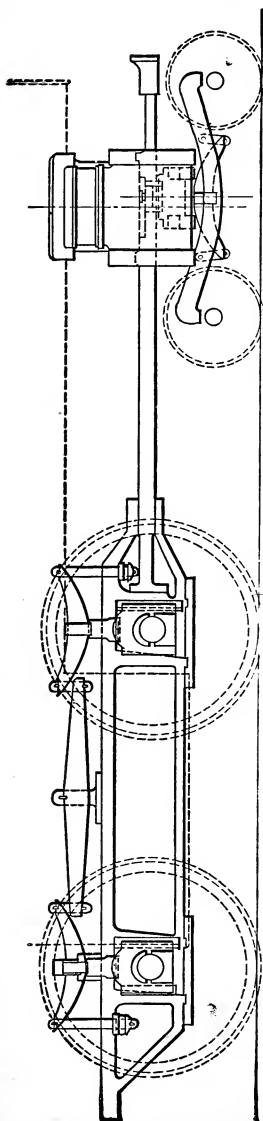


FIG. 196.—EQUALIZING-LEVERS FOR "AMERICAN" TYPE.

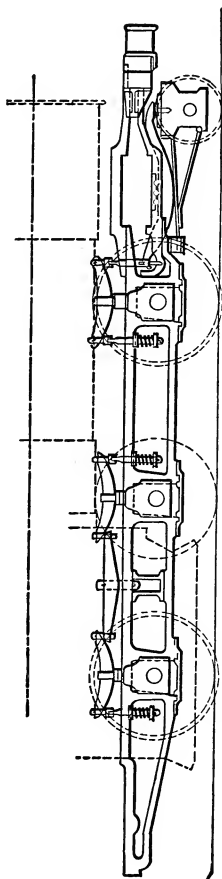


FIG. 197.—EQUALIZING-LEVERS FOR "MOGUL" TYPE.

very rapidly on the comparatively rough tracks which are usually found on light-traffic roads. It is still an open ques-

tion to what extent the neglect of this device is responsible for the statistical fact that average freight-train loads on foreign trains are less in proportion to the weight on the drivers than is the case with American practice. The recent increasing use of this device on foreign heavy freight locomotives is perhaps an acknowledgment of this principle.

325. Counterbalancing. At very high velocities the centrifugal force developed by the weight of the rotating parts becomes a quantity which cannot be safely neglected. These rotating parts include the crank-pin, the crank-pin boss, the side rod, and that part of the weight of the connecting-rod which may be considered as rotating about the center of the crank-driver. As a numerical illustration, a driving-wheel 62" in diameter, running 60 miles per hour, will revolve 325 times per minute. The weights are:

Crank-pin.....	110 lbs.
" boss.....	150 "
One-half side rod.....	240 "
Back end of connecting-rod.....	190 "
Total.....	690 lbs.

If the stroke is 24", the radius of rotation is 12", or 1 foot. Then

$$\frac{Gv^2}{gr} = \frac{690 \times 4\pi^2 1^2 \times 325^2}{32.2 \times 1 \times 60^2} = 24821 \text{ lbs.},$$

which is half as much again as the weight on a driver, 16000 lbs. Therefore if *no* counterbalancing were used, the pressure between the drivers and the rail would always be less (at any velocity) when the crank-pin was at its highest point. At a velocity of about 48 miles per hour the pressure would become zero, and at higher velocities the wheel would actually be thrown from the rail. As an additional objection, when the crank-pin was at the lowest point, the rail pressure would be increased (velocity 60 miles per hour) from 16000 lbs. to nearly 41000 lbs., an objectionably high pressure. These injurious effects are neutralized by "counterbalancing." Since all of the above-mentioned weights can be considered as concentrated at the center of the crank-pin, if a sufficient weight is so placed in the drivers that the center of gravity of the eccentric weight is diametrically opposite to the crank-pin, this centrifugal force can be wholly balanced. This is done by filling up a portion of the space between the spokes. If the center of gravity of the counterbalancing weight is 20" from the center,

then, since the crank-pin radius is 12", the required weight would be $690 \times \frac{12}{20} = 414$ lbs.

In addition to the effect of these revolving parts there is the effect of the sudden acceleration and retardation of the reciprocating parts. In the engine above considered the weights of these reciprocating parts will be:

Front end of connecting-rod.....	150 lbs.
Cross-head.....	174 "
Piston and piston-rod.....	300 "
Total.....	624 lbs.

Assume as before that the reciprocating parts may be considered as concentrated at one point, the point *P* of the diagram in Fig. 198. Since the motion of *P* is horizontal only, the force required to overcome its inertia at any point will exactly equal the *horizontal component* of the force required to overcome the inertia of an *equal weight at S* revolving in

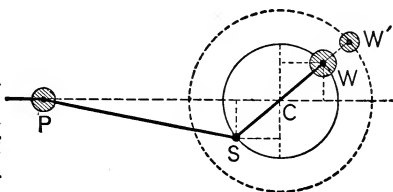


FIG. 198.—ACTION OF COUNTERBALANCE.

a circular path. Then evidently the horizontal component of the force required to keep *W* in the circular path will exactly balance the force required to overcome the inertia of *P*. Of course $W = P$. But a smaller weight *W'*, whose weight is inversely proportional to its radius of rotation, will evidently accomplish the same result. In the above numerical case, if the center of gravity of the counterweights is 20" from the center, the required weight to completely counterbalance the reciprocating parts would be $624 \times \frac{12}{20} = 374.4$ lbs. This counterweight need not be all placed on the driver carrying the main crank-pin, but can be (and is) distributed among all the drivers. Suppose it were divided between the two drivers in the above case. At 60 miles per hour such a counterweight would produce an additional pressure of 11211 lbs. when the counterweight was down, or a lifting force of the same amount when the counterweight was up. Although this is not sufficient to lift the driver from the rail, it would produce an objectionably high pressure on the rail (over 27000 lbs.), thus inducing just what it was desired to avoid on account of the eccentric rotating parts. Therefore a compromise must be made. Only a portion (one half to three fourths) of the weight of the reciprocating parts is balanced. Since the effect of the rotating

weights is to cause variable pressure on the rail, while the effect of the reciprocating parts is to cause a horizontal wobbling or "nosing" of the locomotive, it is impossible to balance both. Enough counterweight is introduced to partially neutralize the effect of the reciprocating parts, still leaving some tendency to horizontal wobbling, while the counterweights which were introduced to reduce the wobbling cause some variation of pressure. By using hollow piston-rods of steel, ribbed cross-heads, and connecting- and side-rods with an I section, the weight of the reciprocating parts may be greatly lessened without reducing their strength, and with a decrease in weight the effect of the unbalanced reciprocating parts and of the "excess balance" (that used to balance the reciprocating parts) is largely reduced.

Current practice is somewhat variable on three features:

(a) The proportion of the weight of the connecting-rod which should be considered as revolving weight.

(b) The proportion of the total reciprocating weight that should be balanced.

(c) The distribution among the drivers of the counterweight to balance the reciprocating parts.

An exact theoretical analysis of (a) shows that it is a function of the weights and dimensions of the reciprocating parts. The weight which may be considered as revolving equals *

$$W_1 \left(\frac{r^2 + k^2 - rd \left(1 + \frac{r}{l} \right)}{l^2 - r^2} \right) + W_2 \frac{r^2}{l^2 - r^2},$$

in which r =radius of the crank, l =length of connecting-rod, k =distance of center of gyration from wrist-pin, d =distance of center of gravity from wrist-pin, W_1 =weight of connecting-rod in pounds, and W_2 =weight of piston, piston-rod, and cross-head in pounds; all dimensions in feet. An application of this formula will show that for the dimensions of usual practice, from 51 to 57% of the weight of the connecting-rod should be considered as revolving weight.

The principal rules which have been formulated for counterbalancing may be stated as follows:

1. Each wheel should be balanced correctly for the revolving parts connected with it.

2. In addition, introduce counterbalance sufficient for 50% of the weight of the reciprocating parts for ordinary engines,

* R. A. Parke, in R. R. Gazette, Feb. 23, 1894.

increasing this to 75% when the reciprocating parts are excessively heavy (as in compound locomotives) or when the engine is light and unable to withstand much lateral strain or when the wheel-base is short.

3. Consider the weight of the connecting-rod as $\frac{1}{2}$ revolving and $\frac{1}{2}$ reciprocating when it is over 8 feet long; when shorter than 8 feet, consider $\frac{6}{10}$ of the weight as revolving and $\frac{4}{10}$ as reciprocating.

4. The part of the weight of the connecting-rod considered as revolving should be entirely balanced in the crank-driver wheel.

5. The "excess balance" should be divided equally among the drivers.

6. Place the counterbalance as near the rim of the wheel as possible and also as near the outside of the wheel as possible in order that the center of gravity shall be as near as possible opposite the center of gravity of the rods, etc., which are all outside of even the plane of the face of the wheel.

In Fig. 199 is shown a section of a locomotive driver with the cavities in the casting for the accommodation of the lead which is used for the counterbalance weight. Incidentally several other features and dimensions are shown in the illustration.

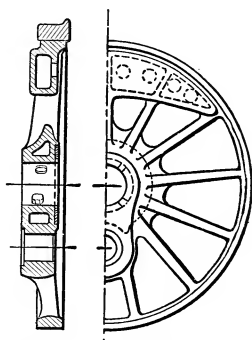


FIG. 199.—SECTION OF LOCOMOTIVE-DRIVER.

326. Mutual relations of the boiler power, tractive power, and cylinder power for various types. The design of a locomotive includes three *distinct* features which are varied in their mutual relations according to the work which the engine is expected to do.

(a) **The boiler power.** This is limited by the rate at which steam may be generated in a boiler of admissible size and weight. Engines which are designed to haul very fast trains which are comparatively light must be equipped with very large grates and heating surfaces so that steam may be developed with great rapidity in order to keep up with the very rapid consumption. Engines for very heavy freight work are run at very much lower velocity and at a lower piston speed in spite of the fact that more strokes are required to cover a given distance and the demand on the boiler for *rapid* steam production is not

as great as with high-speed passenger-engines. The capacity of a boiler to produce steam is therefore limited by the limiting weight of the general type of engine required. Although improvements may be and have been made in the design of fire-boxes so as to increase the steam-producing capacity without adding proportionately to the weight, yet there is a more or less definite limit to the boiler power of an engine of given weight.

(b) **The tractive power.** This is a function of the weight on the drivers. The absolute limit of tractive adhesion between a steel-tired wheel and a steel rail is about one third of the pressure, but not more than one fourth of the weight on the drivers can be depended on for adhesion and wet rails will often reduce this to one fifth and even less. The tractive power is therefore absolutely limited by the practicable weight of the engine. In some designs, when the maximum tractive power is desired, not only is the entire weight of the boiler and running gear thrown on the drivers, but even the tank and fuel-box are loaded on. Such designs are generally employed in switching-engines (or on engines designed for use on abnormally heavy mountain grades) in which the maximum tractive power is required, but in which there is no great tax on the boiler for *rapid* steam production (the speed being always very low), and the boiler and fire-box, which furnish the great bulk of the weight of an engine, are therefore comparatively light, and the requisite weight for traction must, therefore, be obtained by loading the drivers as much as possible. On the other hand, engines of the highest speed cannot possibly produce steam fast enough to maintain the required speed unless the load be cut down to a comparatively small amount. The tractive power required for this comparatively small load will be but a small part of the weight of the engine, and therefore engines of this class have but a small proportion of their weight on the drivers; generally have but two driving-axles and sometimes but one.

(c) **Cylinder power.** The running gear forms a mechanism which is simply a means of transforming the energy of the boiler into tractive force and its power is unlimited, within the practical conditions of the problem. The power of the running gear depends on the steam pressure, on the area of the piston, on the diameter of the drivers, and on the ratio of crank-pin radius to wheel radius, or of stroke to driver diameter. It

is always possible to increase one or more of these elements by a relatively small increase of expenditure until the cylinders are able to make the drivers slip, assuming a sufficiently great resistance. Since the power of the engine is limited by the power of its weakest feature, and since the running gear is the most easily controlled feature, the power of the running gear (or the "cylinder power") is always made somewhat excessive on all well-designed engines. It indicates a badly designed engine if it is stalled and unable to move its drivers, the steam pressure being normal. If it is attempted to use a freight-engine on *fast* passenger service, it will probably fail to attain the desired speed on account of the steam pressure falling. The tractive power and cylinder power are superabundant, but the boiler cannot make steam as fast as it is needed for high speed, especially when the drivers are small. The practical result would be a comparatively low speed kept up with a forced fire. If it is attempted to use a high-speed passenger-engine on heavy freight service, the logical result is a slipping of the drivers until the load is reduced. The boiler power and cylinder power are ample, but the weight on the drivers is so small that the tractive power is only sufficient to draw a comparatively small load.

These relations between boiler, cylinder, and tractive power are illustrated in the following comparative figures referring to a fast passenger-engine, a heavy freight-engine, and a switching-engine. The weights of the passenger- and freight-engines are about the same, but the passenger-engine has only 72% of

Kind.	Cylinders.	Total W ght.	Wt. on Driv'rs	Heat- ing Sur- face, sq. ft.	Grate area sq. ft.	Steam Pres- sure in Boiler.	Stroke. Diam. Driver.
Fast passenger.	19" × 24"	126700	81500	1831.8	26.2	180	$\frac{24}{78} = .31$
Heavy freight.	20" × 24"	128700	112600	1498.3	31.5	140	$\frac{24}{50} = .48$
Switcher.	19" × 24"	109000	109000	1498.0	22.8	160	$\frac{24}{50} = .48$

the tractive power of the freight. But the passenger-engine has 22% more heating-surface and can generate steam much faster; it makes less than two thirds as many strokes in covering a given distance, but it runs at perhaps twice the speed

and probably consumes steam much faster. The switch-engine is lighter in total weight, but the tractive power is nearly as great as the freight and much greater than the passenger-engine. While the heating-surfaces of the freight- and switching-engines are practically identical, the grate area of the switcher is much less; its speed is always low and there is but little necessity for rapid steam development.

While these figures show the general tendency for the relative proportions, and in this respect may be considered as typical, there are large variations. The recent enormous increase in the dead weight of passenger-trains has necessitated greater tractive power. This has been provided sometimes by using "Mogul" and "ten-wheel" engines, which were originally designed for freight work. On the other hand, the demand for fast freight service, and the possibility of safely operating such trains by the use of air-brakes, has required that heavy freight-engines shall be run at comparatively high speeds, and that requires the rapid production of steam, large grate areas, and heating surfaces. But in spite of these variations, the normal standard for passenger service is a four-driver engine carrying about two thirds of the weight of the engine on the drivers, which are very large; the normal standard for freight work is the "consolidation," with perhaps 90% of the weight on the drivers, which are small, but which must have the pony truck for such speed as it uses; and finally the normal standard for switching service has all the weight on the drivers and has comparatively low steam-producing capacity.

327. Life of locomotives. The life of locomotives (as a whole) may be taken as about 800000 miles or about 22 to 24 years. While its life should be and is considered as the period between its construction and its final consignment to the scrap pile, parts of the locomotive may have been renewed more than once. The boiler and fire-box are especially subject to renewal. The mileage life is much longer than formerly. This is due partly to better design and partly to the custom of drawing the fires less frequently and thereby avoiding some of the destructive strains caused by extreme alternations of heat and cold. Recent statistics give the average annual mileage on twenty-three leading roads to be 41000 miles.

CARS.

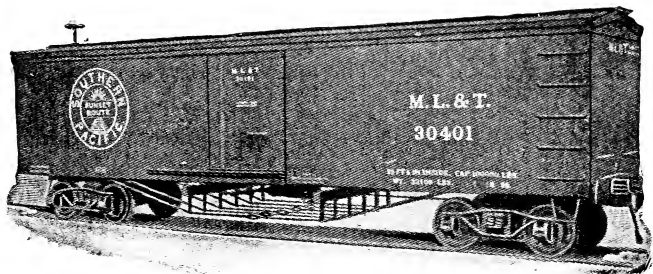
328. Capacity and size of cars. The capacity of freight-cars has been enormously increased of late years. About thirty years ago the usual live-load capacity for a box-car was about 20000 lbs. In 1893 the standard box-car, gondola-cars, etc., of the Pennsylvania Railroad on exhibition at the Chicago Exposition, had a live-load capacity of 60000 lbs. and a dead weight of 30000 to 33000 lbs. With a full load, the weight on each wheel is nearly 12000 lbs., which equals or exceeds the load usually placed on the drivers of ordinary locomotives. But now cars with a live-load capacity of 100,000 lbs. are standard on many roads, and even larger cars are made for special service. (See Fig. 200.)

The limitation of the carrying capacity for some kinds of freight depends somewhat on the amount of live load that can be carried within given dimensions; for the cross-section of a car is limited to the extreme dimensions which may be safely run through the tunnels and through bridges as at present constructed, and the length is somewhat limited by the difficulty of properly supporting an excessively heavy load, distributed over an unusually long span, by a structure which is subjected to excessive jar, concussion, compression, and tension. The cross-sectional limit seems to have been scarcely reached yet, except, perhaps, in the case of furniture and carriage-cars, whose load per cubic foot is not great. The usual width of freight-cars is about 8 to 9 feet, while parlor-cars and sleepers are generally 10 feet wide and sometimes 11 feet. The highest point of a train is usually the smoke-stack of the locomotive which is generally 14 feet above the rails and occasionally over 15 feet. A sleeping-car usually has the highest point of the car about 14 feet above the rails. Box-cars are usually about 8 feet high (above the sills), with a total height of about 11' 3". Refrigerator-cars are usually about 9' high and furniture-cars about 10' above the sills, the truck adding about 3' 3". The usual length is 34 feet, but 35 to 40 feet is not uncommon. Passenger-cars (day coaches) are usually 50 feet long, exclusive of the end platforms and weigh 45000 to 50000 lbs. Sixty passengers at 150 pounds apiece (a high average) will only add 9000 lbs. to the weight. A parlor-car or sleeper is generally about 65 feet long exclusive of the platforms, which add about 6' 6". The weight is anywhere from 60000 to 80000 lbs.

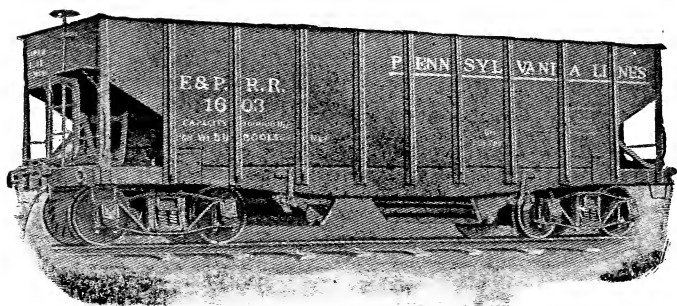
The weight of the 25 or 30 passengers it *may* carry is hardly worth considering in comparison.

329. Stresses to which car-frames are subjected. A car is structurally a truss, supported at points at some distance from the ends and subjected to transverse stress. There is, therefore, a change of flexure at two points between the trucks. Besides this stress the floor is subjected to compression when the cars are suddenly stopped and to tension when in ordinary motion, the tension being greater as the train resistance is greater and as the car is nearer the engine. The shocks, jars, and sudden strains to which the car-frames are subjected are very much harder on them than the mere static strains due to their maximum loads if the loads were quiescent. Consequently any calculations based on the static loads are practically valueless, except as a very rough guide, and previous experience must be relied on in designing car bodies. As evidence of the increasing demand for strength in car-frames, it has been recently observed that freight-cars, built some years ago and built almost entirely of wood, are requiring repairs of wooden parts which have been *crushed* in service, the wood being perfectly sound as regards decay.

330. The use of metal. The use of metal in car construction is very rapidly increasing. The demand for greater strength in car-frames has grown until the wooden framing has become so heavy that it is found possible to make steel frames and trucks at a small additional cost, the steel frames being twice as strong and yet reducing the dead weight of the car about 5000 lbs., a consideration of no small value, especially on roads having heavy grades. Another reason for the increasing use of metal is the great reduction in the price of rolled or pressed steel, while the cost of wood is possibly higher than before. The advocates of the use of steel advise steel floors, sides, etc. For box-cars a wooden floor has advantages. For ore and coal-cars an all-metal construction has advantages. (Fig. 200.) In Germany, where steel frames have been almost exclusively in use for many years, they have not yet been able to determine the normal age limit of such frames; none have yet *worn* out. The life is estimated at 50 to 80 years



100,000-LB. BOX CAR.



STEEL COAL CAR.



WOODEN BOX CAR, STEEL FRAME.

FIG. 200.—SOME HEAVY FREIGHT CARS.

(To face page 394.)



Brake-beams are also best made of metal rather than wood, as was formerly done. Metal brake-beams are generally used on cars having air-brakes, as a wooden beam must be excessively large and heavy in order to have sufficient rigidity.

Truck-frames (see Fig. 201), which were formerly made principally of wood, are now largely made of pressed steel. It makes

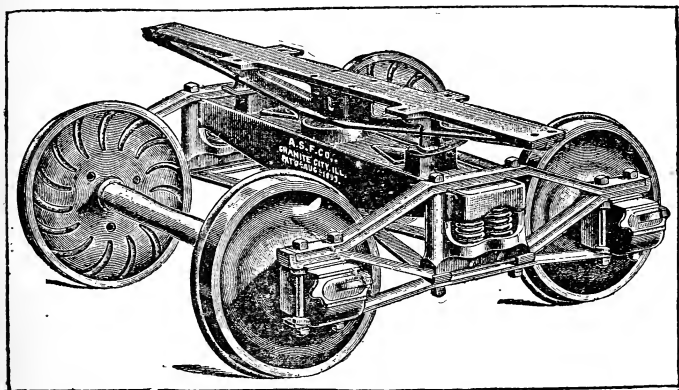


FIG 201.

a reduction in weight of about 3000 lbs. per car. The increased durability is still an uncertain quantity.

331. Draft gear. The enormous increase in the weight and live load capacities of rolling stock have necessitated a corresponding development in draft gear. Even within recent years, "coal-jimmies," carrying a few tons have been made up into trains by dropping a chain of three big links over hooks on the ends of the cars. But the great stresses due to present loadings would tear such hooks from the cars or tear the cars apart if such cars were used in the make-up of long heavy trains as now operated. The next stage in the development of draft gear was the invention of the "spring coupler," by which the energy due to a sudden tensile jerk or the impact of compression may be absorbed by heavy springs and gradually imparted to the car body. Such devices, for which there are many designs, seemed to answer the purpose for cars of 25 to 40 tons capacity. The use of 100,000-pound steel cars soon proved the inadequacy of even spring couplers. The friction-draft gear was then invented. The general principle of such a gear is that, when

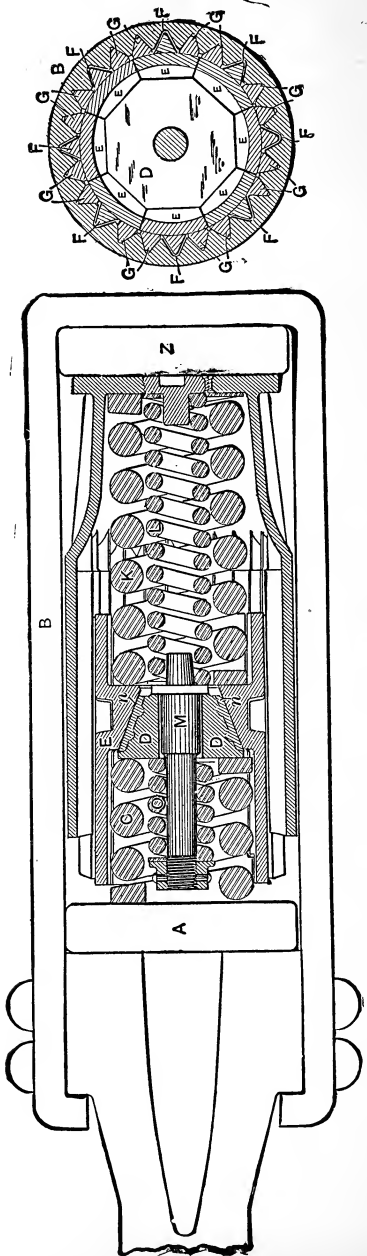


FIG. 202.

acting at or near its maximum capacity, it harmlessly transforms into heat the excessive energy developed by jerks or compression. There are several different designs of such gear, but the general principle underlying all of them may be illustrated by a description of the Westinghouse draft gear. The gear employs springs which have sufficient stiffness to act as ordinary spring-couplers for the ordinary pushing and pulling of train operations. Sections of the gear are shown in Fig. 202, while the method of its application to the framing of a car of the pressed steel type is shown in Fig. 203, *a* and *b*. When the draft gear is in tension the coupler, which is rigidly attached to *B*, is drawn to the left, drawing the follower *Z* with it. Compression is then exerted through the gear mechanism to the follower *A* which, being restrained by the shoulders *RR*, against which it presses, causes the gear to absorb the compression. The coil-spring *C* forces the eight wedges *n* against the eight corresponding segments *E*. The great compression of these surfaces against the outer shell produces a friction which retards the compression of the gear. The total possible movement of the gear, as determined by an official test, was 2.42 inches, when the maximum stress was 180,000 pounds. The work done in producing this stress amounted to 18,399 foot-pounds. Of this total energy 16,666 foot-pounds, or over 90%, represents the amount of energy absorbed and dissipated as heat by the frictional gear. The remaining 10% is given back by the recoil. The main release spring *K* is used for returning the segments and friction strips to their normal position after the force to close them has been removed. It also gives additional capacity to the entire mechanism. The auxiliary spring *L* releases the wedge *D*, while the release pin *M* releases the pressure of the auxiliary spring *L* against the wedge during frictional operation. If we omit from the above design the frictional features and consider only the two followers *A* and *Z*, separated by the springs *C* and *K*, acting as one spring, we have the essential elements of a spring-draft gear. In fact, this gear acts exactly like a spring-draft gear for all ordinary service, the frictional device only acting during severe tension and compression.

332. Gauge of wheels and form of wheel-tread.—In Fig. 204 is shown the standard adopted by the Master Car Builders' Association at their twentieth annual convention. Note the

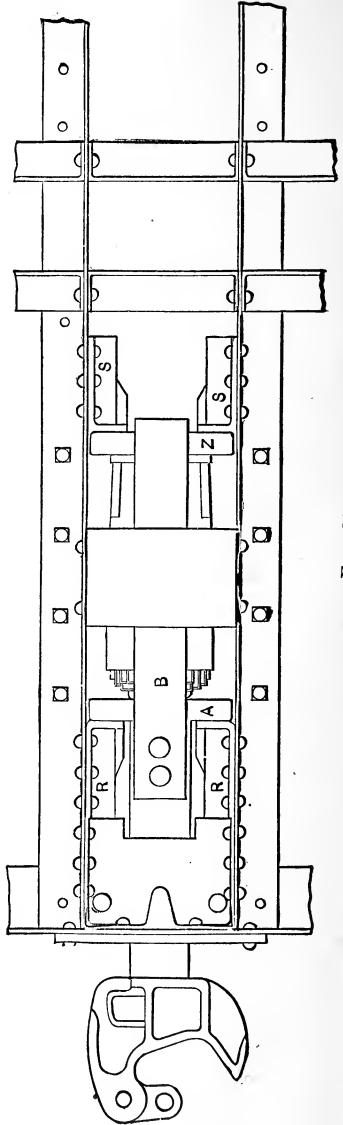
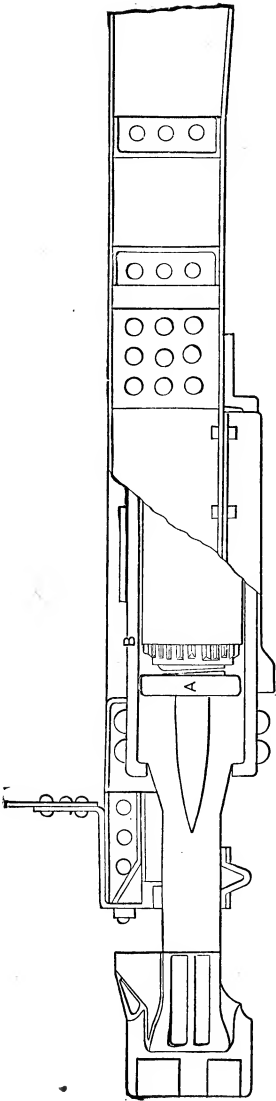


FIG. 203.

normal position of the gauge-line on the wheel-tread. In Fig. 114, p. 259, the relation of rail to wheel-tread is shown on a smaller scale. It should be noted that there is no definite position where the wheel-flange is absolutely "chock-a-block" against the rail. As the pressure increases the wheel mounts a little higher on the rail until a point is soon reached when the resistance is too great for it to mount still higher. By this means is avoided the shock of unyielding impact when the car sways from side to side. When the gauge between the inner faces of the wheels is greater or less than the limits given in the figure, the interchange rules of the Master Car Builders' Association authorize a road to refuse to accept a car from another road for transportation. At junction points of railroads inspectors are detailed to see that this rule (as well as many others) is complied with in respect to all cars offered for transfer.

TRAIN-BRAKES.

333. Introduction. Owing to the very general misapprehension that exists regarding the nature and intensity of the action of brakes, a complete analysis of the problem is considered justifiable. This misapprehension is illustrated by the common notion (and even practice) that the effectiveness of braking a car is proportional to the brake pressure, and therefore a brakeman is frequently seen using a bar to obtain a greater leverage on the brake-wheel and using his utmost strength to obtain the maximum pull on the brake-chain while the car is skidding along with locked wheels.

When a vehicle is moving on a track with a considerable velocity, the mass of the vehicle possesses kinetic energy of translation and the wheels possess kinetic energy of rotation. To stop the vehicle, this energy must be destroyed. The rotary kinetic energy will vary from about 4 to 8% of the kinetic energy of translation, according to the car loading (see § 347). On steam railroads brake action is obtained by pressing brake-shoes against car-wheel treads. As the brake-shoe pressure increases, the brake-shoes retard with increasing force the rotary action of the wheels. As long as the wheels do not slip or "skid" on the rails, the adhesion of the rails forces them to rotate with a circumferential velocity equal to the train velocity. The retarding action of the brake-shoe

checks first the rotative kinetic energy (which is small), and the remainder develops a *tendency* for the wheel to slip on the rail. Since the rotative kinetic energy is such a small percentage of the total, it will hereafter be ignored, except as specifically stated, and it will be assumed for simplicity that the only work of the brakes is to overcome the kinetic energy of translation. The possible effect of grade in assisting or preventing retardation, and the effect of all other track resistances, is also ignored. The amount of the developed force which retards the train movement is limited to the possible adhesion or *static* friction between the wheel and the rail. When the friction between the brake-shoe and the wheel exceeds the adhesion between the wheel and the rail, the wheel skids, and then the friction between the wheel and the rail at once drops to a much less quantity. It must therefore be remembered at the outset that the retarding action of brake-shoes on wheels as a means of stopping a train is absolutely limited by the possible static friction between the braked wheels and the rails.

334. Laws of friction as applied to this problem. Much of the misapprehension regarding this problem arises from a very common and widespread misstatement of the general laws of friction. It is frequently stated that friction is independent of the velocity and of the unit of pressure. The first of these so-called laws is not even approximately true. A very exhaustive series of tests were made by Capt. Douglas Galton on the Brighton Railway in England in 1878 and 1879, and by M. George Marié on the Paris and Lyons Railway in 1879, with trains which were specially fitted with train-brakes and with dynagraphs of various kinds to measure the action of the brakes. Experience proved that variations in the condition of the rails (wet or dry), and numerous irregularities incident to measuring the forces acting on a heavy body moving with a high velocity, were such as to give somewhat discordant results, even when the conditions were made as nearly identical as possible. But the tests were carried so far and so persistently that the general laws stated below were demonstrated beyond question, and even the numerical constants were determined as closely as they may be practically utilized. These laws may be briefly stated as follows:

(a) The coefficient of friction between cast-iron brake-blocks

and steel tires is about .3 when the wheels are "just moving"; it drops to about .16 when the velocity is about 30 miles per hour, and is less than .10 when the velocity is 60 miles per hour. These figures fluctuate considerably with the condition of the rails, wet or dry.

(b) The coefficient of friction is greatest when the brakes are first applied; it then reduces very rapidly, decreasing nearly one third after the brakes have been applied 10 seconds, and dropping to nearly one half in the course of 20 seconds. Although the general truth of this law was established beyond question, the tests to demonstrate the law of the variation of friction with time of application were too few to determine accurately the numerical constants.

(c) The friction of skidded wheels on rails is always very much less than the adhesion when the wheel is rolling on the rail—sometimes less than one third as much.

(d) An analysis of the tests all pointed to a law that the friction developed does *not* increase as rapidly as the *intensity* of pressure increases, but this may hardly be considered as an established law.

(e) The adhesion between the wheel and the rail appears to be independent of velocity. The adhesion here means the force that must be developed before the wheel will slip on the rail.

The practical effect of these laws is shown by the following observed phenomena:

(a) When the brakes are first applied (the velocity being very high), a brake pressure far in excess of the weight on the wheel (even three or four times as much) may be applied without skidding the wheel. This is partly due to the fact that the wheel has a very high rotative kinetic energy (which varies as the square of the velocity, and which must be overcome first), but it is chiefly due to the fact that the coefficient of friction at the higher velocity is very small (at 60 miles per hour it is about .07), while the adhesion between the wheel and the rail is independent of the velocity.

(b) As the velocity decreases the brake pressure must be decreased or the wheels will skid. Although the friction decreases with the time required to stop and increases with the reduction of speed, and these two effects tend to neutralize each other, yet unless the stop is very slow, the increase in friction due to reduction of speed is much greater than the

decrease due to time, and therefore the brake pressure must not be greater than the weight on the wheel, unless momentarily while the speed is still very high.

(c) The adhesion between wheels and rails varies from .20 to .25 and over when the rail is dry. When wet and slippery it may fall to .18 or even .15. The use of sand will always raise it above .20, and on a dry rail, when the sand is not blown away by wind, it may raise it to .35 or even .40.

(d) Experiments were made with an automatic valve by which the brake-shoe pressure against the wheel should be reduced as the friction increased, but since (1) the essential requirement is that the friction produced by the brake-shoes shall not exceed the adhesion between rail and wheel, and since (2) the rail-wheel adhesion is a very variable quantity, depending on whether the rail is wet or dry, it has been found impracticable to use such a valve, and that the best plan is to leave it to the engineer to vary the pressure, if necessary, by the use of the brake-valve.

MECHANISM OF BRAKES.

335. Hand-brakes. The old style of brakes consists of brake-shoes of some type which are pressed against the wheel-treads by means of a brake-beam, which is operated by means of a hand-windlass and chain operating a set of levers. It is desirable that brakes shall not be set so tightly that the wheels shall be locked, and then slide over the track, producing flat places on them, which are very destructive to the rolling-stock and track afterward, on account of the impact occasioned at each revolution. With air-brakes the maximum pressure of the brake-shoes can be quite carefully regulated, and they are so designed that the maximum pressure exerted by any pair of brake-shoes on the wheels of any axle shall not exceed a certain per cent. of the weight carried by that axle when the car is *empty*, 90% being the figure usually adopted for passenger-cars and 70% for freight-cars. Consider the case of a freight-car of 100000 lbs. capacity, weighing 33100 lbs., or 8275 lbs. on an axle, and equipped with a hand-brake which operates the levers and brake-beams, which are sketched in Fig. 205. The dead weight on an axle is 8275 lbs.; 70% of

this is 5792 lbs., which is the maximum allowable pressure per brake-beam, or 2896 lbs. per brake-shoe. With the dimensions shown, such a pressure will be produced by a pull of about 1158 lbs. on the brake-chain. The power gained by the brake-wheel is not equal to the ratio of the brake-wheel diameter to the diameter of the shaft, about which the brake-chain winds, which is about 16 to 1½. The ratio of the circumference of the brake-wheel to the length of chain wound up by one complete turn would be a closer figure. The loss of effi-

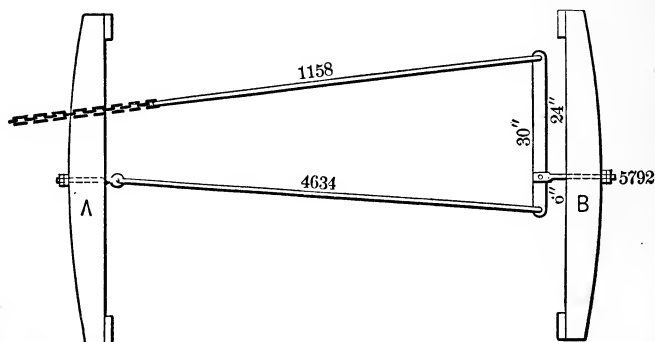


FIG. 205.—SKETCH OF MECHANISM OF HAND-BRAKE.

ciency in such a clumsy mechanism also reduces the effective ratio. Assuming the *effective* ratio as 6:1 it would require a pull of 193 lbs. at the circumference of the brake-wheel to exert 1158 lbs. pull on the brake-chain, or 5792 lbs. pressure on the wheels at B, and even this will not lock the wheels when the car is empty, much less when it is loaded. Note that the pressures at A and B are unequal. This is somewhat objectionable, but it is unavoidable with this simple form of brake-beam. More complicated forms to avoid this are sometimes used. Hand-brakes are, of course, cheapest in first cost, and even with the best of automatic brakes, additional mechanism to operate the brakes by hand in an emergency is always provided, but their slow operation when a quick stop is desired makes it exceedingly dangerous to attempt to run a train at high speed unless some automatic brake directly under the control of the engineer is at hand. The great increase in the

average velocity of trains during recent years has only been rendered possible by the invention of automatic brakes.

336. "Straight" air-brakes. The essential constructive features of this form of brake are (1) an air-pump on the engine, operated by steam, which compresses air into a reservoir on the engine; (2) a "brake-pipe" running from the reservoir to the rear of the engine and pipes running under each car, the pipes having flexible connections at the ends of the cars and engine; (3) a cylinder and piston under each car which operates the brakes by a system of levers, the cylinder being connected to the brake-pipe. The reservoir on the engine holds compressed air at about 45 lbs. pressure. To operate the brakes, a valve on the engine is opened which allows the compressed air to flow from the reservoir through the brake-pipe to each cylinder, moving the piston, which thereby moves the levers and applies the brakes. The *defects* of this system are many: (1) With a long train, considerable time is required for the air to flow from the reservoir on the engine to the rear cars, and for an emergency-stop even this delay would often be fatal; (2) if the train breaks in two, the rear portion is not provided with power for operating the brakes, and a dangerous collision would often be the result; (3) if an air-pipe coupling bursts under any car, the whole system becomes absolutely helpless, and as such a thing might happen during some emergency, the accident would then be especially fatal.

This form of brake has almost, if not entirely, passed out of use. It is here briefly described in order to show the logical development of the form which is now in almost universal use, the automatic.

337. Automatic air-brakes. The above defects have been overcome by a method which may be briefly stated as follows: A reservoir for compressed air is placed under each car and the tender; whenever the pressure in these reservoirs is reduced for any reason, it is automatically replenished from the main reservoir on the engine; whenever the pressure in the brake-pipe is reduced for **any** cause (opening a valve at any point of its length, parting of the train, or bursting of a pipe or coupler), valves are automatically moved under each car to operate the piston and put on the brakes. *All* the brakes on the train are thus applied almost simultaneously. If the train breaks in two, *both* sections will at once have *all* the brakes applied automati-

cally; if a coupling or pipe bursts, the brakes are at once set, and attention is thereby attracted to the defect; if an emergency should arise, such that the conductor desires to stop the train instantly without even taking time to signal the engineer, he can do so by opening a valve placed on each car which admits air to the train-pipe, which will set the brakes on the whole train, and the engineer, being able to determine instantly what had occurred, would shut off steam and whatever else was necessary to stop the train as quickly as possible. The most important and essential detail of this is the "automatic triple valve" placed under each car. According to the Westinghouse Air-brake Company's Instruction Book, "A moderate reduction of air pressure in the train-pipe causes the greater pressure remaining stored in the auxiliary reservoir to force the piston of the triple valve and its slide-valve to a position which will allow the air in the auxiliary reservoir to pass directly into the brake-cylinder and apply the brake. A sudden or violent reduction of the air in the train-pipe produces the same effect, and in addition causes supplementary valves in the triple valve to be opened, permitting the pressure from the train-pipe to also enter the brake-cylinder, thus augmenting the pressure derived from the auxiliary reservoir about 20%, producing practically instantaneous action of the brakes to their highest efficiency throughout the entire train. When the pressure in the brake-pipe is again restored to the normal amount in excess of that remaining in the auxiliary reservoir, the piston- and slide-valves are forced in the opposite direction to their normal position, opening communication from the train-pipe to the auxiliary reservoir, and permitting the air in the brake-cylinder to escape to the atmosphere, thus releasing the brakes. If the engineer wishes to apply the brake, he moves the handle of the engineer's brake-valve to the right, which first closes a port, retaining the pressure in the main reservoir, and then permits a portion of the air in the train-pipe to escape. To release the brakes, he moves the handle to the extreme left, which allows the air in the main reservoir to flow freely into the brake-pipe, restoring the pressure therein."

338. Tests to measure the efficiency of brakes. Let v represent the velocity of a train in feet per second; W , its weight; F , the retarding force due to the brakes; d , the distance in feet required to make a stop; and g , the acceleration of gravity.

feet per square second); then the kinetic energy possessed by the train (disregarding for the present the rotative energy of the wheels) $= \frac{Wv^2}{2g}$. The work done in stopping the train $= Fd$. $\therefore Fd = \frac{Wv^2}{2g}$. The ratio of the retarding force to the weight,

$$\frac{F}{W} = \frac{v^2}{2gd} = .0155 \frac{v^2}{d}.$$

In order to compare tests made under varying conditions, the $F \div W$ should be corrected for the effect of grade (+ or -), γ , and also for the proportion of the weight of the train which is on *braked* wheels. For example, a train weighed 6 lbs., the proportion on braked wheels was 67%, speed 10 feet per second, length of stop 450 feet, track level. Substituting these values in the above formula, we find $(F \div W) = .044$. This value is really unduly favorable, since the ordinary track resistance helps to stop the train. This has a value from 6 to 20 lbs. per ton, averaging say 10 lbs. per ton during the stop, or .005 of the weight. Since the effect of this is small and is nearly constant for all trains, it may be ignored in comparative tests. The grade in this case was level, and therefore grade had no effect. But since only 67% of the weight was on braked wheels, the ratio, on the basis of *all* the wheels braked, or of the weight reduced to that actually on the braked wheels, is $0.124 \div .67 = 0.185$. This was called a "good" although as high a ratio as 0.200 has been obtained.

b. Brake-shoes. Brake-shoes were formerly made of wrought iron, but when it was discovered that cast-iron shoes would answer the purpose, the use of wrought-iron shoes was abandoned, since the cast-iron shoes are so much cheaper. A common practice is to form the brake-shoe and its head in one casting, which is cheaper in first cost, but when the wearing-surfaces are too far gone for further use, the whole casting must be replaced. The "Christie" shoe, adopted by the Master Carriers' Association as standard, has a separate shoe which is bolted to the head by means of a wrought-iron key. The shoe is beveled $\frac{1}{4}$ " in a width of $3\frac{3}{8}$ " to fit the coned wheel. It has a greater bevel than the standard coning of a car-wheel, which is perhaps done to allow for some bending of the brake-

beam and also so that the maximum pressure (and wear) should come on the outside of the tread, rather than next to the flange, where it might tend to produce sharp flanges. By concentrating the brake-shoe wear on the outer side of the tread, the wear on the tread is more nearly equalized, since the rail wears the wheel-tread chiefly near the flange. This same idea is developed still further in the "flange-shoes," which have a curved form to fit the wheel-flange and which bear on the wheel on the flange and on the outside of the tread. It is claimed that by this means the standard form of the tread is better preserved than when the wear is entirely on the tread. The Congdon brake-shoe is one of a type in which wrought-iron pieces are inserted in the face of a cast-iron shoe. It is claimed that these increase the life of the shoe.

CHAPTER XVI.

TRAIN RESISTANCE.

340. Classification of the various forms. The various resistances which must be overcome by the power of the locomotive may be classified as follows:

(a) *Resistances internal to the locomotive*, which include friction of the valve-gear, piston- and connecting-rods, journal friction of the drivers; also all the loss due to radiation, condensation, friction of the steam in the passages, etc. In short, these resistances are the sum-total of the losses by which the power at the circumference of the drivers is less than the power developed by the boiler.

(b) *Velocity resistances*, which include the atmospheric resistances on the ends and sides; oscillation and concussion resistances, due to uneven track, etc.

(c) *Wheel resistances*, which include the rolling friction between the wheels and the rails of *all* the wheels (including the drivers); also the journal friction of all the axles, except those of the drivers.

(d) *Grade and curve resistances*, which include those resistances which are due to grade and to curves, and which are not found on a straight and level track.

(e) *Brake resistances*. As shown later, brakes consume power and to the extent of their use increase the energy to be developed by the locomotive.

(f) *Inertia resistances*. The resistance due to inertia is not generally considered as a train resistance because the energy which is stored up in the train as kinetic energy may be utilized in overcoming future resistances. But in a discussion of the demands on the tractive power of the engine, one of the chief items is the energy required to *rapidly* give to a starting train its normal velocity. This is especially true of suburban trains, which must acquire speed very quickly in order that

their general average speed between termini may be even reasonably fast.

341. Resistances internal to the locomotive. These are resistances which do not tax the adhesion of the drivers to the rails, and hence are frequently considered as not being a part of the train resistance properly so called. If the engine were considered as lifted from the rails and made to drive a belt placed around the drivers, then all the power that reached the belt would be the power that is ordinarily available for adhesion, while the remainder would be that consumed internally by the engine. The power developed by an engine may be obtained by taking indicator diagrams which show the actual steam pressure in a cylinder at any part of a stroke. From such a diagram the average steam pressure is easily obtained, and this average pressure, multiplied by the length of the stroke and by the net area of the piston, gives the energy developed by one half-stroke of one piston. Four times this product divided by 550 times the time in seconds required for one stroke gives the "indicated horse-power." Even this calculation gives merely the power behind the piston, which is several per cent. greater than the power which reaches the circumference of the drivers, owing to the friction of the piston, piston-rod, cross-head, connecting-rod bearings, and driving-wheel journals. (See § 322, Chapter XV.) By measuring the amount of water used and turned into steam, and by noting the boiler pressure, the energy possessed by the steam used is readily computed. The indicator diagrams will show the amount of steam that has been effective in producing power at the cylinders. The steam accounted for by the diagrams will ordinarily amount to 80 or 85% of the steam developed by the boiler, and the other 15 or 20% represents the loss of energy due to radiation, condensation, etc. From actual tests it has been found that the power consumed by an engine running light is about 11% of that required by the engine when working hard in express freight service. But since the engine resistances (friction, etc.) are increased when it is pulling a load, it was estimated, after allowing for this fact, that about 15 or 16% of the power developed by the pistons was consumed by the engine, leaving about 84 to 85% for the train.

342. Velocity resistances. (a) *Atmospheric.* This consists of the head and tail resistances and the side resistance. The head

and tail resistances are nearly constant for all trains of given velocity, varying but slightly with the varying cross-sections of engines and cars. The side resistance varies with the length of the train and the character of the cars, box-cars or flats, etc. Vestibuling cars has a considerable effect in reducing this side resistance by preventing much of the eddying of air-currents between the cars, although this is one of the least of the advantages of vestibuling. Atmospheric resistance is generally assumed to vary as the square of the velocity, and although this may be nearly true, it has been experimentally demonstrated to be at least inaccurate. The head resistance is generally assumed to vary as the area of the cross-section, but this has been definitely demonstrated to be very far from true. A freight-train composed partly of flat-cars and partly of box-cars will encounter considerably more atmospheric resistance than one made exclusively of either kind, other things being equal. The definite information on this subject is very unsatisfactory, but this is possibly due to the fact that it is of little practical importance to know just how much such resistance amounts to.

(b) *Oscillatory and concussive.* These resistances are considered to vary as the square of the velocity. Probably this is nearly, if not quite, correct on the general principle that such resistances are a succession of impacts and the force of impacts varies as the square of the velocity. These impacts are due to the defects of the track, and even though it were possible to make a precise determination of the amount of this resistance in any particular case, the value obtained would only be true for that particular piece of track and for the particular degree of excellence or defect which the track *then* possessed. The general improvement of track maintenance during late years has had a large influence in increasing the possible train-load by decreasing the train resistance. The expenditure of money to improve track will give a road a large advantage over a competing road with a poorer track, by reducing train resistance, and thus reducing the cost of handling traffic.

343. Wheel resistances. (a) *Rolling friction of the wheels.* To determine experimentally the rolling friction of wheels, apart from all journal friction, is a very difficult matter and has never been satisfactorily accomplished. Theory as well as practice shows that the higher and the more perfect the

elasticity of the wheel and the surface, the less will be the rolling friction. But the determination, if made, would be of theoretical interest only.

The combined effect of rolling friction and journal friction is determinable with comparative ease. From the nature of the case no great reduction of the rolling friction by any device is possible. It is only a very insignificant part of the total train resistance.

(b) *Journal friction of the axles.* This form of resistance has been studied quite extensively by means of the measurement of the force required to turn an axle in its bearings under various conditions of pressure, speed, extent of lubrication, and temperature. The following laws have been fairly well established: (1) The coefficient of friction increases as the pressure diminishes; (2) it is higher at very slow speeds, gradually diminishing to a minimum at a speed corresponding to a train velocity of about 10 miles per hour, then slowly increasing with the speed; it is very dependent on the perfection of the lubrication, it being reduced to one sixth or one tenth, when the axle is lubricated by a bath of oil rather than by a mere pad or wad of waste on one side of the journal; (3) it is much lower at higher temperature, and *vice versa*. The practical effect of these laws is shown by the observed facts that (1) loaded cars have a less resistance per ton than unloaded cars, the figures being (for speeds of about 10 to 20 miles per hour):

For passenger- and loaded freight-cars...	4 lbs. per ton
“ empty freight-cars.	6 “ “ “
“ street-cars.	10 “ “ “
“ freight-trucks without load.	14 “ “ “

(2) When starting a train, the resistances are about 20 lbs. per ton, notwithstanding the fact that the velocity resistances are practically zero; at about 2 miles per hour it will drop to 10 lbs. per ton and above 10 miles per hour it may drop to 4 lbs. per ton if the cars are in good condition. (3) The resistance could probably be materially lowered if some practicable form of journal-box could be devised which would give a more perfect lubrication. (4) It is observed that freight-train loads must be cut down in winter by about 10 or 15% of the loads that the same engine can haul over the same track in summer. This is due partly to the extra roughness and inelasticity of the

track in winter, and partly to increased radiation from the engine wasting some energy, but this will not account for all of the loss, and the effect, which is probably due largely to the lower temperature of the journal-boxes, is very marked and costly. It has been suggested that a jacketing of the journal-boxes, which would prevent rapid radiation of heat and enable them to retain some of the heat developed by friction, would result in a saving amply repaying the cost of the device.

Roller journals for cars have been frequently suggested, and experiments have been made with them. It is found that they are very effective at low velocities, greatly reducing the starting resistance, which is very high with the ordinary forms of journals. But the advantages disappear as the velocity increases. The advantages also decrease as the load is increased, so that with heavily loaded cars the gain is small. The excess of cost for construction and maintenance has been found to be more than the gain from power saved.

344. Grade resistance. The amount of this may be computed with mathematical exactness. Assume that the ball or cylinder (see Fig. 206) is being drawn up the plane. If W

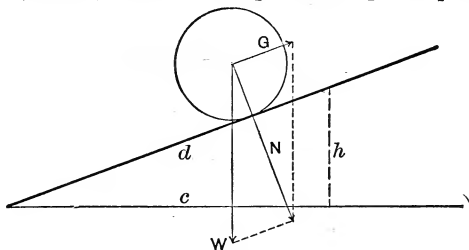


FIG. 206.

is the weight, N the normal pressure against the rail, and G the force required to hold it or to draw it up the plane with uniform velocity, the rolling resistances being considered zero or considered as provided for by other forces, then

$$G:W::h:d, \text{ or } G = \frac{Wh}{d};$$

but for all ordinary railroad grades, $d=c$ to within a tenth of 1%, i.e., $G = \frac{Wh}{c} = W \times \text{rate of grade}$. In order that the student may appreciate the exact amount of this approximation the percentage of slope distance to its horizontal projection is given in the following tabular form:

Grade in per cent.	1	2	3	4	5
$\frac{\text{Slope dist.}}{\text{hor. dist.}} \times 100 \dots \dots$	100.005	100.020	100.045	100.080	100.125

Grade in per cent.	6	7	8	9	10
$\frac{\text{Slope dist.}}{\text{hor. dist.}} \times 100 \dots \dots$	100.180	100.245	100.319	100.404	100.499

This shows also the error on various grades of measuring with the tape on the ground rather than held horizontally. Since almost all railroad grades are less than 2% (where the error is but .02 of 1%), and anything in excess of 4% is unheard of for normal construction, the error in the approximation is generally too small for practical consideration.

If the rate of grade is 1 : 100, $G = W \times \frac{1}{100}$, i.e., $G = 20$ lbs. per ton; $\therefore$ for any per cent. of grade, $G = (20 \times \text{per cent. of grade})$ pounds per ton. When moving up a grade this force G is to be overcome in addition to all the other resistances. When moving down a grade, the force G assists the motion and may be more than sufficient to move the train at its highest allowable velocity. The force required to move a train on a level track at ordinary freight-train speeds (say 20 miles per hour) is about 7 lbs. per ton. A down grade of $\frac{7}{20}$ of 1% will furnish the same power; therefore on a down grade of 0.35%, a freight-train would move indefinitely at about 20 miles per hour. If the grade were higher and the train were allowed to gain speed freely, the speed would increase until the resistance at that speed would equal W times the rate of grade, when the velocity would become uniform and remain so as long as the conditions were constant. If this speed was higher than a safe permissible speed, brakes must be applied and power wasted. The fact that one terminal of a road is considerably higher than the other does not necessarily imply that the extra power needed to overcome the difference of elevation is a total waste of energy, especially if the maximum grades are so low that brakes will never need to be applied to reduce a dangerously high velocity. for although more power must be

used in ascending the grades, there is a considerable saving of power in descending the grades. The amount of this saving will be discussed more fully in Chapter XXIII.

345. Curve resistance. Some of the principal laws will be here given without elaboration. A more detailed discussion will be given in Chapter XXII.

(a) While the total curve resistance increases as the degree of curve increases, the resistance *per degree of curve* is much greater for easy curves than for sharp curves; *e.g.*, the resistance on the excessively sharp curves (radius 90 feet) of the elevated roads of New York City is very much less *per degree of curve* than that on curves of 1° to 5° . (b) Curve resistance increases with the velocity. (c) The total resistance on a curve depends on the central angle rather than on the radius; *i.e.*, two curves of the same central angle but of different radius would cause about the same total curve resistance. This is partly explained by the fact that the longitudinal slipping will be the same in each case. (See § 311, Chapter XV.) In each case also the trucks must be twisted around and the wheels slipped laterally on the rails by the same amount 1° . (See § 312, Chapter XV.)

346. Brake resistances. If a down grade is excessively steep so that brakes must be applied to prevent the train acquiring a dangerous velocity, the energy consumed is hopelessly lost without any compensation. When trains are required to make frequent stops and yet maintain a high average speed, considerable power is consumed by the application of brakes in stopping. All the energy which is thus turned into heat is hopelessly lost, and in addition a very considerable amount of steam is drawn from the boiler to operate the air-brakes, which consume the power already developed. It can be easily demonstrated that engines drawing trains in suburban service, making frequent stops, and yet developing high speed between stops, will consume a very large proportion of the total power developed by the use of brakes. Note the double loss. The brakes consume power already developed and stored in the train as kinetic or potential energy, while the operation of the brakes requires additional steam power from the engine.

347. Inertia resistance. The two forms of train resistance which under some circumstances are the greatest resistances to be overcome by the engine are the grade and inertia resist-

ances, and fortunately both of these resistances may be computed with mathematical precision. The problem may be stated as follows: What constant force P (in addition to the forces required to overcome the various frictional resistances, etc.) will be required to impart to a body a velocity of v feet per second in a distance of s feet? The required number of foot-pounds of energy is evidently Ps . But this work imparts a kinetic energy which may be expressed by $\frac{Wv^2}{2g}$. Equating

these values, we have $Ps = \frac{Wv^2}{2g}$, or

$$P = \frac{Wv^2}{2gs} \quad . \quad . \quad . \quad . \quad . \quad . \quad (138)$$

The force required to increase the velocity from v_1 to v_2 may likewise be stated as $P = \frac{W}{2gs}(v_2^2 - v_1^2)$. Substituting in the formula the values $W = 2000$ lbs. (one ton), $g = 32.16$, and $s = 5280$ feet (one mile), we have

$$P = .00588(v_2^2 - v_1^2).$$

Multiplying by $(5280 \div 3600)^2$ to change the unit of velocity to miles per hour, we have

$$P = .01267(V_2^2 - V_1^2).$$

But this formula must be modified on account of the rotative kinetic energy which must be imparted to the wheels of the cars. The precise additional percentage depends on the particular design of the cars and their loading and also on the design of the locomotive. Consider as an example a box-car, 60000 lbs. capacity, weighing 33000 lbs. The wheels have a diameter of 36" and their radius of gyration is about 13". Each wheel weighs 700 lbs. The rotative kinetic energy of each wheel is 4877 ft.-lbs. when the velocity is 20 miles per hour, and for the eight wheels it is 39016 ft.-lbs. For greater precision (really needless) we may add 192 ft.-lbs. as the rotative kinetic energy of the axles. When the car is fully loaded (weight 93000 lbs.) the kinetic energy of translation is 1,244,340 ft.-lbs.; when empty (weight 33000 lbs.) the energy is 441540 ft.-lbs. The rotative kinetic energy thus adds (for this particular car) 3.15% (when the car is loaded) and 8.9% (when the car is empty) to the kinetic energy of translation. The kinetic

energy which is similarly added, owing to the rotation of the wheels and axles of the locomotive, might be similarly computed. For one type of locomotive it has been figured at about 8%. The variations in design, and particularly the fluctuations of loading, render useless any great precision in these computations. For a train of "empties" the figure would be high, probably 8 to 9%; for a fully loaded train it will not much exceed 3%. Wellington considered that 6% is a good average value to use (actually used 6.14% for "ease of computation"), but considering (a) the increasing proportion of live load to dead load in modern car design, (b) the greater care now used to make up *full* train-loads, and (c) the fact that *full* train-loads are the critical loads, it would appear that 5% is a better average for the conditions of modern practice. Even this figure allows something for the higher percentage for the locomotive and something for a few empties in the train. Therefore, adding 5% to the coefficient in the above equation, we have the true equation

$$P = .0133(V_2^2 - V_1^2), \quad . \quad . \quad . \quad . \quad (139)$$

in which V_2 and V_1 are the higher and lower velocities respectively in *miles per hour*, and P is the force required *per ton* to impart that difference of velocity in a distance of *one mile*. If more convenient, the formula may be used thus:

$$P_1 = \frac{70.224}{s}(V_2^2 - V_1^2), \quad . \quad . \quad . \quad . \quad (140)$$

in which s is the distance in feet and P_1 is the corresponding force.

As a numerical illustration, the force required per ton to impart a kinetic energy due to a velocity of 20 miles per hour in a distance of 1000 feet will equal

$$P_1 = \frac{70.224(400 - 0)}{1000} = 28 \text{ lbs.},$$

which is the equivalent (see § 344) of a 1.4% grade. Since the velocity enters the formula as V^2 , while the distance enters only in the first power, it follows that it will require *four* times the force to produce twice the velocity in the same distance, or that with the *same* force it will require four times the distance to attain twice the velocity.

As another numerical illustration, if a train is to increase its speed from 15 miles per hour to 60 miles per hour in a distance of 2000 feet, the force required (in addition to all the other resistances) will be

$$P_1 = \frac{70.224(3600 - 225)}{2000} = 118.50 \text{ lbs. per ton.}$$

This is equivalent to a 5.9% grade and shows at once that it would be impossible unless there were a very heavy down grade, or that the train was very light and the engine very powerful.

348. Dynamometer tests. These are made by putting a "dynamometer-car" between the engine and the cars to be tested. Suitable mechanism makes an automatic record of the force which is transmitted through the dynamometer at any instant, and also a record of the velocity at any instant. One of the practical difficulties is the accurate determination of the velocity at any instant when the velocity is fluctuating. When the velocity is decreasing, the kinetic energy of the train is being turned into work and the force transmitted through the dynamometer is less than the amount of the resistance which is actually being overcome. On the other hand, when the velocity is increasing, the dynamometer indicates a larger force than that required to overcome the resistances, but the excess force is being stored up in the train as kinetic energy. Grade has a similar effect, and the force indicated by the dynamometer may be greater or less than that required at the given velocity on a level by the force which is derived from, or is turned into, potential energy. Therefore the resistance indicated by the dynamometer of a train will not be that on a level track at uniform velocity, unless the track is actually level and the velocity really uniform.

Dynamometer tests under other circumstances are therefore of no value unless it is possible to determine the true velocity at any instant and its rate of change, and also to determine the grade. Of course, the grade is easily found. An

allowance for an increase or decrease of kinetic or potential energy must therefore be made before it is possible to know how much force is being spent on the ordinary resistances.

349. Gravity or "drop" tests. Dynamometer tests require the use of a dynamometer which is capable of measuring a force of several thousands of pounds, and which therefore cannot determine such values with a close percentage of accuracy, especially if the force is small. A drop test utilizes the force of gravity which may be measured with mathematical accuracy. The general method is to select a stretch of track which has a uniform grade of about 0.7% and which is preferably straight for two or three miles. On such a grade cars with running gear in good condition may be started by a push. The velocity will gradually increase until at some velocity, depending on the resistances encountered, the cars will move uniformly. The only work requiring extreme care with this method is the determination of the velocity. If the velocity is fluctuating, as it is during the time when it is of the greatest importance to know the velocity, it is not sufficient to determine the time required to run some long measured distance, for the average velocity thus obtained would probably differ

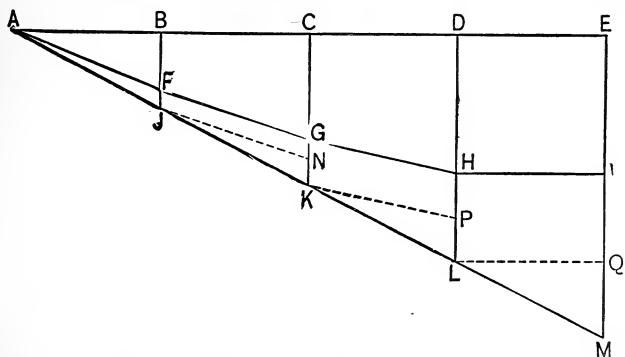


FIG. 207.—LOSS IN VELOCITY-HEAD.

considerably from the velocity at the beginning and end of that space. If the train consists of five cars or more, the velocity may be determined electrically (as described by Wellington in his "Economic Location," etc., p. 793 *et seq.*) from the automatic record made on a chronograph of the passage of the first wheel and the last, the chronograph also recording auto-

matically the ticks of a clock beating seconds. From this the exact time of the passage of the first and last wheels of the train of cars may be determined to the tenth or twentieth of a second.

Velocity-head. From theoretical mechanics we know that if a body descends through any path by the action of gravity, and is unaffected by friction, its velocity at any point in the direction of the path of motion is $V = \sqrt{2gh}$. If the body is retarded by resistances, its velocity at any point will be less than this. If AM , Fig. 207, represents any grade (exaggerated of course), then BJ , CK , etc., represent the actual fall at any point. Let BF represent the fall h_1 , determined from $h_1 = \frac{v_1^2}{2g}$, in which v_1 is the actual observed velocity at J . Then JF = the velocity-head consumed by the resistances between A and J . If the train continues to K , the corresponding h_2 is CG ; the remaining fall GK consists of GN (= JF , which is the velocity-head lost back of J) and NK , the velocity-head lost between J and K . At some velocity (V_n) on any grade, the velocity will not further increase and the line $AFGHI$ will then be horizontal and at a distance (h_n) = EI below $A \dots E$. The grade AM is the "grade of repose" for that velocity (V_n); i.e., it is the grade that would just permit the train to move indefinitely at the velocity V_n . The broken line $AFGHI$ should really be a curve, and the grade of repose at any point is the angle between AM and the tangent to that curve at the given point. The "grade of repose" by its definition gives the total resistance of the train at the particular velocity, or multiplying the grade of repose in per cent. by 20 gives the pounds per ton of resistance. Thus being able to determine the total resistance in pounds per ton at any velocity, the variation of total resistance with velocity may be determined, and then by varying the resistances, using different kinds of cars, empty and loaded, box-cars and flats, the resistances of the different kinds at various velocities may be determined.

350. Formulæ for train resistance. These are generally given in one of the forms

$$\left. \begin{aligned} R &= aV + c, & . & . & . & (1) \\ R &= bV^2 + c, & . & . & . & (2) \\ R &= aV + bV^2 + c, & . & . & . & (3) \end{aligned} \right\} (141)$$

in which R is the resistance in pounds per ton, a and b are coefficients to be determined, V is the velocity in miles per hour, and c is a constant, also to be determined. These formulæ disregard grade and curve resistances, inertia resistance and the active resistance (or assistance) of *wind*, as distinct from mere atmospheric resistance. In short, they are supposed to give the resistance of a train moving at a uniform velocity over a straight and level track, there being no appreciable wind.

The various formulæ are sometimes based directly on experiments made by the proposer of the formula; sometimes they are deduced from a mere study of the results of one or more series of tests made by others. Unfortunately for either method, no one investigator has ever been able to make tests which are so thorough and made under such a wide range of conditions that his results may be considered as conclusive, while a student of the tests of others is handicapped by a lack of knowledge of precise conditions, which, if fully understood, would perhaps permit some reconciliation of the very discordant figures which are reported. As already intimated, the condition of the rolling stock, the unit weight on the axles, the lubrication of the axles, the length of the train in relation to its weight and the condition of the track, which involves the weight of rail, spacing and size of ties, tamping of ties, etc., all have their influence in modifying the apparent resistance. There is also good reason to believe that the effect of grade, curvature, and changing velocity has not been properly allowed for in deducing many of the formulæ. In view of all these considerations, it may be considered as demonstrated that no one formula, and especially a simple formula, will represent the resistance for all conditions. But, since some of the calculations of railroad economics are absolutely dependent on the law of tractive resistance, some law must be deduced with sufficient accuracy for the purpose. Fortunately several of the formulæ are amply accurate for such purposes. A report of a committee of the A. R. E. & M. W. Assoc. (1907) quoted sixty-one different formulæ which have been suggested. Some of these are chiefly of historical value, since they were deduced from tests made many years ago with track and rolling stock very dissimilar from those in use at the present time. Such formulæ will therefore be omitted. For convenience of comparison, all formulæ will be changed (if necessary) from the original statement of them so that they give the

resistance per ton of 2000 pounds. The coefficients of V and V^2 will be given decimally. Other notation occasionally used is as follows:

t = weight of train in tons of 2000 pounds;
 L = length of train in feet;
 n = number of cars in train;
 A = area of front of train in square feet.

(a) Formulæ of the first class: $R = aV + c$. Among those most commonly used are the following:

$$\text{Engineering News,} \quad R = 0.25V + 2.0 \quad . \quad . \quad . \quad . \quad . \quad (142)$$

$$\text{Baldwin locomotive,} \quad R = 0.17V + 3.0 \quad . \quad . \quad . \quad . \quad . \quad (143)$$

$$\text{New York Central,} \quad R = 0.11V + 1.8, \quad . \quad . \quad . \quad . \quad . \quad (144)$$

$$\text{Henderson,} \quad R = 0.25V + \frac{50n}{t} + 0.5. \quad . \quad . \quad . \quad . \quad (145)$$

Although Henderson's formula is in a class by itself, on account of the extra term, and although it is not applicable to general use, when the character of the trains cannot be estimated, it is perhaps more accurate than the others. It is apparently not intended for use at very low velocities.

(b) Formulæ of the second class: $R = bV^2 + c$:

$$\text{Crawford,} \quad R = 0.00214V^2 + 2.5 \quad . \quad . \quad . \quad . \quad . \quad (146)$$

$$\text{Wolff,} \quad R = 0.00357V^2 + 2.7 \quad . \quad . \quad . \quad . \quad . \quad (147)$$

$$\text{Henderson,} \quad R = 0.00461V^2 + 3.0 \quad . \quad . \quad . \quad . \quad . \quad (148)$$

$$\text{Forney,} \quad R = 0.00585V^2 + 4.0 \quad . \quad . \quad . \quad . \quad . \quad (149)$$

$$\text{Wel-} \left\{ \begin{array}{l} R = 0.0056V^2 + \frac{.57V^2}{t} + 3.9 \text{ (for loaded flat cars)} \\ R = 0.0075V^2 + \frac{.64V^2}{t} + 3.9 \text{ (for loaded box cars)} \\ R = 0.0083V^2 + \frac{.57V^2}{t} + 6.0 \text{ (for empty flat cars)} \\ R = 0.0106V^2 + \frac{.64V^2}{t} + 6.0 \text{ (for empty box cars)} \end{array} \right\}, \quad (150)$$

Notice in formulæ (150) the additional journal resistance (indicated by the constant term) for unloaded cars. The second

term evidently indicates the atmospheric resistance. The first term allows for the oscillatory resistances. Assuming the constant term and the coefficients to have been correctly determined, these formulæ should be better than the others, since a choice of formulæ can be made depending on the conditions. A train consisting partly of box-cars and partly of flat-cars will have a higher resistance than is shown by any of the above formulæ (and *not* a mean value), on account of the increased atmospheric resistance acting on the irregular form of the train.

(c) Formulæ of the third class: $R = aV + bV^2 + c$:

$$\text{W. N. Smith,} \quad R = 0.17V + \frac{0.0025AV^2}{t} + 3.0; \quad . \quad . \quad . \quad (151)$$

$$\text{Von Borries,} \quad R = 0.04V + 0.0016V^2 + 3.0; \quad . \quad . \quad . \quad (152)$$

$$\text{Lundie,} \quad R = 0.24V + \frac{4.8V^2}{t} + 4.0; \quad . \quad . \quad . \quad . \quad (153)$$

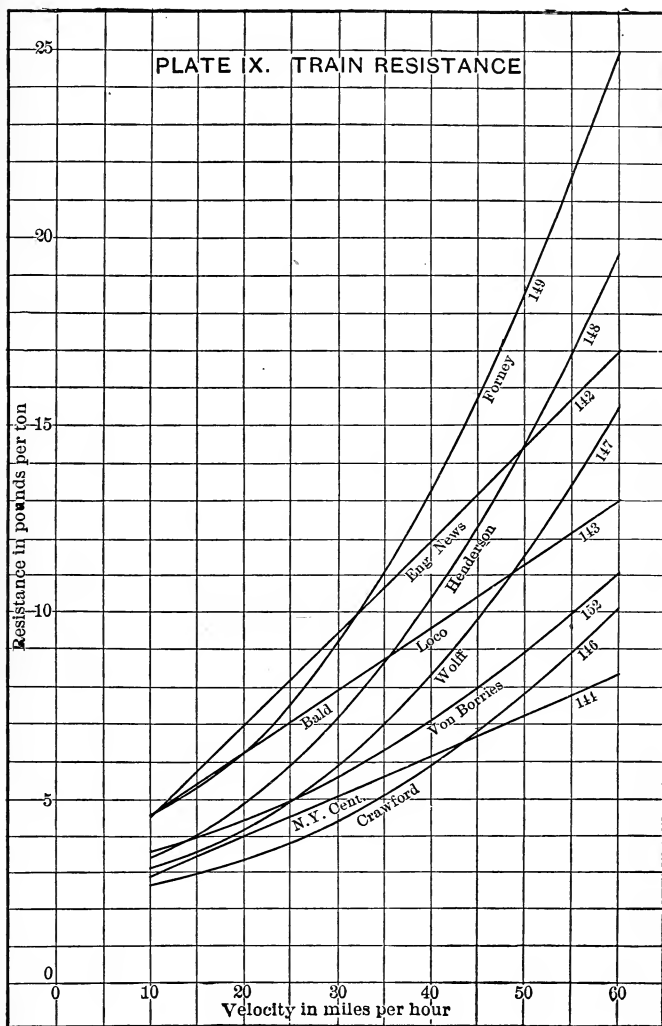
$$\text{Sprague,} \quad R = 0.17V + \frac{0.333V^2}{t} + 4.0. \quad . \quad . \quad . \quad . \quad (154)$$

Although several formulæ have been proposed which involve the area of the front of the train in order to allow more definitely for the atmospheric resistance, only one of these (151) has been quoted. In applying this formula, the proper value to choose for A is somewhat indefinite, since the shape of the front of the train will make a considerable difference in the atmospheric resistance encountered. The area will vary from 80 to 100 square feet. In the comparison of the formulæ given below, A will be assumed as 100 square feet. In order to compare these resistances, the values of R for the various speeds of 10, 20, 30, 40, 50, and 60 miles per hour will be computed by these formulæ on the basis of a train of twelve cars, having a length of 480 feet, and a weight of 600 tons. Therefore in applying the formula, $t=600$, $L=480$, $n=12$, and $A=100$. In order to apply formula (150) to this case, it will be assumed that this train consists of loaded box-cars, and therefore we must apply the second of that group of formulæ. Computing the resistance according to these several formulæ, we may tabulate the results as given below:

Formula.	Velocity in miles per hour.					
	10	20	30	40	50	60
142	2700	4200	5700	7200	8700	10200
143	2800	3800	4800	5800	6800	7800
144	1747	2413	3080	3747	4413	5080
145	2400	3900	5400	6900	8400	9900
146	1628	2014	2656	3554	4710	6122
147	1834	2477	3548	5047	6975	9331
148	2077	2906	4289	6226	8715	11746
149	2751	3804	5559	8116	11175	15036
150	2854	4396	6966	10564	15188	20844
151	2845	3940	5085	6280	7525	8820
152	2136	2664	3384	4296	5400	6696
153	4320	7200	11040	15840	19440	28080
154	3453	4573	5760	7013	8333	9720

Although there is a fair agreement among the results for ordinary velocities, it should be said, in fairness to the proposers of the various formulæ, that some of them evidently were not designed for use at high velocities such as 60 miles per hour.

Another method of comparing formulæ is to plot them on cross-section paper, using velocities as abscissæ and resistances as ordinates. For general use this method may only be applied to formulæ which do not involve the weight, length or area of the train nor the number of cars. All of the above formulæ have thus been plotted on Plate IX, with the exception of Nos. 145, 150, 151, 153, and 154.



CHAPTER XVII

COST OF RAILROADS.

351. General considerations. Although there are many elements in the cost of railroads which are roughly constant per mile of road, yet the published reports of the cost of railroads differ very widely. The variation in the figures is due to several causes. (a) Economy requires that a road shall be operated and placed on an earning basis as soon as possible. Therefore the reported cost of a road during the first few years of its existence is somewhat less than that reported later. This is well illustrated when a long series of consecutive reports from an old-established road is available; nearly every year there will be shown an addition to the previous figures. And this is as it should be. The magnificent road-beds of some old roads cannot be the creation of a single season. It takes many years to produce such settled perfect structures. (b) A large part of the variation is due to a neglect to charge up "permanent improvements" as additions to the cost of the road. For the first few years of the life of a road a great deal of work is done which is in reality a completion of the work of construction, and yet the cost of it is buried under the item "maintenance of way." For example, a long wooden trestle is replaced by an earth embankment and a culvert. Since the original trestle is to be considered a temporary structure, the *excess* of the cost of the permanent structure over that of the temporary structure should evidently be considered as an addition to the cost of the road. But if the filling-in was done slowly, a few train-loads at a time, and the work scattered over many years, the cost of operating the "mud-train" has perhaps been buried under "maintenance" charges. (c) The reports from which many of the following figures were taken have not always analyzed the items of cost with the same detail as has been here attempted, and to that is probably due many of the variations and apparent discrepancies.

The various items of cost will be classified as follows:

1. Preliminary financiering.
2. Surveys and engineering expenses.
3. Land and land damages.
4. Clearing and grubbing.
5. Earthwork.
6. Bridges, trestles, and culverts
7. Trackwork.
8. Buildings and miscellaneous structures.
9. Interest on construction.
10. Telegraph line.

352. Item 1. PRELIMINARY FINANCIERING. The cost of this preliminary work is exceedingly variable. The work includes the clerical and legal work of organization, printing, engraving of stocks and bonds, and (sometimes the most expensive of all) the securing of a charter. This sometimes requires special legislative enactments, or may sometimes be secured from a State railroad commission. It has been estimated that about 2% of the railway capital of Great Britain has been spent in Parliamentary expenses over the charters. These expenses are usually but a small percentage of the total cost of the enterprise, but for important lines the gross cost is large, while the amount of money thus spent by organizations which have never succeeded in constructing their roads is, in the aggregate, an enormous amount, although it is of course not ascertainable by any investigator.

Another occasional feature of the financing of a road must be kept in mind. The promoters of a railroad enterprise frequently endeavor to limit their own personal expenditures to the purely preliminary expenses as mentioned above. The project, after having been surveyed, mapped, and written up in a glowing "prospectus," is submitted to capitalists, in the endeavor to have them furnish money for construction, the money to be secured by bonds. If the project will stand it, the amount of the bond issue is made sufficient to pay the entire cost of the road, even with a discount of perhaps 15%. The bond issue may also provide for a very generous commission to the broker who is the intermediary between the promoters and the capitalists. The bond issue may even provide for repaying the promoters for their preliminary expenses. Frequently a considerable proportion of the capital stock goes to the capitalists

who take the bonds, the promoters retaining only such proportion as may be agreed upon. In such a case, the capital stock is "pure velvet," and costs nothing. Its future value, whatever it may be, is so much clear profit. The effect of such a financial policy is to burden the project with a capitalization which is far in excess of the actual cost of constructing the road. Comparatively few projects will stand such over-capitalization. The apparent financial failure of many railroads, which have gone into the hands of receivers is due to their inability to make returns on an over-capitalization rather than because they could not earn enough to pay the legitimate cost of their construction. These features of financiering are really foreign to the engineer's work, but he should know that many projects which would return a handsome profit on an investment amounting only to the legitimate cost, will be rejected by capitalists because it is apparent that there is not enough "velvet" in it.

353. Item 2. SURVEYS AND ENGINEERING EXPENSES. The comparison of a large number of itemized reports on the cost of construction shows that the cost of the "engineering" will average about 2% of the total cost of construction. This includes the cost of surveys and the cost of laying out and superintending the constructive work. The cost of mere surveying up to the time when construction actually commences has been variously quoted at \$60, \$75, and even \$150 per mile. In exceptional cases the surveying for a few miles through some gorge might cost many times this amount, but \$150 per mile may be considered an ordinary maximum for difficult country. On the other hand, much construction has been done over the western prairies after hasty surveys costing not much over \$10 per mile. In the estimate given at the end of this chapter the cost of "engineering and office expenses" is given at 5% of the cost of the construction work. The item then includes the cost of the very considerable amount of clerical work and superintendence incident to the expenditure of such a large sum of money.

354. Item 3. LAND AND LAND DAMAGES. The cost of this item varies from the extreme, in which not only the land for

right-of-way but also grants of public land adjoining the road are given to the corporation as a subsidy, to the other extreme, where the right-of-way can only be obtained at exorbitant prices. The width required is variable, depending on the width that may be needed for deep cuts or high fills, or the extra land required for yards, stations, etc. A strip of land 1 mile long and 8.25 feet wide contains precisely 1 acre. An average width of 4 rods (66 feet), therefore, requires 8 acres per mile. On the Boston & Albany Railroad the expenditure assigned to "land and land damages" averages over \$25000 per mile. Of course this includes some especially expensive land for terminals and stations in large cities. Less than \$300 per mile was assigned to this item by an unimportant 18-mile road.

355. Item 4. CLEARING AND GRUBBING. The cost of this may vary from zero to 100% for miles at a time, but as an average figure it may be taken as about 3 acres per mile at a cost of say \$50 per acre. The possibility of obtaining valuable timber, which may be utilized for trestles, ties, or otherwise, and the value of which may not only repay the cost of clearing and grubbing, but also some of the cost of the land, should not be forgotten.

356. Item 5. EARTHWORK. This item also includes rockwork. The methods of estimating the cost of earthwork and rockwork have been discussed in Chapter III. The percentage of this item to the total cost is very variable. On a western prairie it might not be more than 5 to 10%. On a road through the mountains it will run up to 20 or 25%, and even more. The item also includes tunneling, which on some roads is a heavy item.

357. Item 6. BRIDGES, TRESTLES, AND CULVERTS. This item will usually amount to 5 or 6% of the total cost of the road. In special cases, where extensive trestling is necessary, or several large bridges are required, the percentage will be much higher. On the other hand, a road whose route avoids the watercourses may have very little except minor culverts. On the Boston & Albany the cost is given as \$5860 per mile; on the Adirondack Railroad, \$2845 per mile. Considering their relative character (double and single track), these figures are relatively what we might expect.

358. Item 7. TRACKWORK. This item will be considered as including everything above subgrade, except as otherwise itemized.

(a) **Ballast.** With an average width, for single track, of 10 feet and an average depth of 15 inches, 2444 cubic yards of ballast will be required. The Pennsylvania Railroad estimate is 2500 yards of gravel per mile of single track. At an estimate of 60 c. per yard, this costs \$1500 per mile. Broken-stone ballast must be filled out over the ends of the ties and therefore more is required; 2800 cubic yards of broken stone at \$1.25 per yard in place will cost \$3500 per mile.

(b) **Ties.** Ties cost anywhere from 80 c. down to 35 c. and even 25 c. At an average figure of 50 c., 2640 ties per mile will cost \$1320 per mile of single track. The cheaper ties are usually smaller and more must be used per mile, and this tends to compensate the difference in cost.

The following tabular form is convenient for reference:

TABLE XV.—NUMBER OF CROSS TIES PER MILE.

Number per 33' rail.	Average spacing center to center.	Number per mile.
22	18.0 inches	3520
21	18.9 "	3360
20	19.8 "	3200
19	20.9 "	3040
18	22.0 "	2880
17	23.3 "	2720
16	24.75 "	2560
15	26.4 "	2400
14	28.3 "	2240
13	30.5 "	2080

(c) **Rails.** The total weight of the rails used per mile may best be seen by the tabular form.

A convenient and useful rule to remember is that the number of *long tons* (2240 lbs.) per mile of single track equals the weight of the rail per yard times $\frac{11}{7}$. The rule is exact. For example, there are 3520 yards of rail in a mile of single track; at 70 lbs. per yard this equals 246400 lbs., or 110 long tons (exactly); but $70 \times \frac{11}{7} = 110$.

Any calculation of the required weight of rail for a given weight of rolling-stock necessarily depends on the assumptions which are made regarding the support which the rails receive from the ties. This depends not only on the width and spacing

TABLE XVI.—TONS PER MILE (WITH COST) OF RAILS OF VARIOUS WEIGHTS.

Weight in lbs. per yd.	Tons (2240 lb.) per mile of single track.	Cost at \$26 per ton.	Cost at \$30 per ton.	Weight in lbs. per yd.	Tons (2240 lb.) per mile of single track.	Cost at \$26 per ton.	Cost at \$30 per ton.
8	12.571	\$326.86	\$377.14	65	102.143	\$2655.71	\$3064.29
10	15.714	408.57	471.43	66	103.714	2696.57	3111.43
12	18.857	490.29	565.71	67	105.286	2737.43	3158.59
14	22.000	572.00	660.00	68	106.857	2778.29	3205.79
16	25.143	653.71	754.20	70	110.000	2860.00	3300.00
20	31.429	817.14	942.86	71	111.571	2900.86	3347.14
25	39.286	1021.43	1178.57	72	113.143	2941.71	3394.29
30	47.143	1225.71	1414.29	73	114.714	2982.57	3441.43
35	55.000	1430.00	1650.00	75	117.857	3064.29	3535.71
40	62.857	1634.29	1885.71	78	122.571	3186.86	3677.14
45	70.714	1838.57	2121.43	80	125.714	3268.57	3771.43
48	75.429	1961.14	2262.86	82	128.857	3350.29	3865.71
50	78.571	2042.86	2357.14	85	133.571	3472.86	4007.14
52	81.714	2124.57	2451.43	88	138.286	3595.43	4148.57
56	88.000	2288.00	2640.00	90	141.429	3677.14	4242.86
57	89.571	2328.86	2687.14	92	144.571	3758.86	4337.14
60	94.286	2451.43	2828.57	95	149.286	3881.43	4478.57
61	95.857	2492.29	2875.71	98	154.000	4004.00	4620.00
63	99.000	2574.00	2970.00	100	157.143	4085.71	4714.29

About two per cent. (2%) extra should be allowed for waste in cutting.

of the ties (which are determinable), but also on the support which the ties receive from the ballast, which is not only very uncertain but variable. No general rule can therefore claim any degree of precision, but the following is given by the Baldwin Locomotive Works: "Each ten pounds weight per yard of ordinary steel rail, properly supported by cross-ties (not less than 14 per 30-foot rail), is capable of sustaining a safe load per wheel of 2240 pounds." For example, a consolidation locomotive with 112600 lbs. on 8 drivers has a load of 14075 lbs. per wheel. This divided by 2240 gives 6.28. According to the rule, the rails for such a locomotive should weigh at least 62.8 lbs. per yard.

(d) **Splice-bars, track-bolts, and spikes.** These are usually sold by the pound, except the patented forms of rail-joints, which are sold by the pair. In any case they are subject to market fluctuations in price. As an approximate value the following prices are quoted: Splice-bars, 1.35 c. per pound; track-bolts, 2.4 c.; spikes, 1.75 c. The weight of the splice-bars will depend on the precise pattern adopted—its cross-section and length.

In Table XVII are quoted from a catalogue of the Illinois Steel Co. the weights per foot of sections of angle-bars which they recommend for various weights of rail and which are designed to fit standard A. S. C. E. rail sections of those weights. The net weight of the angle-bars may be approximated by subtracting about 2.5% to 4% from the gross weight to allow for the bolt-holes. A deduction of 2.5% is usually about right for the heavier sections. Their recommendations regarding lengths of angle-bars do not include those for rails heavier than 50 pounds per yard. On the basis of a length of 23 inches for four-hole splices and of 33 inches for six-hole splices, the weights of splice-bars have been computed for the several styles of splices for heavier rails, allowing 2.5% for the holes. The lengths recommended for track bolts are those which will allow about $\frac{1}{2}$ inch for the nutlock and for margin, except for the lighter rails.

TABLE XVII.—SPlice-BARS FOR VARIOUS WEIGHTS OF RAILS.

Weight of rail.	Length of angle-bar.	Weight per foot.	Weight of pair.	Proper size of track-bolt.	Proper size of spikes.
30	21"	4.49	15.1	2 $\frac{1}{2}$ " × 5"	4" × $\frac{1}{2}$ "
35	21"	4.7	15.9	2 $\frac{3}{4}$ " × 5"	4 $\frac{1}{2}$ " × $\frac{1}{2}$ "
40	21"	5.54	18.8	3" × 5"	5" × $\frac{1}{2}$ "
45	21"	6.3	21.5	3" × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
50	21"	6.97	23.4	3 $\frac{1}{2}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
55	23"	7.5	28.0	3 $\frac{3}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
60	23"	8.4	31.4	3 $\frac{3}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
65	23"	9.2	34.4	4" × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
	33"	9.6	51.5	4 $\frac{1}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
	23"	9.0	33.6	4" × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
70	33"	10.0	53.6	4" × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
	23"	10.68	39.9	4 $\frac{1}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
75	33"	11.9	63.7	4" × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
	23"	10.61	39.7	4 $\frac{1}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
80	33"	14.65	78.5	4 $\frac{1}{2}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
85	33"	12.4	66.4	4 $\frac{1}{2}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
90	33"	13.5	72.3	4 $\frac{3}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
95	33"	14.7	78.7	4 $\frac{3}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "
100	33"	15.78	85.0	4 $\frac{3}{4}$ " × 5"	5 $\frac{1}{2}$ " × $\frac{9}{16}$ "

OR
5" × $\frac{1}{2}$ "
OR
5" × $\frac{9}{16}$ "
OR
5" × $\frac{1}{2}$ "
OR
5" × $\frac{9}{16}$ "

TABLE XVIII.—RAILROAD SPIKES.

Size measured under head.	Average number per keg of 200 pounds	Ties 24" between centers, 4 spikes per tie, number per mile.		Suitable weight of rail.
		Pounds.	Kegs.	
5½" × ⅝"	275	7680	38.40	90 to 100
5½" × ⅞"	375	5632	28.16	45 " 100
5" × ⅞"	400	5280	26.40	40 " 56
5" × 1½"	450	4692	23.46	40
4½" × 1½"	530	3984	19.92	35
4" × 1½"	600	3520	17.60	30
4½" × 7⁄16"	680	3104	15.52	25 to 30

TABLE XIX.—TRACK-BOLTS.

Average number in a keg of 200 pounds.

Size of bolt.	Square nut.	Hexagonal nut.	Suitable rail.
3" × ⅝"	366	395	40 pound
3" × ¾"	250	270	
3½" × ¾"	243	261	
3½" × 1"	236	253	50
3½" × 1¼"	229	244	55 to 60
4" × 1¼"	222	236	65 " 70
4½" × 1¼"	215	228	75
3½" × 7⁄8"	170	180	
3½" × 7⁄8"	165	175	
4" × 7⁄8"	161	170	
4½" × 7⁄8"	157	165	80
4½" × 7⁄8"	153	160	85
4½" × 7⁄8"	149	156	90

(e) **Track-laying.** Much depends on the force of men employed and the use of systematic methods; \$528 per mile is the estimate employed by the Pennsylvania Railroad. \$500 per mile is the estimate given in § 362.

359. Item 8. BUILDINGS AND MISCELLANEOUS STRUCTURES. Except for rough and preliminary estimates, these items must be individually estimated according to the circumstances. The subitems include depots, engine-houses, repair-shops, water-stations, section- and tool-houses, besides a large variety of smaller buildings. The structures include turn-tables, cattle-guards, fencing, road-crossings, overhead bridges, etc. The detailed estimate, given in § 362, illustrates the cost of these smaller items.

360. Item 9. INTEREST ON CONSTRUCTION. The amount of capital that must be spent on a railroad before it has begun to earn anything is so very large that the interest on the cost during the period of construction is a very considerable item. The amount that must be charged to this head depends on the current rate of money on the time required for construction and on the ability of the capitalists to retain their capital where it will be earning something until it is actually needed to pay the company's obligations. Of course, it is not necessary to have the entire capital needed for construction on hand when construction commences. Assuming money to be worth 6%, that the work of construction will require one year, that the money may be retained where it will earn something for an average period of six months after construction commences, or, in other words, it will be out of circulation six months before the road is opened for traffic and begins to earn its way, then we may charge 3% on the total cost of construction.

361. Item 10. TELEGRAPH LINES. This evidently depends on the scale of the road and the magnitude of the business to be operated. In the following estimate it is given as \$200 per mile, which evidently is intended to apply to the business of a small road.

362. Detailed estimate of the cost of a line of road. The following estimate was given in the *Engineering News* of Dec. 27, 1900, of the cost of the Duluth, St. Cloud, Glencoe & Mankato Railroad, 157.2 miles long.

The estimate is exactly as copied from the *Engineering News*. There are some numerical discrepancies. Item 26 should evi-

dently be based on the sum of the first 25 items, and item 27 on the sum of the first 26. The figures in parentheses () are deduced from the figures given.

1. Right-of-way: 1905.3 acres (12.12 acres per mile) @ \$100 per acre.....	\$190530
2. Clearing and grubbing. 144 acres (0.916 acre per mile) @ \$50 per acre.....	7200
3. Earth excavation. 1907590 cu. yds. (12135 cu. yds. per mile) @ 15 c.....	286138
4. Rock excavation. 5100 cu. yds. (32.44 cu. yds. per mile) @ 80 c.....	4080
5. { Wooden-box culverts. 508300 ft. B.M. @ \$30 per M..	\$15249
5. { Iron-pipe culverts: 879840 lbs. @ 3c. per lb.....	26395
6. { Pile trestling: 4600 lin. ft. @ 35 c. per lin. ft.....	1610
6. { Timber trestling: 509300 ft. B.M. @ \$30 per M.....	15279
7. { Bridge masonry: 5520 cu. yds. @ \$8 per cu. yd.....	44160
7. { Bridges, iron, 100 spans, 2000000 lbs. @ 4 c. per lb...	80000
8. Cattle-guards.....	8750
9. Ties (2640 per mile). 419813 (159.02 miles) @ 35 c.....	146935
10. Rails (70 lbs. per yd.): 110 tons per mile, 17492.2 tons (159.02 miles @ \$26.....	384797
11. Rail sidings (70 lbs. per yd.): 110 tons per mile, 3300 tons (30 miles @ \$26.....	85800
12. Switch timbers and ties.....	3300
13. Spikes: 5920 lbs. per mile. 1107040 (187 m.) @ 1.75. c. per lb.....	19373
14. Splice-bars: 2635776 lbs. @ 1.35 c. per lb.....	35583
15. Track-bolts (2 to joint (?)): 188458.3 lbs. @ 2.4 c. per lb.....	4520
16. Track-laying 187.2 miles @ \$500 per mile.....	93600
17. Ballasting: 2152 cu. yds. per mile, 402854 (187.2 m.) @ 60 c..	241712
18. Turn-out and switch furnishings.....	6450
19. Road-crossings, 68040 ft. B.M. @ \$30 per M.....	2041
20. Section and tool-houses, 16 @ \$800.....	12800
21. Water-stations.....	15000
22. Turn-tables, 6 @ \$800.....	4800
23. Depots, grounds, and repair-shops.....	78000
24. Terminal grounds and special land damages.....	150000
25. Fencing, 314 miles (\$150 per mile).....	47100
26. Engineering and office expenses (5% of \$1984458).....	99222
27. Interest on construction (3% of \$2083680).....	62510
28. Rolling-stock (\$5000 per mile).....	786000
29. Telegraph line: 157 miles @ \$200 per mile.....	31400
	<u>\$3060340</u>

Average cost per mile ready for operation, \$19467.

Approximate cost of 130 miles from St. Cloud to Duluth, estimated at \$23000 per mile.

Approximate cost of entire line from Albert Lea to Duluth, 287.2 miles, \$6050340 (\$21060 per mile).

PART II.

RAILROAD ECONOMICS.

CHAPTER XVIII.

INTRODUCTION.

363. The magnitude of railroad business. The gross earnings of railroads for the year ending June 30, 1910, were \$2,750,667,-435. This is greater than the combined value of all the gold, silver, iron, wheat, and corn produced by the country.

About 1,700,000 persons (over one sixtieth of the population) were directly employed by the roads for a compensation of \$1,143,725,306. Probably 5,000,000 to 6,000,000 people were supported by this. Besides all these, probably 8,000,000 employés were kept busy in occupations which are a more or less direct result of railroads, *e.g.*, locomotive- and car-shops, rail-mills, etc. We therefore may estimate that perhaps 25,000,000 people (or over one fourth of our population) are supported by railroads or by occupations which owe their chief existence to railroads.

The "number of passengers carried 1 mile" was 32,338 millions. Calling the population of the United States 92,000,000 for round numbers, it means an average ride of 351 miles for every man, woman, and child.

The "tons carried 1 mile" were 255,017 millions, or nearly 2773 ton-miles per inhabitant. The payments made to the railroads averaged nearly \$30 per inhabitant.

The actual bona-fide cost of the railroads of the country cannot be accurately computed (as will be shown later), but the capital, as represented by stocks and bonds, aggregated

in 1910, \$18,417,132,238 or about \$200 per inhabitant. This is roughly about one sixth of the total national wealth.

The above figures may give some idea of the magnitude of the interests involved in the operation of railroads. No single business in the country approaches it in capital involved, earnings, number of people affected, or effect on other business.

364. Cost of transportation. The importance of railroads may be also indicated by their power of creating cheap transportation. Less than one hundred years ago local famine and overabundant harvests within a radius of a few miles were not unknown. When the transportation of goods depended on actual portorage by human beings, as has been the case but recently in the Klondike, the transportation of 50 lbs. 20 miles might be considered a hard day's labor. At \$2 per day, this equals \$4 per ton-mile. In 1909 the railroads transported freight at an *average* cost to the public of 0.763 c. per ton per mile, and the feeding of Europe with wheat from Manitoba has become a commercial possibility. In 1909 passengers paid an average charge of 1.928 c. per mile, and a trip of 1000 miles inside of 24 hours is now common.

365. Study of railroad economics—its nature and limitations. The multiplicity of the elements involved in most problems in railroad construction preclude the possibility of a solution which is demonstrably perfect. Barring out the comparatively few cases in this country where it is difficult to obtain *any* practicable location, it may be said that a comparatively low order of talent will suffice to locate anywhere a railroad over which it is physically possible to run trains. It may be very badly located for obtaining business, the ruling grades may be excessive, the alignment may be very bad, and the road may be a hopeless financial failure, and yet trains *can* be run. Among the infinite number of possible locations of the road, the engineer must determine the route which will give the best railroad property for the least expenditure of money—the road whose earning capacity is so great that after paying the operating expenses and interests on the bonds, the surplus available for dividends or improvements is a maximum.

An unfortunate part of the problem is that even the blunders are not always readily apparent nor their magnitude. A defective dam or bridge will give way and every one realizes the failure, but a badly located railroad affects chiefly the finances

of the enterprise by a series of leaks which are only perceptible and demonstrable by an expert, and even he can only say that certain changes would probably have a certain financial value.

366. Outline of the engineer's duties. The engineer must realize at the outset the nature and value of the conflicting interests which are involved in variable amount in each possible route.

(a) **The maximum of business must be obtained**, and yet it *may* happen that some of the business may only be obtained by an extravagant expenditure in building the line or by building a line very expensive to operate.

(b) **The ruling grades should be kept low**, and yet this *may* require a sacrifice in business obtained and also *may* cost more than it is worth.

(c) **The alignment should be made as favorable as possible**; favorable alignment reduces the future operating expenses, but it may require a very large immediate outlay.

(d) **The total cost must be kept within the amount at which the earnings will make it a profitable investment.**

(e) **The road must be completed and operated until the "normal" traffic is obtained and the road is self-supporting without exhausting the capital obtainable by the projectors**; for no matter how valuable the property may ultimately become, the projectors will lose nearly, if not quite, all they have invested if they lose control of the enterprise before it becomes a paying investment.

Each new route suggested makes a new combination of the above conflicting elements. The engineer must select a route by first eliminating all lines which are manifestly impracticable and then gradually narrowing the choice to the best routes whose advantages are so nearly equal that a closer detailed comparison is necessary.

The ruling grade and the details of alignment have a large influence on the operating expenses. A large part of this course of instruction therefore consists of a study of operating expenses under average normal conditions, and then a study of the effect on operating expenses of given changes in the alignment.

367. Justification of such methods of computation. It may be argued that the data on which these computations are based are so unreliable (because variable and to some extent non-

computable) that no dependence can be based on the conclusions. This is true to the extent that it is useless to claim great precision in the computation of the value of any proposed change of alignment. Suppose, for example, it is computed that a given improvement in alignment will reduce the operating expenses of 20 trains per day by \$1000 per year. Suppose the change in alignment may be made for \$5000, which may be obtained at 5% interest. Even with large allowances for inaccuracy in the computation of the value, \$1000, it evidently will be better to incur an additional interest charge of \$250 than increase the annual operating expenses by \$1000. Moreover, since traffic is almost sure to increase (and interest charges are generally decreasing), the advantage of the improvement will only increase as time passes. On the other hand, if the improvement cannot be made except by an expenditure of, say, \$50000, the change would evidently be unjustifiable. When the interest on the first cost is practically equal to the annual operating value of the proposed improvement, there is evidently but little choice; no great harm can result from either decision, and the decision frequently will depend on the willingness to increase the total amount invested in the enterprise.

To express the above question more generally, in every computation of the operating value of a proposed improvement, it may always be shown that the true value lies somewhere between some maximum and some minimum. Closer calculations and more reliable data will narrow the range between these extreme values. According as the interest on the cost of the proposed improvement is greater or less than the mean of these limits, we may judge of its advisability. The range of the limits shows the uncertainty. If it lies outside of the limits there is no uncertainty, assuming that the limits have been properly determined. If well within the limits, either decision will answer unless other considerations determine the question. And so, although it is not often possible to obtain precise values, we may generally reach a conclusion which is unquestionable. Even under the most unfavorable circumstances, the computations, when made with the assistance of all the broad common sense and experience that can be brought to bear, will point to a decision which is much better than mere "judgment," which is responsible for very many glaring and

costly railroad blunders. In short, Railroad Economics means the application of systematic methods of work *plus* experience and judgment, rather than a dependence on judgment unsystematically formed. It makes no pretense to furnishing mechanical rules by which all railroad problems may be solved by any one, but it does give a general method of applying principles by which an engineer of experience and judgment can apply his knowledge to better advantage. To the engineer of limited experience the methods are invaluable; without such methods of work his opinions are practically worthless; with them his conclusions are frequently more sound than the unsystematically formed judgments of a man with a glittering record. But the engineer of great experience may use these methods to form the best opinions which are obtainable, for he can apply his experience to make any necessary local modifications in the method of solution. The dangers lie in the extremes, either recklessly applying a rule on the basis of insufficient data to an unwarrantable extent, or, disgusted with such evident unreliability, neglecting altogether such systematic methods of work.

CHAPTER XIX.

THE PROMOTION OF RAILROAD PROJECTS.

368. Method of formation of railroad corporations. Many business enterprises, especially the smaller ones, are financed entirely by the use of money which is put into them directly in the form of stock or mere partnership interest. A railroad enterprise is frequently floated with a comparatively small financial expenditure on the part of the original promoters. The promoters become convinced that a railroad between *A* and *B*, passing through the intermediate towns of *C* and *D*, with others of less importance, will be a paying investment. They organize a company, have surveys made, obtain a charter, and then, being still better able (on account of the additional information obtained) to exploit the financial advantages of their scheme, they issue a prospectus and invite subscriptions to bonds. Sometimes a portion of these bonds are guaranteed, principal and interest, or perhaps the principal alone, by townships or by the national government. The cost of this preliminary work, although large in gross amount if the road is extensive, is yet but an insignificant proportion of the total amount involved. The proportionate amount that *can* be raised by means of bonds varies with the circumstances. In the early history of railroad building, when a road was projected into a new country where the traffic possibilities were great and there was absolutely no competition, the financial success of the enterprise would seem so assured that no difficulty would be experienced in raising from the sale of bonds all the money necessary to construct and equip the road. But the promoters (or stockholders) must furnish all money for the preliminary expenses, and must make up all deficiencies between the proceeds of the sale of the bonds and the capital needed for construction.

“In theory, stocks represent the property of the responsible owners of the road, and bonds are an encumbrance on that

property. According to this theory, a railroad enterprise should begin with an issue of stock somewhere near the value of the property to be created and no more bonds should be issued than are absolutely necessary to complete the enterprise. Now it is not denied that there are instances in which this theory is followed out. In New England, for example, as well as in some of the Southern States, there are a few roads represented wholly by stock or very lightly mortgaged. But this theory does not conform to the general history of railway construction in the United States, nor is it supported by the figures that appear in the summary. The truth is, railroads are built on borrowed capital, and the amount of stock that is issued represents in the majority of cases the difference between the actual cost of the undertaking and the confidence of the public expressed by the amount of bonds it is willing to absorb in the ultimate success of the venture." *

"The same general law obtains and has always obtained throughout the world, that such properties (as railways) are always built on borrowed money up to the limit of what is regarded as the positive and certain minimum value. The risk only—the dubious margin which is dependent upon sagacity, skill, and good management—is assumed and held by the company proper who control and manage the property." †

369. The two classes of financial interests—the security and profits of each. From the above it may be seen that stocks, bonds, car-trust obligations, and even current liabilities represent railroad capital. The issue of the bonds "was one means of collecting the capital necessary to create the property against which the mortgage lies." The variation between these interests lies chiefly in the security and profits of each. The current liabilities are either discharged or, as frequently happens, they accumulate until they are funded and thus become a definite part of the railroad capital.

The growth of this tendency is shown in the following tabular form (see next page):

The bonded interest has greater security than the stock, but less profit. The interest on the bonds must be paid before any money can be disbursed as dividends. If the bond interest

* Henry C. Adams, Statistician, U. S. Int. Con. Commission.

† A. M. Wellington, *Economic Theory of Railway Location*.

Capitalization of Railroads in the United States.	June 30, 1888.		June 30, 1898.		June 30, 1910.	
	Amount, millions.	Per cent.	Amount, millions.	Per cent.	Amount, millions.	Per cent.
Stocks	3864	47.5	5311	44.6	8,113	44.05
Funded debt	3869	47.6	5510	46.3	10,303	55.95
Current liabilities, etc. .	396	4.9	1087	9.1		

is not paid, a receivership, and perhaps a foreclosure and sale of the road, is a probability, and in such case the stockholder's interests are frequently wiped out altogether. The bondholder's real profit is frequently very different from his nominal profit. He sometimes buys the bonds at a very considerable discount, which modifies the rate which the interest received bears to the amount really invested. Even the bondholder's security may suffer if his mortgage is a second (or fifth) mortgage, and the foreclosure sale fails to net sufficient to satisfy all previous claims.

On the other hand, the stockholder, who may have paid in but a small proportion of his subscription, *may*, if the venture is successful, receive a dividend which equals 50 or 100% of the money actually paid in, or, as before stated, his entire holdings may be entirely wiped out by a foreclosure sale. When the road is a great success and the dividends very large, additional issues of stock are generally made, which are distributed to the stockholders in proportion to their holdings, either gratuitously or at rates which give the stockholders a large advantage over outsiders. This is the process known as "watering." While it may sometimes be considered as a legitimate "salting down" of profits, it is frequently a cover for dishonest manipulation of the money market.

For the twelve years between 1887 and 1899 about *two thirds* of all the railroad stock in the United States paid *no* dividends, while of those that paid dividends the average rate varied from 4.96 to 5.74%. The year from June 30, 1898, to June 30, 1899, was the most prosperous year of the group, and yet nearly 60% of all railroad stock paid *no* dividend, and the average rate paid by those which paid at all was 4.96%. The total amount distributed in dividends was greater than ever before, but the average rate is the least of the above group because many roads, which had passed their dividends for many previous

years, distinguished themselves by declaring a dividend, even though small. During that same period but 13.35% of the stock paid over 6% interest. The total dividends paid amounted to but 2.01% of all the capital stock, while investments ordinarily are expected to yield from 4 to 6% (or more) according to the risk. Of course the effect of "watering" stock is to decrease the nominal rate of dividends, but there is no dodging the fact that, watered or not, even in that year of "good times," about 60% of all the stock paid *no* dividends. Unfortunately there are no accurate statistics showing how much of the stock of railroads represents actual paid-in capital and how much is "water." The great complication of railroad finances and the dishonest manipulation to which the finances of some railroads have been subjected would render such a computation practically worthless and hopelessly unreliable now.

During the year ending June 30, 1898 (which may in general be considered as a sample), 15.82% of the funded debt paid no interest. About one third of the funded debt paid between 4 and 5% interest, which is about the average which is paid.

The income from railroads (both interest on bonds and dividends on stock) may be shown graphically by diagrams, such as are given in the annual reports of the Interstate Commerce Commission. They show that while railroad investments are occasionally very profitable, the average return is less than that of ordinary investments *to the investors*. The *indirect* value of railroads in building up a section of country is almost incalculable and is worth many times the cost of the roads. It is a discouraging fact that very few railroads (old enough to have a history) have escaped the experience of a receivership, with the usual financial loss to the then stockholders. But there is probably not a railroad in existence which, however much a financial failure in itself, has not profited the community more than its cost.

370. The small margin between profit and loss to projectors. When a railroad is built entirely from the funds furnished by its promoters (or from the sale of stock) it will generally be a paying investment, although the rate of payment may be very small. The percentage of receipts that is demanded for actual operating expenses is usually about 67%. The remainder will usually pay a reasonable interest on the total capital involved. But the operating expenses are frequently 90 and even 100% of

the gross receipts. In such cases even the bondholders do not get their due and the stockholders have absolutely nothing. Therefore the stockholder's interest is very speculative. A comparatively small change in the business done (as is illustrated numerically in § 372) will not only wipe out altogether the dividend—taken from the last small percentage of the total receipts and which may equal 50% or more of the capital stock *actually paid in*—but it may even endanger the bondholders' security and cause them to foreclose their mortgage. In such a case the stockholders' interest is usually entirely lost. It does not alter the essential character of the above-stated relations that the stockholders sometimes protect themselves somewhat by buying bonds. By so doing they simply decrease their risk and also decrease the possible profit that might result from the investment of a given total amount of capital.

371. Extent to which a railroad is a monopoly. It is a popular fallacy that a railroad, when not subject to the direct competition of another road, has an absolute monopoly—that it controls “all the traffic there is” and that its income will be practically independent of the facilities afforded to the public. The growth of railroad traffic, like the use of the so-called necessities or luxuries of life, depends entirely on the supply and the cost (in money or effort) to obtain it. A large part of railroad traffic belongs to the unnecessary class—such as traveling for pleasure. Such traffic is very largely affected by mere matters of convenience, such as well-built stations, convenient terminals, smooth track, etc. The freight traffic is very largely dependent on the possibility of delivering manufactured articles or produce at the markets so that the *total* cost of production and transportation shall not exceed the total cost in that same market of similar articles obtained elsewhere. The creation of facilities so that a factory or mine may successfully compete with other factories or mines will develop such traffic. The receipts from such a traffic may render it possible to still further develop facilities which will in return encourage further business. On the other hand, even the partial withdrawal of such facilities may render it impossible for the factory or mine to compete successfully with rivals; the traffic furnished by them is completely cut off and the railroad (and indirectly the whole community) suffers correspondingly. The “strictly necessary” traffic is thus so small that few railroads could pay

their operating expenses from it. The dividends of a road come from the last comparatively small percentage of its revenue, and such revenue comes from the "unnecessary" traffic which must be coaxed and which is so easily affected by apparently insignificant "conveniences."

372. Profit resulting from an increase in business done; loss resulting from a decrease. In a subsequent chapter it will be shown that a large portion of the operating expenses are independent of small fluctuations in the business done and that the operating expenses are roughly two thirds of the gross revenue. Assume that by changes in the alignment the business obtained has been increased (or diminished) 10%. Assume for simplicity that the operating expenses on the revised track are the same as on the route originally planned; also that the cost of the track is the same and hence the fixed charges are assumed to be constant for all the cases considered. Assume the fixed charges to be 28%. The additional business, when carried in cars otherwise but partly filled will hardly increase the operating expenses by a measurable amount. When extra cars or extra trains are required, the cost will increase up to about 60% of the average cost per train mile. We may say that 10% increase may in general be carried at a rate of 40% of the average cost of the traffic. A reduction of 10% in traffic may be assumed to reduce expenses a similar amount. The effect of the change in business will therefore be as follows:

	Business increased 10%.	Business decreased 10%.
Operating exp. = 67	$67(1 + 10\% \times 40\%) = 69.68$	$67(1 - 10\% \times 40\%) = 64.32$
Fixed charges = 28	28.00	28.00
	95	92.32
Total income... 100	Income..... 110.00	Income..... 90.00
Available for dividends..... 5	Available for dividends..... 12.32	Deficit..... 2.32

In the one case the increase in business, which may often be obtained by judicious changes in the alignment or even by better management without changing the alignment, more than doubles the amount available for dividends. In the other case the profits are gone, and there is an absolute deficit. The above is a numerical illustration of the argument, previously

stated, of the small margin between profit and loss to the original projectors.

373. Estimation of probable volume of traffic and of probable growth. Since traffic and traffic facilities are mutually interdependent and since a large part of the normal traffic is merely potential until the road is built, it follows that the traffic of a road will not attain its normal volume until a considerable time after it is opened for operation. But the estimation even of this normal volume is a very uncertain problem. The estimate may be approached in three ways:

1st. The actual gross revenue derived by all the railroads in that section of the country (as determined by State or U. S. Gov. reports) may be divided by the total population of the section and thus the average annual expenditure per head of population may be determined. A determination of this value for each one of a series of years will give an idea of the normal rate of growth of the traffic. Multiplying this annual contribution by the population which may be considered as tributary gives a *valuation* of the possible traffic. Such an estimate is unreliable (a) because the *average* annual contribution may not fit that particular locality, (b) because it is very difficult to correctly estimate the number of the true tributary population especially when other railroads encroach more or less into the territory. Since a rough value of this sort may be readily determined, it has its value as a check, if for nothing else.

2d. The actual revenue obtained by some road whose circumstances are as nearly as possible identical with the road to be considered may be computed. The weak point consists in the assumption that the character of the two roads is identical or in incorrectly estimating the allowance to be made for observed differences. The method of course has its value as a check.

3d. A laborious calculation may be made from an actual study of the route—determining the possible output of all factories, mines, etc., the amount of farm produce and of lumber that might be shipped, with an estimate of probable passenger traffic based on that of like towns similarly situated. This method is the best when it is properly done, but there is always the danger of leaving out sources of income—both existent and that to be developed by traffic facilities, or, on the other hand, of overestimating the value of expected traffic. In the

following tabular form are shown the population, gross receipts, receipts per head of population, mileage, earnings per mile of line operated, and mileage per 10,000 of population for the whole United States. It should be noted that the values are only *averages*, that individual variations are large, and that only a very rough dependence may be placed on them as applied to any particular case.

Year.	Population (estimated).	Gross receipts.	Receipts per head of popu- lation.	Mileage†	Earnings per mile of line operated.	Mileage per 10,000 popula- tion.‡
1888...	60,100,000	\$910,621,220	\$15.15	136,884	\$6653	24.94
1889...	61,450,000	964,816,129	15.81	153,385	6290	25.67
1890...	*62,801,571	1051,877,632	16.75	156,404	6725	26.05
1891...	64,150,000	1096,761,395	17.10	161,275	6801	26.28
1892...	65,500,000	1171,407,343	17.89	162,397	7213	26.19
1893...	68,850,000	1220,751,874	18.26	169,780	7190	26.40
1894...	68,200,000	1073,361,797	15.74	175,691	6109	26.20
1895...	69,550,000	1075,371,462	15.46	177,746	6050	25.97
1896...	70,900,000	1150,169,376	16.22	181,983	6320	25.78
1897...	72,350,000	1122,089,773	15.53	183,284	6122	25.53
1898...	73,600,000	1247,325,621	16.95	184,648	6755	25.32
1899...	74,950,000	1313,610,118	17.53	187,535	7005	25.25
1900...	*76,295,220	1487,044,814	19.49	192,556	7722	25.44
1901...	77,863,000	1588,526,037	20.47	195,562	8123	25.52
1902...	79,431,000	1726,380,267	21.88	200,155	8625	25.76
1903...	80,998,000	1900,846,907	23.70	205,314	9258	26.03
1904...	82,566,000	1975,174,091	24.23	212,243	9306	26.34
1905...	84,134,000	2082,482,406	25.15	216,974	9508	26.44
1906...	85,701,000	2325,765,167	27.65	222,340	10460	26.78
1907...	87,279,000	2589,105,578	29.63	227,455	11383	26.38
1908...	88,837,000	2393,805,989	26.95	231,540	10338	26.30
1909...	90,405,000	2418,677,538	26.71	234,800	10301	26.20
1910...	*91,972,266	2750,667,435	29.91	238,609	11528	26.14

* Actual.

† Excludes a small percentage not reporting "gross receipts."

‡ Actual mileage.

The probable growth in traffic, after the traffic has once attained its normal volume, is a small but almost certain quantity. In the above tabular form this is indicated by the gradual growth in "receipts per head of population" from 1897 to 1907. Then the sudden drop due to the panic of 1907 is clearly indicated, and also the gradual growth in the last few years. Even in England, where the population has been nearly stationary for many years, the growth though small is unmistakable. On the other hand the growth in some of the Western States

has been very large. For example, the gross earnings per head of population in the State of Iowa increased from \$1.42 in 1862 to \$10.00 in 1870, and to \$19.46 in 1884.

There will seldom be any justification in building to accommodate a larger business than what is "in sight." Even if it could be anticipated with certainty that a large increase in business would come in ten years, there are many reasons why it would be unwise to build on a scale larger than that required for the business to be immediately handled. Even though it may cost more in the future to provide the added accommodations (*e.g.* larger terminals, engine-houses, etc.), the extra expense will be nearly if not quite offset by the interest saved by avoiding the larger outlay for a period of years which may often prove much longer than was expected. A still more important reason is the avoidance of uselessly sinking money at a time when every cent may be needed to insure the success of the enterprise as a whole.

374. Probable number of trains per day. Increase with growth of traffic. The number of passenger trains per day cannot be determined by dividing the total number of passengers estimated to be carried per day by the capacity of the cars that can be hauled by one engine. There are many small railroads, running three or four passenger trains per day each way, which do not carry as many passengers all told as are carried on one heavy train of a trunk line. But because the bulk of the passenger traffic, especially on such light-traffic roads, is "unnecessary" traffic (see § 371) and must be encouraged and coaxed, the trains must be run much more frequently than mere capacity requires. The minimum number of passenger trains per day on even the lightest-traffic road should be two. These need not necessarily be passenger trains exclusively. They may be mixed trains.

The number required for freight service may be kept more nearly according to the actual tonnage to be moved. At least one local freight will be required, and this is apt to be considerably within the capacity of the engine. Some very light-traffic roads have little else than local freight to handle, and on such there is less chance of economical management. Roads with heavy traffic can load up each engine quite accurately according to its hauling capacity and the resulting economy is great. Fluctuations in traffic are readily allowed for by adding on or drop-

ping off one or more trains. Passenger trains must be run on regular schedule, full or empty. Freight trains are run by train-despatcher's orders. A few freight trains per day may be run on a nominal schedule, but all others will be run as extras. The criterion for an increase in the number of passenger trains is impossible to define by set rules. Since it should always come before it is absolutely demanded by the train capacity being overtaxed, it may be said in general terms that a train should be added when it is believed that the consequent increase in facilities will cause an increase in traffic the value of which will equal or exceed the added expense of the extra train.

375. Effect on traffic of an increase in facilities. The term facilities here includes everything which facilitates the transport of articles from the door of the producer to the door of the consumer. As pointed out before, in many cases of freight transport, the reduction of facilities below a certain point will mean the entire loss of such traffic owing to local inability to successfully compete with more favored localities. Sometimes owing to a lack of facilities a railroad company feels compelled to pay the cartage or to make a corresponding reduction on what would normally be the freight rate. In competitive freight business such a method of procedure is a virtual necessity in order to retain even a respectable share of the business. Even though the railroad has no direct competitor, it must if possible enable its customers to meet their competitors on even terms. In passenger business the effect of facilities is perhaps even more marked. The pleasure travel will be largely cut down if not destroyed.

376. Loss caused by inconvenient terminals and by stations far removed from business centers. This is but a special case of the subject discussed just in the preceding paragraph. The competition once existing between the West Shore and the New York Central was hopeless for the West Shore from the start. The possession of a terminal at the Grand Central Station gave the New York Central an advantage over the West Shore with its inconvenient terminal at Weehawken which could not be compensated by any obtainable advantage by the West Shore. This is especially true of the passenger business. The through freight business passing through or terminating at New York is handled so generally by means of floats that the disadvantage in this respect is not so great. The

enormous expenditure (roughly \$10,000,000) made by the Pennsylvania R. R., on the Broad Street Station (and its approaches) in Philadelphia, a large part of which was made in crossing the Schuylkill River and running to City Hall Square, rather than retain their terminal in West Philadelphia, is an illustration of the policy of a great road on such a question. The fact that the original plan and expenditure has been very largely increased since the first construction proves that the management has not only approved the original large outlay, but saw the wisdom of making a very large increase in the expenditure.

The construction of great terminals is comparatively infrequent and seldom concerns the majority of engineers. But an engineer has frequently to consider the question of the location of a way station with reference to the business center of the town. The following points may (or may not) have to be considered, and the real question consists in striking a proper balance between conflicting considerations.

(1) During the early history of a railroad enterprise it is especially needful to avoid or at least postpone all expenditures which are not demonstrably justifiable.

(2) The ideal place for a railroad station is a location immediately contiguous to the business center of the town. The location of the station even one fourth of a mile from this may result in a loss of business. Increase this distance to one mile and the loss is very serious. Increase it to five miles and the loss approaches 100%.

(3) The cost of the ideal location and the necessary right of way may be a very large sum of money for the new enterprise. On the other hand the increase in property values and in the general prosperity of the town, caused by the railroad itself, will so enhance the value of a more convenient location that its cost at some future time will generally be extravagant if not absolutely prohibitory. The original location is therefore under ordinary conditions a finality.

(4) To some extent the railroad will cause a movement of the business center toward it, especially in the establishment of new business, factories, etc., but the disadvantages caused to business already established is permanent.

(5) In any attempt to compute the loss resulting from a location at a given distance from the business center it must be

recognized that each problem is distinct in itself and that any change or growth in the business of the town changes the amount of this loss.

The argument for locating the station at some distance from the center of the town may be based on (a) the cost of right of way, thus involving the question of a large initial outlay, (b) the cost of very expensive construction (e.g. bridges), again involving a large initial outlay, (c) the avoidance of excessive grade into and out of the town. It sometimes happens that a railroad is following a line which would naturally cause it to pass at a considerable elevation above (rarely below) the town. In this case there is to be considered not only the possible greater initial cost, but the even more important increase in operating cost due to the introduction of a very heavy grade. To study such a case, compute the annual increase in operating expenses due to the additional grade, curvature, and distance; add to this the annual interest on the increased initial cost (if any) and compare this sum with the estimated annual loss due to the inconvenient location. The estimation of the increase in operating expenses is discussed in a subsequent chapter. The loss of business due to inconvenient location can only be guessed at. Wellington says that at a distance of one mile the loss would average 25%, with upper and lower limits of 10 and 40%, depending on the keenness of the competition and other modifying circumstances. For each additional mile reduce 25% of the preceding value. While such estimates are grossly approximate, yet with the aid of sound judgment they are better than nothing and may be used to check gross errors.

377. General principles which should govern the expenditure of money for railroad purposes. It will be shown later that the elimination of grade, curvature, and distance have a positive money value; that the reduction of ruling grade is of far greater value; that the creation of facilities for the handling of a large traffic is of the highest importance and yet the added cost of these improvements is sometimes a large percentage of the cost of *some road* over which it would be physically possible to run trains between the termini.

The subsequent chapters will be largely devoted to a discussion of the value of these details, but the general principles governing the expenditure of money for such purposes may be stated as follows:

1. No money should be spent (beyond the unavoidable minimum) unless it may be shown that the addition is in itself a profitable investment. The additional sum may not wreck the enterprise and it may add something to the value of the road, but unless it adds more than the improvement costs it is not justifiable.

2. If it may be positively demonstrated that an improvement will be more valuable to the road than its cost, it should certainly be made even if the required capital is obtained with difficulty. This is all the more necessary if the neglect to do so will permanently hamper the road with an operating disadvantage which will only grow worse as the traffic increases.

3. This last principle has two exceptions: (a) the cost of the improvement may wreck the whole enterprise and cause a total loss to the original investors. For, unless the original promoters can build the road and operate it until its stock has a market value and the road is beyond immediate danger of a receivership, they are apt to lose the most if not all of their investment; (b) an improvement which is very costly although unquestionably wise may often be postponed by means of a cheap temporary construction. Cases in point are found at many of the changes of alignment of the Pennsylvania R. R., the N. Y., N. H. & H. R. R., and many others. While some of the cases indicate faulty original construction, at many of the places the original construction was wise, considering the then scanty traffic, and now the improvement is wise considering the great traffic.

CHAPTER XX.

OPERATING EXPENSES.

378. **Distribution of gross revenue.** When a railroad comprises but one single property, owned and operated by itself, the distribution of the gross revenue is a comparatively simple matter. The operating expenses then absorb about two thirds of the gross revenue; the fixed charges (chiefly the interest on the bonds) require about 25 or 30% more, leaving perhaps 3 to 8% (more or less) available for dividends. The report on the Fitchburg R. R. for 1898 shows the following:

Operating expenses.....	\$5,083,571	69.1%
Fixed charges.....	1,567,640	21.3%
Available for dividends, surplus, or permanent improvements.....	708,259	9.6%
Total revenue.....	\$7,359,470	100.0%

But the financial statements of a large majority of the railroad corporations are by no means so simple. The great consolidations and reorganizations of recent years have been effected by an exceedingly complicated system of leases and sub-leases, purchases, "mergers," etc., whose forms are various. Railroads in their corporate capacity frequently own stocks and bonds of other corporations (railroad properties and otherwise) and receive, as part of their income, the dividends (or bond interest) from the investments.

In consequence of this complication, the U. S. Interstate Commerce Commission presents a "condensed income account" of which the following is a sample (1910):

Rail operations, operating revenues.....	\$2,750,667,435
" expenses.....	1,822,630,433
Net operating revenue.....	\$ 928,037,002
Net revenue from outside operations.....	2,225,455
Total net revenue.....	930,262,457
Taxes accrued.....	98,034,593
Operating income.....	832,227,864
Other income (chiefly dividends and interest on stocks and bonds owned).....	252,219,946
Gross corporate income.....	\$1,084,447,810

Deductions from gross corporate income (chiefly interest on funded debt, rents for lease of other roads and hire of equipment).....	567,853,088
Net corporate income.....	\$ 516,594,722
Dividends.....	\$283,411,828
Appropriations for additions, betterments, new lines, and extensions .	55,061,675
Appropriations for other reserves....	2,640,893
Total.....	\$341,114,396
Balance to credit of profit and loss.....	175,480,326

In the above account an item of *income* (e.g., lease of road) reported by one road will be reported as a "deduction from income" by the road which leases the other.

The above statement may be reduced to an income account of all the railways *considered as one system*. We then have

Operating revenues.....	\$2,750,667,435
Clear income from investments.....	78,442,027
	2,829,109,462
Operating expenses.....	\$1,822,630,433
Salaries and maintenance of leased lines.....	332,242
Total.....	1,822,962,675
Net revenues and income.....	1,006,146,787
Net interest on funded debt.....	370,092,222
Other interest.....	16,520,342
Taxes.....	103,795,701
Total.....	490,408,265
Available for dividends, adjustments and improvements.....	515,738,522
Net dividends.....	293,836,863
Available for adjustments and improvements.....	221,901,659

Of the balance "available for adjustments and improvements," part was spent in permanent improvements, part was advanced to cover deficits in the operation of weak lines, and more than half was left as "surplus," *i.e.*, working capital.

The percentages of the gross revenue which are devoted to operating expenses, fixed charges, and dividends are not *necessarily* an indication of creditable management or the reverse. Causes utterly beyond the control of the management, such as the local price of coal, may abnormally increase certain

items of expense, while ruinous competition may cut down the gross revenue so that little or nothing is left for dividends. A favorable location will sometimes make a road prosperous in spite of bad management. On the other hand, the highest grade of skill will fail to keep some roads out of the hands of a receiver.

379. Fivefold distribution of operating expenses. The distribution of operating expenses here used is copied from the method of the Interstate Commerce Commission. The aim is to divide the expenses into groups which are as mutually independent and distinct as possible—although, as will be seen later, a change in one item of expense will variously affect other items.

Operating expenses.	1908.	1909.	1910.
Maintenance of way and structures.....	19.73%	19.29%	20.22%
Maintenance of equipment.....	22.06	22.75	22.66
Traffic expenses.....	2.89	3.08	3.07
Transportation expenses.....	52.01	50.90	50.29
General expenses.....	3.13	3.98	3.76
	100.00	100.00	100.00

The above percentages represent the averages given by the reports for the three years from 1908 to 1910 inclusive.

380. Operating expenses per train-mile. The reports of the U. S. Interstate Commerce Commission give the average cost per train-mile for every railroad in the United States. Although there are wide variations in these values, it is remarkable that the very large majority of roads give values which agree to within a small range, and that within this range are found not only the great trunk lines with their enormous train mileage, but also roads with very light traffic.

In the following tabular form is shown a statement taken from the reports for 1904 and for 1910 of ten of the longest railroads in the United States and, in comparison with them, a corresponding statement regarding ten more roads selected *at random*, except in the respect that each had a mileage less of than 100 miles. Although the extreme variations are greater, yet there is no very marked difference in the general values for operating expenses per train-mile, or in the ratio of expenses to earnings. The averages for the ten long roads agree fairly well with the averages

for the whole country, but there would be no trouble (as is shown by some of the individual cases) in finding another group of ten short roads giving either greater or less average values than those given. And yet the tendency to uniform values, regardless of the mileage, is very striking. For comparison the figures for 1910 have been merely added to the figures for 1904, given in previous editions.

OPERATING EXPENSES PER TRAIN-MILE ON LARGE AND SMALL
ROADS (1904 AND 1910).

	Mileage.		Operating expenses per train-mile.		Ratio expenses to earnings per cent.	
	1904.	1910.	1904.	1910.	1904.	1910.
Whole United States.	220,112	240,439	1.314	1.489	67.79	66.29
Canadian Pacific.	8,332	10,271	1.320	1.504	68.72	65.41
C., B. & Q.	8,326	9,040	1.313	1.710	64.35	71.71
Chicago & Northwestern. . .	7,412	7,629	1.136	1.306	66.61	70.31
Southern Railway.	7,197	7,050	1.048	1.234	70.30	67.43
C., R. I. & P.	6,761	7,396	1.199	1.344	72.90	73.07
Northern Pacific.	5,619	6,189	1.392	1.824	52.26	61.71
A., T. & S. F.	5,031	7,460	1.305	1.626	60.05	64.33
Great Northern.	4,489	7,147	1.464	1.808	49.72	60.53
Illinois Central.	4,374	4,551	1.107	1.409	70.02	74.84
Atlantic Coast Line.	4,229	4,491	0.984	1.213	58.95	62.44
Average of ten.			1.227	1.498	63.39	67.18
Montpelier & Wells River. . .	44	50	1.169	1.430	80.73	75.08
Somerset Railway Co.*. . . .	42	94	0.802	1.314	59.37	76.65
Huntingdon & Broadtop Mountain.	66	70	0.950	2.052	52.10	96.40
Lehigh & New England. . . .	96	170	0.793	2.045	69.80	62.84
Ligonier Valley.	11	16	1.427	1.480	69.33	49.15
Newburgh, Dutchess & Connecticut †.	59		0.922		85.09	
Susquehanna & New York. . .	55	80	1.368	1.028	78.47	77.81
Detroit & Charlevoix.	51	51	1.424	1.010	67.52	99.53
Harriman & Northeastern *. .	20	20	2.162	1.733	79.26	63.70
Galveston, Houston & Henderson.	50	50	1.556	1.759	47.27	70.37
Average of ten (or nine). . .			1.257	1.539	68.89	74.61

* Subsidiary road since 1904.

† Merged since 1904; separate figures not available.

The fluctuations of the average cost per train-mile for several years past may be noted from the following tabular form:

Year.	Average cost per train-mile in cents.	Year.	Average cost per train-mile in cents.	Year.	Average cost per train-mile in cents.
1890	96.006	1897	92.918	1904	131.375
1891	95.707	1898	95.635	1905	132.140
1892	96.580	1899	98.390	1906	137.060
1893	97.272	1900	107.288	1907	146.993
1894	93.478	1901	112.292	1908	147.340
1895	91.829	1902	117.960	1909	143.370
1896	93.838	1903	126.604	1910	148.865

The enforced economies after the panic of 1893 are well shown. The reduction generally took the form of a lowering of the standards of maintenance of way and of maintenance of equipment. The marked advance since 1895 is partly due to the necessity for restoring the roads to proper condition, replenishing worn-out equipment and providing additional equipment to handle the greatly increased volume of business. The recent advance is chiefly due to the increase in wages and the generally increased cost of supplies.

In looking over the list, it may be noted that the cases where the operating expenses per train-mile and the ratio of expenses to earnings vary very greatly from the average are almost invariably those of the very small roads or of "junction roads" where the operating conditions are abnormal. For example, one little road, with a total length of 13 miles and total annual operating expenses of \$5342, spent but 22½c. per train-mile, which precisely exhausted its earnings. This precise equality of earnings and expenses suggests jugglery in the bookkeeping. As another abnormal case, a road 44 miles long spent \$3.81 per train-mile, which was nearly *fourteen* times its earnings. In another case a road 13 miles long earned \$7.76 per train-mile and spent \$6.03 (78%) on operating expenses, but the fixed charges were abnormal and the earnings were less than half the sum of the operating expenses and fixed charges. The *normal* case, even for the small road, is that the cost per train-mile and the ratio of operating expenses to earnings will agree fairly well with the average, and when there is a marked difference it is generally due to some abnormal conditions of expenses or of earning capacity.

381. **Reasons for uniformity in expenses per train-mile.** The chief reason is that, although on the heavy-traffic road everything is kept up on a finer scale, better roadbed, heavier rails, better rolling stock, more employees, better buildings,

stations, and terminals, etc., yet the number of trains is so much greater that the divisor is just enough larger to make the average cost about constant. This is but a general statement of a fact which will be discussed in detail under the different items of expense.

382. Detailed classification of expenses with ratios to the total expense. The Interstate Commerce Commission now publishes each year a classification with detailed summation for the cost of each item. These summations are made up from reports furnished by railroads which have (in the reports recently made) represented over 99% of the total traffic handled. In the annexed tabular form (Table XX) are shown the percentages which each item bears to the total. The railroads have been divided into two classes, "large" and "small," as indicated below. Large roads report on 116 items which are combined and condensed with 44 items for small roads.

"Large roads" are those with mileage greater than 250 miles, or those with operating revenues greater than \$1,000,000. Roads subsidiary to "large roads" are also included in this class.

"Small roads" are those with mileage less than 250 miles and also with operating revenues less than \$1,000,000.

383. Amounts and percentages of the various items. The I. C. C. report for the year ending June 30, 1909, was the first to include the distribution of expenses according to the present classification. The items as given are reliable and may be utilized, as far as any such computations are to be depended on, in estimating future expenses. The chief purpose of this discussion is to point out those elements of the cost of operating trains which may be affected by such changes of location as an engineer is able to make. There are some items of expense with which the engineer has not the slightest concern, nor will they be altered by any change in alignment or constructive detail which he may make. In the following discussion such items will be passed over with a brief discussion of the sub-items included.

MAINTENANCE OF WAY AND STRUCTURES.

384. Items 2 to 5. Track material. The relative cost of ballast, ties, rails and other track material, as shown by comparing either the gross amounts or the percentages in Table IX,

is suggestive and instructive. The fact that ties cost considerably more than all other track material combined shows the importance of any possible saving in tie renewals. It is also significant that the relative importance of ties has increased in the last few years, and that the relative increase has not been due to a reduction in the cost of other track material. Apparently the lengthening of the average life of ties, due to preservative processes, the use of tie-plates, and greater care to avoid the premature withdrawal from the track of ties which are still serviceable, has not kept pace with the increase in the average cost per tie. The cost of Bessemer rails per ton has remained almost constant for several years, but the adoption of heavier rails by many roads, necessitated by heavier rolling stock, and also the adoption of open-hearth steel costing about \$2 per ton more for much of the tonnage, have been potent causes in increasing the cost of maintenance.

385. Item 6. Roadway and track. This item is three-eighths of the total cost of maintenance of way and structures. It consists chiefly of the wages of trackmen. There has been an almost steady increase in the daily wages of section foremen and other trackmen since 1900, as shown below:

	1900	1901	1902	1903	1904	1905	1906	1907	1908	1909	1910
Section foremen..	1.68	1.71	1.72	1.78	1.78	1.79	1.80	1.90	1.95	1.96	1.99
Other trackmen..	1.22	1.23	1.25	1.31	1.33	1.32	1.36	1.46	1.45	1.38	1.47
No. of trackmen per 100 miles...	118	122	140	147	136	143	155	162	130	136	157

The average number of section foremen per 100 miles of line has remained almost constant at 18. Although there have been fluctuations in the number of "other trackmen" required per 100 miles of line, there has been in general a very substantial increase. These two causes combined (increased number and increased wages) have had a great influence in producing the regular and steady increase in the average cost of a train-mile, as shown in § 380. The variations in average daily wages in the various Groups for 1910 are shown herewith:

AVERAGE WAGES OF TRACKMEN IN THE SEVERAL GROUPS.

	I.	II.	III.	IV.	V.	VI.	VII.	VIII.	IX.	X.	U.S.
Section foremen..	2.38	2.08	1.99	1.77	1.83	1.90	2.22	1.83	1.88	2.51	1.99
Other trackmen..	1.65	1.52	1.57	1.21	1.13	1.56	1.60	1.37	1.26	1.52	1.47

The classification of expenses for "small roads" combine items 2 to 7. It should be noted that this item (see Table XX A) is over three-fourths of the total for that division, and also that the cost of maintenance of way and structures for "small roads" is nearly 26% of the total operating expenses and that it is about 20% for "large roads." In 1908 and 1909, this difference was even more marked, and the fact should not be neglected when considering the economics of a project which will evidently be, at least for many years, a "small road."

386. Items 8 to 15. Maintenance of track structures. As a matter of economics, the locating engineer has little or no concern with the cost of maintaining track structures. If he is comparing two proposed routes it would be seldom that they would be so different that he would be justified in attempting to compute a train-mile difference in cost of operation, based on differences in these items. Of course, one proposed line might call for one or more tunnels which the alternate line might not have, and the annual cost of maintaining the tunnels would increase the cost of operation. Such a case would justify special consideration. So far as the maintenance of small bridges and culverts are concerned it would usually be sufficiently accurate to consider that a proposed change of line, involving perhaps several miles of road, would require substantially the same number of bridges and culverts, and therefore that the cost of maintaining them would be the same by either line. The error involved in such an assumption would usually be insignificant, unless there was a very large and material difference in the two lines in this respect. Under such conditions special computations should be made. The items total less than 3% for small roads and still less for large roads.

387. Items 16 to 21. The other items of maintenance of way and structures are very small, except No. 16, and under ordinary conditions will not be affected by any changes which the locating engineer may make, except as noted in the chapter on Distance.

MAINTENANCE OF EQUIPMENT.

388. Item 24. Superintendence of Equipment. The item averages about two-thirds of one per cent and has so little fluctuation under ordinary conditions that it may almost be considered as a fixed charge. It includes those fixed charges in

superintendence which do not fluctuate with small variations in business done. It includes the salaries of superintendent of motive power, master mechanic, master car-builder, foreman, etc., but does not include that of road foreman of engines nor enginemen. Although the item will vary with the general scale of business on the road, it does not fluctuate with it and hence will not usually be influenced by any small changes in alinement which the engineer may be considering.

389. Items 25 to 27. Repairs, renewals, and depreciation of steam locomotives. This subject will be treated at greater length in Chapter XXI, Distance. The item is of interest to the locating engineer because he must appreciate the effect on locomotive repairs and renewals of an addition to distance. A large part of the repairs of locomotives are due to the wear of wheels, which is largely caused by curvature. Therefore the value of any reduction of curvature is a matter of importance, and this will be considered in Chapter XXII. A considerable portion of the deterioration of a locomotive is due to grade, and the economic advantages of reductions of grade will be considered in Chapter XXIII.

This item includes the expenses of work whose effect is supposed to last for an indefinite period. It does *not* include the expense of cleaning out boilers, packing cylinders, etc., which occurs regularly and which is charged to items 72 or 81. It does include all current repairs, general overhauling, and even the replacement of old and worn-out locomotives by new ones to the extent of keeping up the original standard and number. Of course additions beyond this should be considered as so much increase in the original capital investment. As a locomotive becomes older the *annual* repair charge becomes a larger percentage on the first cost, and it may become as much as one-fourth and even one-third of the first cost. When a locomotive is in this condition it is usually consigned to the scrap-pile; the annual cost for maintenance becomes too large an item for its annual mileage. The effect on expenses of increasing the weight of engines is too complicated a problem to be solved accurately, but certain elements of it may be readily computed. While the cost of repairs is greater for the heavier engines, the increase is only about one-half as fast as the increase in weight—some of the sub-items not being increased at all.

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390. Items 28 to 30. Repairs, renewals, and depreciation of electric locomotives. The use of electric locomotives as an adjunct to steam railroad service applies to only a small percentage of the roads of the country at present, but the general principles stated in the previous article will apply to these items.

391. Items 31 to 36. Repairs, renewals, and depreciation of passenger-train cars and of freight-cars. Many of the economic features of car construction, especially the effect of modern improvements, such as friction draft-gear, automatic couplers, and air-brakes, have been considered in Chapter XV. Such figures will be utilized in considering the effect on car repairs of additional distance, of variations in curvature, and of grade as discussed later. Although the published figures for these items since 1908 are on a slightly different basis from those published previously, it is evident that these items with respect to passenger-cars have remained fairly constant, while those for freight-cars have substantially increased, not only in gross amount but even in percentage to the total. The *fluctuations* of these items are largely due to accidents rather than to any line of policy under the control of the engineer. It should be understood that for these items, as well as the previous item, the renewal of rolling stock generally means the construction of higher-grade locomotives and higher-grade cars. When this is done the difference in cost between the higher-grade, and probably more expensive, construction and the former cheap construction should be considered as an addition to the capital account rather than a charge against operating expenses. The enormous increase in the movement of freight during the last few years has required a corresponding increase in freight-car equipment. Some roads have been charging up the full cost of the renewing of their old-fashioned light-weight equipment with more expensive equipment under the head of operating expenses, and it is quite possible that a portion of the increase in these items is due to this policy.

392. Items 37 to 52. Electric equipment; floating equipment; work equipment; shop machinery and tools; power-plant equipment; miscellaneous items. The location of the road along the line has no connection with the maintenance of marine equipment. The maintenance of shop machinery and tools can only be affected as the work of repairs of rolling stock fluctuates, and of course in a much smaller ratio. No change which an

engineer can effect will have any appreciable influence on this item.

The other items are too small and have too little connection with location to be here discussed, except as it may be considered that they vary with train-mileage, which an engineer may influence (see Chapter XXIII, Grades).

TRAFFIC.

393. Items 53 to 60. These items have exclusive reference to the work of securing business for the road and have no necessary relation to any questions which the locating engineer must answer.

TRANSPORTATION.

394. Items 61 to 70. These items cover the superintendence of transportation and many of the yard and station expenses.

They require over one-eighth of the entire operating expenses. Although some of them might be somewhat affected by the details of the design of a yard, they are practically unaffected by any changes of alinement which an engineer may make.

395. Items 71 to 76. Yard-engine expenses. By comparing these items with the corresponding items (80 to 85) for road engines, it may be seen that the total expenses assignable to yard engines are about 20% of those road of engines; the relative fuel charge for 1910 was 15.3%. The number of switching locomotives in the United States in 1910 was 9115 or 15.4% of the total number, 58,947. The relative charge for wages of engine-men was 25.7%. This higher proportionate charge is probably due to the fact that the wages for yard engine-men must necessarily be on a per diem basis, but the wages of road engine-men are generally on a mileage basis, as explained later. On the other hand, the mileage of a yard engine is usually comparatively low, and the coal consumed will be correspondingly, although not proportionately, low. It must also be remembered that these figures are exclusive of the work and equipment of switching and terminal companies. †

396. Item 80. Road engine-men. This item requires 6% of the total operating expenses. The engine-men are usually paid on a mileage basis, or by the trip, except on very small railroads.

On very short roads, where a train crew may make two, three, or even four complete round trips per day, they may readily be paid by the day, so many round trips being considered as a day's work, but on roads of great length, where all trains, and especially freight-trains, are run day and night, weekday and Sunday, all trainmen are necessarily paid by the trip or, as it is more usually expressed, by the "run." It is generally found convenient to divide the road into "divisions" which are approximately 100 miles in length. According as the division is greater or less than 100 miles, it is designated as a $1\frac{1}{8}$ run, $1\frac{1}{4}$ run, or perhaps $\frac{7}{8}$ run. The enginemen will then be paid according to the number of runs made per month. There is a considerable fluctuation in the average wages paid in different sections of the United States, as is shown by the tabular form given below, in which the "groups" refer to the sections into which the country has been divided by the Interstate Commerce Commission. This tabular form may be of some assistance in showing the average daily compensation which must be allowed for in the various sections of the country:

AVERAGE DAILY COMPENSATION BY GROUPS—1910.

	I.	II.	III.	IV.	V.	VI.	VII.	VIII.	IX.	X.	U.S.
	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$
Enginemen.....	3.97	4.53	4.34	4.34	4.81	4.58	4.72	4.75	4.87	4.84	4.55
Firemen.....	2.35	2.75	2.57	2.16	2.42	2.84	3.14	3.13	3.03	3.04	2.74

The increase in the average wages paid to enginemen and firemen during eleven years is plainly shown by the following figures:

INCREASE IN DAILY WAGES, FROM 1900 TO 1910.

	1900	1901	1902	1903	1904	1905	1906	1907	1908	1909	1910
	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$
Enginemen.....	3.75	3.78	3.84	4.01	4.10	4.12	4.12	4.30	4.45	4.44	4.55
Firemen.....	2.14	2.16	2.20	2.28	2.35	2.38	2.42	2.54	2.64	2.67	2.74

The fluctuations of this item are of importance to the locating engineer in discussing the economics of differences in distance. This feature will be more fully discussed in Chapter XXI.

397. Item 82. Fuel for road locomotives. This item includes every subitem of the entire cost of the fuel until it is placed in the engine-tender. The cost therefore includes not only the first cost at the point of delivery to the road, but also the expense of hauling it over the road from the point of delivery to the various coaling-stations and the cost of operating the coal-pockets from which it is loaded on to the tenders. Even though the cost may be fairly regular for any one road, the cost for different roads is exceedingly variable. There has been an almost steady increase in the percentage of the cost of this item per train-mile since 1897. Items 73 and 82 amounted to nearly 12% of the total operating expenses in 1910, and required an actual expenditure of nearly \$213,000,000. It is the largest item in the whole cost of railroad operation. Although some roads, which traverse coal-regions and perhaps actually own the coal-mines, are able to obtain their coal for a cost which may be charged up as \$1 per ton or less, there are many roads which are far removed from coal-fields which have to pay \$3 or \$4 per ton, on account of the excessive distance over which the coal must be hauled. Unfortunately the figures published by the Interstate Commerce Commission do not show the variations in the percentage of this item in the different groups. In the succeeding chapters will be shown the effect on fuel consumption of the several variations on location details. The great importance of this item requires that it shall be thoroughly understood and studied by the engineer. It will be shown, contrary to the commonly received opinion, that the fuel consumption is quite largely independent of distance and even of the number of cars hauled.

398. Items 83 to 95. Water, lubricants, and other supplies for road locomotives. These items aggregate about 1.3% of the operating expenses. There seems to be slight tendency for the percentage to increase. Since the consumption of all these supplies will vary nearly as the engine mileage, the engineer is concerned with them directly to the extent to which he may change the engine mileage.

399. Item 88. Road trainmen. This item includes the wages of conductors and "other trainmen." As in the case of all other employees, the average daily wages have advanced since 1900 as shown below:

**AVERAGE DAILY WAGES OF CONDUCTORS AND OTHER TRAINMEN,
1900 TO 1910.**

	1900	1901	1902	1903	1904	1905	1906	1907	1908	1909	1910
	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$
Conductors.....	3.17	3.17	3.21	3.38	3.50	3.50	3.51	3.69	3.81	3.81	3.91
Other trainmen...	1.96	2.00	2.04	2.17	2.27	2.31	2.35	2.54	2.60	2.59	2.69

The following form shows the variation in the daily wages of conductors and trainmen in the several groups for 1910:

AVERAGE DAILY WAGES IN 1910 IN THE SEVERAL GROUPS.

	I.	II.	III.	IV.	V.	VI.	VII.	VIII.	IX.	X.	U.S.
	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$	\$
Conductors.....	3.52	3.74	3.73	3.38	3.79	4.12	4.16	4.38	4.59	4.38	3.91
Other trainmen...	2.49	2.71	2.72	1.89	2.18	2.81	2.86	2.84	3.00	3.04	2.69

These figures are of vital importance from an economic standpoint, since they show a constant tendency to increase and thereby raise the average cost of a train mile. And as there is no present indication of any limit to this increase, all economic calculations which attempt to predict future expenses, even for a few years in advance, must allow for these and other increased expenses.

400. Item 89. Train supplies and expenses. These items, which average about 1.8%, include the large list of consumable supplies such as lubricating oil, illuminating-oil or gas, ice, fuel for heating, cleaning materials, etc., which are used on the cars and not on the locomotives. The consumption of some of these articles is chiefly a matter of time. In other cases it is a function of mileage. The effect of changes which an engineer may make on this item will be considered when estimating the effect of the changes.

401. Items 93, 99 to 103. Clearing wrecks, loss, damage and injuries to persons and property. These expenses are fortuitous and bear no absolute relation either to the number of miles of road or the number of train-miles. While they depend largely on the standards of discipline on the road, even the best of roads have to pay some small proportion of their earnings to these

items. While we might expect that a road with heavy traffic would have a larger proportion of train accidents than a road of light traffic, it is usually true that on the heavy-traffic roads the precautions taken are such that they are usually freer from accidents than the light-traffic roads. During recent years there has been a very perceptible increase in the percentages of these items, particularly in the compensations paid for "injuries to persons." The increase in this item coincides with the increase already noted in the number of passengers killed during recent years. The possible relation between curvature and accidents has already been discussed, but otherwise the locating engineer has no concern with these items.

402. Items 104, 105. Operating joint tracks and facilities, Dr. and Cr. A large part of these debit and credit charges are those for car per diem and mileage charges. This is a charge paid by one road to another for the use of cars, which are chiefly freight-cars. To save the rehandling of freight at junctions, the policy of running freight-cars from one road to another is very extensively adopted. Since the foreign road receives its mileage proportion of the freight charge, it justly pays to the road owning the car at a rate which is supposed to represent the value of the use of the freight-car for the number of miles traveled. The foreign road then loads up the freight-car with freight consigned to some point on the home road and sends it back, paying mileage for the distance traveled on the foreign road, a proportional freight charge having been received for that service. All of these movements of freight-cars are reported to a car association, which, by a clearing-house arrangement, settles the debit and credit accounts of the various roads with each other. Such is the simple theory. In practice the cars are not sent back to the home road at once, but wander off according to the local demand. As long as a strict account is kept of the movements of every car, and as long as the home road is paid the charge which really covers the value of lost service, no harm is done to the home road, except that sometimes, when business has suddenly increased, the home road cannot get enough cars to handle its own business. The value of the car is then abnormally above its ordinary value, and the home road suffers for lack of the rolling stock which belongs to it. Formerly such charges were paid strictly according to the mileage. This developed the intolerable condition that loaded cars would be

run onto a siding and left there for several days, simply because it was not convenient to the consignee to unload the car immediately. On the mileage basis the car would be earning nothing, and, since the road on which the car then was had no particular interest in the car, the car was allowed to stand to suit the convenience of the consignee. To correct this evil a system of per diem charges has been developed, so that a railroad has to pay a per diem charge for every foreign car on its lines. To reduce this charge as much as possible the railroads compel consignees, under penalty of heavy demurrage charges, to unload cars promptly. The running of freight-cars on foreign lines is now settled almost exclusively on the per diem basis, but the earning of passenger-cars over other lines, as is done on account of the advantages of through-car service, as well as the running of Pullmans and other special cars, is still paid for on the mileage basis. To the extent to which this charge is settled on the mileage basis, any change in distance which the engineer may be able to effect in the length of the road will have its influence on this item, but when the freight-car business, which comprises by far the larger part of the running of cars over foreign lines, is settled on the per diem basis no changes in alinement which the engineer may make will affect the item appreciably.

Switching Charges. Where two or more railroads intersect there will be a considerable amount of shifting of cars, chiefly freight-cars, from one road to the other. This shifting at any one junction may be done entirely by the engines of one road or perhaps by those of both roads. A portion of the expense of this work is charged up against the other road by the road which does the work. The total amount of this work is carefully accounted for by a clearing-house arrangement, and the balance is charged up against the road which has done the least work. The item is very small, is fairly uniform year by year, and is seldom, if ever, affected by changes of alinement.

Other Items. All of the remaining items, as stated in Table IX, are of no concern to the locating engineer. They are either general expenses, such as the salaries of general officers, insurance or law expenses, or are special items, such as advertising or the operation of marine equipment which will not be changed by any variations in distance, curvature, or grades which a locating engineer may make. There is therefore no need for their further discussion here.

TABLE XX.—SUMMARY SHOWING CLASSIFICATION OF OPERATING EXPENSES FOR THE YEAR ENDING JUNE 30, 1910, AND PROPORTION OF EACH CLASS TO THE TOTAL.—LARGE ROADS.

	Operating expenses.	Amount.	Per cent
MAINTENANCE OF WAY AND STRUCTURES.			
1	Superintendence of way and structures.....	\$17,058,473	0.957
2	Ballast.....	8,861,334	0.497
3	Ties.....	55,259,585	3.099
4	Rails.....	16,435,349	0.922
5	Other track material.....	20,225,436	1.134
6	Roadway and track.....	134,275,440	7.531
7	Removal of snow, sand, and ice.....	8,297,715	0.465
8	Tunnels.....	1,147,140	0.064
9	Bridges, trestles, and culverts.....	30,471,313	1.709
10	Over and under grade crossings.....	1,086,702	0.061
11	Grade crossings, fences, cattle guards, and signs.	6,105,192	0.343
12	Snow and sand fences and snowsheds.....	385,730	0.022
13	Signals and interlocking plants.....	8,175,641	0.459
14	Telegraph and telephone lines.....	3,407,627	0.191
15	Electric power transmission.....	367,400	0.021
16	Buildings, fixtures, and grounds.....	32,181,561	1.805
17	Docks and wharves.....	3,537,331	0.198
18	Roadway tools and supplies.....	5,141,983	0.288
19	Injuries to persons.....	1,886,865	0.106
20	Stationery and printing.....	714,637	0.040
21	Other expenses.....	333,056	0.018
22	Maintn'g joint tracks, yds., and other facilities, Dr.	12,151,280	0.681
23	Maintn'g joint tracks, yds., and other facilities, Cr.	9,241,467	0.518
	Total maintenance of way and structures....	\$358,265,293	20.093
MAINTENANCE OF EQUIPMENT.			
24	Superintendence of equipment.....	\$11,457,025	0.643
25	Steam locomotives—repairs.....	138,548,039	7.770
26	Steam locomotives—renewals.....	3,272,370	0.184
27	Steam locomotives—depreciation.....	11,755,130	0.659
28	Electric locomotives—repairs.....	217,182	0.012
29	Electric locomotives—renewals.....		
30	Electric locomotives—depreciation.....	22,341	0.001
31	Passenger-train cars—repairs.....	30,695,229	1.721
32	Passenger-train cars—renewals.....	1,608,124	0.090
33	Passenger-train cars—depreciation.....	5,827,607	0.327
34	Freight-train cars—repairs.....	137,846,373	7.731
35	Freight-train cars—renewals.....	12,423,890	0.697
36	Freight-train cars—depreciation.....	30,712,546	1.722
37	Electric equipment of cars—repairs.....	161,624	0.009
38	Electric equipment of cars—renewals.....	27,000	0.002
39	Electric equipment of cars—depreciation.....	42,996	0.003
40	Floating equipment—repairs.....	924,976	0.052
41	Floating equipment—renewals.....	48,144	0.003
42	Floating equipment—depreciation.....	381,129	0.021
43	Work equipment—repairs.....	4,461,503	0.250
44	Work equipment—renewals.....	754,536	0.042
45	Work equipment—depreciation.....	899,128	0.050
46	Shop machinery and tools.....	9,439,056	0.529
47	Power plant equipment.....	167,182	0.010
48	Injuries to persons.....	1,370,388	0.077
49	Stationery and printing.....	978,744	0.055
50	Other expenses.....	841,019	0.047
51	Maintaining joint equipment at terminals Dr...	1,392,411	0.078
52	Maintaining joint equipment at terminals, Cr...	840,895	0.047
	Total maintenance of equipment.....	\$405,434,797	22.738

TABLE XX.—(Continued.)

	Operating expenses.	Amount.	Per cent
TRAFFIC.			
53	Superintendence of traffic.....	\$13,960,333	0.783
54	Outside agencies.....	19,592,049	1.099
55	Advertising.....	8,347,914	0.468
56	Traffic associations.....	1,519,215	0.085
57	Fast freight lines.....	3,984,644	0.224
58	Industrial and immigration bureaus.....	931,212	0.052
59	Stationery and printing.....	6,476,861	0.363
60	Other expenses.....	120,052	0.007
	Total traffic expenses.....	\$54,932,280	3.081
TRANSPORTATION.			
61	Superintendence of transportation.....	21,365,630	1.198
62	Dispatching trains.....	16,250,243	0.911
63	Station employees.....	123,062,288	6.902
64	Weighing and car-service associations.....	2,389,673	0.134
65	Coal and ore docks.....	2,542,882	0.143
66	Station supplies and expenses..	10,302,118	0.578
67	Yardmasters and their clerks.....	14,563,707	0.817
68	Yard conductors and brakemen.....	48,211,995	2.704
69	Yard switch and signal tenders.....	3,888,708	0.218
70	Yard supplies and expenses.....	1,249,844	0.070
71	Yard enginemen.....	27,890,126	1.564
72	Enginehouse expenses—yard.....	8,106,787	0.454
73	Fuel for yard locomotives.....	28,306,698	1.587
74	Water for yard locomotives.....	1,754,359	0.098
75	Lubricants for yard locomotives.....	576,693	0.032
76	Other supplies for yard locomotives.....	644,336	0.036
77	Operating joint yards and terminals—Dr.....	20,875,654	1.171
78	Operating joint yards and terminals—Cr.....	12,998,087	0.727
79	Motormen.....	477,404	0.027
80	Road enginemen.....	108,471,661	6.083
81	Enginehouse expenses—road.....	30,580,907	1.715
82	Fuel for road locomotives.....	184,588,622	10.352
83	Water for road locomotives.....	11,624,756	0.652
84	Lubricants for road locomotives.....	3,573,631	0.200
85	Other supplies for road locomotives.....	3,708,286	0.208
86	Operating power plants.....	726,742	0.041
87	Purchased power.....	413,783	0.023
88	Road trainmen.....	114,122,534	6.400
89	Train supplies and expenses.....	31,795,761	1.783
90	Interlockers & block & other signals—operation.....	8,718,693	0.490
91	Crossing flagmen and gatemen.....	6,319,183	0.354
92	Drawbridge operation.....	897,495	0.050
93	Clearing wrecks.....	4,488,082	0.252
94	Telegraph and telephone—operation.....	5,820,358	0.326
95	Operating floating equipment.....	2,876,653	0.161
96	Express service.....	597	*
97	Stationery and printing.....	8,113,694	0.455
98	Other expenses.....	2,060,571	0.117
99	Loss and damage—freight.....	21,756,671	1.220
100	Loss and damage—baggage.....	360,244	0.020
101	Damage to property.....	4,791,324	0.269
102	Damage to stock on right of way.....	3,650,544	0.206
103	Injuries to persons.....	20,139,285	1.129
104	Operating joint tracks and facilities—Dr.....	4,625,115	0.259
105	Operating joint track and facilities—Cr.....	4,857,256	0.244
	Total transportation expenses.....	\$899,328,994	50.438

* Less than 0.0001 per cent.

TABLE XX.—(Continued.)

	Operating expenses.	Amount.	Per cent
	GENERAL.		
106	Salaries and expenses of general officers. . . .	\$9,206,835	0.516
107	Salaries and expenses of clerks and attendants	25,167,569	1.411
108	General office supplies and expenses.	3,295,407	0.185
109	Law expenses.	10,845,738	0.608
110	Insurance.	7,551,789	0.424
111	Relief department expenses.	680,843	0.038
112	Pensions.	2,007,818	0.113
113	Stationery and printing.	2,853,808	0.160
114	Other expenses.	3,006,289	0.169
115	General administration, joint tracks, yards, and terminals—Dr.	655,875	0.037
116	General administration, joint tracks, yards, and terminals—Cr.	192,378	0.011
	Total general expenses.	\$65,079,593	3.650
	Recapitulation of expenses:		
	I. Maintenance of way and structures. . .	\$358,265,293	20.093
	II. Maintenance of equipment.	405,434,797	22.738
	III. Traffic expenses.	54,932,280	3.081
	IV. Transportation expenses.	899,328,994	50.438
	V. General expenses.	65,079,593	3.650
	Total operating expenses.	\$1,783,040,957	100.000

The "Credit items" 23, 52, 78, 105, and 116 represent receipts from other roads, from switching and terminal companies (which are not included in this or the following table) or from other sources. The amounts and percentages must be deducted from the other expenses to obtain the net expenses.

TABLE XXa.—SUMMARY SHOWING CLASSIFICATION OF OPERATING EXPENSES FOR THE YEAR ENDING JUNE 30, 1910, AND PROPORTION OF EACH CLASS TO THE TOTAL.—SMALL ROADS.

	Operating expenses.	Per cent	Corresponding items in Table XX.
	I. MAINTENANCE OF WAY AND STRUCTURES		
1	Superintendence of way and structures.	1.195	1
2	Maintenance of roadway and track.	19.752	2-7
3	Maintenance of track structures.	2.991	8-15
4	Maintenance of buildings, docks, and wharves.	1.191	16, 17
5	Injuries to persons.	0.042	19
6	Other maintenance of way & structures expenses	0.860	18, 20, 21
7	Maintaining joint tracks, yards, and other facilities—Dr.	0.469	22
8	Maintaining joint tracks, yards, and other facilities—Cr.	0.668	23
	Total, maintenance of way and structures. . .	25.832	

TABLE XXa.—SUMMARY SHOWING CLASSIFICATION OF OPERATING EXPENSES FOR THE YEAR ENDING JUNE 30, 1910, AND PROPORTION OF EACH CLASS TO THE TOTAL.—SMALL ROADS.—
(Continued.)

	Operating expenses.	Per cent.	Corresponding items in Table XX.
	II. MAINTENANCE OF EQUIPMENT.		
9	Superintendence of equipment.....	0.919	24
10	Locomotives—repairs.....	6.275	25, 28
11	Cars—repairs.....	6.510	31, 34, 37
12	Floating equipment—repairs.....	0.044	40
13	Work equipment—repairs.....	0.165	43
14	Equipment—renewals.....	0.608	26, 29, 32, 35, 38, 41, 44
15	Equipment—depreciation.....	4.214	27, 30, 33, 36, 39, 42, 45
16	Injuries to persons.....	0.025	48
17	Other maintenance of equipment expenses....	0.785	46, 47, 49, 50
18	Maintaining joint equipment at terminals—Dr.	0.026	51
19	Maintaining joint equipment at terminals—Cr.	0.063	52
	Total, maintenance of equipment.....	19.508	
	III. TRAFFIC EXPENSES.		
20	Traffic expenses.....	2.418	53-60
	IV. TRANSPORTATION EXPENSES.		
21	Superintendence and dispatching trains.....	2.200	61, 62
22	Station service.....	6.185	63-66
23	Yard enginemen.....	0.823	71
24	Other yard employees.....	1.259	67-69
25	Fuel for yard locomotives.....	1.276	73
26	All other yard expenses.....	0.376	70, 72, 74-76
27	Operating joint yards and terminals—Dr.....	0.743	77
28	Operating joint yards and terminals—Cr.....		78
29	Road enginemen and motormen.....	5.977	79, 80
30	Fuel for road locomotives.....	11.184	82
31	Other road locomotive supplies and expenses...	2.850	81, 83-87
32	Road trainmen.....	6.313	88
33	Train supplies and expenses.....	0.936	89
34	Injuries to persons.....	0.619	103
35	Loss and damage.....	0.471	99, 100
36	Other casualties.....	0.616	93, 101, 102
37	All other transportation expenses.....	1.916	90-92, 94-98 ¹
38	Operating joint tracks and facilities—Dr.....	0.481	104
39	Operating joint tracks and facilities—Cr.....	0.135	105
		43.472	
	V. GENERAL EXPENSES.		
40	Administration.....	7.088	106-109
41	Insurance.....	0.749	110
42	Other general expenses.....	0.968	111-114
43	General administration, joint tracks, yards, and terminals—Dr.....	0.031	115
44	General administration, joint tracks, yards, and terminals—Cr.....	0.063	116
		8.770	

CHAPTER XXI.

DISTANCE.

403. Relation of distance to rates and expenses. Rates are usually based on distance traveled, on the apparent hypotheses that each additional mile of distance adds its proportional amount not only to the service rendered but also to the expense of rendering it. Neither hypothesis is true. The value of the service of transporting a passenger or a ton of freight from *A* to *B* is a more or less uncertain gross amount depending on the necessities of the case and independent of the exact distance. Except for that very small part of passenger traffic which is undertaken for the mere pleasure of traveling, the general object to be attained in either passenger or freight traffic is the transportation from *A* to *B*, however it is attained. A mile greater distance does not improve the service rendered; in fact, it consumes valuable time of the passengers and perhaps deteriorates the freight. From the standpoint of service rendered, the railroad which adopts a more costly construction and thereby saves a mile or more in the route between two places is thereby fairly entitled to additional compensation rather than have it cut down as it would be by a strict mileage rate. The actual value of the service rendered may therefore vary from an insignificant amount which is less than any reasonable charge (which therefore discourages such traffic) and its value in cases of necessity—a value which can hardly be measured in money. If the passenger charge between New York and Philadelphia were raised to \$5, \$10, or even \$20, there would still be some passengers who would pay it and go, because *to them* it would be worth \$5, \$10, or \$20, or even more. Therefore, when they pay \$2.25 they are not paying what the service is worth to them. The service rendered cannot therefore be made a measure of the charge, nor is the service rendered proportional to the miles of distance.

The idea that the cost of transportation is proportional to

the distance is much more prevalent and is in some respects more justifiable, but it is still far from true. This is especially true of passenger service. The extra cost of transporting a single passenger is but little more than the cost of printing his ticket. Once aboard the train, it makes but little difference to the railroad whether he travels one mile or a hundred. Of course there are certain very large expenses due to the passenger traffic which must be paid for by a tariff which is rightfully demanded, but such expenses have but little relation to the cost of an additional mile or so of distance inserted between stations. The same is true to a slightly less degree of the freight traffic. As shown later, the items of expense in the total cost of a train-mile, which are directly affected by a small increase in distance, are but a small proportion of the total cost.

404. The conditions other than distance that affect the cost; reasons why rates are usually based on distance. Curvature and minor grades have a considerable influence on the cost of transportation, as will be shown in detail in succeeding chapters, but they are never considered in making rates. Ruling grades have a very large influence on the cost, but they are likewise disregarded in making rates. An accurate measure of the effect of these elements is difficult and complicated and would not be appreciated by the general public. Mere distance is easily calculated; the public is satisfied with such a method of calculation; and the railroads therefore adopt a tariff which pays expenses and profits even though the charges are not in accordance with the expenses or the service rendered.

An addition to the length of the line may (and generally does) involve curvature and grade as well as added distance. In this chapter is considered merely the effect of the added distance. The effect of grade and curvature must be considered separately, according to the methods outlined in succeeding chapters. The additional length considered is likewise assumed *not* to affect the business done nor the number of stations, but that it is a mere addition to length of track.

EFFECT OF DISTANCE ON OPERATING EXPENSES.

405. Effect of changes in distance on maintenance of way. The items of maintenance of way are more nearly affected in proportion to the distance than any other group of items. In

fact it will be easier to note the exceptions from a full 100% addition for all increases of distance. The cost of track labor, which is such a large percentage of the total cost of Item 6 (see Table XX in Chapter XX), and also the cost of all track material, will vary almost exactly in accordance with the distance. If the track-labor was so perfectly organized that there were no more laborers than could precisely accomplish the necessary work, working full time, then any additional labor would necessarily require a greater expenditure for laborers. Although a division of a road is divided into sections of such a length that a gang of say six or seven men will be employed as steadily as possible in maintaining the track in proper condition, the addition of a few feet of track would not probably have the effect of increasing the number of sections, nor would it even require the addition of another man to the track-gang. It might require a little harder work in maintaining a section, it *might* even mean a slight lowering in the standard of work done in order that the whole section should be covered. The fact remains that the cost of track-labor will not inevitably and necessarily be increased in a strict proportion to the increase in distance. On the other hand, it would not be wise to rely on any definite reduction or discount from the full 100% of work required, since to do so implies that, with the lessened distance there would be some loafing on the part of the track-gang, or that with the added distance the men would be over-worked or would be compelled to slight their work. The items, renewals of rails and renewals of ties, should certainly be considered as changing in direct proportion to the distance. Therefore it is only safe to allow the full 100% addition for Items 1 to 7.

The repairs and renewals of tunnels, bridges, culverts, fences, road crossings, signs, cattle-guards, buildings and fixtures, docks and wharves, etc. (Items 8 to 17), may perhaps be considered in the same way, although there are some of these items on which the effect is more doubtful. If a proposed change in line does not involve any difference in the number of streams crossed, then the number of the bridges and culverts will not be altered, and although the size *may* be altered, the effect of the change on the cost for repairs will probably be too insignificant for notice. For small changes of distance it may very readily happen that no bridge or culvert is involved. For

great changes of distance, especially those which would involve an entire change of route for a distance of many miles, it might be proper to consider Item 9 to be affected fully 100%. Although Items 9 and 10 are small, averaging about 1.8%, the error involved in these items by considering that the change amounts to 100% for great distances and zero for small distances will be almost inappreciable. For Items 11 to 13 the full 100% will be allowed for all changes of distance, for the same reason as previously given for repairs of roadway. Item 16 will usually be absolutely unaffected by a small change in distance, since it does not usually involve any buildings or fixtures. Larger changes of distance will probably require some change in the number of minor buildings required, but such buildings will be the more insignificant buildings, and we are therefore making ample allowance, if, under ordinary conditions, we estimate that 20% of the average cost of all buildings (which include terminals, etc.) is allowed for this item. Under ordinary conditions Item 17 will be absolutely unaffected by any changes in alinement which the engineer may make. An addition to distance will not usually affect the telegraph system, except as it adds to the number of telegraph-poles and to the amount of wiring and pole fixtures. Therefore any addition to distance will not add more than 50% to the average cost of Item 14. Items 18 to 21 are insignificant in amount, and can hardly be said to be affected by any small difference in distance which would ordinarily be measured in feet. Larger differences, which are measured in miles and which may involve, for instance, all the blank forms required for the reports of an additional section-gang, additional pay-rolls, etc., will be increased to practically their full proportion. Therefore there is but little error involved in allowing 100% on these items for changes of distance measured in miles.

406. Effect on maintenance of equipment. The relation between an increase in length of line and the expenses of Items 24, and 46 to 50 are quite indefinite. In some respects they would be unaffected by slight changes of distance, and yet it is difficult to prove that the expenses should not be considered proportionate for the distance. For example, the added train-mileage will increase repairs of rolling-stock, and will therefore hasten the deterioration and increase the cost of "repairs and renewals of shop machinery and tools" (Item 46). Fortunately,

all these items are so small, even in the aggregate, that little error will be involved, whatever decision is made. It will therefore be assumed that these items are affected 100% for large additions in distance and 50% for small additions. Items 40 to 42 are evidently unaffected by any change of distance. Electrical equipment, items 28-30 and 37-39, which are used on so few steam railroads, is ignored in this discussion.

There only remain the four groups of items, the repairs, renewals and depreciation of steam locomotives and of passenger and freight-cars and of work equipment. The deterioration of rolling stock, which requires its repair and finally its ultimate abandonment and therefore renewal, is caused by a combination of a large number of causes, of which the mere distance they travel on the road is but one cause. They deteriorate first with age; second, on account of the strains due to stopping and starting; third, on account of the strains and wear of wheels due to curved track; fourth, on account of the additional stresses due to grade and change of grade, and fifth, on account of the work of pulling on a straight level track. In addition to this, locomotives suffer considerable deterioration due to expansion and contraction, especially of the fire-box when the fires are drawn and the fire-box and boiler become cold, and again when the fire is started up. A large part of the expenses of maintaining passenger-cars is the expense of painting, which is a matter of mere time. Considering that the changes of distance, whose economic value the engineer tries to compute, will never make a difference in the number of round trips the engine or car would make in a day or month, the added distance which may be traveled does not add to the exposure of the car to the weather. Therefore, whatever deterioration of the car paint is due to weather, it will be incurred regardless of whether the length of the division of the road is 100 miles or 99 or 101. That element of the cost of car maintenance is absolutely independent of the precise length of that division of the road. On the other hand, the wear of car- and engine-wheels, although largely affected by curvature, is certainly affected to some extent by wear on a straight tangent. To determine the proportion of total wear due to these various causes is a matter of estimation and judgment. An approach to accuracy may be made by a compilation of the shop records of rolling-stock, repairs, showing the amount which is spent

in various kinds of repairs, and estimating as closely as possible what is the cause of each form of deterioration. A check on any such estimate is the consideration that the total deterioration is simply the summation of the deterioration due to all causes combined. It is therefore a question of dividing 100% into as many portions as there are contributing causes, and to assign to each cause its relative importance in per cent, so that the sum total shall reach 100. A. M. Wellington, in his "Economic Theory of Railway Location," distributed the cost of engine repairs to its various contributing causes, as shown in the following tabular form. He did not claim that such an estimate was accurate and applicable to all cases, but he did claim that the error was probably not sufficient to be of importance. A comparison of these percentages, with the data given by shop records on any particular road, would not probably show a very material difference, and the writer will not attempt to claim that any figures he might obtain will be any more accurate in general, although they might be more accurate as applied to some particular conditions.

DISTRIBUTION OF THE COST OF ENGINE REPAIRS TO ITS VARIOUS CONTRIBUTING CAUSES. (Copied from Wellington.)

Item.	Total cost of item.	Distribution.				Distance on tangent between stations, per cent.
		Effect of time, age, and exposure, per cent.	Stopping and starting at way stations per cent.	Terminal: getting up steam making up trains, per cent.	Curvature and grades, per cent. (App'ximate average.)	
Boiler.....	20.0		2	7	4	7
Running gear.....	20.0		4	2	7	7
Machinery.....	30.0	1	7	3	5	14
Mountings.....						
Lagging and painting..	12.0	4		2		6
Smoke-box, etc.....	5.0	1		1		3
Tender:						
Running gear.....	10.0		2	1	3	4
Body and tank.....	3.0	1		1		1
Total.....	100.0	7	15	17	19	42

It may be noted from the above table that 42% of engine repairs has been assigned to distance on a tangent between sta-

tions. If the added distance does not imply an extra stoppage of the train, there is but little, if any, reason to differentiate between the effect on repairs of a large or small addition to distance. We will therefore consider Items 25-27 to be affected in this ratio of 42%.

Wellington similarly distributed the cost of freight-car repairs to its various causes, and by a very similar method estimated that 36% of such repairs was due to distance on a tangent between stations. The considerable transformation in the construction of freight-cars, since the time that Wellington compiled this table, has certainly utterly changed the absolute cost of car repairs, even if it has not changed the relative percentage of the cost of the various items. In the lack of any better figures this same figure will be for Items 34-36. Although there are evidently enormous differences between Items 34-36 and 43-45, Items 43-45 are so small that it is hardly worth the calculation of any more precise figures, and therefore the same ratio, 36%, will be used for Items 43-45. Wellington made no definite calculations for the itemized cost of passenger-car repairs, but contented himself with using the same figure as for freight-cars, 36%. Such a percentage is probably very much too high, since it is estimated that about one-half the cost of passenger-car repairs is due to the work of painting, inside and out, and of maintaining the seats and upholstery in proper condition. Such repairs are chiefly a function of time, and are but little, if any, dependent on mere distance between stations. It is therefore considered that Items 31-33 will not be affected more than 20% by any addition of distance.

Traffic expenses will be unaffected.

407. Effect on conducting transportation. Items 61-78, superintendence and dispatching, all station and yard expenses, and the joint expenses of yards and terminals, may be considered as unaffected. Item 79, motormen's wages, is ignored.

Item 80. Road enginemen. In a previous chapter, §396 the wages of road enginemen were discussed. The discussion showed that the enginemen are rarely, although sometimes, paid on a strict mileage basis. They are usually paid on a trip basis, in accordance with which a slight change in the distance will not affect the classification of the trip, and therefore would make no difference in the wages. We will therefore say that

for small changes of distance, especially such as would be measured in feet, this item will be unaffected; but that for larger changes, such as would be measured in miles, this item will be affected by the full amount of the enginemen's wages.

Although there might be some justification for saying that the engine-house expenses for road engines (Item 81) might be somewhat increased by additions to distance, the addition may be considered as already covered by the increase in maintenance charges (Items 25-27), and no further allowance will be made.

Item 82. Fuel. A surprisingly large percentage of the fuel consumed is not utilized in drawing a train along the road. A portion of this percentage is used in firing-up. A portion is wasted when the engine is standing still, which is a considerable proportion of the whole time. The policy of banking fires instead of drawing them reduces the injury resulting from great fluctuations in temperature, but in a general way we may say that there is but little, if any, saving in fuel by banking the fires, and therefore we may consider that almost a fire-box full of coal is wasted whether the fires are banked or drawn. Some tests were made on the Santa Fé, in which some large locomotives consumed from 1200 to 1660 pounds of coal merely in firing-up. But even the amount of coal required to produce the required steam-pressure in the boiler from cold water does not represent the total loss. The train-dispatcher, in his anxiety that engines shall be ready when needed, will sometimes order out the locomotives which remain somewhere in the yard, perhaps exposed to cold weather, and blow off steam for several hours before they make an actual start. Of course the amount thus attributable to firing-up is a very variable one, depending on the management, and therefore no precise figures are obtainable. But it has been estimated that it amounts to from 5 to 10% of the total consumption. A freight-train, especially on a single-track road, will usually spend several hours during the day on sidings, and when a single-track road is being run to the limit of its capacity, or when the management is not good, the time will be still greater. It has been found that the amount of coal required by an engine merely to keep up steam will amount to from 25 to 50 pounds per hour. If, in addition to this, steam escapes through the safety-valve, the loss is much larger. It is estimated that the amount lost through a 2½-inch safety-

valve in one minute would represent the consumption of 15 pounds of coal, which would be sufficient to haul 100 tons on a mile of track with easy grades. Again we see that the amount thus lost is exceedingly variable and almost non-computable, although as a rough estimate the amount has been placed at from 3 to 6% of the total. Another very large subitem of loss of useful energy is that occasioned by stopping and starting. A train running 30 miles per hour has enough kinetic energy to move it on a straight level track for more than two miles. Therefore, every time a train running at 30 miles per hour is stopped, enough energy is consumed by the brakes to run it about two miles. There is a double loss, not only due to the fact of the loss of energy, but also because the power of the locomotive has been consumed in operating the brakes. When the train is again started, this kinetic energy must be restored to the train in addition to the ordinary resistances which are even greater, on account of the greater resistance at very low velocities. Of course, the proportion of fuel thus consumed depends on the frequency of the stops. It was demonstrated by some tests on the Manhattan Elevated Road in New York City, where the stops average one in every three-eighths of a mile, that this cause alone would account for the consumption of nearly three-fourths of the fuel. On ordinary railroads the proportion, of course, will not be nearly so great, but there is reason to believe that 10 to 20% is not excessive as an average figure. The amount of fuel which is consumed on account of curvature is, of course, a function of the curvature and will vary with each case. The only possible basis for a calculation of this amount must be somewhat as follows: Since all of the above calculations consider the average cost of a train-mile throughout the country, we must consider what is the average amount of curvature per mile of track throughout the country. By estimating, as will be developed in the next chapter, the proportion of the fuel expenditure which is due to this average amount of curvature and subtracting this, as well as the other subitems enumerated above, from the total average cost of a train-mile, we would then have the desired quantity, the cost of fuel per mile of straight level track. Although it is not easy to obtain reliable statistics showing the average curvature per mile or road throughout the United States, there is reason to believe that it is not far from 35° per mile. According to the

method of calculations given in the next chapter, to determine the additional fuel consumed by the added resistance due to 35° of curvature per mile, we obtain the approximate value that about 4% of the fuel consumption will be due to this cause. To obtain an average figure for the resistance due to grade is perhaps even harder, but on the approximate basis that the average amount of rise and fall per mile of track is about 11 feet per mile, it would seem as if 25% of the consumption of fuel were due to grades. Summarizing the above items we would have

Firing.....	5 to 10%	
Loss by radiation, etc.....	3 to 6%	
Stopping and starting.....	10 to 20%	
Average curvature.....	4 to 4%	
Average grade.....	25 to 25%	
	47 to 65%	
Direct hauling.....	53 to 35%	Average 44%
	100	100

This gives, as an average figure for the increased consumption of fuel due to one additional mile of straight level track, 44% of the average consumption per mile.

Items 83, 84, and 85. Minor engine-supplies. If water is obtained from municipal supplies and paid for at meter rates, then the cost will be strictly according to the consumption, which will be nearly according to the number of engine-miles. Almost the only waste would be that occasioned by blowing off steam. Under such circumstances the increase in this item would be very nearly 100%, but, on the other hand, where the supply is obtained from the company's own plant, there is hardly any appreciable increase in expense due to the extra draft on the tanks. Of course the cost of pumping would be somewhat affected. Considering that the sum of Items 83, 84, and 85 is only 1.06%, very little error would be involved if we consider as an average figure that this item will be increased the same as fuel, 44%.

Item 88 includes the wages of trainmen other than engine-men. Their wages are paid on very much the same basis as the enginemen, which means usually that small additions of distance will not affect their wages. Large additions will affect them 100%.

Item 89. Train-supplies and expenses include a very large matter of small subitems, the consumption of which is partly a matter of mere time and partly a matter of mileage. It is sufficiently precise in this case to say that 50% of these sub-items will be affected directly as the mileage.

Items 90, 91. Signals, flagmen, and gatemen. The necessity for flagmen and switchmen may be said in general to increase with the mileage, although it might readily happen that a given change in distance which is under consideration might not effect the slightest change in this item. It is quite unlikely that the number of switchmen would be affected. It is probable that 25% of this item is sufficient as an average figure.

Item 94. Telegraph expenses include the wages of operators at stations (which are unaffected) and the special expenses due to offices and telegraph stations and to operating the line, the maintenance of the line being charged to Item 14. Although it will theoretically require more battery material to transmit telegrams over a longer line, the added expense is so very slight that it may be utterly ignored.

Items 86 and 87, operating power plants and purchased power, which depend on electric traction, are ignored. Item 92, drawbridge operation, and Item 95, operating floating equipment, are considered to be unaffected. Items 96 to 98 are small miscellaneous expenses which are considered unaffected, as are likewise Items 104 and 105, the operation of joint tracks and facilities. This leaves Items 93 and 99 to 103, which are the fortuitous expenses due to wrecks, damage and mishaps. While it cannot be definitely predicted that these will happen with any stated regularity, it is only proper to consider that they will increase with added distance and to charge them as strictly proportional to distance.

The general expenses, Items 106 to 116, are considered to be unaffected.

408. Estimate of total effect on expenses of small changes in distance measured in feet; also estimate for distances measured in miles. Collecting the above percentages for the various items we have Table XXI, which shows that the average cost of operating a small additional distance will be about 33% of the average cost per unit distance. If the additional distance amounts to several miles, the added cost will amount to about 50% of the unit cost. These figures may also be con-

TABLE XXI. EFFECT ON OPERATING EXPENSES OF GREAT AND SMALL CHANGES IN DISTANCE.

No. of item.	Normal average.	Per cent affected.		Cost per mile, per cent.	
		Great.	Small.	Great.	Small.
*1-7	14.61	100	100	14.61	14.61
8	0.06	0	0	0	0
9, 10	1.77	100	0	1.77	0
11-13	0.82	100	100	0.82	0.82
14	0.49	50	50	0.10	0.10
15	0.02	0	0	0	0
16	1.81	20	0	0.36	0
17	0.20	0	0	0	0
18-21	0.45	100	0	0.45	0
22, 23	0.16	0	0	0	0
	20.09			18.11	15.53
24	0.64	100	50	0.64	0.32
25-27	8.62	42	42	3.62	3.62
28-30	0.01	0	0	0	0
31-33	2.14	20	20	0.43	0.43
34-36	10.15	36	36	3.65	3.65
37-39	0.01	0	0	0	0
40-42	0.08	0	0	0	0
43-45	0.34	36	36	0.12	0.12
46-50	0.72	100	50	0.72	0.36
51, 52	0.03	0	0	0	0
	22.74			9.18	8.50
53-60	3.08	0	0	0	0
61-79	17.92	0	0	0	0
80	6.08	100	0	6.08	0
81	1.72	0	0	0	0
82-85	11.41	44	44	5.02	5.02
86, 87	0.06	0	0	0	0
88	6.40	100	0	6.40	0
89	1.78	50	50	0.89	0.89
90, 91	0.85	25	25	0.21	0.21
92	0.05	0	0	0	0
93	0.25	100	100	0.25	0.25
94-98	1.06	0	0	0	0
99-103	2.84	100	100	2.84	2.84
104, 105	0.02	0	0	0	0
	50.44			21.69	9.21
106-116	3.65	0	0	0	0
	100.00			48.98	33.24

* For the significance of the items, see Table XX.

sidered as the saving in operating expenses resulting from a shortening of the line, and thus gives a measure of the operating value of reducing the length of the line. The average cost of a train-mile since 1890 has varied between 91.829 c. in 1895 to

148.865 c. in 1910. The cost has been rising almost steadily since 1897. Whether the cost will continue to rise or whether it will recede during the next few years is of course a matter of pure conjecture. Even if the cost recedes somewhat from the high value of recent years, it is quite certain that it will never again sink to the low value of 1895. If we adopt the round number of \$1.50 as the probable cost of a train-mile during the next few years, we can reduce the above percentages to cents per train-mile, which will come to 50 and 73 c. per train-mile respectively. Some trains run 365 days per year; others run but 313 days. The tendency, however, is toward the larger figure, especially in the case of freight service, which comprises about 52% of the number of train-miles. The added cost per daily train per year for *each foot* of distance would therefore be

$$\frac{50 \times 365 \times 2}{5280} = 6.91 \text{ c.}$$

When the distance is measured in miles the added cost per daily train per year for *each mile* of distance would be

$$.73 \times 365 \times 2 = \$533.$$

Of course, if such calculations are made for a light traffic road which only runs trains on week-days, we should use 313 in the above equations instead of 365. It should be noted that the subitems in the above table which are the most uncertain are those whose absolute value is the smallest, and that even if we make very large variations in the most uncertain items, the final result will not be very materially altered. On the other hand, the very largest items are those which are capable of fairly precise calculations.

EFFECT OF DISTANCE ON RECEIPTS.

409. Classification of traffic. There are various methods of classifying traffic, according to the use it is intended to make of the classification. The method here adopted will have reference to its competitive or non-competitive character and also to the method of division of the receipts on through traffic. Traffic may be classified first as "through" and "local"—

through traffic being that traveling over two (or [more) lines, no matter how short or non-competitive it may be; "local" traffic is that confined entirely to one road. A fivefold classification is however necessary—which is:

A. Non-competitive local—on one road with no choice of routes.

B. Non-competitive through—on two (or more) roads, but with no choice.

C. Competitive local—a choice of two (or more) routes, but the entire haul may be made on the home road.

D. Competitive through—direct competition between two or more routes each passing over two or more lines.

E. Semi-competitive through—a non-competitive haul on the home road and a competitive haul on foreign roads.

There are other possible combinations, but they all reduce to one of the above forms so far as their essential effect is concerned.

410. Method of division of through rates between the roads run over. Through rates are divided between the roads run over in proportion to the mileage. There may be terminal charges and possibly other more or less arbitrary deductions to be taken from the total amount received, but when the final division is made the remainder is divided according to the mileage. On account of this method of division and also because non-competitive rates are always fixed according to the distance, there results the unusual feature that, unlike curvature and grade, there is a compensating advantage in increased distance, which applies to all the above kinds of traffic except one (competitive local), and that the compensation is sometimes sufficient to make the added distance an actual source of profit. It has just been proved that the cost of hauling a train an additional mile is only 33 to 49% of the average cost. Therefore in all non-competitive business (local or through) where the rate is according to the distance, there is an actual profit in all such added distance. In competitive local business, in which the rate is fixed by competition and has practically no relation to distance, any additional distance is dead loss. In competitive through business the profit or loss depends on the distances involved. This may best be demonstrated by examples.

411. Effect of a change in the length of the home road on its receipts from through competitive traffic. Suppose the

home road is 100 miles long and the foreign road is 150 miles long. Then the home road will receive $\frac{100}{100+150}=40\%$ of the through rate.

Suppose the home road is lengthened 5 miles; then it will receive $\frac{105}{105+150}=41.176\%$ of the through rate. The traffic being competitive, the rate will be a fixed quantity regardless of this change of distance. By the first plan the rate received is 0.4% per mile; adding 5 miles, the rate for the original 100 miles may be considered *the same* as before; and that the additional 5 miles receive 1.176%, or 0.235% per mile. This is 59% of the original rate per mile, and since this is more than the cost per mile for the additional distance (see § 408), the added distance is evidently *in this case* a source of distinct profit. On the other hand, if the line is shortened 5 miles, it may be similarly shown that not only are the receipts lessened, but that the saving in operating expenses by the shorter distance is less than the reduction in receipts.

A second example will be considered to illustrate another phase. Suppose the home road is 200 miles long and the foreign road is 50 miles long. In this case the home road will receive

$\frac{200}{200+50}=80\%$ of the through rate. Suppose the home road is

lengthened 5 miles; then it will receive $\frac{205}{205+50}=80.392\%$ of the through rate. By the first plan the rate received is 0.400% per mile; adding 5 miles, there is a surplus of 0.392, or 0.0784 per mile, which is but 19.6% of the original rate. At this rate the extra distance evidently is not profitable, although it is not a dead loss—there is some compensation.

412. The most advantageous conditions for roads forming part of a through competitive route. From the above it may be seen that when a road is but a short link in a long competitive through route, an addition to its length will increase its receipts and increase them more than the addition to the operating expenses.

As the proportionate length of the home road increases the less will this advantage become, until at some proportion an increase in distance will just pay for itself. As the proportionate length grows greater the advantage becomes a disadvantage

until, when the competitive haul is entirely on the home road, any increase in distance becomes a net loss without any compensation. It is therefore advantageous for a road to be a short link in a long competitive route; an increase in that link will be financially advantageous; if the total length is less than that of the competing line, the advantage is still greater, for then the rate received per mile will be greater.

413. Effect of the variations in the length of haul and the classes of the business actually done. The above distances refer to particular lengths of haul and are not necessarily the total lengths of the road. Each station on the road has traffic relations with an indefinite number of traffic points all over the country. The traffic between each station on the road and any other station in the country between which traffic may pass therefore furnishes a new combination, the effect of which will be an element in the total effect of a change of distance. In consequence of this, any *exact* solution of such a problem becomes impracticable, but a sufficiently accurate solution for all practical purposes is frequently obtainable. For it frequently happens that the great bulk of a road's business is non-competitive, or, on the other hand, it may be competitive-through, and that the proportion of one or more definite kinds of traffic is so large as to overshadow the other miscellaneous traffic. In such cases an approximate but sufficiently accurate solution is possible.

414. General conclusions regarding a change in distance.

(a) In *all* non-competitive business (local and through) the added distance is actually profitable. Sometimes practically all of the business of the road is non-competitive; a considerable proportion of it is always non-competitive.

(b) When the competitive local business is very large and the competitive through business has a very large average home haul compared with the foreign haul, the added distance is a source of loss. Such situations are unusual and are generally confined to trunk lines.

(c) The above may be still further condensed to the general conclusion that there is always *some* compensation for the added cost of operating an added length of line and that it frequently is a source of actual profit.

(d) There is, however, a limitation which should not be lost sight of. The above argument may be carried to the logical

conclusion that, if added distance is profitable, the engineer should purposely lengthen the line. But added distance means added operating expenses. A sufficient tariff to meet these is a tax on the community—a tax which more or less discourages traffic. It is contrary to public policy to burden a community with an avoidable expense. But, on the other hand, a railroad is not a charitable organization, but a money-making enterprise, and cannot be expected to unduly load up its first cost in order that subsequent operating expenses may be unduly cheapened and the tariff unduly lowered. A common reason for increased distance is the saving of the first cost of a very expensive although shorter line.

(e) Finally, although there is a considerable and uncompensated loss resulting from curvature and grade which will justify a considerable expenditure to avoid them, there is by no means as much justification to incur additional expenditure to avoid distance. Of course needless lengthening should be avoided. A moderate expenditure to shorten the line may be justifiable, but large expenditures to decrease distance are never justifiable except when the great bulk of the traffic is exceedingly heavy and is competitive.

415. Justification of decreasing distance to save time. It should be recalled that the changes which an engineer may make which are physically or financially possible will ordinarily have but little effect on the time required for a trip. The time which can thus be saved will have practically no value for the freight business—at least any value which would justify changing the route. When there is a large directly competitive passenger traffic between two cities (*e.g.* New York to Philadelphia) a difference of even 10 minutes in the time required for a run might have considerable financial importance, but such cases are comparatively rare. It may therefore be concluded that the value of the time saved by shortening distance will not ordinarily be a justification for increased expense to accomplish it.

416. Effect of change of distance on the business done. The above discussion is based on the assumption that the business done is unaffected by any proposed change in distance. If a proposed reduction in distance involves a loss of business obtained, it is almost certainly unwise. But if by increasing the distance the original cost of the road is decreased (because

the construction is of less expensive character), and if the receipts are greater, and are increased still more by an increase in business done, then the change is probably wise. While it is almost impossible in a subject of such complexity to give a general rule, the following is generally safe: Adopt a route of such length that the annual traffic per mile of line is a maximum. This statement may be improved by allowing the element of original cost to enter and say, adopt a route of such length that the annual traffic per mile of line divided by the average cost per mile is a maximum. Even in the above the operating cost per mile, as affected by the curvature and grades on the various routes, does not enter, but any attempt to formulate a general rule which would allow for variable operating expenses would evidently be too complicated for practical application.

CHAPTER XXII.

CURVATURE.

417. General objections to curvature. In the popular mind curvature is one of the most objectionable features of railroad alignment. The cause of this is plain. The objectionable qualities are on the surface, and are apparent to the non-technical mind. They may be itemized as follows:

1. Curvature increases **operating expenses** by increasing (a) the required tractive force, (b) the wear and tear of roadbed and track, (c) the wear and tear of equipment, and (d) the required number of track-walkers and watchmen.

2. It may affect the **operation of trains** (a) by limiting the length of trains, and (b) by preventing the use of the heaviest types of engines.

3. It may affect **travel** (a) by the difficulty of making time, (b) on account of rough riding, and (c) on account of the apprehension of danger.

4. There is actually an increased **danger** of collision, derailment, or other form of accident.

Some of these objections are quite definite and their true value may be computed. Others are more general and vague and are usually exaggerated. These objections will be discussed in inverse order.

418. Financial value of the danger of accident due to curvature. At the outset it should be realized that in general the problem is *not* one of curvature *vs.* no curvature, but simply sharp curvature *vs.* easier curvature (the central angle remaining the same), or a greater or less percentage of elimination of the degrees of central angle. A straight road between termini is in general a financial (if not a physical) impossibility. The practical question is then, how much is the financial value of such diminution of danger that may result from such eliminations of curvature as an engineer is able to make?

In the year 1898 there were 2228 railroad accidents reported by the *Railroad Gazette*, whose lists of all accidents worth reporting are very complete. Of these a very large proportion clearly had no relation whatever to curvature. But suppose we assume that 50% (or 1114 accidents) were directly caused by curvature. Since there are approximately 200,000 curves on the railroads of the country, there was on the average an accident for every 179 curves during the year. Therefore we may say, according to the theory of probabilities, that the chances are even that an accident may happen on any particular curve in 179 years. This assumes all curves to be equally dangerous, which is not true, but we may temporarily consider it to be true. If, at the time of the construction of the road, \$1.00 were placed at compound interest at 5% for 179 years, it would produce in that time \$620.89 for each dollar saved, wherewith to pay all damages, while the amount necessary to eliminate that curvature, even if it were possible, would probably be several thousand dollars. The number of passengers carried one mile for one killed in 1898-99 was 61,051,580. If a passenger were to ride continuously at the rate of sixty miles per hour, day and night, year after year, he would need to ride for more than 116 years before he had covered such a mileage, and even then the probabilities of his death being due to curvature or to such a reduction of curvature as an engineer might accomplish are very small. Of course particular curves are often, for special reasons, a source of danger and justify the employment of special watchmen. They would also justify very large expenditures for their elimination if possible. But as a general proposition it is evidently impossible to assign a definite money value to the danger of a serious accident happening on a particular curve which has no special elements of danger.

Another element of safety on curved track is that trait of human nature to exercise greater care where the danger is more apparent. Many accidents are on record which have been caused by a carelessness of locomotive engineers on a *straight* track when the extra watchfulness usually observed on a curved track would have avoided them.

419. Effect of curvature on travel. (a) **Difficulty in making time.** The growing use of transition curves has largely eliminated the necessity for reducing speed on curves, and even when the speed is reduced it is done so easily and quickly by means

of air-brakes that but little time is lost. If two parallel lines were competing sharply for passenger traffic, the handicap of sharp curvature on one road and easy curvature on the other might have a considerable financial value, but ordinarily the *mere reduction* of time due to sharp curvature will not have any computable financial value.

(b) **On account of rough riding.** Again, this is much reduced by the use of transition curves. Some roads suffer from a general reputation for crookedness, but in such cases the excessive curvature is practically unavoidable. This cause probably does have some effect in influencing competitive passenger traffic.

(c) **On account of the apprehension of danger.** This doubtless has its influence in deterring travel. The amount of its influence is hardly computable. When the track is in good condition and transition curves are used so that the riding is smooth, even the apprehension of danger will largely disappear.

Travel is doubtless more or less affected by curvature, but it is impossible to say how much. Nevertheless the engineer should not ordinarily give this item any financial weight whatsoever. Freight traffic (two thirds of the total) is unaffected by it. It chiefly affects that limited class of sharply competitive passenger traffic—a traffic of which most roads have not a trace.

420. Effect on operation of trains. (a) **Limiting the length of trains.** When curvature actually limits the length of trains, as is sometimes true, the objection is valid and serious. But this can generally be avoided. If a curve occurs on a ruling grade without a reduction of the grade sufficient to compensate for the curvature, then the resistance on that curve will be a maximum and that curve will limit the trains to even a less weight than that which may be hauled on the ruling grade. In such cases the unquestionably correct policy is to “compensate for curvature,” as explained later (see §§ 427, 428), and not allow such an objection to exist. It is *possible* for curvature to limit the length of trains even without the effect of grade. On the Hudson River R. R. the total net fall from Albany to New York is so small that it has practically no influence in determining grade. On the other hand, a considerable portion of the route follows a steep rocky river bank which is so crooked that much curvature is unavoidable and very sharp curvature

can only be avoided by very large expenditure. As a consequence sharp curvature has been used and the resistance on the curves is far greater than that of any fluctuations of grade which it was necessary to use. Or, at least, a comparatively small expenditure would suffice to cut down any grade so that its resistance would be less than that of some curve which could not be avoided except at an enormous cost. And as a result, since the length of trains is really limited by curvature, minor grades of 0.3 to 0.5% have been freely introduced which might be removed at comparatively small expense. The above case is very unusual. Low grades are usually associated with generally level country where curvature is easily avoided—as in the Camden and Atlantic R. R. Even in the extreme case of the Hudson River road the maximum curvature is only equivalent to a comparatively low ruling grade.

(b) **Preventing the use of the heaviest types of engines.** The validity of this objection depends somewhat on the degree of curvature and the detailed construction of the engine. While some types of engines might have difficulty on curves of extremely short radius, yet the objection is ordinarily invalid. This will best be appreciated when it is recalled that the "Consolidation" type was originally designed for use on the sharp curvature of the mountain divisions of the Lehigh Valley R. R., and that the type has been found so satisfactory that it has been extensively employed elsewhere. It should also be remembered that during the Civil War an immense traffic daily passed over a hastily constructed trestle near Petersburg, Va., the track having a radius of 50 feet. As a result of a test made at Renovo on the Philadelphia and Erie R. R. by Mr. Isaac Dripps, Gen. Mast. Mech., in 1875,* it was claimed that a Consolidation engine encountered less resistance per ton than one of the "American" type. Whether the test was strictly reliable or not, it certainly demonstrated that there was no trouble in using these heavy engines on very sharp curvature, and we may therefore consider that, except in the most extreme cases, this objection has no force whatsoever.

* Seventh An. Rep. Am. Mast. Mech. Assn.

EFFECT OF CURVATURE ON OPERATING EXPENSES.

421. Relation of radius of curvature and of degrees of central angle to operating expenses. The smallest consideration will show that the sharper the curvature the greater will be the tractive force required, also the greater per unit of track length will be the rail wear and the general wear and tear on roadbed and rolling stock. But it would be inconvenient to use a relation between operating expenses and radius of curvature, because even when such a relation was found there would be two elements to consider in each problem—the radius and the length of the curve. The method which will be here developed cannot claim to be strictly accurate or even strictly logical, but, as will be shown later, the most uncertain elements of the computation have but a small influence on the final result, and the method is in general the only possible method of solution. The outline of the method is as follows:

(1) For reasons given in detail later, it is found that the expenses, wear, etc., on the track from *A* to *B* will be substantially the same whether by the route *M* or *N*. The wear, etc.,

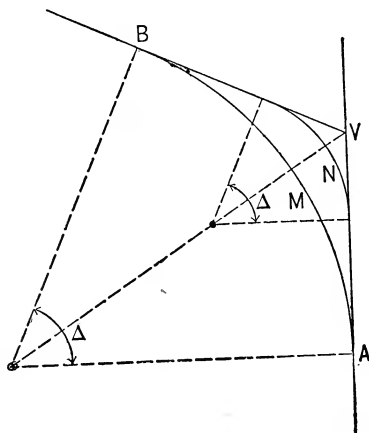


FIG. 208.

per foot at *N* is of course greater, but the length of curve is less. Therefore the effect of the curvature depends on the degrees of central angle Δ and is independent of the radius.

(2) At what degree of curvature is the total train resistance double its value on a tangent? Probably no one figure would be exact for all conditions. Train resistance varies with the velocity and with the various conditions of train loading even on a tangent, and it is by no means certain (or even probable) that the ratio would be exactly the same for all conditions. As an average figure we may say that a train running at average velocity on a 10° curve will encounter a resistance due to curvature which is approximately equal to the average resistance found on a level tangent. On a 10° curve therefore the resistance is doubled.

(3) A train-mile costs about so much—approximately \$1.35. Doubling the tractive resistance will increase certain items of expenditure about so much. Their combined value is so much *per cent* of the cost of a train-mile. A mile of continuous 10° curve contains 528° of central angle. A mile of such track would add so much per cent to the average train-mile expenses, and each degree of central angle is responsible for $\frac{1}{528}$ of this increase. Since the increase is irrespective of radius and depends only on the degrees of central angle, we therefore say that each degree of central angle of a curve will add so much to the average operating expenses of a train-mile.

The "cost per train-mile" considered above should be considered as the cost of a mile of level tangent. If we for a moment consider that all the railroads of the country were made absolutely straight and level, it is apparent that the average cost per train-mile instead of being about \$1.35, would be somewhat less. The percentage should therefore be applied to this reduced value, but the net effect of this change would evidently be small.

422. Effect of curvature on maintenance of way. A very large proportion of the items of expense in a train-mile are absolutely unaffected by curvature. It will therefore simplify matters somewhat if we at once throw out all the unaffected items. Of the items of maintenance of way and structures all but items 2 to 6 may be thrown out. Item 9 will be somewhat affected when bridges or trestles occur on a curve. But when it is considered what a very small percentage of this small item (1.709%) could be ascribed to curvature, since the very large majority of bridges and trestles are purposely made

straight, and since culverts, etc., are not affected, we may evidently ignore any variation in the item.

Item 3. RENEWALS OF TIES. Curvature affects ties by increasing the "rail cutting" and on account of the more frequent respiking, which "spike-kills" the ties even before they have decayed. Wellington estimates that a tie which will last nine years on a tangent will last but six years on a 10° curve. He adds 50% for tie renewals. He considers the decrease in tie life to be proportional to the degree of curve and therefore again verifies the general statement made above regarding the expense of curvature.

Item 4. RENEWALS OF RAILS. The *excess* wear due to curvature has never been determined with satisfactory conclusiveness. Some tests have been made within the last few years on the Northern Pacific Railroad, which have perhaps followed the only practical method for determining the law of rail wear on curves. Selected rails on several tangents and curves of varying degrees of curvature were annually taken up, cleaned and weighed, and the annual loss due to wear was noted. The results indicated a loss of weight on curves varying nearly according to the degree of curve, and that the *excess* wear on a curve is 22.6% per degree of curve over that on a tangent. For a 10° curve, this would mean an excess wear of 226%.

Item 6. Repairs of Roadway. A very large proportion of the subitems are absolutely unaffected. The care of embankments and slopes, the ditching, weeding, etc., are evidently unaffected. The track-labor on rails and ties and the work of surfacing will evidently be somewhat increased, and yet it is very seldom that the length of a track section would be decreased simply on account of excessive curvature throughout that section. We are here trying to estimate how much this item, which consists largely of track-labor, will be affected by 528° of central angle per mile. In the previous chapter an approximate estimate was made that the average curvature per mile of road for the whole United States is about 35° . 528° of curvature in a mile probably does not frequently occur. It would mean the equivalent of nearly $1\frac{1}{2}$ complete circles, and yet it is probably a generous estimate to say that the track-labor and other expenses belonging to this item would not be increased more than 25% for such an amount of curvature. Items 2 and 5 are also allowed 25%.

423. Effect of curvature on maintenance of equipment. All items except the repairs, renewals and depreciation of steam locomotives, passenger-, freight- and work-cars, and shop machinery and tools, will be considered as unaffected. As before, electric equipment is ignored.

Items 25-27. Repairs and renewals of locomotives. Curvature affects locomotive repairs by increasing very largely the wear on tires and wheels, and also the wear and strain due to the additional power required. Since the resistance due to curvature is very small compared with that due to even a moderate grade, this last cause may be neglected altogether. Referring to the tabulation in § 406, we find an estimate that 19% of the cost of engine repairs is assigned to curvature and grades combined. Of this amount two-thirds, or, say, 13%, should be assigned to curvature alone. On the basis that the average curvature of the roads of the country is about 35° per mile, which is about one-fifteenth of the 528° of curvature per mile which we are considering, then, if 35° is responsible for 13% increase in engine repairs, 528° would be responsible for 196%. It must be admitted that the above computation is grossly approximate, and that it contains the unwarrantable assumption that the extra cost of engine repairs which is due to curvature will be strictly in proportion to the degrees of curve. Although it is probably not true that 528° of curvature would increase the cost of engine repairs by 15 times the extra cost of 35° of curvature, yet it is probably true that for ordinary variations from that average of 35° per mile the increased cost of engine repairs will be approximately as the number of degrees of curve, and therefore our final value is not necessarily far out of the way. If 35° is responsible for an increase of 13%, 1° would be responsible for about .37 of 1%. In allowing an increase of 196% for 528° we are also allowing .37 of 1% per degree of central angle.

Items 31-36, 43-45. By a similar course of reasoning to that above given, the estimates for Items 34-36 and 43-45 will be made 100%, while that for Items 31-33 will be made only 50%, because such a large proportion of the expenses of Items 31-33 are due to painting and maintaining upholstery, which have no relation to variations in alinement.

Item 46. The repairs and renewals of shop machinery and tools will not be increased more than 50% per mile

for the additional repairs required on the above equipment.

424. Effect of curvature on conducting transportation. An inspection of the items under this general heading will show us that we may at once throw out as unaffected all the items except those which concern engine-supplies for road engines, flagmen, and watchmen, and the group which refers to accidents.

Items 82, 83, 84, and 85. The required quantities in this calculation are the additions to cost resulting from the introduction of 528° of central angle into a mile of track. We have assumed that this amount of curvature will exactly double the resistance. We found in Chapter XXI (§ 407) that the average increase in fuel consumption due to direct hauling amounts to about 44%. We have here assumed that the added curvature exactly doubles the work. We will therefore charge 44% of the average cost of a train-mile for this extra curvature. Since the consumption of water and other engine-supplies is roughly proportional to the consumption of coal, there will evidently be no error worth considering in assigning this same percentage to Items 83, 84, and 85.

Flagmen. There are many cases where a dangerous curve justifies and requires the employment of a special flagman to give timely notice of any dangerous condition of the track. Such special cases would, of course, justify a considerable expenditure to eliminate the dangerous features of that particular location, but such a charge should not be made against curvature in general. Ordinarily the elimination or retention of a curve will not involve the question of watchmen and flagmen in any way. We are therefore justified in disregarding this item altogether as a general proposition, if we keep in mind that the item should be included when we are considering the elimination of some particularly dangerous curve.

Items 99, 99-103. This group of items, which refer to accidents and the increased danger of accident due to curvature, and therefore the amount of money which might be justifiably spent to avoid this danger, has already been discussed in § 418. It was there shown that, although there might be special cases which would justify considerable expenditure on account of specific dangers in the situation, we cannot ordinarily give any definite financial value to this item as applied to curvature in

general. We therefore must eliminate these items as affecting the cost of curvature in general.

General expenses. Items 106 to 116 will also be unaffected.

425. Estimate of total effect per degree of central angle. Compiling the above estimates we have Table XXII. Accord-

TABLE XXII. EFFECT ON OPERATING EXPENSES OF CHANGES IN CURVATURE.

Item number.	Item (abbreviated).	Normal average.	Per cent affected.	Cost per mile, per cent.
2	Ballast.....	0.50	25	0.12
3	Ties.....	3.10	50	1.55
4	Rails.....	0.92	226	2.10
5	Other track material.....	1.13	25	0.28
6	Roadway and track.....	7.53	25	1.88
	(All other items).....	6.91	0	0
	Maintenance of way.....	20.09		5.93
25-27	Locomotives.....	8.61	196	16.88
31-33	Passenger-cars.....	2.14	50	1.07
34-36	Freight-cars.....	10.15	100	10.15
43-45	Work-cars.....	0.34	100	0.34
46	Shop machinery; tools.....	0.53	50	0.26
	(All other items).....	0.97	0	0
	Maintenance of equipment.....	22.74		28.70
53-60	Traffic.....	3.08	0	0
82-85	Fuel and other supplies for road engines.....	11.41	44	5.02
	(All other items).....	39.03	0	0
	Transportation.....	50.44		5.02
106-116	General expenses.....	3.65	0	0
		100.00		39.65

ing to the table 528° of curvature in one mile will increase the expenses of each train passing over it by 39.65% of the average cost of a train-mile, and, according to the general principles laid down in § 425, one degree of central angle of any curve, no

matter what the radius, will increase the expenses by $\frac{1}{528}$ of

39.65%, or .0751% per degree. Therefore, the cost per year per daily train each way, at the average rate of \$1.50 per train-mile would be

$$150 \times .0751\% \times 2 \times 365 = 82.23 \text{ c.}$$

For a round number we will call this 82 cents.

To forestall one kind of objection to the foregoing course of reasoning, it should be remembered that many of the estimates of additional cost, instead of being actually based on the effects of a continuous 10° curve, were based on the observed effects of lighter curvature, which were then multiplied by a factor to obtain the effect of a continuous 10° curve, as if the effects of curvature were strictly proportional to the curvature; but since we afterward divide our final result by 528 to obtain the effect of one degree of curvature, and then multiply this constant by the number of degrees of central angle, as found in ordinary practice, our calculations are not thereby vitiated because the effect of curvature is not strictly proportional to the rate of curvature. It is probably true that within the ordinary limits of variations in rate of curvature the above calculations are substantially true. In extreme cases they are probably in error, although it is likewise probably true that extreme curvature will have a variable effect on the rate of increase of the various items, and that even in extreme cases the error will not be very large.

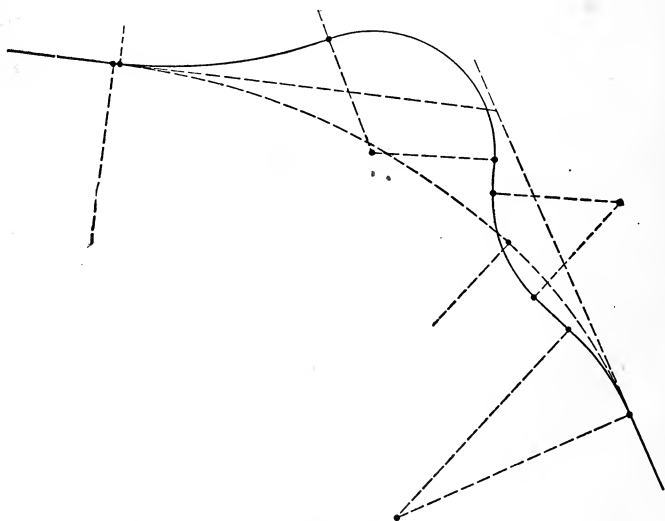


FIG. 209.

As a simple illustration (a more extended one will be given later), suppose that by using greater freedom with regard to

earthwork the crooked line sketched may be reduced to the simple curve shown and a curvature of, say, 110° may be reduced to, say, 60° .

Note that since the extreme tangents are identical, the saving in central angle results from the elimination of the reversed curvature and of that part of the direct curvature necessary to balance the reversed curvature. Assume that there are six daily trains each way. Then the annual saving is

$$50 \times .82 \times 6 = \$246,$$

which at 5% would justify an expenditure of \$4920. If the extra cost of construction does not exceed this, the improvement is justifiable, and is made all the more so if the probabilities are great that the future traffic will largely exceed six trains per day. At the same time the warning regarding "discounting the future" with respect to expected traffic should not be neglected. The possible effect of change of distance has not been referred to in the above problem. In any case it is a distinct problem. According to the above sketch, the difference in distance is probably very slight, and considering the compensating character of extra distance, such small differences may usually be disregarded. The possible effect of change of grade will be discussed in the next chapter. Assuming that there is no difference to be considered on account of either grade or distance, the question hinges on the advisability of spending \$4920 for the improvement.

426. Reliability and value of the above estimate. It should be realized at the outset that no extreme accuracy is claimed for the above estimate. The effect of curvature is somewhat variable as well as uncertain, but such estimates have this great value. Vary the estimates of individual items as you please (within reason), and the final result is still about the same and may be used to guide the judgment. As an illustration, suppose that the item of renewals of rails is assumed to be affected 300% rather than 226%, the justifiable expenditure to avoid the curvature in the above case may similarly be computed as \$5017, an increase of about 2%. But, after all, the real question is not whether the improvement is worth \$4800 or \$5017. The extra work involved may perhaps be done for \$500 or it may require \$10000. The above general method furnishes

reliance on vague judgment that it should not be ignored.

COMPENSATION FOR CURVATURE.

427. Reasons for compensation. The effect of curvature on a grade is to increase the resistance by an amount which is equivalent to a material addition to that grade. On minor grades the addition is of little importance, but when the grade is nearly or quite the ruling grade of the road, then the additional resistance induced by a curve will make that curve a place of maximum resistance and the real maximum will be a "virtual grade" somewhat higher than the nominal maximum. If, in Fig. 210,

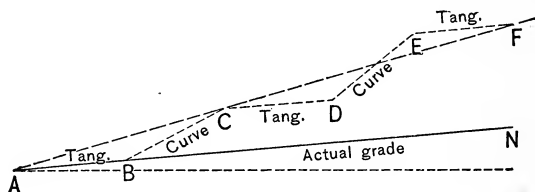


FIG. 210.

AN represents an actual *uniform* grade consisting of tangents and curves, the "virtual grade" on curves at BC and DE may be represented by BC and DE . If BC and DE are very long, or if a stop becomes necessary on the curve, then the full disadvantage of the curve becomes developed. If the whole grade may be operated without stoppage, then, as elaborated further in the next chapter, the whole grade may be operated as if equal to the average grade, AF , which is better than BC , although much worse than AN . The process of "compensation" consists in reducing the grade on every curve by such an amount that the actual resistance on each curve, due to both curvature and grade, shall precisely equal the resistance on the tangent. The practical effect of such reduction is that the "virtual" grade is kept constant, while the nominal grade fluctuates.

One effect of this is that (see Fig. 211) instead of accomplishing the vertical rise from A to G (i.e., HG) in the horizontal distance AH , it requires the horizontal distance AK . Such an addition to the horizontal distance can usually be obtained by proper development, and it should always be done on a ruling

grade. Of course it is possible that it will cost more to accomplish this than it is worth, but the engineer should be sure of this before allowing this virtual increase of the grade.

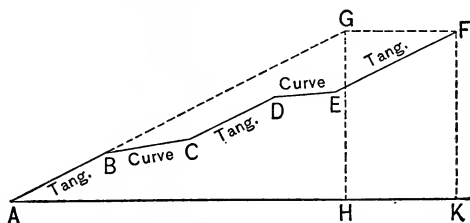


FIG. 211.

European engineers early realized the significance of unreduced curvature and the folly of laying out a uniform ruling grade regardless of the curvature encountered. Curve compensation is now quite generally allowed for in this country, but thousands of miles have been laid out without any compensation. A very common limitation of curvature and grade has been the alliterative figures 6° curvature and 60 feet per mile of grade, either singly or in combination. Assuming that the resistance on a 6° curve is equivalent to a 0.3% grade (15.84 feet per mile), then a 6° curve occurring on a 60-foot grade would develop more resistance than a 75-foot grade on a tangent. The "mountain cut-off" of the Lehigh Valley Railroad near Wilkesbarre is a fine example of a heavy grade compensated for curvature, and yet so laid out that the virtual grade is uniform from bottom to top, a distance of several miles.

428. The proper rate of compensation. This evidently is the rate of grade of which the resistance just equals the resistance due to the curve. But such resistance is variable. It is greater as the velocity is lower; it is generally about 2 lbs. per ton (equivalent to a 0.1% grade) per degree of curve when starting a train. On this account, the compensation for a curve which occurs at a known stopping-place for the heaviest trains should be 0.1% per degree of curve. The resistance is not even strictly proportional to the degree of curvature, although it is usually considered to be so. In fact most formulæ for curve resistance are based on such a relation. But if the experimentally determined resistances for low curvatures are applied to the excessive

curvature of the New York Elevated road, for example, the rules become ridiculous. On this account the compensation *per degree of curve* may be made less on a sharp curve than on an easy curve. The compensation actually required for very fast trains is less than for slow trains, say 0.02 or 0.03% per degree of curve; but since the comparatively slow and heavy freight trains are the trains which are chiefly limited by ruling grade, the compensation must be made with respect to those trains. From 0.04 to 0.05% per degree is the rate of compensation most usually employed for average conditions. Curves which occur *below* a known stopping-place for *all* trains need not be compensated, for the extra resistance of the curve will be simply utilized in place of brakes to stop the train. If a curve occurs just *above* a stopping-place, it is very serious and should be amply compensated. Of course the down-grade traffic need not be considered.

It sometimes happens that the ordinary rate of compensation will consume so much of the vertical height (especially if the curvature is excessive) that a steeper through grade must be adopted than was first computed, and then the trains might stall on the tangents rather than on the curves. In such cases a slight reduction in the rate of compensation might be justifiable. The proper rate of compensation can therefore be estimated from the following rules:

(1) On the upper side of a stopping-place for the heaviest trains compensate 0.10% per degree of curve.

(2) On the lower side of such a stopping-place do not compensate at all.

(3) Ordinarily compensate about 0.05% per degree of curve.

(4) Reduce this rate to 0.04% or even 0.03% per degree of curve if the grade on tangents must be increased to reach the required summit.

(5) Reduce the rate somewhat for curvature above 8° or 10°.

(6) Curves on minor grades need not be compensated, unless the minor grade is so heavy that the added resistance of the curve would make the total resistance greater than that of the ruling grade, or unless there is some ground to believe that the ruling grade may sometime be reduced below that of the minor grade under consideration.

429. The limitations of maximum curvature. What is the maximum degree of curvature which should be allowed on any

road? It has been shown that sharp curvature does not prevent the use of the heaviest types of engines, and although a sharp curve unquestionably increases operating expenses, the increase is but one of degree with hardly any definite limit. The general character of the country and the gross capital available (or the probable earnings) are generally the true criterions.

A portion of the road from Denver to Leadville, Col., is an example of the necessity of considering sharp curvature. The traffic that might be expected on the line was so meagre and yet the general character of the country was so forbidding that a road built according to the usual standards would have cost very much more than the traffic could possibly pay for. The line as adopted cost about \$20,000 per mile, and yet in a stretch of 11.2 miles there are about 127 curves. One is a 25° 20' curve, twenty-four are 24° curves, twenty-five are 20° curves, and seventy-two are sharper than 10° . If 10° had been made the limit (a rather high limit according to usual ideas), it is probable that the line would have been found impracticable (except with prohibitive grades) unless four or five times as much per mile had been spent on it, and this would have ruined the project financially.

For many years the main-line traffic of the Baltimore and Ohio R. R. has passed over a 300-foot curve (19° 10') and a 400-foot curve (14° 22') at Harper's Ferry. A few years ago some reduction was made in this by means of a tunnel, but the fact that such a road thought it wise to construct and operate such curves (and such illustrations on the heaviest-traffic roads are quite common) shows how foolish it is for an engineer to sacrifice money or (which is much more common) sacrifice gradients in order to reduce the *rate* of curvature on a road which at its best is but a second- or third-class road.

Of course such belittling of the effects of curvature may be (and sometimes is) carried to an extreme and cause an engineer to fail to give to curvature its due consideration. Degrees of central angle should always be reduced by all the ingenuity of the engineer, and should only be limited by the general relation between the financial and topographical conditions of the problem. Easy curvature is in general better than sharp curvature and should be adopted when it may be done at a small financial sacrifice, especially since it reduces distance generally and may even cut down the initial cost of that section of the

road. But large financial expenditures are rarely, if ever, justifiable where the net result is a mere increase in radius without a reduction in central angle. An analysis of the changes which have been so extensively made during late years on the Penn. R. R. and the N. Y., N. H. & H. R. R. will show invariably a reduction of distance, or of central angle, or both, and perhaps incidentally an increase in radius of curvature. There are but few, if any, cases where the sole object to be attained by the improvement is a mere increase in radius.

The requirements of standard M. C. B. car-couplers have virtually placed a limitation on the radius on account of the corners of adjacent cars striking each other on very sharp curves. This limitation has been crystallized into a rule on the P. R. R. that no curve, even that of a siding, can have a less radius than 175 feet, which is nearly the radius of a 33° curve. Of course only the most peremptory requirements of yard work would justify the employment of such a radius.

CHAPTER XXIII.

GRADE.

430. Two distinct effects of grade. The effects of grade on train expenses are of two distinct kinds; one possible effect is very costly and should be limited even at considerable expenditure; the other is of comparatively little importance, its cost being slight. As long as the length of the train is not limited, the occurrence of a grade on a road simply means that the engine is required to develop so many foot-pounds of work in raising the train so many feet of vertical height. For example, if a freight train weighing 600 tons (1,200,000 lbs.) climbs a hill 50 feet high, the engine performs an *additional* work of creating 60,000,000 foot-pounds of potential energy. If this height is surmounted in 2 miles and in 6 minutes of actual time (20 miles per hour), the *extra* work is 10,000,000 foot-pounds per minute, or about 303 horse-power. But the disadvantages of such a rise are always largely compensated. Except for the fact that one terminus of a road is generally higher than the other, every up grade is followed, more or less directly, by a down grade which is operated partly by the potential energy acquired during the previous climb. But when we consider the trains running in both directions even the difference of elevation of the termini is largely neutralized. If we could eliminate frictional resistances and particularly the use of brakes, the *net* effect of minor grades on the operation of minor grades in both directions would be zero. Whatever was lost on any up grade would be regained on a succeeding down grade, or at any rate on the return trip. On the very lowest grades (the limits of which are defined later) we may consider this to be literally true, viz., that nothing is lost by their presence; whatever is temporarily lost in climbing them is either immediately regained on a subsequent light down grade or is regained on the return trip. If a stop is required at the bottom of a sag, there is a net and uncompensated loss of energy.

On the other hand, if the length of trains is limited by the grade, it will require more trains to handle a given traffic. The receipts from the traffic are a definite sum. The cost of handling it will be nearly in proportion to the number of trains. Anticipating a more complete discussion, it may be said as an example that increasing the ruling grade from 1.20% (63.36 feet per mile) to 1.55% (81.84 feet per mile—an increase of about 18.5 feet per mile) will be sufficient to increase the required number of trains for a given gross traffic about 25%, i.e., five trains will be required to handle the traffic which four trains would have handled before at a cost slightly more than four-fifths as much. The effect of this on dividends may readily be imagined.

431. Application to the movement of trains of the laws of accelerated motion. When a train starts from rest and acquires its normal velocity, it overcomes not only the usual tangent resistances (and perhaps curve and grade resistances), but it also performs work in storing into the train a vast fund of kinetic energy. This work is not lost, for every foot-pound of such energy may later be utilized in overcoming resistances, provided it is not wasted by the action of train-brakes. If for a moment we consider that a train runs without any friction, then, when running at a velocity of v feet per second, it possesses a kinetic energy which would raise it to a height h feet, when

$h = \frac{v^2}{2g}$, in which g is the acceleration of gravity = 32.16. Assuming

that the engine is exerting just enough energy to overcome the frictional resistances, the train would climb a grade until the train was raised h feet above the point where its velocity was v . When it had climbed a height h' (less than h) it would have velocity $v_1 = \sqrt{2g(h-h')}$. As a numerical illustration, assume

$v = 30$ miles per hour = 44 feet per second. Then $h = \frac{v^2}{2g} = 30.1$ feet,

and assuming that the engine was exerting just enough force to overcome the rolling resistances on a level, the kinetic energy in the train would carry it for two miles up a grade of 15 feet per mile, or half a mile up a grade of 60 feet per mile. When the train had climbed 20 feet, there would still be 10.1 feet left and its velocity would be $v_1 = \sqrt{2g(10.1)} = 25.49$ feet per second = 17.4 miles per hour. These figures, however, must be slightly modified on account of the weight and the revolving

action of the wheels, which form a considerable percentage of the total weight of the train. When train velocity is being acquired, part of the work done is spent in imparting the energy of rotation to the driving-wheels and various truck-wheels of the train. Since these wheels run on the rails and must turn as the train moves, their rotative kinetic energy is just as effective—as far as it goes—in becoming transformed back into useful work. The proportion of this energy to the total kinetic energy has already been demonstrated (see Chapter XVI, § 347). The value of this correction is variable, but an average value of 5% has been adopted for use in the accompanying tabular form (Table XXIII), in which is given the corrected “velocity head” corresponding to various velocities in miles per hour. The table is computed from the following formula:

$$\text{Velocity head} = \frac{v^2 \text{ in ft. per sec.}}{64.32} = \frac{2.151v^2 \text{ in m. per h.}}{64.32} = 0.03344v^2$$

adding 5% for the rotative kinetic energy of the wheels, $0.00167v^2$

The corrected velocity head therefore equals $0.03511v^2$

Part of the figures of Table XXIII were obtained by interpolation and the final *hundredth* may be in error by one unit, but it may readily be shown that the final hundredth is of no practical importance. It is also true that the chief use made of this table is with velocities much less than 50 miles per hour. Corresponding figures may be obtained for higher velocities, if desired, by multiplying the figure for *half* the velocity by *four*.

432. Construction of a virtual profile. The following simple demonstration will be made on the basis that the ordinary tractive resistances and also the tractive force of the locomotive are independent of velocity. For a considerable range of velocity which includes the most common freight-train velocities this assumption is so nearly correct that the method will give an approximately correct result, but for higher velocities and for more accurate results a more complicated method (given later) must be used. The following demonstration will serve well as a preliminary to the more accurate method. It may best be illustrated by considering a simple numerical example.

Assuming that a train is passing *A* (see Fig. 212), running at 30 miles per hour. Assume that the throttle is not changed or any brakes applied, but that the engine continues to exert the

TABLE XXIII.—VELOCITY HEAD (REPRESENTING THE KINETIC ENERGY) OF TRAINS MOVING AT VARIOUS VELOCITIES.

Vel. mi. hr.	0.0	0.1	0.2	0.3	0.4	0.5	0.6	0.7	0.8	0.9
10	3.51	3.58	3.65	3.72	3.79	3.87	3.95	4.02	4.10	4.17
11	4.25	4.33	4.41	4.49	4.57	4.65	4.73	4.81	4.89	4.97
12	5.06	5.15	5.23	5.32	5.41	5.50	5.58	5.67	5.75	5.84
13	5.93	6.02	6.12	6.21	6.31	6.40	6.50	6.59	6.69	6.78
14	6.88	6.98	7.08	7.19	7.29	7.39	7.49	7.60	7.70	7.80
15	7.90	8.00	8.11	8.22	8.33	8.44	8.55	8.66	8.77	8.88
16	8.99	9.10	9.21	9.32	9.43	9.55	9.67	9.79	9.91	10.03
17	10.15	10.27	10.39	10.51	10.63	10.75	10.87	10.99	11.12	11.25
18	11.38	11.50	11.63	11.76	11.89	12.02	12.15	12.28	12.41	12.55
19	12.68	12.81	12.95	13.08	13.22	13.35	13.49	13.63	13.77	13.91
20	14.05	14.19	14.33	14.47	14.61	14.75	14.89	15.04	15.19	15.34
21	15.49	15.64	15.79	15.94	16.09	16.24	16.39	16.54	16.69	16.84
22	17.00	17.15	17.30	17.46	17.62	17.78	17.94	18.10	18.26	18.42
23	18.58	18.74	18.90	19.06	19.22	19.38	19.55	19.72	19.89	20.06
24	20.23	20.40	20.57	20.74	20.91	21.08	21.25	21.42	21.59	21.77
25	21.95	22.12	22.30	22.48	22.66	22.84	23.02	23.20	23.38	23.56
26	23.74	23.92	24.10	24.28	24.46	24.65	24.84	25.03	25.22	25.41
27	25.60	25.79	25.98	26.17	26.36	26.55	26.74	26.93	27.13	27.33
28	27.53	27.73	27.93	28.13	28.33	28.53	28.73	28.93	29.13	29.33
29	29.53	29.73	29.93	30.13	30.34	30.55	30.76	30.97	31.18	31.39
30	31.60	31.81	32.02	32.23	32.44	32.65	32.86	33.08	33.30	33.52
31	33.74	33.96	34.18	34.40	34.62	34.84	35.06	35.28	35.50	35.72
32	35.95	36.17	36.39	36.62	36.85	37.08	37.31	37.54	37.77	38.00
33	38.23	38.46	38.69	38.92	39.15	39.38	39.62	39.86	40.10	40.34
34	40.58	40.82	41.06	41.30	41.54	41.78	42.02	42.26	42.51	42.76
35	43.01	43.26	43.51	43.76	44.01	44.26	44.51	44.76	45.01	45.26
36	45.51	45.76	46.01	46.26	46.52	46.78	47.04	47.30	47.56	47.82
37	48.08	48.34	48.60	48.86	49.12	49.38	49.64	49.91	50.18	50.45
38	50.72	50.99	51.26	51.53	51.80	52.07	52.34	52.61	52.88	53.15
39	53.42	53.69	53.96	54.23	54.51	54.79	55.07	55.35	55.63	55.91
40	56.19	56.47	56.75	57.03	57.31	57.59	57.87	58.16	58.45	58.74
41	59.03	59.32	59.61	59.90	60.19	60.48	60.77	61.06	61.35	61.64
42	61.94	62.23	62.52	62.82	63.12	63.42	63.72	64.02	64.32	64.62
43	64.92	65.22	65.52	65.82	66.12	66.43	66.74	67.05	67.36	67.67
44	67.98	68.29	68.60	68.91	69.22	69.53	69.84	70.15	70.46	70.78
45	71.10	71.42	71.74	72.06	72.38	72.70	73.02	73.34	73.66	73.98
46	74.30	74.62	74.94	75.26	75.59	75.92	76.25	76.58	76.91	77.24
47	77.57	77.90	78.23	78.56	78.89	79.22	79.55	79.89	80.23	80.57
48	80.91	81.25	81.59	81.93	82.27	82.61	82.95	83.29	83.63	83.97
49	84.32	84.66	85.00	85.34	85.69	86.04	86.39	86.74	87.09	87.44
50	87.79	88.14	88.49	88.85	89.20	89.55	89.91	90.26	90.61	90.97

same draw-bar pull. At *A* its "velocity head" is that due to 30 miles per hour, or 31.60 feet. At *B* it has gained 40 feet more, and its velocity is that due to a velocity head of 71.60 feet, or slightly over 45 miles per hour. At *B'* its velocity is again 30 miles per hour and velocity head 31.60 feet. At *C* the velocity head is but 6.60 feet and the velocity about 13.7 miles per hour.

As the train runs from *C* to *D* its velocity increases to 30 miles at *C'* and to over 45 miles per hour at *D*. At *E* the velocity again becomes 30 miles per hour. Although there will be some slight modifications of the above figures in actual practice, yet the above is not a fanciful theoretical sketch. Thousands of just such undulations of grade are daily operated in such a way, without disturbing the throttle or applying brakes, and the draw-bar pull, if measured by a dynamometer, would be found to be practically constant. Of course the above case assumes that

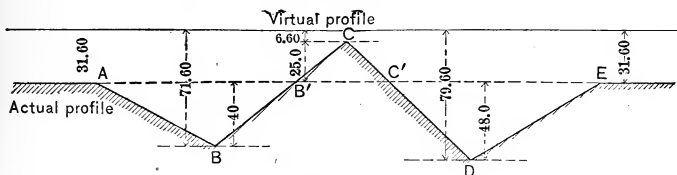


FIG. 212.

there are no stoppages and that the speed through the sags is not so great that safety requires the application of brakes. Observe that the "virtual profile" is here a straight line—as it always is when the draw-bar pull is constant. The virtual profile (in this case as well as in every other case, illustrations of which will follow) is found by adding to the actual profile at any point an ordinate which represents the "velocity head" due to the velocity of the train at that point.

As another case, assume that a train is climbing the grade *AE* and exerting a pull just sufficient to maintain a constant velocity up that grade. Then *A'B'* (parallel to *AB*) is the virtual profile, *AA'* representing the velocity head. A stop being required at *C*, steam is shut off and brakes are applied at *B*, and the velocity head *BB'* reduces to zero at *C*. The train

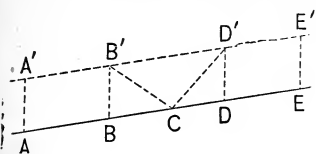


FIG. 213.

starts from *C*, and at *D* attains a velocity corresponding to the ordinate *DD'*. At *D* the throttle may be slightly closed so that the velocity will be uniform and the virtual grade is *D'E'*, parallel to *DE*.

From the above it may be seen that a virtual profile has the following properties:

(a) When the velocity is *uniform*, the virtual profile is parallel with the actual.

(b) When the velocity is increasing the profiles are separating; when decreasing the profiles are approaching.

(c) When the velocity is zero the profiles coincide.

433. Use, value, and possible misuse. The essential feature respecting grades is the demand on the locomotive. From the foregoing it may readily be seen that the ruling grade of a road is not necessarily the steepest nominal grade. When a grade may be operated by momentum, i.e., when every train has an opportunity to take "a run at the hill," it may become a very harmless grade and not limit the length of trains, while another grade, actually much less, which occurs at a stopping-place for the heaviest trains, will require such extra exertion to get trains started that it may be the worst place on the road. Therefore the true way to consider the value of the grade at any critical place on the road is to construct a virtual profile for that section of the road. The required length of such a profile is variable, but in general may be said to be limited by points on each side of the critical section at which the velocity is definite, as at a stopping-place (velocity zero), or a long heavy grade where it is the minimum permissible, say 10 or 15 miles per hour.

Since the velocities of different trains vary, each train will have its own virtual profile at any particular place. The fast passenger trains are generally unaffected, practically. The requirement of high average speed necessitates the use of powerful engines, and grades which would stall a heavy freight will only cause a momentary and harmless reduction of speed of the fast passenger train.

A possible misuse of virtual profiles lies in the chance that a station or railroad grade crossing may be subsequently located on a heavy grade that was designed to be operated by momentum. But this should not be used as an argument against the employment of a virtual profile. The virtual profile shows the *actual state of the case* and only points out the necessity, if an unexpected requirement for a full stoppage of trains at a critical point has developed, of changing the location (if a station), or of changing the grade by regrading or by using an overhead crossing. Examples of such modifications are given in Chapter XXIV, The Improvement of Old Lines.

434. Undulatory grades. Advantages. Money can generally be saved by adopting an actual profile which is not strictly uniform—the matter of compensation for curvature being here

ignored. Its effect on the operation of trains is harmless provided the sag or hump is not too great. In Fig. 214 the undulatory grade may actually be operated as a uniform grade AG . The sag at C must be considered as a sag, even though BC is actually an up grade. But the engine is supposed to be working

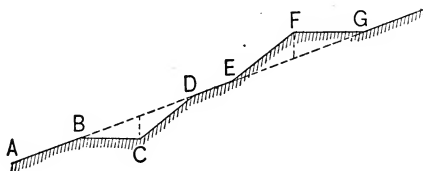


FIG. 214.

hard enough to carry a train at uniform velocity up a grade AG . Therefore it *gains* in velocity from B to C , and from C to D loses an equal amount. It may even be proven that the *time* required to pass the sag will be slightly less than the time required to run the uniform grade.

Disadvantages. The hump at F is dangerous in that, if the velocity at E is not equal to that corresponding to the extra velocity-head ordinate at F , the train will be stalled before reaching F . In practice there should be considerable margin. Any train should have a velocity of at least 10 miles per hour in passing any summit. This corresponds to a velocity head of 3.51 feet. An extra heavy head wind, slippery rails, etc., may use up any smaller margin and stall the train. If the grade AG is a ruling grade, then *no* hump should be allowed under any circumstances. For the heaviest trains are supposed to be so made up that the engine will *just* haul them up the ruling grades—of course with some margin for safety. Any increase of this grade, however short, would probably stall the train.

Safe limits. It is quite possible to have a sag so deep that it is not safe to allow freight trains to rush through them without the use of brakes. The use of brakes of course adds a distinct element of cost. To illustrate: If a freight train is running at a velocity of 20 miles per hour (velocity head 14.05 feet) and encounters a sag of 25 feet, the velocity head at the bottom of the sag will be 39.05 feet, which corresponds to a velocity of 33.3 miles per hour. This approaches the limit of safe speed for freight trains, and certainly passes the limit for trains not equipped with air-brakes and automatic couplers.

The term "safe limits" as used here, refers to the limits within which a freight train may be safely operated without the application of brakes or varying the work of the engine. Of course much greater undulations are frequently necessary and are safely operated, but it should be remembered that they add a distinct element to the cost of operating trains and that they must not be considered as harmless or that they should be introduced unless really necessary.

MINOR GRADES.

435. Basis of cost of minor grades. The basis of the computation of this least objectionable form of grade is as follows: The resistance encountered by a train on a level straight track is somewhat variable, depending on the velocity and the number and character of the cars, but for average velocities we may consider that 10 lbs. per ton is a reasonable figure. This value agrees fairly well with the results of some dynamometer tests made by Mr. P. H. Dudley, using a passenger train of 313 tons running at about 50 miles per hour. It also agrees with the *Engineering News* formula (Eq. 143) for the resistance of a train at a velocity of 32 miles per hour. Ten pounds per ton is the grade resistance of a 0.5% grade, or that of 26.4 feet per mile. On the above basis, a 0.5% grade will just double the tractive resistance on a level straight track. We may compute, as in the previous chapter, the cost of doubling the tractive resistance for one mile. But since the extra resistance is due to lifting the train through 26.4 feet of elevation, we may divide the extra cost of a mile of 0.5% grade by 26.4 and we have the cost of *one foot* of difference of elevation, and then (disregarding the limiting effect of grades) we may say that this cost of one foot of difference of elevation will be independent of the rate of grade. There are, however, limitations to this general proposition which will be developed in the next section.

436. Classification of minor grades. These are classified with reference to their effect on the operation of trains. In the first class are grades which may be operated without changing the work of the engine and which have practically no other effect than a harmless fluctuation of the velocity. But a grade which belongs to this class when considering a fast passenger train will belong to another class when considering a slow but heavy

freight train. And since it is the slow heavy freight trains which must be chiefly considered, a grade will usually be classified with respect to them. The limit of class A (the harmless class) therefore depends on the maximum allowable speed. The effect of a sag on speed will depend on the vertical feet of drop rather than on the rate of grade, for with the engine working as usual on even a light down grade a train

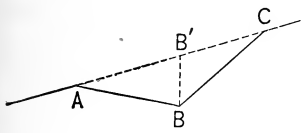


FIG. 215.

would soon exceed permissible speed. Assume that a freight train runs at an average speed of 15 miles per hour with a minimum of 10 miles and a permissible maximum of 30 miles per hour. Assume that a train runs up the grade at A with a uniform velocity of 15 miles per hour, i.e., the engine is working so that the velocity *would* be uniform to C. How much sag (BB') can there be without the speed exceeding 30 miles per hour?

Velocity head for 30 miles per hour.....	31.60
“ “ “ 15 “ “ “	<u>7.90</u>
The drop BB' will therefore be.....	23.70

While each case must be figured by itself, considering the probable velocity of approach and the maximum permissible velocity, we may say that a sag of *about* 24 feet will ordinarily mark the limit of this class. With a higher velocity of approach even this limit will be much reduced.

The classification therefore applies to sags and humps and to the vertical feet of drop or climb which are involved, rather than to grade *per se*. The practical application of these principles is necessarily confined to humps or sags which are possibly removable and does not apply to the long grades which are essential to connect predetermined points of the route—grades which are irreducible except by development and which must be studied as ruling grades (see §§ 440–445).

The application therefore consists in the comparative study of two proposed grades, noting the relative energy required to operate them and the probable cost. The depth in feet saved would be the maximum difference between the grades, and the classification will depend on the necessary method of operating the trains.

The next classification (*B*) applies to drops so deep that steam must be shut off when descending the grade, while the work required of the engine when ascending the opposite grade is correspondingly increased. The loss is not so serious as in the next case, but the inability of the engine to work continuously may result in a failure to accumulate sufficient kinetic energy to carry the train over a succeeding summit.

The third class (*C*) includes the grades so long that brakes must be applied to prevent excessive velocity. The loss involved is very heavy; the brakes require power for their application, they wear the brake-shoes and wheel-tires, they destroy kinetic or potential energy which had previously been created, while the tax on the locomotive on the corresponding ascending grade is very great. The ascending grade may or may not be a ruling grade.

437. Effect on operating expenses. As in the previous chapter we may at once throw out a large proportion of the items of expense of an average train-mile. In "maintenance of way and structures," Items 2 to 6 will be variously affected according to the classification of grades. The other items are evidently unaffected.

Item 3. Renewals of ties. The extra wear of ties on grades is considered by Wellington as being somewhat compensated by the improved drainage of the roadbed which is found on a track which is not quite level, the better drainage tending to increase the life of the ties. On this account Wellington made an allowance of 5% increase for class *C* and no increase for the other classes.

Item 4. Renewals of rails. Observations of rail wear on heavy grades show that the wear is much greater than on level tangents. The heavy grades of a mountain division of a road are usually operated with shorter trains or with the help of pusher-engines, and in such cases the proportion of engine tonnage to the total is much greater than on minor grades, and, since the engine has much greater effect on rail wear than an equal total tonnage of cars, partly on account of the use of sand and excess of engine tonnage, the grade will have a marked effect on rail wear. But such circumstances apply to ruling grades and very little, if at all, to minor grades. The use of sand on up-grades and the possible skidding of wheels on down-grades will wear the rails considerably. Even the slipping

of the drivers, although sand is not used, will wear the rails. Wellington allows 10% increase for class *C* and 5% for class *B*.

Item 6. Roadway and track. It is very plain that a large proportion of the subitems are absolutely unaffected by minor grades. In fact it is a little difficult to ascribe any definite increase to any subitem. Rail wear is somewhat increased and this will have some effect in increasing the track-work. Wellington allows 5% increase as a liberal estimate for class *C*, and makes no allowance for classes *A* and *B*. Items 2 and 5 are allowed the same.

Maintenance of equipment. It is evident that none of these items will be affected except repairs and renewals and depreciation of road engines and cars. The chief effect on these items will evidently be the repairs and renewals of wheels and brake-shoes. The draw-bars are also apt to suffer somewhat, on account of the increased work which they are required to do. It would appear that the extra stress on the locomotive mechanism will have some effect on locomotive repairs, and the expense of boiler repairs may be increased on account of the greater range of the demands on it. It would seem as if such effects would be quite large, but if the records of engine and car repairs on mountain divisions and on comparatively level divisions of the same road are examined and compared, there appears to be no such difference as might be expected. Considering the very small proportion of locomotive and car repairs which are affected at all by such circumstances, and also the very small percentage of these subitems which can be said to be affected by these two classes of minor grades, Wellington allows only an increase of 4% on each of these four items for class *C* and 1% for class *B*. This is reasonable in view of the 19% allowed for grades and curvature (see § 406) of which 13% was estimated for curves, leaving only 6% for *all* grades.

Conducting transportation. It is apparent that the only items in conducting transportation which will be affected will be the four items of engine supplies (82 to 85). As in Chapter XXII, since we are considering that the resistance is doubled, we will assume that there is an increase of 44% as the cost of the extra fuel used in climbing 26.4 feet, but the total cost of both rise and fall is to be considered. In class *B*, although steam is shut off, fuel will be wasted by mere radiation. On this account we will add 5%. For class *C* we must allow in

addition the energy spent in applying the brakes, which we may assume as 5% more. Therefore we will allow 49% for class *B* and 54% for class *C*. The other small items of supplies, 83, 84, and 85, will be estimated similarly. The items of general expense are evidently unaffected.

438. Estimate of cost of one foot of change of elevation. Collecting the above estimates we have Table XXIV, showing

TABLE XXIV.—EFFECT ON OPERATING EXPENSES OF 26.4 FEET OF RISE AND FALL.

Item number	Item (abbreviated).	Normal average	Class B.		Class C.	
			Per cent affected	Cost per mile per cent.	Per cent affected.	Cost per mile per cent.
2	Ballast.....	0.50	0	0	5	0.02
3	Ties.....	3.10	0	0	5	0.15
4	Rails.....	0.92	5	0.05	10	0.09
5	Other track material..	1.13	0	0	5	0.06
6	Roadway and track...	7.53	0	0	5	0.38
	(All other items).....	6.91	0	0	0	0
	Maintenance of way...	20.09		0.05		0.70
25-27, 31-36, 43-45	{ Locomotives, passen- ger cars, freight-cars and work-cars..... (All other items).....	21.64 1.50	1 0	0.21 0	4 0	0.85 0
	Maintenance of equip- ment.....	22.74		0.21		0.85
53-60	Traffic.....	3.08	0	0	0	0
82-85	Fuel and other supplies for road engines..... (All other items).....	11.41 39.03	49 0	5.59 0	54 0	6.16 0
	Transportation.....	50.44		5.59		6.16
106-116	General expenses.....	3.65	0	0	0	0
		100.00		5.85		7.71

that the percentage of *increase* for operating grades of classes *B* or *C* will be 5.85% and 7.71% respectively on the average cost of a train-mile. On the basis of an average cost of \$1.50 per train-mile, the additional cost for the 26.4 feet of rise and fall in one mile would be 8.77 c. and 11.56 c., or 0.33 c. and 0.44 c.

per foot for the two classes respectively. For each train per day each way per year the value per foot of difference of elevation is

For class *B*: $2 \times 365 \times \$0.0033 = \2.41 .

For class *C*: $2 \times 365 \times \$0.0044 = \$3.21.$

In assuming the number of trains running over a given grade the character of the trains must be carefully considered, since it will quite frequently happen that a hump or sag must be classified as belonging to class *C*, so far as heavy freight-trains are concerned, but may be classified as *B* or perhaps even *A*, the harmless class, for passenger-trains. The above values should likewise be modified when the trains are run only 313 days per year instead of 365.

439. Operating value of the filling of a sag in a grade. Assume that the sag is 4000 feet long and that its depth in the center is 35 feet, as sketched in Fig. 216. Assume that a freight train is approaching the sag from the right-hand side (running to the left), the speed of passing *D* being 25 miles per hour. The 0.3% grade to the left will furnish a gravity pull of 6 pounds per ton, which may be more than half the force required to pull the train, and the locomotive would have but little to do, even if there were no sag, to maintain the speed of 25 miles per hour. Assuming that the locomotive is doing just this amount of work in running to the left through the sag, it will gain in velocity. Its velocity head at *D* is that corresponding to 25 miles per hour, or 21.95 feet. Adding 35 feet, the depth of the sag, we have 56.95 feet, which is the velocity head at *C*, which

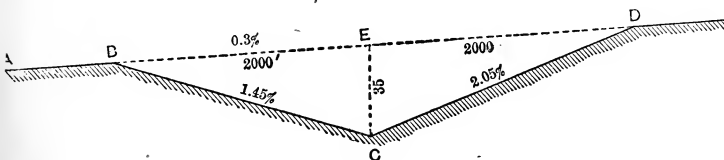


FIG. 216.

means that the velocity of the train would be 40.3 miles per hour. For passenger trains this will not be an objectionably high velocity, and even freight trains, which are provided with air-brakes and standard M. C. B. couplers, which are now

nearly universal, may be safely run at this velocity. Therefore for all trains which may run at a speed of over 40 miles per hour, this sag will belong to class *A*, or the harmless class, so far as trains moving to the left are concerned. The effect on trains moving to the right will depend partly on the rate of ruling grade on that section of the road. The conditions of class *A* pre-suppose that the draw-bar pull is *constant*, but horse-power is measured by the product of pull times velocity. Assume that *AB* is a ruling grade—0.3% has recently been adopted as ruling grade for revision work on the Erie R. R. Then, if 20 miles per hour were the speed of approach (running to the right) the speed at *C* would have to be 37.4 miles per hour. But since the draw-bar pull is to be constant, the horse-power developed would have to be nearly doubled $\left(\frac{37.4}{20}\right)$. Since

this would be impossible, it would mean that such a sag would not only be a serious matter but would be prohibitive on a ruling grade. For ordinary roads, 0.3% will not be a ruling grade, and the possibility of temporarily increasing the horse-power developed by the locomotive while running through the sag, so that the draw-bar pull will remain constant, will be far greater. The ability to develop such horse-power is very apt to be the criterion as to whether a sag belongs to class *A* rather than the danger that the speed may be prohibitive. The criterion as to whether the grade belongs to class *B* or *C* for trains moving to the left, depends on whether brakes must be applied before reaching the bottom of the sag. A sharp curve at or near *C* might require the use of brakes to prevent a dangerous velocity. For trains moving to the right, there is no definite criterion between classes *B* and *C*, but a 2.05% grade is a very severe tax on a locomotive, especially when the assistance of momentum has been wasted by shutting off steam or by the application of brakes. Ignoring the possible limiting effect of a ruling grade (which is a separate matter) the value of the 35-foot sag is evidently

$35 \times \$2.41 = \84.35 per daily round-trip train for class *B*
and

$35 \times \$3.21 = \112.35 per daily round-trip train for class *C*.

Assume that there are six daily trains each way, for which the grade would be classified as grade *B*, and four others, for

which the sag would be classified as involving class *C* grade. Then on the above basis, the total annual cost would be

$$6 \times \$84.35 = \$506.10$$

$$4 \times \$112.35 = \$449.40$$

$$\$955.50$$

This annual cost capitalized at 5% equals \$19,110, which is the justifiable expenditure to fill up the sag. The amount of fill in such a sag, roughly calculated, is 125,000 cubic yards. Assuming that it would cost 30 c. per cubic yard to make the fill, this would require an expenditure of \$37,500, which apparently would not be justifiable.

But another solution may be considered. It has been shown above that a sag may be made harmless for all classes of trains provided the depth is not greater than some uncertain limit, which depends on the particular circumstances of the case. The volume of earth in this fill is great on account of the great depth in the center. It may be readily computed that by filling up the lower part of the sag so that its maximum depth below the grade line, *BED*, is about one-half of *CE*, or about 17 feet, that the amount of earthwork required will be only about 25,000 yards rather than 125,000 yards. This would make the cost of such a fill practically \$7500 rather than \$37,500. If it could be shown that a sag of about 18 feet could be operated on the class *A* basis by all trains, then it would certainly be justifiable to expend \$7500 in order to secure a reduction in the operating expenses, whose capitalized value according to the above calculation, is \$19,110.

RULING GRADES.

440. Definition. Ruling grades are those which limit the weight of the train of cars which may be hauled by one engine. The subject of "pusher grades" will be considered later. For the present it will suffice to say that on all well-designed roads the large majority of the grades on any one division are kept below some limit which is considered the ruling grade. If a heavier grade is absolutely necessary no special expense will be made to keep it below a rate where the resistance is twice

(or possibly three times) the resistance on the ruling grade, and then the trains can be hauled unbroken up these few special grades with the help of one (or two) pusher engines. So far as limitation of train length is concerned, these pusher grades are no worse than the regular ruling grades and, except for the expense of operating the pusher engines (which is a separate matter), they are not appreciably more expensive than any ruling grade. As before stated, the engineer cannot alter very greatly the ruling grade of the road when the general route has been decided on. He may remove sags or humps, or he may lower the natural grade of the route by development in order to bring the grade within the adopted limit of ruling grade. The financial value of removing sags and humps has been considered. It now remains to determine the financial relation between the lowest permissible ruling grade and the money which may profitably be spent to secure it.

441. Choice of ruling grade. It is of course impracticable for an engine to drop off or pick up cars according to the grades which may be encountered along the line. A train load is made up at one terminus of a division and must run to the other terminus. Excluding from consideration any short but steep grades which may *always* be operated by momentum, and also all pusher grades, the maximum grade on that division is the ruling grade.

It will evidently be economy to reduce the few grades which naturally would be a little higher than the great majority of others until such a large amount of grade is at some uniform limit that a reduction at all these places would cost more than it is worth. The precise determination of this limit is practically impossible, but an approximate value may be at once determined from a general survey of the route. The distance apart of the termini of the division into their difference of elevation is a first trial figure for the rate of the grade. If a grade even approximately uniform is impossible owing to the elevations of predetermined intermediate points, the worst place may be selected and the natural grade of that part of the route determined. If this grade is much steeper than the general run of the natural grades, it may be policy to reduce it by development or to boldly plan to operate that place as a pusher grade. The choice of possible grades thus has large limitations, and it justifies very close study to determine the best

combination of grades and pusher grades. When the choice has narrowed down to two limits, the lower of which may be obtained by the expenditure of a definite extra sum, the choice may be readily computed, as will be developed.

442. Maximum train load on any grade. The tractive power of a locomotive has been discussed in Chap. XV, § 322. The *net* train load which may be placed behind any engine is the difference between the weight of the engine itself and the *gross* load which can be handled under the given circumstances, with a given weight on the drivers. Since the design of locomotives is so variable, it is impracticable to show in tabular form the power of all kinds of locomotives on all grades. In Table XXV are given the tractive powers of locomotives of a wide range of types and weights and with various ratios of adhesion. They may be accepted as typical figures and will serve to compute the effect of variations of grade on train load. In Table XXVI is given the total train resistance in pounds per ton for various grades and for various values of track resistances. By a combination

TABLE XXV.—TRACTION POWER OF VARIOUS TYPES OF STANDARD-GAUGE LOCOMOTIVES AT VARIOUS RATES OF ADHESION.

Type of locomotive.	Total weight of engine and tender.		Weight of engine only.	Weight on the drivers.	Tractive power when ratio of adhesion is		
	Lbs.	Tons.			$\frac{1}{4}$	$\frac{3}{10}$	$\frac{1}{2}$
Atlantic, 4-4-2 . . .	340,000	170.0	199,400	105,540	26,385	23,740	21,100
Atlantic, 4-4-2, four cylinder compound	368,800	184.4	206,000	115,000	28,750	25,875	23,000
Pacific, 4-6-2	343,600	171.8	218,000	142,000	35,500	31,950	28,400
Pacific, 4-6-2	403,780	201.9	226,700	151,900	37,975	34,180	30,380
Ten-wheel, 4-6-0 . . .	321,000	160.5	201,000	154,000	38,500	34,650	30,800
Prairie, 2-6-2	366,500	183.2	212,500	154,000	38,500	34,650	30,800
Consolidation, 2-8-0	214,000	107.0	120,000	106,000	26,500	23,850	21,200
Consolidation, 2-8-0	366,700	183.3	221,500	197,500	49,375	44,440	39,500
Mikado, 2-8-2	405,500	202.7	259,000	196,000	40,000	44,100	39,200

of these two tables the net train load on any grade under given conditions may be quickly determined. For example, an ordinary consolidation engine having a weight of 106000 pounds on the drivers (see Table XXV) will have a tractive force of 26500 pounds under fair conditions of track, when the adhesion ratio is $\frac{1}{4}$. When climbing slowly up a grade of 1.30% the tractive resistance will be about 32 pounds per ton if the rolling-stock and track are fair—assuming a tractive resistance on

a level of 6 pounds per ton. Dividing 26500 by 32 we have 828 tons, the gross train load. Subtracting 107 tons, the weight of the engine and tender in working order, we have 721 tons, the *net* load. Incidentally we may note that, cutting down the grade to 0.90% (a reduction of only 21.12 feet per mile), the resistance per ton is reduced to 24 pounds and the gross train load is increased to 1104 tons and the net load to 997 tons—an increase of about 38%.

As another numerical example, consider a contractor's locomotive (not referred to in Table XXV), a light four-wheel-connected-tank narrow-gauge engine, with a total weight of 12000 pounds, all on the drivers. On the rough temporary track used by contractors the tractive ratio may be as low as $\frac{1}{5}$. The tractive adhesion should therefore be taken as 2400 pounds. Assume that the grade when hauling "empties" is 4.7% and that the tractive resistance on such a track *on a level* is 10 pounds per ton. By Table XXVI, the total train resistance is therefore (by interpolation) 104 pounds per ton. $2400 \div 104 = 23$ tons; subtracting the weight of the engine we have 17 tons, the net load of empty cars—perhaps twenty cars weighing 1700 pounds per car.

In general, and to compute accurately the train load under conditions not exactly given in the tables, the maximum train load at slow speed may be computed according to the following rule:

The maximum load behind an engine on any grade may be found by multiplying the weight on the drivers by the ratio of adhesion and dividing this by the sum of the grade and tractive resistances per ton; this gives the gross load, from which the weight of the engine and tender must be subtracted to find the net load.

443. Proportion of the traffic affected by the ruling grade. Some very light traffic roads are not so fortunate as to have a traffic which will be largely affected by the rate of the ruling grade. When passenger traffic is light, and when, for the sake of encouraging traffic, more frequent trains are run than are required from the standpoint of engine capacity, it may happen that no passenger trains are really limited by any grade on the road—i.e., an extra passenger car *could* be added if needed. The maximum grade then has no worse effect (for passenger trains) than to cause a harmless reduction of speed at a few points.

TABLE XXVI.—TOTAL TRAIN RESISTANCE PER TON (OF 2000 POUNDS) ON VARIOUS GRADES.

Grade.		When tractive resistance <i>on a level</i> in pounds per ton is					Grade.		When tractive resistance <i>on a level</i> in pounds per ton is				
Rate per cent.	Feet per mile.	6	7	8	9	10	Rate per cent.	Feet per mile.	6	7	8	9	10
0.00	0.00	6	7	8	9	10	2.00	105.60	46	47	48	49	50
.05	2.64	7	8	9	10	11	.05	108.24	47	48	49	50	51
.10	5.28	8	9	10	11	12	.10	110.88	48	49	50	51	52
.15	7.92	9	10	11	12	13	.15	113.52	49	50	51	52	53
.20	10.56	10	11	12	13	14	.20	116.16	50	51	52	53	54
0.25	13.20	11	12	13	14	15	2.25	118.80	51	52	53	54	55
.30	15.84	12	13	14	15	16	.30	121.44	52	53	54	55	56
.35	18.48	13	14	15	16	17	.35	124.08	53	54	55	56	57
.40	21.12	14	15	16	17	18	.40	126.72	54	55	56	57	58
.45	23.76	15	16	17	18	19	.45	129.36	55	56	57	58	59
0.50	26.40	16	17	18	19	20	2.50	132.00	56	57	58	59	60
.55	29.04	17	18	19	20	21	.55	134.64	57	58	59	60	61
.60	31.68	18	19	20	21	22	.60	137.28	58	59	60	61	62
.65	34.32	19	20	21	22	23	.65	139.92	59	60	61	62	63
.70	36.96	20	21	22	23	24	.70	142.56	60	61	62	63	64
0.75	39.60	21	22	23	24	25	2.75	145.20	61	62	63	64	65
.80	42.24	22	23	24	25	26	.80	147.84	62	63	64	65	66
.85	44.88	23	24	25	26	27	.85	150.48	63	64	65	66	67
.90	47.52	24	25	26	27	28	.90	153.12	64	65	66	67	68
0.95	50.16	25	26	27	28	29	.95	155.76	65	66	67	68	69
1.00	52.80	26	27	28	29	30	3.00	158.40	66	67	68	69	70
.05	55.44	27	28	29	30	31	.05	161.04	67	68	69	70	71
.10	58.08	28	29	30	31	32	.10	163.68	68	69	70	71	72
.15	60.72	29	30	31	32	33	.15	166.32	69	70	71	72	73
.20	63.36	30	31	32	33	34	.20	168.96	70	71	72	73	74
1.25	66.00	31	32	33	34	35	3.25	171.60	71	72	73	74	75
.30	68.64	32	33	34	35	36	.30	174.24	72	73	74	75	76
.35	71.28	33	34	35	36	37	.35	176.88	73	74	75	76	77
.40	73.92	34	35	36	37	38	.40	179.52	74	75	76	77	78
.45	76.56	35	36	37	38	39	.45	182.16	75	76	77	78	79
1.50	79.20	36	37	38	39	40	3.50	184.80	76	77	78	79	80
.55	81.84	37	38	39	40	41	4.00	211.20	86	87	88	89	90
.60	84.48	38	39	40	41	42	4.50	237.60	96	97	98	99	100
.65	87.12	39	40	41	42	43	5.00	264.00	106	107	108	109	110
.70	89.76	40	41	42	43	44	5.50	290.40	116	117	118	119	120
1.75	92.40	41	42	43	44	45	6.00	316.80	126	127	128	129	130
.80	95.04	42	43	44	45	46	6.50	343.20	136	137	138	139	140
.85	97.68	43	44	45	46	47	7.00	369.60	146	147	148	149	150
.90	100.32	44	45	46	47	48	8.00	422.40	166	167	168	169	170
1.95	102.96	45	46	47	48	49	9.00	475.20	186	187	188	189	190
2.00	105.60	46	47	48	49	50	10.00	528.00	206	207	208	209	210

The local freight business is frequently affected in practically the same way. All coal, mineral, or timber roads are affected

by the rate of ruling grade as far as such traffic is concerned. Likewise the through business in general merchandise, especially of the heavy traffic roads, will generally be affected by the rate of ruling grade. Therefore in computing the effect of ruling grade, the total number of trains on the road should not ordinarily be considered, but only the trains to which cars are added, until the limit of the hauling power of the engine on the ruling grades is reached.

444. Financial value of increasing the train load. The gross receipts for transporting a given amount of freight is a definite sum regardless of the number of train loads. The cost of a train mile is nearly constant. If it were exactly so, the saving in operating expenses would be strictly proportional to the number of trains saved. How will the cost per train mile vary when by a reduction in ruling grade more cars are handled in one train than before? First, compute the effect of increasing the train load so that one less engine will handle the traffic, or, for example, that an engine can haul 11 cars instead of 10 or 44 instead of 40—that 10 engines will do the work for which 11 engines would be required with the steeper grade. What will be the relative cost of running 10 heavy trains rather than 11 lighter trains, or, rather, what will be the extra cost of the extra engine?

We will estimate as before the difference in the cost of operating, say, four light trains on heavy grades, or three heavier trains on the lighter grades. In either case the gross tonnage of cars, with their contents, is supposed to be the same. The difference consists in the cost of operating the extra engine and also the extra cost for train service, etc., which is a function of the number of trains on the road rather than of their tonnage. The additional cost of maintenance of way is confined to the effect of the extra engine, and this will evidently effect only Items 2 to 6 and 9. On the basis that an engine produces one-half of the track deterioration, we may allow 50% of those items as the total effect on maintenance of way.

Maintenance of equipment. The effect on maintenance of equipment will be practically confined to the repairs, renewals, and depreciation of steam locomotives (Items 25 to 27) and the same items for freight-cars (Items 34 to 36). Very few roads have a passenger traffic which is affected by the rate of the ruling grade, since, for the encouragement of traffic, passenger-

trains are usually added to the schedule in advance of the physical capacity of a locomotive to haul one or more additional cars. Therefore, in general, no allowance need be made for any effect on passenger-trains, or on the maintenance of the passenger-cars. But the cost of maintaining the freight-cars is actually reduced by having more trains and less cars per train. This means that, although the maximum draw-bar pull will be the same in both cases, and will equal the maximum capacity of the locomotive, while on the ruling grades, the draw-bar pull while on the light grades, level track and down grades, which may mean 90% of the length of the road, will average very much less. It is impossible to make an accurate estimate of the amount of this saving which would be generally applicable. Wellington estimated it at 10%. Considering the very large proportion of freight-car maintenance charges which are evidently independent of draw-bar pull, the estimate is probably large enough and the error in adopting that figure is not very great.



Although, for either system of grades, the locomotive is supposed to work to its full capacity while on the ruling grades, there is also some saving for each locomotive when hauling a lighter train over the light grades, level sections and down grades. If, on account of the reduction in average draw-bar pull, the repairs of each of four locomotives were reduced 5% below the repairs of the three locomotives which could haul the same number of cars over lower ruling grades, then the repairs of the four locomotives would cost 4×95 , or 380 compared with 3×100 , or 300 for the three locomotives. This would mean that the additional locomotive should be assigned an added expenditure of 80% of the average cost for one. If the saving by the reduction of grade was only half as much, or one train in eight, so that seven trains were required to do the work of eight on the steeper grade, then the saving per engine would be correspondingly less. If it were 2.5% instead of 5, we would have, for the cost of eight engines, 8×97.5 , or 780 compared with 7×100 , or 700 for the seven engines. Again, we would have 80% as the additional net cost of the repairs on the additional engine. Of course the above estimates of 5% and 2.5%, as the saving on one engine, are merely guesswork, but the above demonstration shows that if the saving in repairs is proportional to the reduction in number of trains which is

made possible by the reduction of grade, as is quite probable, then the cost of repairs of the extra engine is the same, whether it is one engine out of four, or one engine out of eight, or one engine out of any other number. Therefore, although the estimate of 5% per engine as made above is a guess, it is probably near enough to the truth, so that there is a comparatively little error in using the figure of 80% for the additional cost of repairs of the additional engine.

Conducting transportation. Items 61-79, chiefly yard work, will be practically unaffected. Items 80 and 81 will be given their full value. The additional cost for the fuel for the added engine may be computed somewhat on the same basis as the cost of engine repairs. The fuel used by four engines will average somewhat less than that used by three engines, since the four engines will do far less work on the level and on very light grades, while on the heavier grades the engines are working to the limit of their capacity in either case. The loss of heat, due to radiation and the other causes, which are independent of the direct work done by the engine in hauling, will be the same in either case. These causes have already been discussed in §407. It is impossible to make any general calculations as to the relative consumption of fuel in the two cases, since so much depends on the proportion of track which is level or which has a very light grade. If the four engines operating lighter trains each burn 5% less fuel than three engines operating heavier trains, we will find, by the same method as before, that the extra engine may be charged with 80% of the average fuel consumption of the other three engines. If, as before, we assume that this variation in fuel consumption is proportional to the variation in the number of trains required to handle a given traffic, then the extra engine would be responsible for 80% of the average fuel consumption, regardless of whether the number of trains saved was one in four or one in ten. Although it is true that the value 80% is a mere guess, that no general value is obtainable, and that the value for any particular and special case could only be computed with great difficulty, it is evident that the error is not very great, and we will therefore assume 80% as the fuel consumption assignable to the extra engine. The consumption of other engine supplies, water, oil, waste, etc., is not strictly proportional to the con-

sumption of fuel, but we will assume it to be so in this case, and that 80% of Items 83-85 are allowable for the extra engine.

Items 88 to 94, which concern train-service, will be considered as varying according to the number of train-miles, and we will therefore add 100% for all these items. Items 97 and 98 will be allowed 50%. Items 99, 101 to 103, which refer to damages, might be considered from one standpoint to be unaffected, while from other standpoints the effect might be considered as 100%. The risk of train operations varies very largely with the number of trains, and yet in some respects the danger is independent of whether there are 15 or 20 cars in a train. There will be little error in assigning 50% extra for this item. Items 104 and 105 will be allowed 100% of their *net* value. The general expenses are evidently unaffected. Collecting these various items we have Table XXVII.

If we assume that the average cost of a train-mile is \$1.50, then the operating value per mile of saving the use of the additional engine equals 41.63% of \$1.50, or 62.45 c.

445. Operating value of a reduction in the rate of the ruling grade. As a practical application of the above figures, assume that on a constructed and operated road the ruling grade on a 100-mile division is 1.6%; the actual traffic affected by ruling grade is 8 daily trains with a net load of 552 tons or 4416 tons. It is found that with an expenditure of \$400,000 the ruling grade may be reduced to 1.2%. Will it pay? At 1.2% grade the net load behind an 80-ton consolidation engine, with 48 tons on the drivers, adhesion $\frac{1}{4}$, and 6 pounds per ton normal resistance, is 720 tons. The traffic (4416 tons) may therefore be hauled by 6 engines, the balance, less than 100 tons, being taken care of by lighter trains not affected by the ruling grade. There is therefore the saving due to not operating two engines. Since the *additional* cost of the two engines drawing lighter trains is 62.45 c. per mile, the annual saving is therefore $2 \times \$0.6245 \times 100 \times 365 = \$45,588.50$, which capitalized at 5% = \$911,770. This shows that if the improvement can be accomplished for \$400,000 it is worth while.

As in other similar problems, it must be reiterated that although there are some more or less uncertain elements in the above estimates, yet with a considerable margin for error in

TABLE XXVII. ADDITIONAL COST OF OPERATING A GIVEN FREIGHT TONNAGE WITH $(n+1)$ ENGINES ON HEAVY RULING GRADES INSTEAD OF WITH n ENGINES ON LIGHTER GRADES.

Item number.	Item (abbreviated).	Normal average.	Per cent affected.	Cost per mile, per cent.
2	Ballast, Ties, Rails, Other track material, Roadway and track, Bridges, trestles and culverts (All other items)	14.89	50	7.45
3				
4				
5				
6				
9		5.20	0	0
	Maintenance of way	20.09		7.45
25-27	Steam locomotives	8.61	80	6.89
34-36	Freight cars	10.15	-10	-1.01
	(All other items)	3.98	0	0
	Maintenance of equipment	22.74		5.88
53-60	Traffic	3.08	0	0
80	Road enginemen	6.08	100	6.08
81	Enginehouse expenses, road	1.72	100	1.72
82-85	Fuel, and other supplies for road engines	11.41	80	9.13
88-94	Train service, etc.	9.66	100	9.66
97, 98	Stationery, printing, etc.	0.57	50	0.28
99, 101, 103	Loss, damage, etc.	2.82	50	1.41
104, 105	Operating joint tracks, <i>net</i>	0.02	100	0.02
	(All other items)	18.16	0	0
	Transportation	50.44		28.30
106-116	General expenses	3.65	0	0
		100.00		41.63

individual items the value of the whole improvement will not be very greatly altered and the estimate will be infinitely better than an indefinite reliance on vague "judgment." Of course certain items in the above estimates are somewhat variable and should be altered to fit the particular case to be computed.

PUSHER GRADES.

446. General principles underlying the use of pusher engines: On nearly all roads there are some grades which are greatly in excess of the general average rate of grade and these heavy

grades cannot usually be materially reduced without an expenditure which is excessive and beyond the financial capacity of the road. If no pusher engines are used, the length of all heavy trains is limited by these grades. The financial value of the reduction of such ruling grades has already been shown. But in the operation of pusher grades there is incurred the additional cost of pusher-engine service, for a pusher engine must run *twice* over the grade for each train which is assisted. It is possible for this additional expense to equal or even exceed the advantage to be gained. In any case it means the adoption of the lesser of two evils, or the adoption of the more economical method. A simple example will illustrate the point. Assume that at one point on the road there is a grade of 1.9% which is five miles long. Assume that all other grades are less than 0.92%. If pushers are not to be used the net capacity of a 107-ton consolidation engine with 53 tons on the drivers, assuming $\frac{9}{40}$ adhesion and 6 pounds per ton for normal resistance, will be 435 tons, and that will be the maximum weight of train allowable. By using pusher engines on this one 5-mile grade the train load is at once doubled and the number of trains cut down one half. This double load, 870 tons, can easily be hauled by one engine up the 0.92% grades. As a rough comparison, free from details and allowances, we may say:

(a) 10 trains per day over a 100-mile division, 435 tons net per train, will require 1000 engine miles daily.

(b) 5 trains per day handling the same traffic, 870 tons net per train, with $2 \times 5 \times 5$ pusher-engine miles, will require $(5 \times 100) + (2 \times 5 \times 5) = 550$ engine miles daily. There is thus a large saving in the number of engine miles and also in the number of the engines required for the work. Moreover, the engines are working to the limit of their capacity for a much larger proportion of the time, and their work is therefore more economically done. The work of overcoming the normal resistances of so many loaded cars over so many miles of track and of lifting so many tons up the gross differences of elevation of predetermined points of the line is approximately the same whatever the exact route, and if the grades are so made that fewer engines working more constantly can accomplish the work as well as more engines which are not hard worked for a considerable proportion of the time, the economy is very ap-

parent and unquestionable. Wellington expresses it concisely: "It is a truth of the first importance that the objection to high gradients is not the work which the engines have to do on them, but it is the work which they do *not* do when they thunder over the track with a light train behind them, from end to end of a division, in order that the needed power may be at hand at a few scattered points where alone it is needed."

447. Balance of grades for pusher service. In the above illustration the "through" grade and the "pusher" grade are "balanced" for the use of one equal pusher. It is therefore evident that if some intermediate grade (such as 1.4%) were permitted, it could only be operated by (a) making it the ruling grade and cutting down all train loads from 870 tons to 594 tons, or (b) operating it as a pusher grade, although with a loss of economy, since two engines would have much more power than necessary. The proper plan in such a case would be to strive to reduce the 1.4% grade to 0.92%, or, if that seemed impracticable, to attempt to get an operating advantage at the expense of an increase of the 1.4% grade to anything short of 1.9%. For the increase in *rate* of grade would cost almost nothing, and some advantage might be obtained which would practically compensate for the introduction of a pusher grade. *Another possible* solution would be to operate the 1.9% with two pushers, adopt a corresponding grade for use with one pusher and a corresponding ruling grade for through trains. With the above data these three grades would be 1.90%, 1.27%, and 0.54%, obtained as follows:

Tractive power of three engines = $106000 \times \frac{9}{40} \times 3 = 71550$ pounds.

Resistance on 1.9% grade = $6 + (20 \times 1.9) = 44$ lbs. per ton.

$71550 \div 44 = 1626$ = gross load in tons.

$1626 - (3 \times 107) = 1305$ = net load in tons.

$1305 + (2 \times 107) = 1519$ = gross load on the one-pushers grade.

Tractive power of two engines = 47700 lbs.

$47700 \div 1519 = 31.40$ = possible tractive force in lbs. per ton.

$(31.40 - 6) \div 20 = 1.27\%$ = permissible grade for one pusher.

$1305 + 107 = 1412$ = gross load on the through grade.

Tractive power of one engine = 23850 lbs.

$23850 \div 1412 = 16.89$ = possible tractive force in lbs. per ton.

$(16.89 - 6) \div 20 = 0.54\%$ = permissible through grade.

It should be realized that, assuming the accuracy of the normal resistance (6 lbs.) and the normal adhesion ($\frac{9}{40}$) and with the use of 107-ton locomotives with 53 tons on the drivers, the above figures are *precisely* what is required for hauling with one, two, and three engines. Other types of engines, other values for resistance and adhesion will vary considerably the gross load in tons which may be hauled up those grades, but starting with 0.54% as a through grade, the corresponding values for one and for two pushers would vary but slightly from those given. To show the tendency of these variations, the corresponding values have been computed as follows:

Adhesion.	Resistance per ton.	Load on drivers.	Through grade.	One-pusher grade.	Two-pusher grade.
$\frac{9}{40}$	6 lbs.	53 tons.	0.54%	1.27%	1.90%
$\frac{8}{40}$	7 "	53 "	.54%	1.31%	1.96%
$\frac{7}{40}$	6 "	53 "	.54%	1.28%	1.93%
$\frac{6}{40}$	6 "	53 "	.54%	1.26%	1.86%
$\frac{5}{40}$	7 "	53 "	.54%	1.29%	1.92%

The above form shows that *increasing* the resistance per ton and *decreasing* the adhesion have opposite effects on altering the ratio of these grades, and as a storm, for example, would increase the resistance and decrease the adhesion, the changes in the ratio would be compensating although the absolute reduction in train load might be considerable.

In Table XXVIII is shown a series of "balanced" grades on which a given *net* train load may be operated by means of one or two pusher engines. For example, assuming a track resistance of 6 pounds per ton, a consolidation engine of the type shown in the table can haul a train weighing 977 tons (exclusive of the engine) up a grade of 0.80%. If this is the maximum *through* grade, pusher grades as high as 1.70% for one pusher, or 2.46% for two pushers, may be introduced and the same *net* load may be hauled up these grades.

The ratios of pusher grade to through grade, as given in Table XXVIII, are exactly true only for the conditions named as to weight and type of engine, ratio of adhesion, and normal track resistance. But a little comparative study of the two

TABLE XXVIII.—BALANCED GRADES FOR ONE, TWO, AND
THREE ENGINES.

Basis.—Through and pusher engines alike; consolidation type; total weight, 107 tons; weight on drivers, 53 tons; adhesion, $\frac{3}{8}$, giving a tractive force for each engine of 23850 lbs.; normal track resistance, 6 (also 8) lbs. per ton.

Through grade.	Track resistance, 6 lbs.			Track resistance, 8 lbs.		
	Net load for one engine in tons (2000 lbs.).	Corresponding pusher grade for same net load.		Net load for one engine in tons (2000 lbs.).	Corresponding pusher grade for same net load.	
		One pusher.	Two pushers.		One pusher.	Two pushers.
Level.	3868 tons	0.28%	0.55%	2874 tons	0.37%	0.72%
0.10%	2874 "	0.47%	0.82%	2278 "	0.56%	0.98%
0.20%	2278 "	0.66%	1.08%	1880 "	0.74%	1.23%
0.30%	1880 "	0.84%	1.33%	1596 "	0.92%	1.47%
0.40%	1596 "	1.02%	1.57%	1384 "	1.09%	1.70%
0.50%	1384 "	1.19%	1.80%	1218 "	1.27%	1.92%
0.60%	1218 "	1.37%	2.02%	1085 "	1.44%	2.14%
0.70%	1085 "	1.54%	2.24%	977 "	1.60%	2.36%
0.80%	977 "	1.70%	2.46%	887 "	1.77%	2.56%
0.90%	887 "	1.87%	2.66%	810 "	1.93%	2.76%
1.00%	810 "	2.03%	2.86%	745 "	2.09%	2.96%
1.10%	745 "	2.19%	3.06%	688 "	2.24%	3.15%
1.20%	688 "	2.34%	3.25%	638 "	2.40%	3.33%
1.30%	638 "	2.50%	3.43%	594 "	2.55%	3.51%
1.40%	594 "	2.65%	3.61%	555 "	2.70%	3.68%
1.50%	555 "	2.80%	3.78%	521 "	2.85%	3.85%
1.60%	521 "	2.95%	3.95%	489 "	2.99%	4.02%
1.70%	489 "	3.09%	4.12%	461 "	3.13%	4.17%
1.80%	461 "	3.23%	4.27%	435 "	3.27%	4.33%
1.90%	435 "	3.37%	4.43%	411 "	3.42%	4.49%
2.00%	411 "	3.52%	4.59%	390 "	3.55%	4.63%
2.10%	390 "	3.65%	4.73%	370 "	3.68%	4.78%
2.20%	370 "	3.78%	4.88%	352 "	3.81%	4.92%
2.30%	352 "	3.91%	5.02%	335 "	3.94%	5.05%
2.40%	335 "	4.04%	5.15%	319 "	4.07%	5.19%
2.50%	319 "	4.17%	5.29%	304 "	4.20%	5.32%

halves of Table XXVIII and of the tabular form given on page 535 will show that although the net load which can be hauled on any grade varies considerably with the normal track resistance and also with the ratio of adhesion, yet the *ratios* of through to pusher grade, for either one or two pushers, varies but slightly with ordinary changes in these conditions. Therefore when the precise conditions are unknown or variable, the

figures of Table XXVIII may be considered as applicable to any ordinary practice, especially for preliminary computations. For final calculations on any proposed ruling grade and pusher grade, the whole problem should be worked out on the principles outlined above and on the basis of the best data obtainable.

Problem: If the *through* ruling grade for the road has been established at 1.12%, what pusher grades are permissible?

Answer: Interpolating in Table XXVIII, we may employ a grade of 2.22% if the track and road-bed are to be such that a tractive resistance of 6 pounds per ton can be expected. With a poorer track, the normal resistance assumed as 8 pounds per ton, the rate is raised to 2.27%. The *increase* in rate of pusher grade with increase of resistance is due to the fact that the net load hauled is less—so much less that on the pusher grade a larger part of the adhesion is available to overcome a grade resistance.

448. Operation of pusher engines. The maximum efficiency in operating pusher engines is obtained when the pusher engine is kept constantly at work, and this is facilitated when the pusher grade is as long as possible, i.e., when the heavy grades and the great bulk of the difference of elevation to be surmounted is at one place. For example, a pusher grade of three miles followed by a comparatively level stretch of three miles and then by another pusher grade of two miles cannot all be operated as cheaply as a continuous pusher grade of five miles. Either the two grades must be operated as a continuous grade of eight miles (sixteen pusher miles per trip) or else as two short pusher grades, in which case there would be a very great loss of time and a difficulty in so arranging the schedules that a train need not wait for a pusher or the pushers need not waste too much time in idleness waiting for trains. If the level stretch were imperative, the two grades would probably be operated as one, but an effort should be made to bring the grades together. It is not necessary to bring the trains to a stop to uncouple the pusher engine, but a stop is generally made for coupling on, and the actual cost in loss of energy and in wear and tear of stopping and starting a heavy train is as great as the cost of running an engine light for several miles.

There are two ways in which it is *possible* to economize in the use of pusher engines. (a) When the traffic of a road is

so very light that a pusher engine will not be kept reasonably busy on the pusher grade it *may* be worth while to place a siding long enough for the longest trains both at top and bottom of the pusher grade and then take up the train in sections. Perhaps the worst objection to this method is the time lost while the engine runs the extra mileage, but with such very light traffic roads a little time more or less is of small consequence. On light traffic roads this method of surmounting a heavy grade will be occasionally adopted even if pushers are never used. If the traffic is fluctuating, the method has the advantage of only requiring such operation when it is needed and avoiding the purchase and operation of a pusher engine which has but little to do and which might be idle for a considerable proportion of the year. (b) The second possible method of economizing is only practicable when a pusher grade begins or ends at or near a station yard where switching-engines are required. In such cases there is a possible economy in utilizing the switching-engines as pushers, especially when the work in each class is small, and thus obtain a greater useful mileage. But such cases are special and generally imply small traffic.

A telegraph-station at top and bottom of a pusher grade is generally indispensable to effective and safe operation.

449. Length of a pusher grade. The virtual length of the pusher grade, as indicated by the mileage of the pusher engine, is always somewhat in excess of the true length of the grade as shown on the profile, and sometimes the excess length is very great. If a station is located on a lower grade within a mile or so of the top or bottom of a pusher grade, it will ordinarily be advisable to couple or uncouple at or near the station, since the telegraph-station, switching, and signaling may be more economically operated at a regular station. If the extra engine is coupled on ahead of the through engine (as is sometimes required by law for passenger trains) the uncoupling at the top of the grade may be accomplished by running the assistant engine ahead at greater speed after it is uncoupled, and, after running it on a siding, clearing the track for the train. But this requires considerable extra track at the top of the grade. Therefore, when estimating the length of the pusher grade, the most desirable position for the terminal sidings must be studied and the length determined accordingly rather than

by measuring the mere length of the grade on the profile. Of course these odd distances are always *excess*; the coupling or uncoupling should not be done while on the grade.

450. The cost of pusher-engine service. When we analyze the elements of cost, we will find that many of them are dependent only on time, while others are dependent upon mileage. Still others are dependent on both. Very much will depend on the constancy of the service, and this in turn depends on the train schedule and on a variety of local conditions which must be considered for each particular case. The effect of a pusher-engine on maintenance of way may be considered to be the same as the cost of an additional engine to handle a given traffic, as developed in § 198. The same total allowance for the expenses of maintenance of way (7.45%) will therefore be made. Although the cost of repairs and renewals of engines is evidently a function of the mileage, and would therefore be somewhat less for a pusher-engine which did little work than for an engine which was worked to the limit of its capacity, yet it is only safe to make the same allowance as for other engines. Other items of maintenance of equipment are evidently to be ignored. The item of wages of enginemen will evidently depend upon the system employed on the particular road. Whatever the precise system the general result is to pay the enginemen as much in wages as the average payment for regular service, and therefore the full allowance for Item 80 will be made. Similarly we must allow the full cost of the items for engine supplies. While the engine is doing its heavy work in climbing up the grade, the consumption of fuel and water is certainly greater than the average; but, on the other hand, on the return trip, when the engine is running light, it probably runs for a considerable portion of the distance actually without steam, and therefore the consumption of fuel and water will nearly, if not quite, average the consumption for an engine running up and down grade along the whole line. That portion of fuel consumption which is due to radiation, blowing-off steam, and the many other causes previously enumerated, will be the same regardless of the work done. We therefore allow 100% for all of these items of engine supplies. In general we must add 100% for Items 90, 91, and 94, the cost of switchmen and telegraphic service. While there might be cases where there would be no actual addition

TABLE XXIX.—COST FOR EACH MILE OF PUSHER-ENGINE SERVICE.

Item number.	Item (abbreviated).	Normal average.	Per cent affected.	Cost per engine mile, per cent.
2-6, 9	Track material, labor, bridges. .	14.89%	50	7.45
25-27	Steam locomotives.....	8.61	100	8.61
80, 81	Road enginemen and engine-house expenses.....	7.80	100	7.80
82-85	Fuel and other engine supplies..	11.41	100	11.41
90, 91, 194	{ Signaling, flagmen, and telegraph.....	1.17	100	1.17
				45.05

to the pay-rolls or the operating expenses on account of these items, we are not justified in general in neglecting to add the full quota for such service. Collecting these items we will have 45.05% of the average cost of a train-mile for the cost of each mile run by the pusher engine. Using the same figure as before, \$1.50, for the cost of one train mile, we have 67.57 c. for each mile run.

451. Numerical comparison of pusher and through grades. In § 445 the computation was made of the desirability of reducing a 1.6% ruling grade to a 1.2% grade. Suppose it is found that by keeping the 1.6% grades as pusher grades having a total length of 20 miles on a 100-mile division, the other grades may be reduced to a grade not exceeding 0.713% (the corresponding through grade) for an expenditure of \$200,000. Will it pay? The saving by cutting down trains from 8 to 4, computed as before, would be (see § 445), $4 \times \$0.6245 \times 100 \times 365 = \$91,177$. But this saving is only accomplished by the employment of pushers making four round trips over 20 miles of pusher grades at a cost of $4 \times 20 \times 2 \times 365 \times \$0.6757 = \$39,461$.

The net annual saving is therefore \$51,716, which when capitalized at 5% = \$1,034,320.

The above estimate probably has this defect. The total daily pusher-engine mileage is but $2 \times 4 \times 20 = 160$, scarcely work enough for two pushers. Unless the pusher grades were bunched into two groups of about 10 miles each, two pusher engines could not do the work. If the number of trains was

much larger, then the above method of calculation would be more exact even though the 20 miles of pusher grade was divided among four or five different grades. Therefore with the above data the annual cost of the pusher service would probably be much more, say, \$60,000, and the annual saving about \$30,000, which would justify an expenditure of \$600,000. But even this would very amply justify the assumed expenditure of \$200,000 which would accomplish this result.

The above computation is but an illustration of the general truth which has been previously stated. In spite of the uncertainties and the variations of many items in the above estimates it will generally be possible to make a computation which will show unquestionably, as in the above instance, what is the best and the most economical method of procedure. When the capitalized valuations of both methods are so nearly equal that a proper choice is more difficult, the question will frequently be determined by the relative ease of raising additional capital.

BALANCE OF GRADES FOR UNEQUAL TRAFFIC.

452. Nature of the subject. It sometimes happens, as when a road runs into a mountainous country for the purpose of hauling therefrom the natural products of lumber or minerals, that the heavy grades are all in one direction—that the whole line consists of a more or less unbroken climb having perhaps a few comparatively level stretches, but no down grade (except possibly a slight sag) in the direction of the general up grade. With such lines this present topic has no concern. But the majority of railroads have termini at nearly the same level (500 feet in 500 miles has no practical effect on grade) and consist of up and down grades in nearly equal amounts and rates. The general rate of ruling grade is determined by the character of the country and the character and financial backing of the road to be built. It is always possible to reduce the grade at some point by “development” or in general by the expenditure of more money. It has been tacitly assumed in the previous discussions that when the ruling grade has been determined all grades in either direction are cut down to that limit. If the traffic in both directions were the same this would be the proper policy and sometimes is so. But it has developed,

especially on the great east and west trunk lines, that the *weight* of the eastbound freight traffic is enormously greater than that of the westbound—that westbound trains consist very largely of “empties” and that an engine which could haul twenty loaded cars up a given grade in eastbound traffic could haul the same cars empty up a much higher grade when running west. As an illustration of the large disproportion which may exist, the eastbound ton-mileage on the P. R. R. between the years 1851 and 1885 was 3.7 times the westbound ton-mileage. Between the years 1876 and 1880 the ratio rose to more than 4.5 to 1. On such a basis it is as important and necessary to obtain, say, a 0.6% ruling grade against the eastbound traffic as to have, say, a 1.0% grade against the westbound traffic. This is the basis of the following discussion. It now remains to estimate the probable ratio of the traffic in the two directions and from that to determine the proper “balance” of the opposite ruling grades.

453. Computation of the theoretical balance. Assume first, for simplicity, that the exact business in either direction is accurately known. A little thought will show the truth of the following statements.

1. The locomotive and passenger-car traffic in both directions is equal.

2. Except as a road may carry emigrants, the passenger traffic in both directions is equal. Of course there are innumerable individual instances in which the return trip is made by another route, but it is seldom if ever that there is any marked tendency to uniformity in this. Considering that a car load of, say, 50 passengers at 150 pounds apiece weigh but 7500 pounds, which is $\frac{1}{6}$ of the 45000 pounds which the car may weigh, even a considerable variation in the number of passengers will not appreciably affect the hauling of cars on grades. On parlor-cars and sleepers the ratio of live load to dead load (say 20 passengers, 3000 pounds, and the car, 75000 pounds) is even more insignificant. The effect of passenger traffic on balance of grades may therefore be disregarded.

3. Empty cars have a greater resistance *per ton* than loaded cars. Therefore in computing the hauling capacity of a locomotive hauling so many tons of “empties,” a larger figure must be used for the ordinary tractive resistances—say four pounds per ton greater.

4. Owing to greater or less imperfections of management a small percentage of cars will run empty or but partly full in the direction of greatest traffic.

5. Freight having great bulk and weight (such as grain, lumber, coal, etc.) is run from the rural districts toward the cities and manufacturing districts.

6. The return traffic—manufactured products—although worth as much or more, do not weigh as much.

As a simple numerical illustration assume that the weight of the cars is $\frac{1}{3}$ and the live load $\frac{2}{3}$ of the total load when the cars are "full"—although not loaded to their absolute limit of capacity. Assume that the relative weight of live load to be hauled in the other direction is but $\frac{1}{3}$; assume that the grade against the heaviest traffic is 0.9%. Since the tractive resistance per ton is considerably greater in the case of unloaded cars than it is in the case of loaded cars, allowance must be made for this in calculating the train resistance. Mr. A. C. Dennis, of the Canadian Pacific Railway Company, has made some elaborate tests of train resistance for trains which were alternately loaded and empty, and found that the tractive resistance of loaded cars was very uniform at 4.7 pounds per ton, when the weight of the empty cars was $\frac{1}{3}$ of the total weight. He also found that the tractive resistance of empty cars was very uniform at 8.9 pounds per ton. Although the live load capacity of a box-car is usually considerably more than twice the weight of the empty car, it will probably coincide more nearly with actual running conditions to consider that the live load is just twice the dead load. Assume that these loads are being hauled by a consolidation engine with a total weight, including engine and tender, of 107 tons, of which 106000 pounds is on the drivers. We will assume that the tractive resistance of the locomotive is likewise 4.7 pounds per ton. On the 0.9% grade, the grade resistance will be 18 pounds per ton, and therefore the total resistance is 22.7 pounds per ton. Assume that this engine is working with a tractive adhesion of $\frac{1}{4}$; the tractive power at the circumference of the drivers will be $\frac{1}{4}$ of 106000 pounds, or 26500 pounds. Dividing this by 22.7, we obtain 1167 as the gross load of the train in tons. Subtracting the weight of the locomotive, 107 tons, we have 1060 tons as the weight of the loaded cars which could be hauled by this locomotive up a 0.9% grade, assuming an adhesion of $\frac{1}{4}$. Since

the traffic in the other direction is but $\frac{1}{3}$, we will assume that $\frac{2}{3}$ of the return cars are empty. We then have 353 tons of loaded cars with a locomotive weighing 107 tons, and 236 tons of empty cars in the return train. The loaded cars with the locomotive will weigh 460 tons, and their tractive resistance will be 4.7 pounds per ton, or 2162 pounds. The 236 tons of empty cars will have a resistance of 8.9 pounds per ton, or a total tractive resistance of 2100 pounds. This makes a total of 4262 pounds of tractive resistance. Subtracting this from the 26500 of total adhesion of the drivers, we have left 22238 as the amount of pull available for grade. But the return train weighs 696 tons. Dividing this into 22238, we find that 32 pounds per ton is available for grade, which is the resistance on a 1.60% grade. Therefore, *under the above conditions*, a 0.9% grade against the heaviest traffic will correspond with a 1.60% grade against the lighter traffic.

Of course these figures will be slightly modified by variations in the assumptions as to the tractive resistance of loaded and unloaded cars, and more especially by variations in the ratio of live load to dead load in the two directions. Therefore no great accuracy can be claimed for the ratio of these two grades in opposite directions, nevertheless the above calculation shows unmistakably that under the given conditions, a very considerable variation in the rate of grade in opposite directions is not only justifiable, but a neglect to allow for it would be a great economic error.

454. Computation of relative traffic. Some of the principal elements have already been referred to, but in addition the following facts should be considered.

(a) The greatest disparity in traffic occurs through the handling of large amounts of coal, lumber, iron ore, grain, etc. On roads which handle but little of these articles or on which for local reasons coal is hauled one way and large shipments of grain the other way the disparity will be less and will perhaps be insignificant.

(b) A marked change in the development of the country may, and often does, cause a marked difference in the disparity of traffic. The heaviest traffic (in mere weight) is always toward manufacturing regions and away from agricultural regions. But when a region, from being purely agricultural or mineral, becomes largely manufacturing, or when a manufacturing region

develops an industry which will cause a growth of heavy freight traffic from it, a marked change in the relative freight movement will be the result.

(c) Very great fluctuations in the relative traffic may be expected for prolonged intervals.

(d) An estimate of the relative traffic may be formed by the same general method used in computing the total traffic of the road (see § 373, Chap. XIX) or by noting the relative traffic on existing roads which may be assumed to have practically the same traffic as the proposed road will obtain.

CHAPTER XXIV.

THE IMPROVEMENT OF OLD LINES.

455. Classification of improvements. The improvements here considered are only those of alignment—horizontal and vertical. Strictly there is no definite limit, either in kind or magnitude, to the improvements which may be made. But since a railroad cannot ordinarily obtain money, even for improvements, to an amount greater than some small proportion of the previously invested capital, it becomes doubly necessary to expend such money to the greatest possible advantage. It has been previously shown that securing additional business and increasing the train load are the two most important factors in increasing dividends. After these, and of far less importance, come reductions of curvature, reductions of distance (frequently of doubtful policy, see Chap. XXI, § 414), and elimination of sags and humps. These various improvements will be briefly discussed.

(a) **Securing additional business.** It is not often possible by any small modification of alignment to materially increase the business of a road. The cases which do occur are usually those in which a gross error of judgment was committed during the original construction. For instance, in the early history of railroad construction many roads were largely aided by the towns through which the road passed, part of the money necessary for construction being raised by the sale of bonds, which were assumed or guaranteed and subsequently paid by the towns. Such aid was often demanded and exacted by the promoters. Instances are not unknown where a failure to come to an agreement has caused the promoters to deliberately pass by the town at a distance of some miles, to the mutual disadvantage of the road and the town. If the town subsequently grew in spite of this disadvantage, the *annual* loss of business might readily amount to more than the original sum in dispute.

Such an instance would be a legitimate opportunity for study of the advisability of a re-location.

As another instance (the original location being justifiable) a railroad might have been located along the bank of a considerable river too wide to be crossed except at considerable expense. When originally constructed the enterprise would not justify the two extra bridges needed to reach the town. A growth in prosperity and in the business obtainable might subsequently make such extra expense a profitable investment.

(b) **Increasing the train load.** On account of its importance this will be separately considered in § 458 *et seq.*

(c) **Reductions in curvature and distance and the elimination of sags and humps.** The financial value of these improvements has already been discussed in Chapters XXI, XXII, and XXIII. Such improvements are constantly being made by all progressive roads. The need for such changes occurs in some cases because the original location was very faulty, the revised location being no more expensive than the original, and in other cases because the original location was the best that was then financially possible and because the present expanded business will justify a change.

(d) **Changing the location of stations or of passing sidings.** The station may sometimes be re-located so as to bring it nearer to the business center and thus increase the business done. But the principal reasons for re-locating stations or passing sidings is that starting trains may have an easier grade on which to overcome the additional resistances of starting. Such changes will be discussed in detail in § 460.

456. Advantages of re-locations. There are certain undoubted advantages possessed by the engineer who is endeavoring to improve an old line.

(a) The gross traffic to be handled is definitely known.

(b) The actual cost per train-mile for that road (which *may* differ very greatly from the average) is also known, and therefore the value of the proposed improvement can be more accurately determined.

(c) The actual performance of such locomotives as are used on the road may be studied at leisure and more reliable data may be obtained for the computations.

457. Disadvantages of re-locations. The disadvantages are generally more apparent and frequently appear practically

insuperable—more so than they prove to be on closer inspection. (a) It frequently means the abandonment of a greater or less length of old line and the construction of new line. At first thought it might seem as if a change of line such as would permit an increase of train-load of 50 or perhaps 100% could never be obtained, or at least that it could not be done except at an impracticable expense. On the contrary a change of 10% of the old line is frequently all that is necessary to reduce the grades so that the train-loads hauled by one engine may be nearly if not quite doubled. And when it is considered that the cost of a road to sub-grade is generally not more than one-third of the total cost of construction and equipment per mile, it becomes plain that an expenditure of but a small percentage of the original outlay, expended where it will do the most good, will often suffice to increase enormously the earning capacity.

(b) One of the most difficult matters is to convince the financial backers of the road that the proposed improvement will be justifiable. The cause is simple. The disadvantages of the original construction lie in the large increase of certain items of expense which are necessary to handle a given traffic. And yet the fact that the expenditures are larger than they need be are only apparent to the expert, and the fact that a saving may be made is considered to be largely a matter of opinion until it is demonstrated by actual trial. On the other hand the cost of the proposed changes is definite, and the very fact that the road has been uneconomically worked and is in a poor financial condition makes it difficult to obtain money for improvements.

(c) The legal right to abandon a section of operated line and thus reduce the value of some adjoining property has sometimes been successfully attacked. A common instance would be that of a factory which was located adjoining the right of way for convenience of transportation facilities. The abandonment of that section of the right of way would probably be fatal to the successful operation of the factory. The objection may be largely eliminated by the maintenance of the old right of way as a long siding (although the business of the factory might not be worth it), but it is not always so easy of solution, and this phase of the question must always be considered.

REDUCTION OF VIRTUAL GRADE.

458. Obtaining data for computations. As developed in the last chapter (§§ 432–434) the real object to be attained is the reduction of the *virtual grade*. The method of comparing grades under various assumed conditions was there discussed. When the road is still “on paper” some such method is all that is possible; but when the road is in actual operation the virtual grade of the road at various critical points, with the rolling stock actually in use, may be determined by a simple test and the effect of a proposed change may be reliably computed. Bearing in mind the general principle that the virtual grade line is the locus of points determined by adding to the actual grade profile ordinates equal to the velocity head of the train, it only becomes necessary to measure the velocity at various points. Since the velocity is *not* usually uniform, its precise determination at any instant is almost impossible, but it will generally be found to be sufficiently precise to assume the velocity to be uniform for a short distance, and then observe the time required to pass that short space. Suppose that an ordinary watch is used and the time taken to the nearest second. At 30 miles per hour, the velocity is 44 feet per second. To obtain the time to within 1%, the time would need to be 100 seconds and the space 4400 feet. But with variable velocity there would be too great error in assuming the velocity as uniform for 4400 feet or for the time of 100 seconds. Using a stop-watch registering fifths of a second, a 1% accuracy would require but 20 seconds and a space of 880 feet, at 30 miles per hour. Wellington suggests that the space be made 293 feet 4 inches, or $\frac{1}{18}$ of a mile; then the speed in miles per hour equals $200 \div s$, in which s is the time in seconds required to traverse the 293' 4". For instance, suppose the time required to pass the interval is 12.5 seconds. $\frac{1}{18}$ mile in 12.5 seconds = one mile in 225 seconds, or 16 miles per hour. But likewise $200 \div 12.5 = 16$, the required velocity. The following features should be noted when obtaining data for the computations:

(a) All critical grades on the road should be located and their profiles obtained—by a survey if necessary.

(b) At the bottom and top of all long grades (and perhaps at intermediate points if the grades are very long) spaces of known

length (preferably $293\frac{1}{2}$ feet) should be measured off and marked by flags, painted boards, or any other serviceable targets.

(c) Provided with a stop-watch marking fifths of seconds the observer should ride on the trains affected by these grades and note the exact interval of time required to pass these spaces. If the space is $293\frac{1}{2}$ feet, the velocity in miles per hour $= 200 \div$ interval in seconds. In general,

$$V = \frac{\text{distance in feet} \times 3600}{\text{time in seconds} \times 5280}.$$

(d) Since these critical grades are those which require the greatest tax on the power of the locomotive, the conditions under which the locomotive is working must be known—i.e., the steam pressure, point of cut-off, and position of the throttle. Economy of coal consumption as well as efficient working at high speeds requires that steam be used expansively (using an early cut-off), and even that the throttle be partly closed; but when an engine is slowly climbing up a maximum grade with a full load it is not exerting its maximum tractive power unless it has its maximum steam pressure, wide-open throttle, and is cutting off nearly at full stroke. These data must therefore be obtained so as to know whether the engine is developing at a critical place all the tractive force of which it is capable. The condition of the track (wet and slippery or dry) and the approximate direction and force of the wind should be noted with sufficient accuracy to judge whether the test has been made under ordinary conditions rather than under conditions which are exceptionally favorable or unfavorable.

(e) The train-loading should be obtained as closely as possible. Of course the dead weight of the cars is easily found, and the records of the freight department will usually give the live load with all sufficient accuracy.

459. Use of the data obtained. A very brief inspection of the results, freed from refined calculations or uncertainties, will demonstrate the following truths:

(a) If, on a uniform grade, the velocity increases, it shows that, under those conditions of engine working, the load is less than the engine can handle on that grade

(b) If the velocity decreases, it shows that the load is greater than the engine can handle on an indefinite length of such

grade. It shows that such a grade is being operated by momentum. From the rate of decrease of velocity the maximum practicable length of such a grade (starting with a given velocity) may be easily computed.

(c) By combining results under different conditions of grade but with practically the same engine working, the tractive power of the engine may be determined (according to the principles previously demonstrated) for any grade and velocity. For example: On an examination of the profile of a division of a road the maximum grade was found to be 1.62% (85.54 feet per mile). At the bottom and near the top of this grade two lengths of 293' 4" are laid off. The distance between the centers of these lengths is 6000 feet. A freight train moving up the grade is timed at $9\frac{2}{3}$ seconds on the lower stretch and $7\frac{3}{4}$ seconds on the upper. These times correspond to $\frac{200}{9.4}$ and $\frac{200}{7.6}$ or 21.3 and 26.3 miles per hour respectively. It is at once observed that the velocity has increased and that the engine could draw even a heavier load up such a grade for an indefinite distance. How much heavier might the load be?

For simplicity we will assume that the conditions were normal, neither exceptionally favorable nor unfavorable, and that the engine was worked to its maximum capacity. The engine is a "consolidation" weighing 128700 pounds, with 112600 pounds on the drivers. The train-load behind the engine consists of ten loaded cars weighing 465 tons and eleven empties weighing 183 tons, thus making a total train-weight of 712 tons. Applying Eq. 140, we find that the *additional* force which the engine has actually exerted per ton in increasing the velocity from 21.3 to 26.3 miles per hour in a distance of 6000 feet is

$$P = \frac{70.224}{6000}(26.3^2 - 21.3^2) = 2.78 \text{ pounds per ton}$$

The grade resistance on a 1.62% grade is 32.4 pounds per ton. The average train resistance may be computed similarly to the method adopted in § 439:

465	}	tons at 4.7 pounds per ton = 2486 pounds						
64								
183	"	"	8.9	"	"	"	= 1629	"
—	"						—	
712							4115	"

The average tractive resistance is therefore $4115 \div 712 = 5.78$ pounds per ton. Adding the grade resistance (32.4) we have a total train resistance of 38.18 pounds per ton. But, computing from the increase in velocity, the locomotive is evidently exerting a pull of 2.78 pounds per ton in excess of the computed required pull on that grade, or a total pull of 40.96 pounds per ton. Therefore the train load might have been increased proportionately and might have been made

$$712 \times \frac{2.78 + 38.18}{38.18} = 764 \text{ tons.}$$

This shows that 52 tons additional might have been loaded on to the train, or say, three more empties or one additional loaded car.

A pull of 40.96 pounds per ton means a total adhesion at the drivers of 29164 pounds, which is about 26% of the weight on the drivers—112600 pounds. This indicates average conditions as to traction, although better conditions than can be depended on for regular service.

The above calculation should of course be considered simply as a "single observation." The performance of the same engine on the same grade (as well as on many other grades) on succeeding days should also be noted. It may readily happen that variations in the condition of the track or of the handling of the engine may make considerable variation in the results of the several calculations, but when the work is properly done it is always possible to draw definite and very positive deductions.

460. Reducing the starting grade at stations. The resistance to starting a train is augmented from two causes: (a) the tractive resistances are usually about 20 pounds per ton instead of, say, 6 pounds, and (b) the inertia resistance must be overcome. The inertia resistance of a freight train (see § 347) which is expected to attain a velocity of 15 miles per hour in a distance of 1000 feet is (see Eq. 140)

$$P = \frac{70.224}{1000} (15^2 - 0) = 15.8 \text{ pounds per ton, which is the equivalent of a } 0.79\% \text{ grade.}$$

Adding this to a grade which nearly or quite equals the ruling grade, it virtually creates a new and higher ruling grade. Of course that additional force can be greatly reduced at the expense of slower acceleration, but even

this cannot be done indefinitely, and an acceleration to only 15 miles per hour in 1000 feet is as slow as should be allowed for. With perhaps 14 pounds per ton additional tractive resistance, we have about 30 pounds per ton additional—equiva-

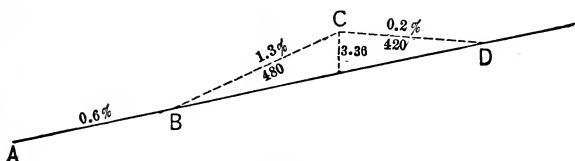


FIG. 217.

lent to a 1.5% grade. Instances are known where it has proven wise to create a *hump* (in what was otherwise a uniform grade) at a station. The effect of this on high-speed passenger trains moving *up* the grade would be merely to reduce their speed very slightly. No harm is done to trains moving *down* the grade. Freight trains moving *up* the grade and intending to stop at the station will merely have their velocity reduced as they approach the station and will actually save part of the wear and tear otherwise resulting from applying brakes. When the trains start they are assisted by the short down grade, just where they need assistance most. Even if the grade *CD* is still an up grade, the pull required at starting is *less* than that required on the uniform grade by an amount equal to 20 times the difference of the grade in per cent.

APPENDIX.

THE ADJUSTMENTS OF INSTRUMENTS.

THE accuracy of instrumental work may be vitiated by any one of a large number of inaccuracies in the geometrical relations of the parts of the instruments. Some of these relations are so apt to be altered by ordinary usage of the instrument that the makers have provided adjusting-screws so that the inaccuracies may be readily corrected. There are other possible defects, which, however, will seldom be found to exist, provided the instrument was properly made and has never been subjected to treatment sufficiently rough to distort it. Such defects, when found, can only be corrected by a competent instrument-maker or repairer.

A WARNING is necessary to those who would test the accuracy of instruments, and especially to those whose experience in such work is small. Lack of skill in handling an instrument will often indicate an apparent error of adjustment when the real error is very different or perhaps non-existent. It is always a safe plan when testing an adjustment to note the amount of the apparent error; then, beginning anew, make another independent determination of the amount of the error. When two or more *perfectly independent* determinations of such an error are made it will generally be found that they differ by an appreciable amount. The *differences* may be due in variable measure to careless inaccurate manipulation and to instrumental *defects* which are wholly independent of the particular test being made. Such careful determinations of the amounts of the errors are generally advisable in view of the next paragraph.

DO NOT DISTURB THE ADJUSTING-SCREWS ANY MORE THAN NECESSARY. Although metals are apparently rigid, they are really elastic and yielding. If some parts of a complicated mechanism, which is held together largely by friction, are subjected to greater internal stresses than other parts of the mech-

anism, the jarring resulting from handling will frequently cause a slight readjustment in the parts which will tend to more nearly equalize the internal stresses. Such action frequently occurs with the adjusting mechanism of instruments. One screw may be strained more than others. The friction of parts may prevent the opposing screw from *immediately* taking up an equal stress. Perhaps the adjustment appears perfect under these conditions. Jarring diminishes the friction between the parts, and the unequal stresses tend to equalize. A motion takes place which, although microscopically minute, is sufficient to indicate an error of adjustment. A readjustment made by unskillful hands may not make the final adjustment any more perfect. The frequent shifting of adjusting-screws wears them badly, and when the screws are worn it is still more difficult to keep them from moving enough to vitiate the adjustments. It is therefore preferable in many cases to refrain from disturbing the adjusting-screws, especially as the accuracy of the work done is not *necessarily* affected by errors of adjustment, as may be illustrated:

(a) Certain operations are *absolutely* unaffected by certain errors of adjustment.

(b) Certain operations are so slightly affected by certain *small* errors of adjustment that their effect may properly be neglected.

(c) Certain errors of adjustment may be readily allowed for and neutralized so that no error results from the use of the unadjusted instrument. Illustrations of all these cases will be given under their proper heads.

ADJUSTMENTS OF THE TRANSIT.

1. *To have the plate-bubbles in the center of the tubes when the axis is vertical.* Clamp the upper plate and, with the lower clamp loose, swing the instrument so that the plate-bubbles are parallel to the lines of opposite leveling-screws. Level up until both bubbles are central. Swing the instrument 180° . If the bubbles again settle at the center, the adjustment is perfect. If either bubble does not settle in the center, move the leveling-screws until the bubble is *half-way* back to the center. Then, before touching the adjusting-screws, note carefully the position of the bubbles and observe whether the bubbles always settle at the *same* place in the tube, no matter to what position the in-

strument may be rotated. When the instrument is so leveled, the axis is truly vertical and the discrepancies between this constant position of the bubbles and the centers of the tubes measure the errors of adjustment. By means of the adjusting-screws bring each bubble to the center of the tube. If this is done so skillfully that the true level of the instrument is not disturbed, the bubbles should settle in the center for all positions of the instrument. Under unskillful hands, two or more such trials may be necessary.

When the plates are not horizontal, the measured angle is greater than the true horizontal angle by the difference between the measured angle and its projection on a horizontal plane. When this angle of inclination is small, the difference is insignificant. Therefore when the plate-bubbles are *very nearly* in adjustment, the error of measurement of horizontal angles may be far within the lowest unit of measurement used. A *small* error of adjustment of the plate-bubble *perpendicular* to the telescope will affect the horizontal angles by only a small proportion of the error, which will be perhaps imperceptible. Vertical angles will be affected by the same insignificant amount. A *small* error of adjustment of the plate-bubble *parallel* to the telescope will affect horizontal angles very slightly, but will affect vertical angles by the full amount of the error.

All error due to unadjusted plate-bubbles may be avoided by noting in what positions in the tubes the bubbles will remain fixed for all positions of azimuth and then keeping the bubbles adjusted to these positions, for the axis is then truly vertical. It will often save time to work in this way temporarily rather than to stop to make the adjustments. This should especially be done when accurate vertical angles are required.

When the bubbles are truly adjusted, they should remain stationary regardless of whether the telescope is revolved with the upper plate loose and the lower plate clamped or whether the whole instrument is revolved, the plates being clamped together. If there is any appreciable difference, it shows that the two vertical axes or "centers" of the plates are not concentric. This may be due to cheap and faulty construction or to the excessive wear that may be sometimes observed in an old instrument originally well made. In either case it can only be corrected by a maker.

2. *To make the revolving axis of the telescope perpendicular to the vertical axis of the instrument.* This is best tested by using a long plumb-line, so placed that the telescope must be pointed upward at an angle of about 45° to sight at the top of the plumb-line and downward about the same amount, if possible, to sight at the lower end. The vertical axis of the transit must be made truly vertical. Sight at the upper part of the line, clamping the horizontal plates. Swing the telescope down and see if the cross-wire again bisects the cord. If so, the adjustment is *probably* perfect (a conceivable exception will be

noted later); if not, raise or lower one end of the axis by means of the adjusting-screws, placed at the top of one of the standards, until the cross-wire will bisect the cord both at top and bottom. The plumb-bob may be steadied, if necessary, by hanging it in a pail of water. As many telescopes cannot be focused on an object nearer than 6 or 8 feet from the telescope, this method requires a long plumb-line swung from a high point, which may be inconvenient.

Another method is to set up the instrument about 10 feet from a high wall. After leveling, sight at some convenient mark high up on the wall. Swing the telescope down and make a mark (when working alone some convenient natural mark may generally be found) low down on the wall. Plunge the telescope and revolve the instrument about its vertical axis and again sight at the upper mark. Swing down to the lower mark. If the wire again bisects it, the adjustment is perfect. If not, fix a point *half-way* between the two positions of the lower mark. The plane of this point, the upper point, and the center of the instrument is truly vertical. Adjust the axis to these upper and lower points as when using the plumb-line.

3. *To make the line of collimation perpendicular to the revolving axis of the telescope.* With the instrument level and the telescope nearly horizontal point at some well-defined point at a distance of 200 feet or more. Plunge the telescope and establish a point in the opposite direction. Turn the whole instrument about the vertical axis until it again points at the first mark. Again plunge to "direct position" (*i.e.*, with the level-tube *under* the telescope). If the vertical cross-wire again points at the second mark, the adjustment is perfect. If not, the error is *one-fourth* of the distance between the two positions of the second mark. Loosen the capstan screw on one side of the telescope and tighten it on the other side until the vertical wire is set at the one-fourth mark. Turn the whole instrument by means of the tangent screw until the vertical wire is *midway* between the two positions of the second mark. Plunge the telescope. If the adjusting has been skillfully done, the cross-wire should come exactly to the first mark. As an "erecting eyepiece" reinverts an image already inverted, the ring carrying the cross-wires must be moved in the *same* direction as the *apparent* error in order to correct that error.

The necessity for the third adjustment lies principally in the practice of producing a line by plunging the telescope, but when this is required to be done with great accuracy it is always better to obtain the forward point by reversion (as described above for making the test) and take the *mean* of the two forward points. Horizontal and vertical angles are practically unaffected by *small* errors of this adjustment, unless, in the case of horizontal angles, the vertical angles to the points observed are very different.

Unnecessary motion of the adjusting-screws may sometimes be avoided by carefully establishing the forward point on line by repeated reversions of the instrument, and thus determining by repeated trials the exact amount of the error. *Differences* in the amount of error determined would be evidence of inaccuracy in manipulating the instrument, and would show that an adjustment based on the first trial would *probably* prove unsatisfactory.

The 2d and 3d adjustments are mutually dependent. If either adjustment is badly out, the other adjustment cannot be made except as follows:

(a) The second adjustment can be made regardless of the third when the lines to the high point and the low point make *equal* angles with the horizontal.

(b) The third adjustment can be made regardless of the second when the front and rear points are *on a level* with the instrument.

When both of these requirements are *nearly* fulfilled, and especially when the error of either adjustment is small, no trouble will be found in perfecting either adjustment on account of a small error in the other adjustment.

If the test for the second adjustment is made by means of the plumb-line and the vertical cross-wire intersects the line at all points as the telescope is raised or lowered, it not only demonstrates at once the accuracy of that adjustment, but also shows that the third adjustment is either perfect or has so small an error that it does not affect the second.

4. *To have the bubble of the telescope-level in the center of the tube when the line of collimation is horizontal.* The line of collimation should coincide with the optical axis of the telescope. If the object-glass and eyepiece have been properly centered, the previous adjustment will have brought the vertical cross-wire to the center of the field of view. The horizontal cross-wire should also be brought to the center of the field of view, and the bubble should be adjusted to it.

a. *Peg method.* Set up the transit at one end of a nearly level stretch of about 300 feet. Clamp the telescope with its bubble in the center. Drive a stake vertically under the eyepiece of the transit, and another about 300 feet away. Observe the height of the center of the eyepiece (the telescope being level) above the stake (calling it *a*); observe the reading of the rod when held on the other stake (calling it *b*); take the instrument to the other stake and set it up so that the eyepiece is

vertically over the stake, observing the height, c ; take a reading on the first stake, calling it d . If this adjustment is perfect, then

$$a - d = b - c,$$

$$\text{or} \quad (a - d) - (b - c) = 0.$$

$$\text{Call} \quad (a - d) - (b - c) = 2m.$$

When m is positive, the line points downward;

“ m “ negative, “ “ “ upward.

To adjust: if the line points *up*, sight the horizontal cross-wire (by moving the vertical tangent screw) at a point which is m lower, then adjust the bubble so that it is in the center.

By taking several independent values for a , b , c , and d , a mean value for m is obtained, which is more reliable and which may save much unnecessary working of the adjusting-screws.

b. Using an auxiliary level. When a carefully adjusted level is at hand, this adjustment may sometimes be more easily made by setting up the transit and level, so that their lines of collimation are as nearly as possible at the same height. If a point may be found which is half a mile or more away and which is on the horizontal cross-wire of the level, the horizontal cross-wire of the transit may be pointed directly at it, and the bubble adjusted accordingly. Any slight difference in the heights of the lines of collimation of the transit and level (say $\frac{1}{4}''$) may almost be disregarded at a distance of $\frac{1}{2}$ mile or more, or, if the difference of level would have an appreciable effect, even this may be practically eliminated by making an estimated allowance when sighting at the distant point. Or, if a distant point is not available, a level-rod with target may be used at a distance of (say) 300 feet, making allowance for the carefully determined difference of elevation of the two lines of collimation.

5. Zero of vertical circle. When the line of collimation is truly horizontal and the vertical axis is truly vertical, the reading of the vertical circle should be 0° . If the arc is adjustable, it should be brought to 0° . If it is not adjustable, the *index error* should be observed, so that it may be applied to all readings of vertical angles.

ADJUSTMENTS OF THE WYE LEVEL.

1. To make the line of collimation coincide with the center of the rings. Point the intersection of the cross-wires at some

well-defined point which is at a considerable distance. The instrument need not be level, which allows much greater liberty in choosing a convenient point. The vertical axis should be clamped, and the clips over the wyes should be loosened and raised. Rotate the telescope in the wyes. The intersection of the cross-wires should be continually on the point. If it is not, it requires adjustment. Rotate the telescope 180° and adjust *one-half* of the error by means of the capstan-headed screws that move the cross-wire ring. It should be remembered that, with an erecting telescope, on account of the inversion of the image, the ring should be moved in the direction of the *apparent* error. Adjust the other half of the error with the leveling-screws. Then rotate the telescope 90° from its usual position, sight accurately at the point, and then rotate 180° from that position and adjust any error as before. It may require several trials, but it is necessary to adjust the ring until the intersection of the cross-wires will remain on the point for any position of rotation.

If such a test is made on a very distant point and again on a point only 10 or 15 feet from the instrument, the adjustment may be found correct for one point and incorrect for the other. This indicates that the object-slide is improperly centered. Usually this defect can only be corrected by an instrument-maker. If the difference is very small it may be ignored, but the adjustment should then be made on a point which is at about the mean distance for usual practice—say 150 feet.

If the whole image appears to shift as the telescope is rotated, it indicates that the eyepiece is improperly adjusted. This defect is likewise usually corrected only by the maker. It does not interfere with instrumental accuracy, but it usually causes the intersection of the cross-wires to be eccentric with the field of view.

2. *To make the axis of the level-tube parallel to the line of collimation.* Raise the clips as far as possible. Swing the level so that it is parallel to a pair of opposite leveling-screws and clamp it. Bring the bubble to the middle of the tube by means of the leveling-screws. Take the telescope out of the wyes and replace it end for end, using *extreme care* that the wyes are not jarred by the action. If the bubble does not come to the center, correct *one-half* of the error by the vertical adjusting-screws at one end of the bubble. Correct the other half by the leveling-screws. Test the work by again changing the telescope end for end in the wyes.

Care should be taken while making this adjustment to see

that the level-tube is vertically under the telescope. With the bubble in the center of the tube, rotate the telescope in the wyes for a considerable angle each side of the vertical. If the first half of the adjustment has been made and the bubble moves, it shows that the axis of the wyes and the axis of the level-tube are not in the same vertical plane although both have been made horizontal. By moving one end of the level-tube *sidewise* by means of the horizontal screws at one end of the tube, the two axes may be brought into the same plane. As this adjustment is liable to disturb the other, both should be alternately tested until both requirements are complied with.

By these methods the axis of the bubble is made parallel to the axis of the wyes; and as this has been made parallel to the lines of collimation by means of the previous adjustment, the axis of the bubble is therefore parallel to the line of collimation.

3. *To make the line of collimation perpendicular to the vertical axis.* Level up so that the instrument is approximately level over both sets of leveling-screws. Then, after leveling carefully over one pair of screws, revolve the telescope 180° . If it is not level, adjust half of the error by means of the capstan-headed screw under one of the wyes, and the other half by the leveling-screws. Reverse again as a test.

When the first two adjustments have been accurately made, good leveling may always be done by bringing the bubble to the center by means of the leveling-screws, at every sight if necessary, even if the third adjustment is not made. Of course this third adjustment should be made as a matter of convenience, so that the line of collimation may be always level no matter in what direction it may be pointed, but it is not *necessary* to stop work to make this adjustment every time it is found to be defective.

ADJUSTMENTS OF THE DUMPY LEVEL.

1. *To make the axis of the level-tube perpendicular to the vertical axis.* Level up so that the instrument is approximately level over both sets of leveling-screws. Then, after leveling carefully over one pair of screws, revolve the telescope 180° . If it is not level, adjust *one-half* of the error by means of the adjusting-screws at one end of the bubble, and the other half by means of the leveling-screws. Reverse again as a test.

2. *To make the line of collimation perpendicular to the vertical axis.* The method of adjustment is identical with that for the transit (No. 4, p. 505) except that the cross-wire must be

adjusted to agree with the level-bubble rather than *vice versa*, as is the case with the corresponding adjustment of the transit; i.e., with the level-bubble in the center, raise or lower the horizontal cross-wire until it points at the mark known to be on a level with the center of the instrument.

If the instrument has been well made and has not been distorted by rough usage, the cross-wires will intersect at the center of the field of view when adjusted as described. If they do not, it indicates an error which ordinarily can only be corrected by an instrument-maker. The error may be due to any one of several causes, which are

(a) faulty centering of object-slide;

(b) faulty centering of eyepiece;

(c) distortion of instrument so that the geometric axis of the telescope is not perpendicular to the vertical axis. If the error is only just perceptible, it will not probably cause any error in the work.

EXPLANATORY NOTE ON THE USE OF THE TABLES

The logarithms here given are "five-place," but the last figure sometimes has a special mark over it (*e.g.*, $\bar{6}$) which indicates that one-half a unit in the last place should be *added*. For example

the value	includes all values between
.69586	.6958575000 + and .6958624999...
.6958 $\bar{6}$	.6958625000 + and .6958674999...

The maximum error in any one value therefore does not exceed one-quarter of a fifth-place unit.

When adding or subtracting such logarithms allow a half-unit for such a sign. For example

.69586	.69586	.6958 $\bar{6}$
.10841	.1084 $\bar{1}$	.1084 $\bar{1}$
.1294 $\bar{7}$	.1294 $\bar{7}$	.1294 $\bar{7}$
.9337 $\bar{4}$	.93375	.9337 $\bar{5}$

All other logarithmic operations are performed as usual and are supposed to be understood by the student.

TABLE I.—RADII OF CURVES.

Deg	0°		1°		2°		3°		Deg
Min	Radius.	Log R	Radius.	Log R	Radius.	Log R	Radius.	Log R	Min
0	∞	∞	5729.6	3.75813	2864.9	3.45711	1910.1	3.28105	0
1	343775	5.53627	5635.7	.75095	2841.3	.45351	1899.5	.27864	1
2	171887	5.23524	5544.8	.74389	2818.0	.44993	1889.1	.27625	2
3	114592	5.05915	5456.8	.73694	2795.1	.44639	1878.8	.27387	3
4	85944	4.93421	5371.6	.73010	2772.5	.44287	1868.6	.27151	4
5	68755	4.83730	5288.9	.72336	2750.4	.43939	1858.5	.26915	5
6	57296	4.75812	5208.8	3.71673	2728.5	3.43593	1848.5	3.26681	6
7	49111	.69117	5131.0	.71020	2707.0	.43249	1838.6	.26448	7
8	42972	.63318	5055.6	.70377	2685.9	.42909	1828.8	.26217	8
9	38197	.58203	4982.3	.69743	2665.1	.42571	1819.1	.25986	9
10	34377	.53627	4911.2	.69118	2644.6	.42235	1809.6	.25757	10
11	31252	4.49488	4842.0	3.68502	2624.4	3.41903	1800.1	3.25529	11
12	28648	.45709	4774.7	.67895	2604.5	.41572	1790.7	.25303	12
13	26444	.42233	4709.3	.67296	2584.9	.41245	1781.5	.25077	13
14	24555	.39014	4645.7	.66705	2565.6	.40919	1772.3	.24853	14
15	22918	.36018	4583.8	.66122	2546.6	.40597	1763.2	.24629	15
16	21486	4.33215	4523.4	3.65547	2527.9	3.40276	1754.2	3.24407	16
17	20222	.30582	4464.7	.64979	2509.5	.39958	1745.3	.24186	17
18	19099	.28100	4407.5	.64419	2491.3	.39642	1736.5	.23967	18
19	18093	.25752	4351.7	.63865	2473.4	.39329	1727.8	.23748	19
20	17189	.23524	4297.3	.63319	2455.7	.39017	1719.1	.23530	20
21	16370	4.21405	4244.2	3.62780	2438.3	3.38708	1710.6	3.23314	21
22	15626	.19385	4192.5	.62247	2421.1	.38401	1702.1	.23098	22
23	14947	.17454	4142.0	.61720	2404.2	.38097	1693.7	.22884	23
24	14324	.15606	4092.7	.61200	2387.5	.37794	1685.4	.22670	24
25	13751	.13833	4044.5	.60686	2371.0	.37494	1677.2	.22458	25
26	13222	4.12130	3997.5	3.60178	2354.8	3.37195	1669.1	3.22247	26
27	12732	.10491	3951.5	.59676	2338.8	.36899	1661.0	.22037	27
28	12278	.08911	3906.6	.59180	2323.0	.36604	1653.0	.21827	28
29	11854	.07387	3862.7	.58689	2307.4	.36312	1645.1	.21619	29
30	11459	.05915	3819.8	.58204	2292.0	.36021	1637.3	.21412	30
31	11090	4.04491	3777.9	3.57724	2276.8	3.35733	1629.5	3.21206	31
32	10743	.03112	3736.8	.57250	2261.9	.35446	1621.8	.21000	32
33	10417	.01776	3696.6	.56780	2247.1	.35162	1614.2	.20796	33
34	10111	4.00479	3657.3	.56316	2232.5	.34879	1606.7	.20593	34
35	9822.2	3.99221	3618.8	.55856	2218.1	.34598	1599.2	.20390	35
36	9549.3	3.97997	3581.1	3.55401	2203.9	3.34318	1591.8	3.20189	36
37	9291.3	.96807	3544.2	.54951	2189.8	.34041	1584.5	.19988	37
38	9046.7	.95649	3508.0	.54506	2176.0	.33765	1577.2	.19789	38
39	8814.8	.94521	3472.6	.54065	2162.3	.33491	1570.0	.19590	39
40	8594.4	.93421	3437.9	.53629	2148.8	.33219	1562.9	.19392	40
41	8384.8	3.92349	3403.8	3.53197	2135.4	3.32949	1555.8	3.19195	41
42	8185.2	.91302	3370.5	.52769	2122.3	.32680	1548.8	.18999	42
43	7994.8	.90281	3337.7	.52345	2109.2	.32412	1541.9	.18804	43
44	7813.1	.89282	3305.7	.51925	2096.4	.32147	1535.0	.18610	44
45	7639.5	.88306	3274.2	.51510	2083.7	.31883	1528.2	.18417	45
46	7473.4	3.87352	3243.3	3.51098	2071.1	3.31621	1521.4	3.18224	46
47	7314.4	.86418	3213.0	.50691	2058.7	.31360	1514.7	.18032	47
48	7162.0	.85503	3183.2	.50287	2046.5	.31101	1508.1	.17842	48
49	7015.9	.84608	3154.0	.49883	2034.4	.30843	1501.5	.17652	49
50	6875.6	.83731	3125.4	.49490	2022.4	.30587	1495.0	.17462	50
51	6740.7	3.82871	3097.2	3.49097	2010.6	3.30332	1488.5	3.17274	51
52	6611.1	.82027	3069.8	.48707	1998.9	.30079	1482.1	.17087	52
53	6486.4	.81200	3042.4	.48321	1987.3	.29827	1475.7	.16900	53
54	6366.3	.80388	3015.7	.47939	1975.9	.29577	1469.4	.16714	54
55	6250.5	.79591	2989.5	.47559	1964.6	.29328	1463.2	.16529	55
56	6138.9	3.78809	2963.7	3.47183	1953.5	3.29081	1457.0	3.16344	56
57	6031.2	.78040	2938.4	.46811	1942.4	.28835	1450.8	.16161	57
58	5927.2	.77285	2913.5	.46441	1931.5	.28590	1444.7	.15978	58
59	5826.8	.76542	2889.0	.46075	1920.7	.28347	1438.7	.15796	59
60	5729.6	.75813	2864.9	.45711	1910.1	.28105	1432.7	.15615	60

TABLE I.—RADII OF CURVES.

Deg	4°		5°		6°		7°		Deg
Min	Radius.	Log R	Radius.	Log R	Radius.	Log R	Radius.	Log R	Min
0	1432.7	3.15615	1146.3	3.05929	955.37	2.98017	819.02	2.91329	0
1	1426.7	.15434	1142.5	.05784	952.72	.97896	817.08	.91226	1
2	1420.8	.15255	1138.7	.05640	950.09	.97776	815.14	.91123	2
3	1415.0	.15076	1134.9	.05497	947.48	.97657	813.22	.91021	3
4	1409.2	.14897	1131.2	.05354	944.88	.97537	811.30	.90918	4
5	1403.5	.14720	1127.5	.05211	942.29	.97418	809.40	.90816	5
6	1397.8	3.14543	1123.8	3.05069	939.72	2.97300	807.50	2.90714	6
7	1392.1	.14367	1120.2	.04928	937.16	.97181	805.61	.90612	7
8	1386.5	.14191	1116.5	.04787	934.62	.97063	803.73	.90511	8
9	1380.9	.14017	1112.9	.04646	932.09	.96945	801.86	.90410	9
10	1375.4	.13843	1109.3	.04506	929.57	.96828	800.00	.90309	10
11	1369.9	3.13669	1105.8	3.04366	927.07	2.96711	798.14	2.90208	11
12	1364.5	.13497	1102.2	.04227	924.58	.96594	796.30	.90107	12
13	1359.1	.13325	1098.7	.04088	922.10	.96478	794.46	.90007	13
14	1353.8	.13154	1095.2	.03949	919.64	.96361	792.63	.89907	14
15	1348.4	.12983	1091.7	.03811	917.19	.96246	790.81	.89807	15
16	1343.2	3.12813	1088.3	3.03674	914.75	2.96130	789.00	2.89708	16
17	1338.0	.12644	1084.8	.03537	912.33	.96015	787.20	.89608	17
18	1332.8	.12475	1081.4	.03400	909.92	.95900	785.41	.89509	18
19	1327.6	.12307	1078.1	.03264	907.52	.95785	783.62	.89410	19
20	1322.5	.12140	1074.7	.03128	905.13	.95671	781.84	.89312	20
21	1317.5	3.11974	1071.3	3.02992	902.76	2.95557	780.07	2.89213	21
22	1312.4	.11808	1068.0	.02857	900.40	.95443	778.31	.89115	22
23	1307.4	.11642	1064.7	.02723	898.05	.95330	776.55	.89017	23
24	1302.5	.11477	1061.4	.02589	895.71	.95217	774.81	.88919	24
25	1297.6	.11313	1058.2	.02455	893.39	.95104	773.07	.88821	25
26	1292.7	3.11150	1054.9	3.02322	891.08	2.94991	771.34	2.88724	26
27	1287.9	.10987	1051.7	.02189	888.78	.94879	769.61	.88627	27
28	1283.1	.10825	1048.5	.02056	886.49	.94767	767.90	.88530	28
29	1278.3	.10663	1045.3	.01924	884.21	.94655	766.19	.88433	29
30	1273.6	.10502	1042.1	.01792	881.95	.94544	764.49	.88337	30
31	1268.9	3.10341	1039.0	3.01661	879.69	2.94433	762.80	2.88241	31
32	1264.2	.10182	1035.9	.01530	877.45	.94322	761.11	.88145	32
33	1259.6	.10022	1032.8	.01400	875.22	.94212	759.43	.88049	33
34	1255.0	.09864	1029.7	.01270	873.00	.94101	757.76	.87953	34
35	1250.4	.09705	1026.6	.01140	870.80	.93991	756.10	.87858	35
36	1245.9	3.09548	1023.5	3.01010	868.60	2.93882	754.44	2.87762	36
37	1241.4	.09391	1020.5	.00882	866.41	.93772	752.80	.87668	37
38	1236.9	.09234	1017.5	.00753	864.24	.93663	751.16	.87573	38
39	1232.5	.09079	1014.5	.00625	862.07	.93554	749.52	.87478	39
40	1228.1	.08923	1011.5	.00497	859.92	.93446	747.89	.87384	40
41	1223.7	3.08769	1008.6	3.00370	857.78	2.93337	746.27	2.87290	41
42	1219.4	.08614	1005.6	.00242	855.65	.93229	744.66	.87196	42
43	1215.1	.08461	1002.7	3.00116	853.53	.93122	743.06	.87102	43
44	1210.8	.08308	999.76	2.99989	851.42	.93014	741.46	.87008	44
45	1206.6	.08155	996.87	.99863	849.32	.92907	739.86	.86915	45
46	1202.4	3.08003	993.99	2.99738	847.23	2.92800	738.28	2.86822	46
47	1198.2	.07852	991.13	.99613	845.15	.92693	736.70	.86729	47
48	1194.0	.07701	988.28	.99488	843.08	.92587	735.13	.86636	48
49	1189.9	.07550	985.45	.99363	841.02	.92480	733.56	.86544	49
50	1185.8	.07400	982.64	.99239	838.97	.92374	732.01	.86451	50
51	1181.7	3.07251	979.84	2.99115	836.93	2.92269	730.45	2.86359	51
52	1177.7	.07102	977.06	.98992	834.90	.92163	728.91	.86267	52
53	1173.6	.06954	974.29	.98869	832.89	.92058	727.37	.86175	53
54	1169.7	.06806	971.54	.98746	830.88	.91953	725.84	.86084	54
55	1165.7	.06658	968.81	.98624	828.88	.91849	724.31	.85992	55
56	1161.8	3.06511	966.09	2.98501	826.89	2.91744	722.79	2.85901	56
57	1157.9	.06365	963.39	.98380	824.91	.91640	721.28	.85810	57
58	1154.0	.06219	960.70	.98258	822.93	.91536	719.77	.85719	58
59	1150.1	.06074	958.03	.98137	820.97	.91433	718.27	.85629	59
60	1146.3	.05929	955.37	.98017	819.02	.91329	716.78	.85538	60

TABLE I.—RADII OF CURVES.

Deg.	8°		9°		10°		11°		Deg.
Min.	Radius.	Log R	Radius.	Log R	Radius.	Log R	Radius.	Log R	Min
0	716.78	2.85538	637.27	2.80432	573.69	2.75867	521.67	2.71739	0
1	715.29	.85448	636.10	.80352	572.73	.75795	520.88	.71674	1
2	713.81	.85358	634.93	.80272	571.78	.75723	520.10	.71608	2
3	712.34	.85268	633.76	.80192	570.84	.75651	519.32	.71543	3
4	710.87	.85178	632.60	.80113	569.90	.75579	518.54	.71478	4
5	709.40	.85089	631.44	.80033	568.96	.75508	517.76	.71413	5
6	707.95	2.85000	630.29	2.79954	568.02	2.75436	516.99	2.71348	6
7	706.49	.84911	629.14	.79874	567.09	.75365	516.21	.71283	7
8	705.05	.84822	627.99	.79795	566.16	.75293	515.44	.71218	8
9	703.61	.84733	626.85	.79716	565.23	.75222	514.68	.71153	9
10	702.17	.84644	625.71	.79637	564.31	.75151	513.91	.71088	10
11	700.75	2.84556	624.58	2.79558	563.38	2.75080	513.15	2.71024	11
12	699.33	.84468	623.45	.79480	562.47	.75009	512.38	.70959	12
13	697.91	.84380	622.32	.79401	561.55	.74939	511.63	.70895	13
14	696.50	.84292	621.20	.79323	560.64	.74868	510.87	.70831	14
15	695.09	.84204	620.09	.79245	559.73	.74798	510.11	.70767	15
16	693.70	2.84117	618.97	2.79167	558.82	2.74727	509.36	2.70702	16
17	692.30	.84029	617.87	.79089	557.92	.74657	508.61	.70638	17
18	690.91	.83942	616.76	.79011	557.02	.74587	507.86	.70575	18
19	689.53	.83855	615.66	.78934	556.12	.74517	507.12	.70511	19
20	688.16	.83768	614.56	.78856	555.23	.74447	506.38	.70447	20
21	686.78	2.83682	613.47	2.78779	554.34	2.74377	505.64	2.70383	21
22	685.42	.83595	612.38	.78702	553.45	.74307	504.90	.70320	22
23	684.06	.83509	611.30	.78625	552.56	.74238	504.16	.70257	23
24	682.70	.83423	610.21	.78548	551.68	.74168	503.42	.70193	24
25	681.35	.83337	609.14	.78471	550.80	.74099	502.69	.70130	25
26	680.01	2.83251	608.06	2.78395	549.92	2.74030	501.96	2.70067	26
27	678.67	.83166	606.99	.78318	549.05	.73961	501.23	.70004	27
28	677.34	.83080	605.93	.78242	548.17	.73892	500.51	.69941	28
29	676.01	.82995	604.86	.78165	547.30	.73823	499.78	.69878	29
30	674.69	.82910	603.80	.78089	546.44	.73754	499.06	.69815	30
31	673.37	2.82825	602.75	2.78013	545.57	2.73685	498.34	2.69752	31
32	672.06	.82740	601.70	.77938	544.71	.73617	497.62	.69690	32
33	670.75	.82656	600.65	.77862	543.86	.73548	496.91	.69627	33
34	669.45	.82571	599.61	.77786	543.00	.73480	496.19	.69565	34
35	668.15	.82487	598.57	.77711	542.15	.73412	495.48	.69503	35
36	666.86	2.82403	597.53	2.77636	541.30	2.73343	494.77	2.69440	36
37	665.57	.82319	596.50	.77561	540.45	.73275	494.07	.69378	37
38	664.29	.82235	595.47	.77486	539.61	.73207	493.36	.69316	38
39	663.01	.82152	594.44	.77411	538.76	.73140	492.66	.69254	39
40	661.74	.82068	593.42	.77336	537.92	.73072	491.96	.69192	40
41	660.47	2.81985	592.40	2.77261	537.09	2.73004	491.26	2.69131	41
42	659.21	.81902	591.38	.77187	536.25	.72937	490.56	.69069	42
43	657.95	.81819	590.37	.77112	535.42	.72869	489.86	.69007	43
44	656.69	.81736	589.36	.77038	534.59	.72802	489.17	.68946	44
45	655.45	.81653	588.36	.76964	533.77	.72735	488.48	.68884	45
46	654.20	2.81571	587.36	2.76890	532.94	2.72668	487.79	2.68823	46
47	652.96	.81489	586.36	.76816	532.12	.72601	487.10	.68762	47
48	651.73	.81406	585.36	.76742	531.30	.72534	486.42	.68701	48
49	650.50	.81324	584.37	.76669	530.49	.72467	485.73	.68640	49
50	649.27	.81243	583.38	.76595	529.67	.72401	485.05	.68579	50
51	648.05	2.81161	582.40	2.76522	528.86	2.72334	484.37	2.68518	51
52	646.84	.81079	581.42	.76449	528.05	.72267	483.69	.68457	52
53	645.63	.80998	580.44	.76376	527.25	.72201	483.02	.68396	53
54	644.42	.80917	579.47	.76303	526.44	.72135	482.34	.68335	54
55	643.22	.80836	578.49	.76230	525.64	.72069	481.67	.68275	55
56	642.02	2.80755	577.53	2.76157	524.84	2.72003	481.00	2.68214	56
57	640.83	.80674	576.56	.76084	524.05	.71937	480.33	.68154	57
58	639.64	.80593	575.60	.76012	523.25	.71871	479.67	.68094	58
59	638.45	.80513	574.64	.75939	522.46	.71805	479.00	.68033	59
60	637.27	.80432	573.69	.75867	521.67	.71739	478.34	.67973	60

TABLE I.—RADII OF CURVES.

Deg.	Radius.	Log R	Deg.	Radius.	Log R	Deg.	Radius.	Log R	Deg.	Radius	Log R
12°	478.34	2.67973	14°	410.28	2.61307	16°	359.26	2.55541	21°	274.37	2.43833
2	477.02	.67853	2	409.31	.61205	5	357.42	.55317	10	272.23	.43494
4	475.71	.67734	4	408.34	.61102	10	355.59	.55094	20	270.13	.43157
6	474.40	.67614	6	407.38	.61000	15	353.77	.54872	30	268.06	.42823
8	473.10	.67495	8	406.42	.60898	20	351.98	.54652	40	266.02	.42492
10	471.81	2.67376	10	405.47	2.60796	25	350.21	.54432	50	264.02	.42163
12	470.53	.67258	12	404.53	.60694	30	348.45	2.54214	22°	262.04	2.41837
14	469.25	.67140	14	403.58	.60593	35	346.71	.53997	10	260.10	.41513
16	467.98	.67022	16	402.65	.60492	40	344.99	.53780	20	258.18	.41192
18	466.72	.66905	18	401.71	.60391	45	343.29	.53565	30	256.29	.40873
20	465.46	2.66788	20	400.78	2.60291	50	341.60	.53351	40	254.43	.40557
22	464.21	.66671	22	399.86	.60190	55	339.93	.53138	50	252.60	.40243
24	462.97	.66555	24	398.94	.60090	17°	338.27	2.52927	23°	250.79	2.39931
26	461.73	.66439	26	398.02	.59990	5	336.64	.52716	10	249.01	.39622
28	460.50	.66323	28	397.11	.59891	10	335.01	.52506	20	247.26	.39315
30	459.28	2.66207	30	396.20	2.59791	15	333.41	.52297	30	245.53	.39010
32	458.06	.66092	32	395.30	.59692	20	331.82	.52090	40	243.82	.38707
34	456.85	.65977	34	394.40	.59593	25	330.24	.51883	50	242.14	.38407
36	455.65	.65863	36	393.50	.59494	30	328.68	2.51677	24°	240.49	2.38109
38	454.45	.65748	38	392.61	.59396	35	327.13	.51472	10	238.85	.37813
40	453.26	2.65634	40	391.72	2.59298	40	325.60	.51269	20	237.24	.37519
42	452.07	.65521	42	390.84	.59199	45	324.09	.51066	30	235.65	.37227
44	450.89	.65407	44	389.96	.59102	50	322.59	.50864	40	234.08	.36937
46	449.72	.65294	46	389.08	.59004	55	321.10	.50663	50	232.54	.36649
48	448.56	.65181	48	388.21	.58907	18°	319.62	2.50464	25°	231.01	2.36363
50	447.40	2.65069	50	387.34	2.58809	5	318.16	.50265	30	229.55	.35517
52	446.24	.64957	52	386.48	.58713	10	316.71	.50067	26°	227.27	.34688
54	445.09	.64845	54	385.62	.58616	15	315.28	.49869	30	218.15	.33875
56	443.95	.64733	56	384.77	.58519	20	313.86	.49673	27°	214.18	2.33078
58	442.81	.64622	58	383.91	.58423	25	312.45	.49478	30	210.36	.32296
13°	441.68	2.64511	15°	383.06	2.58327	30	311.06	2.49284	28°	206.68	.31529
2	440.56	.64400	2	382.22	.58231	35	309.67	.49090	30	203.13	.30776
4	439.44	.64290	4	381.38	.58135	40	308.30	.48898	29°	199.70	2.30037
6	438.33	.64180	6	380.54	.58040	45	306.95	.48706	30	196.38	.29310
8	437.22	.64070	8	379.71	.57945	50	305.60	.48515	30°	193.19	.28597
10	436.12	2.63960	10	378.88	2.57850	55	304.27	.48325	30	190.09	.27896
12	435.02	.63851	12	378.05	.57755	19°	302.94	2.48136	31°	187.10	2.27207
14	433.93	.63742	14	377.23	.57661	5	301.63	.47948	32	181.40	.25863
16	432.84	.63633	16	376.41	.57566	10	300.33	.47760	33	176.05	.24563
18	431.76	.63524	18	375.60	.57472	15	299.04	.47573	34	171.02	.23303
20	430.69	2.63416	20	374.79	2.57378	20	297.77	.47388	35	166.28	.22083
22	429.62	.63308	22	373.98	.57284	25	296.50	.47203	36	161.80	2.20899
24	428.56	.63201	24	373.17	.57191	30	295.25	2.47018	37	157.58	.19749
26	427.50	.63093	26	372.37	.57097	35	294.00	.46835	38	153.58	.18633
28	426.44	.62986	28	371.57	.57004	40	292.77	.46652	39	149.79	.17547
30	425.40	2.62879	30	370.78	2.56911	45	291.55	.46471	40	146.19	.16492
32	424.35	.62773	32	369.99	.56819	50	290.33	.46289	41	142.77	2.15464
34	423.32	.62666	34	369.20	.56726	55	289.13	.46109	42	139.52	.14464
36	422.28	.62560	36	368.42	.56634	20°	287.94	2.45930	43	136.43	.13489
38	421.26	.62454	38	367.64	.56542	5	286.76	.45751	44	133.47	.12539
40	420.23	2.62349	40	366.86	2.56450	10	285.58	.45573	45	130.66	.11613
42	419.22	.62243	42	366.09	.56358	15	284.42	.45396	46	127.97	2.10709
44	418.20	.62138	44	365.31	.56266	20	283.27	.45219	47	125.39	.09827
46	417.19	.62034	46	364.55	.56175	25	282.12	.45044	48	122.93	.08965
48	416.19	.61929	48	363.78	.56084	30	280.99	2.44869	49	120.57	.08124
50	415.19	2.61825	50	363.02	2.55993	35	279.86	.44694	50	118.31	.07302
52	414.20	.61721	52	362.26	.55902	40	278.75	.44521	52	114.06	2.05713
54	413.21	.61617	54	361.51	.55812	45	277.64	.44348	54	110.13	.04192
56	412.23	.61514	56	360.76	.55721	50	276.54	.44176	56	106.50	.02736
58	411.25	.61410	58	360.01	.55631	55	275.45	.44004	58	103.13	.01340
14°	410.28	2.61307	16°	359.26	2.55541	21°	274.37	2.43833	60	100.00	2.00000

TABLE II.—TANGENTS, EXTERNAL DISTANCES, AND LONG CHORDS
FOR A 1° CURVE.

Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>	Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>	Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>
1°	50.00	0.218	100.00	11°	551.70	26.500	1098.3	21°	1061.9	97.58	2088.3
10'	58.34	0.297	116.67	10	560.11	27.313	1114.9	10	1070.6	99.15	2104.7
20	66.67	0.388	133.33	20	568.53	28.137	1131.5	20	1079.2	100.75	2121.1
30	75.01	0.491	150.00	30	576.95	28.974	1148.1	30	1087.8	102.35	2137.4
40	83.34	0.606	166.66	40	585.36	29.824	1164.7	40	1096.4	103.97	2153.8
50	91.68	0.733	183.33	50	593.79	30.686	1181.2	50	1105.1	105.60	2170.2
2°	100.01	0.873	199.99	12°	602.21	31.561	1197.8	22°	1113.7	107.24	2186.5
10	108.35	1.024	216.66	10	610.64	32.447	1214.4	10	1122.4	108.90	2202.9
20	116.68	1.188	233.32	20	619.07	33.347	1231.0	20	1131.0	110.57	2219.2
30	125.02	1.364	249.98	30	627.50	34.259	1247.5	30	1139.7	112.25	2235.6
40	133.36	1.552	266.65	40	635.93	35.183	1264.1	40	1148.4	113.95	2251.9
50	141.70	1.752	283.31	50	644.37	36.120	1280.7	50	1157.0	115.66	2268.3
3°	150.04	1.964	299.97	13°	652.81	37.069	1297.2	23°	1165.7	117.38	2284.6
10	158.38	2.188	316.63	10	661.25	38.031	1313.8	10	1174.4	119.12	2301.0
20	166.72	2.425	333.29	20	669.70	39.006	1330.3	20	1183.1	120.87	2317.3
30	175.06	2.674	349.95	30	678.15	39.993	1346.9	30	1191.8	122.63	2333.6
40	183.40	2.934	366.61	40	686.60	40.992	1363.4	40	1200.5	124.41	2349.9
50	191.74	3.207	383.27	50	695.06	42.004	1380.0	50	1209.2	126.20	2366.2
4°	200.08	3.492	399.92	14°	703.51	43.029	1396.5	24°	1217.9	128.00	2382.5
10	208.43	3.790	416.58	10	711.97	44.066	1413.1	10	1226.6	129.82	2398.8
20	216.77	4.099	433.24	20	720.44	45.116	1429.6	20	1235.3	131.65	2415.1
30	225.12	4.421	449.89	30	728.90	46.178	1446.2	30	1244.0	133.50	2431.4
40	233.47	4.755	466.54	40	737.37	47.253	1462.7	40	1252.8	135.36	2447.7
50	241.81	5.100	483.20	50	745.85	48.341	1479.2	50	1261.5	137.23	2464.0
5°	250.16	5.459	499.85	15°	754.32	49.441	1495.7	25°	1270.2	139.11	2480.2
10	258.51	5.829	516.50	10	762.80	50.554	1512.3	10	1279.0	141.01	2496.5
20	266.86	6.211	533.15	20	771.29	51.679	1528.8	20	1287.7	142.93	2512.8
30	275.21	6.606	549.80	30	779.77	52.818	1545.3	30	1296.5	144.85	2529.0
40	283.57	7.013	566.44	40	788.26	53.969	1561.8	40	1305.3	146.79	2545.3
50	291.92	7.432	583.09	50	796.75	55.132	1578.3	50	1314.0	148.75	2561.5
6°	300.28	7.863	599.73	16°	805.25	56.309	1594.8	26°	1322.8	150.71	2577.8
10	308.64	8.307	616.38	10	813.75	57.498	1611.3	10	1331.6	152.69	2594.0
20	316.99	8.762	633.02	20	822.25	58.699	1627.8	20	1340.4	154.69	2610.3
30	325.35	9.230	649.66	30	830.76	59.914	1644.3	30	1349.2	156.70	2626.5
40	333.71	9.710	666.30	40	839.27	61.141	1660.8	40	1358.0	158.72	2642.7
50	342.08	10.202	682.94	50	847.78	62.381	1677.3	50	1366.8	160.76	2658.9
7°	350.44	10.707	699.57	17°	856.30	63.634	1693.8	27°	1375.6	162.81	2675.1
10	358.81	11.224	716.21	10	864.82	64.900	1710.3	10	1384.4	164.87	2691.3
20	367.17	11.753	732.84	20	873.35	66.178	1726.8	20	1393.2	166.95	2707.5
30	375.54	12.294	749.47	30	881.88	67.470	1743.2	30	1402.0	169.04	2723.7
40	383.91	12.847	766.10	40	890.41	68.774	1759.7	40	1410.9	171.15	2739.9
50	392.28	13.413	782.73	50	898.95	70.091	1776.2	50	1419.7	173.27	2756.1
8°	400.66	13.991	799.36	18°	907.49	71.421	1792.6	28°	1428.6	175.41	2772.3
10	409.03	14.582	815.99	10	916.03	72.764	1809.1	10	1437.4	177.55	2788.4
20	417.41	15.184	832.61	20	924.58	74.119	1825.5	20	1446.3	179.72	2804.6
30	425.79	15.799	849.23	30	933.13	75.488	1842.0	30	1455.1	181.89	2820.7
40	434.17	16.426	865.85	40	941.69	76.869	1858.4	40	1464.0	184.08	2836.9
50	442.55	17.066	882.47	50	950.25	78.264	1874.9	50	1472.9	186.29	2853.0
9°	450.93	17.717	899.09	19°	958.81	79.671	1891.3	29°	1481.8	188.51	2869.2
10	459.32	18.381	915.70	10	967.38	81.092	1907.8	10	1490.7	190.74	2885.3
20	467.71	19.058	932.31	20	975.96	82.525	1924.2	20	1499.6	192.99	2901.4
30	476.10	19.746	948.92	30	984.53	83.972	1940.6	30	1508.5	195.25	2917.6
40	484.49	20.447	965.53	40	993.12	85.431	1957.1	40	1517.4	197.53	2933.7
50	492.88	21.161	982.14	50	1001.70	86.904	1973.5	50	1526.3	199.82	2949.8
10°	501.28	21.886	998.74	20°	1010.29	88.389	1989.9	30°	1535.3	202.12	2965.9
10	509.68	22.624	1015.35	10	1018.89	89.888	2006.3	10	1544.2	204.44	2982.0
20	518.08	23.375	1031.95	20	1027.49	91.399	2022.7	20	1553.1	206.77	2998.1
30	526.48	24.138	1048.54	30	1036.09	92.924	2039.1	30	1562.1	209.12	3014.2
40	534.89	24.913	1065.14	40	1044.70	94.462	2055.5	40	1571.0	211.48	3030.2
50	543.29	25.700	1081.73	50	1053.31	96.013	2071.9	50	1580.0	213.86	3046.3
11°	551.70	26.500	1098.33	21°	1061.93	97.577	2088.3	31°	1589.0	216.25	3062.4

**TABLE II.—TANGENTS, EXTERNAL DISTANCES, AND LONG CHORDS
FOR A 1° CURVE.**

Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>	Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>	Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>
31°	1589.0	216.25	3062.4	41°	2142.2	387.38	4013.1	51°	2732.9	618.39	4933.4
10	1598.0	218.66	3078.4	10	2151.7	390.71	4028.7	10	2743.1	622.81	4948.4
20	1606.9	221.08	3094.5	20	2161.2	394.06	4044.3	20	2753.4	627.24	4963.4
30	1615.9	223.51	3110.5	30	2170.8	397.43	4059.9	30	2763.7	631.69	4978.4
40	1624.9	225.96	3126.6	40	2180.3	400.82	4075.5	40	2773.9	636.16	4993.4
50	1633.9	228.42	3142.6	50	2189.9	404.22	4091.1	50	2784.2	640.66	5008.4
32°	1643.0	230.90	3158.6	42°	2199.4	407.64	4106.6	52°	2794.5	645.17	5023.4
10	1652.0	233.39	3174.6	10	2209.0	411.07	4122.2	10	2804.9	649.70	5038.4
20	1661.0	235.90	3190.6	20	2218.6	414.52	4137.7	20	2815.2	654.25	5053.4
30	1670.0	238.43	3206.6	30	2228.1	417.99	4153.3	30	2825.6	658.83	5068.3
40	1679.1	240.96	3222.6	40	2237.7	421.48	4168.8	40	2835.9	663.42	5083.3
50	1688.1	243.52	3238.6	50	2247.3	424.98	4184.3	50	2846.3	668.03	5098.2
33°	1697.2	246.08	3254.6	43°	2257.0	428.50	4199.8	53°	2856.7	672.66	5113.1
10	1706.3	248.66	3270.6	10	2266.6	432.04	4215.3	10	2867.1	677.32	5128.0
20	1715.3	251.26	3286.6	20	2276.2	435.59	4230.8	20	2877.5	681.99	5142.9
30	1724.4	253.87	3302.5	30	2285.9	439.16	4246.3	30	2888.0	686.68	5157.8
40	1733.5	256.50	3318.5	40	2295.6	442.75	4261.8	40	2898.4	691.40	5172.7
50	1742.6	259.14	3334.4	50	2305.2	446.35	4277.3	50	2908.9	696.13	5187.6
34°	1751.7	261.80	3350.4	44°	2314.9	449.98	4292.7	54°	2919.4	700.89	5202.4
10	1760.8	264.47	3366.3	10	2324.6	453.62	4308.2	10	2929.9	705.66	5217.3
20	1770.0	267.16	3382.2	20	2334.3	457.27	4323.6	20	2940.4	710.46	5232.1
30	1779.1	269.86	3398.2	30	2344.1	460.95	4339.0	30	2951.0	715.28	5246.9
40	1788.2	272.58	3414.1	40	2353.8	464.64	4354.5	40	2961.5	720.11	5261.7
50	1797.4	275.31	3430.0	50	2363.5	468.35	4369.9	50	2972.1	724.97	5276.5
35°	1806.6	278.05	3445.9	45°	2373.3	472.08	4385.3	55°	2982.7	729.85	5291.3
10	1815.7	280.82	3461.8	10	2383.1	475.82	4400.7	10	2993.3	734.76	5306.1
20	1824.9	283.60	3477.7	20	2392.8	479.59	4416.1	20	3003.9	739.68	5320.9
30	1834.1	286.39	3493.5	30	2402.6	483.37	4431.4	30	3014.5	744.62	5335.6
40	1843.3	289.20	3509.4	40	2412.4	487.16	4446.8	40	3025.2	749.59	5350.4
50	1852.5	292.02	3525.3	50	2422.3	490.98	4462.2	50	3035.8	754.57	5365.1
36°	1861.7	294.86	3541.1	46°	2432.1	494.82	4477.5	56°	3046.5	759.58	5379.8
10	1870.9	297.72	3557.0	10	2441.9	498.67	4492.8	10	3057.2	764.61	5394.5
20	1880.1	300.59	3572.8	20	2451.8	502.54	4508.2	20	3067.9	769.66	5409.2
30	1889.4	303.47	3588.6	30	2461.7	506.42	4523.5	30	3078.7	774.73	5423.9
40	1898.6	306.37	3604.5	40	2471.5	510.33	4538.8	40	3089.4	779.83	5438.6
50	1907.9	309.29	3620.3	50	2481.4	514.25	4554.1	50	3100.2	784.94	5453.3
37°	1917.1	312.22	3636.1	47°	2491.3	518.20	4569.4	57°	3110.9	790.08	5467.9
10	1926.4	315.17	3651.9	10	2501.2	522.16	4584.7	10	3121.7	795.24	5482.5
20	1935.7	318.13	3667.7	20	2511.2	526.13	4599.9	20	3132.6	800.42	5497.2
30	1945.0	321.11	3683.5	30	2521.1	530.13	4615.2	30	3143.4	805.62	5511.8
40	1954.3	324.11	3699.3	40	2531.1	534.15	4630.4	40	3154.2	810.85	5526.4
50	1963.6	327.12	3715.0	50	2541.0	538.18	4645.7	50	3165.1	816.10	5541.0
38°	1972.9	330.15	3730.8	48°	2551.0	542.23	4660.9	58°	3176.0	821.37	5555.6
10	1982.2	333.19	3746.5	10	2561.0	546.30	4676.1	10	3186.9	826.66	5570.2
20	1991.5	336.25	3762.3	20	2571.0	550.39	4691.3	20	3197.8	831.98	5584.7
30	2000.8	339.32	3778.0	30	2581.0	554.50	4706.5	30	3208.8	837.31	5599.3
40	2010.2	342.41	3793.8	40	2591.1	558.63	4721.7	40	3219.7	842.67	5613.8
50	2019.6	345.52	3809.5	50	2601.1	562.77	4736.9	50	3230.7	848.06	5628.3
39°	2029.0	348.64	3825.2	49°	2611.2	566.94	4752.1	59°	3241.7	853.46	5642.8
10	2038.4	351.78	3840.9	10	2621.2	571.12	4767.3	10	3252.7	858.89	5657.3
20	2047.8	354.94	3856.6	20	2631.3	575.32	4782.4	20	3263.7	864.34	5671.8
30	2057.2	358.11	3872.3	30	2641.4	579.54	4797.5	30	3274.8	869.82	5686.3
40	2066.6	361.29	3888.0	40	2651.5	583.78	4812.7	40	3285.8	875.32	5700.8
50	2076.0	364.50	3903.6	50	2661.6	588.04	4827.8	50	3296.9	880.84	5715.2
40°	2085.4	367.72	3919.3	50°	2671.8	592.32	4842.9	60°	3308.0	886.38	5729.7
10	2094.9	370.95	3935.0	10	2681.9	596.62	4858.0	10	3319.1	891.95	5744.1
20	2104.3	374.20	3950.6	20	2692.1	600.93	4873.1	20	3330.3	897.54	5758.5
30	2113.8	377.47	3966.3	30	2702.3	605.27	4888.2	30	3341.4	903.15	5772.9
40	2123.3	380.76	3981.9	40	2712.5	609.62	4903.2	40	3352.6	908.79	5787.3
50	2132.7	384.06	3997.5	50	2722.7	614.00	4918.3	50	3363.8	914.45	5801.7
41°	2142.2	387.38	4013.1	51°	2732.9	618.39	4933.4	61°	3375.0	920.14	5816.0

TABLE II.—TANGENTS, EXTERNAL DISTANCES, AND LONG CHORDS
FOR A 1° CURVE.

Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>	Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>	Δ	Tang. <i>T.</i>	Ext. Dist. <i>E.</i>	Long Chord <i>LC.</i>
61°	3375.0	920.14	5816.0	71°	4086.9	1308.2	6654.4	81°	4893.6	1805.3	7442.2
10'	3386.3	925.85	5830.4	10	4099.5	1315.5	6668.0	10	4908.0	1814.7	7454.9
20	3397.5	931.58	5844.7	20	4112.1	1322.9	6681.6	20	4922.5	1824.1	7467.5
30	3408.8	937.34	5859.1	30	4124.8	1330.3	6695.1	30	4937.0	1833.6	7480.2
40	3420.1	943.12	5873.4	40	4137.4	1337.7	6708.6	40	4951.5	1843.1	7492.8
50	3431.4	948.92	5887.7	50	4150.1	1345.1	6722.1	50	4966.1	1852.6	7505.4
62°	3442.7	954.75	5902.0	72°	4162.8	1352.6	6735.6	82°	4980.7	1862.2	7518.0
10	3454.1	960.60	5916.3	10	4175.6	1360.1	6749.1	10	4995.4	1871.8	7530.5
20	3465.4	966.48	5930.5	20	4188.4	1367.6	6762.5	20	5010.0	1881.5	7543.1
30	3476.8	972.39	5944.8	30	4201.2	1375.2	6776.0	30	5024.8	1891.2	7555.6
40	3488.2	978.31	5959.0	40	4214.0	1382.8	6789.4	40	5039.5	1900.9	7568.2
50	3499.7	984.27	5973.3	50	4226.8	1390.4	6802.8	50	5054.3	1910.7	7580.7
63°	3511.1	990.24	5987.5	73°	4239.7	1398.0	6816.3	83°	5069.2	1920.5	7593.2
10	3522.6	996.24	6001.7	10	4252.6	1405.7	6829.6	10	5084.0	1930.4	7605.6
20	3534.1	1002.3	6015.9	20	4265.6	1413.5	6843.0	20	5099.0	1940.3	7618.1
30	3545.6	1008.3	6030.0	30	4278.5	1421.2	6856.4	30	5113.9	1950.3	7630.5
40	3557.2	1014.4	6044.2	40	4291.5	1429.0	6869.7	40	5128.9	1960.2	7643.0
50	3568.7	1020.5	6058.4	50	4304.6	1436.8	6883.1	50	5143.9	1970.3	7655.4
64°	3580.3	1026.6	6072.5	74°	4317.6	1444.6	6896.4	84°	5159.0	1980.4	7667.8
10	3591.9	1032.8	6086.6	10	4330.7	1452.5	6909.7	10	5174.1	1990.5	7680.1
20	3603.5	1039.0	6100.7	20	4343.8	1460.4	6923.0	20	5189.3	2000.6	7692.5
30	3615.1	1045.2	6114.8	30	4356.9	1468.4	6936.2	30	5204.4	2010.8	7704.9
40	3626.8	1051.4	6128.9	40	4370.1	1476.4	6949.5	40	5219.7	2021.1	7717.2
50	3638.5	1057.7	6143.0	50	4383.3	1484.4	6962.8	50	5234.9	2031.4	7729.5
65°	3650.2	1063.9	6157.1	75°	4396.5	1492.4	6976.0	85°	5250.3	2041.7	7741.8
10	3661.9	1070.2	6171.1	10	4409.8	1500.5	6989.2	10	5265.6	2052.1	7754.1
20	3673.7	1076.6	6185.2	20	4423.1	1508.6	7002.4	20	5281.0	2062.5	7766.3
30	3685.4	1082.9	6199.2	30	4436.4	1516.7	7015.6	30	5296.4	2073.0	7778.6
40	3697.2	1089.3	6213.2	40	4449.7	1524.9	7028.8	40	5311.9	2083.5	7790.8
50	3709.0	1095.7	6227.2	50	4463.1	1533.1	7041.9	50	5327.4	2094.1	7803.0
66°	3720.9	1102.2	6241.2	76°	4476.5	1541.4	7055.0	86°	5343.0	2104.7	7815.2
10	3732.7	1108.6	6255.2	10	4489.9	1549.7	7068.2	10	5358.6	2115.3	7827.4
20	3744.6	1115.1	6269.1	20	4503.4	1558.0	7081.3	20	5374.2	2126.0	7839.6
30	3756.5	1121.7	6283.1	30	4516.9	1566.3	7094.4	30	5389.9	2136.7	7851.7
40	3768.5	1128.2	6297.0	40	4530.4	1574.7	7107.5	40	5405.6	2147.5	7863.8
50	3780.4	1134.8	6310.9	50	4544.0	1583.1	7120.5	50	5421.4	2158.4	7876.0
67°	3792.4	1141.4	6324.8	77°	4557.6	1591.6	7133.6	87°	5437.2	2169.2	7888.1
10	3804.4	1148.0	6338.7	10	4571.2	1600.1	7146.6	10	5453.1	2180.2	7900.1
20	3816.4	1154.7	6352.6	20	4584.8	1608.6	7159.6	20	5469.0	2191.1	7912.2
30	3828.4	1161.3	6366.4	30	4598.5	1617.1	7172.6	30	5484.9	2202.2	7924.3
40	3840.5	1168.1	6380.3	40	4612.2	1625.7	7185.6	40	5500.9	2213.2	7936.3
50	3852.6	1174.8	6394.1	50	4626.0	1634.4	7198.6	50	5517.0	2224.3	7948.3
68°	3864.7	1181.6	6408.0	78°	4639.8	1643.0	7211.6	88°	5533.1	2235.5	7960.3
10	3876.8	1188.4	6421.8	10	4653.6	1651.7	7224.5	10	5549.2	2246.7	7972.3
20	3889.0	1195.2	6435.6	20	4667.4	1660.5	7237.4	20	5565.4	2258.0	7984.2
30	3901.2	1202.0	6449.4	30	4681.3	1669.2	7250.4	30	5581.6	2269.3	7996.2
40	3913.4	1208.9	6463.1	40	4695.2	1678.1	7263.3	40	5597.8	2280.6	8008.1
50	3925.6	1215.8	6476.9	50	4709.2	1686.9	7276.1	50	5614.2	2292.0	8020.0
69°	3937.9	1222.7	6490.6	79°	4723.2	1695.8	7289.0	89°	5630.5	2303.5	8031.9
10	3950.2	1229.7	6504.4	10	4737.2	1704.7	7301.9	10	5646.9	2315.0	8043.8
20	3962.5	1236.7	6518.1	20	4751.2	1713.7	7314.7	20	5663.4	2326.6	8055.7
30	3974.8	1243.7	6531.8	30	4765.3	1722.7	7327.5	30	5679.9	2338.2	8067.5
40	3987.2	1250.8	6545.5	40	4779.4	1731.7	7340.3	40	5696.4	2349.8	8079.3
50	3999.5	1257.9	6559.1	50	4793.6	1740.8	7353.1	50	5713.0	2361.5	8091.2
70°	4011.9	1265.0	6572.8	80°	4808.7	1749.9	7365.9	90°	5729.7	2373.3	8103.0
10	4024.4	1272.1	6586.4	10	4822.0	1759.0	7378.7	10	5746.3	2385.1	8114.7
20	4036.8	1279.3	6600.1	20	4836.2	1768.2	7391.4	20	5763.1	2397.0	8126.5
30	4049.3	1286.5	6613.7	30	4850.5	1777.4	7404.1	30	5779.9	2408.9	8138.2
40	4061.8	1293.7	6627.3	40	4864.8	1786.7	7416.8	40	5796.7	2420.9	8150.0
50	4074.4	1300.9	6640.9	50	4879.2	1796.0	7429.5	50	5813.6	2432.9	8161.7
71°	4086.9	1308.2	6654.4	81°	4893.6	1805.3	7442.2	91°	5830.5	2444.9	8173.4

TABLE III.—SWITCH-LEADS AND DISTANCES.

LEAD-RAILS CIRCULAR THROUGHOUT; GAUGE 4' 8½". See § 262.

Frog No. (n)	Frog Angle (F).	Lead (L) (Eq. 79)	Chord (QT) (Eq. 77)	Radius of Lead-rails (r, Eq. 78)	Log r.	Length of Switch-rails (QK, Eq. 81).	Frog No. (n)
4	14° 15' 00"	37.67	37.38	150.67	2.17801	11.73	4
5	11 25 16	47.08	46.85	235.42	.37183	14.65	5
6	9 31 38	56.50	56.30	339.00	.53020	17.62	6
7	8 10 16	65.92	65.75	461.42	.66409	20.53	7
8	7° 09' 10	75.33	75.19	602.67	.78007	23.48	8
9	6 21 35	84.75	84.62	762.75	.88238	26.43	9
9½	6 01 32	89.46	89.33	849.85	.92934	27.97	9½
10	5 43 29	94.17	94.05	941.67	2.97389	29.37	10
11	5° 12' 18"	103.58	103.47	1139.42	3.05662	32.31	11
12	4 46 19	113.00	112.90	1356.00	.13226	35.25	12
15	3 49 06	141.25	141.17	2118.75	.32608	44.06	15
16	3 34 47	150.67	150.59	2410.67	.38213	46.98	16
18	3° 10' 56"	169.50	169.43	3051.00	.48444	52.87	18
20	2 51 51	188.33	188.27	3766.67	.57595	58.76	20
24	2° 23' 13"	226.00	225.95	5424.00	3.73432	70.52	24

TURNOUTS WITH STRAIGHT POINT-RAILS AND STRAIGHT FROG RAILS; GAUGE 4' 8½". See §§ 265 and 276.

Frog No. (n)	Frog Bluntness.	Frog.		Switch.		Switch Dimensions.				
		Wing Rail. (f)	Total Length. (HK)	Length. (DM)	Angle. (α)	Radius. (r)	Degree of Lead Curve. (D)	L'gth. (L)	Closure.	
									Str'ght Rail.	Curv'd Rail.
	ft.	ft. in.	ft. in.	ft. in.	° ' "	ft.	° ' "	ft.	ft.	ft.
4	0.17	3 2	8 6	11 0	2 36 19	112.26	52 53 56	37.22	22.88	23.29
5	0.21	3 7	10 0	11 0	2 36 19	183.22	31 40 24	42.98	28.19	28.55
6	0.25	4 0	11 0	11 0	2 36 19	273.95	21 01 58	48.36	33.11	33.38
7	0.29	4 5	12 6	16 6	1 44 11	364.88	15 47 19	62.23	41.02	41.24
8	0.33	4 9	13 6	16 6	1 44 11	488.71	11 44 40	67.80	46.22	46.42
9	0.37	6 0	16 0	16 6	1 44 11	616.27	9 18 27	72.61	49.74	49.92
9½	0.40	6 0	16 0	16 6	1 44 11	699.97	8 11 33	75.30	52.40	52.58
10	0.42	6 0	16 6	16 6	1 44 11	790.25	7 15 18	77.93	55.01	55.17
11	0.46	6 0	17 6	22 0	1 18 08	940.21	6 05 48	92.52	64.06	64.20
12	0.50	6 5	18 6	22 0	1 18 08	1136.34	5 02 38	97.75	68.83	68.96
15	0.62	7 8	22 6	33 0	0 52 05	1744.38	3 17 01	133.64	92.36	92.46
16	0.67	8 0	24 0	33 0	0 52 05	2005.98	2 51 24	136.62	94.95	95.05
18	0.75	8 10	26 6	33 0	0 52 05	2587.66	2 12 52	147.13	104.54	104.61
20	0.83	9 8	29 0	33 0	0 52 05	3262.98	1 45 22	157.18	113.68	113.76
24	1.00	11 4	34 6	33 0	0 52 05	4932.77	1 09 42	176.09	130.66	130.77

TABLE III. SWITCH LEAD AND DISTANCES.—*Continued.*

TRIGONOMETRICAL FUNCTIONS OF THE FROG ANGLES.

Frog No. (n)	Frog Angle (F).	Nat. sin F.	Nat. cos F.	Log sin F.	Log cos F.	Log cot F.	Log vers F.	Frog No. (n)
4	14° 15' 00''	.24615	.96923	9.39120	9.98642	10.59522	8.48811	4
5	11 25 16	.19802	.98020	.29670	.99131	.69461	.29670	5
6	9 31 38	.16552	.98621	.21884	.99397	.77513	.13966	6
7	8 10 16	.14213	.98985	.15268	.99557	.84288	8.00655	7
8	7 09 10	.12452	.99222	.09522	.99660	.90138	7.89110	8
9	6 21 35	.11077	.99385	.04442	.99732	.95289	.78915	9
9½	6 01 32	.10497	.99448	9.02107	.99759	.97652	.74232	9½
10	5 43 29	.09975	.99501	8.99891	.99783	10.99892	.69787	10
11	5 12 18	.09072	.99588	.95770	.99820	11.04050	.61527	11
12	4 46 19	.08319	.99653	.92007	.99849	.07842	.53986	12
15	3 49 06	.06659	.99778	.82343	.99903	.17560	.34631	15
16	3 34 47	.06244	.99805	.79543	.99915	.20370	.29028	16
18	3 10 56	.05551	.99846	.74438	.99933	.25494	.18807	18
20	2 51 51	.04997	.99875	.69869	.99945	.30076	7.09663	20
24	2° 23' 13''	.04165	.99913	8.61959	9.99962	11.38003	6.93834	24

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Point.	Total Central Angle ϕ	Nat. sin ϕ	Nat. cos ϕ	Log sin ϕ	Log cos ϕ	Log vers ϕ	α	Log α	ψ	Point.
1	0° 07' 30"	.0022	one	7.33878	10.00000	4.37654	0.027	8.43568	25.000	1
2	0 22 30	.0065	one	7.81590	9.99999	5.33078	0.136	9.13465	50.000	2
3	0 45 00	.0131	.9999	8.11692	9.99996	5.93282	0.382	9.58181	74.999	3
4	1 15 00	.0218	.9997	8.33875	9.99985	6.37653	0.818	9.91280	99.995	4
5	1 52 30	.0327	.9994	8.51480	9.99976	6.72869	1.500	0.17602	124.985	5
6	2 37 30	.0458	.9980	8.66085	9.99954	7.02091	2.481	0.39467	149.966	6
7	3 30 00	.0610	.9981	8.78567	9.99919	7.27072	3.817	0.58171	174.930	7
8	4 30 00	.0784	.9969	8.89464	9.99866	7.48892	5.561	0.74514	199.870	8
9	5 37 30	.0980	.9952	8.99130	9.99790	7.68262	7.792	0.89164	224.772	9
10	6 52 30	.1197	.9928	9.07810	9.99686	7.85675	10.489	1.02061	249.623	10

0° 30'-per-25-foot spiral. PART A.

Deflections from the tangent at the point occupied when the instrument is at—

sighting at	Q	1	2	3	4	5	6	7	8	9	10
Q	0° 0' 0"	0° 3' 45"	0° 13' 07"	0° 27' 30"	0° 46' 52"	1° 11' 15"	1° 40' 37"	2° 15' 00"	2° 54' 22"	3° 38' 45"	4° 28' 10"
1	0 3 45	0 0 00	0 07 30	0 20 37	0 38 45	1 01 52	1 30 00	2 03 07	2 41 15	3 24 22	4 12 30
2	0 9 22	0 7 30	0 00 00	0 11 15	0 28 07	0 50 00	1 16 52	1 48 45	2 25 37	3 07 30	3 54 22
3	0 17 30	0 16 52	0 11 15	0 00 00	0 15 00	0 35 37	1 01 15	1 31 52	2 07 30	2 48 07	3 33 45
4	0 28 07	0 28 45	0 24 22	0 15 00	0 00 00	0 18 45	0 43 07	1 12 30	1 46 52	2 26 15	3 10 37
5	0 41 15	0 43 07	0 40 00	0 31 52	0 18 45	0 00 00	0 22 30	0 50 37	1 23 45	2 01 52	2 45 00
6	0 58 52	1 00 00	0 58 07	0 51 15	0 39 22	0 22 30	0 00 00	0 26 15	0 58 07	1 35 00	2 16 52
7	1 15 00	1 19 22	1 18 45	1 13 07	1 02 30	0 46 52	0 26 15	0 00 00	0 30 00	1 05 37	1 46 15
8	1 35 37	1 41 15	1 41 52	1 37 30	1 28 07	1 13 45	0 54 22	0 30 00	0 00 00	0 33 45	1 13 07
9	1 58 45	2 05 37	2 07 30	2 04 22	1 58 15	1 43 07	1 25 00	1 01 52	0 33 45	0 00 00	0 37 30
10	2 24 20	2 32 30	2 35 37	2 33 45	2 26 52	2 15 00	1 58 07	1 36 15	1 09 22	0 37 30	0 00 00

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Dimensions of various 0° 30'-per-25-feet spirals.—Part B.

Deg. of curve.	L'gth of spiral	ZK	QK	A'N	NQ	A' to Z.		Deg. of curve.
						Defl.	Dist.	
1° 00'	25	0.03	25.00	0.01	12.50	0° 04'	12.50	1° 00'
10	50	0.14	50.00	0.03	17.86	0 11	32.14	10
20	50	0.14	50.00	0.04	21.88	0 11	28.13	20
30	50	0.14	50.00	0.05	25.00	0 11	25.00	30
40	75	0.38	75.00	0.09	30.00	0 22½	45.00	40
50	75	0.38	75.00	0.11	43.09	0 22½	40.91	50
2° 00'	75	0.38	75.00	0.14	37.50	0° 22½	37.50	2° 00'
10	100	0.82	100.00	0.19	42.30	0 37½	57.69	10
20	100	0.82	100.00	0.23	46.43	0 37½	53.57	20
30	100	0.82	100.00	0.27	50.00	0 37½	50.00	30
40	125	1.50	124.99	0.35	54.68	0 56	70.31	40
50	125	1.50	124.99	0.42	58.81	0 56	66.18	50
3° 00'	125	1.50	124.99	0.48	62.49	0° 56'	62.50	3° 00'
10	150	2.48	149.97	0.58	67.09	1 19	82.59	10
20	150	2.48	149.97	0.68	71.24	1 19	78.75	20
30	150	2.48	149.97	0.76	74.98	1 19	75.00	30
40	175	3.82	174.93	0.90	79.52	1 45	95.45	40
50	175	3.82	174.93	1.03	83.67	1 45	91.30	50
4° 00'	175	3.82	174.93	1.15	87.47	1° 45'	87.50	4° 00'
10	200	5.56	199.87	1.32	91.96	2 15	108.00	10
20	200	5.56	199.87	1.48	96.11	2 15	103.85	20
30	200	5.56	199.87	1.63	99.95	2 15	100.00	30
40	225	7.79	224.77	1.88	104.40	2 49	120.54	40
50	225	7.79	224.77	2.08	108.55	2 49	116.38	50
5° 00'	225	7.79	224.77	2.27	112.42	2° 49'	112.50	5° 00'
10	250	10.49	249.62	2.51	116.83	3 26	133.06	10
20	250	10.49	249.62	2.76	120.98	3 26	128.91	20
30	250	10.49	249.62	3.00	124.88	3 26	125.00	30

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Dimensions of various 0° 30'-per-25-feet spirals.—Part C.

VALUES OF AN .

Δ	Degree of curve.					
	1°	2°	3°	4°	5°	5° 30'
2°	0.00	0.00	0.01	0.02	0.04	0.05
4	0.00	0.01	0.02	0.04	0.08	0.10
6	0.00	0.01	0.03	0.06	0.12	0.16
8	0.00	0.01	0.03	0.08	0.16	0.21
10	0.00	0.01	0.04	0.10	0.20	0.28
12°	0.00	0.01	0.05	0.12	0.24	0.32
14	0.00	0.02	0.06	0.14	0.28	0.37
16	0.00	0.02	0.07	0.16	0.32	0.42
18	0.00	0.02	0.08	0.18	0.36	0.47
20	0.00	0.02	0.08	0.20	0.40	0.53
22°	0.00	0.03	0.09	0.22	0.44	0.58
24	0.00	0.03	0.10	0.24	0.48	0.64
26	0.00	0.03	0.11	0.26	0.52	0.69
28	0.00	0.03	0.12	0.29	0.57	0.75
30	0.00	0.04	0.13	0.31	0.61	0.80
32°	0.00	0.04	0.14	0.33	0.65	0.86
34	0.00	0.04	0.15	0.35	0.69	0.92
36	0.00	0.04	0.16	0.37	0.74	0.97
38	0.00	0.05	0.16	0.39	0.78	1.03
40	0.00	0.05	0.17	0.42	0.83	1.09
42°	0.00	0.05	0.18	0.44	0.87	1.15
44	0.00	0.06	0.19	0.46	0.92	1.21
46	0.00	0.06	0.20	0.49	0.96	1.27
48	0.00	0.06	0.21	0.51	1.01	1.34
50	0.00	0.06	0.22	0.53	1.06	1.40
52°	0.00	0.07	0.23	0.56	1.11	1.46
54	0.01	0.07	0.24	0.58	1.16	1.53
56	0.01	0.07	0.25	0.61	1.21	1.59
58	0.01	0.08	0.26	0.64	1.26	1.66
60	0.01	0.08	0.28	0.66	1.31	1.73
62°	0.01	0.08	0.29	0.69	1.36	1.80
64	0.01	0.09	0.30	0.72	1.42	1.87
66	0.01	0.09	0.31	0.74	1.47	1.95
68	0.01	0.09	0.32	0.77	1.53	2.02
70	0.01	0.10	0.33	0.80	1.59	2.10
72°	0.01	0.10	0.35	0.83	1.65	2.18
74	0.01	0.10	0.36	0.83	1.71	2.26
76	0.01	0.11	0.37	0.90	1.77	2.34
78	0.01	0.11	0.39	0.93	1.84	2.43
80	0.01	0.11	0.40	0.96	1.91	2.51
82°	0.01	0.12	0.42	1.00	1.97	2.60
84	0.01	0.12	0.43	1.03	2.04	2.70
86	0.01	0.13	0.45	1.07	2.12	2.79
88	0.01	0.13	0.46	1.10	2.19	2.89
90	0.01	0.14	0.48	1.15	2.27	3.00
92°	0.01	0.14	0.49	1.19	2.35	3.10
94	0.01	0.15	0.51	1.23	2.44	3.21
96	0.01	0.15	0.53	1.27	2.52	3.33
98	0.01	0.16	0.55	1.32	2.61	3.45
100	0.01	0.16	0.57	1.37	2.71	3.57

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Point.	Total Central Angle ϕ	Nat. sin ϕ	Nat. cos ϕ	Log sin ϕ	Log cos ϕ	Log vers ϕ	x	Log x	y	Point.
1	0° 15'	.0043	one	7.63981	9.99999	4.97860	.055	8.73672	25.000	1
2	0 45	.0131	.9999	8.11692	9.99996	5.93284	.273	9.43616	49.999	2
3	1 30	.0262	.9996	8.41792	9.99985	6.53488	.763	9.88252	74.994	3
4	2 30	.0436	.9990	8.63968	9.99958	6.97853	1.636	0.21378	99.979	4
5	3 45	.0654	.9978	8.81560	9.99907	7.33063	2.999	0.47697	124.942	5
6	5 15	.0915	.9958	8.96143	9.99817	7.62274	4.960	0.69548	149.865	6
7	7 00	.1213	.9925	9.08589	9.99675	7.87238	7.628	0.88241	174.722	7
8	9 00	.1564	.9877	9.19433	9.99462	8.09031	11.107	1.04559	199.479	8
9	11 15	.1951	.9808	9.29023	9.99157	8.28363	15.502	1.19039	224.090	9
10	13 45	.2377	.9713	9.37600	9.98737	8.45724	20.913	1.32041	248.497	10

*V rrvd 'rvids 100f-52-rvd-01

Deflections from the tangent at the point occupied when the instrument is at—

—sighting at—	Q										
	Q	1	2	3	4	5	6	7	8	9	10
Q	0° 0' 0"	0° 07' 30"	0° 26' 15"	0° 55' 0"	1° 33' 45"	2° 22' 30"	3° 21' 15"	4° 30' 00"	5° 48' 47"	7° 17' 34"	8° 56' 22"
1	0 07 30	0 00 00	0 15 00	0 41 15	1 17 30	2 03 45	3 00 00	4 06 15	5 22 30	6 48 47	8 25 05
2	0 18 45	0 15 00	0 00 00	0 22 30	0 56 15	1 40 00	2 33 45	3 37 30	4 51 15	6 15 02	7 48 49
3	0 35 00	0 33 45	0 22 30	0 00 00	0 30 00	1 11 15	2 02 30	3 03 45	4 15 00	5 36 15	7 07 32
4	0 56 15	0 57 30	0 48 45	0 30 00	0 00 00	0 37 30	1 26 15	2 25 00	3 33 45	4 52 30	6 21 17
5	1 22 30	1 26 15	1 20 00	1 03 45	0 37 30	0 00 00	0 45 00	1 41 15	2 47 30	4 03 45	5 30 00
6	1 53 45	2 00 00	1 56 15	1 42 30	1 18 45	0 45 00	0 00 00	0 52 30	1 56 15	3 10 00	4 33 45
7	2 30 00	2 38 45	2 37 30	2 26 15	2 05 00	1 33 45	0 52 30	0 00 00	1 00 00	2 11 15	3 32 30
8	3 11 13	3 22 30	3 23 45	3 15 00	2 56 15	2 27 30	1 48 45	1 00 00	0 00 00	1 07 30	2 26 15
9	3 57 26	4 11 13	4 14 58	4 08 45	3 52 30	3 26 15	2 50 00	2 03 45	1 07 30	0 00 00	1 15 00
10	4 48 38	5 04 55	5 11 11	5 07 28	4 53 43	4 30 00	3 56 15	3 12 30	2 18 45	1 15 00	0 00 00

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Dimensions of various 1°-per-25-foot spirals.—Part B.

Deg. of curve.	L'gth of spiral	ZK	QK	A'N	NQ	A' to Z.		Deg. of curve.
						Defl.	Dist.	
2° 00'	25	0.06	25.00	0.03	12.50	0° 07 $\frac{1}{2}$ '	12.50	2° 00'
10	50	0.27	50.00	0.05	15.38	0 22 $\frac{1}{2}$ '	34.62	10
20	50	0.27	50.00	0.06	17.86	0 22 $\frac{1}{2}$ '	32.14	20
30	50	0.27	50.00	0.08	20.00	0 22 $\frac{1}{2}$ '	30.00	30
40	50	0.27	50.00	0.09	21.87	0 22 $\frac{1}{2}$ '	28.13	40
50	50	0.27	50.00	0.10	23.53	0 22 $\frac{1}{2}$ '	26.17	50
3° 00'	50	0.27	50.00	0.11	25.00	0° 22 $\frac{1}{2}$ '	25.00	3° 00'
10	75	0.76	74.99	0.14	27.63	0 45	47.37	10
20	75	0.76	74.99	0.17	29.99	0 45	45.00	20
30	75	0.76	74.99	0.20	32.13	0 45	42.86	30
40	75	0.76	74.99	0.23	34.08	0 45	40.19	40
50	75	0.76	74.99	0.25	35.86	0 45	39.13	50
4° 00'	75	0.76	74.99	0.27	37.49	0° 45'	37.50	4° 00'
10	100	1.64	99.98	0.33	39.98	1 15	60.00	10
20	100	1.64	99.98	0.38	42.29	1 15	57.69	20
30	100	1.64	99.98	0.42	44.43	1 15	55.56	30
40	100	1.64	99.98	0.47	46.41	1 15	53.57	40
50	100	1.64	99.98	0.51	48.26	1 15	51.72	50
5° 00'	100	1.64	99.98	0.55	49.98	1° 15'	50.00	5° 00'
10	125	3.00	124.94	0.62	52.39	1 52 $\frac{1}{2}$ '	72.58	10
20	125	3.00	124.94	0.70	54.65	1 52 $\frac{1}{2}$ '	70.31	20
30	125	3.00	124.94	0.77	56.78	1 52 $\frac{1}{2}$ '	68.18	30
40	125	3.00	124.94	0.83	58.79	1 52 $\frac{1}{2}$ '	66.18	40
50	125	3.00	124.94	0.90	60.67	1 52 $\frac{1}{2}$ '	64.29	50
6° 00'	125	3.00	124.94	0.95	62.46	1° 52 $\frac{1}{2}$ '	62.50	6° 00'
10	150	4.96	149.87	1.06	64.81	2 37 $\frac{1}{2}$ '	85.14	10
20	150	4.96	149.87	1.16	67.04	2 37 $\frac{1}{2}$ '	82.89	20
30	150	4.96	149.87	1.26	69.17	2 37 $\frac{1}{2}$ '	80.77	30
40	150	4.96	149.87	1.35	71.18	2 37 $\frac{1}{2}$ '	78.75	40
50	150	4.96	149.87	1.44	73.10	2 37 $\frac{1}{2}$ '	76.83	50
7° 00'	150	4.96	149.87	1.52	74.92	2° 37 $\frac{1}{2}$ '	75.00	7° 00'
10	175	7.63	174.72	1.67	77.23	3 30	97.67	10
20	175	7.63	174.72	1.80	79.44	3 30	95.45	20
30	175	7.63	174.72	1.93	81.55	3 30	93.33	30
40	175	7.63	174.72	2.05	83.58	3 30	91.30	40
50	175	7.63	174.72	2.17	85.51	3 30	89.36	50
8° 00'	175	7.63	174.72	2.29	87.37	3° 30'	87.50	8° 00'
10	200	11.11	199.48	2.46	89.64	4 30	110.20	10
20	200	11.11	199.48	2.64	91.83	4 30	108.00	20
30	200	11.11	199.48	2.80	93.94	4 30	105.88	30
40	200	11.11	199.48	2.96	95.96	4 30	103.85	40
50	200	11.11	199.48	3.10	97.91	4 30	101.89	50
9° 00'	200	11.11	199.48	3.26	99.79	4° 30'	100.00	9° 00'
10	225	15.50	224.09	3.48	102.03	5 37 $\frac{1}{2}$ '	122.73	10
20	225	15.50	224.09	3.69	104.20	5 37 $\frac{1}{2}$ '	120.54	20
30	225	15.50	224.09	3.90	106.29	5 37 $\frac{1}{2}$ '	118.42	30
40	225	15.50	224.09	4.10	108.32	5 37 $\frac{1}{2}$ '	116.38	40
50	225	15.50	224.09	4.29	110.28	5 37 $\frac{1}{2}$ '	114.41	50
10° 00'	225	15.50	224.09	4.48	112.17	5° 37 $\frac{1}{2}$ '	112.50	10° 00'
10	250	20.91	248.50	4.74	114.37	6 52 $\frac{1}{2}$ '	135.25	10
20	250	20.91	248.50	5.00	116.53	6 52 $\frac{1}{2}$ '	133.06	20
30	250	20.91	248.50	5.25	118.62	6 52 $\frac{1}{2}$ '	130.95	30
40	250	20.91	248.50	5.50	120.64	6 52 $\frac{1}{2}$ '	128.91	40
50	250	20.91	248.50	5.73	122.60	6 52 $\frac{1}{2}$ '	126.92	50
11° 00'	250	20.91	248.50	5.96	124.50	6° 52 $\frac{1}{2}$ '	125.00	11° 00'

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Dimensions of various 1°-per-25-feet spirals.—Part C.

VALUES OF AN .

Δ	Degree of curve.									
	2°	3°	4°	5°	6°	7°	8°	9°	10°	11°
2	0.00	0.00	0.00	0.01	0.02	0.03	0.04	0.06	0.08	0.11
4	0.00	0.00	0.01	0.02	0.03	0.05	0.08	0.11	0.16	0.21
6	0.00	0.01	0.01	0.03	0.05	0.08	0.12	0.17	0.23	0.31
8	0.00	0.01	0.02	0.04	0.07	0.11	0.16	0.23	0.31	0.42
10	0.00	0.01	0.02	0.05	0.08	0.13	0.20	0.29	0.39	0.52
12	0.00	0.01	0.03	0.06	0.10	0.16	0.24	0.34	0.47	0.63
14	0.00	0.01	0.03	0.07	0.12	0.19	0.28	0.40	0.55	0.73
16	0.00	0.02	0.04	0.08	0.13	0.21	0.32	0.46	0.63	0.84
18	0.00	0.02	0.04	0.09	0.15	0.24	0.36	0.52	0.71	0.94
20	0.00	0.02	0.05	0.10	0.17	0.27	0.40	0.57	0.79	1.05
22	0.01	0.02	0.05	0.11	0.19	0.30	0.44	0.63	0.87	1.16
24	0.01	0.02	0.06	0.12	0.20	0.32	0.49	0.69	0.95	1.27
26	0.01	0.03	0.06	0.13	0.22	0.35	0.53	0.75	1.03	1.38
28	0.01	0.03	0.07	0.14	0.24	0.38	0.57	0.81	1.11	1.49
30	0.01	0.03	0.07	0.15	0.26	0.41	0.61	0.87	1.20	1.60
32	0.01	0.03	0.08	0.16	0.27	0.44	0.65	0.93	1.28	1.71
34	0.01	0.03	0.08	0.17	0.29	0.47	0.70	1.00	1.37	1.82
36	0.01	0.04	0.09	0.18	0.31	0.49	0.74	1.06	1.45	1.94
38	0.01	0.04	0.09	0.19	0.33	0.52	0.79	1.12	1.54	2.05
40	0.01	0.04	0.10	0.20	0.35	0.55	0.83	1.19	1.63	2.17
42	0.01	0.04	0.10	0.21	0.37	0.58	0.88	1.25	1.72	2.28
44	0.01	0.04	0.11	0.22	0.38	0.62	0.92	1.32	1.81	2.41
46	0.01	0.05	0.12	0.23	0.40	0.65	0.97	1.38	1.90	2.53
48	0.01	0.05	0.12	0.24	0.42	0.68	1.02	1.45	1.99	2.65
50	0.01	0.05	0.13	0.25	0.44	0.71	1.07	1.52	2.09	2.78
52	0.01	0.05	0.13	0.27	0.46	0.74	1.11	1.59	2.18	2.91
54	0.01	0.06	0.14	0.28	0.49	0.78	1.16	1.66	2.28	3.04
56	0.01	0.06	0.14	0.29	0.51	0.81	1.21	1.74	2.38	3.17
58	0.02	0.06	0.15	0.30	0.53	0.85	1.27	1.81	2.48	3.31
60	0.02	0.06	0.16	0.31	0.55	0.88	1.32	1.88	2.58	3.44
62	0.02	0.07	0.16	0.33	0.57	0.92	1.37	1.96	2.69	3.58
64	0.02	0.07	0.17	0.34	0.60	0.95	1.43	2.04	2.80	3.73
66	0.02	0.07	0.18	0.35	0.62	0.99	1.48	2.12	2.91	3.87
68	0.02	0.07	0.18	0.37	0.64	1.03	1.54	2.20	3.02	4.02
70	0.02	0.08	0.19	0.38	0.67	1.07	1.60	2.28	3.13	4.18
72	0.02	0.08	0.20	0.40	0.69	1.11	1.66	2.37	3.25	4.33
74	0.02	0.08	0.20	0.41	0.72	1.15	1.72	2.46	3.38	4.49
76	0.02	0.09	0.21	0.43	0.74	1.19	1.79	2.55	3.50	4.66
78	0.02	0.09	0.22	0.44	0.77	1.23	1.85	2.64	3.63	4.83
80	0.02	0.09	0.23	0.46	0.80	1.28	1.92	2.74	3.76	5.00
82	0.02	0.09	0.24	0.47	0.83	1.32	1.99	2.83	3.89	5.18
84	0.03	0.10	0.24	0.49	0.86	1.37	2.06	2.94	4.03	5.37
86	0.03	0.10	0.25	0.51	0.89	1.42	2.13	3.04	4.18	5.56
88	0.03	0.11	0.26	0.53	0.92	1.47	2.21	3.15	4.33	5.76
90	0.03	0.11	0.27	0.55	0.95	1.52	2.29	3.26	4.48	5.96
92	0.03	0.11	0.28	0.56	0.99	1.58	2.37	3.38	4.64	6.17
94	0.03	0.12	0.29	0.58	1.02	1.63	2.45	3.50	4.80	6.39
96	0.03	0.12	0.30	0.61	1.06	1.69	2.54	3.62	4.97	6.62
98	0.03	0.13	0.31	0.63	1.10	1.75	2.63	3.75	5.15	6.86
100	0.03	0.13	0.32	0.65	1.13	1.82	2.72	3.89	5.34	7.11

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Point.	Total Central Angle ϕ	Nat. sin ϕ	Nat. cos ϕ	Log sin ϕ	Log cos ϕ	Log vers ϕ	x	Log x	y	Point.
1	0° 30'	.0087	.9999	7.94084	9.99998	5.53066	0.109	9.03774	25.000	1
2	1 30	.0262	.9996	8.41792	9.99985	6.53488	0.545	9.73670	49.996	2
3	3 00	.0523	.9986	8.71880	9.99940	7.13687	1.527	0.18388	74.977	3
4	5 00	.0871	.9962	8.94029	8.99834	7.58039	3.271	0.51468	99.916	4
5	7 30	.1305	.9914	9.11570	9.99627	7.93222	5.992	0.77762	124.767	5
6	10 30	.1822	.9832	9.26063	9.99266	8.22389	9.903	0.99579	149.459	6
7	14 00	.2419	.9703	9.38367	9.98390	8.47282	15.208	1.18207	173.890	7
8	18 00	.3090	.9510	9.48998	9.97320	8.68969	22.099	1.34437	197.922	8
9	22 30	.3827	.9239	9.58284	9.96561	8.88150	30.752	1.48787	221.376	9
10	27 30	.4617	.8870	9.66440	9.94793	9.05303	41.317	1.61613	244.034	10

2°-per-35-foot spiral. PART A.

—sight-
ing
at—

Deflections from the tangent at the point occupied when the instrument is at—

Q	1	2	3	4	5	6	7	8	9	10
0° 00' 00"	0° 15' 00"	0° 52' 30"	1° 50' 00"	3° 07' 30"	4° 45' 00"	6° 42' 30"	9° 00' 05"	11° 37' 45"	14° 35' 30"	17° 53' 25"
1 1	0 15 00	0 30 00	1 22 30	2 35 00	4 07 30	6 00 01	8 12 34	10 45 10	13 37 52	16 50 42
2 2	0 37 30	0 45 00	0 45 00	1 52 30	3 20 00	5 07 30	7 15 02	9 42 36	12 30 15	15 38 00
3 3	1 10 00	0 30 00	0 00 00	1 00 00	2 22 30	4 05 00	6 07 30	8 30 04	11 12 39	14 15 20
4 4	1 52 30	1 55 00	1 00 00	0 00 00	1 15 00	2 52 30	4 50 00	7 07 30	9 45 05	12 42 42
5 5	2 45 00	2 52 30	2 07 30	1 15 00	0 00 00	1 30 00	3 22 30	5 35 00	8 07 30	11 00 06
6 6	3 47 28	3 59 59	3 52 30	2 37 30	1 30 00	0 00 00	1 45 00	3 52 30	6 20 00	9 07 30
7 7	4 59 53	5 17 26	4 52 30	4 10 00	3 07 30	1 45 00	0 00 00	2 00 00	4 22 30	7 05 00
8 8	6 22 15	6 44 50	6 29 56	5 52 30	4 55 00	3 37 30	2 00 00	0 00 00	2 15 00	4 52 30
9 9	7 54 30	8 22 08	8 17 21	7 44 55	6 52 30	5 40 00	4 07 30	2 15 00	0 00 00	2 30 00
10 10	9 36 35	10 09 17	10 14 40	9 47 18	8 59 54	7 52 30	6 25 00	4 37 30	2 30 00	0 00 00

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Dimensions of various 2°-per-25-feet spirals.—Part B.

Deg. of curve.	L'gth of spiral	ZK	QK	A'N	NQ	A' to Z.		Deg. of curve.
						Defl.	Dist.	
4° 00'	25	0.11	25.00	0.05	12.50	0° 15'	12.50	4° 00'
20	50	0.55	50.00	0.09	15.38	0 45	34.62	20
40	50	0.55	50.00	0.12	17.85	0 45	32.14	40
5° 00'	50	0.55	50.00	0.15	19.99	0 45	30.00	5° 00'
20	50	0.55	50.00	0.18	21.86	0 45	28.13	20
40	50	0.55	50.00	0.20	23.52	0 45	26.47	40
6° 00'	50	0.55	50.00	0.22	24.99	0° 45'	25.00	6° 00'
20	75	1.52	74.98	0.29	27.61	1 30	47.37	20
40	75	1.52	74.98	0.35	29.97	1 30	45.00	40
7° 00'	75	1.52	74.98	0.40	32.11	1 30	42.86	7° 00'
20	75	1.52	74.98	0.46	34.06	1 30	40.91	20
40	75	1.52	74.98	0.50	35.84	1 30	39.13	40
8° 00'	75	1.52	74.98	0.54	37.46	1° 30'	37.50	8° 00'
20	100	3.27	99.92	0.65	39.94	2 30	60.00	20
40	100	3.27	99.92	0.75	42.24	2 30	57.69	40
9° 00'	100	3.27	99.92	0.85	44.37	2 30	55.56	9° 00'
20	100	3.27	99.92	0.93	46.35	2 30	53.57	20
40	100	3.27	99.92	1.01	48.20	2 30	51.72	40
10° 00'	100	3.27	99.92	1.09	49.92	2° 30'	50.00	10° 00'
20	125	5.99	124.77	1.24	52.30	3 45	72.58	20
40	125	5.99	124.77	1.39	54.55	3 45	70.31	40
11° 00'	125	5.99	124.77	1.53	56.68	3 45	68.18	11° 00'
20	125	5.99	124.77	1.66	58.67	3 45	66.18	20
40	125	5.99	124.77	1.78	60.55	3 45	64.29	40
12° 00'	125	5.99	124.77	1.90	62.33	3° 45'	62.50	12° 00'
20	150	9.90	149.46	2.11	64.68	5 15	85.14	20
40	150	9.90	149.46	2.31	66.86	5 15	82.89	40
13° 00'	150	9.90	149.46	2.51	68.97	5 15	80.77	13° 00'
20	150	9.90	149.46	2.69	70.97	5 15	78.75	20
40	150	9.90	149.46	2.87	72.88	5 15	76.83	40
14° 00'	150	9.90	149.46	3.03	74.69	5° 15'	75.00	14° 00'
20	175	15.21	173.89	3.30	76.93	7 00	97.67	20
40	175	15.21	173.89	3.57	79.12	7 00	95.45	40
15° 00'	175	15.21	173.89	3.83	81.22	7 00	93.33	15° 00'
20	175	15.21	173.89	4.08	83.22	7 00	91.30	20
40	175	15.21	173.89	4.31	85.14	7 00	89.36	40
16° 00'	175	15.21	173.89	4.54	86.98	7° 00'	87.50	16° 00'
20	200	22.10	197.92	4.87	89.15	9 00	110.20	20
40	200	22.10	197.92	5.21	91.31	9 00	108.00	40
17° 00'	200	22.10	197.92	5.54	93.39	9 00	105.88	17° 00'
20	200	22.10	197.92	5.86	95.38	9 00	103.85	20
40	200	22.10	197.92	6.16	97.30	9 00	101.89	40
18° 00'	200	22.10	197.92	6.45	99.15	9° 00'	100.00	18° 00'
20	225	30.75	221.38	6.86	101.27	11 15	122.73	20
40	225	30.75	221.38	7.28	103.39	11 15	120.54	40
19° 00'	225	30.75	221.38	7.69	105.45	11 15	118.42	19° 00'
20	225	30.75	221.38	8.09	107.43	11 15	116.38	20
40	225	30.75	221.38	8.47	109.34	11 15	114.41	40
20° 00'	225	30.75	221.38	8.83	111.19	11° 15'	112.50	20° 00'
20	250	41.32	244.03	9.31	113.23	13 45	135.25	20
40	250	41.32	244.03	9.82	115.32	13 45	133.06	40
21° 00'	250	41.32	244.03	10.32	117.35	13 45	130.95	21° 00'
20	250	41.32	244.03	10.80	119.30	13 45	128.91	20
40	250	41.32	244.03	11.26	121.20	13 45	126.92	40
22° 00'	250	41.32	244.03	11.71	123.04	13° 45'	125.00	22° 00'

TABLE IV.—ELEMENTS OF TRANSITION CURVES.

Dimensions of various 2°-per-25-feet spirals.—Part C.

VALUES OF AN .

Δ	Degree of curve.									
	4°	6°	8°	10°	12°	14°	16°	18°	20°	22°
4°	0.00	0.01	0.02	0.04	0.07	0.11	0.16	0.23	0.31	0.41
6	0.00	0.01	0.03	0.06	0.10	0.16	0.24	0.34	0.46	0.61
8	0.00	0.02	0.04	0.08	0.13	0.21	0.32	0.45	0.62	0.82
10	0.00	0.02	0.05	0.10	0.17	0.27	0.40	0.56	0.77	1.02
12	0.01	0.02	0.06	0.11	0.20	0.32	0.48	0.68	0.93	1.23
14	0.01	0.03	0.07	0.13	0.23	0.37	0.56	0.79	1.08	1.44
16	0.01	0.03	0.08	0.15	0.27	0.43	0.64	0.91	1.24	1.65
18	0.01	0.03	0.09	0.17	0.30	0.48	0.72	1.02	1.40	1.85
20	0.01	0.04	0.10	0.19	0.34	0.53	0.80	1.14	1.56	2.06
22	0.01	0.04	0.11	0.21	0.37	0.59	0.88	1.25	1.72	2.28
24	0.01	0.05	0.12	0.23	0.40	0.64	0.96	1.37	1.88	2.49
26	0.01	0.05	0.13	0.25	0.44	0.70	1.05	1.49	2.04	2.70
28	0.01	0.05	0.14	0.27	0.47	0.76	1.13	1.61	2.20	2.92
30	0.01	0.06	0.15	0.29	0.51	0.81	1.22	1.73	2.37	3.14
32	0.02	0.06	0.16	0.31	0.54	0.87	1.30	1.85	2.53	3.36
34	0.02	0.07	0.17	0.33	0.58	0.93	1.39	1.97	2.70	3.58
36	0.02	0.07	0.18	0.35	0.62	0.99	1.47	2.10	2.87	3.80
38	0.02	0.08	0.19	0.37	0.65	1.04	1.56	2.22	3.04	4.03
40	0.02	0.08	0.20	0.40	0.69	1.10	1.65	2.35	3.22	4.26
42	0.02	0.08	0.21	0.42	0.73	1.16	1.74	2.48	3.39	4.49
44	0.02	0.09	0.22	0.44	0.77	1.23	1.83	2.61	3.57	4.73
46	0.02	0.09	0.23	0.46	0.81	1.29	1.93	2.74	3.75	4.97
48	0.02	0.10	0.24	0.48	0.85	1.35	2.02	2.87	3.93	5.21
50	0.03	0.10	0.25	0.51	0.89	1.41	2.11	3.01	4.12	5.46
52	0.03	0.11	0.27	0.53	0.93	1.48	2.21	3.15	4.31	5.71
54	0.03	0.11	0.28	0.55	0.97	1.54	2.31	3.29	4.50	5.97
56	0.03	0.12	0.29	0.58	1.01	1.61	2.41	3.43	4.70	6.23
58	0.03	0.12	0.30	0.60	1.05	1.68	2.51	3.58	4.90	6.49
60	0.03	0.13	0.31	0.63	1.10	1.75	2.62	3.73	5.10	6.76
62	0.03	0.13	0.33	0.65	1.14	1.82	2.73	3.88	5.31	7.04
64	0.03	0.14	0.34	0.68	1.19	1.90	2.83	4.03	5.52	7.32
66	0.03	0.14	0.35	0.71	1.23	1.97	2.95	4.19	5.74	7.60
68	0.04	0.15	0.37	0.73	1.28	2.05	3.06	4.35	5.96	7.90
70	0.04	0.15	0.38	0.76	1.33	2.12	3.18	4.52	6.19	8.20
72	0.04	0.16	0.40	0.78	1.38	2.20	3.30	4.69	6.42	8.51
74	0.04	0.16	0.41	0.81	1.43	2.28	3.42	4.86	6.66	8.82
76	0.04	0.17	0.43	0.84	1.48	2.37	3.54	5.04	6.90	9.15
78	0.04	0.18	0.44	0.88	1.54	2.46	3.67	5.22	7.15	9.48
80	0.05	0.18	0.46	0.91	1.59	2.54	3.81	5.41	7.41	9.82
82	0.05	0.19	0.47	0.94	1.65	2.64	3.94	5.61	7.68	10.18
84	0.05	0.20	0.49	0.98	1.71	2.73	4.08	5.81	7.95	10.54
86	0.05	0.20	0.51	1.02	1.77	2.83	4.23	6.02	8.24	10.92
88	0.05	0.21	0.53	1.05	1.83	2.93	4.38	6.23	8.53	11.31
90	0.05	0.22	0.55	1.09	1.90	3.03	4.54	6.45	8.83	11.71
92	0.06	0.23	0.56	1.13	1.97	3.14	4.70	6.68	9.15	12.12
94	0.06	0.23	0.58	1.17	2.04	3.25	4.86	6.92	9.47	12.56
96	0.06	0.24	0.61	1.21	2.11	3.37	5.04	7.17	9.81	13.00
98	0.06	0.25	0.63	1.25	2.19	3.49	5.22	7.42	10.16	13.47
100	0.06	0.26	0.65	1.30	2.26	3.61	5.41	7.69	10.53	13.95

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
100	00 000	043	087	130	173	216	260	303	346	389	
101	432	475	518	561	604	646	689	732	775	817	.1 4.3
102	860	902	945	987	*030	*072	*114	*157	*199	*241	.2 8.7
103	01 283	326	368	410	452	494	536	578	619	661	.3 13.0
104	703	745	787	828	870	911	953	994	*036	*077	.4 17.4
105	02 119	160	201	243	284	325	366	407	448	489	.5 21.7
106	530	571	612	653	694	735	775	816	857	898	.6 26.1
107	938	979	*019	*060	*100	*141	*181	*221	*262	*302	.7 30.4
108	03 342	382	422	463	503	543	583	623	663	703	.8 34.8
109	742	782	822	862	901	941	981	*020	*060	*100	.9 39.1
110	04 139	178	218	257	297	336	375	415	454	493	
111	532	571	610	649	688	727	766	805	844	883	.1 4.0
112	922	960	999	*038	*076	*115	*154	*192	*231	*269	.2 8.1
113	05 308	346	384	423	461	499	538	576	614	652	.3 12.1
114	690	728	766	804	842	880	918	956	994	*032	.4 16.2
115	06 070	107	145	183	220	258	296	333	371	408	.5 20.2
116	446	483	520	558	595	632	670	707	744	781	.6 24.3
117	818	855	893	930	967	*004	*040	*077	*114	*151	.7 28.3
118	07 188	225	261	298	335	372	408	445	481	518	.8 32.4
119	554	591	627	664	700	737	773	809	845	882	.9 36.4
120	918	954	990	*026	*062	*098	*134	*170	*206	*242	
121	08 278	314	350	386	422	457	493	529	564	600	.1 3.7
122	636	671	707	742	778	813	849	884	920	955	.2 7.5
123	990	*026	*061	*096	*131	*166	*202	*237	*272	*307	.3 11.2
124	09 342	377	412	447	482	517	552	586	621	656	.4 15.0
125	691	725	760	795	830	864	899	933	968	*002	.5 18.7
126	10 037	071	106	140	174	209	243	277	312	346	.6 22.5
127	380	414	448	483	517	551	585	619	653	687	.7 26.2
128	721	755	789	822	856	890	924	958	991	*025	.8 30.0
129	11 059	092	126	160	193	227	260	294	327	361	.9 33.7
130	394	427	461	494	528	561	594	627	661	694	
131	727	760	793	826	859	892	925	958	991	*024	.1 3.4
132	12 057	090	123	156	189	221	254	287	320	352	.2 6.9
133	385	418	450	483	515	548	580	613	645	678	.3 10.3
134	710	743	775	807	840	872	904	937	969	*001	.4 13.8
135	13 033	065	097	130	162	194	226	258	290	322	.5 17.2
136	354	386	417	449	481	513	545	577	608	640	.6 20.7
137	672	703	735	767	798	830	862	893	925	956	.7 24.1
138	988	*019	*051	*082	*113	*145	*176	*207	*239	*270	.8 27.6
139	14 301	332	364	395	426	457	488	519	550	582	.9 31.0
140	613	644	675	706	736	767	798	829	860	891	
141	922	952	983	*014	*045	*075	*106	*137	*167	*198	.1 3.1
142	15 229	259	290	320	351	381	412	442	473	503	.2 6.3
143	533	564	594	624	655	685	715	745	776	806	.3 9.4
144	836	866	896	926	956	986	*017	*047	*077	*107	.4 12.6
145	16 137	166	196	226	256	286	316	346	376	405	.5 15.7
146	435	465	494	524	554	584	613	643	672	702	.6 18.9
147	731	761	791	820	849	879	908	938	967	997	.7 22.0
148	17 026	055	085	114	143	172	202	231	260	289	.8 25.2
149	318	348	377	406	435	464	493	522	551	580	.9 28.3
150	609	638	667	696	725	753	782	811	840	869	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
150	17 609	638	667	696	725	753	782	811	840	869	
151	897	926	955	984	*012	*041	*070	*098	*127	*156	.1 2.9 2.8 2.7
152	18 184	213	241	270	298	327	355	384	412	440	.2 5.8 5.6 5.4
153	469	497	526	554	582	611	639	667	695	724	.3 8.7 8.4 8.1
154	752	780	808	836	864	893	921	949	977	*005	.4 11.6 11.2 10.8
155	19 033	061	089	117	145	173	201	229	256	284	.5 14.5 14.0 13.5
156	312	340	368	396	423	451	479	507	534	562	.6 17.4 16.8 16.2
157	590	617	645	673	700	728	755	783	810	838	.7 20.3 19.6 18.9
158	865	893	920	948	975	*003	*030	*057	*085	*112	.8 23.2 22.4 21.6
159	20 139	167	194	221	249	276	303	330	357	385	.9 26.1 25.2 24.3
160	412	439	466	493	520	547	574	601	628	655	
161	682	709	736	763	790	817	844	871	898	924	.1 2.6 2.6
162	951	978	*005	*032	*058	*085	*112	*139	*165	*192	.2 5.3 5.2
163	21 219	245	272	298	325	352	378	405	431	458	.3 7.9 7.8
164	484	511	537	564	590	616	643	669	695	722	.4 10.6 10.4
165	748	774	801	827	853	880	906	932	958	984	.5 13.2 13.0
166	22 011	037	063	089	115	141	167	193	219	245	.6 15.9 15.6
167	271	297	323	349	375	401	427	453	479	505	.7 18.5 18.2
168	531	557	582	608	634	660	686	711	737	763	.8 21.2 20.8
169	788	814	840	865	891	917	942	968	994	*019	.9 23.8 23.4
170	23 045	070	096	121	147	172	198	223	249	274	
171	299	325	350	375	401	426	451	477	502	527	.1 2.5 2.5 2.4
172	553	578	603	628	653	679	704	729	754	779	.2 5.1 5.0 4.8
173	804	829	855	880	905	930	955	980	*005	*030	.3 7.6 7.5 7.2
174	24 055	080	105	129	154	179	204	229	254	279	.4 10.2 10.0 9.6
175	304	328	353	378	403	427	452	477	502	526	.5 12.7 12.5 12.0
176	551	576	600	625	650	674	699	723	748	773	.6 15.3 15.0 14.4
177	797	822	846	871	895	920	944	968	993	*017	.7 17.8 17.5 16.8
178	25 042	066	091	115	139	164	188	212	237	261	.8 20.4 20.0 19.2
179	285	309	334	358	382	406	430	455	479	503	.9 22.9 22.5 21.6
180	527	551	575	599	623	647	672	696	720	744	
181	768	792	816	840	863	887	911	935	959	983	.1 2.3 2.3
182	26 007	031	055	078	102	126	150	174	197	221	.2 4.7 4.6
183	245	269	292	316	340	363	387	411	434	458	.3 7.0 6.9
184	482	505	529	552	576	599	623	646	670	693	.4 9.4 9.2
185	717	740	764	787	811	834	858	881	904	928	.5 11.7 11.5
186	951	974	998	*021	*044	*068	*091	*114	*137	*161	.6 14.1 13.8
187	27 184	207	230	254	277	300	323	346	369	392	.7 16.4 16.1
188	416	439	462	485	508	531	554	577	600	623	.8 18.8 18.4
189	646	669	692	715	738	761	784	806	829	852	.9 21.1 20.7
190	875	898	921	944	966	989	*012	*035	*058	*080	
191	28 103	126	149	171	194	217	239	262	285	307	.1 2.2 2.2 2.1
192	330	352	375	398	420	443	465	488	510	533	.2 4.5 4.4 4.3
193	555	578	600	623	645	668	690	713	735	758	.3 6.7 6.6 6.4
194	780	802	825	847	869	892	914	936	959	981	.4 9.0 8.8 8.6
195	29 003	025	048	070	092	114	137	159	181	203	.5 11.2 11.0 10.7
196	225	248	270	292	314	336	358	380	402	424	.6 13.5 13.2 12.9
197	446	468	490	512	534	556	578	600	622	644	.7 15.7 15.4 15.0
198	666	688	710	732	754	776	798	820	841	863	.8 18.0 17.6 17.2
199	885	907	929	950	972	994	*016	*038	*059	*081	.9 20.2 19.8 19.3
200	30 103	124	146	168	190	211	233	254	276	298	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
200	30 103	124	146	168	190	211	233	254	276	298	
201	319	341	363	384	406	427	449	470	492	513	.1 2.2 2.1
202	535	556	578	599	621	642	664	685	707	728	.2 4.4 4.2
203	749	771	792	813	835	856	878	899	920	941	.3 6.6 6.3
204	963	984	*005	*027	*048	*069	*090	*112	*133	*154	.4 8.8 8.4
205	31 175	196	217	239	260	281	302	323	344	365	.5 11.0 10.5
206	386	408	429	450	471	492	513	534	555	576	.6 13.2 12.6
207	597	618	639	660	681	702	722	743	764	785	.7 15.4 14.7
208	806	827	848	869	890	910	931	952	973	994	.8 17.6 16.8
209	32 014	035	056	077	097	118	139	160	180	201	.9 19.8 18.9
210	222	242	263	284	304	325	346	366	387	407	
211	428	449	469	490	510	531	551	572	592	613	.1 2.0 2.0
212	633	654	674	695	715	736	756	776	797	817	.2 4.1 4.0
213	838	858	878	899	919	940	960	980	*001	*021	.3 6.1 6.0
214	33 041	061	082	102	122	142	163	183	203	223	.4 8.2 8.0
215	244	264	284	304	324	344	365	385	405	425	.5 10.2 10.0
216	445	465	485	505	525	546	566	586	606	626	.6 12.3 12.0
217	646	666	686	706	726	746	766	786	806	825	.7 14.3 14.0
218	845	865	885	905	925	945	965	985	*004	*024	.8 16.4 16.0
219	34 044	064	084	104	123	143	163	183	203	222	.9 18.4 18.0
220	242	262	281	301	321	341	360	380	400	419	
221	439	459	478	498	518	537	557	576	596	615	.1 1.9 1.9
222	635	655	674	694	713	733	752	772	791	811	.2 3.9 3.8
223	830	850	869	889	908	928	947	966	986	*005	.3 5.8 5.7
224	35 025	044	063	083	102	121	141	160	179	199	.4 7.8 7.6
225	218	237	257	276	295	314	334	353	372	391	.5 9.7 9.5
226	411	430	449	468	487	507	526	545	564	583	.6 11.7 11.4
227	602	621	641	660	679	698	717	736	755	774	.7 13.6 13.3
228	793	812	831	850	869	888	907	926	945	964	.8 15.6 15.2
229	983	*002	*021	*040	*059	*078	*097	*116	*135	*154	.9 17.5 17.1
230	36 173	191	210	229	248	267	286	305	323	342	
231	361	380	399	417	436	455	474	492	511	530	.1 1.8 1.8
232	549	567	586	605	623	642	661	679	698	717	.2 3.7 3.6
233	735	754	773	791	810	828	847	866	884	903	.3 5.5 5.4
234	921	940	958	977	996	*014	*033	*051	*070	*088	.4 7.4 7.2
235	37 107	125	143	162	180	199	217	236	254	273	.5 9.2 9.0
236	291	309	328	346	364	383	401	420	438	456	.6 11.1 10.8
237	475	493	511	530	548	566	584	603	621	639	.7 12.9 12.6
238	657	676	694	712	730	749	767	785	803	821	.8 14.8 14.4
239	840	858	876	894	912	930	948	967	985	*003	.9 16.6 16.2
240	38 021	039	057	075	093	111	129	147	165	183	
241	201	219	237	255	273	291	309	327	345	363	.1 1.7 1.7
242	381	399	417	435	453	471	489	507	525	543	.2 3.5 3.4
243	560	578	596	614	632	650	667	685	703	721	.3 5.2 5.1
244	739	757	774	792	810	828	845	863	881	899	.4 7.0 6.8
245	916	934	952	970	987	*005	*023	*040	*058	*076	.5 8.7 8.5
246	39 093	111	129	146	164	181	199	217	234	252	.6 10.5 10.2
247	269	287	305	322	340	357	375	392	410	427	.7 12.2 11.9
248	445	462	480	497	515	532	550	567	585	602	.8 14.0 13.6
249	620	637	655	672	689	707	724	742	759	776	.9 15.7 15.3
250	794	811	828	846	863	881	898	915	933	950	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
250	39 794	811	828	846	863	881	898	915	933	950	
251	967	984	*002	*019	*036	*054	*071	*088	*105	*123	
252	40 140	157	174	191	209	226	243	260	277	295	17 17
253	312	329	346	363	380	398	415	432	449	466	.1 1.7 1.7
254	483	500	517	534	551	569	586	603	620	637	.2 3.5 3.4
255	654	671	688	705	722	739	756	773	790	807	.3 5.2 5.1
256	824	841	858	875	892	908	925	942	959	976	.4 7.0 6.8
257	993	*010	*027	*044	*061	*077	*094	*111	*128	*145	.5 8.7 8.5
258	41 162	179	195	212	229	246	263	279	296	313	.6 10.5 10.2
259	330	346	363	380	397	413	430	447	464	480	.7 12.2 11.9
260	497	514	530	547	564	581	597	614	631	647	.8 14.0 13.6
261	664	680	697	714	730	747	764	780	797	813	.9 15.7 15.3
262	830	846	863	880	896	913	929	946	962	979	
263	995	*012	*028	*045	*061	*078	*094	*111	*127	*144	
264	42 160	177	193	209	226	242	259	275	292	308	16 16
265	324	341	357	373	390	406	423	439	455	472	.1 1.6 1.6
266	488	504	521	537	553	569	586	602	618	635	.2 3.3 3.2
267	651	667	683	700	716	732	748	765	781	797	.3 4.9 4.8
268	813	829	846	862	878	894	910	927	943	959	.4 6.6 6.4
269	975	991	*007	*023	*040	*056	*072	*088	*104	*120	.5 8.2 8.0
270	43 136	152	168	184	200	216	233	249	265	281	.6 9.9 9.6
271	297	313	329	345	361	377	393	409	425	441	.7 11.5 11.2
272	457	473	489	505	520	536	552	568	584	600	.8 13.2 12.8
273	616	632	648	664	680	695	711	727	743	759	.9 14.8 14.4
274	775	791	806	822	838	854	870	886	901	917	
275	933	949	965	980	996	*012	*028	*043	*059	*075	
276	44 091	106	122	138	154	169	185	201	216	232	
277	248	263	279	295	310	326	342	357	373	389	
278	404	420	435	451	467	482	498	513	529	545	15 15
279	560	576	591	607	622	638	653	669	685	700	.1 1.5 1.5
280	716	731	747	762	778	793	809	824	839	855	.2 3.1 3.0
281	870	886	901	917	932	948	963	978	994	*009	.3 4.6 4.5
282	45 025	040	055	071	086	102	117	132	148	163	.4 6.2 6.0
283	178	194	209	224	240	255	270	286	301	316	.5 7.7 7.5
284	332	347	362	377	393	408	423	438	454	469	.6 9.3 9.0
285	484	499	515	530	545	560	576	591	606	621	.7 10.8 10.5
286	636	652	667	682	697	712	727	743	758	773	.8 12.4 12.0
287	788	803	818	833	848	864	879	894	909	924	.9 13.9 13.5
288	939	954	969	984	999	*014	*029	*044	*059	*075	
289	46 090	105	120	135	150	165	180	195	210	225	
290	240	255	269	284	299	314	329	344	359	374	14 14
291	389	404	419	434	449	464	479	493	508	523	.1 1.4 1.4
292	538	553	568	583	597	612	627	642	657	672	.2 2.9 2.8
293	687	701	716	731	746	761	775	790	805	820	.3 4.3 4.2
294	834	849	864	879	894	908	923	938	952	967	.4 5.8 5.6
295	982	997	*011	*026	*041	*055	*070	*085	*100	*114	.5 7.2 7.0
296	47 129	144	158	173	188	202	217	232	246	261	.6 8.7 8.4
297	275	290	305	319	334	348	363	378	392	407	.7 10.1 9.8
298	421	436	451	465	480	494	509	523	538	552	.8 11.6 11.2
299	567	581	596	610	625	639	654	668	683	697	.9 13.0 12.6
300	712	726	741	755	770	784	799	813	828	842	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
300	47 712	726	741	755	770	784	799	813	828	842	
301	856	871	885	900	914	928	943	957	972	986	
302	48 000	015	029	044	058	072	087	101	115	130	
303	144	158	173	187	201	216	230	244	259	273	
304	287	301	316	330	344	358	373	387	401	415	
305	430	444	458	472	487	501	515	529	543	558	
306	572	586	600	614	629	643	657	671	685	699	
307	714	728	742	756	770	784	798	812	827	841	
308	855	869	883	897	911	925	939	953	967	982	
309	996	*010	*024	*038	*052	*066	*080	*094	*108	*122	
310	49 136	150	164	178	192	206	220	234	248	262	
311	276	290	304	318	332	346	359	373	387	401	
312	415	429	443	457	471	485	499	513	526	540	
313	554	568	582	596	610	624	637	651	665	679	
314	693	707	720	734	748	762	776	789	803	817	
315	831	845	858	872	886	900	913	927	941	955	
316	968	982	996	*010	*023	*037	*051	*065	*078	*092	
317	50 106	119	133	147	160	174	188	201	215	229	
318	242	256	270	283	297	311	324	338	352	365	
319	379	392	406	420	433	447	460	474	488	501	
320	515	528	542	555	569	583	596	610	623	637	
321	650	664	677	691	704	718	731	745	758	772	
322	785	799	812	826	839	853	866	880	893	907	
323	920	933	947	960	974	987	*001	*014	*027	*041	
324	51 054	068	081	094	108	121	135	148	161	175	
325	188	201	215	228	242	255	268	282	295	308	
326	322	335	348	361	375	388	401	415	428	441	
327	455	468	481	494	508	521	534	547	561	574	
328	587	600	614	627	640	653	667	680	693	706	
329	719	733	746	759	772	785	798	812	825	838	
330	851	864	877	891	904	917	930	943	956	969	
331	983	996	*009	*022	*035	*048	*061	*074	*087	*100	
332	52 114	127	140	153	166	179	192	205	218	231	
333	244	257	270	283	296	309	322	335	348	361	
334	374	387	400	413	426	439	452	465	478	491	
335	504	517	530	543	556	569	582	595	608	621	
336	634	647	660	672	685	698	711	724	737	750	
337	763	776	789	801	814	827	840	853	866	879	
338	891	904	917	930	943	956	968	981	994	*007	
339	53 020	033	045	058	071	084	097	109	122	135	
340	148	160	173	186	199	211	224	237	250	262	
341	275	288	301	313	326	339	352	364	377	390	
342	402	415	428	440	453	466	478	491	504	516	
343	529	542	554	567	580	592	605	618	630	643	
344	656	668	681	693	706	719	731	744	756	769	
345	782	794	807	819	832	845	857	870	882	895	
346	907	920	932	945	958	970	983	995	*008	*020	
347	54 033	045	058	070	083	095	108	120	133	145	
348	158	170	183	195	208	220	232	245	257	270	
349	282	295	307	320	332	344	357	369	382	394	
350	407	419	431	444	456	469	481	493	506	518	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

14 14
.1 1.4 1.4
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TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
350	54 407	419	431	444	456	469	481	493	506	518	
351	530	543	555	568	580	592	605	617	629	642	.1 1.2
352	654	666	679	691	703	716	728	740	753	765	.2 2.5
353	777	790	802	814	826	839	851	863	876	888	.3 3.7
354	900	912	925	937	949	961	974	986	998	*010	.4 5.0
355	55 023	035	047	059	071	084	096	108	120	133	.5 6.2
356	145	157	169	181	194	206	218	230	242	254	.6 7.5
357	267	279	291	303	315	327	340	352	364	376	.7 8.7
358	388	400	412	424	437	449	461	473	485	497	.8 10.0
359	509	521	533	545	558	570	582	594	606	618	.9 11.2
360	630	642	654	666	678	690	702	714	726	738	
361	750	762	775	787	799	811	823	835	847	859	.1 1.2
362	871	883	895	907	919	931	943	955	966	978	.2 2.4
363	990	*002	*014	*026	*038	*050	*062	*074	*086	*098	.3 3.6
364	56 110	122	134	146	158	170	181	193	205	217	.4 4.8
365	229	241	253	265	277	288	300	312	324	336	.5 6.0
366	348	360	372	383	395	407	419	431	443	455	.6 7.2
367	466	478	490	502	514	525	537	549	561	573	.7 8.4
368	585	596	608	620	632	643	655	667	679	691	.8 9.6
369	702	714	726	738	749	761	773	785	796	808	.9 10.8
370	820	832	843	855	867	879	890	902	914	925	
371	937	949	961	972	984	996	*007	*019	*031	*042	.1 1.1
372	57 054	066	077	089	101	112	124	136	147	159	.2 2.3
373	171	182	194	206	217	229	240	252	264	275	.3 3.4
374	287	299	310	322	333	345	357	368	380	391	.4 4.6
375	403	414	426	438	449	461	472	484	495	507	.5 5.7
376	519	530	542	553	565	576	588	599	611	622	.6 6.9
377	634	645	657	668	680	691	703	714	726	737	.7 8.0
378	749	760	772	783	795	806	818	829	841	852	.8 9.2
379	864	875	887	898	909	921	932	944	955	967	.9 10.3
380	978	990	*001	*012	*024	*035	*047	*058	*069	*081	
381	58 092	104	115	126	138	149	161	172	183	195	.1 1.1
382	206	217	229	240	252	263	274	286	297	308	.2 2.2
383	320	331	342	354	365	376	388	399	410	422	.3 3.3
384	433	444	455	467	478	489	501	512	523	535	.4 4.4
385	546	557	568	580	591	602	613	625	636	647	.5 5.5
386	658	670	681	692	703	715	726	737	748	760	.6 6.6
387	771	782	793	804	816	827	838	849	861	872	.7 7.7
388	883	894	905	916	928	939	950	961	972	984	.8 8.8
389	995	*006	*017	*028	*039	*050	*062	*073	*084	*095	.9 9.9
390	59 106	117	128	140	151	162	173	184	195	206	
391	217	229	240	251	262	273	284	295	306	317	.1 1.0
392	328	339	351	362	373	384	395	406	417	428	.2 2.1
393	439	450	461	472	483	494	505	516	527	538	.3 3.1
394	549	560	571	582	593	604	615	626	637	648	.4 4.2
395	659	670	681	692	703	714	725	736	747	758	.5 5.2
396	769	780	791	802	813	824	835	846	857	868	.6 6.3
397	879	890	901	912	923	933	944	955	966	977	.7 7.3
398	988	999	*010	*021	*032	*043	*053	*064	*075	*086	.8 8.4
399	60 097	108	119	130	141	151	162	173	184	195	.9 9.4
400	206	217	227	238	249	260	271	282	293	303	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
400	60 206	217	227	238	249	260	271	282	293	303	
401	314	325	336	347	357	368	379	390	401	412	
402	422	433	444	455	466	476	487	498	509	519	
403	530	541	552	563	573	584	595	606	616	627	11
404	638	649	659	670	681	692	702	713	724	735	.1 1.1
405	745	756	767	777	788	799	810	820	831	842	.2 2.2
406	852	863	874	884	895	906	916	927	938	949	.3 3.3
407	959	970	981	991	*002	*013	*023	*034	*044	*055	.4 4.4
408	61 066	076	087	098	108	119	130	140	151	161	.5 5.5
409	172	183	193	204	215	225	236	246	257	268	.6 6.6
410	278	289	299	310	320	331	342	352	363	373	.7 7.7
411	384	394	405	416	426	437	447	458	468	479	.8 8.8
412	489	500	511	521	532	542	553	563	574	584	.9 9.9
413	595	605	616	626	637	647	658	668	679	689	
414	700	710	721	731	742	752	763	773	784	794	
415	805	815	825	836	846	857	867	878	888	899	10
416	909	920	930	940	951	961	972	982	993	*003	.1 1.0
417	62 013	024	034	045	055	065	076	086	097	107	.2 2.1
418	117	128	138	149	159	169	180	190	200	211	.3 3.1
419	221	232	242	252	263	273	283	294	304	314	.4 4.2
420	325	335	345	356	366	376	387	397	407	418	.5 5.2
421	428	438	449	459	469	480	490	500	510	521	.6 6.3
422	531	541	552	562	572	582	593	603	613	624	.7 7.3
423	634	644	654	665	675	685	695	706	716	726	.8 8.4
424	736	747	757	767	777	788	798	808	818	828	.9 9.4
425	839	849	859	869	879	890	900	910	920	931	
426	941	951	961	971	981	992	*002	*012	*022	*032	
427	63 043	053	063	073	083	093	104	114	124	134	10
428	144	154	164	175	185	195	205	215	225	235	.1 1.0
429	245	256	266	276	286	296	306	316	326	336	.2 2.0
430	347	357	367	377	387	397	407	417	427	437	.3 3.0
431	447	458	468	478	488	498	508	518	528	538	.4 4.0
432	548	558	568	578	588	598	608	618	628	639	.5 5.0
433	649	659	669	679	689	699	709	719	729	739	.6 6.0
434	749	759	769	779	789	799	809	819	829	839	.7 7.0
435	849	859	869	879	889	899	909	919	928	938	.8 8.0
436	948	958	968	978	988	998	*008	*018	*028	*038	.9 9.0
437	64 048	058	068	078	088	098	107	117	127	137	
438	147	157	167	177	187	197	207	217	226	236	
439	246	256	266	276	286	296	306	315	325	335	9
440	345	355	365	375	384	394	404	414	424	434	.1 0.9
441	444	453	463	473	483	493	503	512	522	532	.2 1.9
442	542	552	562	571	581	591	601	611	621	630	.3 2.8
443	640	650	660	670	679	689	699	709	718	728	.4 3.8
444	738	748	758	767	777	787	797	806	816	826	.5 4.7
445	836	846	855	865	875	885	894	904	914	923	.6 5.7
446	933	943	953	962	972	982	992	*001	*011	*021	.7 6.6
447	65 031	040	050	060	069	079	089	098	108	118	.8 7.6
448	128	137	147	157	166	176	186	195	205	215	.9 8.5
449	224	234	244	253	263	273	282	292	302	311	
450	321	331	340	350	360	369	379	389	398	408	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.	
450	65 321	331	340	350	360	369	379	389	398	408		
451	417	427	437	446	456	466	475	485	494	504		
452	514	523	533	542	552	562	571	581	590	600		
453	610	619	629	638	648	657	667	677	686	696	.1	1.0
454	705	715	724	734	744	753	763	772	782	791	.2	2.0
455	801	810	820	830	839	849	858	868	877	887	.3	3.0
456	896	906	915	925	934	944	953	963	972	982	.4	4.0
457	991	*001	*010	*020	*029	*039	*048	*058	*067	*077	.5	5.0
458	66 086	096	105	115	124	134	143	153	162	172	.6	6.0
459	181	190	200	209	219	228	238	247	257	266	.7	7.0
											.8	8.0
											.9	9.0
460	276	285	294	304	313	323	332	342	351	360		
461	370	379	389	398	408	417	426	436	445	455		
462	464	473	483	492	502	511	520	530	539	548		
463	558	567	577	586	595	605	614	623	633	642		
464	652	661	670	680	689	698	708	717	726	736		
465	745	754	764	773	782	792	801	810	820	829		
466	838	848	857	866	876	885	894	904	913	922	.1	0.9
467	931	941	950	959	969	978	987	996	*006	*015	.2	1.9
468	67 024	034	043	052	061	071	080	089	099	108	.3	2.8
469	117	126	136	145	154	163	173	182	191	200	.4	3.8
											.5	4.7
											.6	5.7
											.7	6.6
											.8	7.6
											.9	8.5
470	210	219	228	237	246	256	265	274	283	293		
471	302	311	320	329	339	348	357	366	376	385		
472	394	403	412	422	431	440	449	458	467	477		
473	486	495	504	513	523	532	541	550	559	568		
474	578	587	596	605	614	623	633	642	651	660		
475	669	678	687	697	706	715	724	733	742	751		
476	760	770	779	788	797	806	815	824	833	842		
477	852	861	870	879	888	897	906	915	924	933		
478	943	952	961	970	979	988	997	*006	*015	*024		
479	68 033	042	051	060	070	079	088	097	106	115	.1	9
											.2	0.9
											.3	1.8
											.4	2.7
											.5	3.6
											.6	4.5
											.7	5.4
											.8	6.3
											.9	7.2
												8.1
480	124	133	142	151	160	169	178	187	196	205		
481	214	223	232	241	250	259	268	277	286	295		
482	304	313	322	331	340	349	358	367	376	385		
483	394	403	412	421	430	439	448	457	466	475		
484	484	493	502	511	520	529	538	547	556	565		
485	574	583	592	601	610	619	628	637	646	654		
486	663	672	681	690	699	708	717	726	735	744		
487	753	762	770	779	788	797	806	815	824	833		
488	842	851	860	868	877	886	895	904	913	922		
489	931	940	948	957	966	975	984	993	*002	*010		
490	69 019	028	037	046	055	064	073	081	090	099		
491	108	117	126	134	143	152	161	170	179	187	.1	0.8
492	196	205	214	223	232	240	249	258	267	276	.2	1.7
493	284	293	302	311	320	328	337	346	355	364	.3	2.5
494	372	381	390	399	408	416	425	434	443	451	.4	3.4
495	460	469	478	487	495	504	513	522	530	539	.5	4.2
496	548	557	565	574	583	592	600	609	618	627	.6	5.1
497	635	644	653	662	670	679	688	697	705	714	.7	5.9
498	723	731	740	749	758	766	775	784	792	801	.8	6.8
499	810	819	827	836	845	853	862	871	879	888	.9	7.6
500	897	905	914	923	931	940	949	958	966	975		
N.	0	1	2	3	4	5	6	7	8	9	P. P.	

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
500	69 897	905	914	923	931	940	949	958	966	975	
501	984	992	*001	*010	*018	*027	*036	*044	*053	*061	
502	70 070	079	087	096	105	113	122	131	139	148	9
503	157	165	174	182	191	200	208	217	226	234	.1 0.9
504	243	251	260	269	277	286	294	303	312	320	.2 1.8
505	329	337	346	355	363	372	380	389	398	406	.3 2.7
506	415	423	432	441	449	458	466	475	483	492	.4 3.6
507	501	509	518	526	535	543	552	560	569	578	.5 4.5
508	586	595	603	612	620	629	637	646	654	663	.6 5.4
509	672	680	689	697	706	714	723	731	740	748	.7 6.3
510	757	765	774	782	791	799	808	816	825	833	.8 7.2
511	842	850	859	867	876	884	893	901	910	918	.9 8.1
512	927	935	944	952	961	969	978	986	995	*003	
513	71 011	020	028	037	045	054	062	071	079	088	
514	096	105	113	121	130	138	147	155	164	172	8
515	180	189	197	206	214	223	231	239	248	256	.1 0.8
516	265	273	282	290	298	307	315	324	332	340	.2 1.7
517	349	357	366	374	382	391	399	408	416	424	.3 2.5
518	433	441	449	458	466	475	483	491	500	508	.4 3.4
519	516	525	533	542	550	558	567	575	583	592	.5 4.2
520	600	608	617	625	633	642	650	659	667	675	.6 5.1
521	684	692	700	709	717	725	734	742	750	758	.7 5.9
522	767	775	783	792	800	808	817	825	833	842	.8 6.8
523	850	858	867	875	883	891	900	908	916	925	.9 7.6
524	933	941	949	958	966	974	983	991	999	*007	
525	72 016	024	032	040	049	057	065	074	082	090	
526	098	107	115	123	131	140	148	156	164	173	8
527	181	189	197	206	214	222	230	238	247	255	.1 0.8
528	263	271	280	288	296	304	312	321	329	337	.2 1.6
529	345	354	362	370	378	386	395	403	411	419	.3 2.4
530	427	436	444	452	460	468	476	485	493	501	.4 3.2
531	509	517	526	534	542	550	558	566	575	583	.5 4.0
532	591	599	607	615	624	632	640	648	656	664	.6 4.8
533	672	681	689	697	705	713	721	729	738	746	.7 5.6
534	754	762	770	778	786	795	803	811	819	827	.8 6.4
535	835	843	851	859	868	876	884	892	900	908	.9 7.2
536	916	924	932	941	949	957	965	973	981	989	
537	997	*005	*013	*021	*030	*038	*046	*054	*062	*070	
538	73 078	086	094	102	110	118	126	134	143	151	
539	159	167	175	183	191	199	207	215	223	231	
540	239	247	255	263	271	279	287	295	303	311	7
541	319	328	336	344	352	360	368	376	384	392	.1 0.7
542	400	408	416	424	432	440	448	456	464	472	.2 1.5
543	480	488	496	504	512	520	528	536	544	552	.3 2.2
544	560	568	576	584	592	600	608	615	623	631	.4 3.0
545	639	647	655	663	671	679	687	695	703	711	.5 3.7
546	719	727	735	743	751	759	767	775	783	791	.6 4.5
547	798	806	814	822	830	838	846	854	862	870	.7 5.2
548	878	886	894	902	909	917	925	933	941	949	.8 6.0
549	957	965	973	981	989	997	*004	*012	*020	*028	.9 6.7
550	74 036	044	052	060	068	075	083	091	099	107	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
550	74 036	044	052	060	068	075	083	091	099	107	
551	115	123	131	139	146	154	162	170	178	186	
552	194	202	209	217	225	233	241	249	257	264	
553	272	280	288	296	304	312	319	327	335	343	
554	351	359	366	374	382	390	398	406	413	421	
555	429	437	445	453	460	468	476	484	492	499	
556	507	515	523	531	538	546	554	562	570	577	
557	585	593	601	609	616	624	632	640	648	655	
558	663	671	679	687	694	702	710	718	725	733	.1 0.8
559	741	749	756	764	772	780	788	795	803	811	.2 1.6
											.3 2.4
560	819	826	834	842	850	857	865	873	881	888	.4 3.2
											.5 4.0
561	896	904	912	919	927	935	942	950	958	966	.6 4.8
562	973	981	989	997	*004	*012	*020	*027	*035	*043	.7 5.6
563	75 051	058	066	074	081	089	097	105	112	120	.8 6.4
564	128	135	143	151	158	166	174	182	189	197	.9 7.2
565	205	212	220	228	235	243	251	258	266	274	
566	281	289	297	304	312	320	327	335	343	350	
567	358	366	373	381	389	396	404	412	419	427	
568	435	442	450	458	465	473	480	488	496	503	
569	511	519	526	534	541	549	557	564	572	580	
570	587	595	602	610	618	625	633	641	648	656	
571	663	671	679	686	694	701	709	717	724	732	.1 0.7
572	739	747	755	762	770	777	785	792	800	808	.2 1.5
573	815	823	830	838	846	853	861	868	876	883	.3 2.2
574	891	899	906	914	921	929	936	944	951	959	.4 3.0
575	967	974	982	989	997	*004	*012	*019	*027	*034	.5 3.7
576	76 042	050	057	065	072	080	087	095	102	110	.6 4.5
577	117	125	132	140	147	155	162	170	178	185	.7 5.2
578	193	200	208	215	223	230	238	245	253	260	.8 6.0
579	268	275	283	290	298	305	313	320	328	335	.9 6.7
580	343	350	358	365	372	380	387	395	402	410	
581	417	425	432	440	447	455	462	470	477	485	
582	492	500	507	514	522	529	537	544	552	559	
583	567	574	582	589	596	604	611	619	626	634	
584	641	648	656	663	671	678	686	693	700	708	
585	715	723	730	738	745	752	760	767	775	782	
586	790	797	804	812	819	827	834	841	849	856	
587	864	871	878	886	893	901	908	915	923	930	
588	937	945	952	960	967	974	982	989	997	*004	.1 0.7
589	77 011	019	026	033	041	048	055	063	070	078	.2 1.4
											.3 2.1
590	085	092	100	107	114	122	129	136	144	151	.4 2.8
											.5 3.5
591	158	166	173	181	188	195	203	210	217	225	.6 4.2
592	232	239	247	254	261	269	276	283	291	298	.7 4.9
593	305	313	320	327	335	342	349	356	364	371	.8 5.6
594	378	386	393	400	408	415	422	430	437	444	.9 6.3
595	451	459	466	473	481	488	495	503	510	517	
596	524	532	539	546	554	561	568	575	583	590	
597	597	604	612	619	626	634	641	648	655	663	
598	670	677	684	692	699	706	713	721	728	735	
599	742	750	757	764	771	779	786	793	800	808	
600	815	822	829	837	844	851	858	866	873	880	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
600	77 815	822	829	837	844	851	858	866	873	880	
601	887	894	902	909	916	923	931	938	945	952	
602	959	967	974	981	988	995	*003	*010	*017	*024	
603	78 031	039	046	053	060	067	075	082	089	096	
604	103	111	118	125	132	139	147	154	161	168	
605	175	182	190	197	204	211	218	226	233	240	
606	247	254	261	269	276	283	290	297	304	311	
607	319	326	333	340	347	354	362	369	376	383	
608	390	397	404	412	419	426	433	440	447	454	
609	461	469	476	483	490	497	504	511	518	526	
610	533	540	547	554	561	568	575	583	590	597	
611	604	611	618	625	632	639	646	654	661	668	
612	675	682	689	696	703	710	717	725	732	739	
613	746	753	760	767	774	781	788	795	802	810	
614	817	824	831	838	845	852	859	866	873	880	
615	887	894	901	908	915	923	930	937	944	951	
616	958	965	972	979	986	993	*000	*007	*014	*021	
617	79 028	035	042	049	056	063	070	078	085	092	
618	099	106	113	120	127	134	141	148	155	162	
619	169	176	183	190	197	204	211	218	225	232	
620	239	246	253	260	267	274	281	288	295	302	
621	309	316	323	330	337	344	351	358	365	372	
622	379	386	393	400	407	414	421	428	435	442	
623	449	456	462	469	476	483	490	497	504	511	
624	518	525	532	539	546	553	560	567	574	581	
625	588	595	602	609	616	622	629	636	643	650	
626	657	664	671	678	685	692	699	706	713	720	
627	727	733	740	747	754	761	768	775	782	789	
628	796	803	810	816	823	830	837	844	851	858	
629	865	872	879	886	892	899	906	913	920	927	
630	934	941	948	954	961	968	975	982	989	996	
631	80 003	010	016	023	030	037	044	051	058	065	
632	071	078	085	092	099	106	113	120	126	133	
633	140	147	154	161	168	174	181	188	195	202	
634	209	216	222	229	236	243	250	257	263	270	
635	277	284	291	298	304	311	318	325	332	339	
636	345	352	359	366	373	380	386	393	400	407	
637	414	421	427	434	441	448	455	461	468	475	
638	482	489	495	502	509	516	523	529	536	543	
639	550	557	563	570	577	584	591	597	604	611	
640	618	625	631	638	645	652	658	665	672	679	
641	686	692	699	706	713	719	726	733	740	746	
642	753	760	767	774	780	787	794	801	807	814	
643	821	828	834	841	848	855	861	868	875	882	
644	888	895	902	909	915	922	929	936	942	949	
645	956	962	969	976	983	989	996	*003	*010	*016	
646	81 023	030	036	043	050	057	063	070	077	083	
647	090	097	104	110	117	124	130	137	144	151	
648	157	164	171	177	184	191	197	204	211	218	
649	224	231	238	244	251	258	264	271	278	284	
650	291	298	304	311	318	324	331	338	345	351	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

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TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
650	81 291	298	304	311	318	324	331	338	345	351	
651	358	365	371	378	385	391	398	405	411	418	
652	425	431	438	444	451	458	464	471	478	484	
653	491	498	504	511	518	524	531	538	544	551	
654	558	564	571	577	584	591	597	604	611	617	
655	624	631	637	644	650	657	664	670	677	684	
656	690	697	703	710	717	723	730	736	743	750	
657	756	763	770	776	783	789	796	803	809	816	
658	822	829	836	842	849	855	862	869	875	882	
659	888	895	901	908	915	921	928	934	941	948	
660	954	961	967	974	980	987	994	*000	*007	*013	
661	82 020	026	033	040	046	053	059	066	072	079	
662	086	092	099	105	112	118	125	131	138	145	
663	151	158	164	171	177	184	190	197	203	210	
664	217	223	230	236	243	249	256	262	269	275	
665	282	288	295	302	308	315	321	328	334	341	
666	347	354	360	367	373	380	386	393	399	406	
667	412	419	425	432	438	445	451	458	464	471	
668	477	484	490	497	503	510	516	523	529	536	
669	542	549	555	562	568	575	581	588	594	601	
670	607	614	620	627	633	640	646	653	659	666	
671	672	678	685	691	698	704	711	717	724	730	
672	737	743	750	756	763	769	775	782	788	795	
673	801	808	814	821	827	834	840	846	853	859	
674	866	872	879	885	892	898	904	911	917	924	
675	930	937	943	949	956	962	969	975	982	988	
676	994	*001	*007	*014	*020	*027	*033	*039	*046	*052	
677	83 059	065	071	078	084	091	097	103	110	116	
678	123	129	136	142	148	155	161	168	174	180	
679	187	193	200	206	212	219	225	231	238	244	
680	251	257	263	270	276	283	289	295	302	308	
681	314	321	327	334	340	346	353	359	365	372	
682	378	385	391	397	404	410	416	423	429	435	
683	442	448	455	461	467	474	480	486	493	499	
684	505	512	518	524	531	537	543	550	556	562	
685	569	575	581	588	594	600	607	613	619	626	
686	632	638	645	651	657	664	670	676	683	689	
687	695	702	708	714	721	727	733	740	746	752	
688	759	765	771	778	784	790	796	803	809	815	
689	822	828	834	841	847	853	859	866	872	878	
690	885	891	897	904	910	916	922	929	935	941	
691	948	954	960	966	973	979	985	992	998	*004	
692	84 010	017	023	029	035	042	048	054	061	067	
693	073	079	086	092	098	104	111	117	123	129	
694	136	142	148	154	161	167	173	179	186	192	
695	198	204	211	217	223	229	236	242	248	254	
696	261	267	273	279	286	292	298	304	311	317	
697	323	329	335	342	348	354	360	367	373	379	
698	385	392	398	404	410	416	423	429	435	441	
699	447	454	460	466	472	479	485	491	497	503	
700	510	516	522	528	534	541	547	553	559	565	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

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TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
700	84 510	516	522	528	534	541	547	553	559	565	
701	572	578	584	590	596	603	609	615	621	627	
702	633	640	646	652	658	664	671	677	683	689	
703	695	701	708	714	720	726	732	739	745	751	
704	757	763	769	776	782	788	794	800	806	813	
705	819	825	831	837	843	849	856	862	868	874	
706	880	886	893	899	905	911	917	923	929	936	
707	942	948	954	960	966	972	979	985	991	997	
708	85 003	009	015	021	028	034	040	046	052	058	.1 0.6
709	064	070	077	083	089	095	101	107	113	119	.2 1.3
710	126	132	138	144	150	156	162	168	174	181	.3 1.9
711	187	193	199	205	211	217	223	229	236	242	.4 2.6
712	248	254	260	266	272	278	284	290	297	303	.5 3.2
713	309	315	321	327	333	339	345	351	357	363	.6 3.9
714	370	376	382	388	394	400	406	412	418	424	.7 4.5
715	430	436	443	449	455	461	467	473	479	485	.8 5.2
716	491	497	503	509	515	521	527	533	540	546	.9 5.8
717	552	558	564	570	576	582	588	594	600	606	
718	612	618	624	630	636	642	648	655	661	667	
719	673	679	685	691	697	703	709	715	721	727	
720	733	739	745	751	757	763	769	775	781	787	
721	793	799	805	811	817	823	829	835	841	847	6
722	853	859	865	872	878	884	890	896	902	908	.1 0.6
723	914	920	926	932	938	944	950	956	962	968	.2 1.2
724	974	980	986	992	998	*004	*010	*016	*022	*028	.3 1.8
725	86 034	040	046	052	058	063	069	075	081	087	.4 2.4
726	093	099	105	111	117	123	129	135	141	147	.5 3.0
727	153	159	165	171	177	183	189	195	201	207	.6 3.6
728	213	219	225	231	237	243	249	255	261	267	.7 4.2
729	273	278	284	290	296	302	308	314	320	326	.8 4.8
730	332	338	344	350	356	362	368	374	380	386	.9 5.4
731	391	397	403	409	415	421	427	433	439	445	
732	451	457	463	469	475	481	486	492	498	504	
733	510	516	522	528	534	540	546	552	558	563	
734	569	575	581	587	593	599	605	611	617	623	
735	628	634	640	646	652	658	664	670	676	682	
736	688	693	699	705	711	717	723	729	735	741	
737	746	752	758	764	770	776	782	788	794	800	
738	805	811	817	823	829	835	841	847	852	858	.1 0.5
739	864	870	876	882	888	894	899	905	911	917	.2 1.1
740	923	929	935	941	946	952	958	964	970	976	.3 1.6
741	982	987	993	999	*005	*011	*017	*023	*028	*034	.4 2.2
742	87 040	046	052	058	064	069	075	081	087	093	.5 2.7
743	099	104	110	116	122	128	134	140	145	151	.6 3.3
744	157	163	169	175	180	186	192	198	204	210	.7 3.8
745	215	221	227	233	239	245	250	256	262	268	.8 4.4
746	274	279	285	291	297	303	309	314	320	326	.9 4.9
747	332	338	343	349	355	361	367	372	378	384	
748	390	396	402	407	413	419	425	431	436	442	
749	448	454	460	465	471	477	483	489	494	500	
750	506	512	517	523	529	535	541	546	552	558	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
750	87 506	512	517	523	529	535	541	546	552	558	
751	564	570	575	581	587	593	598	604	610	616	
752	622	627	633	639	645	650	656	662	668	673	
753	679	685	691	697	702	708	714	720	725	731	
754	737	743	748	754	760	766	771	777	783	789	
755	794	800	806	812	817	823	829	835	840	846	
756	852	858	863	869	875	881	886	892	898	904	
757	909	915	921	927	932	938	944	949	955	961	
758	967	972	978	984	990	995	*001	*007	*012	*018	
759	88 024	030	035	041	047	053	058	064	070	075	.1 6 .2 0.6 .3 1.2 .4 1.8 .5 2.4 .6 3.0 .7 3.6 .8 4.2 .9 4.8
760	081	087	093	098	104	110	115	121	127	133	.9 5.4
761	138	144	150	155	161	167	172	178	184	190	
762	195	201	207	212	218	224	229	235	241	247	
763	252	258	264	269	275	281	286	292	298	303	
764	309	315	320	326	332	337	343	349	355	360	
765	366	372	377	383	389	394	400	406	411	417	
766	423	428	434	440	445	451	457	462	468	474	
767	479	485	491	496	502	508	513	519	525	530	
768	536	542	547	553	558	564	570	575	581	587	
769	592	598	604	609	615	621	626	632	638	643	
770	649	654	660	666	671	677	683	688	694	700	
771	705	711	716	722	728	733	739	745	750	756	
772	761	767	773	778	784	790	795	801	806	812	
773	818	823	829	835	840	846	851	857	863	868	
774	874	879	885	891	896	902	907	913	919	924	
775	930	936	941	947	952	958	964	969	975	980	
776	986	992	997	*003	*008	*014	*019	*025	*031	*036	
777	89 042	047	053	059	064	070	075	081	087	092	.1 5 .2 0.5 .3 1.1 .4 1.6 .5 2.2 .6 2.7 .7 3.3 .8 3.8 .9 4.4
778	098	103	109	114	120	126	131	137	142	148	.9 4.9
779	153	159	165	170	176	181	187	193	198	204	
780	209	215	220	226	231	237	243	248	254	259	
781	265	270	276	282	287	293	298	304	309	315	
782	320	326	332	337	343	348	354	359	365	370	
783	376	381	387	393	398	404	409	415	420	426	
784	431	437	442	448	454	459	465	470	476	481	
785	487	492	498	503	509	514	520	525	531	536	
786	542	548	553	559	564	570	575	581	586	592	
787	597	603	608	614	619	625	630	636	641	647	
788	652	658	663	669	674	680	685	691	696	702	
789	707	713	718	724	729	735	740	746	751	757	.1 5 .2 0.5 .3 1.0 .4 1.5 .5 2.0 .6 2.5 .7 3.0 .8 3.5 .9 4.0
790	762	768	773	779	784	790	795	801	806	812	.9 4.5
791	817	823	828	834	839	845	850	856	861	867	
792	872	878	883	889	894	900	905	911	916	922	
793	927	933	938	943	949	954	960	965	971	976	
794	982	987	993	998	*004	*009	*015	*020	*026	*031	
795	90 036	042	047	053	058	064	069	075	080	086	
796	091	097	102	107	113	118	124	129	135	140	
797	146	151	156	162	167	173	178	184	189	195	
798	200	205	211	216	222	227	233	238	244	249	
799	254	260	265	271	276	282	287	292	298	303	
800	309	314	320	325	330	336	341	347	352	358	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
800	90 309	314	320	325	330	336	341	347	352	358	
801	363	368	374	379	385	390	396	401	406	412	
802	417	423	428	433	439	444	450	455	460	466	
803	471	477	482	488	493	498	504	509	515	520	
804	525	531	536	542	547	552	558	563	569	574	
805	579	585	590	596	601	606	612	617	622	628	
806	633	639	644	649	655	660	666	671	676	682	
807	687	692	698	703	709	714	719	725	730	736	
808	741	746	752	757	762	768	773	778	784	789	
809	795	800	805	811	816	821	827	832	838	843	
810	848	854	859	864	870	875	880	886	891	896	
811	902	907	913	918	923	929	934	939	945	950	
812	955	961	966	971	977	982	987	993	998	*003	
813	91	009	014	019	025	030	036	041	046	052	
814	062	068	073	078	084	089	094	100	105	110	
815	116	121	126	131	137	142	147	153	158	163	
816	169	174	179	185	190	195	201	206	211	217	
817	222	227	233	238	243	249	254	259	264	270	
818	275	280	286	291	296	302	307	312	318	323	
819	328	333	339	344	349	355	360	365	371	376	
820	381	386	392	397	402	408	413	418	423	429	
821	434	439	445	450	455	461	466	471	476	482	
822	487	492	497	503	508	513	519	524	529	534	
823	540	545	550	556	561	566	571	577	582	587	
824	592	598	603	608	614	619	624	629	635	640	
825	645	650	656	661	666	671	677	682	687	692	
826	698	703	708	714	719	724	729	735	740	745	
827	750	756	761	766	771	777	782	787	792	798	
828	803	808	813	819	824	829	834	839	845	850	
829	855	860	866	871	876	881	887	892	897	902	
830	908	913	918	923	928	934	939	944	949	955	
831	960	965	970	976	981	986	991	996	*002	*007	
832	92	012	017	023	028	033	038	043	049	054	
833	064	069	075	080	085	090	096	101	106	111	
834	116	122	127	132	137	142	148	153	158	163	
835	168	174	179	184	189	194	200	205	210	215	
836	220	226	231	236	241	246	252	257	262	267	
837	272	277	283	288	293	298	303	309	314	319	
838	324	329	335	340	345	350	355	360	366	371	
839	376	381	386	391	397	402	407	412	417	423	
840	428	433	438	443	448	454	459	464	469	474	
841	479	485	490	495	500	505	510	515	521	526	
842	531	536	541	546	552	557	562	567	572	577	
843	583	588	593	598	603	608	613	619	624	629	
844	634	639	644	649	655	660	665	670	675	680	
845	685	691	696	701	706	711	716	721	727	732	
846	737	742	747	752	757	762	768	773	778	783	
847	788	793	798	803	809	814	819	824	829	834	
848	839	844	850	855	860	865	870	875	880	885	
849	891	896	901	906	911	916	921	926	931	937	
850	942	947	952	957	962	967	972	977	982	988	
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TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
850	92 942	947	952	957	962	967	972	977	982	988	
851	993	998	*003	*008	*013	*018	*023	*028	*034	*039	
852	93 044	049	054	059	064	069	074	079	084	090	
853	095	100	105	110	115	120	125	130	135	140	
854	146	151	156	161	166	171	176	181	186	191	
855	196	201	207	212	217	222	227	232	237	242	
856	247	252	257	262	267	272	278	283	288	293	
857	298	303	308	313	318	323	328	333	338	343	
858	348	354	359	364	369	374	379	384	389	394	
859	399	404	409	414	419	424	429	434	439	445	
860	450	455	460	465	470	475	480	485	490	495	
861	500	505	510	515	520	525	530	535	540	545	
862	550	556	561	566	571	576	581	586	591	596	
863	601	606	611	616	621	626	631	636	641	646	
864	651	656	661	666	671	676	681	686	691	696	
865	701	706	711	716	721	726	731	736	742	747	
866	752	757	762	767	772	777	782	787	792	797	
867	802	807	812	817	822	827	832	837	842	847	
868	852	857	862	867	872	877	882	887	892	897	
869	902	907	912	917	922	927	932	937	942	947	
870	952	957	962	967	972	977	982	987	992	997	
871	94 002	007	012	017	022	026	031	036	041	046	
872	051	056	061	066	071	076	081	086	091	096	
873	101	106	111	116	121	126	131	136	141	146	
874	151	156	161	166	171	176	181	186	191	196	
875	201	206	210	215	220	225	230	235	240	245	
876	250	255	260	265	270	275	280	285	290	295	
877	300	305	310	315	320	324	329	334	339	344	
878	349	354	359	364	369	374	379	384	389	394	
879	399	404	409	413	418	423	428	433	438	443	
880	448	453	458	463	468	473	478	483	487	492	
881	497	502	507	512	517	522	527	532	537	542	
882	547	552	556	561	566	571	576	581	586	591	
883	596	601	606	611	615	620	625	630	635	640	
884	645	650	655	660	665	670	674	679	684	689	
885	694	699	704	709	714	719	724	728	733	738	
886	743	748	753	758	763	768	773	777	782	787	
887	792	797	802	807	812	817	821	826	831	836	
888	841	846	851	856	861	865	870	875	880	885	
889	890	895	900	905	909	914	919	924	929	934	
890	939	944	949	953	958	963	968	973	978	983	
891	988	992	997	*002	*007	*012	*017	*022	*026	031	
892	95 036	041	046	051	056	061	065	070	075	080	
893	085	090	095	099	104	109	114	119	124	129	
894	134	138	143	148	153	158	163	167	172	177	
895	182	187	192	197	201	206	211	216	221	226	
896	231	235	240	245	250	255	260	264	269	274	
897	279	284	289	294	298	303	308	313	318	323	
898	327	332	337	342	347	352	356	361	366	371	
899	376	381	385	390	395	400	405	410	414	419	
900	424	429	434	438	443	448	453	458	463	467	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

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TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
900	95 424	429	434	438	443	448	453	458	463	467	
901	472	477	482	487	492	496	501	506	511	516	
902	520	525	530	535	540	544	549	554	559	564	
903	569	573	578	583	588	593	597	602	607	612	
904	617	621	626	631	636	641	645	650	655	660	
905	665	669	674	679	684	689	693	698	703	708	
906	713	717	722	727	732	737	741	746	751	756	
907	760	765	770	775	780	784	789	794	799	804	
908	808	813	818	823	827	832	837	842	847	851	
909	856	861	866	870	875	880	885	890	894	899	
910	904	909	913	918	923	928	933	937	942	947	
911	952	956	961	966	971	975	980	985	990	994	
912	999	*004	*009	*014	*018	*023	*028	*033	*037	*042	.1 0.5
913	047	052	056	061	066	071	075	080	085	090	.2 1.0
914	094	099	104	109	113	118	123	128	132	137	.3 1.5
915	142	147	151	156	161	166	170	175	180	185	.4 2.0
916	189	194	199	204	208	213	218	222	227	232	.5 2.5
917	237	241	246	251	256	260	265	270	275	279	.6 3.0
918	284	289	293	298	303	308	312	317	322	327	.7 3.5
919	331	336	341	345	350	355	360	364	369	374	.8 4.0
920	379	383	388	393	397	402	407	412	416	421	.9 4.5
921	426	430	435	440	445	449	454	459	463	468	
922	473	478	482	487	492	496	501	506	511	515	
923	520	525	529	534	539	543	548	553	558	562	
924	567	572	576	581	586	590	595	600	605	609	
925	614	619	623	628	633	637	642	647	651	656	
926	661	666	670	675	680	684	689	694	698	703	
927	708	712	717	722	726	731	736	741	745	750	
928	755	759	764	769	773	778	783	787	792	797	
929	801	806	811	815	820	825	829	834	839	843	
930	848	853	857	862	867	871	876	881	885	890	
931	895	899	904	909	913	918	923	927	932	937	
932	941	946	951	955	960	965	969	974	979	983	.1 0.4
933	988	993	997	*002	*007	*011	*016	*020	*025	*030	.2 0.9
934	034	039	044	048	053	058	062	067	072	076	.3 1.3
935	081	086	090	095	099	104	109	113	118	123	.4 1.8
936	127	132	137	141	146	151	155	160	164	169	.5 2.2
937	174	178	183	188	192	197	202	206	211	215	.6 2.7
938	220	225	229	234	239	243	248	252	257	262	.7 3.1
939	266	271	276	280	285	289	294	299	303	308	.8 3.6
940	313	317	322	326	331	336	340	345	349	354	.9 4.0
941	359	363	368	373	377	382	386	391	396	400	
942	405	409	414	419	423	428	432	437	442	446	
943	451	456	460	465	469	474	479	483	488	492	
944	497	502	506	511	515	520	525	529	534	538	
945	543	548	552	557	561	566	570	575	580	584	
946	589	593	598	603	607	612	616	621	626	630	
947	635	639	644	649	653	658	662	667	671	676	
948	681	685	690	694	699	703	708	713	717	722	
949	726	731	736	740	745	749	754	758	763	768	
950	772	777	781	786	790	795	800	804	809	813	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
950	97 772	777	781	786	790	795	800	804	809	813	
951	818	822	827	831	836	841	845	850	854	859	
952	863	868	873	877	882	886	891	895	900	904	
953	909	914	918	923	927	932	936	941	945	950	
954	955	959	964	968	973	977	982	986	991	996	
955	98 000	005	009	014	018	023	027	032	036	041	
956	046	050	055	059	064	068	073	077	082	086	
957	091	095	100	105	109	114	118	123	127	132	
958	136	141	145	150	154	159	163	168	173	177	.1 0.5
959	182	186	191	195	200	204	209	213	218	222	.2 1.0
											.3 1.5
960	227	231	236	240	245	249	254	259	263	268	.4 2.0
											.5 2.5
961	272	277	281	286	290	295	299	304	308	313	.6 3.0
962	317	322	326	331	335	340	344	349	353	358	.7 3.5
963	362	367	371	376	380	385	389	394	398	403	.8 4.0
964	407	412	416	421	425	430	434	439	443	448	.9 4.5
965	452	457	461	466	470	475	479	484	488	493	
966	497	502	506	511	515	520	524	529	533	538	
967	542	547	551	556	560	565	569	574	578	583	
968	587	592	596	601	605	610	614	619	623	628	
969	632	637	641	646	650	655	659	663	668	672	
970	677	681	686	690	695	699	704	708	713	717	
971	722	726	731	735	740	744	749	753	757	762	.1 4
972	766	771	775	780	784	789	793	798	802	807	.2 0.4
973	811	815	820	824	829	833	838	842	847	851	.3 0.9
974	856	860	865	869	873	878	882	887	891	896	.4 1.3
975	900	905	909	914	918	922	927	931	936	940	.5 1.8
976	945	949	954	958	963	967	971	976	980	985	.6 2.2
977	989	994	998	*003	*007	*011	*016	*020	*025	*029	.7 2.7
978	99 034	038	043	047	051	056	060	065	069	074	.8 3.1
979	078	082	087	091	096	100	105	109	113	118	.9 3.6
980	122	127	131	136	140	145	149	153	158	162	.9 4.0
981	167	171	176	180	184	189	193	198	202	206	
982	211	215	220	224	229	233	237	242	246	251	
983	255	260	264	268	273	277	282	286	290	295	
984	299	304	308	312	317	321	326	330	335	339	
985	343	348	352	357	361	365	370	374	379	383	
986	387	392	396	401	405	409	414	418	423	427	
987	431	436	440	445	449	453	458	462	467	471	
988	475	480	484	489	493	497	502	506	511	515	.1 4
989	519	524	528	533	537	541	546	550	554	559	.2 0.8
											.3 1.2
990	563	568	572	576	581	585	590	594	598	603	.4 1.6
											.5 2.0
991	607	611	616	620	625	629	633	638	642	647	.6 2.4
992	651	655	660	664	668	673	677	682	686	690	.7 2.8
993	695	699	703	708	712	717	721	725	730	734	.8 3.2
994	738	743	747	751	756	760	765	769	773	778	.9 3.6
995	782	786	791	795	800	804	808	813	817	821	
996	826	830	834	839	843	847	852	856	861	865	
997	869	874	878	882	887	891	895	900	904	908	
998	913	917	922	926	930	935	939	943	948	952	
999	956	961	965	969	974	978	982	987	991	995	
1000	00 000	004	008	013	017	021	026	030	034	039	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
1000	000 000	043	087	130	173	217	260	304	347	390	
01	434	477	521	564	607	651	694	737	781	824	
02	867	911	954	997	*041	*084	*127	*171	*214	*257	
03	001 301	344	387	431	474	517	560	604	647	690	
04	733	777	820	863	906	950	993	*036	*079	*123	
05	002 166	209	252	295	339	382	425	468	511	555	
06	598	641	684	727	770	814	857	900	943	986	
07	003 029	072	115	159	202	245	288	331	374	417	
08	460	503	546	590	633	676	719	762	805	848	
09	891	934	977	*020	*063	*106	*149	*192	*235	*278	
1010	004 321	364	407	450	493	536	579	622	665	708	
11	751	794	837	880	923	966	*009	*051	*094	*137	
12	005 180	223	266	309	352	395	438	481	523	566	
13	609	652	695	738	781	824	866	909	952	995	
14	006 038	081	123	166	209	252	295	337	380	423	
15	466	509	551	594	637	680	722	765	808	851	
16	893	936	979	*022	*064	*107	*150	*193	*235	*278	
17	007 321	363	406	449	491	534	577	620	662	705	
18	748	790	833	875	918	961	*003	*046	*089	131	
19	008 174	217	259	302	344	387	430	472	515	557	
1020	600	642	685	728	770	813	855	898	940	983	
21	009 025	068	111	153	196	238	281	323	366	408	
22	451	493	536	578	621	663	706	748	790	833	
23	875	918	960	*003	*045	*088	*130	*172	*215	*257	
24	010 300	342	385	427	469	512	554	596	639	681	
25	724	766	808	851	893	935	978	*020	*062	*105	
26	011 147	189	232	274	316	359	401	443	486	528	
27	570	612	655	697	739	782	824	866	908	951	
28	993	*035	*077	*120	*162	*204	*246	*288	*331	*373	
29	012 415	457	500	542	584	626	668	710	753	795	
1030	837	879	921	963	*006	*048	*090	*132	174	216	
31	013 258	301	343	385	427	469	511	553	595	637	
32	679	722	764	806	848	890	932	974	*016	*058	
33	014 100	142	184	226	268	310	352	394	436	478	
34	520	562	604	646	688	730	772	814	856	898	
35	940	982	*024	*066	*108	*150	*192	*234	*276	*318	
36	015 360	401	443	485	527	569	611	653	695	737	
37	779	820	862	904	946	988	*030	*072	*113	155	
38	016 197	239	281	323	364	406	448	490	532	573	
39	615	657	699	741	782	824	866	908	950	991	
1040	017 033	075	117	158	200	242	284	325	367	409	
41	450	492	534	576	617	659	701	742	784	826	
42	867	909	951	992	*034	*076	*117	*159	*201	*242	
43	018 284	326	367	409	451	492	534	575	617	659	
44	700	742	783	825	867	908	950	991	*033	*074	
45	019 116	158	199	241	282	324	365	407	448	490	
46	531	573	614	656	697	739	780	822	863	905	
47	946	988	*029	*071	*112	*154	*195	*237	*278	*320	
48	020 361	402	444	485	527	568	610	651	692	734	
49	775	817	858	899	941	982	*024	*065	*106	*148	
1050	021 189	230	272	313	354	396	437	478	520	561	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

43 43
 .1 4.3 4.3
 .2 8.7 8.6
 .3 13.0 12.9
 .4 17.4 17.2
 .5 21.7 21.5
 .6 26.1 25.8
 .7 30.4 30.1
 .8 34.8 34.4
 .9 39.1 38.7

42 42
 .1 4.2 4.2
 .2 8.5 8.4
 .3 12.7 12.6
 .4 17.0 16.8
 .5 21.2 21.0
 .6 25.5 25.2
 .7 29.7 29.4
 .8 34.0 33.6
 .9 38.2 37.8

41 41
 .1 4.1 4.1
 .2 8.3 8.2
 .3 12.4 12.3
 .4 16.6 16.4
 .5 20.7 20.5
 .6 24.9 24.6
 .7 29.0 28.7
 .8 33.2 32.8
 .9 37.3 36.9

TABLE V.—LOGARITHMS OF NUMBERS.

N.	0	1	2	3	4	5	6	7	8	9	P. P.
1050	021 189	230	272	313	354	396	437	478	520	561	
51	602	644	685	726	768	809	850	892	933	974	.1 4.1
52	022 015	057	098	139	181	222	263	304	346	387	.2 8.3
53	428	469	511	552	593	634	676	717	758	799	.3 12.4
54	840	882	923	964	*005	*046	*088	*129	*170	*211	.4 16.6
55	023 252	293	335	376	417	458	499	540	581	623	.5 20.7
56	664	705	746	787	828	869	910	951	993	*034	.6 24.9
57	024 075	116	157	198	239	280	321	362	403	444	.7 29.0
58	485	526	568	609	650	691	732	773	814	855	.8 33.2
59	896	937	978	*019	*060	*101	*142	*183	*224	*265	.9 37.3
1060	025 306	347	388	429	469	510	551	592	633	674	
61	715	756	797	838	879	920	961	*002	*042	*083	.1 4.1
62	026 124	165	206	247	288	329	370	410	451	492	.2 8.2
63	533	574	615	656	696	737	778	819	860	901	.3 12.3
64	941	982	*023	*064	*105	*145	*186	*227	*268	*309	.4 16.4
65	027 349	390	431	472	512	553	594	635	675	716	.5 20.5
66	757	798	838	879	920	961	*001	*042	*083	*123	.6 24.6
67	028 164	205	246	286	327	368	408	449	490	530	.7 28.7
68	571	612	652	693	734	774	815	856	896	937	.8 32.8
69	977	*018	*059	*099	*140	*181	*221	*262	*302	*343	.9 36.9
1070	029 384	424	465	505	546	586	627	668	708	749	
71	789	830	870	911	951	992	*032	*073	*114	*154	.1 4.0
72	030 195	235	276	316	357	397	438	478	519	559	.2 8.1
73	599	640	680	721	761	802	842	883	923	964	.3 12.1
74	031 004	044	085	125	166	206	247	287	327	368	.4 16.2
75	408	449	489	529	570	610	651	691	731	772	.5 20.2
76	812	852	893	933	973	*014	*054	*094	*135	*175	.6 24.3
77	032 215	256	296	336	377	417	457	498	538	578	.7 28.3
78	619	659	699	739	780	820	860	900	941	981	.8 32.4
79	033 021	061	102	142	182	222	263	303	343	383	.9 36.4
1080	424	464	504	544	584	625	665	705	745	785	
81	825	866	906	946	986	*026	*066	*107	147	187	.1 4.0
82	034 227	267	307	347	388	428	468	508	548	588	.2 8.0
83	628	668	708	748	789	829	869	909	949	989	.3 12.0
84	035 029	069	109	149	189	229	269	309	349	389	.4 16.0
85	429	470	510	550	590	630	670	710	750	790	.5 20.0
86	830	870	910	950	990	*029	*069	*109	*149	*189	.6 24.0
87	036 229	269	309	349	389	429	469	509	549	589	.7 28.0
88	629	669	708	748	788	828	868	908	948	988	.8 32.0
89	037 028	068	107	147	187	227	267	307	347	386	.9 36.0
1090	426	466	506	546	586	625	665	705	745	785	
91	825	864	904	944	984	*023	*063	*103	143	183	.1 3.9
92	038 222	262	302	342	381	421	461	501	540	580	.2 7.9
93	620	660	699	739	779	819	858	898	938	977	.3 11.8
94	039 017	057	096	136	176	216	255	295	335	374	.4 15.8
95	414	454	493	533	572	612	652	691	731	771	.5 19.7
96	810	850	890	929	969	*008	*048	*088	*127	*167	.6 23.7
97	040 206	246	286	325	365	404	444	483	523	563	.7 27.6
98	602	642	681	721	760	800	839	879	918	958	.8 31.6
99	997	*037	*076	*116	*155	*195	*234	*274	*313	*353	.9 35.5
1100	041 392	432	471	511	550	590	629	669	708	748	
N.	0	1	2	3	4	5	6	7	8	9	P. P.

TABLE VI.—LOGARITHMIC SINES AND TANGENTS OF SMALL ANGLES.

$$\begin{aligned}\log \sin \phi &= \log \phi'' + S. \\ \log \tan \phi &= \log \phi'' + T.\end{aligned}$$

0°

$$\begin{aligned}\log \phi'' &= \log \sin \phi + S'. \\ \log \phi'' &= \log \tan \phi + T'.\end{aligned}$$

"	'	S	T	Log. Sin.	S'	T'	Log. Tan.
0	0	4.685 57	57	— ∞	5.314 42	42	— ∞
60	1	57	57	6.46 372	42	42	6.46 372
120	2	57	57	.76 475	42	42	.76 475
180	3	57	57	.94 084	42	42	.94 084
240	4	57	57	7.06 578	42	42	7.06 578
300	5	4.685 57	57	7.16 269	5.314 42	42	7.16 269
360	6	57	57	.24 187	42	42	.24 188
420	7	57	57	.30 882	42	42	.30 882
480	8	57	57	.36 681	42	42	.36 681
540	9	57	57	.41 797	42	42	.41 797
600	10	4.685 57	57	7.46 372	5.314 42	42	7.46 372
660	11	57	57	.50 512	42	42	.50 512
720	12	57	57	.54 290	42	42	.54 291
780	13	57	57	.57 767	42	42	.57 767
840	14	57	57	.60 985	42	42	.60 985
900	15	4.685 57	58	7.63 981	5.314 42	42	7.63 982
960	16	57	58	.66 784	42	42	.66 785
1020	17	57	58	.69 417	42	42	.69 418
1080	18	57	58	.71 899	42	42	.71 900
1140	19	57	58	.74 248	42	42	.74 248
1200	20	4.685 57	58	7.76 475	5.314 43	42	7.76 476
1260	21	57	58	.78 594	43	42	.78 595
1320	22	57	58	.80 614	43	42	.80 615
1380	23	57	58	.82 545	43	42	.82 546
1440	24	57	58	.84 393	43	42	.84 394
1500	25	4.685 57	58	7.86 166	5.314 43	41	7.86 167
1560	26	57	58	.87 869	43	41	.87 871
1620	27	57	58	.89 508	43	41	.89 510
1680	28	57	58	.91 088	43	41	.91 089
1740	29	57	58	.92 612	43	41	.92 613
1800	30	4.685 57	58	7.94 084	5.314 43	41	7.94 086
1860	31	57	58	.95 508	43	41	.95 510
1920	32	57	58	.96 887	43	41	.96 889
1980	33	57	59	.98 223	43	41	.98 225
2040	34	57	59	.99 520	43	41	.99 522
2100	35	4.685 56	59	8.00 778	5.314 43	41	8.00 781
2160	36	56	59	.02 002	43	41	.02 004
2220	37	56	59	.03 192	43	41	.03 194
2280	38	56	59	.04 350	43	40	.04 352
2340	39	56	59	.05 478	43	40	.05 481
2400	40	4.685 56	59	8.06 577	5.314 43	40	8.06 580
2460	41	56	59	.07 650	43	40	.07 653
2520	42	56	59	.08 696	43	40	.08 699
2580	43	56	60	.09 718	43	40	.09 721
2640	44	56	60	.10 716	43	40	.10 720
2700	45	4.685 56	60	8.11 692	5.314 44	40	8.11 696
2760	46	56	60	.12 647	44	40	.12 651
2820	47	56	60	.13 581	44	40	.13 585
2880	48	56	60	.14 495	44	39	.14 499
2940	49	56	60	.15 390	44	39	.15 395
3000	50	4.685 56	60	8.16 268	5.314 44	39	8.16 272
3060	51	56	60	.17 128	44	39	.17 133
3120	52	56	61	.17 971	44	39	.17 976
3180	53	56	61	.18 798	44	39	.18 803
3240	54	55	61	.19 610	44	39	.19 615
3300	55	4.685 55	61	8.20 407	5.314 44	39	8.20 412
3360	56	55	61	.21 189	44	38	.21 195
3420	57	55	61	.21 958	44	38	.21 964
3480	58	55	61	.22 713	44	38	.22 719
3540	59	55	62	.23 455	44	38	.23 462

TABLE VI.—LOGARITHMIC SINES AND TANGENTS OF SMALL ANGLES.

$$\begin{aligned}\log \sin \phi &= \log \phi'' + S. \\ \log \tan \phi &= \log \phi'' + T.\end{aligned}$$

1°

$$\begin{aligned}\log \phi'' &= \log \sin \phi + S'. \\ \log \phi'' &= \log \tan \phi + T'.\end{aligned}$$

"	'	S	T	Log. Sin.	S'	T'	Log. Tan.
3600	0	4.685 55	62	8.24 185	5.314 44	38	8.24 192
3660	1	55	62	.24 903	45	38	.24 910
3720	2	55	62	.25 609	45	38	.25 616
3780	3	55	62	.26 304	45	37	.26 311
3840	4	55	62	.26 988	45	37	.26 995
3900	5	4.685 55	62	8.27 661	5.314 45	37	8.27 669
3960	6	55	63	.28 324	45	37	.28 332
4020	7	54	63	.28 977	45	37	.28 985
4080	8	54	63	.29 620	45	37	.29 629
4140	9	54	63	.30 254	45	36	.30 263
4200	10	4.685 54	63	8.30 879	5.314 45	36	8.30 888
4260	11	54	63	.31 495	45	36	.31 504
4320	12	54	64	.32 102	45	36	.32 112
4380	13	54	64	.32 701	46	36	.32 711
4440	14	54	64	.33 292	46	36	.33 302
4500	15	4.685 54	64	8.33 875	5.314 46	35	8.33 885
4560	16	54	64	.34 450	46	35	.34 461
4620	17	54	65	.35 018	46	35	.35 029
4680	18	54	65	.35 578	46	35	.35 589
4740	19	53	65	.36 131	46	35	.36 143
4800	20	4.685 53	65	8.36 677	5.314 46	34	8.36 689
4860	21	53	65	.37 217	46	34	.37 229
4920	22	53	65	.37 750	46	34	.37 762
4980	23	53	66	.38 276	46	34	.38 289
5040	24	53	66	.38 796	47	34	.38 809
5100	25	4.685 53	66	8.39 310	5.314 47	33	8.39 323
5160	26	53	66	.39 818	47	33	.39 831
5220	27	53	67	.40 320	47	33	.40 334
5280	28	52	67	.40 816	47	33	.40 830
5340	29	52	67	.41 307	47	33	.41 321
5400	30	4.685 52	67	8.41 792	5.314 47	32	8.41 807
5460	31	52	67	.42 271	47	32	.42 287
5520	32	52	68	.42 746	47	32	.42 762
5580	33	52	68	.43 215	48	32	.43 231
5640	34	52	68	.43 680	48	31	.43 696
5700	35	4.685 52	68	8.44 139	5.314 48	31	8.44 156
5760	36	52	69	.44 594	48	31	.44 611
5820	37	51	69	.45 044	48	31	.45 061
5880	38	51	69	.45 489	48	30	.45 507
5940	39	51	69	.45 930	48	30	.45 948
6000	40	4.685 51	69	8.46 366	5.314 48	30	8.46 385
6060	41	51	70	.46 798	49	30	.46 817
6120	42	51	70	.47 226	49	30	.47 245
6180	43	51	70	.47 650	49	29	.47 669
6240	44	51	70	.48 069	49	29	.48 089
6300	45	4.685 50	71	8.48 485	5.314 49	29	8.48 505
6360	46	50	71	.48 896	49	28	.48 917
6420	47	50	71	.49 304	49	28	.49 325
6480	48	50	72	.49 708	49	28	.49 729
6540	49	50	72	.50 108	50	28	.50 130
6600	50	4.685 50	72	8.50 504	5.314 50	27	8.50 526
6660	51	50	72	.50 897	50	27	.50 920
6720	52	50	73	.51 286	50	27	.51 310
6780	53	49	73	.51 672	50	27	.51 696
6840	54	49	73	.52 055	50	26	.52 079
6900	55	4.685 49	73	8.52 434	5.314 50	26	8.52 458
6960	56	49	74	.52 810	51	26	.52 835
7020	57	49	74	.53 183	51	25	.53 208
7080	58	49	74	.53 552	51	25	.53 578
7140	59	49	75	.53 918	51	25	.53 944

TABLE VI.—LOGARITHMIC SINES AND TANGENTS OF SMALL ANGLES

$\log \sin \phi = \log \phi'' + S.$
 $\log \tan \phi = \log \phi'' + T.$

2°

$\log \phi'' = \log \sin \phi + S'.$
 $\log \phi'' = \log \tan \phi + T'.$

"	'	S	T	Log. Sin.	S'	T'	Log. Tan.
7200	0	4.685 48	75	8.54 282	5.314 51	25	8.54 308
7260	1	48	75	.54 642	51	24	.54 669
7320	2	48	75	.54 999	51	24	.55 027
7380	3	48	76	.55 354	52	24	.55 381
7440	4	48	76	.55 705	52	23	.55 733
7500	5	4.685 48	76	8.56 054	5.314 52	23	8.56 083
7560	6	48	77	.56 400	52	23	.56 429
7620	7	47	77	.56 743	52	22	.56 772
7680	8	47	77	.57 083	52	22	.57 113
7740	9	47	78	.57 421	52	22	.57 452
7800	10	4.685 47	78	8.57 756	5.314 53	22	8.57 787
7860	11	47	78	.58 089	53	21	.58 121
7920	12	47	79	.58 419	53	21	.58 451
7980	13	46	79	.58 747	53	21	.58 779
8040	14	46	79	.59 072	53	20	.59 105
8100	15	4.685 46	80	8.59 395	5.314 53	20	8.59 428
8160	16	46	80	.59 715	54	20	.59 749
8220	17	46	80	.60 033	54	19	.60 067
8280	18	46	81	.60 349	54	19	.60 384
8340	19	45	81	.60 662	54	19	.60 698
8400	20	4.685 45	81	8.60 973	5.314 54	18	8.61 009
8460	21	45	82	.61 282	54	18	.61 319
8520	22	45	82	.61 589	55	18	.61 626
8580	23	45	82	.61 893	55	17	.61 931
8640	24	45	83	.62 196	55	17	.62 234
8700	25	4.685 44	83	8.62 496	5.314 55	16	8.62 535
8760	26	44	83	.62 795	55	16	.62 834
8820	27	44	84	.63 091	55	16	.63 131
8880	28	44	84	.63 385	56	15	.63 425
8940	29	44	84	.63 677	56	15	.63 718
9000	30	4.685 43	85	8.63 968	5.314 56	15	8.64 009
9060	31	43	85	.64 256	56	14	.64 298
9120	32	43	86	.64 543	56	14	.64 585
9180	33	43	86	.64 827	57	14	.64 870
9240	34	43	86	.65 110	57	13	.65 153
9300	35	4.685 43	87	8.65 391	5.314 57	13	8.65 435
9360	36	42	87	.65 670	57	12	.65 715
9420	37	42	87	.65 947	57	12	.65 993
9480	38	42	88	.66 223	58	12	.66 269
9540	39	42	88	.66 497	58	11	.66 543
9600	40	4.685 42	89	8.66 769	5.314 58	11	8.66 816
9660	41	41	89	.67 039	58	10	.67 087
9720	42	41	89	.67 308	58	10	.67 356
9780	43	41	90	.67 575	59	10	.67 624
9840	44	41	90	.67 840	59	09	.67 890
9900	45	4.685 41	91	8.68 104	5.314 59	09	8.68 154
9960	46	40	91	.68 366	59	08	.68 417
10020	47	40	91	.68 627	59	08	.68 678
10080	48	40	92	.68 886	60	08	.68 938
10140	49	40	92	.69 144	60	07	.69 196
10200	50	4.685 40	93	8.69 400	5.314 60	07	8.69 453
10260	51	39	93	.69 654	60	06	.69 708
10320	52	39	93	.69 907	60	06	.69 961
10380	53	39	94	.70 159	61	06	.70 214
10440	54	39	94	.70 409	61	05	.70 464
10500	55	4.685 38	95	8.70 657	5.314 61	05	8.70 714
10560	56	38	95	.70 905	61	04	.70 962
10620	57	38	96	.71 150	61	04	.71 208
10680	58	38	96	.71 395	62	03	.71 453
10740	59	38	97	.71 638	62	03	.71 697

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

0°

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'	Log. Sin.	D	Log. Tan.	Com. D.	Log. Cot.	Log. Cos.	
0	— ∞		— ∞		+ ∞	0.00 000	60
1	6.46 372	30103	6.46 372	30103	3.53 627	0.00 000	59
2	6.76 475	17609	6.76 475	17609	3.23 524	0.00 000	58
3	6.94 084	12494	6.94 084	12494	3.05 915	0.00 000	57
4	7.06 578		7.06 578		2.93 421	0.00 000	56
5	7.16 269	9691	7.16 269	9691	2.83 730	0.00 000	55
6	7.24 187	7918	7.24 188	7918	2.75 812	0.00 000	54
7	7.30 882	6695	7.30 882	6694	2.69 117	0.00 000	53
8	7.36 681	5799	7.36 681	5799	2.63 318	0.00 000	52
9	7.41 797	5115	7.41 797	5115	2.58 203	0.00 000	51
10	7.46 372	4575	7.46 372	4575	2.53 627	0.00 000	50
11	7.50 512	4139	7.50 512	4139	2.49 488	0.00 000	49
12	7.54 290	3778	7.54 291	3779	2.45 709	9.99 999	48
13	7.57 767	3476	7.57 767	3476	2.42 233	9.99 999	47
14	7.60 985	3218	7.60 985	3218	2.39 014	9.99 999	46
15	7.63 981	2996	7.63 982	2996	2.36 018	9.99 999	45
16	7.66 784	2803	7.66 785	2803	2.33 215	9.99 999	44
17	7.69 417	2633	7.69 418	2633	2.30 582	9.99 999	43
18	7.71 899	2482	7.71 900	2482	2.28 099	9.99 999	42
19	7.74 248	2348	7.74 248	2348	2.25 751	9.99 999	41
20	7.76 475	2227	7.76 476	2227	2.23 524	9.99 999	40
21	7.78 594	2119	7.78 595	2119	2.21 405	9.99 999	39
22	7.80 614	2020	7.80 615	2020	2.19 384	9.99 999	38
23	7.82 545	1930	7.82 546	1930	2.17 454	9.99 999	37
24	7.84 393	1848	7.84 394	1848	2.15 605	9.99 999	36
25	7.86 166	1772	7.86 167	1773	2.13 832	9.99 999	35
26	7.87 869	1703	7.87 871	1703	2.12 129	9.99 999	34
27	7.89 508	1639	7.89 510	1639	2.10 490	9.99 998	33
28	7.91 088	1579	7.91 089	1579	2.08 910	9.99 998	32
29	7.92 612	1524	7.92 613	1524	2.07 386	9.99 998	31
30	7.94 084	1472	7.94 086	1472	2.05 914	9.99 998	30
31	7.95 508	1424	7.95 510	1424	2.04 490	9.99 998	29
32	7.96 887	1379	7.96 889	1379	2.03 111	9.99 998	28
33	7.98 223	1336	7.98 225	1336	2.01 774	9.99 998	27
34	7.99 520	1296	7.99 522	1296	2.00 478	9.99 998	26
35	8.00 778	1258	8.00 781	1259	1.99 219	9.99 997	25
36	8.02 002	1223	8.02 004	1223	1.97 995	9.99 997	24
37	8.03 192	1190	8.03 194	1190	1.96 805	9.99 997	23
38	8.04 350	1158	8.04 352	1158	1.95 647	9.99 997	22
39	8.05 478	1128	8.05 481	1128	1.94 519	9.99 997	21
40	8.06 577	1099	8.06 580	1099	1.93 419	9.99 997	20
41	8.07 650	1072	8.07 653	1072	1.92 347	9.99 997	19
42	8.08 696	1046	8.08 699	1046	1.91 300	9.99 997	18
43	8.09 718	1022	8.09 721	1022	1.90 278	9.99 996	17
44	8.10 716	998	8.10 720	999	1.89 279	9.99 996	16
45	8.11 692	976	8.11 696	976	1.88 303	9.99 996	15
46	8.12 647	954	8.12 651	954	1.87 349	9.99 996	14
47	8.13 581	934	8.13 585	934	1.86 415	9.99 996	13
48	8.14 495	914	8.14 499	914	1.85 500	9.99 996	12
49	8.15 390	895	8.15 395	895	1.84 605	9.99 995	11
50	8.16 268	877	8.16 272	877	1.83 727	9.99 995	10
51	8.17 128	860	8.17 133	860	1.82 867	9.99 995	9
52	8.17 971	843	8.17 976	843	1.82 023	9.99 995	8
53	8.18 798	827	8.18 803	827	1.81 196	9.99 995	7
54	8.19 610	811	8.19 615	812	1.80 384	9.99 994	6
55	8.20 407	797	8.20 412	797	1.79 587	9.99 994	5
56	8.21 189	782	8.21 195	783	1.78 804	9.99 994	4
57	8.21 958	768	8.21 964	768	1.78 036	9.99 994	3
58	8.22 713	755	8.22 719	755	1.77 280	9.99 994	2
59	8.23 455	742	8.23 462	742	1.76 538	9.99 993	1
60	8.24 185	730	8.24 192	730	1.75 808	9.99 993	0
	Log. Cos.	D	Log. Cot.	Com. D.	Log. Tan.	Log. Sin.	'

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

1°

178°

'	Log. Sin.	D	Log. Tan.	Com. D.	Log. Cot.	Log. Cos.	
0	8.24 185	718	8.24 192	718	1.75 808	9.99 993	60
1	8.24 903	706	8.24 910	706	1.75 090	9.99 993	59
2	8.25 609	694	8.25 616	695	1.74 383	9.99 993	58
3	8.26 304	684	8.26 311	684	1.73 688	9.99 992	57
4	8.26 988	673	8.26 995	673	1.73 004	9.99 992	56
5	8.27 661	663	8.27 669	663	1.72 331	9.99 992	55
6	8.28 324	653	8.28 332	653	1.71 667	9.99 992	54
7	8.28 977	643	8.28 985	643	1.71 014	9.99 992	53
8	8.29 620	634	8.29 629	634	1.70 371	9.99 991	52
9	8.30 254	625	8.30 263	625	1.69 736	9.99 991	51
10	8.30 879	616	8.30 888	616	1.69 111	9.99 991	50
11	8.31 495	607	8.31 504	607	1.68 495	9.99 990	49
12	8.32 102	599	8.32 112	599	1.67 888	9.99 990	48
13	8.32 701	591	8.32 711	591	1.67 288	9.99 990	47
14	8.33 292	583	8.33 302	583	1.66 697	9.99 990	46
15	8.33 875	575	8.33 885	575	1.66 114	9.99 989	45
16	8.34 450	567	8.34 461	568	1.65 539	9.99 989	44
17	8.35 018	560	8.35 029	560	1.64 971	9.99 989	43
18	8.35 578	553	8.35 589	553	1.64 410	9.99 989	42
19	8.36 131	546	8.36 143	546	1.63 857	9.99 988	41
20	8.36 677	539	8.36 689	539	1.63 310	9.99 988	40
21	8.37 217	533	8.37 229	533	1.62 771	9.99 988	39
22	8.37 750	526	8.37 762	527	1.62 238	9.99 987	38
23	8.38 276	520	8.38 289	520	1.61 711	9.99 987	37
24	8.38 796	514	8.38 809	514	1.61 191	9.99 987	36
25	8.39 310	508	8.39 323	508	1.60 676	9.99 986	35
26	8.39 818	502	8.39 831	502	1.60 168	9.99 986	34
27	8.40 320	496	8.40 334	496	1.59 666	9.99 986	33
28	8.40 816	491	8.40 830	491	1.59 169	9.99 986	32
29	8.41 307	485	8.41 321	485	1.58 678	9.99 985	31
30	8.41 792	479	8.41 807	480	1.58 193	9.99 985	30
31	8.42 271	474	8.42 287	475	1.57 713	9.99 985	29
32	8.42 746	469	8.42 762	469	1.57 238	9.99 984	28
33	8.43 215	464	8.43 231	464	1.56 768	9.99 984	27
34	8.43 680	459	8.43 696	460	1.56 304	9.99 984	26
35	8.44 139	454	8.44 156	455	1.55 844	9.99 983	25
36	8.44 594	450	8.44 611	450	1.55 389	9.99 983	24
37	8.45 044	445	8.45 061	445	1.54 938	9.99 982	23
38	8.45 489	440	8.45 507	441	1.54 493	9.99 982	22
39	8.45 930	436	8.45 948	437	1.54 052	9.99 982	21
40	8.46 366	432	8.46 385	432	1.53 615	9.99 981	20
41	8.46 798	428	8.46 817	428	1.53 183	9.99 981	19
42	8.47 226	423	8.47 245	424	1.52 754	9.99 981	18
43	8.47 650	419	8.47 669	419	1.52 330	9.99 980	17
44	8.48 069	415	8.48 089	416	1.51 911	9.99 980	16
45	8.48 485	411	8.48 505	412	1.51 495	9.99 979	15
46	8.48 896	407	8.48 917	408	1.51 083	9.99 979	14
47	8.49 304	404	8.49 325	404	1.50 675	9.99 979	13
48	8.49 708	400	8.49 729	400	1.50 270	9.99 978	12
49	8.50 108	396	8.50 130	396	1.49 870	9.99 978	11
50	8.50 504	393	8.50 526	393	1.49 473	9.99 978	10
51	8.50 897	389	8.50 920	390	1.49 080	9.99 977	9
52	8.51 286	386	8.51 310	386	1.48 690	9.99 977	8
53	8.51 672	382	8.51 696	383	1.48 304	9.99 976	7
54	8.52 055	379	8.52 079	379	1.47 921	9.99 976	6
55	8.52 434	375	8.52 458	376	1.47 541	9.99 975	5
56	8.52 810	373	8.52 835	373	1.47 165	9.99 975	4
57	8.53 183	369	8.53 208	370	1.46 792	9.99 975	3
58	8.53 552	366	8.53 578	366	1.46 422	9.99 974	2
59	8.53 918	363	8.53 944	364	1.46 055	9.99 974	1
60	8.54 282		8.54 308		1.45 691	9.99 973	0
	Log. Cos.	D	Log. Cot.	Com. D.	Log. Tan.	Log. Sin.	

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

177°

	Log. Sin.	D	Log. Tan.	Com. D.	Log. Cot.	Log. Cos.	
0	8.54 282	360	8.54 308	360	1.45 691	9.99 973	60
1	8.54 642	357	8.54 669	358	1.45 831	9.99 973	59
2	8.54 999	354	8.55 027	354	1.44 973	9.99 972	58
3	8.55 354	351	8.55 381	352	1.44 618	9.99 972	57
4	8.55 705	348	8.55 733	349	1.44 265	9.99 971	56
5	8.56 054	346	8.56 083	346	1.43 917	9.99 971	55
6	8.56 400	343	8.56 429	343	1.43 571	9.99 971	54
7	8.56 743	340	8.56 772	341	1.43 227	9.99 970	53
8	8.57 083	338	8.57 113	338	1.42 886	9.99 970	52
9	8.57 421	335	8.57 452	335	1.42 548	9.99 969	51
10	8.57 756	332	8.57 787	333	1.42 212	9.99 969	50
11	8.58 090	330	8.58 121	330	1.41 879	9.99 968	49
12	8.58 419	327	8.58 451	328	1.41 548	9.99 968	48
13	8.58 747	325	8.58 779	325	1.41 220	9.99 967	47
14	8.59 072	323	8.59 105	323	1.40 895	9.99 967	46
15	8.59 395	320	8.59 428	320	1.40 571	9.99 966	45
16	8.59 715	318	8.59 749	318	1.40 251	9.99 966	44
17	8.60 033	316	8.60 067	316	1.39 932	9.99 965	43
18	8.60 349	313	8.60 384	314	1.39 616	9.99 965	42
19	8.60 662	311	8.60 698	311	1.39 302	9.99 964	41
20	8.60 973	309	8.61 009	309	1.38 990	9.99 964	40
21	8.61 282	306	8.61 319	307	1.38 681	9.99 963	39
22	8.61 589	304	8.61 626	305	1.38 374	9.99 963	38
23	8.61 893	302	8.61 931	303	1.38 068	9.99 962	37
24	8.62 196	300	8.62 234	300	1.37 765	9.99 962	36
25	8.62 496	298	8.62 535	299	1.37 465	9.99 961	35
26	8.62 795	296	8.62 834	297	1.37 166	9.99 961	34
27	8.63 091	294	8.63 131	294	1.36 869	9.99 960	33
28	8.63 385	292	8.63 425	293	1.36 574	9.99 959	32
29	8.63 677	290	8.63 718	291	1.36 281	9.99 959	31
30	8.63 968	288	8.64 009	288	1.35 990	9.99 958	30
31	8.64 256	286	8.64 298	287	1.35 702	9.99 958	29
32	8.64 543	284	8.64 585	285	1.35 414	9.99 957	28
33	8.64 827	282	8.64 870	283	1.35 129	9.99 957	27
34	8.65 110	281	8.65 153	281	1.34 846	9.99 956	26
35	8.65 391	279	8.65 435	280	1.34 565	9.99 956	25
36	8.65 670	277	8.65 715	278	1.34 285	9.99 955	24
37	8.65 947	275	8.65 993	276	1.34 007	9.99 954	23
38	8.66 223	274	8.66 269	274	1.33 731	9.99 954	22
39	8.66 497	272	8.66 543	272	1.33 456	9.99 953	21
40	8.66 769	270	8.66 816	271	1.33 184	9.99 953	20
41	8.67 039	268	8.67 087	269	1.32 913	9.99 952	19
42	8.67 308	267	8.67 356	267	1.32 643	9.99 952	18
43	8.67 575	265	8.67 624	266	1.32 376	9.99 951	17
44	8.67 840	264	8.67 890	264	1.32 110	9.99 950	16
45	8.68 104	262	8.68 154	262	1.31 845	9.99 950	15
46	8.68 366	260	8.68 417	261	1.31 583	9.99 949	14
47	8.68 627	259	8.68 678	259	1.31 321	9.99 948	13
48	8.68 886	257	8.68 938	258	1.31 062	9.99 948	12
49	8.69 144	256	8.69 196	256	1.30 803	9.99 947	11
50	8.69 400	254	8.69 453	255	1.30 547	9.99 947	10
51	8.69 654	253	8.69 708	253	1.30 292	9.99 946	9
52	8.69 907	251	8.69 961	252	1.30 038	9.99 945	8
53	8.70 159	250	8.70 214	250	1.29 786	9.99 945	7
54	8.70 409	248	8.70 464	249	1.29 535	9.99 944	6
55	8.70 657	247	8.70 714	248	1.29 286	9.99 943	5
56	8.70 905	245	8.70 962	246	1.29 038	9.99 943	4
57	8.71 150	244	8.71 208	245	1.28 791	9.99 942	3
58	8.71 395	243	8.71 453	243	1.28 546	9.99 942	2
59	8.71 638	241	8.71 697	242	1.28 303	9.99 941	1
60	8.71 880	241	8.71 939	242	1.28 060	9.99 940	0
	Log. Cos.	D	Log. Cot.	Com. D.	Log. Tan.	Log. Sin.	

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**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

										P. P.			
'	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.							
0	8.71 880		8.71 939	241	1.28 060	9.99 940	60			330	320	310	300
1	8.72 120	240	8.72 180	240	1.27 819	9.99 940	59	6	33.0	32.0	31.0	30.0	
2	8.72 359	239	8.72 420	238	1.27 579	9.99 939	58	7	38.5	37.3	36.1	35.0	
3	8.72 597	237	8.72 659	237	1.27 341	9.99 938	57	8	44.0	42.6	41.3	40.0	
4	8.72 833	236	8.72 896	237	1.27 104	9.99 938	56	9	49.5	48.0	46.5	45.0	
5	8.73 069	235	8.73 131	235	1.26 868	9.99 937	55	10	55.0	53.3	51.6	50.0	
6	8.73 302	233	8.73 366	233	1.26 633	9.99 936	54	20	110.0	106.6	103.3	100.0	
7	8.73 535	233	8.73 599	233	1.26 400	9.99 935	53	30	165.0	160.0	155.0	150.0	
8	8.73 766	231	8.73 831	232	1.26 168	9.99 935	52	40	220.0	213.3	206.6	200.0	
9	8.73 997	230	8.74 062	231	1.25 937	9.99 934	51	50	275.0	266.6	258.3	250.0	
10	8.74 226	229	8.74 292	229	1.25 708	9.99 933	50			290	280	270	260
11	8.74 453	227	8.74 520	228	1.25 479	9.99 933	49	6	29.0	28.0	27.0	26.0	
12	8.74 680	226	8.74 748	227	1.25 252	9.99 932	48	7	33.8	32.6	31.5	30.3	
13	8.74 905	225	8.74 974	226	1.25 026	9.99 931	47	8	38.6	37.3	36.0	34.6	
14	8.75 129	224	8.75 199	225	1.24 801	9.99 931	46	9	43.5	42.0	40.5	39.0	
15	8.75 353	223	8.75 422	223	1.24 577	9.99 930	45	10	48.3	46.6	45.0	43.3	
16	8.75 574	221	8.75 645	223	1.24 354	9.99 929	44	20	96.6	93.3	90.0	86.6	
17	8.75 795	219	8.75 867	220	1.24 133	9.99 928	43	30	145.0	140.0	135.0	130.0	
18	8.76 015	218	8.76 087	219	1.23 913	9.99 928	42	40	193.3	186.6	180.0	173.3	
19	8.76 233	217	8.76 306	218	1.23 693	9.99 927	41	50	241.6	233.3	225.0	216.6	
20	8.76 451	216	8.76 524	217	1.23 475	9.99 926	40			250	240	230	220
21	8.76 667	215	8.76 741	216	1.23 258	9.99 925	39	6	25.0	24.0	23.0	22.0	
22	8.76 883	214	8.76 958	215	1.23 042	9.99 925	38	7	29.1	28.0	26.8	25.6	
23	8.77 097	213	8.77 172	214	1.22 827	9.99 924	37	8	33.3	32.0	30.6	29.3	
24	8.77 310	212	8.77 386	213	1.22 613	9.99 923	36	9	37.5	36.0	34.5	33.0	
25	8.77 522	211	8.77 599	212	1.22 400	9.99 922	35	10	41.6	40.0	38.3	36.6	
26	8.77 733	210	8.77 811	211	1.22 188	9.99 922	34	20	83.3	80.0	76.6	73.3	
27	8.77 943	209	8.78 022	210	1.21 978	9.99 921	33	30	125.0	120.0	115.0	110.0	
28	8.78 152	208	8.78 232	209	1.21 768	9.99 920	32	40	166.6	160.0	153.3	146.6	
29	8.78 360	207	8.78 441	208	1.21 559	9.99 919	31	50	208.3	200.0	191.6	183.3	
30	8.78 567	206	8.78 648	207	1.21 351	9.99 919	30			210	200	190	180
31	8.78 773	205	8.78 855	206	1.21 144	9.99 918	29	6	21.0	20.0	19.0	18.0	
32	8.78 978	204	8.79 061	205	1.20 938	9.99 917	28	7	24.5	23.3	22.1	21.0	
33	8.79 183	203	8.79 266	204	1.20 734	9.99 916	27	8	28.0	26.6	25.3	24.0	
34	8.79 386	202	8.79 470	203	1.20 530	9.99 916	26	9	31.5	30.0	28.5	27.0	
35	8.79 588	201	8.79 673	202	1.20 327	9.99 915	25	10	35.0	33.3	31.6	30.0	
36	8.79 789	200	8.79 875	201	1.20 125	9.99 914	24	20	70.0	66.6	63.3	60.0	
37	8.79 989	199	8.80 076	200	1.19 923	9.99 913	23	30	105.0	100.0	95.0	90.0	
38	8.80 189	198	8.80 276	199	1.19 723	9.99 912	22	40	140.0	133.3	126.6	120.0	
39	8.80 387	197	8.80 476	198	1.19 524	9.99 912	21	50	175.0	166.6	158.3	150.0	
40	8.80 585	196	8.80 674	197	1.19 326	9.99 911	20			9	8	7	6
41	8.80 782	195	8.80 871	196	1.19 128	9.99 910	19	6	0.9	0.9	0.8	0.7	0.5
42	8.80 977	194	8.81 068	195	1.18 931	9.99 909	18	7	1.1	1.0	0.9	0.8	0.6
43	8.81 172	193	8.81 264	194	1.18 736	9.99 908	17	8	1.2	1.1	1.0	0.9	0.7
44	8.81 366	192	8.81 459	193	1.18 541	9.99 907	16	9	1.4	1.3	1.2	1.0	0.9
45	8.81 560	191	8.81 653	192	1.18 347	9.99 907	15	10	1.6	1.5	1.3	1.1	1.0
46	8.81 752	190	8.81 846	191	1.18 154	9.99 906	14	20	3.1	3.0	2.6	2.3	2.0
47	8.81 943	189	8.82 038	190	1.17 961	9.99 905	13	30	4.7	4.5	4.0	3.5	3.0
48	8.82 134	188	8.82 230	189	1.17 770	9.99 904	12	40	6.3	6.0	5.3	4.6	4.0
49	8.82 324	187	8.82 420	188	1.17 579	9.99 903	11	50	7.9	7.5	6.6	5.8	5.0
50	8.82 513	186	8.82 610	187	1.17 389	9.99 902	10			4	3	2	1
51	8.82 701	185	8.82 799	186	1.17 201	9.99 902	9	6	0.4	0.4	0.3	0.2	0.1
52	8.82 888	184	8.82 987	185	1.17 012	9.99 901	8	7	0.5	0.4	0.3	0.2	0.1
53	8.83 075	183	8.83 175	184	1.16 825	9.99 900	7	8	0.6	0.5	0.4	0.3	0.1
54	8.83 260	182	8.83 361	183	1.16 638	9.99 899	6	9	0.7	0.6	0.4	0.3	0.1
55	8.83 445	181	8.83 547	182	1.16 453	9.99 898	5	10	0.7	0.6	0.4	0.3	0.1
56	8.83 629	180	8.83 732	181	1.16 268	9.99 897	4	20	1.5	1.3	1.0	0.6	0.3
57	8.83 813	179	8.83 916	180	1.16 083	9.99 896	3	30	2.2	2.0	1.5	1.0	0.5
58	8.83 995	178	8.84 100	179	1.15 900	9.99 895	2	40	3.0	2.6	2.0	1.3	0.6
59	8.84 177	177	8.84 282	178	1.15 717	9.99 895	1	50	3.7	3.3	2.5	1.6	0.8
60	8.84 358	176	8.84 464	177	1.15 535	9.99 894	0						
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.				P. P.			

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.		P. P.				
0	8.84 358		8.84 464		1.15 535	9.99 894	60		181	180	178	176
1	8.84 538	180	8.84 645	181	1.15 354	9.99 893	59	6	18.1	18.0	17.8	17.6
2	8.84 718	180	8.84 826	180	1.15 174	9.99 892	58	7	21.1	21.0	20.7	20.5
3	8.84 897	178	8.85 005	179	1.14 994	9.99 891	57	8	24.1	24.0	23.7	23.4
4	8.85 075	178	8.85 184	179	1.14 815	9.99 890	56	9	27.1	27.0	26.7	26.4
5	8.85 252	177	8.85 363	178	1.14 637	9.99 889	55	10	30.1	30.0	29.6	29.3
6	8.85 429	176	8.85 540	177	1.14 459	9.99 888	54	20	60.3	60.0	59.3	58.6
7	8.85 605	176	8.85 717	176	1.14 283	9.99 888	53	30	90.5	90.0	89.0	88.0
8	8.85 780	175	8.85 893	176	1.14 107	9.99 887	52	40	120.6	120.0	118.6	117.3
9	8.85 954	174	8.86 068	175	1.13 931	9.99 886	51	50	150.8	150.0	148.3	146.6
10	8.86 128	174	8.86 243	175	1.13 756	9.99 885	50		174	172	170	168
11	8.86 301	173	8.86 417	174	1.13 582	9.99 884	49	6	17.4	17.2	17.0	16.8
12	8.86 474	172	8.86 590	173	1.13 409	9.99 883	48	7	20.3	20.0	19.8	19.6
13	8.86 645	171	8.86 763	172	1.13 237	9.99 882	47	8	23.2	22.9	22.6	22.4
14	8.86 816	171	8.86 935	172	1.13 065	9.99 881	46	9	26.1	25.8	25.5	25.2
15	8.86 987	170	8.87 106	171	1.12 893	9.99 880	45	10	29.0	28.6	28.3	28.0
16	8.87 156	169	8.87 277	170	1.12 723	9.99 879	44	20	58.0	57.3	56.6	56.0
17	8.87 325	168	8.87 447	169	1.12 553	9.99 878	43	30	87.0	86.0	85.0	84.0
18	8.87 494	167	8.87 616	169	1.12 384	9.99 877	42	40	116.0	114.6	113.3	112.0
19	8.87 661	167	8.87 785	169	1.12 215	9.99 876	41	50	145.0	143.3	141.6	140.0
20	8.87 828	167	8.87 953	168	1.12 047	9.99 875	40		166	164	162	160
21	8.87 995	166	8.88 120	167	1.11 880	9.99 874	39	6	16.6	16.4	16.2	16.0
22	8.88 160	165	8.88 287	167	1.11 713	9.99 874	38	7	19.3	19.1	18.9	18.6
23	8.88 326	165	8.88 453	166	1.11 547	9.99 873	37	8	22.1	21.8	21.6	21.3
24	8.88 490	164	8.88 618	165	1.11 381	9.99 872	36	9	24.9	24.6	24.3	24.0
25	8.88 654	163	8.88 783	165	1.11 216	9.99 871	35	10	27.6	27.3	27.0	26.6
26	8.88 817	163	8.88 947	164	1.11 052	9.99 870	34	20	55.3	54.6	54.0	53.3
27	8.88 980	162	8.89 111	163	1.10 889	9.99 869	33	30	83.0	82.0	81.0	80.0
28	8.89 142	161	8.89 274	162	1.10 726	9.99 868	32	40	110.6	109.3	108.0	106.6
29	8.89 303	161	8.89 436	162	1.10 563	9.99 867	31	50	138.3	136.6	135.0	133.3
30	8.89 464	160	8.89 598	161	1.10 401	9.99 866	30		158	156	154	152
31	8.89 624	159	8.89 759	161	1.10 240	9.99 865	29	6	15.8	15.6	15.4	15.2
32	8.89 784	159	8.89 920	160	1.10 079	9.99 864	28	7	18.4	18.2	17.9	17.7
33	8.89 943	158	8.90 080	159	1.09 919	9.99 863	27	8	21.0	20.8	20.5	20.2
34	8.90 101	158	8.90 240	159	1.09 760	9.99 862	26	9	23.7	23.4	23.1	22.8
35	8.90 259	157	8.90 398	158	1.09 601	9.99 861	25	10	26.3	26.0	25.6	25.3
36	8.90 417	156	8.90 557	157	1.09 443	9.99 860	24	20	52.6	52.0	51.3	50.6
37	8.90 573	156	8.90 714	157	1.09 285	9.99 859	23	30	79.0	78.0	77.0	76.0
38	8.90 729	156	8.90 872	156	1.09 128	9.99 858	22	40	105.3	104.0	102.6	101.3
39	8.90 885	156	8.91 028	156	1.08 971	9.99 857	21	50	131.6	130.0	128.3	126.6
40	8.91 040	155	8.91 184	155	1.08 815	9.99 856	20		150	149	148	147
41	8.91 195	154	8.91 340	155	1.08 660	9.99 855	19	6	15.0	14.9	14.8	14.7
42	8.91 349	153	8.91 495	154	1.08 505	9.99 853	18	7	17.5	17.4	17.2	17.1
43	8.91 502	153	8.91 649	154	1.08 350	9.99 852	17	8	20.0	19.8	19.7	19.6
44	8.91 655	153	8.91 803	154	1.08 196	9.99 851	16	9	22.5	22.3	22.2	22.0
45	8.91 807	152	8.91 957	153	1.08 043	9.99 850	15	10	25.0	24.8	24.6	24.5
46	8.91 959	151	8.92 109	152	1.07 890	9.99 849	14	20	50.0	49.6	49.3	49.0
47	8.92 110	151	8.92 262	151	1.07 738	9.99 848	13	30	75.0	74.5	74.0	73.5
48	8.92 261	150	8.92 413	151	1.07 586	9.99 847	12	40	100.0	99.3	98.6	98.0
49	8.92 411	150	8.92 565	151	1.07 435	9.99 846	11	50	125.0	124.1	123.3	122.5
50	8.92 561	149	8.92 715	150	1.07 284	9.99 845	10		146	145	144	143
51	8.92 710	148	8.92 866	149	1.07 134	9.99 844	9	6	14.6	14.5	14.4	14.3
52	8.92 858	148	8.93 015	149	1.06 984	9.99 843	8	7	17.0	16.9	16.8	16.7
53	8.93 007	147	8.93 164	149	1.06 835	9.99 842	7	8	19.4	19.3	19.2	19.1
54	8.93 154	147	8.93 313	148	1.06 686	9.99 841	6	9	21.9	21.7	21.6	21.5
55	8.93 301	146	8.93 461	148	1.06 538	9.99 840	5	10	24.3	24.1	24.0	23.9
56	8.93 448	146	8.93 609	147	1.06 390	9.99 839	4	20	48.6	48.3	48.0	47.7
57	8.93 594	146	8.93 756	146	1.06 243	9.99 837	3	30	73.0	72.5	72.0	71.5
58	8.93 740	145	8.93 903	146	1.06 097	9.99 836	2	40	97.3	96.6	96.0	95.3
59	8.93 885	145	8.94 049	146	1.05 950	9.99 835	1	50	121.6	120.8	120.0	119.2
60	8.94 029	144	8.94 195	145	1.05 805	9.99 834	0		146	145	144	143
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.		P. P.				

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**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

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'	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.		P. P.				
0	8.94 029	141	8.94 195	145	1.05 805	9.99 834	60	145	144	143	142	141
1	8.94 174	143	8.94 340	144	1.05 659	9.99 833	59	6 14.5	14.4	14.3	14.2	14.1
2	8.94 317	143	8.94 485	144	1.05 515	9.99 832	58	7 16.9	16.8	16.7	16.5	16.4
3	8.94 460	143	8.94 629	144	1.05 370	9.99 831	57	8 19.3	19.2	19.0	18.9	18.8
4	8.94 603	143	8.94 773	144	1.05 226	9.99 830	56	9 21.7	21.6	21.4	21.3	21.1
5	8.94 745	142	8.94 917	143	1.05 083	9.99 829	55	10 24.1	24.0	23.8	23.6	23.5
6	8.94 887	141	8.95 059	142	1.04 940	9.99 827	54	20 48.3	48.0	47.6	47.3	47.0
7	8.95 028	141	8.95 202	142	1.04 798	9.99 826	53	30 72.5	72.0	71.5	71.0	70.5
8	8.95 169	140	8.95 344	141	1.04 656	9.99 825	52	40 96.6	96.0	95.3	94.6	94.0
9	8.95 310	140	8.95 485	141	1.04 514	9.99 824	51	50 120.8	120.0	119.1	118.3	117.5
10	8.95 450	139	8.95 626	141	1.04 373	9.99 823	50	140	139	138	137	136
11	8.95 589	139	8.95 767	141	1.04 232	9.99 822	49	6 14.0	13.9	13.8	13.7	13.6
12	8.95 728	138	8.95 907	140	1.04 092	9.99 821	48	7 16.3	16.2	16.1	16.0	15.8
13	8.95 867	138	8.96 046	140	1.03 952	9.99 819	47	8 18.6	18.5	18.4	18.2	18.1
14	8.96 005	138	8.96 187	139	1.03 813	9.99 818	46	9 21.0	20.8	20.7	20.5	20.4
15	8.96 143	138	8.96 325	139	1.03 674	9.99 817	45	10 23.3	23.1	23.0	22.8	22.6
16	8.96 280	137	8.96 464	138	1.03 536	9.99 816	44	20 46.6	46.3	46.0	45.6	45.3
17	8.96 417	135	8.96 602	137	1.03 398	9.99 815	43	30 70.0	69.5	69.0	68.5	68.0
18	8.96 553	136	8.96 739	137	1.03 260	9.99 814	42	40 93.3	92.6	92.0	91.3	90.6
19	8.96 689	136	8.96 876	137	1.03 123	9.99 813	41	50 116.6	115.8	115.0	114.1	113.3
20	8.96 825	135	8.97 013	137	1.02 986	9.99 811	40	135	134	133	132	
21	8.96 960	134	8.97 149	136	1.02 850	9.99 810	39	6 13.5	13.4	13.3	13.2	
22	8.97 094	134	8.97 285	135	1.02 714	9.99 809	38	7 15.7	15.6	15.5	15.4	
23	8.97 229	134	8.97 421	135	1.02 579	9.99 808	37	8 18.0	17.8	17.7	17.6	
24	8.97 363	133	8.97 556	134	1.02 444	9.99 807	36	9 20.2	20.1	19.9	19.8	
25	8.97 496	133	8.97 690	134	1.02 309	9.99 805	35	10 22.5	22.3	22.1	22.0	
26	8.97 629	132	8.97 825	133	1.02 175	9.99 804	34	20 45.0	44.6	44.3	44.0	
27	8.97 762	132	8.97 958	133	1.02 041	9.99 803	33	30 67.5	67.0	66.5	66.0	
28	8.97 894	132	8.98 092	133	1.01 908	9.99 802	32	40 90.0	89.3	88.6	88.0	
29	8.98 026	132	8.98 225	133	1.01 775	9.99 801	31	50 112.5	111.6	110.8	110.0	
30	8.98 157	131	8.98 357	132	1.01 642	9.99 799	30	131	130	129	128	
31	8.98 288	131	8.98 490	131	1.01 510	9.99 798	29	6 13.1	13.0	12.9	12.8	
32	8.98 419	130	8.98 621	131	1.01 378	9.99 797	28	7 15.3	15.1	15.0	14.9	
33	8.98 549	130	8.98 753	131	1.01 247	9.99 796	27	8 17.4	17.3	17.2	17.0	
34	8.98 679	129	8.98 884	131	1.01 116	9.99 794	26	9 19.6	19.5	19.3	19.2	
35	8.98 808	129	8.99 015	130	1.00 985	9.99 793	25	10 21.8	21.6	21.5	21.3	
36	8.98 937	128	8.99 145	130	1.00 855	9.99 792	24	20 43.6	43.3	43.0	42.6	
37	8.99 066	128	8.99 275	129	1.00 725	9.99 791	23	30 65.5	65.0	64.5	64.0	
38	8.99 194	127	8.99 404	129	1.00 595	9.99 789	22	40 87.3	86.6	86.0	85.3	
39	8.99 322	127	8.99 533	129	1.00 466	9.99 788	21	50 109.1	108.3	107.5	106.6	
40	8.99 449	127	8.99 662	129	1.00 337	9.99 787	20	127	126	125	124	123
41	8.99 577	126	8.99 791	128	1.00 209	9.99 786	19	6 12.7	12.6	12.5	12.4	12.3
42	8.99 703	126	8.99 919	127	1.00 081	9.99 784	18	7 14.8	14.7	14.6	14.4	14.3
43	8.99 830	126	9.00 046	127	0.99 953	9.99 783	17	8 16.9	16.8	16.6	16.5	16.4
44	8.99 956	126	9.00 174	127	0.99 826	9.99 782	16	9 19.0	18.9	18.7	18.6	18.4
45	9.00 081	125	9.00 300	126	0.99 699	9.99 781	15	10 21.1	21.0	20.8	20.6	20.5
46	9.00 207	125	9.00 427	126	0.99 573	9.99 779	14	20 42.3	42.0	41.6	41.3	41.0
47	9.00 332	124	9.00 553	125	0.99 446	9.99 778	13	30 63.5	63.0	62.5	62.0	61.5
48	9.00 456	124	9.00 679	125	0.99 321	9.99 777	12	40 84.6	84.0	83.3	82.6	82.0
49	9.00 580	124	9.00 804	125	0.99 195	9.99 776	11	50 105.8	105.0	104.1	103.3	102.5
50	9.00 704	123	9.00 930	124	0.99 070	9.99 774	10	122	121	120	119	118
51	9.00 828	123	9.01 054	124	0.98 945	9.99 773	9	6 12.2	12.1	12.0	0.1	0.0
52	9.00 951	122	9.01 179	124	0.98 821	9.99 772	8	7 14.2	14.1	14.0	0.2	0.1
53	9.01 073	122	9.01 303	124	0.98 697	9.99 770	7	8 16.2	16.1	16.0	0.2	0.1
54	9.01 196	122	9.01 427	124	0.98 573	9.99 769	6	9 18.3	18.1	18.0	0.2	0.1
55	9.01 318	122	9.01 550	123	0.98 450	9.99 768	5	10 20.3	20.1	20.0	0.2	0.1
56	9.01 440	121	9.01 673	123	0.98 327	9.99 766	4	20 40.6	40.3	40.0	0.5	0.3
57	9.01 561	121	9.01 796	122	0.98 204	9.99 765	3	30 61.0	60.5	60.0	0.7	0.5
58	9.01 682	120	9.01 918	122	0.98 081	9.99 764	2	40 81.3	80.6	80.0	1.0	0.6
59	9.01 803	120	9.02 040	121	0.97 959	9.99 763	1	50 101.6	100.8	100.0	1.2	0.8
60	9.01 923	120	9.02 162	121	0.97 838	9.99 761	0	121	120	119	118	117
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.	'	P. P.				

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.		P. P.				
0	9.01 923	120	9.02 162	121	0.97 838	9.99 761	60	121	121	120	119	118
1	9.02 043	119	9.02 283	121	0.97 716	9.99 760	59	6 12.1	12.1	12.0	11.9	11.8
2	9.02 163	119	9.02 404	120	0.97 595	9.99 759	58	7 14.2	14.1	14.0	13.9	13.7
3	9.02 282	119	9.02 525	120	0.97 475	9.99 757	57	8 16.2	16.1	16.0	15.8	15.7
4	9.02 401	119	9.02 645	120	0.97 354	9.99 756	56	9 18.2	18.1	18.0	17.8	17.7
5	9.02 520	119	9.02 765	120	0.97 234	9.99 754	55	10 20.2	20.1	20.0	19.8	19.6
6	9.02 638	118	9.02 885	119	0.97 115	9.99 753	54	20 40.5	40.3	40.0	39.6	39.3
7	9.02 756	118	9.03 004	119	0.96 995	9.99 752	53	30 60.7	60.5	60.0	59.5	59.0
8	9.02 874	117	9.03 123	119	0.96 876	9.99 750	52	40 81.0	80.6	80.0	79.3	78.6
9	9.02 992	117	9.03 242	119	0.96 757	9.99 749	51	50 101.2	100.8	100.0	99.1	98.3
10	9.03 109	117	9.03 361	118	0.96 639	9.99 748	50	117 117 116 115				
1	9.03 225	116	9.03 479	118	0.96 521	9.99 746	49	6 11.7	11.7	11.6	11.5	
2	9.03 342	116	9.03 597	117	0.96 403	9.99 745	48	7 13.7	13.6	13.5	13.4	
3	9.03 458	116	9.03 714	117	0.96 285	9.99 744	47	8 15.6	15.6	15.4	15.3	
4	9.03 574	116	9.03 831	117	0.96 168	9.99 742	46	9 17.6	17.5	17.4	17.2	
5	9.03 689	115	9.03 948	117	0.96 051	9.99 741	45	10 19.6	19.5	19.3	19.1	
6	9.03 805	114	9.04 065	116	0.95 935	9.99 739	44	20 39.1	39.0	38.6	38.3	
7	9.03 919	114	9.04 181	116	0.95 818	9.99 738	43	30 58.7	58.5	58.0	57.5	
8	9.04 034	114	9.04 297	116	0.95 702	9.99 737	42	40 78.3	78.0	77.3	76.6	
9	9.04 148	114	9.04 413	115	0.95 587	9.99 735	41	50 97.9	97.5	96.6	95.8	
20	9.04 262	114	9.04 528	115	0.95 471	9.99 734	40	114 114 113 112 111				
21	9.04 376	113	9.04 643	114	0.95 356	9.99 733	39	6 11.4	11.4	11.3	11.2	11.1
22	9.04 489	113	9.04 758	114	0.95 242	9.99 731	38	7 13.3	13.3	13.2	13.0	12.9
23	9.04 602	113	9.04 872	114	0.95 127	9.99 730	37	8 15.2	15.2	15.0	14.9	14.8
24	9.04 715	113	9.04 987	114	0.95 013	9.99 728	36	9 17.2	17.1	16.9	16.8	16.6
25	9.04 828	112	9.05 101	114	0.94 899	9.99 727	35	10 19.1	19.0	18.8	18.6	18.5
26	9.04 940	112	9.05 214	113	0.94 785	9.99 725	34	20 38.1	38.0	37.6	37.3	37.0
27	9.05 052	111	9.05 327	113	0.94 672	9.99 724	33	30 57.2	57.0	56.5	56.0	55.5
28	9.05 163	111	9.05 440	113	0.94 559	9.99 723	32	40 76.3	76.0	75.3	74.6	74.0
29	9.05 275	111	9.05 553	113	0.94 446	9.99 721	31	50 95.4	95.0	94.1	93.3	92.5
30	9.05 388	111	9.05 666	112	0.94 334	9.99 720	30	110 110 109 108				
31	9.05 496	110	9.05 778	112	0.94 222	9.99 718	29	6 11.0	11.0	10.9	10.8	
32	9.05 607	110	9.05 890	111	0.94 110	9.99 717	28	7 12.9	12.8	12.7	12.6	
33	9.05 717	110	9.06 001	111	0.93 998	9.99 715	27	8 14.7	14.6	14.5	14.4	
34	9.05 827	110	9.06 113	111	0.93 887	9.99 714	26	9 16.6	16.5	16.3	16.2	
35	9.05 936	109	9.06 224	111	0.93 776	9.99 712	25	10 18.4	18.3	18.1	18.0	
36	9.06 046	109	9.06 335	110	0.93 665	9.99 711	24	20 36.8	36.6	36.3	36.0	
37	9.06 155	109	9.06 445	110	0.93 554	9.99 710	23	30 55.2	55.0	54.5	54.0	
38	9.06 264	109	9.06 555	110	0.93 444	9.99 708	22	40 73.6	73.3	72.6	72.0	
39	9.06 372	108	9.06 665	110	0.93 334	9.99 707	21	50 92.1	91.6	90.8	90.0	
40	9.06 480	108	9.06 775	109	0.93 225	9.99 705	20	107 107 106 105 104				
41	9.06 588	108	9.06 884	109	0.93 115	9.99 704	19	6 10.7	10.7	10.6	10.5	10.4
42	9.06 696	107	9.06 994	109	0.93 006	9.99 702	18	7 12.5	12.5	12.3	12.2	12.1
43	9.06 803	107	9.07 102	109	0.92 897	9.99 701	17	8 14.3	14.2	14.1	14.0	13.8
44	9.06 910	107	9.07 211	109	0.92 788	9.99 699	16	9 16.1	16.0	15.9	15.7	15.6
45	9.07 017	107	9.07 319	108	0.92 680	9.99 698	15	10 17.9	17.8	17.6	17.5	17.3
46	9.07 124	106	9.07 428	108	0.92 572	9.99 696	14	20 35.8	35.6	35.3	35.0	34.6
47	9.07 230	106	9.07 535	107	0.92 464	9.99 695	13	30 53.7	53.5	53.0	52.5	52.0
48	9.07 336	106	9.07 643	107	0.92 357	9.99 693	12	40 71.6	71.3	70.6	70.0	69.3
49	9.07 442	106	9.07 750	107	0.92 249	9.99 692	11	50 89.6	89.1	88.3	87.5	86.6
50	9.07 548	105	9.07 857	107	0.92 142	9.99 690	10	103 103 2 1 1				
51	9.07 653	105	9.07 964	106	0.92 035	9.99 689	9	6 10.3	10.3	10.2	10.1	10.0
52	9.07 758	104	9.08 071	106	0.91 929	9.99 687	8	7 12.1	12.0	11.9	11.8	11.7
53	9.07 863	104	9.08 177	106	0.91 822	9.99 686	7	8 13.8	13.7	13.6	13.5	13.4
54	9.07 967	104	9.08 283	106	0.91 716	9.99 684	6	9 15.5	15.4	15.3	15.2	15.1
55	9.08 072	104	9.08 389	105	0.91 611	9.99 683	5	10 17.2	17.1	17.0	16.9	16.8
56	9.08 176	103	9.08 494	105	0.91 505	9.99 681	4	20 34.5	34.3	34.0	33.6	33.3
57	9.08 279	103	9.08 600	105	0.91 400	9.99 679	3	30 51.7	51.5	51.0	50.7	50.5
58	9.08 383	103	9.08 705	105	0.91 295	9.99 678	2	40 69.0	68.6	68.1	67.3	66.6
59	9.08 486	103	9.08 810	104	0.91 190	9.99 676	1	50 86.2	85.8	85.1	84.2	83.3
60	9.08 589	103	9.08 914	104	0.91 085	9.99 675	0	P. P.				
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.						

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**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

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'	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.	
0	9.08 589	102	9.08 314	104	0.91 085	9.99 675	60
1	9.08 692	102	9.09 018	104	0.90 981	9.99 673	59
2	9.08 794	102	9.09 123	103	0.90 877	9.99 672	58
3	9.08 897	102	9.09 226	103	0.90 773	9.99 670	57
4	9.08 999	102	9.09 330	103	0.90 670	9.99 669	56
5	9.09 101	101	9.09 433	103	0.90 566	9.99 667	55
6	9.09 202	101	9.09 536	103	0.90 463	9.99 665	54
7	9.09 303	101	9.09 639	102	0.90 360	9.99 664	53
8	9.09 404	101	9.09 742	102	0.90 258	9.99 662	52
9	9.09 505	101	9.09 844	102	0.90 155	9.99 661	51
10	9.09 606	100	9.09 947	102	0.90 053	9.99 659	50
11	9.09 706	100	9.10 048	101	0.89 951	9.99 658	49
12	9.09 806	100	9.10 150	101	0.89 849	9.99 656	48
13	9.09 906	99	9.10 252	101	0.89 748	9.99 654	47
14	9.10 006	99	9.10 353	101	0.89 647	9.99 653	46
15	9.10 105	99	9.10 454	101	0.89 546	9.99 651	45
16	9.10 205	98	9.10 555	100	0.89 445	9.99 650	44
17	9.10 303	99	9.10 655	100	0.89 344	9.99 648	43
18	9.10 402	98	9.10 756	100	0.89 244	9.99 646	42
19	9.10 501	98	9.10 856	100	0.89 144	9.99 645	41
20	9.10 599	98	9.10 956	100	0.89 044	9.99 643	40
21	9.10 697	97	9.11 055	99	0.88 944	9.99 641	39
22	9.10 795	97	9.11 155	99	0.88 845	9.99 640	38
23	9.10 892	97	9.11 254	99	0.88 745	9.99 638	37
24	9.10 990	97	9.11 353	98	0.88 646	9.99 637	36
25	9.11 087	96	9.11 452	98	0.88 548	9.99 635	35
26	9.11 184	97	9.11 550	98	0.88 449	9.99 633	34
27	9.11 281	96	9.11 649	98	0.88 351	9.99 632	33
28	9.11 377	96	9.11 747	98	0.88 253	9.99 630	32
29	9.11 473	96	9.11 845	98	0.88 155	9.99 628	31
30	9.11 570	95	9.11 943	97	0.88 057	9.99 627	30
31	9.11 665	96	9.12 040	97	0.87 959	9.99 625	29
32	9.11 761	95	9.12 137	97	0.87 862	9.99 623	28
33	9.11 856	95	9.12 235	96	0.87 765	9.99 622	27
34	9.11 952	95	9.12 331	96	0.87 668	9.99 620	26
35	9.12 047	94	9.12 428	96	0.87 571	9.99 618	25
36	9.12 141	94	9.12 525	96	0.87 475	9.99 617	24
37	9.12 236	94	9.12 621	96	0.87 379	9.99 615	23
38	9.12 330	94	9.12 717	96	0.87 283	9.99 613	22
39	9.12 425	93	9.12 813	95	0.87 187	9.99 611	21
40	9.12 518	94	9.12 908	95	0.87 091	9.99 610	20
41	9.12 612	93	9.13 004	95	0.86 996	9.99 608	19
42	9.12 706	93	9.13 099	95	0.86 900	9.99 606	18
43	9.12 799	93	9.13 194	95	0.86 805	9.99 605	17
44	9.12 892	93	9.13 289	94	0.86 710	9.99 603	16
45	9.12 985	92	9.13 384	94	0.86 616	9.99 601	15
46	9.13 078	92	9.13 478	94	0.86 521	9.99 600	14
47	9.13 170	92	9.13 572	94	0.86 427	9.99 598	13
48	9.13 263	92	9.13 666	94	0.86 333	9.99 596	12
49	9.13 355	92	9.13 760	94	0.86 239	9.99 594	11
50	9.13 447	91	9.13 854	93	0.86 146	9.99 593	10
51	9.13 538	92	9.13 947	93	0.86 052	9.99 591	9
52	9.13 630	91	9.14 041	93	0.85 959	9.99 589	8
53	9.13 721	91	9.14 134	93	0.85 866	9.99 587	7
54	9.13 813	90	9.14 227	92	0.85 773	9.99 586	6
55	9.13 903	91	9.14 319	92	0.85 680	9.99 584	5
56	9.13 994	90	9.14 412	92	0.85 588	9.99 582	4
57	9.14 085	90	9.14 504	92	0.85 495	9.99 580	3
58	9.14 175	90	9.14 596	92	0.85 403	9.99 579	2
59	9.14 265	90	9.14 688	92	0.85 311	9.99 577	1
60	9.14 355	90	9.14 780	92	0.85 219	9.99 575	0
Log. Cos.		d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.	'

P. P.

	104	103	102	101
6	10.4	10.3	10.2	10.1
7	12.1	12.0	11.9	11.8
8	13.8	13.7	13.6	13.4
9	15.6	15.4	15.3	15.1
10	17.3	17.1	17.0	16.8
20	34.6	34.3	34.0	33.6
30	52.0	51.5	51.0	50.5
40	69.3	68.6	68.0	67.3
50	86.6	85.8	85.0	84.1

	100	100	99	98
6	10.0	10.0	9.9	9.8
7	11.7	11.6	11.5	11.4
8	13.4	13.3	13.2	13.0
9	15.1	15.0	14.8	14.7
10	16.7	16.6	16.5	16.3
20	33.5	33.3	33.0	32.6
30	50.2	50.0	49.5	49.0
40	67.0	66.6	66.0	65.3
50	83.7	83.3	82.5	81.6

	97	97	96	95
6	9.7	9.7	9.6	9.5
7	11.4	11.3	11.2	11.1
8	13.0	12.9	12.8	12.6
9	14.6	14.5	14.4	14.2
10	16.2	16.1	16.0	15.8
20	32.5	32.3	32.0	31.6
30	48.7	48.5	48.0	47.5
40	65.0	64.6	64.0	63.3
50	81.2	80.8	80.0	79.1

	94	94	93	92
6	9.4	9.4	9.3	9.2
7	11.0	10.9	10.8	10.7
8	12.6	12.5	12.4	12.2
9	14.2	14.1	13.9	13.8
10	15.7	15.6	15.5	15.3
20	31.5	31.3	31.0	30.6
30	47.2	47.0	46.5	46.0
40	63.0	62.6	62.0	61.3
50	78.7	78.3	77.5	76.6

	91	91	90	2	1
6	9.1	9.1	9.0	0.2	0.1
7	10.7	10.6	10.5	0.2	0.1
8	12.2	12.1	12.0	0.2	0.2
9	13.7	13.6	13.5	0.3	0.2
10	15.2	15.1	15.0	0.3	0.2
20	30.5	30.3	30.0	0.6	0.5
30	45.7	45.5	45.0	1.0	0.7
40	61.0	60.6	60.0	1.3	1.0
50	76.2	75.8	75.0	1.6	1.2

P. P.

97°

82°

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

171°

	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.		P. P.
0	9.14 355		9.14 780	91	0.85 219	9.99 575	60	
1	9.14 445	90	9.14 872	91	0.85 128	9.99 573	59	
2	9.14 535	89	9.14 963	91	0.85 037	9.99 571	58	
3	9.14 624	89	9.15 054	91	0.84 945	9.99 570	57	
4	9.14 713	89	9.15 145	91	0.84 854	9.99 568	56	
5	9.14 802	89	9.15 236	91	0.84 763	9.99 566	55	
6	9.14 891	88	9.15 327	90	0.84 673	9.99 564	54	
7	9.14 980	88	9.15 417	90	0.84 582	9.99 563	53	
8	9.15 068	88	9.15 507	90	0.84 492	9.99 561	52	
9	9.15 157	88	9.15 598	90	0.84 402	9.99 559	51	
0	9.15 245	88	9.15 687	89	0.84 312	9.99 557	50	
1	9.15 333	88	9.15 777	89	0.84 222	9.99 555	49	
2	9.15 421	88	9.15 867	89	0.84 133	9.99 553	48	
3	9.15 508	87	9.15 956	89	0.84 043	9.99 552	47	
4	9.15 595	87	9.16 045	89	0.83 954	9.99 550	46	
5	9.15 683	87	9.16 134	89	0.83 865	9.99 548	45	
6	9.15 770	87	9.16 223	89	0.83 776	9.99 546	44	
7	9.15 857	87	9.16 312	89	0.83 687	9.99 544	43	
8	9.15 943	86	9.16 401	88	0.83 599	9.99 542	42	
9	9.16 030	86	9.16 489	88	0.83 511	9.99 541	41	
0	9.16 116	86	9.16 577	88	0.83 422	9.99 539	40	
1	9.16 202	86	9.16 665	88	0.83 334	9.99 537	39	
2	9.16 288	86	9.16 753	87	0.83 247	9.99 535	38	
3	9.16 374	85	9.16 841	87	0.83 159	9.99 533	37	
4	9.16 460	85	9.16 928	87	0.83 071	9.99 531	36	
5	9.16 545	85	9.17 015	87	0.82 984	9.99 529	35	
6	9.16 630	85	9.17 103	87	0.82 897	9.99 528	34	
7	9.16 716	85	9.17 190	86	0.82 810	9.99 526	33	
8	9.16 801	84	9.17 276	86	0.82 723	9.99 524	32	
9	9.16 885	84	9.17 363	87	0.82 636	9.99 522	31	
0	9.16 970	84	9.17 450	86	0.82 550	9.99 520	30	
1	9.17 054	84	9.17 536	86	0.82 464	9.99 518	29	
2	9.17 139	84	9.17 622	86	0.82 377	9.99 516	28	
3	9.17 223	84	9.17 708	85	0.82 291	9.99 514	27	
4	9.17 307	84	9.17 794	86	0.82 206	9.99 512	26	
5	9.17 391	83	9.17 880	85	0.82 120	9.99 511	25	
6	9.17 474	83	9.17 965	85	0.82 034	9.99 509	24	
7	9.17 558	83	9.18 051	85	0.81 949	9.99 507	23	
8	9.17 641	83	9.18 136	85	0.81 864	9.99 505	22	
9	9.17 724	83	9.18 221	85	0.81 779	9.99 503	21	
0	9.17 807	83	9.18 306	85	0.81 694	9.99 501	20	
1	9.17 890	82	9.18 390	84	0.81 609	9.99 499	19	
2	9.17 972	82	9.18 475	84	0.81 525	9.99 497	18	
3	9.18 055	82	9.18 559	84	0.81 440	9.99 495	17	
4	9.18 137	82	9.18 644	84	0.81 356	9.99 493	16	
5	9.18 219	82	9.18 728	84	0.81 272	9.99 491	15	
6	9.18 301	82	9.18 812	84	0.81 188	9.99 489	14	
7	9.18 383	81	9.18 896	83	0.81 104	9.99 487	13	
8	9.18 465	81	9.18 979	83	0.81 020	9.99 485	12	
9	9.18 546	81	9.19 063	83	0.80 937	9.99 484	11	
0	9.18 628	81	9.19 146	83	0.80 854	9.99 482	10	
1	9.18 709	81	9.19 229	83	0.80 770	9.99 480	9	
2	9.18 790	80	9.19 312	83	0.80 687	9.99 478	8	
3	9.18 871	80	9.19 395	82	0.80 604	9.99 476	7	
4	9.18 952	81	9.19 478	82	0.80 522	9.99 474	6	
5	9.19 032	80	9.19 560	82	0.80 439	9.99 472	5	
6	9.19 113	80	9.19 643	82	0.80 357	9.99 470	4	
7	9.19 193	80	9.19 725	82	0.80 274	9.99 468	3	
8	9.19 273	80	9.19 807	82	0.80 192	9.99 466	2	
9	9.19 353	80	9.19 889	82	0.80 110	9.99 464	1	
0	9.19 433	79	9.19 971	82	0.80 028	9.99 462	0	
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.		P. P.

	91	91	90	89
6	9.1	9.1	9.0	8.9
7	10.7	10.6	10.5	10.4
8	12.2	12.1	12.0	11.8
9	13.7	13.6	13.5	13.3
10	15.2	15.1	15.0	14.8
20	30.5	30.3	30.0	29.6
30	45.7	45.5	45.0	44.5
40	61.0	60.6	60.0	59.3
50	76.2	75.8	75.0	74.1

	88	88	87	86
6	8.8	8.8	8.7	8.6
7	10.3	10.2	10.1	10.0
8	11.8	11.7	11.6	11.4
9	13.3	13.2	13.0	12.9
10	14.7	14.6	14.5	14.3
20	29.5	29.3	29.0	28.6
30	44.2	44.0	43.5	43.0
40	59.0	58.6	58.0	57.3
50	73.7	73.3	72.5	71.6

	85	85	84	83
6	8.5	8.5	8.4	8.3
7	10.0	9.9	9.8	9.7
8	11.4	11.3	11.2	11.0
9	12.8	12.7	12.6	12.4
10	14.2	14.1	14.0	13.8
20	28.5	28.3	28.0	27.6
30	42.7	42.5	42.0	41.5
40	57.0	56.6	56.0	55.3
50	71.2	70.8	70.0	69.1

	82	82	81	80
6	8.2	8.2	8.1	8.0
7	9.6	9.5	9.4	9.3
8	11.0	10.9	10.8	10.6
9	12.4	12.3	12.1	12.0
10	13.7	13.6	13.5	13.3
20	27.5	27.3	27.0	26.6
30	41.2	41.0	40.5	40.0
40	55.0	54.6	54.0	53.3
50	68.7	68.3	67.5	66.6

	79	2	1
6	7.9	0.2	0.1
7	9.3	0.2	0.2
8	10.6	0.2	0.2
9	11.9	0.3	0.2
10	13.2	0.3	0.2
20	26.5	0.6	0.5
30	39.7	1.0	0.7
40	53.0	1.3	1.0
50	66.2	1.6	1.2

P. P.

88°

81°

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

9°

170°

'	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.	
0	9.19 433	80	9.19 971	81	0.80 028	9.99 462	60
1	9.19 513	79	9.20 053	81	0.79 947	9.99 460	59
2	9.19 592	79	9.20 134	81	0.79 865	9.99 458	58
3	9.19 672	79	9.20 216	81	0.79 784	9.99 456	57
4	9.19 751	79	9.20 297	81	0.79 703	9.99 454	56
5	9.19 830	79	9.20 378	81	0.79 622	9.99 452	55
6	9.19 909	79	9.20 459	81	0.79 541	9.99 450	54
7	9.19 988	79	9.20 540	81	0.79 460	9.99 448	53
8	9.20 066	78	9.20 620	80	0.79 379	9.99 446	52
9	9.20 145	78	9.20 701	81	0.79 298	9.99 444	51
10	9.20 223	78	9.20 781	80	0.79 218	9.99 442	50
11	9.20 301	78	9.20 862	80	0.79 138	9.99 440	49
12	9.20 379	78	9.20 942	80	0.79 058	9.99 437	48
13	9.20 457	78	9.21 022	80	0.78 978	9.99 435	47
14	9.20 535	78	9.21 102	80	0.78 898	9.99 433	46
15	9.20 613	77	9.21 181	79	0.78 818	9.99 431	45
16	9.20 690	77	9.21 261	79	0.78 739	9.99 429	44
17	9.20 768	77	9.21 340	79	0.78 659	9.99 427	43
18	9.20 845	77	9.21 420	79	0.78 580	9.99 425	42
19	9.20 922	77	9.21 499	79	0.78 501	9.99 423	41
20	9.20 999	77	9.21 578	79	0.78 422	9.99 421	40
21	9.21 076	77	9.21 657	79	0.78 343	9.99 419	39
22	9.21 152	76	9.21 735	78	0.78 264	9.99 417	38
23	9.21 229	76	9.21 814	78	0.78 186	9.99 415	37
24	9.21 305	76	9.21 892	78	0.78 107	9.99 413	36
25	9.21 382	76	9.21 971	78	0.78 029	9.99 411	35
26	9.21 458	76	9.22 049	78	0.77 951	9.99 408	34
27	9.21 534	75	9.22 127	78	0.77 873	9.99 406	33
28	9.21 609	75	9.22 205	78	0.77 795	9.99 404	32
29	9.21 685	75	9.22 283	78	0.77 717	9.99 402	31
30	9.21 761	75	9.22 360	77	0.77 639	9.99 400	30
31	9.21 836	75	9.22 438	77	0.77 562	9.99 398	29
32	9.21 911	75	9.22 515	77	0.77 484	9.99 396	28
33	9.21 987	75	9.22 593	77	0.77 407	9.99 394	27
34	9.22 062	75	9.22 670	77	0.77 330	9.99 392	26
35	9.22 136	74	9.22 747	77	0.77 253	9.99 389	25
36	9.22 211	74	9.22 824	77	0.77 176	9.99 387	24
37	9.22 286	74	9.22 900	76	0.77 099	9.99 385	23
38	9.22 360	74	9.22 977	76	0.77 022	9.99 383	22
39	9.22 435	74	9.23 054	76	0.76 946	9.99 381	21
40	9.22 509	74	9.23 130	76	0.76 870	9.99 379	20
41	9.22 583	74	9.23 206	76	0.76 793	9.99 377	19
42	9.22 657	73	9.23 282	76	0.76 717	9.99 374	18
43	9.22 731	73	9.23 358	76	0.76 641	9.99 372	17
44	9.22 805	73	9.23 434	76	0.76 565	9.99 370	16
45	9.22 878	73	9.23 510	75	0.76 489	9.99 368	15
46	9.22 952	73	9.23 586	75	0.76 414	9.99 366	14
47	9.23 025	73	9.23 661	75	0.76 338	9.99 364	13
48	9.23 098	73	9.23 737	75	0.76 263	9.99 361	12
49	9.23 171	73	9.23 812	75	0.76 188	9.99 359	11
50	9.23 244	72	9.23 887	75	0.76 113	9.99 357	10
51	9.23 317	72	9.23 962	75	0.76 038	9.99 355	9
52	9.23 390	72	9.24 037	75	0.75 963	9.99 353	8
53	9.23 462	72	9.24 112	74	0.75 888	9.99 350	7
54	9.23 535	72	9.24 186	74	0.75 813	9.99 348	6
55	9.23 607	72	9.24 261	74	0.75 739	9.99 346	5
56	9.23 679	72	9.24 335	74	0.75 664	9.99 344	4
57	9.23 751	72	9.24 409	74	0.75 590	9.99 342	3
58	9.23 823	72	9.24 484	74	0.75 516	9.99 339	2
59	9.23 895	72	9.24 558	74	0.75 442	9.99 337	1
60	9.23 967	71	9.24 632	74	0.75 368	9.99 335	0
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.	

P. P.

	81	81	80	79
6	8.1	8.1	8.0	7.9
7	9.5	9.4	9.3	9.2
8	10.8	10.8	10.6	10.5
9	12.2	12.1	12.0	11.8
10	13.6	13.5	13.3	13.1
20	27.1	27.0	26.6	26.3
30	40.7	40.5	40.0	39.5
40	54.3	54.0	53.3	52.6
50	67.9	67.5	66.6	65.8

	78	78	77
6	7.8	7.8	7.7
7	9.1	9.1	9.0
8	10.4	10.4	10.2
9	11.8	11.7	11.5
10	13.1	13.0	12.8
20	26.1	26.0	25.6
30	39.2	39.0	38.5
40	52.3	52.0	51.3
50	65.4	65.0	64.1

	76	76	75	74
6	7.6	7.6	7.5	7.4
7	8.9	8.8	8.7	8.6
8	10.2	10.1	10.0	9.8
9	11.5	11.4	11.2	11.1
10	12.7	12.6	12.5	12.3
20	25.5	25.3	25.0	24.6
30	38.2	38.0	37.5	37.0
40	51.0	50.6	50.0	49.4
50	63.7	63.3	62.5	61.6

	73	73	72
6	7.3	7.3	7.2
7	8.6	8.5	8.4
8	9.8	9.7	9.6
9	11.0	10.9	10.8
10	12.2	12.1	12.0
20	24.5	24.3	24.0
30	36.7	36.5	36.0
40	49.0	48.6	48.0
50	61.2	60.8	60.0

	71	71	70	69
6	7.1	7.1	7.0	6.9
7	8.3	8.3	8.2	8.1
8	9.5	9.4	9.3	9.2
9	10.7	10.6	10.4	10.3
10	11.9	11.8	11.6	11.5
20	23.8	23.6	23.3	23.0
30	35.7	35.5	35.1	34.7
40	47.6	47.3	46.8	46.3
50	59.6	59.1	58.4	57.7

P. P.

99°

614.

80°

**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

10°

169°

	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.		P. P.			
0	9.23 967		9.24 632		0.75 368	9.99 335	60		74	73	73
1	9.24 038	71	9.24 705	73	0.75 294	9.99 333	59	6	7.4	7.3	7.3
2	9.24 110	71	9.24 779	74	0.75 220	9.99 330	58	7	8.6	8.6	8.5
3	9.24 181	71	9.24 853	73	0.75 147	9.99 328	57	8	9.8	9.8	9.7
4	9.24 252	71	9.24 926	73	0.75 073	9.99 326	56	9	11.1	11.0	10.9
5	9.24 323	71	9.25 000	73	0.75 000	9.99 324	55	10	12.3	12.2	12.1
6	9.24 394	71	9.25 073	73	0.74 927	9.99 321	54	20	24.6	24.5	24.3
7	9.24 465	71	9.25 146	73	0.74 854	9.99 319	53	30	37.0	36.7	36.5
8	9.24 536	70	9.25 219	73	0.74 781	9.99 317	52	40	49.3	49.0	48.6
9	9.24 607	70	9.25 292	73	0.74 708	9.99 315	51	50	61.6	61.2	60.8
10	9.24 677	70	9.25 365	72	0.74 635	9.99 312	50		72	72	71
11	9.24 748	70	9.25 437	72	0.74 562	9.99 310	49	6	7.2	7.2	7.1
12	9.24 818	70	9.25 510	72	0.74 490	9.99 308	48	7	8.4	8.4	8.3
13	9.24 888	70	9.25 582	72	0.74 417	9.99 306	47	8	9.6	9.6	9.5
14	9.24 958	69	9.25 654	72	0.74 345	9.99 303	46	9	10.9	10.8	10.7
15	9.25 028	70	9.25 727	72	0.74 273	9.99 301	45	10	12.1	12.0	11.9
16	9.25 098	69	9.25 799	72	0.74 201	9.99 299	44	20	24.1	24.0	23.8
17	9.25 167	70	9.25 871	72	0.74 129	9.99 296	43	30	36.2	36.0	35.7
18	9.25 237	69	9.25 943	71	0.74 057	9.99 294	42	40	48.3	48.0	47.6
19	9.25 306	69	9.26 014	72	0.73 985	9.99 292	41	50	60.4	60.0	59.6
20	9.25 376	69	9.26 086	71	0.73 913	9.99 290	40		70	70	69
21	9.25 445	69	9.26 158	71	0.73 842	9.99 287	39	6	7.0	7.0	6.9
22	9.25 514	69	9.26 229	71	0.73 771	9.99 285	38	7	8.2	8.1	8.0
23	9.25 583	69	9.26 300	71	0.73 699	9.99 283	37	8	9.4	9.3	9.2
24	9.25 652	68	9.26 371	71	0.73 628	9.99 280	36	9	10.6	10.5	10.4
25	9.25 721	69	9.26 443	71	0.73 557	9.99 278	35	10	11.7	11.6	11.5
26	9.25 790	68	9.26 514	70	0.73 486	9.99 276	34	20	23.5	23.3	23.1
27	9.25 858	68	9.26 584	71	0.73 415	9.99 273	33	30	35.2	35.0	34.7
28	9.25 927	68	9.26 655	70	0.73 344	9.99 271	32	40	47.0	46.6	46.3
29	9.25 995	68	9.26 726	70	0.73 274	9.99 269	31	50	58.7	58.3	57.9
30	9.26 063	68	9.26 796	70	0.73 203	9.99 266	30		68	68	67
31	9.26 131	68	9.26 867	70	0.73 133	9.99 264	29	6	6.8	6.8	6.7
32	9.26 199	68	9.26 937	70	0.73 062	9.99 262	28	7	8.0	7.9	7.8
33	9.26 267	68	9.27 007	70	0.72 992	9.99 259	27	8	9.1	9.0	8.9
34	9.26 335	67	9.27 078	70	0.72 922	9.99 257	26	9	10.3	10.2	10.1
35	9.26 402	67	9.27 148	70	0.72 852	9.99 255	25	10	11.4	11.3	11.2
36	9.26 470	67	9.27 218	69	0.72 782	9.99 252	24	20	22.8	22.6	22.5
37	9.26 537	67	9.27 287	70	0.72 712	9.99 250	23	30	34.2	34.0	33.7
38	9.26 605	67	9.27 357	69	0.72 642	9.99 248	22	40	45.6	45.3	45.0
39	9.26 672	67	9.27 427	69	0.72 573	9.99 245	21	50	57.1	56.6	56.2
40	9.26 739	67	9.27 496	69	0.72 503	9.99 243	20		66	66	65
41	9.26 806	67	9.27 566	69	0.72 434	9.99 240	19	6	6.6	6.6	6.5
42	9.26 873	66	9.27 635	69	0.72 365	9.99 238	18	7	7.7	7.7	7.6
43	9.26 940	67	9.27 704	69	0.72 295	9.99 236	17	8	8.8	8.8	8.7
44	9.27 007	66	9.27 773	69	0.72 226	9.99 233	16	9	10.0	9.9	9.8
45	9.27 073	66	9.27 842	69	0.72 157	9.99 231	15	10	11.1	11.0	10.9
46	9.27 140	66	9.27 911	68	0.72 088	9.99 228	14	20	22.1	22.0	21.8
47	9.27 206	66	9.27 980	69	0.72 020	9.99 226	13	30	33.2	33.0	32.7
48	9.27 272	66	9.28 049	68	0.71 951	9.99 224	12	40	44.3	44.0	43.6
49	9.27 339	66	9.28 117	68	0.71 882	9.99 221	11	50	55.4	55.0	54.6
50	9.27 405	66	9.28 186	68	0.71 814	9.99 219	10				
51	9.27 471	65	9.28 254	68	0.71 746	9.99 216	9				
52	9.27 536	66	9.28 322	68	0.71 677	9.99 214	8				
53	9.27 602	65	9.28 390	68	0.71 609	9.99 212	7				
54	9.27 668	65	9.28 459	68	0.71 541	9.99 209	6				
55	9.27 733	65	9.28 527	68	0.71 473	9.99 207	5				
56	9.27 799	65	9.28 594	68	0.71 405	9.99 204	4				
57	9.27 864	65	9.28 662	68	0.71 337	9.99 202	3				
58	9.27 929	65	9.28 730	67	0.71 270	9.99 199	2				
59	9.27 995	65	9.28 797	67	0.71 202	9.99 197	1				
60	9.28 060	65	9.28 865	67	0.71 135	9.99 194	0				
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.		P. P.			

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	P. P.
0	9.28 060		9.28 865		0.71 135	9.99 194	60
1	9.28 125	65	9.28 932	67	0.71 067	9.99 192	59
2	9.28 189	64	9.29 000	67	0.71 000	9.99 189	58
3	9.28 254	65	9.29 067	67	0.70 933	9.99 187	57
4	9.28 319	64	9.29 134	67	0.70 866	9.99 185	56
5	9.28 383	64	9.29 201	67	0.70 798	9.99 182	55
6	9.28 448	64	9.29 268	66	0.70 732	9.99 180	54
7	9.28 512	64	9.29 335	67	0.70 665	9.99 177	53
8	9.28 576	64	9.29 401	66	0.70 598	9.99 175	52
9	9.28 641	64	9.29 468	67	0.70 531	9.99 172	51
10	9.28 705	64	9.29 535	66	0.70 465	9.99 170	50
11	9.28 769	63	9.29 601	66	0.70 398	9.99 167	49
12	9.28 832	63	9.29 667	66	0.70 332	9.99 165	48
13	9.28 896	64	9.29 734	66	0.70 266	9.99 162	47
14	9.28 960	63	9.29 800	66	0.70 200	9.99 160	46
15	9.29 023	63	9.29 866	66	0.70 134	9.99 157	45
16	9.29 087	63	9.29 932	66	0.70 068	9.99 155	44
17	9.29 150	63	9.29 998	66	0.70 002	9.99 152	43
18	9.29 213	63	9.30 064	65	0.69 936	9.99 150	42
19	9.29 277	63	9.30 129	65	0.69 870	9.99 147	41
20	9.29 340	63	9.30 195	65	0.69 805	9.99 145	40
21	9.29 403	63	9.30 260	65	0.69 739	9.99 142	39
22	9.29 466	62	9.30 326	65	0.69 674	9.99 139	38
23	9.29 528	63	9.30 391	65	0.69 608	9.99 137	37
24	9.29 591	62	9.30 456	65	0.69 543	9.99 134	36
25	9.29 654	62	9.30 522	65	0.69 478	9.99 132	35
26	9.29 716	62	9.30 587	65	0.69 413	9.99 129	34
27	9.29 779	62	9.30 652	65	0.69 348	9.99 127	33
28	9.29 841	62	9.30 717	64	0.69 283	9.99 124	32
29	9.29 903	62	9.30 781	64	0.69 218	9.99 122	31
30	9.29 965	62	9.30 846	65	0.69 153	9.99 119	30
31	9.30 027	62	9.30 911	64	0.69 089	9.99 116	29
32	9.30 089	62	9.30 975	64	0.69 024	9.99 114	28
33	9.30 151	61	9.31 040	64	0.68 960	9.99 111	27
34	9.30 213	62	9.31 104	64	0.68 896	9.99 109	26
35	9.30 275	61	9.31 168	64	0.68 831	9.99 106	25
36	9.30 336	61	9.31 232	64	0.68 767	9.99 104	24
37	9.30 398	61	9.31 297	64	0.68 703	9.99 101	23
38	9.30 459	61	9.31 361	63	0.68 639	9.99 098	22
39	9.30 520	61	9.31 424	63	0.68 575	9.99 096	21
40	9.30 582	61	9.31 488	64	0.68 511	9.99 093	20
41	9.30 643	61	9.31 552	64	0.68 447	9.99 091	19
42	9.30 704	61	9.31 616	63	0.68 384	9.99 088	18
43	9.30 765	61	9.31 679	63	0.68 320	9.99 085	17
44	9.30 826	60	9.31 743	63	0.68 257	9.99 083	16
45	9.30 886	61	9.31 806	63	0.68 193	9.99 080	15
46	9.30 947	60	9.31 869	63	0.68 130	9.99 077	14
47	9.31 008	60	9.31 933	63	0.68 067	9.99 075	13
48	9.31 068	60	9.31 996	63	0.68 004	9.99 072	12
49	9.31 129	60	9.32 059	63	0.67 941	9.99 069	11
50	9.31 189	60	9.32 122	63	0.67 878	9.99 067	10
51	9.31 249	60	9.32 185	63	0.67 815	9.99 064	9
52	9.31 309	60	9.32 248	62	0.67 752	9.99 062	8
53	9.31 370	60	9.32 310	63	0.67 689	9.99 059	7
54	9.31 429	60	9.32 373	62	0.67 626	9.99 056	6
55	9.31 489	60	9.32 436	62	0.67 564	9.99 054	5
56	9.31 549	59	9.32 498	62	0.67 501	9.99 051	4
57	9.31 609	59	9.32 560	62	0.67 439	9.99 048	3
58	9.31 669	59	9.32 623	62	0.67 377	9.99 046	2
59	9.31 728	59	9.32 685	62	0.67 314	9.99 043	1
60	9.31 788	59	9.32 747	62	0.67 252	9.99 040	0
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	

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'	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.		P. P.
0	9.31 788	59	9.32 747	62	0.67 252	9.99 040	60	
1	9.31 847	59	9.32 809	62	0.67 190	9.99 038	59	
2	9.31 906	59	9.32 871	62	0.67 128	9.99 035	58	
3	9.31 966	59	9.32 933	62	0.67 066	9.99 032	57	
4	9.32 025	59	9.32 995	61	0.67 004	9.99 029	56	
5	9.32 084	59	9.33 057	61	0.66 943	9.99 027	55	
6	9.32 143	59	9.33 118	61	0.66 881	9.99 024	54	
7	9.32 202	59	9.33 180	61	0.66 819	9.99 021	53	
8	9.32 260	59	9.33 242	61	0.66 758	9.99 019	52	
9	9.32 319	58	9.33 303	61	0.66 696	9.99 016	51	
10	9.32 378	58	9.33 364	61	0.66 635	9.99 013	50	
11	9.32 436	58	9.33 426	61	0.66 574	9.99 010	49	
12	9.32 495	58	9.33 487	61	0.66 513	9.99 008	48	
13	9.32 553	58	9.33 548	61	0.66 452	9.99 005	47	
14	9.32 611	58	9.33 609	60	0.66 390	9.99 002	46	
15	9.32 670	58	9.33 670	61	0.66 330	9.98 999	45	
16	9.32 728	58	9.33 731	61	0.66 269	9.98 997	44	
17	9.32 786	58	9.33 792	60	0.66 208	9.98 994	43	
18	9.32 844	58	9.33 852	61	0.66 147	9.98 991	42	
19	9.32 902	58	9.33 913	60	0.66 086	9.98 988	41	
20	9.32 960	57	9.33 974	60	0.66 026	9.98 986	40	
21	9.33 017	57	9.34 034	60	0.65 965	9.98 983	39	
22	9.33 075	57	9.34 095	60	0.65 905	9.98 980	38	
23	9.33 133	57	9.34 155	60	0.65 845	9.98 977	37	
24	9.33 190	57	9.34 215	60	0.65 784	9.98 975	36	
25	9.33 248	57	9.34 275	60	0.65 724	9.98 972	35	
26	9.33 305	57	9.34 336	60	0.65 664	9.98 969	34	
27	9.33 362	57	9.34 396	60	0.65 604	9.98 966	33	
28	9.33 419	57	9.34 456	59	0.65 544	9.98 963	32	
29	9.33 476	57	9.34 515	60	0.65 484	9.98 961	31	
30	9.33 533	57	9.34 575	60	0.65 424	9.98 958	30	
31	9.33 590	57	9.34 635	59	0.65 364	9.98 955	29	
32	9.33 647	57	9.34 695	59	0.65 305	9.98 952	28	
33	9.33 704	56	9.34 754	59	0.65 245	9.98 949	27	
34	9.33 761	56	9.34 814	59	0.65 186	9.98 947	26	
35	9.33 817	56	9.34 873	59	0.65 126	9.98 944	25	
36	9.33 874	56	9.34 933	59	0.65 067	9.98 941	24	
37	9.33 930	56	9.34 992	59	0.65 008	9.98 938	23	
38	9.33 987	56	9.35 051	59	0.64 948	9.98 935	22	
39	9.34 043	56	9.35 110	59	0.64 889	9.98 933	21	
40	9.34 099	56	9.35 169	59	0.64 830	9.98 930	20	
41	9.34 156	56	9.35 228	59	0.64 771	9.98 927	19	
42	9.34 212	56	9.35 287	59	0.64 712	9.98 924	18	
43	9.34 268	56	9.35 346	59	0.64 653	9.98 921	17	
44	9.34 324	55	9.35 405	58	0.64 594	9.98 918	16	
45	9.34 379	55	9.35 464	58	0.64 536	9.98 915	15	
46	9.34 435	55	9.35 522	58	0.64 477	9.98 913	14	
47	9.34 491	55	9.35 581	58	0.64 418	9.98 910	13	
48	9.34 547	55	9.35 640	58	0.64 360	9.98 907	12	
49	9.34 602	55	9.35 698	58	0.64 302	9.98 904	11	
50	9.34 658	55	9.35 756	58	0.64 243	9.98 901	10	
51	9.34 713	55	9.35 815	58	0.64 185	9.98 898	9	
52	9.34 768	55	9.35 873	58	0.64 127	9.98 895	8	
53	9.34 824	55	9.35 931	58	0.64 068	9.98 892	7	
54	9.34 879	55	9.35 989	58	0.64 010	9.98 890	6	
55	9.34 934	55	9.36 047	58	0.63 952	9.98 887	5	
56	9.34 989	55	9.36 105	57	0.63 894	9.98 884	4	
57	9.35 044	54	9.36 163	58	0.63 837	9.98 881	3	
58	9.35 099	55	9.36 221	57	0.63 779	9.98 878	2	
59	9.35 154	55	9.36 278	57	0.63 721	9.98 875	1	
60	9.35 209	55	9.36 336	58	0.63 663	9.98 872	0	
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.		P. P.

62	61	61
6 6.2	6.1	6.1
7 7.2	7.2	7.1
8 8.2	8.2	8.1
9 9.3	9.2	9.1
10 10.3	10.2	10.1
20 20.6	20.5	20.3
30 31.0	30.7	30.5
40 41.3	41.0	40.6
50 51.6	51.2	50.8

60	60	59	59
6 6.0	6.0	5.9	5.9
7 7.0	7.0	6.9	6.9
8 8.0	8.0	7.9	7.8
9 9.1	9.0	8.9	8.8
10 10.1	10.0	9.9	9.8
20 20.1	20.0	19.8	19.6
30 30.2	30.0	29.7	29.5
40 40.3	40.0	39.6	39.3
50 50.4	50.0	49.6	49.1

58	58	57	57
6 5.8	5.8	5.7	5.7
7 6.8	6.7	6.7	6.6
8 7.8	7.7	7.6	7.6
9 8.8	8.7	8.6	8.5
10 9.7	9.6	9.6	9.5
20 19.5	19.3	19.1	19.0
30 29.2	29.0	28.7	28.5
40 39.0	38.6	38.3	38.0
50 48.7	48.3	47.9	47.5

56	56	55	55
6 5.6	5.6	5.5	5.5
7 6.6	6.5	6.5	6.4
8 7.5	7.4	7.4	7.3
9 8.5	8.4	8.3	8.2
10 9.4	9.3	9.2	9.1
20 18.8	18.6	18.5	18.3
30 28.2	28.0	27.7	27.5
40 37.6	37.3	37.0	36.6
50 47.1	46.6	46.2	45.8

54	3	2
6 5.4	0.3	0.2
7 6.3	0.3	0.3
8 7.2	0.4	0.3
9 8.2	0.4	0.4
10 9.1	0.5	0.4
20 18.1	1.0	0.8
30 27.2	1.5	1.2
40 36.3	2.0	1.6
50 45.4	2.5	2.1

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.		P. P.
0	9.35 209		9.36 336		0.63 663	9.98 872	60	
1	9.35 263	54	9.36 394	57	0.63 606	9.98 869	59	
2	9.35 318	54	9.36 451	57	0.63 548	9.98 866	58	
3	9.35 372	54	9.36 509	57	0.63 491	9.98 863	57	
4	9.35 427	54	9.36 566	57	0.63 433	9.98 860	56	
5	9.35 481	54	9.36 623	57	0.63 376	9.98 858	55	
6	9.35 536	54	9.36 681	57	0.63 319	9.98 855	54	
7	9.35 590	54	9.36 738	57	0.63 262	9.98 852	53	
8	9.35 644	54	9.36 795	57	0.63 204	9.98 849	52	
9	9.35 698	54	9.36 852	57	0.63 147	9.98 846	51	
10	9.35 752	54	9.36 909	57	0.63 090	9.98 843	50	
11	9.35 806	54	9.36 966	56	0.63 033	9.98 840	49	
12	9.35 860	53	9.37 023	57	0.62 977	9.98 837	48	
13	9.35 914	54	9.37 080	56	0.62 920	9.98 834	47	
14	9.35 968	53	9.37 136	57	0.62 863	9.98 831	46	
15	9.36 021	53	9.37 193	56	0.62 806	9.98 828	45	
16	9.36 075	53	9.37 250	56	0.62 750	9.98 825	44	
17	9.36 128	53	9.37 306	56	0.62 693	9.98 822	43	
18	9.36 182	53	9.37 363	56	0.62 637	9.98 819	42	
19	9.36 235	53	9.37 419	56	0.62 580	9.98 816	41	
20	9.36 289	53	9.37 475	56	0.62 524	9.98 813	40	
21	9.36 342	53	9.37 532	56	0.62 468	9.98 810	39	
22	9.36 395	53	9.37 588	56	0.62 412	9.98 807	38	
23	9.36 448	53	9.37 644	56	0.62 356	9.98 804	37	
24	9.36 501	53	9.37 700	56	0.62 299	9.98 801	36	
25	9.36 554	53	9.37 756	55	0.62 243	9.98 798	35	
26	9.36 607	53	9.37 812	56	0.62 188	9.98 795	34	
27	9.36 660	52	9.37 868	55	0.62 132	9.98 792	33	
28	9.36 713	53	9.37 924	55	0.62 076	9.98 789	32	
29	9.36 766	52	9.37 979	56	0.62 020	9.98 786	31	
30	9.36 818	52	9.38 035	55	0.61 964	9.98 783	30	
31	9.36 871	52	9.38 091	55	0.61 909	9.98 780	29	
32	9.36 923	52	9.38 146	55	0.61 853	9.98 777	28	
33	9.36 976	52	9.38 202	55	0.61 798	9.98 774	27	
34	9.37 028	52	9.38 257	55	0.61 742	9.98 771	26	
35	9.37 081	52	9.38 313	55	0.61 687	9.98 768	25	
36	9.37 133	52	9.38 368	55	0.61 632	9.98 765	24	
37	9.37 185	52	9.38 423	55	0.61 576	9.98 762	23	
38	9.37 237	52	9.38 478	55	0.61 521	9.98 759	22	
39	9.37 289	52	9.38 533	55	0.61 466	9.98 755	21	
40	9.37 341	52	9.38 589	55	0.61 411	9.98 752	20	
41	9.37 393	51	9.38 644	54	0.61 356	9.98 749	19	
42	9.37 445	52	9.38 698	55	0.61 301	9.98 746	18	
43	9.37 497	51	9.38 753	55	0.61 246	9.98 743	17	
44	9.37 548	52	9.38 808	54	0.61 191	9.98 740	16	
45	9.37 600	51	9.38 863	55	0.61 137	9.98 737	15	
46	9.37 652	51	9.38 918	54	0.61 082	9.98 734	14	
47	9.37 703	51	9.38 972	54	0.61 027	9.98 731	13	
48	9.37 755	51	9.39 027	54	0.60 973	9.98 728	12	
49	9.37 806	51	9.39 081	54	0.60 918	9.98 725	11	
50	9.37 857	51	9.39 136	54	0.60 864	9.98 721	10	
51	9.37 909	51	9.39 190	54	0.60 809	9.98 718	9	
52	9.37 960	51	9.39 244	54	0.60 755	9.98 715	8	
53	9.38 011	51	9.39 299	54	0.60 701	9.98 712	7	
54	9.38 062	51	9.39 353	54	0.60 647	9.98 709	6	
55	9.38 113	51	9.39 407	54	0.60 592	9.98 706	5	
56	9.38 164	50	9.39 461	54	0.60 538	9.98 703	4	
57	9.38 215	51	9.39 515	54	0.60 484	9.98 700	3	
58	9.38 266	51	9.39 569	54	0.60 430	9.98 696	2	
59	9.38 317	51	9.39 623	54	0.60 376	9.98 693	1	
60	9.38 367	50	9.39 677	53	0.60 322	9.98 690	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.		P. P.

	57	57	56	56
6	5.7	5.7	5.6	5.6
7	6.7	6.6	6.6	6.5
8	7.6	7.6	7.5	7.4
9	8.6	8.5	8.5	8.4
10	9.6	9.5	9.4	9.3
20	19.1	19.0	18.8	18.6
30	28.7	28.5	28.2	28.0
40	38.3	38.0	37.6	37.3
50	47.9	47.5	47.1	46.6

	55	55	54	54
6	5.5	5.5	5.4	5.4
7	6.5	6.4	6.3	6.3
8	7.4	7.3	7.2	7.2
9	8.3	8.2	8.2	8.1
10	9.2	9.1	9.1	9.0
20	18.5	18.3	18.1	18.0
30	27.7	27.5	27.2	27.0
40	37.0	36.6	36.3	36.0
50	46.2	45.8	45.4	45.0

	53	53	52	52
6	5.3	5.3	5.2	5.2
7	6.2	6.2	6.1	6.0
8	7.1	7.0	7.0	6.9
9	8.0	7.9	7.9	7.8
10	8.9	8.8	8.7	8.6
20	17.8	17.6	17.5	17.3
30	26.7	26.5	26.2	26.0
40	35.6	35.3	35.0	34.6
50	44.6	44.1	43.7	43.3

	51	51	50
6	5.1	5.1	5.0
7	6.0	5.9	5.9
8	6.8	6.8	6.7
9	7.7	7.6	7.6
10	8.6	8.5	8.4
20	17.1	17.0	16.8
30	25.7	25.5	25.2
40	34.3	34.0	33.6
50	42.9	42.5	42.1

	3	3	2
6	0.3	0.3	0.2
7	0.4	0.3	0.3
8	0.4	0.4	0.3
9	0.5	0.4	0.4
10	0.6	0.5	0.4
20	1.1	1.0	0.8
30	1.7	1.5	1.2
40	2.3	2.0	1.6
50	2.9	2.5	2.1

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS, AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.38 367		9.39 677		0.60 323	9.98 690		60	
1	9.38 418	50	9.39 731	54	0.60 269	9.98 687	50	59	
2	9.38 468	50	9.39 784	54	0.60 215	9.98 684	50	58	
3	9.38 519	50	9.39 838	54	0.60 161	9.98 681	50	57	
4	9.38 569	50	9.39 892	53	0.60 108	9.98 678	50	56	
5	9.38 620	50	9.39 945	53	0.60 054	9.98 674	50	55	
6	9.38 670	50	9.39 999	53	0.60 001	9.98 671	50	54	
7	9.38 720	50	9.40 052	53	0.59 947	9.98 668	50	53	
8	9.38 771	50	9.40 106	53	0.59 894	9.98 665	50	52	
9	9.38 821	50	9.40 159	53	0.59 841	9.98 662	50	51	
10	9.38 871	50	9.40 212	53	0.59 787	9.98 658	50	50	
11	9.38 921	50	9.40 265	53	0.59 734	9.98 655	50	49	
12	9.38 971	50	9.40 318	53	0.59 681	9.98 652	50	48	
13	9.39 021	50	9.40 372	53	0.59 628	9.98 649	50	47	
14	9.39 071	50	9.40 425	53	0.59 575	9.98 646	50	46	
15	9.39 120	49	9.40 478	53	0.59 522	9.98 642	50	45	
16	9.39 170	49	9.40 531	52	0.59 469	9.98 639	50	44	
17	9.39 220	49	9.40 583	52	0.59 416	9.98 636	50	43	
18	9.39 269	49	9.40 636	52	0.59 363	9.98 633	50	42	
19	9.39 319	49	9.40 689	52	0.59 311	9.98 630	50	41	
20	9.39 368	49	9.40 742	52	0.59 258	9.98 626	50	40	
21	9.39 418	49	9.40 794	52	0.59 205	9.98 623	50	39	
22	9.39 467	49	9.40 847	52	0.59 153	9.98 620	50	38	
23	9.39 516	49	9.40 899	52	0.59 100	9.98 617	50	37	
24	9.39 566	49	9.40 952	52	0.59 048	9.98 613	50	36	
25	9.39 615	49	9.41 004	52	0.58 995	9.98 610	50	35	
26	9.39 664	49	9.41 057	52	0.58 943	9.98 607	50	34	
27	9.39 713	49	9.41 109	52	0.58 891	9.98 604	50	33	
28	9.39 762	49	9.41 161	52	0.58 838	9.98 600	50	32	
29	9.39 811	49	9.41 213	52	0.58 786	9.98 597	50	31	
30	9.39 860	49	9.41 266	52	0.58 734	9.98 594	50	30	
31	9.39 909	48	9.41 318	52	0.58 682	9.98 591	50	29	
32	9.39 957	48	9.41 370	52	0.58 630	9.98 587	50	28	
33	9.40 006	48	9.41 422	52	0.58 578	9.98 584	50	27	
34	9.40 055	48	9.41 474	51	0.58 526	9.98 581	50	26	
35	9.40 103	48	9.41 525	52	0.58 474	9.98 578	50	25	
36	9.40 152	48	9.41 577	52	0.58 422	9.98 574	50	24	
37	9.40 200	48	9.41 629	51	0.58 370	9.98 571	50	23	
38	9.40 249	48	9.41 681	51	0.58 319	9.98 568	50	22	
39	9.40 297	48	9.41 732	51	0.58 267	9.98 564	50	21	
40	9.40 345	48	9.41 784	51	0.58 216	9.98 561	50	20	
41	9.40 394	48	9.41 836	51	0.58 164	9.98 558	50	19	
42	9.40 442	48	9.41 887	51	0.58 112	9.98 554	50	18	
43	9.40 490	48	9.41 938	51	0.58 061	9.98 551	50	17	
44	9.40 538	48	9.41 990	51	0.58 010	9.98 548	50	16	
45	9.40 586	48	9.42 041	51	0.57 958	9.98 544	50	15	
46	9.40 634	48	9.42 092	51	0.57 907	9.98 541	50	14	
47	9.40 682	48	9.42 144	51	0.57 856	9.98 538	50	13	
48	9.40 730	47	9.42 195	51	0.57 805	9.98 534	50	12	
49	9.40 777	47	9.42 246	51	0.57 753	9.98 531	50	11	
50	9.40 825	47	9.42 297	51	0.57 702	9.98 528	50	10	
51	9.40 873	47	9.42 348	51	0.57 651	9.98 524	50	9	
52	9.40 920	47	9.42 399	51	0.57 600	9.98 521	50	8	
53	9.40 968	47	9.42 450	50	0.57 549	9.98 518	50	7	
54	9.41 015	47	9.42 501	50	0.57 499	9.98 514	50	6	
55	9.41 063	47	9.42 552	50	0.57 448	9.98 511	50	5	
56	9.41 110	47	9.42 602	50	0.57 397	9.98 508	50	4	
57	9.41 158	47	9.42 653	50	0.57 346	9.98 504	50	3	
58	9.41 205	47	9.42 704	50	0.57 296	9.98 501	50	2	
59	9.41 252	47	9.42 754	50	0.57 245	9.98 498	50	1	
60	9.41 299	47	9.42 805	50	0.57 195	9.98 494	50	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

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**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.41 299		9.42 805	51	0.57 195	9.98 494	60	
1	9.41 346	47	9.42 856	50	0.57 144	9.98 491	59	
2	9.41 394	47	9.42 906	50	0.57 094	9.98 487	58	
3	9.41 441	47	9.42 956	50	0.57 043	9.98 484	57	
4	9.41 488	47	9.43 007	50	0.56 993	9.98 481	56	
5	9.41 534	46	9.43 057	50	0.56 942	9.98 477	55	
6	9.41 581	47	9.43 107	50	0.56 892	9.98 474	54	
7	9.41 628	47	9.43 157	50	0.56 842	9.98 470	53	
8	9.41 675	46	9.43 208	50	0.56 792	9.98 467	52	
9	9.41 721	46	9.43 258	50	0.56 742	9.98 464	51	
10	9.41 768	47	9.43 308	50	0.56 692	9.98 460	50	
11	9.41 815	46	9.43 358	50	0.56 642	9.98 457	49	
12	9.41 861	46	9.43 408	50	0.56 592	9.98 453	48	
13	9.41 908	46	9.43 458	50	0.56 542	9.98 450	47	
14	9.41 954	46	9.43 508	50	0.56 492	9.98 446	46	
15	9.42 000	46	9.43 557	49	0.56 442	9.98 443	45	
16	9.42 047	46	9.43 607	49	0.56 392	9.98 439	44	
17	9.42 093	46	9.43 657	49	0.56 343	9.98 436	43	
18	9.42 139	46	9.43 706	50	0.56 293	9.98 433	42	
19	9.42 185	46	9.43 756	49	0.56 243	9.98 429	41	
20	9.42 232	46	9.43 806	49	0.56 194	9.98 426	40	
21	9.42 278	46	9.43 855	49	0.56 144	9.98 422	39	
22	9.42 324	45	9.43 905	49	0.56 095	9.98 419	38	
23	9.42 369	46	9.43 954	49	0.56 045	9.98 415	37	
24	9.42 415	46	9.44 003	49	0.55 996	9.98 412	36	
25	9.42 461	46	9.44 053	49	0.55 947	9.98 408	35	
26	9.42 507	45	9.44 102	49	0.55 898	9.98 405	34	
27	9.42 553	45	9.44 151	49	0.55 848	9.98 401	33	
28	9.42 598	46	9.44 200	49	0.55 799	9.98 398	32	
29	9.42 644	45	9.44 249	49	0.55 750	9.98 394	31	
30	9.42 690	45	9.44 299	49	0.55 701	9.98 391	30	
31	9.42 735	45	9.44 348	49	0.55 652	9.98 387	29	
32	9.42 781	45	9.44 397	49	0.55 603	9.98 384	28	
33	9.42 826	45	9.44 446	48	0.55 554	9.98 380	27	
34	9.42 871	45	9.44 494	48	0.55 505	9.98 377	26	
35	9.42 917	45	9.44 543	49	0.55 456	9.98 373	25	
36	9.42 962	45	9.44 592	48	0.55 407	9.98 370	24	
37	9.43 007	45	9.44 641	49	0.55 359	9.98 366	23	
38	9.43 052	45	9.44 690	48	0.55 310	9.98 363	22	
39	9.43 098	45	9.44 738	48	0.55 261	9.98 359	21	
40	9.43 143	45	9.44 787	48	0.55 213	9.98 356	20	
41	9.43 188	45	9.44 835	48	0.55 164	9.98 352	19	
42	9.43 233	45	9.44 884	48	0.55 116	9.98 348	18	
43	9.43 278	44	9.44 932	48	0.55 067	9.98 345	17	
44	9.43 322	45	9.44 981	48	0.55 019	9.98 341	16	
45	9.43 367	44	9.45 029	48	0.54 970	9.98 338	15	
46	9.43 412	45	9.45 077	48	0.54 922	9.98 334	14	
47	9.43 457	44	9.45 126	48	0.54 874	9.98 331	13	
48	9.43 501	44	9.45 174	48	0.54 825	9.98 327	12	
49	9.43 546	44	9.45 222	48	0.54 777	9.98 324	11	
50	9.43 591	44	9.45 270	48	0.54 729	9.98 320	10	
51	9.43 635	44	9.45 318	48	0.54 681	9.98 316	9	
52	9.43 680	44	9.45 367	48	0.54 633	9.98 313	8	
53	9.43 724	44	9.45 415	48	0.54 585	9.98 309	7	
54	9.43 768	44	9.45 463	48	0.54 537	9.98 306	6	
55	9.43 813	44	9.45 510	47	0.54 489	9.98 302	5	
56	9.43 857	44	9.45 558	48	0.54 441	9.98 298	4	
57	9.43 901	44	9.45 606	47	0.54 393	9.98 295	3	
58	9.43 945	44	9.45 654	48	0.54 346	9.98 291	2	
59	9.43 989	44	9.45 702	48	0.54 298	9.98 288	1	
60	9.44 034	44	9.45 749	47	0.54 250	9.98 284	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.44 034	44	9.45 749	48	0.54 250	9.98 284	60	
1	9.44 078	44	9.45 797	47	0.54 202	9.98 280	59	
2	9.44 122	44	9.45 845	47	0.54 155	9.98 277	58	
3	9.44 166	43	9.45 892	47	0.54 107	9.98 273	57	
4	9.44 209	43	9.45 940	47	0.54 060	9.98 269	56	
5	9.44 253	44	9.45 987	47	0.54 012	9.98 266	55	
6	9.44 297	43	9.46 035	47	0.53 965	9.98 262	54	
7	9.44 341	43	9.46 082	47	0.53 917	9.98 258	53	
8	9.44 384	44	9.46 129	47	0.53 870	9.98 255	52	
9	9.44 428	43	9.46 177	47	0.53 823	9.98 251	51	
10	9.44 472	43	9.46 224	47	0.53 776	9.98 247	50	
11	9.44 515	43	9.46 271	47	0.53 728	9.98 244	49	
12	9.44 559	43	9.46 318	47	0.53 681	9.98 240	48	
13	9.44 602	43	9.46 366	47	0.53 634	9.98 236	47	
14	9.44 646	43	9.46 413	47	0.53 587	9.98 233	46	
15	9.44 689	43	9.46 460	47	0.53 540	9.98 229	45	
16	9.44 732	43	9.46 507	47	0.53 493	9.98 225	44	
17	9.44 776	43	9.46 554	47	0.53 446	9.98 222	43	
18	9.44 819	43	9.46 601	46	0.53 399	9.98 218	42	
19	9.44 862	43	9.46 647	47	0.53 352	9.98 214	41	
20	9.44 905	43	9.46 694	47	0.53 305	9.98 211	40	
21	9.44 948	43	9.46 741	46	0.53 258	9.98 207	39	
22	9.44 991	43	9.46 788	46	0.53 212	9.98 203	38	
23	9.45 034	43	9.46 834	47	0.53 165	9.98 200	37	
24	9.45 077	43	9.46 881	46	0.53 118	9.98 196	36	
25	9.45 120	42	9.46 928	46	0.53 072	9.98 192	35	
26	9.45 163	43	9.46 974	46	0.53 025	9.98 188	34	
27	9.45 206	43	9.47 021	46	0.52 979	9.98 185	33	
28	9.45 249	42	9.47 067	46	0.52 932	9.98 181	32	
29	9.45 291	42	9.47 114	46	0.52 886	9.98 177	31	
30	9.45 334	43	9.47 160	46	0.52 839	9.98 173	30	
31	9.45 377	42	9.47 207	46	0.52 793	9.98 170	29	
32	9.45 419	42	9.47 253	46	0.52 747	9.98 166	28	
33	9.45 462	42	9.47 299	46	0.52 700	9.98 162	27	
34	9.45 504	42	9.47 345	46	0.52 654	9.98 158	26	
35	9.45 547	42	9.47 392	46	0.52 608	9.98 155	25	
36	9.45 589	42	9.47 438	46	0.52 562	9.98 151	24	
37	9.45 631	42	9.47 484	46	0.52 516	9.98 147	23	
38	9.45 674	42	9.47 530	46	0.52 469	9.98 143	22	
39	9.45 716	42	9.47 576	46	0.52 423	9.98 140	21	
40	9.45 758	42	9.47 622	46	0.52 377	9.98 136	20	
41	9.45 800	42	9.47 668	45	0.52 331	9.98 132	19	
42	9.45 842	42	9.47 714	46	0.52 286	9.98 128	18	
43	9.45 885	42	9.47 760	46	0.52 240	9.98 124	17	
44	9.45 927	42	9.47 806	45	0.52 194	9.98 121	16	
45	9.45 969	42	9.47 851	46	0.52 148	9.98 117	15	
46	9.46 011	41	9.47 897	45	0.52 102	9.98 113	14	
47	9.46 052	42	9.47 943	46	0.52 057	9.98 109	13	
48	9.46 094	42	9.47 989	45	0.52 011	9.98 105	12	
49	9.46 136	41	9.48 034	45	0.51 965	9.98 102	11	
50	9.46 178	42	9.48 080	45	0.51 920	9.98 098	10	
51	9.46 220	41	9.48 125	45	0.51 874	9.98 094	9	
52	9.46 261	41	9.48 171	45	0.51 829	9.98 090	8	
53	9.46 303	42	9.48 216	45	0.51 783	9.98 086	7	
54	9.46 345	41	9.48 262	45	0.51 738	9.98 082	6	
55	9.46 386	41	9.48 307	45	0.51 692	9.98 079	5	
56	9.46 428	41	9.48 353	45	0.51 647	9.98 075	4	
57	9.46 469	41	9.48 398	45	0.51 602	9.98 071	3	
58	9.46 511	41	9.48 443	45	0.51 556	9.98 067	2	
59	9.46 552	41	9.48 488	45	0.51 511	9.98 063	1	
60	9.46 593	41	9.48 534	45	0.51 466	9.98 059	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.46 593		9.48 534		0.51 466	9.98 059	60	
1	9.46 635	41	9.48 579	45	0.51 421	9.98 056	59	
2	9.46 676	41	9.48 624	45	0.51 376	9.98 052	58	
3	9.46 717	41	9.48 669	45	0.51 330	9.98 048	57	
4	9.46 758	41	9.48 714	45	0.51 285	9.98 044	56	
5	9.46 799	41	9.48 759	45	0.51 240	9.98 040	55	
6	9.46 840	41	9.48 804	44	0.51 195	9.98 036	54	
7	9.46 881	41	9.48 849	44	0.51 151	9.98 032	53	
8	9.46 922	41	9.48 894	45	0.51 106	9.98 028	52	
9	9.46 963	41	9.48 939	45	0.51 061	9.98 024	51	
10	9.47 004	41	9.48 984	45	0.51 016	9.98 021	50	
11	9.47 045	41	9.49 028	44	0.50 971	9.98 017	49	
12	9.47 086	40	9.49 073	44	0.50 926	9.98 013	48	
13	9.47 127	41	9.49 118	44	0.50 882	9.98 009	47	
14	9.47 168	40	9.49 162	44	0.50 837	9.98 005	46	
15	9.47 208	40	9.49 207	45	0.50 792	9.98 001	45	
16	9.47 249	41	9.49 252	44	0.50 748	9.97 997	44	
17	9.47 290	40	9.49 296	44	0.50 703	9.97 993	43	
18	9.47 330	40	9.49 341	44	0.50 659	9.97 989	42	
19	9.47 371	40	9.49 385	44	0.50 614	9.97 985	41	
20	9.47 411	40	9.49 430	44	0.50 570	9.97 981	40	
21	9.47 452	40	9.49 474	44	0.50 525	9.97 977	39	
22	9.47 492	40	9.49 518	44	0.50 481	9.97 973	38	
23	9.47 532	40	9.49 563	44	0.50 437	9.97 969	37	
24	9.47 573	40	9.49 607	44	0.50 392	9.97 966	36	
25	9.47 613	40	9.49 651	44	0.50 348	9.97 962	35	
26	9.47 653	40	9.49 695	44	0.50 304	9.97 958	34	
27	9.47 694	40	9.49 740	44	0.50 260	9.97 954	33	
28	9.47 734	40	9.49 784	44	0.50 216	9.97 950	32	
29	9.47 774	40	9.49 828	44	0.50 172	9.97 946	31	
30	9.47 814	40	9.49 872	44	0.50 128	9.97 942	30	
31	9.47 854	40	9.49 916	44	0.50 083	9.97 938	29	
32	9.47 894	40	9.49 960	43	0.50 039	9.97 934	28	
33	9.47 934	40	9.50 004	44	0.49 996	9.97 930	27	
34	9.47 974	40	9.50 048	44	0.49 952	9.97 926	26	
35	9.48 014	40	9.50 092	44	0.49 908	9.97 922	25	
36	9.48 054	39	9.50 136	43	0.49 864	9.97 918	24	
37	9.48 093	39	9.50 179	44	0.49 820	9.97 914	23	
38	9.48 133	39	9.50 223	43	0.49 776	9.97 910	22	
39	9.48 173	39	9.50 267	43	0.49 733	9.97 906	21	
40	9.48 213	39	9.50 311	43	0.49 689	9.97 902	20	
41	9.48 252	39	9.50 354	43	0.49 645	9.97 898	19	
42	9.48 292	39	9.50 398	43	0.49 602	9.97 894	18	
43	9.48 331	39	9.50 442	43	0.49 558	9.97 890	17	
44	9.48 371	39	9.50 485	43	0.49 514	9.97 886	16	
45	9.48 410	39	9.50 529	43	0.49 471	9.97 881	15	
46	9.48 450	39	9.50 572	43	0.49 427	9.97 877	14	
47	9.48 489	39	9.50 616	43	0.49 384	9.97 873	13	
48	9.48 529	39	9.50 659	43	0.49 340	9.97 869	12	
49	9.48 568	39	9.50 702	43	0.49 297	9.97 865	11	
50	9.48 607	39	9.50 746	43	0.49 254	9.97 861	10	
51	9.48 646	39	9.50 789	43	0.49 210	9.97 857	9	
52	9.48 686	39	9.50 832	43	0.49 167	9.97 853	8	
53	9.48 725	39	9.50 876	43	0.49 124	9.97 849	7	
54	9.48 764	39	9.50 919	43	0.49 081	9.97 845	6	
55	9.48 803	39	9.50 962	43	0.49 038	9.97 841	5	
56	9.48 842	39	9.51 005	43	0.48 994	9.97 837	4	
57	9.48 881	39	9.51 048	43	0.48 951	9.97 833	3	
58	9.48 920	39	9.51 091	43	0.48 908	9.97 829	2	
59	9.48 959	39	9.51 134	43	0.48 865	9.97 824	1	
60	9.48 998	38	9.51 177	43	0.48 822	9.97 820	0	
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.	d.	P. P.

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.48 998		9.51 177		0.48 822	9.97 820		60	
1	9.49 037	39	9.51 220	43	0.48 779	9.97 816	4	59	
2	9.49 076	39	9.51 263	43	0.48 736	9.97 812	4	58	
3	9.49 114	38	9.51 306	43	0.48 693	9.97 808	4	57	
4	9.49 153	39	9.51 349	43	0.48 650	9.97 804	4	56	
5	9.49 192	38	9.51 392	42	0.48 608	9.97 800	4	55	
6	9.49 231	39	9.51 435	43	0.48 565	9.97 796	4	54	
7	9.49 269	38	9.51 477	42	0.48 522	9.97 792	4	53	
8	9.49 308	38	9.51 520	43	0.48 479	9.97 787	4	52	
9	9.49 346	38	9.51 563	42	0.48 437	9.97 783	4	51	
10	9.49 385	38	9.51 605	42	0.48 394	9.97 779	4	50	
11	9.49 423	38	9.51 648	43	0.48 351	9.97 775	4	49	
12	9.49 462	38	9.51 691	42	0.48 309	9.97 771	4	48	
13	9.49 500	38	9.51 733	42	0.48 266	9.97 767	4	47	
14	9.49 539	38	9.51 776	42	0.48 224	9.97 763	4	46	
15	9.49 577	38	9.51 818	42	0.48 181	9.97 758	4	45	
16	9.49 615	38	9.51 861	42	0.48 139	9.97 754	4	44	
17	9.49 653	38	9.51 903	42	0.48 096	9.97 750	4	43	
18	9.49 692	38	9.51 946	42	0.48 054	9.97 746	4	42	
19	9.49 730	38	9.51 988	42	0.48 012	9.97 742	4	41	
20	9.49 768	38	9.52 030	42	0.47 969	9.97 737	4	40	
21	9.49 806	38	9.52 073	42	0.47 927	9.97 733	4	39	
22	9.49 844	38	9.52 115	42	0.47 885	9.97 729	4	38	
23	9.49 882	38	9.52 157	42	0.47 842	9.97 725	4	37	
24	9.49 920	38	9.52 199	42	0.47 800	9.97 721	4	36	
25	9.49 958	38	9.52 241	42	0.47 758	9.97 716	4	35	
26	9.49 996	38	9.52 284	42	0.47 716	9.97 712	4	34	
27	9.50 034	37	9.52 326	42	0.47 674	9.97 708	4	33	
28	9.50 072	38	9.52 368	42	0.47 632	9.97 704	4	32	
29	9.50 110	38	9.52 410	42	0.47 590	9.97 700	4	31	
30	9.50 147	37	9.52 452	42	0.47 548	9.97 695	4	30	
31	9.50 185	38	9.52 494	42	0.47 506	9.97 691	4	29	
32	9.50 223	37	9.52 536	42	0.47 464	9.97 687	4	28	
33	9.50 260	37	9.52 578	42	0.47 422	9.97 683	4	27	
34	9.50 298	38	9.52 619	41	0.47 380	9.97 678	4	26	
35	9.50 336	37	9.52 661	42	0.47 338	9.97 674	4	25	
36	9.50 373	37	9.52 703	42	0.47 296	9.97 670	4	24	
37	9.50 411	37	9.52 745	42	0.47 255	9.97 666	4	23	
38	9.50 448	37	9.52 787	42	0.47 213	9.97 661	4	22	
39	9.50 486	37	9.52 828	41	0.47 171	9.97 657	4	21	
40	9.50 523	37	9.52 870	41	0.47 130	9.97 653	4	20	
41	9.50 561	37	9.52 912	42	0.47 088	9.97 649	4	19	
42	9.50 598	37	9.52 953	41	0.47 046	9.97 644	4	18	
43	9.50 635	37	9.52 995	41	0.47 005	9.97 640	4	17	
44	9.50 672	37	9.53 036	41	0.46 963	9.97 636	4	16	
45	9.50 710	37	9.53 078	41	0.46 922	9.97 632	4	15	
46	9.50 747	37	9.53 119	41	0.46 880	9.97 627	4	14	
47	9.50 784	37	9.53 161	41	0.46 839	9.97 623	4	13	
48	9.50 821	37	9.53 202	41	0.46 797	9.97 619	4	12	
49	9.50 858	37	9.53 244	41	0.46 756	9.97 614	4	11	
50	9.50 895	37	9.53 285	41	0.46 714	9.97 610	4	10	
51	9.50 932	37	9.53 326	41	0.46 673	9.97 606	4	9	
52	9.50 969	37	9.53 368	41	0.46 632	9.97 601	4	8	
53	9.51 006	37	9.53 409	41	0.46 591	9.97 597	4	7	
54	9.51 043	37	9.53 450	41	0.46 549	9.97 593	4	6	
55	9.51 080	37	9.53 491	41	0.46 508	9.97 588	4	5	
56	9.51 117	36	9.53 533	41	0.46 467	9.97 584	4	4	
57	9.51 154	37	9.53 574	41	0.46 426	9.97 580	4	3	
58	9.51 190	36	9.53 615	41	0.46 385	9.97 575	4	2	
59	9.51 227	37	9.53 656	41	0.46 344	9.97 571	4	1	
60	9.51 264	36	9.53 697	41	0.46 303	9.97 567	4	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

	43	42	42
6	4.3	4.2	4.2
7	5.0	4.9	4.9
8	5.7	5.6	5.6
9	6.4	6.4	6.3
10	7.1	7.1	7.0
20	14.3	14.1	14.0
30	21.5	21.2	21.0
40	28.6	28.3	28.0
50	35.8	35.4	35.0

	41	41
6	4.1	4.1
7	4.8	4.8
8	5.5	5.4
9	6.2	6.1
10	6.9	6.8
20	13.8	13.6
30	20.7	20.5
40	27.6	27.3
50	34.6	34.1

	39	38	38
6	3.9	3.8	3.8
7	4.5	4.5	4.4
8	5.2	5.1	5.0
9	5.8	5.8	5.7
10	6.5	6.4	6.3
20	13.0	12.8	12.6
30	19.5	19.2	19.0
40	26.0	25.6	25.3
50	32.5	32.1	31.6

	37	37	36
6	3.7	3.7	3.6
7	4.4	4.3	4.2
8	5.0	4.9	4.8
9	5.6	5.5	5.5
10	6.2	6.1	6.1
20	12.5	12.3	12.1
30	18.7	18.5	18.2
40	25.0	24.6	24.3
50	31.2	30.8	30.4

	4	4
6	0.4	0.4
7	0.5	0.4
8	0.6	0.5
9	0.7	0.6
10	0.7	0.6
20	1.5	1.3
30	2.2	2.0
40	3.0	2.6
50	3.7	3.3

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.51 264	37	9.53 697	41	0.46 303	9.97 567	4	60
1	9.51 301	36	9.53 738	41	0.46 262	9.97 562	4	59
2	9.51 337	36	9.53 779	41	0.46 221	9.97 558	4	58
3	9.51 374	36	9.53 820	41	0.46 180	9.97 554	4	57
4	9.51 410	36	9.53 861	41	0.46 139	9.97 549	4	56
5	9.51 447	36	9.53 902	41	0.46 098	9.97 545	4	55
6	9.51 483	36	9.53 943	40	0.46 057	9.97 541	4	54
7	9.51 520	36	9.53 983	41	0.46 016	9.97 536	4	53
8	9.51 556	36	9.54 024	41	0.45 975	9.97 532	4	52
9	9.51 593	36	9.54 065	40	0.45 934	9.97 527	4	51
10	9.51 629	36	9.54 106	41	0.45 894	9.97 523	4	50
11	9.51 665	36	9.54 147	40	0.45 853	9.97 519	4	49
12	9.51 702	36	9.54 187	40	0.45 812	9.97 514	4	48
13	9.51 738	36	9.54 228	41	0.45 772	9.97 510	4	47
14	9.51 774	36	9.54 269	40	0.45 731	9.97 505	4	46
15	9.51 810	36	9.54 309	40	0.45 690	9.97 501	4	45
16	9.51 847	36	9.54 350	40	0.45 650	9.97 497	4	44
17	9.51 883	36	9.54 390	40	0.45 609	9.97 492	4	43
18	9.51 919	36	9.54 431	40	0.45 569	9.97 488	4	42
19	9.51 955	36	9.54 471	40	0.45 528	9.97 483	4	41
20	9.51 991	36	9.54 512	40	0.45 488	9.97 479	4	40
21	9.52 027	36	9.54 552	40	0.45 447	9.97 475	4	39
22	9.52 063	36	9.54 593	40	0.45 407	9.97 470	4	38
23	9.52 099	36	9.54 633	40	0.45 367	9.97 466	4	37
24	9.52 135	36	9.54 673	40	0.45 326	9.97 461	4	36
25	9.52 170	35	9.54 714	40	0.45 286	9.97 457	4	35
26	9.52 206	36	9.54 754	40	0.45 246	9.97 452	4	34
27	9.52 242	36	9.54 794	40	0.45 205	9.97 448	4	33
28	9.52 278	35	9.54 834	40	0.45 165	9.97 443	4	32
29	9.52 314	36	9.54 874	40	0.45 125	9.97 439	4	31
30	9.52 349	35	9.54 915	40	0.45 085	9.97 434	4	30
31	9.52 385	35	9.54 955	40	0.45 045	9.97 430	4	29
32	9.52 421	36	9.54 995	40	0.45 005	9.97 425	4	28
33	9.52 456	35	9.55 035	40	0.44 965	9.97 421	4	27
34	9.52 492	35	9.55 075	40	0.44 925	9.97 416	4	26
35	9.52 527	35	9.55 115	39	0.44 884	9.97 412	4	25
36	9.52 563	35	9.55 155	40	0.44 845	9.97 407	4	24
37	9.52 598	35	9.55 195	40	0.44 805	9.97 403	4	23
38	9.52 634	35	9.55 235	40	0.44 765	9.97 398	4	22
39	9.52 669	35	9.55 275	40	0.44 725	9.97 394	4	21
40	9.52 704	35	9.55 315	40	0.44 685	9.97 389	4	20
41	9.52 740	35	9.55 355	39	0.44 645	9.97 385	4	19
42	9.52 775	35	9.55 394	40	0.44 605	9.97 380	4	18
43	9.52 810	35	9.55 434	39	0.44 565	9.97 376	4	17
44	9.52 846	35	9.55 474	40	0.44 526	9.97 371	4	16
45	9.52 881	35	9.55 514	39	0.44 486	9.97 367	4	15
46	9.52 916	35	9.55 553	40	0.44 446	9.97 362	4	14
47	9.52 951	35	9.55 593	39	0.44 406	9.97 358	4	13
48	9.52 986	35	9.55 633	39	0.44 367	9.97 353	4	12
49	9.53 021	35	9.55 672	39	0.44 327	9.97 349	4	11
50	9.53 056	35	9.55 712	39	0.44 288	9.97 344	4	10
51	9.53 091	35	9.55 751	40	0.44 248	9.97 340	5	9
52	9.53 126	35	9.55 791	39	0.44 209	9.97 335	4	8
53	9.53 161	35	9.55 831	39	0.44 169	9.97 330	4	7
54	9.53 196	35	9.55 870	39	0.44 129	9.97 326	4	6
55	9.53 231	34	9.55 909	39	0.44 090	9.97 321	4	5
56	9.53 266	35	9.55 949	39	0.44 051	9.97 317	4	4
57	9.53 301	34	9.55 988	39	0.44 011	9.97 312	4	3
58	9.53 335	35	9.56 028	39	0.43 972	9.97 308	5	2
59	9.53 370	34	9.56 067	39	0.43 932	9.97 303	4	1
60	9.53 405		9.56 106	39	0.43 893	9.97 298	4	0
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.

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'	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.53 405	35	9.56 106	39	0.43 893	9.97 298	4	60	
1	9.53 440	34	9.56 146	39	0.43 854	9.97 294	4	59	
2	9.53 474	34	9.56 185	39	0.43 815	9.97 289	4	58	
3	9.53 509	35	9.56 224	39	0.43 775	9.97 285	5	57	
4	9.53 544	34	9.56 263	39	0.43 736	9.97 280	4	56	
5	9.53 578	34	9.56 303	39	0.43 697	9.97 275	4	55	
6	9.53 613	34	9.56 342	39	0.43 658	9.97 271	4	54	
7	9.53 647	34	9.56 381	39	0.43 619	9.97 266	5	53	
8	9.53 682	34	9.56 420	39	0.43 580	9.97 261	4	52	
9	9.53 716	34	9.56 459	39	0.43 540	9.97 257	4	51	
10	9.53 750	34	9.56 498	39	0.43 501	9.97 252	4	50	
11	9.53 785	34	9.56 537	39	0.43 462	9.97 248	5	49	
12	9.53 819	34	9.56 576	39	0.43 423	9.97 243	4	48	
13	9.53 854	34	9.56 615	39	0.43 384	9.97 238	4	47	
14	9.53 888	34	9.56 654	38	0.43 346	9.97 234	4	46	
15	9.53 922	34	9.56 693	39	0.43 307	9.97 229	5	45	
16	9.53 956	34	9.56 732	39	0.43 268	9.97 224	4	44	
17	9.53 990	34	9.56 771	39	0.43 229	9.97 220	5	43	
18	9.54 025	34	9.56 810	38	0.43 190	9.97 215	4	42	
19	9.54 059	34	9.56 848	39	0.43 151	9.97 210	4	41	
20	9.54 093	34	9.56 887	38	0.43 112	9.97 206	5	40	
21	9.54 127	34	9.56 926	39	0.43 074	9.97 201	4	39	
22	9.54 161	34	9.56 965	39	0.43 035	9.97 196	4	38	
23	9.54 195	34	9.57 003	38	0.42 996	9.97 191	5	37	
24	9.54 229	33	9.57 042	39	0.42 958	9.97 187	4	36	
25	9.54 263	34	9.57 081	38	0.42 919	9.97 182	5	35	
26	9.54 297	34	9.57 119	38	0.42 880	9.97 177	4	34	
27	9.54 331	34	9.57 158	38	0.42 842	9.97 173	5	33	
28	9.54 365	33	9.57 196	38	0.42 803	9.97 168	4	32	
29	9.54 398	34	9.57 235	38	0.42 765	9.97 163	4	31	
30	9.54 432	34	9.57 274	39	0.42 726	9.97 159	5	30	
31	9.54 466	33	9.57 312	38	0.42 687	9.97 154	4	29	
32	9.54 500	34	9.57 350	38	0.42 649	9.97 149	5	28	
33	9.54 534	33	9.57 389	38	0.42 611	9.97 144	4	27	
34	9.54 567	33	9.57 427	38	0.42 572	9.97 140	4	26	
35	9.54 601	33	9.57 466	38	0.42 534	9.97 135	5	25	
36	9.54 634	34	9.57 504	38	0.42 495	9.97 130	4	24	
37	9.54 668	33	9.57 542	38	0.42 457	9.97 125	5	23	
38	9.54 702	33	9.57 581	38	0.42 419	9.97 121	4	22	
39	9.54 735	33	9.57 619	38	0.42 380	9.97 116	4	21	
40	9.54 769	33	9.57 657	38	0.42 342	9.97 111	5	20	
41	9.54 802	33	9.57 696	38	0.42 304	9.97 106	4	19	
42	9.54 836	33	9.57 734	38	0.42 266	9.97 102	5	18	
43	9.54 869	33	9.57 772	38	0.42 227	9.97 097	4	17	
44	9.54 902	33	9.57 810	38	0.42 189	9.97 092	5	16	
45	9.54 936	33	9.57 848	38	0.42 151	9.97 087	4	15	
46	9.54 969	33	9.57 886	38	0.42 113	9.97 082	5	14	
47	9.55 002	33	9.57 925	38	0.42 075	9.97 078	4	13	
48	9.55 036	33	9.57 963	38	0.42 037	9.97 073	5	12	
49	9.55 069	33	9.58 001	38	0.41 999	9.97 068	4	11	
50	9.55 102	33	9.58 039	38	0.41 961	9.97 063	5	10	
51	9.55 135	33	9.58 077	38	0.41 923	9.97 058	4	9	
52	9.55 168	33	9.58 115	38	0.41 885	9.97 054	5	8	
53	9.55 202	33	9.58 153	37	0.41 847	9.97 049	4	7	
54	9.55 235	33	9.58 190	38	0.41 809	9.97 044	5	6	
55	9.55 268	33	9.58 228	38	0.41 771	9.97 039	4	5	
56	9.55 301	33	9.58 266	38	0.41 733	9.97 034	5	4	
57	9.55 334	33	9.58 304	37	0.41 695	9.97 029	4	3	
58	9.55 367	33	9.58 342	38	0.41 658	9.97 025	5	2	
59	9.55 400	33	9.58 380	37	0.41 620	9.97 020	4	1	
60	9.55 433	33	9.58 417	37	0.41 582	9.97 015	5	0	
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.	d.		P. P.

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS, AND COTANGENTS.

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'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.55 433	33	9.58 417	38	0.41 582	9.97 015	4	60	
1	9.55 466	32	9.58 455	37	0.41 544	9.97 010	5	59	
2	9.55 498	33	9.58 493	38	0.41 507	9.97 005	5	58	
3	9.55 531	33	9.58 531	38	0.41 469	9.97 000	5	57	
4	9.55 564	33	9.58 568	37	0.41 431	9.96 995	4	56	
5	9.55 597	32	9.58 606	38	0.41 394	9.96 991	5	55	
6	9.55 630	32	9.58 644	37	0.41 356	9.96 986	5	54	
7	9.55 662	33	9.58 681	37	0.41 318	9.96 981	5	53	
8	9.55 695	32	9.58 719	37	0.41 281	9.96 976	4	52	
9	9.55 728	32	9.58 756	37	0.41 243	9.96 971	5	51	
10	9.55 760	32	9.58 794	37	0.41 206	9.96 966	5	50	
11	9.55 793	33	9.58 831	37	0.41 168	9.96 961	5	49	
12	9.55 826	32	9.58 869	37	0.41 131	9.96 956	4	48	
13	9.55 858	32	9.58 906	37	0.41 093	9.96 952	4	47	
14	9.55 891	32	9.58 944	37	0.41 056	9.96 947	5	46	
15	9.55 923	32	9.58 981	37	0.41 018	9.96 942	5	45	
16	9.55 956	32	9.59 019	37	0.40 981	9.96 937	5	44	
17	9.55 988	32	9.59 056	37	0.40 944	9.96 932	5	43	
18	9.56 020	32	9.59 093	37	0.40 906	9.96 927	4	42	
19	9.56 053	32	9.59 131	37	0.40 869	9.96 922	5	41	
20	9.56 085	32	9.59 168	37	0.40 832	9.96 917	5	40	
21	9.56 118	32	9.59 205	37	0.40 794	9.96 912	5	39	
22	9.56 150	32	9.59 242	37	0.40 757	9.96 907	5	38	
23	9.56 182	32	9.59 280	37	0.40 720	9.96 902	5	37	
24	9.56 214	32	9.59 317	37	0.40 683	9.96 897	5	36	
25	9.56 247	32	9.59 354	37	0.40 646	9.96 892	5	35	
26	9.56 279	32	9.59 391	37	0.40 608	9.96 887	5	34	
27	9.56 311	32	9.59 428	37	0.40 571	9.96 882	5	33	
28	9.56 343	32	9.59 465	37	0.40 534	9.96 877	4	32	
29	9.56 375	32	9.59 502	37	0.40 497	9.96 873	5	31	
30	9.56 407	32	9.59 540	37	0.40 460	9.96 868	5	30	
31	9.56 439	32	9.59 577	37	0.40 423	9.96 863	5	29	
32	9.56 471	32	9.59 614	37	0.40 386	9.96 858	5	28	
33	9.56 503	32	9.59 651	37	0.40 349	9.96 853	5	27	
34	9.56 535	32	9.59 688	36	0.40 312	9.96 848	5	26	
35	9.56 567	32	9.59 724	37	0.40 275	9.96 843	5	25	
36	9.56 599	32	9.59 761	37	0.40 238	9.96 838	5	24	
37	9.56 631	31	9.59 798	37	0.40 201	9.96 833	5	23	
38	9.56 663	32	9.59 835	36	0.40 164	9.96 828	5	22	
39	9.56 695	32	9.59 872	37	0.40 128	9.96 823	5	21	
40	9.56 727	31	9.59 909	37	0.40 091	9.96 818	5	20	
41	9.56 758	32	9.59 946	36	0.40 054	9.96 813	5	19	
42	9.56 790	31	9.59 982	37	0.40 017	9.96 808	5	18	
43	9.56 822	32	9.60 019	36	0.39 980	9.96 802	5	17	
44	9.56 854	31	9.60 056	37	0.39 944	9.96 797	5	16	
45	9.56 885	31	9.60 093	37	0.39 907	9.96 792	5	15	
46	9.56 917	32	9.60 129	36	0.39 870	9.96 787	5	14	
47	9.56 949	31	9.60 166	36	0.39 833	9.96 782	5	13	
48	9.56 980	31	9.60 203	36	0.39 797	9.96 777	5	12	
49	9.57 012	31	9.60 239	36	0.39 760	9.96 772	5	11	
50	9.57 043	31	9.60 276	36	0.39 724	9.96 767	5	10	
51	9.57 075	31	9.60 312	37	0.39 687	9.96 762	5	9	
52	9.57 106	31	9.60 349	36	0.39 650	9.96 757	5	8	
53	9.57 138	31	9.60 386	36	0.39 614	9.96 752	5	7	
54	9.57 169	31	9.60 422	36	0.39 577	9.96 747	5	6	
55	9.57 201	31	9.60 459	36	0.39 541	9.96 742	5	5	
56	9.57 232	31	9.60 495	36	0.39 504	9.96 737	5	4	
57	9.57 263	31	9.60 531	36	0.39 468	9.96 732	5	3	
58	9.57 295	31	9.60 568	36	0.39 432	9.96 727	5	2	
59	9.57 326	31	9.60 604	36	0.39 395	9.96 721	5	1	
60	9.57 357	31	9.60 641	36	0.39 359	9.96 716	5	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

6	3.8	3.7	3.7
7	4.4	4.4	4.3
8	5.0	5.0	4.9
9	5.7	5.6	5.5
10	6.3	6.2	6.1
20	12.6	12.5	12.3
30	19.0	18.7	18.5
40	25.3	25.0	24.6
50	31.6	31.2	30.8

6	3.6	3.6
7	4.2	4.2
8	4.8	4.8
9	5.5	5.4
10	6.1	6.0
20	12.1	12.0
30	18.2	18.0
40	24.3	24.0
50	30.4	30.0

6	3.3	3.2	3.2
8	3.8	3.8	3.7
7	4.4	4.3	4.2
9	4.9	4.9	4.8
10	5.5	5.4	5.3
20	11.0	10.8	10.6
30	16.5	16.2	16.0
40	22.0	21.6	21.3
50	27.5	27.1	26.6

6	3.1	3.1
7	3.7	3.6
8	4.2	4.1
9	4.7	4.6
10	5.2	5.1
20	10.5	10.3
30	15.7	15.5
40	21.0	20.6
50	26.2	25.8

6	0.5	0.5	0.4
7	0.6	0.6	0.5
8	0.7	0.6	0.6
9	0.8	0.7	0.7
10	0.9	0.8	0.7
20	1.8	1.6	1.5
30	2.7	2.5	2.2
40	3.6	3.3	3.0
50	4.6	4.1	3.7

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

22°

157°

	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.57357	31	9.60641	36	0.39359	9.96716	5	60	
1	9.57389	31	9.60677	36	0.39322	9.96711	5	59	
2	9.57420	31	9.60713	36	0.39286	9.96706	5	58	
3	9.57451	31	9.60750	36	0.39250	9.96701	5	57	
4	9.57482	31	9.60786	36	0.39213	9.96696	5	56	36 36
5	9.57513	31	9.60822	36	0.39177	9.96691	5	55	6 3.6 3.6
6	9.57544	31	9.60859	36	0.39141	9.96686	5	54	7 4.2 4.2
7	9.57576	31	9.60895	36	0.39105	9.96681	5	53	8 4.8 4.8
8	9.57607	31	9.60931	36	0.39069	9.96675	5	52	9 5.5 5.4
9	9.57638	31	9.60967	36	0.39032	9.96670	5	51	10 6.1 6.0
10	9.57669	31	9.61003	36	0.38996	9.96665	5	50	20 12.1 12.0
11	9.57700	31	9.61039	36	0.38960	9.96660	5	49	30 18.2 18.0
12	9.57731	31	9.61076	36	0.38924	9.96655	5	48	40 24.3 24.0
13	9.57762	31	9.61112	36	0.38888	9.96650	5	47	50 30.4 30.0
14	9.57792	30	9.61148	36	0.38852	9.96644	5	46	
15	9.57823	31	9.61184	36	0.38816	9.96639	5	45	35 35
16	9.57854	31	9.61220	36	0.38780	9.96634	5	44	6 3.5 3.5
17	9.57885	30	9.61256	36	0.38744	9.96629	5	43	7 4.1 4.1
18	9.57916	31	9.61292	36	0.38708	9.96624	5	42	8 4.7 4.6
19	9.57947	31	9.61328	36	0.38672	9.96619	5	41	9 5.3 5.2
20	9.57977	30	9.61364	36	0.38636	9.96613	5	40	10 5.9 5.8
21	9.58008	31	9.61400	36	0.38600	9.96608	5	39	20 11.8 11.6
22	9.58039	30	9.61436	36	0.38564	9.96603	5	38	30 17.7 17.5
23	9.58070	31	9.61472	36	0.38528	9.96598	5	37	40 23.6 23.3
24	9.58100	30	9.61507	35	0.38492	9.96593	5	36	50 29.6 29.1
25	9.58131	30	9.61543	36	0.38456	9.96587	5	35	
26	9.58162	31	7.61579	35	0.38420	9.96582	5	34	31 31
27	9.58192	30	9.61615	36	0.38385	9.96577	5	33	6 3.1 3.1
28	9.58223	30	9.61651	36	0.38349	9.96572	5	32	7 3.7 3.6
29	9.58253	30	9.61686	35	0.38313	9.96567	5	31	8 4.2 4.1
30	9.58284	30	9.61722	35	0.38277	9.96561	5	30	9 4.7 4.6
31	9.58314	30	9.61758	36	0.38242	9.96556	5	29	10 5.2 5.1
32	9.58345	30	9.61794	36	0.38206	9.96551	5	28	20 10.5 10.3
33	9.58375	30	9.61829	35	0.38170	9.96546	5	27	30 15.7 15.5
34	9.58406	30	9.61865	36	0.38135	9.96540	5	26	40 21.0 20.6
35	9.58436	30	9.61901	36	0.38099	9.96535	5	25	50 26.2 25.8
36	9.58466	30	9.61936	36	0.38063	9.96530	5	24	
37	9.58497	30	9.61972	36	0.38028	9.96525	5	23	
38	9.58527	30	9.62007	36	0.37992	9.96519	5	22	30 30 29
39	9.58557	30	9.62043	36	0.37957	9.96514	5	21	6 3.0 3.0
40	9.58587	30	9.62078	35	0.37921	9.96509	5	20	7 3.5 3.5
41	9.58618	30	9.62114	36	0.37886	9.96503	5	19	8 4.0 4.0
42	9.58648	30	9.62149	36	0.37850	9.96498	5	18	9 4.6 4.5
43	9.58678	30	9.62185	35	0.37815	9.96493	5	17	10 5.1 5.0
44	9.58708	30	9.62220	35	0.37779	9.96488	5	16	20 10.1 10.0
45	9.58738	30	9.62256	35	0.37744	9.96482	5	15	30 15.2 15.0
46	9.58769	30	9.62291	35	0.37708	9.96477	5	14	40 20.3 20.0
47	9.58799	30	9.62327	35	0.37673	9.96472	5	13	50 25.4 25.0
48	9.58829	30	9.62362	35	0.37637	9.96466	5	12	
49	9.58859	30	9.62397	35	0.37602	9.96461	5	11	
50	9.58889	30	9.62433	35	0.37567	9.96456	5	10	5 5
51	9.58919	30	9.62468	35	0.37531	9.96450	5	9	6 0.5 0.5
52	9.58949	30	9.62503	35	0.37496	9.96445	5	8	7 0.6 0.6
53	9.58979	30	9.62539	35	0.37461	9.96440	5	7	8 0.7 0.7
54	9.59009	29	9.62574	35	0.37426	9.96434	5	6	9 0.8 0.8
55	9.59038	30	9.62609	35	0.37391	9.96429	5	5	10 0.9 0.9
56	9.59068	30	9.62644	35	0.37355	9.96424	5	4	20 1.8 1.6
57	9.59098	29	9.62679	35	0.37320	9.96418	5	3	30 2.7 2.5
58	9.59128	30	9.62715	35	0.37285	9.96413	5	2	40 3.6 3.3
59	9.59158	30	9.62750	35	0.37250	9.96408	5	1	50 4.6 4.1
60	9.59188	30	9.62785	35	0.37215	9.96402	5	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.59 188	29	9.62 785	35	0.37 215	9.96 402	51	60	
1	9.59 217	30	9.62 820	35	0.37 179	9.96 397	50	59	
2	9.59 247	30	9.62 855	35	0.37 144	9.96 392	49	58	
3	9.59 277	29	9.62 890	35	0.37 109	9.96 386	48	57	
4	9.59 306	29	9.62 925	35	0.37 074	9.96 381	47	56	
5	9.59 336	30	9.62 960	35	0.37 039	9.96 375	46	55	
6	9.59 366	29	9.62 995	35	0.37 004	9.96 370	45	54	
7	9.59 395	29	9.63 030	35	0.36 969	9.96 365	44	53	
8	9.59 425	29	9.63 065	35	0.36 934	9.96 359	43	52	
9	9.59 454	29	9.63 100	35	0.36 899	9.96 354	42	51	
10	9.59 484	29	9.63 135	35	0.36 864	9.96 349	41	50	
11	9.59 513	29	9.63 170	35	0.36 829	9.96 343	40	49	
12	9.59 543	29	9.63 205	34	0.36 794	9.96 338	39	48	
13	9.59 572	29	9.63 240	35	0.36 760	9.96 332	38	47	
14	9.59 602	29	9.63 275	35	0.36 725	9.96 327	37	46	
15	9.59 631	29	9.63 310	34	0.36 690	9.96 321	36	45	
16	9.59 661	29	9.63 344	35	0.36 655	9.96 316	35	44	
17	9.59 690	29	9.63 379	35	0.36 620	9.96 311	34	43	
18	9.59 719	29	9.63 414	34	0.36 585	9.96 305	33	42	
19	9.59 749	29	9.63 449	35	0.36 551	9.96 300	32	41	
20	9.59 778	29	9.63 484	34	0.36 516	9.96 294	31	40	
21	9.59 807	29	9.63 518	34	0.36 481	9.96 289	30	39	
22	9.59 837	29	9.63 553	35	0.36 447	9.96 283	29	38	
23	9.59 866	29	9.63 588	34	0.36 412	9.96 278	28	37	
24	9.59 895	29	9.63 622	34	0.36 377	9.96 272	27	36	
25	9.59 924	29	9.63 657	35	0.36 343	9.96 267	26	35	
26	9.59 953	29	9.63 692	34	0.36 308	9.96 261	25	34	
27	9.59 982	29	9.63 726	34	0.36 273	9.96 256	24	33	
28	9.60 012	29	9.63 761	34	0.36 239	9.96 251	23	32	
29	9.60 041	29	9.63 795	34	0.36 204	9.96 245	22	31	
30	9.60 070	29	9.63 830	34	0.36 170	9.96 240	21	30	
31	9.60 099	29	9.63 864	34	0.36 135	9.96 234	20	29	
32	9.60 128	29	9.63 899	34	0.36 101	9.96 229	19	28	
33	9.60 157	29	9.63 933	34	0.36 066	9.96 223	18	27	
34	9.60 186	29	9.63 968	34	0.36 032	9.96 218	17	26	
35	9.60 215	29	9.64 002	34	0.35 997	9.96 212	16	25	
36	9.60 244	29	9.64 037	34	0.35 963	9.96 206	15	24	
37	9.60 273	28	9.64 071	34	0.35 928	9.96 201	14	23	
38	9.60 301	28	9.64 106	34	0.35 894	9.96 195	13	22	
39	9.60 330	29	9.64 140	34	0.35 859	9.96 190	12	21	
40	9.60 359	29	9.64 174	34	0.35 825	9.96 184	11	20	
41	9.60 388	28	9.64 209	34	0.35 791	9.96 179	10	19	
42	9.60 417	28	9.64 243	34	0.35 756	9.96 173	9	18	
43	9.60 445	29	9.64 277	34	0.35 722	9.96 168	8	17	
44	9.60 474	29	9.64 312	34	0.35 688	9.96 162	7	16	
45	9.60 503	28	9.64 346	34	0.35 653	9.96 157	6	15	
46	9.60 532	28	9.64 380	34	0.35 619	9.96 151	5	14	
47	9.60 560	28	9.64 415	34	0.35 585	9.96 146	4	13	
48	9.60 589	28	9.64 449	34	0.35 551	9.96 140	3	12	
49	9.60 618	29	9.64 483	34	0.35 517	9.96 134	2	11	
50	9.60 646	28	9.64 517	34	0.35 482	9.96 129	1	10	
51	9.60 675	28	9.64 551	34	0.35 448	9.96 123	0	9	
52	9.60 703	28	9.64 585	34	0.35 414	9.96 118		8	
53	9.60 732	28	9.64 620	34	0.35 380	9.96 112		7	
54	9.60 760	28	9.64 654	34	0.35 346	9.96 106		6	
55	9.60 789	28	9.64 688	34	0.35 312	9.96 101		5	
56	9.60 817	28	9.64 722	34	0.35 278	9.96 095		4	
57	9.60 846	28	9.64 756	34	0.35 244	9.96 090		3	
58	9.60 874	28	9.64 790	34	0.35 209	9.96 084		2	
59	9.60 903	28	9.64 824	34	0.35 175	9.96 078		1	
60	9.60 931	28	9.64 858	34	0.35 141	9.96 073		0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

24°

155°

	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.60 931	28	9.64 858	34	0.35 141	9.96 073	51	60	
1	9.60 959	28	9.64 892	34	0.35 107	9.96 067	51	59	
2	9.60 988	28	9.64 926	33	0.35 073	9.96 062	51	58	
3	9.61 016	28	9.64 960	34	0.35 040	9.96 056	51	57	
4	9.61 044	28	9.64 994	34	0.35 006	9.96 050	51	56	
5	9.61 073	28	9.65 028	34	0.34 972	9.96 045	51	55	
6	9.61 101	28	9.65 062	34	0.34 938	9.96 039	51	54	
7	9.61 129	28	9.65 096	33	0.34 904	9.96 033	51	53	
8	9.61 157	28	9.65 129	34	0.34 870	9.96 028	51	52	
9	9.61 186	28	9.65 163	34	0.34 836	9.96 022	51	51	
10	9.61 214	28	9.65 197	33	0.34 802	9.96 016	51	50	
11	9.61 242	28	9.65 231	34	0.34 769	9.96 011	51	49	
12	9.61 270	28	9.65 265	34	0.34 735	9.96 005	51	48	
13	9.61 298	28	9.65 299	33	0.34 701	9.95 999	51	47	
14	9.61 326	28	9.65 332	33	0.34 667	9.95 994	51	46	
15	9.61 354	28	9.65 366	34	0.34 633	9.95 988	51	45	
16	9.61 382	28	9.65 400	33	0.34 600	9.95 982	51	44	
17	9.61 410	28	9.65 433	34	0.34 566	9.95 977	51	43	
18	9.61 438	28	9.65 467	33	0.34 532	9.95 971	51	42	
19	9.61 466	28	9.65 501	33	0.34 499	9.95 965	51	41	
20	9.61 494	28	9.65 535	34	0.34 465	9.95 959	51	40	
21	9.61 522	27	9.65 568	33	0.34 431	9.95 954	51	39	
22	9.61 550	28	9.65 602	33	0.34 398	9.95 948	51	38	
23	9.61 578	28	9.65 635	33	0.34 364	9.95 942	51	37	
24	9.61 606	28	9.65 669	34	0.34 331	9.95 937	51	36	
25	9.61 634	27	9.65 703	33	0.34 297	9.95 931	51	35	
26	9.61 661	28	9.65 736	33	0.34 263	9.95 925	51	34	
27	9.61 689	27	9.65 770	33	0.34 230	9.95 919	51	33	
28	9.61 717	28	9.65 803	33	0.34 196	9.95 914	51	32	
29	9.61 745	28	9.65 837	33	0.34 163	9.95 908	51	31	
30	9.61 772	27	9.65 870	33	0.34 129	9.95 902	51	30	
31	9.61 800	28	9.65 904	33	0.34 096	9.95 896	51	29	
32	9.61 828	27	9.65 937	33	0.34 062	9.95 891	51	28	
33	9.61 856	28	9.65 971	33	0.34 029	9.95 885	51	27	
34	9.61 883	27	9.66 004	33	0.33 996	9.95 879	51	26	
35	9.61 911	27	9.66 037	33	0.33 962	9.95 873	51	25	
36	9.61 938	27	9.66 071	33	0.33 929	9.95 867	51	24	
37	9.61 966	28	9.66 104	33	0.33 895	9.95 862	51	23	
38	9.61 994	27	9.66 137	33	0.33 862	9.95 856	51	22	
39	9.62 021	27	9.66 171	33	0.33 829	9.95 850	51	21	
40	9.62 049	27	9.66 204	33	0.33 795	9.95 844	51	20	
41	9.62 076	27	9.66 237	33	0.33 762	9.95 838	51	19	
42	9.62 104	27	9.66 271	33	0.33 729	9.95 833	51	18	
43	9.62 131	27	9.66 304	33	0.33 696	9.95 827	51	17	
44	9.62 158	27	9.66 337	33	0.33 662	9.95 821	51	16	
45	9.62 186	27	9.66 370	33	0.33 629	9.95 815	51	15	
46	9.62 213	27	9.66 404	33	0.33 596	9.95 809	51	14	
47	9.62 241	27	9.66 437	33	0.33 563	9.95 804	51	13	
48	9.62 268	27	9.66 470	33	0.33 529	9.95 798	51	12	
49	9.62 295	27	9.66 503	33	0.33 496	9.95 792	51	11	
50	9.62 323	27	9.66 536	33	0.33 463	9.95 786	51	10	
51	9.62 350	27	9.66 570	33	0.33 430	9.95 780	51	9	
52	9.62 377	27	9.66 603	33	0.33 397	9.95 774	51	8	
53	9.62 404	27	9.66 636	33	0.33 364	9.95 768	51	7	
54	9.62 432	27	9.66 669	33	0.33 331	9.95 763	51	6	
55	9.62 459	27	9.66 702	33	0.33 298	9.95 757	51	5	
56	9.62 486	27	9.66 735	33	0.33 265	9.95 751	51	4	
57	9.62 513	27	9.66 768	33	0.33 232	9.95 745	51	3	
58	9.62 540	27	9.66 801	33	0.33 199	9.95 739	51	2	
59	9.62 567	27	9.66 834	33	0.33 165	9.95 733	51	1	
60	9.62 595	27	9.66 867	33	0.33 132	9.95 727	51	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

	34	33	33
6	3.4	3.3	3.3
7	3.9	3.9	3.8
8	4.5	4.4	4.4
9	5.1	5.0	4.9
10	5.6	5.6	5.5
20	11.3	11.1	11.0
30	17.0	16.7	16.5
40	22.6	22.3	22.0
50	28.3	27.9	27.5

	28	28
6	2.8	2.8
7	3.3	3.2
8	3.8	3.7
9	4.3	4.2
10	4.7	4.6
20	9.5	9.3
30	14.2	14.0
40	19.0	18.6
50	23.7	23.3

	27	27
6	2.7	2.7
7	3.2	3.1
8	3.6	3.6
9	4.1	4.0
10	4.6	4.5
20	9.1	9.0
30	13.7	13.5
40	18.3	18.0
50	22.9	22.5

	6	5
6	0.6	0.5
7	0.7	0.6
8	0.8	0.7
9	0.9	0.8
10	1.0	0.9
20	2.0	1.8
30	3.0	2.7
40	4.0	3.6
50	5.0	4.6

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.62 595		9.66 867		0.33 132	9.95 727		60	
1	9.62 622	27	9.66 900	32	0.33 100	9.95 721	6	59	
2	9.62 649	27	9.66 933	33	0.33 067	9.95 716	5	58	
3	9.62 676	27	9.66 966	33	0.33 034	9.95 710	6	57	
4	9.62 703	27	9.66 999	33	0.33 001	9.95 704	6	56	
5	9.62 730	27	9.67 032	33	0.32 968	9.95 698	6	55	
6	9.62 757	27	9.67 065	33	0.32 935	9.95 692	6	54	
7	9.62 784	27	9.67 097	32	0.32 902	9.95 686	6	53	
8	9.62 811	27	9.67 130	33	0.32 869	9.95 680	5	52	
9	9.62 838	27	9.67 163	33	0.32 836	9.95 674	6	51	
10	9.62 864	26	9.67 196	33	0.32 803	9.95 668	6	50	
11	9.62 891	27	9.67 229	32	0.32 771	9.95 662	6	49	
12	9.62 918	27	9.67 262	32	0.32 738	9.95 656	6	48	
13	9.62 945	27	9.67 294	32	0.32 705	9.95 650	6	47	
14	9.62 972	26	9.67 327	33	0.32 672	9.95 644	6	46	
15	9.62 999	27	9.67 360	32	0.32 640	9.95 638	6	45	
16	9.63 025	26	9.67 393	32	0.32 607	9.95 632	6	44	
17	9.63 052	27	9.67 425	33	0.32 574	9.95 627	5	43	
18	9.63 079	26	9.67 458	32	0.32 541	9.95 621	6	42	
19	9.63 106	27	9.67 491	32	0.32 509	9.95 615	6	41	
20	9.63 132	26	9.67 523	32	0.32 476	9.95 609	6	40	
21	9.63 159	27	9.67 556	33	0.32 443	9.95 603	6	39	
22	9.63 186	26	9.67 589	32	0.32 411	9.95 597	6	38	
23	9.63 212	26	9.67 621	32	0.32 378	9.95 591	6	37	
24	9.63 239	26	9.67 654	33	0.32 345	9.95 585	6	36	
25	9.63 266	27	9.67 687	32	0.32 313	9.95 579	6	35	
26	9.63 292	26	9.67 719	32	0.32 280	9.95 573	6	34	
27	9.63 319	26	9.67 752	32	0.32 248	9.95 567	6	33	
28	9.63 345	26	9.67 784	32	0.32 215	9.95 561	6	32	
29	9.63 372	26	9.67 817	32	0.32 183	9.95 555	6	31	
30	9.63 398	26	9.67 849	32	0.32 150	9.95 549	6	30	
31	9.63 425	26	9.67 882	32	0.32 118	9.95 543	6	29	
32	9.63 451	26	9.67 914	32	0.32 085	9.95 537	6	28	
33	9.63 478	26	9.67 947	32	0.32 053	9.95 530	6	27	
34	9.63 504	26	9.67 979	32	0.32 020	9.95 524	6	26	
35	9.63 530	26	9.68 012	32	0.31 988	9.95 518	6	25	
36	9.63 557	26	9.68 044	32	0.31 955	9.95 512	6	24	
37	9.63 583	26	9.68 077	32	0.31 923	9.95 506	6	23	
38	9.63 609	26	9.68 109	32	0.31 891	9.95 500	6	22	
39	9.63 636	26	9.68 141	32	0.31 858	9.95 494	6	21	
40	9.63 662	26	9.68 174	32	0.31 826	9.95 488	6	20	
41	9.63 688	26	9.68 206	32	0.31 793	9.95 482	6	19	
42	9.63 715	26	9.68 238	32	0.31 761	9.95 476	6	18	
43	9.63 741	26	9.68 271	32	0.31 729	9.95 470	6	17	
44	9.63 767	26	9.68 303	32	0.31 696	9.95 464	6	16	
45	9.63 793	26	9.68 335	32	0.31 664	9.95 458	6	15	
46	9.63 819	26	9.68 368	32	0.31 632	9.95 452	6	14	
47	9.63 846	26	9.68 400	32	0.31 600	9.95 445	6	13	
48	9.63 872	26	9.68 432	32	0.31 567	9.95 439	6	12	
49	9.63 898	26	9.68 464	32	0.31 535	9.95 433	6	11	
50	9.63 924	26	9.68 497	32	0.31 503	9.95 427	6	10	
51	9.63 950	26	9.68 529	32	0.31 471	9.95 421	6	9	
52	9.63 976	26	9.68 561	32	0.31 439	9.95 415	6	8	
53	9.64 002	26	9.68 593	32	0.31 406	9.95 409	6	7	
54	9.64 028	26	9.68 625	32	0.31 374	9.95 403	6	6	
55	9.64 054	26	9.68 657	32	0.31 342	9.95 397	6	5	
56	9.64 080	26	9.68 690	32	0.31 310	9.95 390	6	4	
57	9.64 106	26	9.68 722	32	0.31 278	9.95 384	6	3	
58	9.64 132	26	9.68 754	32	0.31 246	9.95 378	6	2	
59	9.64 158	26	9.68 786	32	0.31 214	9.95 372	6	1	
60	9.64 184	25	9.68 818	32	0.31 182	9.95 366	6	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

	33	32	32
6	3.3	3.2	3.2
7	3.8	3.8	3.7
8	4.4	4.3	4.2
9	4.9	4.9	4.8
10	5.5	5.4	5.3
20	11.0	10.8	10.6
30	16.5	16.2	16.0
40	22.0	21.6	21.3
50	27.5	27.1	26.8

	27
6	2.7
7	3.1
8	3.6
9	4.0
10	4.5
20	9.0
30	13.5
40	18.0
50	22.5

	26	26	25
6	2.6	2.6	2.5
7	3.1	3.0	3.0
8	3.5	3.4	3.4
9	4.0	3.9	3.8
10	4.4	4.3	4.2
20	8.8	8.6	8.5
30	13.2	13.0	12.7
40	17.6	17.3	17.0
50	22.1	21.6	21.2

	6	6	5
6	0.6	0.6	0.5
7	0.7	0.7	0.6
8	0.8	0.8	0.7
9	1.0	0.9	0.8
10	1.1	1.0	0.9
20	2.1	2.0	1.8
30	3.2	3.0	2.7
40	4.3	4.0	3.6
50	5.4	5.0	4.6

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**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.64 184	26	9.68 818	32	0.31 182	9.95 366	6	60	
1	9.64 210	26	9.68 850	32	0.31 150	9.95 360	6	59	
2	9.64 236	26	9.68 882	32	0.31 117	9.95 353	6	58	
3	9.64 262	26	9.68 914	32	0.31 085	9.95 347	6	57	
4	9.64 287	25	9.68 946	32	0.31 053	9.95 341	6	56	
5	9.64 313	26	9.68 978	32	0.31 021	9.95 335	6	55	32 32
6	9.64 339	26	9.69 010	32	0.30 989	9.95 329	6	54	6 3.2 3.2
7	9.64 365	25	9.69 042	31	0.30 957	9.95 323	6	53	7 3.8 3.7
8	9.64 391	25	9.69 074	31	0.30 926	9.95 316	6	52	8 4.3 4.2
9	9.64 416	25	9.69 106	32	0.30 894	9.95 310	6	51	9 4.9 4.8
10	9.64 442	26	9.69 138	32	0.30 862	9.95 304	6	50	10 5.4 5.3
11	9.64 468	25	9.69 170	32	0.30 830	9.95 298	6	49	20 10.8 10.6
12	9.64 493	25	9.69 202	32	0.30 798	9.95 292	6	48	30 16.2 16.0
13	9.64 519	26	9.69 234	32	0.30 766	9.95 285	6	47	40 21.6 21.3
14	9.64 545	25	9.69 265	31	0.30 734	9.95 279	6	46	50 27.1 26.6
15	9.64 570	25	9.69 297	32	0.30 702	9.95 273	6	45	
16	9.64 596	25	9.69 329	32	0.30 670	9.95 267	6	44	
17	9.64 622	26	9.69 361	31	0.30 639	9.95 260	6	43	
18	9.64 647	25	9.69 393	32	0.30 607	9.95 254	6	42	
19	9.64 673	25	9.69 425	32	0.30 575	9.95 248	6	41	
20	9.64 698	25	9.69 456	31	0.30 543	9.95 242	6	40	31 31
21	9.64 724	25	9.69 488	32	0.30 511	9.95 235	6	39	6 3.1 3.1
22	9.64 749	25	9.69 520	31	0.30 480	9.95 229	6	38	7 3.7 3.6
23	9.64 775	25	9.69 552	32	0.30 448	9.95 223	6	37	8 4.2 4.1
24	9.64 800	25	9.69 583	31	0.30 416	9.95 217	6	36	9 4.7 4.6
25	9.64 826	25	9.69 615	32	0.30 384	9.95 210	6	35	10 5.2 5.1
26	9.64 851	25	9.69 647	31	0.30 353	9.95 204	6	34	20 10.5 10.3
27	9.64 876	25	9.69 678	32	0.30 321	9.95 198	6	33	30 15.7 15.5
28	9.64 902	25	9.69 710	32	0.30 289	9.95 191	6	32	40 21.0 20.6
29	9.64 927	25	9.69 742	31	0.30 258	9.95 185	6	31	50 26.2 25.8
30	9.64 952	25	9.69 773	31	0.30 226	9.95 179	6	30	
31	9.64 978	25	9.69 805	32	0.30 194	9.95 173	6	29	
32	9.65 003	25	9.69 837	31	0.30 163	9.95 166	6	28	
33	9.65 028	25	9.69 868	31	0.30 131	9.95 160	6	27	26 25 25
34	9.65 054	25	9.69 900	31	0.30 100	9.95 154	6	26	6 2.6 2.5
35	9.65 079	25	9.69 931	31	0.30 068	9.95 147	6	25	7 3.0 3.0
36	9.65 104	25	9.69 963	31	0.30 037	9.95 141	6	24	8 3.4 3.4
37	9.65 129	25	9.69 994	31	0.30 005	9.95 135	6	23	9 3.9 3.8
38	9.65 155	25	9.70 026	31	0.29 973	9.95 128	6	22	10 4.3 4.2
39	9.65 180	25	9.70 058	31	0.29 942	9.95 122	6	21	20 8.6 8.5
40	9.65 205	25	9.70 089	31	0.29 910	9.95 116	6	20	30 13.0 12.7
41	9.65 230	25	9.70 121	31	0.29 879	9.95 109	6	19	40 17.3 17.0
42	9.65 255	25	9.70 152	31	0.29 847	9.95 103	6	18	50 21.6 21.2
43	9.65 280	25	9.70 183	31	0.29 816	9.95 097	6	17	
44	9.65 305	25	9.70 215	31	0.29 785	9.95 090	6	16	
45	9.65 331	25	9.70 246	31	0.29 753	9.95 084	6	15	
46	9.65 356	25	9.70 278	31	0.29 722	9.95 078	6	14	
47	9.65 381	25	9.70 309	31	0.29 690	9.95 071	6	13	24 6 6
48	9.65 406	25	9.70 341	31	0.29 659	9.95 065	6	12	6 2.4 0.6
49	9.65 431	25	9.70 372	31	0.29 628	9.95 058	6	11	7 2.8 0.7
50	9.65 456	25	9.70 403	31	0.29 596	9.95 052	6	10	8 3.2 0.8
51	9.65 481	25	9.70 435	31	0.29 565	9.95 046	6	9	9 3.7 1.0
52	9.65 506	24	9.70 466	31	0.29 533	9.95 039	6	8	10 4.1 1.1
53	9.65 530	25	9.70 497	31	0.29 502	9.95 033	6	7	20 8.1 2.1
54	9.65 555	25	9.70 529	31	0.29 471	9.95 026	6	6	30 12.3 3.2
55	9.65 580	25	9.70 560	31	0.29 439	9.95 020	6	5	40 16.3 4.3
56	9.65 605	24	9.70 591	31	0.29 408	9.95 014	6	4	50 20.4 5.4
57	9.65 630	25	9.70 623	31	0.29 377	9.95 007	6	3	
58	9.65 655	25	9.70 654	31	0.29 346	9.95 001	6	2	
59	9.65 680	24	9.70 685	31	0.29 314	9.94 994	6	1	
60	9.65 704	24	9.70 716	31	0.29 283	9.94 988	6	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

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**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

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'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot	Log. Cos.	d.		P. P.
0	9.65 704	25	9.70 716	31	0.29 283	9.94 988	60		
1	9.65 729	24	9.70 748	31	0.29 252	9.94 981	59		
2	9.65 754	24	9.70 779	31	0.29 221	9.94 975	58		
3	9.65 779	25	9.70 810	31	0.29 190	9.94 969	57		
4	9.65 803	24	9.70 841	31	0.29 158	9.94 962	56		
5	9.65 828	25	9.70 872	31	0.29 127	9.94 956	55		
6	9.65 853	24	9.70 903	31	0.29 096	9.94 949	54		
7	9.65 878	25	9.70 935	31	0.29 065	9.94 943	53		
8	9.65 902	24	9.70 966	31	0.29 034	9.94 936	52		
9	9.65 927	24	9.70 997	31	0.29 003	9.94 930	51		
10	9.65 951	25	9.71 028	31	0.28 972	9.94 923	50		
11	9.65 976	24	9.71 059	31	0.28 940	9.94 917	49		
12	9.66 001	24	9.71 090	31	0.28 909	9.94 910	48		
13	9.66 025	24	9.71 121	31	0.28 878	9.94 904	47		
14	9.66 050	24	9.71 152	31	0.28 847	9.94 897	46		
15	9.66 074	24	9.71 183	31	0.28 816	9.94 891	45		
16	9.66 099	24	9.71 214	31	0.28 785	9.94 884	44		
17	9.66 123	24	9.71 245	31	0.28 754	9.94 878	43		
18	9.66 148	24	9.71 276	31	0.28 723	9.94 871	42		
19	9.66 172	24	9.71 307	31	0.28 692	9.94 865	41		
20	9.66 197	24	9.71 338	31	0.28 661	9.94 858	40		
21	9.66 221	24	9.71 369	31	0.28 630	9.94 852	39		
22	9.66 246	24	9.71 400	31	0.28 599	9.94 845	38		
23	9.66 270	24	9.71 431	31	0.28 568	9.94 839	37		
24	9.66 294	24	9.71 462	31	0.28 537	9.94 832	36		
25	9.66 319	24	9.71 493	30	0.28 506	9.94 825	35		
26	9.66 343	24	9.71 524	31	0.28 476	9.94 819	34		
27	9.66 367	24	9.71 555	31	0.28 445	9.94 812	33		
28	9.66 392	24	9.71 586	31	0.28 414	9.94 806	32		
29	9.66 416	24	9.71 617	31	0.28 383	9.94 799	31		
30	9.66 440	24	9.71 647	30	0.28 352	9.94 793	30		
31	9.66 465	24	9.71 678	31	0.28 321	9.94 786	29		
32	9.66 489	24	9.71 709	30	0.28 290	9.94 779	28		
33	9.66 513	24	9.71 740	31	0.28 260	9.94 773	27		
34	9.66 537	24	9.71 771	31	0.28 229	9.94 766	26		
35	9.66 561	24	9.71 801	30	0.28 198	9.94 760	25		
36	9.66 586	24	9.71 832	31	0.28 167	9.94 753	24		
37	9.66 610	24	9.71 863	30	0.28 136	9.94 746	23		
38	9.66 634	24	9.71 894	31	0.28 106	9.94 740	22		
39	9.66 658	24	9.71 925	30	0.28 075	9.94 733	21		
40	9.66 682	24	9.71 955	30	0.28 044	9.94 727	20		
41	9.66 706	24	9.71 986	31	0.28 014	9.94 720	19		
42	9.66 730	24	9.72 017	30	0.27 983	9.94 713	18		
43	9.66 754	24	9.72 047	31	0.27 952	9.94 707	17		
44	9.66 778	24	9.72 078	31	0.27 921	9.94 700	16		
45	9.66 802	24	9.72 109	30	0.27 891	9.94 693	15		
46	9.66 826	24	9.72 139	30	0.27 860	9.94 687	14		
47	9.66 850	24	9.72 170	31	0.27 830	9.94 680	13		
48	9.66 874	24	9.72 201	30	0.27 799	9.94 674	12		
49	9.66 898	24	9.72 231	30	0.27 768	9.94 667	11		
50	9.66 922	24	9.72 262	30	0.27 738	9.94 660	10		
51	9.66 946	24	9.72 292	30	0.27 707	9.94 654	9		
52	9.66 970	23	9.72 323	30	0.27 677	9.94 647	8		
53	9.66 994	23	9.72 354	30	0.27 646	9.94 640	7		
54	9.67 018	24	9.72 384	30	0.27 615	9.94 633	6		
55	9.67 042	24	9.72 415	30	0.27 585	9.94 627	5		
56	9.67 066	23	9.72 445	30	0.27 554	9.94 620	4		
57	9.67 089	24	9.72 476	30	0.27 524	9.94 613	3		
58	9.67 113	23	9.72 506	30	0.27 493	9.94 607	2		
59	9.67 137	23	9.72 537	30	0.27 463	9.94 600	1		
60	9.67 161	24	9.72 567	30	0.27 432	9.94 593	0		
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

31 **31** **30**
 6 3.1 3.1 3.0
 7 3.7 3.6 3.5
 8 4.2 4.1 4.0
 9 4.7 4.6 4.6
 10 5.2 5.1 5.1
 20 10.5 10.3 10.1
 30 15.7 15.5 15.2
 40 21.0 20.6 20.3
 50 26.2 25.8 25.4

25
 6 2.5
 7 2.9
 8 3.3
 9 3.7
 10 4.1
 20 8.3
 30 12.5
 40 16.6
 50 20.8

24 **24** **23**
 6 2.4 2.4 2.3
 7 2.8 2.8 2.7
 8 3.2 3.2 3.1
 9 3.7 3.6 3.5
 10 4.1 4.0 3.9
 20 8.1 8.0 7.8
 30 12.2 12.0 11.7
 40 16.3 16.0 15.6
 50 20.4 20.0 19.6

7 **6** **6**
 6 0.7 0.6 0.6
 7 0.8 0.7 0.7
 8 0.9 0.8 0.8
 9 1.0 1.0 0.9
 10 1.1 1.1 1.0
 20 2.3 2.1 2.0
 30 3.5 3.2 3.0
 40 4.6 4.3 4.0
 50 5.8 5.4 5.0

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.67 161	23	9.72 567	30	0.27 432	9.94 593	60	
1	9.67 184	24	9.72 598	30	0.27 402	9.94 587	59	
2	9.67 208	23	9.72 628	30	0.27 371	9.94 580	58	
3	9.67 232	24	9.72 659	30	0.27 341	9.94 573	57	
4	9.67 256	23	9.72 689	30	0.27 311	9.94 566	56	
5	9.67 279	23	9.72 719	30	0.27 280	9.94 560	55	30 3.0 2.9
6	9.67 303	24	9.72 750	30	0.27 250	9.94 553	54	6 3.0 3.0
7	9.67 327	23	9.72 780	30	0.27 219	9.94 546	53	7 3.5 3.4
8	9.67 350	23	9.72 811	30	0.27 189	9.94 539	52	8 4.0 4.0
9	9.67 374	23	9.72 841	30	0.27 159	9.94 533	51	9 4.6 4.5
10	9.67 397	23	9.72 871	30	0.27 128	9.94 526	50	10 5.1 5.0
11	9.67 421	23	9.72 902	30	0.27 098	9.94 519	49	20 10.1 10.0
12	9.67 445	23	9.72 932	30	0.27 067	9.94 512	48	30 15.2 15.0
13	9.67 468	23	9.72 962	30	0.27 037	9.94 506	47	40 20.3 20.0
14	9.67 492	23	9.72 993	30	0.27 007	9.94 499	46	50 25.4 25.0
15	9.67 515	23	9.73 023	30	0.26 976	9.94 492	45	
16	9.67 539	23	9.73 053	30	0.26 946	9.94 485	44	
17	9.67 562	23	9.73 084	30	0.26 916	9.94 478	43	
18	9.67 586	23	9.73 114	30	0.26 886	9.94 472	42	
19	9.67 609	23	9.73 144	30	0.26 855	9.94 465	41	
20	9.67 633	23	9.73 174	30	0.26 825	9.94 458	40	24 2.4
21	9.67 656	23	9.73 205	30	0.26 795	9.94 451	39	6 2.8
22	9.67 679	23	9.73 235	30	0.26 765	9.94 444	38	7 3.2
23	9.67 703	23	9.73 265	30	0.26 734	9.94 437	37	8 3.6
24	9.67 726	23	9.73 295	30	0.26 704	9.94 431	36	9 4.0
25	9.67 750	23	9.73 325	30	0.26 674	9.94 424	35	10 8.0
26	9.67 773	23	9.73 356	30	0.26 644	9.94 417	34	20 12.0
27	9.67 796	23	9.73 386	30	0.26 614	9.94 410	33	30 16.0
28	9.67 819	23	9.73 416	30	0.26 584	9.94 403	32	40 20.0
29	9.67 843	23	9.73 446	30	0.26 553	9.94 396	31	50
30	9.67 866	23	9.73 476	30	0.26 523	9.94 390	30	
31	9.67 889	23	9.73 506	30	0.26 493	9.94 383	29	
32	9.67 913	23	9.73 536	30	0.26 463	9.94 376	28	
33	9.67 936	23	9.73 567	30	0.26 433	9.94 369	27	
34	9.67 959	23	9.73 597	30	0.26 403	9.94 362	26	23 2.3 2.3
35	9.67 982	23	9.73 627	30	0.26 373	9.94 355	25	6 2.7 2.6
36	9.68 005	23	9.73 657	30	0.26 343	9.94 348	24	7 3.0 3.0
37	9.68 029	23	9.73 687	30	0.26 313	9.94 341	23	8 3.5 3.4
38	9.68 052	23	9.73 717	30	0.26 283	9.94 335	22	9 3.8 3.7
39	9.68 075	23	9.73 747	30	0.26 253	9.94 328	21	10 7.6 7.5
40	9.68 098	23	9.73 777	30	0.26 223	9.94 321	20	20 11.7 11.2
41	9.68 121	23	9.73 807	30	0.26 193	9.94 314	19	30 15.6 15.3
42	9.68 144	23	9.73 837	30	0.26 163	9.94 307	18	40 19.6 19.1
43	9.68 167	23	9.73 867	30	0.26 133	9.94 300	17	50
44	9.68 190	23	9.73 897	30	0.26 103	9.94 293	16	
45	9.68 213	23	9.73 927	30	0.26 073	9.94 286	15	
46	9.68 236	23	9.73 957	30	0.26 043	9.94 279	14	
47	9.68 259	23	9.73 987	30	0.26 013	9.94 272	13	
48	9.68 282	23	9.74 017	30	0.25 983	9.94 265	12	7 6 6
49	9.68 305	23	9.74 047	30	0.25 953	9.94 258	11	6 0.7 0.6
50	9.68 328	23	9.74 076	29	0.25 923	9.94 251	10	7 0.8 0.7
51	9.68 351	23	9.74 106	30	0.25 893	9.94 245	9	8 0.9 0.8
52	9.68 374	22	9.74 136	30	0.25 863	9.94 238	8	9 1.0 1.0
53	9.68 397	23	9.74 166	30	0.25 833	9.94 231	7	10 1.1 1.1
54	9.68 420	23	9.74 196	29	0.25 804	9.94 224	6	20 2.3 2.2
55	9.68 443	23	9.74 226	30	0.25 774	9.94 217	5	30 3.5 3.3
56	9.68 466	23	9.74 256	30	0.25 744	9.94 210	4	40 4.6 4.3
57	9.68 488	23	9.74 286	29	0.25 714	9.94 203	3	50 5.8 5.4
58	9.68 511	23	9.74 315	30	0.25 684	9.94 196	2	
59	9.68 534	23	9.74 345	30	0.25 654	9.94 189	1	
60	9.68 557	22	9.74 375	29	0.25 625	9.94 182	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.68 557	23	9.74 379	30	0.25 625	9.94 182	7	60
1	9.68 580	22	9.74 405	29	0.25 595	9.94 175	7	59
2	9.68 602	22	9.74 435	29	0.25 565	9.94 168	7	58
3	9.68 625	22	9.74 464	29	0.25 535	9.94 161	7	57
4	9.68 648	22	9.74 494	29	0.25 505	9.94 154	7	56
5	9.68 671	23	9.74 524	29	0.25 476	9.94 147	7	55
6	9.68 693	22	9.74 554	29	0.25 446	9.94 140	7	54
7	9.68 716	22	9.74 583	29	0.25 416	9.94 133	7	53
8	9.68 739	22	9.74 613	29	0.25 387	9.94 126	7	52
9	9.68 761	22	9.74 643	30	0.25 357	9.94 118	7	51
10	9.68 784	23	9.74 672	29	0.25 327	9.94 111	7	50
11	9.68 807	22	9.74 702	29	0.25 297	9.94 104	7	49
12	9.68 829	22	9.74 732	29	0.25 268	9.94 097	7	48
13	9.68 852	22	9.74 761	29	0.25 238	9.94 090	7	47
14	9.68 874	22	9.74 791	30	0.25 208	9.94 083	7	46
15	9.68 897	22	9.74 821	29	0.25 179	9.94 076	7	45
16	9.68 920	23	9.74 850	29	0.25 149	9.94 069	7	44
17	9.68 942	22	9.74 880	29	0.25 120	9.94 062	7	43
18	9.68 965	22	9.74 909	29	0.25 090	9.94 055	7	42
19	9.68 987	22	9.74 939	30	0.25 060	9.94 048	7	41
20	9.69 010	22	9.74 969	29	0.25 031	9.94 041	7	40
21	9.69 032	22	9.74 998	29	0.25 001	9.94 034	7	39
22	9.69 055	22	9.75 028	29	0.24 972	9.94 026	7	38
23	9.69 077	22	9.75 057	29	0.24 942	9.94 019	7	37
24	9.69 099	22	9.75 087	29	0.24 913	9.94 012	7	36
25	9.69 122	22	9.75 116	29	0.24 883	9.94 005	7	35
26	9.69 144	22	9.75 146	29	0.24 854	9.93 998	7	34
27	9.69 167	22	9.75 175	29	0.24 824	9.93 991	7	33
28	9.69 189	22	9.75 205	29	0.24 795	9.93 984	7	32
29	9.69 211	22	9.75 234	29	0.24 765	9.93 977	7	31
30	9.69 234	22	9.75 264	29	0.24 736	9.93 969	7	30
31	9.69 256	22	9.75 293	29	0.24 706	9.93 962	7	29
32	9.69 278	22	9.75 323	29	0.24 677	9.93 955	7	28
33	9.69 301	22	9.75 352	29	0.24 647	9.93 948	7	27
34	9.69 323	22	9.75 382	29	0.24 618	9.93 941	7	26
35	9.69 345	22	9.75 411	29	0.24 588	9.93 934	7	25
36	9.69 367	22	9.75 441	29	0.24 559	9.93 926	7	24
37	9.69 390	22	9.75 470	29	0.24 529	9.93 919	7	23
38	9.69 412	22	9.75 499	29	0.24 500	9.93 912	7	22
39	9.69 434	22	9.75 529	29	0.24 471	9.93 905	7	21
40	9.69 456	22	9.75 558	29	0.24 441	9.93 898	7	20
41	9.69 478	22	9.75 588	29	0.24 412	9.93 891	7	19
42	9.69 500	22	9.75 617	29	0.24 383	9.93 883	7	18
43	9.69 523	22	9.75 646	29	0.24 353	9.93 876	7	17
44	9.69 545	22	9.75 676	29	0.24 324	9.93 869	7	16
45	9.69 567	22	9.75 705	29	0.24 295	9.93 862	7	15
46	9.69 589	22	9.75 734	29	0.24 265	9.93 854	7	14
47	9.69 611	22	9.75 764	29	0.24 236	9.93 847	7	13
48	9.69 633	22	9.75 793	29	0.24 207	9.93 840	7	12
49	9.69 655	22	9.75 822	29	0.24 177	9.93 833	7	11
50	9.69 677	22	9.75 851	29	0.24 148	9.93 826	7	10
51	9.69 699	22	9.75 881	29	0.24 119	9.93 818	7	9
52	9.69 721	22	9.75 910	29	0.24 090	9.93 811	7	8
53	9.69 743	22	9.75 939	29	0.24 060	9.93 804	7	7
54	9.69 765	22	9.75 968	29	0.24 031	9.93 796	7	6
55	9.69 787	22	9.75 998	29	0.24 002	9.93 789	7	5
56	9.69 809	22	9.76 027	29	0.23 973	9.93 782	7	4
57	9.69 831	22	9.76 056	29	0.23 943	9.93 775	7	3
58	9.69 853	21	9.76 085	29	0.23 914	9.93 767	7	2
59	9.69 875	22	9.76 115	29	0.23 885	9.93 760	7	1
60	9.69 897	22	9.76 144	29	0.23 856	9.93 753	7	0
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.

	30	29	29
6	3.0	2.9	2.9
7	8.5	3.4	3.4
8	4.0	3.9	3.8
9	4.5	4.4	4.3
10	5.0	4.9	4.8
20	10.0	9.8	9.6
30	15.0	14.7	14.5
40	20.0	19.6	19.3
50	25.0	24.6	24.1

	23
6	2.3
7	2.7
8	3.0
9	3.4
10	3.8
20	7.6
30	11.5
40	15.3
50	19.1

	22	22	21
6	2.2	2.2	2.1
7	2.6	2.5	2.5
8	3.0	2.9	2.8
9	3.4	3.3	3.2
10	3.7	3.6	3.6
20	7.5	7.3	7.1
30	11.2	11.0	10.7
40	15.0	14.6	14.3
50	18.7	18.3	17.9

	7	7
6	0.7	0.7
7	0.9	0.8
8	1.0	0.9
9	1.1	1.0
10	1.2	1.1
20	2.5	2.3
30	3.7	3.5
40	5.0	4.6
50	6.2	5.8

TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.		
0	9.69 897		9.76 144		0.23 856	9.93 753	60			
1	9.69 919	22	9.76 173	29	0.23 827	9.93 746	59			
2	9.69 940	21	9.76 202	29	0.23 797	9.93 738	58			
3	9.69 962	22	9.76 231	29	0.23 768	9.93 731	57			
4	9.69 984	22	9.76 260	29	0.23 739	9.93 724	56			
5	9.70 006	21	9.76 289	29	0.23 710	9.93 716	55			
6	9.70 028	22	9.76 319	29	0.23 681	9.93 709	54			
7	9.70 050	22	9.76 348	29	0.23 652	9.93 702	53			
8	9.70 071	21	9.76 377	29	0.23 623	9.93 694	52			
9	9.70 093	22	9.76 406	29	0.23 594	9.93 687	51			
10	9.70 115	21	9.76 435	29	0.23 565	9.93 680	50			
11	9.70 137	22	9.76 464	29	0.23 535	9.93 672	49			
12	9.70 158	21	9.76 493	29	0.23 506	9.93 665	48			
13	9.70 180	22	9.76 522	29	0.23 477	9.93 658	47			
14	9.70 202	22	9.76 551	29	0.23 448	9.93 650	46			
15	9.70 223	21	9.76 580	29	0.23 419	9.93 643	45			
16	9.70 245	21	9.76 609	29	0.23 390	9.93 635	44			
17	9.70 267	22	9.76 638	29	0.23 361	9.93 628	43			
18	9.70 288	21	9.76 667	29	0.23 332	9.93 621	42			
19	9.70 310	21	9.76 696	29	0.23 303	9.93 613	41			
20	9.70 331	22	9.76 725	29	0.23 274	9.93 606	40			
21	9.70 353	22	9.76 754	29	0.23 245	9.93 599	39			
22	9.70 375	21	9.76 783	29	0.23 216	9.93 591	38			
23	9.70 396	21	9.76 812	29	0.23 187	9.93 584	37			
24	9.70 418	21	9.76 841	29	0.23 158	9.93 576	36			
25	9.70 439	21	9.76 870	29	0.23 129	9.93 569	35			
26	9.70 461	21	9.76 899	29	0.23 101	9.93 562	34			
27	9.70 482	22	9.76 928	29	0.23 072	9.93 554	33			
28	9.70 504	21	9.76 957	29	0.23 043	9.93 547	32			
29	9.70 525	21	9.76 986	29	0.23 014	9.93 539	31			
30	9.70 547	21	9.77 015	29	0.22 985	9.93 532	30			
31	9.70 568	21	9.77 043	29	0.22 956	9.93 524	29			
32	9.70 590	21	9.77 072	29	0.22 927	9.93 517	28			
33	9.70 611	21	9.77 101	29	0.22 898	9.93 509	27			
34	9.70 632	21	9.77 130	29	0.22 869	9.93 502	26			
35	9.70 654	21	9.77 159	29	0.22 841	9.93 495	25			
36	9.70 675	21	9.77 188	29	0.22 812	9.93 487	24			
37	9.70 696	21	9.77 217	29	0.22 783	9.93 480	23			
38	9.70 718	21	9.77 245	29	0.22 754	9.93 472	22			
39	9.70 739	21	9.77 274	29	0.22 725	9.93 465	21			
40	9.70 760	21	9.77 303	29	0.22 696	9.93 457	20			
41	9.70 782	21	9.77 332	29	0.22 668	9.93 450	19			
42	9.70 803	21	9.77 361	29	0.22 639	9.93 442	18			
43	9.70 824	21	9.77 389	29	0.22 610	9.93 435	17			
44	9.70 846	21	9.77 418	29	0.22 581	9.93 427	16			
45	9.70 867	21	9.77 447	29	0.22 553	9.93 420	15			
46	9.70 888	21	9.77 476	29	0.22 524	9.93 412	14			
47	9.70 909	21	9.77 504	29	0.22 495	9.93 405	13			
48	9.70 930	21	9.77 533	29	0.22 466	9.93 397	12			
49	9.70 952	21	9.77 562	29	0.22 438	9.93 390	11			
50	9.70 973	21	9.77 591	29	0.22 409	9.93 382	10			
51	9.70 994	21	9.77 619	29	0.22 380	9.93 374	9			
52	9.71 015	21	9.77 648	29	0.22 352	9.93 367	8			
53	9.71 036	21	9.77 677	29	0.22 323	9.93 359	7			
54	9.71 057	21	9.77 705	29	0.22 294	9.93 352	6			
55	9.71 078	21	9.77 734	29	0.22 266	9.93 344	5			
56	9.71 099	21	9.77 763	29	0.22 237	9.93 337	4			
57	9.71 121	21	9.77 791	29	0.22 208	9.93 329	3			
58	9.71 142	21	9.77 820	29	0.22 180	9.93 321	2			
59	9.71 163	21	9.77 849	29	0.22 151	9.93 314	1			
60	9.71 184	21	9.77 877	29	0.22 122	9.93 306	0			
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.		

	29	29	28
6	2.9	2.9	2.8
7	3.4	3.4	3.3
8	3.9	3.8	3.8
9	4.4	4.3	4.3
10	4.9	4.8	4.7
20	9.8	9.6	9.5
30	14.7	14.5	14.2
40	19.6	19.3	19.0
50	24.6	24.1	23.7
	22	21	21
6	2.2	2.1	2.1
7	2.5	2.5	2.4
8	2.9	2.8	2.8
9	3.3	3.2	3.1
10	3.6	3.6	3.5
20	7.3	7.1	7.0
30	11.0	10.7	10.5
40	14.6	14.3	14.0
50	18.3	17.9	17.5
	8	7	7
6	0.8	0.7	0.7
7	0.9	0.9	0.8
8	1.0	1.0	0.9
9	1.2	1.1	1.0
10	1.3	1.2	1.1
20	2.6	2.5	2.3
30	4.0	3.7	3.5
40	5.3	5.0	4.6
50	6.6	6.2	5.8

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS.

31°

AND COTANGENTS.

148°

	Log. Sin.	d.	Log. Tan.	c.d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.71 184	21	9.77 877	28	0.22 122	9.93 306	7	60
1	9.71 205	21	9.77 906	28	0.22 094	9.93 299	7	59
2	9.71 226	21	9.77 934	28	0.22 065	9.93 291	7	58
3	9.71 247	21	9.77 963	28	0.22 037	9.93 284	7	57
4	9.71 268	21	9.77 992	29	0.22 008	9.93 276	8	56
5	9.71 289	21	9.78 020	28	0.21 979	9.93 268	7	55
6	9.71 310	21	9.78 049	28	0.21 951	9.93 261	7	54
7	9.71 331	21	9.78 077	28	0.21 922	9.93 253	7	53
8	9.71 351	20	9.78 106	28	0.21 894	9.93 245	8	52
9	9.71 372	21	9.78 134	28	0.21 865	9.93 238	7	51
10	9.71 393	21	9.78 163	28	0.21 837	9.93 230	7	50
11	9.71 414	21	9.78 191	28	0.21 808	9.93 223	7	49
12	9.71 435	20	9.78 220	28	0.21 780	9.93 215	8	48
13	9.71 456	21	9.78 248	28	0.21 751	9.93 207	7	47
14	9.71 477	21	9.78 277	28	0.21 723	9.93 200	7	46
15	9.71 498	21	9.78 305	28	0.21 694	9.93 192	8	45
16	9.71 518	21	9.78 334	28	0.21 666	9.93 184	7	44
17	9.71 539	20	9.78 362	28	0.21 637	9.93 177	7	43
18	9.71 560	21	9.78 391	28	0.21 609	9.93 169	8	42
19	9.71 581	20	9.78 419	28	0.21 580	9.93 161	7	41
20	9.71 601	21	9.78 448	28	0.21 552	9.93 153	8	40
21	9.71 622	21	9.78 476	28	0.21 523	9.93 146	7	39
22	9.71 643	21	9.78 505	28	0.21 495	9.93 138	7	38
23	9.71 664	20	9.78 533	28	0.21 467	9.93 130	8	37
24	9.71 684	21	9.78 561	28	0.21 438	9.93 123	7	36
25	9.71 705	21	9.78 590	28	0.21 410	9.93 115	8	35
26	9.71 726	20	9.78 618	28	0.21 381	9.93 107	7	34
27	9.71 746	21	9.78 647	28	0.21 353	9.93 100	7	33
28	9.71 767	20	9.78 675	28	0.21 325	9.93 092	8	32
29	9.71 788	21	9.78 703	28	0.21 296	9.93 084	7	31
30	9.71 808	20	9.78 732	28	0.21 268	9.93 076	8	30
31	9.71 829	21	9.78 760	28	0.21 239	9.93 069	7	29
32	9.71 849	21	9.78 788	28	0.21 211	9.93 061	8	28
33	9.71 870	20	9.78 817	28	0.21 183	9.93 053	7	27
34	9.71 891	21	9.78 845	28	0.21 154	9.93 045	8	26
35	9.71 911	20	9.78 873	28	0.21 126	9.93 038	7	25
36	9.71 932	21	9.78 902	28	0.21 098	9.93 030	8	24
37	9.71 952	20	9.78 930	28	0.21 070	9.93 022	7	23
38	9.71 973	21	9.78 958	28	0.21 041	9.93 014	8	22
39	9.71 993	20	9.78 987	28	0.21 013	9.93 006	7	21
40	9.72 014	21	9.79 015	28	0.20 985	9.92 999	8	20
41	9.72 034	20	9.79 043	28	0.20 956	9.92 991	7	19
42	9.72 055	21	9.79 071	28	0.20 928	9.92 983	8	18
43	9.72 075	20	9.79 100	28	0.20 900	9.92 975	7	17
44	9.72 096	21	9.79 128	28	0.20 872	9.92 967	8	16
45	9.72 116	20	9.79 156	28	0.20 843	9.92 960	7	15
46	9.72 136	21	9.79 184	28	0.20 815	9.92 952	8	14
47	9.72 157	20	9.79 213	28	0.20 787	9.92 944	7	13
48	9.72 177	21	9.79 241	28	0.20 759	9.92 936	8	12
49	9.72 198	20	9.79 269	28	0.20 731	9.92 928	7	11
50	9.72 218	21	9.79 297	28	0.20 702	9.92 920	8	10
51	9.72 238	20	9.79 325	28	0.20 674	9.92 913	7	9
52	9.72 259	21	9.79 354	28	0.20 646	9.92 905	8	8
53	9.72 279	20	9.79 382	28	0.20 618	9.92 897	7	7
54	9.72 299	21	9.79 410	28	0.20 590	9.92 889	8	6
55	9.72 319	20	9.79 438	28	0.20 561	9.92 881	7	5
56	9.72 340	21	9.79 466	28	0.20 533	9.92 873	8	4
57	9.72 360	20	9.79 494	28	0.20 505	9.92 865	7	3
58	9.72 380	21	9.79 522	28	0.20 477	9.92 858	8	2
59	9.72 400	20	9.79 551	28	0.20 449	9.92 850	7	1
60	9.72 421	21	9.79 579	28	0.20 421	9.92 842	8	0
	Log. Cos.	d.	Log. Cot.	c.d.	Log. Tan.	Log. Sin.	d.	P. P.

	29	28	28
6	2.9	2.8	2.8
7	3.4	3.3	3.2
8	3.8	3.8	3.7
9	4.3	4.3	4.2
10	4.8	4.7	4.6
20	9.6	9.5	9.3
30	14.5	14.2	14.0
40	19.3	19.0	18.6
50	24.1	23.7	23.3

	21	20	20
6	2.1	2.0	2.0
7	2.4	2.4	2.3
8	2.8	2.7	2.6
9	3.1	3.1	3.0
10	3.5	3.4	3.3
20	7.0	6.8	6.6
30	10.5	10.2	10.0
40	14.0	13.6	13.3
50	17.5	17.1	16.6

	8	7
6	10.8	10.7
7	10.9	10.9
8	1.0	1.0
9	1.2	1.1
10	1.3	1.2
20	2.6	2.5
30	4.0	3.7
40	5.3	5.0
50	6.6	6.2

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.
0	9.72 421	20	9.79 579	28	0.20 421	9.92 842	8
1	9.72 441	20	9.79 607	28	0.20 393	9.92 834	7
2	9.72 461	20	9.79 635	28	0.20 365	9.92 826	8
3	9.72 481	20	9.79 663	28	0.20 337	9.92 818	8
4	9.72 501	20	9.79 691	28	0.20 308	9.92 810	8
5	9.72 522	20	9.79 719	28	0.20 280	9.92 802	8
6	9.72 542	20	9.79 747	28	0.20 252	9.92 794	8
7	9.72 562	20	9.79 775	28	0.20 224	9.92 786	8
8	9.72 582	20	9.79 803	28	0.20 196	9.92 778	8
9	9.72 602	20	9.79 831	28	0.20 168	9.92 771	7
10	9.72 622	20	9.79 859	28	0.20 140	9.92 763	8
11	9.72 642	20	9.79 887	28	0.20 112	9.92 755	8
12	9.72 662	20	9.79 915	28	0.20 084	9.92 747	8
13	9.72 682	20	9.79 943	28	0.20 056	9.92 739	8
14	9.72 702	20	9.79 971	28	0.20 028	9.92 731	8
15	9.72 723	20	9.79 999	28	0.20 000	9.92 723	8
16	9.72 743	20	9.80 027	28	0.19 972	9.92 715	8
17	9.72 763	20	9.80 055	28	0.19 944	9.92 707	8
18	9.72 783	19	9.80 083	28	0.19 916	9.92 699	8
19	9.72 802	20	9.80 111	28	0.19 888	9.92 691	8
20	9.72 822	20	9.80 139	28	0.19 860	9.92 683	8
21	9.72 842	20	9.80 167	28	0.19 832	9.92 675	8
22	9.72 862	20	9.80 195	28	0.19 804	9.92 667	8
23	9.72 882	20	9.80 223	28	0.19 776	9.92 659	8
24	9.72 902	20	9.80 251	28	0.19 748	9.92 651	8
25	9.72 922	19	9.80 279	27	0.19 721	9.92 643	8
26	9.72 942	20	9.80 307	28	0.19 693	9.92 635	8
27	9.72 962	20	9.80 335	28	0.19 665	9.92 627	8
28	9.72 982	20	9.80 363	28	0.19 637	9.92 619	8
29	9.73 002	20	9.80 391	28	0.19 609	9.92 611	8
30	9.73 021	19	9.80 418	27	0.19 581	9.92 603	8
31	9.73 041	20	9.80 446	28	0.19 553	9.92 595	8
32	9.73 061	20	9.80 474	28	0.19 525	9.92 587	8
33	9.73 081	19	9.80 502	28	0.19 497	9.92 579	8
34	9.73 101	20	9.80 530	27	0.19 470	9.92 570	8
35	9.73 120	19	9.80 558	28	0.19 442	9.92 562	8
36	9.73 140	20	9.80 586	28	0.19 414	9.92 554	8
37	9.73 160	19	9.80 613	27	0.19 386	9.92 546	8
38	9.73 180	20	9.80 641	28	0.19 358	9.92 538	8
39	9.73 199	19	9.80 669	28	0.19 330	9.92 530	8
40	9.73 219	20	9.80 697	27	0.19 303	9.92 522	8
41	9.73 239	19	9.80 725	28	0.19 275	9.92 514	8
42	9.73 258	19	9.80 752	27	0.19 247	9.92 506	8
43	9.73 278	20	9.80 780	28	0.19 219	9.92 498	8
44	9.73 298	19	9.80 808	28	0.19 191	9.92 489	8
45	9.73 317	19	9.80 836	27	0.19 164	9.92 481	8
46	9.73 337	20	9.80 864	28	0.19 136	9.92 473	8
47	9.73 357	19	9.80 891	27	0.19 108	9.92 465	8
48	9.73 376	19	9.80 919	28	0.19 080	9.92 457	8
49	9.73 396	19	9.80 947	27	0.19 053	9.92 449	8
50	9.73 415	19	9.80 975	28	0.19 025	9.92 441	8
51	9.73 435	20	9.81 002	27	0.18 997	9.92 433	8
52	9.73 455	19	9.81 030	27	0.18 970	9.92 424	8
53	9.73 474	19	9.81 058	27	0.18 942	9.92 416	8
54	9.73 494	19	9.81 085	27	0.18 914	9.92 408	8
55	9.73 513	19	9.81 113	28	0.18 886	9.92 400	8
56	9.73 533	19	9.81 141	27	0.18 859	9.92 392	8
57	9.73 552	19	9.81 168	27	0.18 831	9.92 383	8
58	9.73 572	19	9.81 196	28	0.18 803	9.92 375	8
59	9.73 591	19	9.81 224	27	0.18 776	9.92 367	8
60	9.73 611	19	9.81 251	27	0.18 748	9.92 359	8
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.

P. P.

	28	28	27
6	2.8	2.8	2.7
7	3.3	3.2	3.2
8	3.8	3.7	3.6
9	4.3	4.2	4.1
10	4.7	4.6	4.6
20	9.5	9.3	9.1
30	14.2	14.0	13.7
40	19.0	18.6	18.3
50	23.7	23.3	22.9

	20	20	19
6	2.0	2.0	1.9
7	2.4	2.3	2.3
8	2.7	2.6	2.6
9	3.1	3.0	2.9
10	3.4	3.3	3.2
20	6.8	6.6	6.5
30	10.2	10.0	9.7
40	13.6	13.3	13.0
50	17.1	16.6	16.2

	8	8	7
6	0.8	0.8	0.7
7	1.0	0.9	0.9
8	1.1	1.0	1.0
9	1.3	1.2	1.1
10	1.4	1.3	1.2
20	2.8	2.6	2.5
30	4.4	4.0	3.7
40	5.8	5.3	5.0
50	7.1	6.6	6.2

P. P.

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.73 611	19	9.81 251	28	0.18 748	9.92 359	8	60	
1	9.73 630	19	9.81 279	27	0.18 720	9.92 351	8	59	
2	9.73 650	19	9.81 307	27	0.18 693	9.92 342	8	58	
3	9.73 669	19	9.81 334	28	0.18 665	9.92 334	8	57	
4	9.73 688	19	9.81 362	27	0.18 637	9.92 326	8	56	
5	9.73 708	19	9.81 390	27	0.18 610	9.92 318	8	55	
6	9.73 727	19	9.81 417	27	0.18 582	9.92 310	8	54	
7	9.73 746	19	9.81 445	28	0.18 555	9.92 301	8	53	
8	9.73 766	19	9.81 473	27	0.18 527	9.92 293	8	52	
9	9.73 785	19	9.81 500	27	0.18 499	9.92 285	8	51	
10	9.73 805	19	9.81 528	27	0.18 472	9.92 277	8	50	
11	9.73 824	19	9.81 555	27	0.18 444	9.92 268	8	49	
12	9.73 843	19	9.81 583	27	0.18 417	9.92 260	8	48	
13	9.73 862	19	9.81 610	27	0.18 389	9.92 252	8	47	
14	9.73 882	19	9.81 638	28	0.18 362	9.92 244	8	46	
15	9.73 901	19	9.81 666	27	0.18 334	9.92 235	8	45	
16	9.73 920	19	9.81 693	27	0.18 306	9.92 227	8	44	
17	9.73 940	19	9.81 721	27	0.18 279	9.92 219	8	43	
18	9.73 959	19	9.81 748	27	0.18 251	9.92 210	8	42	
19	9.73 978	19	9.81 776	27	0.18 224	9.92 202	8	41	
20	9.73 997	19	9.81 803	27	0.18 196	9.92 194	8	40	
21	9.74 016	19	9.81 831	27	0.18 169	9.92 185	8	39	
22	9.74 036	19	9.81 858	27	0.18 141	9.92 177	8	38	
23	9.74 055	19	9.81 886	27	0.18 114	9.92 169	8	37	
24	9.74 074	19	9.81 913	27	0.18 086	9.92 160	8	36	
25	9.74 093	19	9.81 941	27	0.18 059	9.92 152	8	35	
26	9.74 112	19	9.81 968	27	0.18 031	9.92 144	8	34	
27	9.74 131	19	9.81 996	27	0.18 004	9.92 135	8	33	
28	9.74 151	19	9.82 023	27	0.17 976	9.92 127	8	32	
29	9.74 170	19	9.82 051	27	0.17 949	9.92 119	8	31	
30	9.74 189	19	9.82 078	27	0.17 921	9.92 110	8	30	
31	9.74 208	19	9.82 105	27	0.17 894	9.92 102	8	29	
32	9.74 227	19	9.82 133	27	0.17 867	9.92 094	8	28	
33	9.74 246	19	9.82 160	27	0.17 839	9.92 085	8	27	
34	9.74 265	19	9.82 188	27	0.17 812	9.92 077	8	26	
35	9.74 284	19	9.82 215	27	0.17 784	9.92 069	8	25	
36	9.74 303	19	9.82 243	27	0.17 757	9.92 060	8	24	
37	9.74 322	19	9.82 270	27	0.17 729	9.92 052	8	23	
38	9.74 341	18	9.82 297	27	0.17 702	9.92 043	8	22	
39	9.74 360	18	9.82 325	27	0.17 675	9.92 035	8	21	
40	9.74 379	19	9.82 352	27	0.17 647	9.92 027	8	20	
41	9.74 398	19	9.82 380	27	0.17 620	9.92 018	8	19	
42	9.74 417	19	9.82 407	27	0.17 593	9.92 010	8	18	
43	9.74 436	19	9.82 434	27	0.17 565	9.92 001	8	17	
44	9.74 455	19	9.82 462	27	0.17 538	9.91 993	8	16	
45	9.74 474	19	9.82 489	27	0.17 510	9.91 984	8	15	
46	9.74 493	18	9.82 516	27	0.17 483	9.91 976	8	14	
47	9.74 511	18	9.82 544	27	0.17 456	9.91 967	8	13	
48	9.74 530	19	9.82 571	27	0.17 428	9.91 959	8	12	
49	9.74 549	19	9.82 598	27	0.17 401	9.91 951	8	11	
50	9.74 568	19	9.82 626	27	0.17 374	9.91 942	8	10	
51	9.74 587	18	9.82 653	27	0.17 347	9.91 934	8	9	
52	9.74 606	19	9.82 680	27	0.17 319	9.91 925	8	8	
53	9.74 625	18	9.82 708	27	0.17 292	9.91 917	8	7	
54	9.74 643	18	9.82 735	27	0.17 265	9.91 908	8	6	
55	9.74 662	19	9.82 762	27	0.17 237	9.91 900	8	5	
56	9.74 681	18	9.82 789	27	0.17 210	9.91 891	8	4	
57	9.74 700	19	9.82 817	27	0.17 183	9.91 883	8	3	
58	9.74 718	18	9.82 844	27	0.17 156	9.91 874	8	2	
59	9.74 737	18	9.82 871	27	0.17 128	9.91 866	8	1	
60	9.74 756	18	9.82 898	27	0.17 101	9.91 857	8	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

28	27	27
6	2.8	2.7
7	3.2	3.1
8	3.7	3.6
9	4.2	4.1
10	4.6	4.5
20	9.3	9.1
30	14.0	13.7
40	18.6	18.3
50	23.3	22.9

19	19	18
6	1.9	1.8
7	2.3	2.1
8	2.6	2.5
9	2.9	2.8
10	3.2	3.1
20	6.5	6.3
30	9.7	9.5
40	13.0	12.6
50	16.2	15.8

8	8
6	0.8
7	1.0
8	1.1
9	1.3
10	1.4
20	2.6
30	4.0
40	5.5
50	7.1

**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

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'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.74 756	19	9.82 898	27	0.17 101	9.91 857	60	
1	9.74 775	18	9.82 926	27	0.17 074	9.91 849	59	
2	9.74 793	18	9.82 953	27	0.17 047	9.91 840	58	
3	9.74 812	18	9.82 980	27	0.17 019	9.91 832	57	
4	9.74 831	18	9.83 007	27	0.16 992	9.91 823	56	
5	9.74 849	18	9.83 035	27	0.16 965	9.91 814	55	
6	9.74 868	18	9.83 062	27	0.16 938	9.91 806	54	
7	9.74 887	18	9.83 089	27	0.16 910	9.91 797	53	
8	9.74 905	18	9.83 116	27	0.16 883	9.91 789	52	
9	9.74 924	19	9.83 143	27	0.16 856	9.91 780	51	
10	9.74 943	18	9.83 171	27	0.16 829	9.91 772	50	
11	9.74 961	18	9.83 198	27	0.16 802	9.91 763	49	
12	9.74 980	18	9.83 225	27	0.16 774	9.91 755	48	
13	9.74 998	18	9.83 252	27	0.16 747	9.91 746	47	
14	9.75 017	18	9.83 279	27	0.16 720	9.91 737	46	
15	9.75 036	19	9.83 307	27	0.16 693	9.91 729	45	
16	9.75 054	18	9.83 334	27	0.16 666	9.91 720	44	
17	9.75 073	18	9.83 361	27	0.16 639	9.91 712	43	
18	9.75 091	18	9.83 388	27	0.16 612	9.91 703	42	
19	9.75 110	18	9.83 415	27	0.16 584	9.91 694	41	
20	9.75 128	18	9.83 442	27	0.16 557	9.91 686	40	
21	9.75 147	18	9.83 469	27	0.16 530	9.91 677	39	
22	9.75 165	18	9.83 496	27	0.16 503	9.91 668	38	
23	9.75 184	18	9.83 524	27	0.16 476	9.91 660	37	
24	9.75 202	18	9.83 551	27	0.16 449	9.91 651	36	
25	9.75 221	18	9.83 578	27	0.16 422	9.91 642	35	
26	9.75 239	18	9.83 605	27	0.16 395	9.91 634	34	
27	9.75 257	18	9.83 632	27	0.16 368	9.91 625	33	
28	9.75 276	18	9.83 659	27	0.16 340	9.91 616	32	
29	9.75 294	18	9.83 686	27	0.16 313	9.91 608	31	
30	9.75 313	18	9.83 713	27	0.16 286	9.91 599	30	
31	9.75 331	18	9.83 740	27	0.16 259	9.91 590	29	
32	9.75 349	18	9.83 767	27	0.16 232	9.91 582	28	
33	9.75 368	18	9.83 794	27	0.16 205	9.91 573	27	
34	9.75 386	18	9.83 821	27	0.16 178	9.91 564	26	
35	9.75 404	18	9.83 848	27	0.16 151	9.91 556	25	
36	9.75 423	18	9.83 875	27	0.16 124	9.91 547	24	
37	9.75 441	18	9.83 902	27	0.16 097	9.91 538	23	
38	9.75 459	18	9.83 929	27	0.16 070	9.91 529	22	
39	9.75 478	18	9.83 957	27	0.16 043	9.91 521	21	
40	9.75 496	18	9.83 984	27	0.16 016	9.91 512	20	
41	9.75 514	18	9.84 011	27	0.15 989	9.91 503	19	
42	9.75 532	18	9.84 038	27	0.15 962	9.91 495	18	
43	9.75 551	18	9.84 065	27	0.15 935	9.91 486	17	
44	9.75 569	18	9.84 091	26	0.15 908	9.91 477	16	
45	9.75 587	18	9.84 118	27	0.15 881	9.91 468	15	
46	9.75 605	18	9.84 145	27	0.15 854	9.91 460	14	
47	9.75 623	18	9.84 172	27	0.15 827	9.91 451	13	
48	9.75 642	18	9.84 199	27	0.15 800	9.91 442	12	
49	9.75 660	18	9.84 226	27	0.15 773	9.91 433	11	
50	9.75 678	18	9.84 253	27	0.15 746	9.91 424	10	
51	9.75 696	18	9.84 280	27	0.15 719	9.91 416	9	
52	9.75 714	18	9.84 307	27	0.15 692	9.91 407	8	
53	9.75 732	18	9.84 334	27	0.15 665	9.91 398	7	
54	9.75 750	18	9.84 361	26	0.15 639	9.91 389	6	
55	9.75 769	18	9.84 388	27	0.15 612	9.91 380	5	
56	9.75 787	18	9.84 415	27	0.15 585	9.91 372	4	
57	9.75 805	18	9.84 442	27	0.15 558	9.91 363	3	
58	9.75 823	18	9.84 469	27	0.15 531	9.91 354	2	
59	9.75 841	18	9.84 496	27	0.15 504	9.91 345	1	
60	9.75 859	18	9.84 522	26	0.15 477	9.91 336	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.

6	2.7	2.7	2.6
7	3.2	3.1	3.1
8	3.6	3.6	3.5
9	4.1	4.5	4.0
10	4.6	4.5	4.4
20	9.1	9.0	8.8
30	13.7	13.5	13.2
40	18.3	18.0	17.6
50	22.9	22.5	22.1

	19	18	18
6	1.9	1.8	1.8
7	2.2	2.1	2.1
8	2.5	2.4	2.4
9	2.8	2.8	2.7
10	3.1	3.1	3.0
20	6.3	6.1	6.0
30	9.5	9.2	9.0
40	12.6	12.3	12.0
50	15.8	15.4	15.0

	9	8
6	0.9	0.8
7	1.0	1.0
8	1.2	1.1
9	1.3	1.3
10	1.5	1.4
20	3.0	2.8
30	4.5	4.2
40	6.0	5.6
50	7.5	7.1

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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°	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.
0	9.75 859	18	9.84 522	27	0.15 477	9.91 336	60	
1	9.75 877	18	9.84 549	27	0.15 450	9.91 327	59	
2	9.75 895	18	9.84 576	27	0.15 423	9.91 318	58	
3	9.75 913	18	9.84 603	27	0.15 396	9.91 310	57	
4	9.75 931	18	9.84 630	26	0.15 370	9.91 301	56	
5	9.75 949	18	9.84 657	27	0.15 343	9.91 292	55	
6	9.75 967	18	9.84 684	27	0.15 316	9.91 283	54	
7	9.75 985	18	9.84 711	27	0.15 289	9.91 274	53	
8	9.76 003	18	9.84 737	26	0.15 262	9.91 265	52	
9	9.76 021	18	9.84 764	27	0.15 235	9.91 256	51	
10	9.76 039	18	9.84 791	27	0.15 208	9.91 247	50	
11	9.76 057	18	9.84 818	26	0.15 182	9.91 239	49	
12	9.76 075	18	9.84 845	27	0.15 155	9.91 230	48	
13	9.76 092	17	9.84 871	26	0.15 128	9.91 221	47	
14	9.76 110	18	9.84 898	27	0.15 101	9.91 212	46	
15	9.76 128	18	9.84 925	27	0.15 074	9.91 203	45	
16	9.76 146	18	9.84 952	26	0.15 048	9.91 194	44	
17	9.76 164	17	9.84 979	27	0.15 021	9.91 185	43	
18	9.76 182	18	9.85 005	26	0.14 994	9.91 176	42	
19	9.76 200	18	9.85 032	27	0.14 967	9.91 167	41	
20	9.76 217	17	9.85 059	27	0.14 940	9.91 158	40	
21	9.76 235	18	9.85 086	26	0.14 914	9.91 149	39	
22	9.76 253	18	9.85 113	27	0.14 887	9.91 140	38	
23	9.76 271	17	9.85 139	26	0.14 860	9.91 131	37	
24	9.76 289	18	9.85 166	27	0.14 833	9.91 122	36	
25	9.76 306	17	9.85 193	26	0.14 807	9.91 113	35	
26	9.76 324	18	9.85 220	27	0.14 780	9.91 104	34	
27	9.76 342	17	9.85 246	26	0.14 753	9.91 095	33	
28	9.76 360	18	9.85 273	27	0.14 726	9.91 086	32	
29	9.76 377	17	9.85 300	26	0.14 700	9.91 077	31	
30	9.76 395	18	9.85 327	27	0.14 673	9.91 068	30	
31	9.76 413	17	9.85 353	26	0.14 646	9.91 059	29	
32	9.76 431	18	9.85 380	27	0.14 620	9.91 050	28	
33	9.76 448	17	9.85 407	26	0.14 593	9.91 041	27	
34	9.76 466	18	9.85 433	27	0.14 566	9.91 032	26	
35	9.76 484	17	9.85 460	26	0.14 539	9.91 023	25	
36	9.76 501	18	9.85 487	27	0.14 513	9.91 014	24	
37	9.76 519	17	9.85 513	26	0.14 486	9.91 005	23	
38	9.76 536	18	9.85 540	27	0.14 459	9.90 996	22	
39	9.76 554	17	9.85 567	26	0.14 433	9.90 987	21	
40	9.76 572	18	9.85 594	27	0.14 406	9.90 978	20	
41	9.76 589	17	9.85 620	26	0.14 379	9.90 969	19	
42	9.76 607	18	9.85 647	27	0.14 353	9.90 960	18	
43	9.76 624	17	9.85 673	26	0.14 326	9.90 951	17	
44	9.76 642	18	9.85 700	27	0.14 299	9.90 942	16	
45	9.76 660	17	9.85 727	26	0.14 273	9.90 933	15	
46	9.76 677	18	9.85 753	27	0.14 246	9.90 923	14	
47	9.76 695	17	9.85 780	26	0.14 219	9.90 914	13	
48	9.76 712	18	9.85 807	27	0.14 193	9.90 905	12	
49	9.76 730	17	9.85 833	26	0.14 166	9.90 896	11	
50	9.76 747	18	9.85 860	27	0.14 140	9.90 887	10	
51	9.76 765	17	9.85 887	26	0.14 113	9.90 878	9	
52	9.76 782	18	9.85 913	27	0.14 086	9.90 869	8	
53	9.76 800	17	9.85 940	26	0.14 060	9.90 860	7	
54	9.76 817	18	9.85 966	27	0.14 033	9.90 850	6	
55	9.76 835	17	9.85 993	26	0.14 007	9.90 841	5	
56	9.76 852	18	9.86 020	27	0.13 980	9.90 832	4	
57	9.76 869	17	9.86 046	26	0.13 953	9.90 823	3	
58	9.76 887	18	9.86 073	27	0.13 927	9.90 814	2	
59	9.76 904	17	9.86 099	26	0.13 900	9.90 805	1	
60	9.76 922	17	9.86 126	26	0.13 874	9.90 796	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS, AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.76 922	17	9.86 126	26	0.13 874	9.90 796	9	60	
1	9.76 939	17	9.86 152	26	0.13 847	9.90 786	9	59	
2	9.76 956	17	9.86 179	26	0.13 821	9.90 777	9	58	
3	9.76 974	17	9.86 206	26	0.13 794	9.90 768	9	57	
4	9.76 991	17	9.86 232	26	0.13 767	9.90 759	9	56	
5	9.77 008	17	9.86 259	26	0.13 741	9.90 750	9	55	
6	9.77 026	17	9.86 285	26	0.13 714	9.90 740	9	54	
7	9.77 043	17	9.86 312	26	0.13 688	9.90 731	9	53	
8	9.77 060	17	9.86 338	26	0.13 661	9.90 722	9	52	
9	9.77 078	17	9.86 365	26	0.13 635	9.90 713	9	51	
10	9.77 095	17	9.86 391	26	0.13 608	9.90 703	9	50	
11	9.77 112	17	9.86 418	26	0.13 582	9.90 694	9	49	
12	9.77 130	17	9.86 444	26	0.13 555	9.90 685	9	48	
13	9.77 147	17	9.86 471	26	0.13 529	9.90 676	9	47	
14	9.77 164	17	9.86 497	26	0.13 502	9.90 666	9	46	
15	9.77 181	17	9.86 524	26	0.13 476	9.90 657	9	45	
16	9.77 198	17	9.86 550	26	0.13 449	9.90 648	9	44	
17	9.77 216	17	9.86 577	26	0.13 423	9.90 639	9	43	
18	9.77 233	17	9.86 603	26	0.13 396	9.90 629	9	42	
19	9.77 250	17	9.86 630	26	0.13 370	9.90 620	9	41	
20	9.77 267	17	9.86 656	26	0.13 343	9.90 611	9	40	
21	9.77 284	17	9.86 683	26	0.13 317	9.90 602	9	39	
22	9.77 302	17	9.86 709	26	0.13 290	9.90 592	9	38	
23	9.77 319	17	9.86 736	26	0.13 264	9.90 583	9	37	
24	9.77 336	17	9.86 762	26	0.13 237	9.90 574	9	36	
25	9.77 353	17	9.86 788	26	0.13 211	9.90 564	9	35	
26	9.77 370	17	9.86 815	26	0.13 185	9.90 555	9	34	
27	9.77 387	17	9.86 841	26	0.13 158	9.90 546	9	33	
28	9.77 404	17	9.86 868	26	0.13 132	9.90 536	9	32	
29	9.77 421	17	9.86 894	26	0.13 105	9.90 527	9	31	
30	9.77 439	17	9.86 921	26	0.13 079	9.90 518	9	30	
31	9.77 456	17	9.86 947	26	0.13 052	9.90 508	9	29	
32	9.77 473	17	9.86 973	26	0.13 026	9.90 499	9	28	
33	9.77 490	17	9.87 000	26	0.13 000	9.90 490	9	27	
34	9.77 507	17	9.87 026	26	0.12 973	9.90 480	9	26	
35	9.77 524	17	9.87 053	26	0.12 947	9.90 471	9	25	
36	9.77 541	17	9.87 079	26	0.12 920	9.90 461	9	24	
37	9.77 558	17	9.87 105	26	0.12 894	9.90 452	9	23	
38	9.77 575	17	9.87 132	26	0.12 868	9.90 443	9	22	
39	9.77 592	17	9.87 158	26	0.12 841	9.90 433	9	21	
40	9.77 609	17	9.87 185	26	0.12 815	9.90 424	9	20	
41	9.77 626	17	9.87 211	26	0.12 789	9.90 414	9	19	
42	9.77 643	17	9.87 237	26	0.12 762	9.90 405	9	18	
43	9.77 660	17	9.87 264	26	0.12 736	9.90 396	9	17	
44	9.77 677	17	9.87 290	26	0.12 709	9.90 386	9	16	
45	9.77 693	16	9.87 316	26	0.12 683	9.90 377	9	15	
46	9.77 710	17	9.87 343	26	0.12 657	9.90 367	9	14	
47	9.77 727	17	9.87 369	26	0.12 630	9.90 358	9	13	
48	9.77 744	17	9.87 395	26	0.12 604	9.90 348	9	12	
49	9.77 761	17	9.87 422	26	0.12 578	9.90 339	9	11	
50	9.77 778	16	9.87 448	26	0.12 551	9.90 330	9	10	
51	9.77 795	17	9.87 474	26	0.12 525	9.90 320	9	9	
52	9.77 812	17	9.87 501	26	0.12 499	9.90 311	9	8	
53	9.77 828	16	9.87 527	26	0.12 472	9.90 301	9	7	
54	9.77 845	17	9.87 553	26	0.12 446	9.90 292	9	6	
55	9.77 862	17	9.87 580	26	0.12 420	9.90 282	9	5	
56	9.77 879	16	9.87 606	26	0.12 393	9.90 273	9	4	
57	9.77 896	17	9.87 632	26	0.12 367	9.90 263	9	3	
58	9.77 913	17	9.87 659	26	0.12 341	9.90 254	9	2	
59	9.77 929	16	9.87 685	26	0.12 315	9.90 244	9	1	
60	9.77 946	17	9.87 711	26	0.12 288	9.90 235	9	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

	27	26	26
6	2.7	2.6	2.6
7	3.1	3.1	3.0
8	3.6	3.5	3.4
9	4.0	4.0	3.9
10	4.5	4.4	4.3
20	9.0	8.8	8.6
30	13.5	13.2	13.0
40	18.0	17.6	17.3
50	22.5	22.1	21.6

	17	17	16
6	1.7	1.7	1.6
7	2.0	2.0	1.9
8	2.3	2.2	2.2
9	2.6	2.5	2.5
10	2.9	2.8	2.7
20	5.8	5.6	5.5
30	8.7	8.5	8.2
40	11.6	11.3	11.0
50	14.6	14.1	13.7

	9	9
6	0.9	0.9
7	1.1	1.0
8	1.2	1.2
9	1.4	1.3
10	1.6	1.5
20	3.1	3.0
30	4.7	4.5
40	6.3	6.0
50	7.9	7.5

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**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.78 934		9.89 281		0.10 719	9.89 653		60	
1	9.78 950	16	9.89 307	26	0.10 693	9.89 643	9	59	
2	9.78 966	16	9.89 333	26	0.10 667	9.89 633	10	58	
3	9.78 982	16	9.89 359	26	0.10 641	9.89 623	10	57	
4	9.78 999	16	9.89 385	26	0.10 615	9.89 613	10	56	
5	9.79 015	16	9.89 411	26	0.10 589	9.89 604	9	55	
6	9.79 031	16	9.89 437	26	0.10 563	9.89 594	10	54	
7	9.79 047	16	9.89 463	26	0.10 537	9.89 584	10	53	
8	9.79 063	16	9.89 489	26	0.10 511	9.89 574	10	52	
9	9.79 079	16	9.89 515	26	0.10 485	9.89 564	10	51	
10	9.79 095	16	9.89 541	26	0.10 459	9.89 554	10	50	
11	9.79 111	16	9.89 567	26	0.10 433	9.89 544	9	49	
12	9.79 127	16	9.89 593	26	0.10 407	9.89 534	10	48	
13	9.79 143	16	9.89 619	26	0.10 381	9.89 524	10	47	
14	9.79 159	16	9.89 645	26	0.10 355	9.89 514	10	46	
15	9.79 175	16	9.89 671	26	0.10 329	9.89 504	10	45	
16	9.79 191	16	9.89 697	26	0.10 303	9.89 494	10	44	
17	9.79 207	16	9.89 723	26	0.10 277	9.89 484	10	43	
18	9.79 223	16	9.89 749	26	0.10 251	9.89 474	10	42	
19	9.79 239	16	9.89 775	26	0.10 225	9.89 464	10	41	
20	9.79 255	16	9.89 801	26	0.10 199	9.89 454	10	40	
21	9.79 271	16	9.89 827	26	0.10 173	9.89 444	10	39	
22	9.79 287	16	9.89 853	26	0.10 147	9.89 434	10	38	
23	9.79 303	16	9.89 879	26	0.10 121	9.89 424	10	37	
24	9.79 319	16	9.89 905	26	0.10 095	9.89 414	10	36	
25	9.79 335	16	9.89 931	26	0.10 069	9.89 404	10	35	
26	9.79 351	16	9.89 957	26	0.10 043	9.89 394	10	34	
27	9.79 367	16	9.89 982	25	0.10 017	9.89 384	10	33	
28	9.79 383	15	9.90 008	26	0.09 991	9.89 374	10	32	
29	9.79 399	16	9.90 034	26	0.09 965	9.89 364	10	31	
30	9.79 415	16	9.90 060	26	0.09 939	9.89 354	10	30	
31	9.79 431	16	9.90 086	26	0.09 913	9.89 344	10	29	
32	9.79 446	15	9.90 112	26	0.09 887	9.89 334	10	28	
33	9.79 462	16	9.90 138	26	0.09 861	9.89 324	10	27	
34	9.79 478	16	9.90 164	25	0.09 836	9.89 314	10	26	
35	9.79 494	15	9.90 190	26	0.09 810	9.89 304	10	25	
36	9.79 510	16	9.90 216	26	0.09 784	9.89 294	10	24	
37	9.79 526	16	9.90 242	26	0.09 758	9.89 284	10	23	
38	9.79 541	15	9.90 268	26	0.09 732	9.89 274	10	22	
39	9.79 557	16	9.90 294	26	0.09 706	9.89 264	10	21	
40	9.79 573	16	9.90 319	25	0.09 680	9.89 253	10	20	
41	9.79 589	15	9.90 345	26	0.09 654	9.89 243	10	19	
42	9.79 605	16	9.90 371	26	0.09 628	9.89 233	10	18	
43	9.79 620	15	9.90 397	26	0.09 602	9.89 223	10	17	
44	9.79 636	16	9.90 423	25	0.09 577	9.89 213	10	16	
45	9.79 652	15	9.90 449	26	0.09 551	9.89 203	10	15	
46	9.79 668	16	9.90 475	26	0.09 525	9.89 193	10	14	
47	9.79 683	15	9.90 501	26	0.09 499	9.89 182	10	13	
48	9.79 699	16	9.90 526	25	0.09 473	9.89 172	10	12	
49	9.79 715	15	9.90 552	26	0.09 447	9.89 162	10	11	
50	9.79 730	15	9.90 578	26	0.09 421	9.89 152	10	10	
51	9.79 746	16	9.90 604	26	0.09 395	9.89 142	10	9	
52	9.79 762	15	9.90 630	25	0.09 370	9.89 132	10	8	
53	9.79 777	15	9.90 656	26	0.09 344	9.89 121	10	7	
54	9.79 793	16	9.90 682	26	0.09 318	9.89 111	10	6	
55	9.79 809	15	9.90 707	25	0.09 292	9.89 101	10	5	
56	9.79 824	15	9.90 733	26	0.09 266	9.89 091	10	4	
57	9.79 840	16	9.90 759	26	0.09 240	9.89 081	10	3	
58	9.79 856	15	9.90 785	26	0.09 214	9.89 070	10	2	
59	9.79 871	15	9.90 811	25	0.09 189	9.89 060	10	1	
60	9.79 887	15	9.90 837	26	0.09 163	9.89 050	10	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

26 25
6 2.6 2.5
7 3.0 3.0
8 3.4 3.4
9 3.9 3.8
10 4.3 4.2
20 8.6 8.5
30 13.0 12.7
40 17.3 17.0
50 21.6 21.2

16 16 15
6 1.6 1.6 1.5
7 1.9 1.8 1.8
8 2.2 2.1 2.0
9 2.5 2.4 2.3
10 2.7 2.6 2.6
20 5.5 5.3 5.1
30 8.2 8.0 7.7
40 11.0 10.6 10.3
50 13.7 13.3 12.9

10 10 9
6 1.0 1.0 0.9
7 1.2 1.1 1.1
8 1.4 1.3 1.2
9 1.6 1.5 1.4
10 1.7 1.6 1.6
20 3.5 3.3 3.1
30 5.2 5.0 4.7
40 7.0 6.6 6.3
50 8.7 8.3 7.9

128°

643

51°

'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.79 887	16	9.90 837	26	0.09 163	9.89 050	10	60	
1	9.79 903	15	9.90 863	25	0.09 137	9.89 040	10	59	
2	9.79 918	15	9.90 888	26	0.09 111	9.89 030	10	58	
3	9.79 934	15	9.90 914	25	0.09 085	9.89 019	10	57	
4	9.79 949	15	9.90 940	25	0.09 060	9.89 009	10	56	
5	9.79 965	15	9.90 966	26	0.09 034	9.88 999	10	55	
6	9.79 980	15	9.90 992	25	0.09 008	9.88 989	10	54	
7	9.79 996	15	9.91 017	26	0.08 982	9.88 978	10	53	
8	9.80 011	15	9.91 043	26	0.08 956	9.88 968	10	52	
9	9.80 027	15	9.91 069	25	0.08 930	9.88 958	10	51	
10	9.80 042	15	9.91 095	25	0.08 905	9.88 947	10	50	26 25
11	9.80 058	15	9.91 121	26	0.08 879	9.88 937	10	49	6 2.6 2.5
12	9.80 073	15	9.91 146	25	0.08 853	9.88 927	10	48	7 3.0 3.0
13	9.80 089	15	9.91 172	26	0.08 827	9.88 917	10	47	8 3.4 3.4
14	9.80 104	15	9.91 198	25	0.08 802	9.88 906	10	46	9 3.9 3.8
15	9.80 120	15	9.91 224	26	0.08 776	9.88 896	10	45	10 4.3 4.2
16	9.80 135	15	9.91 250	25	0.08 750	9.88 886	10	44	20 8.6 8.5
17	9.80 151	15	9.91 275	26	0.08 724	9.88 875	10	43	30 13.0 12.7
18	9.80 166	15	9.91 301	25	0.08 698	9.88 865	10	42	40 17.3 17.0
19	9.80 182	15	9.91 327	25	0.08 673	9.88 855	10	41	50 21.6 21.2
20	9.80 197	15	9.91 353	26	0.08 647	9.88 844	10	40	
21	9.80 213	15	9.91 378	25	0.08 621	9.88 834	10	39	
22	9.80 228	15	9.91 404	26	0.08 595	9.88 823	10	38	
23	9.80 243	15	9.91 430	25	0.08 570	9.88 813	10	37	
24	9.80 259	15	9.91 456	26	0.08 544	9.88 803	10	36	
25	9.80 274	15	9.91 481	25	0.08 518	9.88 792	10	35	
26	9.80 289	15	9.91 507	26	0.08 492	9.88 782	10	34	
27	9.80 305	15	9.91 533	25	0.08 467	9.88 772	10	33	
28	9.80 320	15	9.91 559	26	0.08 441	9.88 761	10	32	
29	9.80 335	15	9.91 584	25	0.08 415	9.88 751	10	31	
30	9.80 351	15	9.91 610	26	0.08 389	9.88 740	10	30	
31	9.80 366	15	9.91 636	25	0.08 364	9.88 730	10	29	
32	9.80 381	15	9.91 662	26	0.08 338	9.88 720	10	28	
33	9.80 397	15	9.91 687	25	0.08 312	9.88 709	10	27	
34	9.80 412	15	9.91 713	26	0.08 286	9.88 699	10	26	
35	9.80 427	15	9.91 739	25	0.08 261	9.88 688	10	25	
36	9.80 443	15	9.91 765	26	0.08 235	9.88 678	10	24	
37	9.80 458	15	9.91 790	25	0.08 209	9.88 667	10	23	
38	9.80 473	15	9.91 816	26	0.08 183	9.88 657	10	22	
39	9.80 488	15	9.91 842	25	0.08 158	9.88 646	10	21	
40	9.80 504	15	9.91 867	26	0.08 132	9.88 636	10	20	
41	9.80 519	15	9.91 893	25	0.08 106	9.88 625	10	19	
42	9.80 534	15	9.91 919	26	0.08 081	9.88 615	10	18	
43	9.80 549	15	9.91 945	25	0.08 055	9.88 604	10	17	
44	9.80 564	15	9.91 970	26	0.08 029	9.88 594	10	16	
45	9.80 580	15	9.91 996	25	0.08 004	9.88 583	10	15	
46	9.80 595	15	9.92 022	26	0.07 978	9.88 573	10	14	
47	9.80 610	15	9.92 047	25	0.07 952	9.88 562	10	13	
48	9.80 625	15	9.92 073	26	0.07 926	9.88 552	10	12	
49	9.80 640	15	9.92 099	25	0.07 901	9.88 541	10	11	
50	9.80 655	15	9.92 124	26	0.07 875	9.88 531	10	10	
51	9.80 671	15	9.92 150	25	0.07 849	9.88 520	10	9	
52	9.80 686	15	9.92 176	26	0.07 824	9.88 510	10	8	
53	9.80 701	15	9.92 201	25	0.07 798	9.88 499	10	7	
54	9.80 716	15	9.92 227	26	0.07 772	9.88 489	10	6	
55	9.80 731	15	9.92 253	25	0.07 747	9.88 478	10	5	
56	9.80 746	15	9.92 278	26	0.07 721	9.88 467	10	4	
57	9.80 761	15	9.92 304	25	0.07 695	9.88 457	10	3	
58	9.80 776	15	9.92 330	26	0.07 670	9.88 446	10	2	
59	9.80 791	15	9.92 355	25	0.07 644	9.88 436	10	1	
60	9.80 806	15	9.92 381	26	0.07 618	9.88 425	10	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

26 25
6 2.6 2.5
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8 3.4 3.4
9 3.9 3.8
10 4.3 4.2
20 8.6 8.5
30 13.0 12.7
40 17.3 17.0
50 21.6 21.2

16 15 15
6 1.6 1.5 1.5
7 1.8 1.8 1.7
8 2.1 2.0 2.0
9 2.4 2.3 2.2
10 2.6 2.6 2.5
20 5.3 5.1 5.0
30 8.0 7.7 7.5
40 10.6 10.3 10.0
50 13.3 12.9 12.5

11 10 10
6 1.1 1.0 1.0
7 1.3 1.2 1.1
8 1.4 1.4 1.3
9 1.6 1.6 1.5
10 1.8 1.7 1.6
20 3.6 3.5 3.3
30 5.5 5.2 5.0
40 7.3 7.0 6.6
50 9.1 8.7 8.3

**TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.**

40°

139°

	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.	P. P.		
0	9.80 806		9.92 381		0.07 618	9.88 425	60			
1	9.80 822	15	9.92 407	25	0.07 593	9.88 415	59			
2	9.80 837	15	9.92 432	25	0.07 567	9.88 404	58			
3	9.80 852	15	9.92 458	25	0.07 541	9.88 393	57			
4	9.80 867	15	9.92 484	25	0.07 516	9.88 383	56			
5	9.80 882	15	9.92 509	25	0.07 490	9.88 372	55			
6	9.80 897	15	9.92 535	25	0.07 465	9.88 361	54			
7	9.80 912	15	9.92 561	25	0.07 439	9.88 351	53			
8	9.80 927	15	9.92 586	25	0.07 413	9.88 340	52			
9	9.80 942	15	9.92 612	25	0.07 388	9.88 329	51			
10	9.80 957	15	9.92 638	25	0.07 362	9.88 319	50			
11	9.80 972	15	9.92 663	25	0.07 336	9.88 308	49			
12	9.80 987	15	9.92 689	25	0.07 311	9.88 297	48			
13	9.81 001	14	9.92 714	26	0.07 285	9.88 287	47			
14	9.81 016	15	9.92 740	25	0.07 259	9.88 276	46			
15	9.81 031	15	9.92 766	25	0.07 234	9.88 265	45			
16	9.81 046	15	9.92 791	25	0.07 208	9.88 255	44			
17	9.81 061	15	9.92 817	25	0.07 183	9.88 244	43			
18	9.81 076	15	9.92 842	25	0.07 157	9.88 233	42			
19	9.81 091	14	9.92 868	26	0.07 131	9.88 223	41			
20	9.81 106	15	9.92 894	25	0.07 106	9.88 212	40			
21	9.81 121	15	9.92 919	25	0.07 080	9.88 201	39			
22	9.81 136	15	9.92 945	25	0.07 055	9.88 190	38			
23	9.81 150	14	9.92 971	26	0.07 029	9.88 180	37			
24	9.81 165	15	9.92 996	25	0.07 003	9.88 169	36			
25	9.81 180	15	9.93 022	25	0.06 978	9.88 158	35			
26	9.81 195	14	9.93 047	25	0.06 952	9.88 147	34			
27	9.81 210	15	9.93 073	25	0.06 927	9.88 137	33			
28	9.81 225	15	9.93 098	25	0.06 901	9.88 126	32			
29	9.81 239	14	9.93 124	26	0.06 875	9.88 115	31			
30	9.81 254	15	9.93 150	25	0.06 850	9.88 104	30			
31	9.81 269	14	9.93 175	25	0.06 824	9.88 094	29			
32	9.81 284	15	9.93 201	25	0.06 799	9.88 083	28			
33	9.81 299	15	9.93 226	25	0.06 773	9.88 072	27			
34	9.81 313	14	9.93 252	25	0.06 748	9.88 061	26			
35	9.81 328	15	9.93 278	26	0.06 722	9.88 050	25			
36	9.81 343	14	9.93 303	25	0.06 696	9.88 039	24			
37	9.81 358	15	9.93 329	25	0.06 671	9.88 029	23			
38	9.81 372	14	9.93 354	25	0.06 645	9.88 018	22			
39	9.81 387	14	9.93 380	25	0.06 620	9.88 007	21			
40	9.81 402	15	9.93 405	25	0.06 594	9.87 996	20			
41	9.81 416	14	9.93 431	25	0.06 569	9.87 985	19			
42	9.81 431	15	9.93 456	25	0.06 543	9.87 974	18			
43	9.81 446	14	9.93 482	25	0.06 518	9.87 963	17			
44	9.81 460	14	9.93 508	26	0.06 492	9.87 953	16			
45	9.81 475	15	9.93 533	25	0.06 466	9.87 942	15			
46	9.81 490	14	9.93 559	25	0.06 441	9.87 931	14			
47	9.81 504	14	9.93 584	25	0.06 415	9.87 920	13			
48	9.81 519	15	9.93 610	25	0.06 390	9.87 909	12			
49	9.81 534	14	9.93 635	25	0.06 364	9.87 898	11			
50	9.81 548	14	9.93 661	25	0.06 339	9.87 887	10			
51	9.81 563	14	9.93 686	25	0.06 313	9.87 876	9			
52	9.81 578	15	9.93 712	25	0.06 288	9.87 865	8			
53	9.81 592	14	9.93 737	25	0.06 262	9.87 854	7			
54	9.81 607	14	9.93 763	25	0.06 237	9.87 844	6			
55	9.81 621	14	9.93 788	25	0.06 211	9.87 833	5			
56	9.81 636	14	9.93 814	26	0.06 186	9.87 822	4			
57	9.81 650	14	9.93 840	25	0.06 160	9.87 811	3			
58	9.81 665	15	9.93 865	25	0.06 134	9.87 800	2			
59	9.81 680	15	9.93 891	25	0.06 109	9.87 789	1			
60	9.81 694	14	9.93 916	25	0.06 083	9.87 778	0			
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.	P. P.		

26 25
6 2.6 2.5
7 3.0 3.0
8 3.4 3.4
9 3.9 3.8
10 4.3 4.2
20 8.6 8.5
30 13.0 12.7
40 17.3 17.0
50 21.6 21.2

15 15 14
6 1.5 1.5 1.4
7 1.8 1.7 1.7
8 2.0 2.0 1.9
9 2.3 2.2 2.2
10 2.6 2.5 2.4
20 5.1 5.0 4.8
30 7.7 7.5 7.2
40 10.3 10.0 9.6
50 12.9 12.5 12.1

11 10
6 1.1 1.0
7 1.3 1.2
8 1.4 1.4
9 1.6 1.6
10 1.8 1.7
20 3.6 3.5
30 5.5 5.2
40 7.3 7.0
50 9.1 8.7

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

41°

138°

'	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.81 694	14	9.93 916	25	0.06 083	9.87 778	11	60	
1	9.81 709	14	9.93 942	25	0.06 058	9.87 767	11	59	
2	9.81 723	14	9.93 967	25	0.06 032	9.87 756	11	58	
3	9.81 738	14	9.93 993	25	0.06 007	9.87 745	11	57	
4	9.81 752	14	9.94 018	25	0.05 981	9.87 734	11	56	
5	9.81 767	14	9.94 044	25	0.05 956	9.87 723	11	55	
6	9.81 781	14	9.94 069	25	0.05 930	9.87 712	11	54	
7	9.81 796	14	9.94 095	25	0.05 905	9.87 701	11	53	
8	9.81 810	14	9.94 120	25	0.05 879	9.87 690	11	52	
9	9.81 824	14	9.94 146	25	0.05 854	9.87 679	11	51	
10	9.81 839	14	9.94 171	25	0.05 828	9.87 668	11	50	
11	9.81 853	14	9.94 197	25	0.05 803	9.87 657	11	49	
12	9.81 868	14	9.94 222	25	0.05 777	9.87 645	11	48	
13	9.81 882	14	9.94 248	25	0.05 752	9.87 634	11	47	
14	9.81 897	14	9.94 273	25	0.05 726	9.87 623	11	46	
15	9.81 911	14	9.94 299	25	0.05 701	9.87 612	11	45	
16	9.81 925	14	9.94 324	25	0.05 675	9.87 601	11	44	
17	9.81 940	14	9.94 350	25	0.05 650	9.87 590	11	43	
18	9.81 954	14	9.94 375	25	0.05 625	9.87 579	11	42	
19	9.81 969	14	9.94 400	25	0.05 599	9.87 568	11	41	
20	9.81 983	14	9.94 426	25	0.05 574	9.87 557	11	40	
21	9.81 997	14	9.94 451	25	0.05 548	9.87 546	11	39	
22	9.82 012	14	9.94 477	25	0.05 523	9.87 535	11	38	
23	9.82 026	14	9.94 502	25	0.05 497	9.87 523	11	37	
24	9.82 040	14	9.94 528	25	0.05 472	9.87 512	11	36	
25	9.82 055	14	9.94 553	25	0.05 446	9.87 501	11	35	
26	9.82 069	14	9.94 579	25	0.05 421	9.87 490	11	34	
27	9.82 083	14	9.94 604	25	0.05 395	9.87 479	11	33	
28	9.82 098	14	9.94 630	25	0.05 370	9.87 468	11	32	
29	9.82 112	14	9.94 655	25	0.05 344	9.87 457	11	31	
30	9.82 126	14	9.94 681	25	0.05 319	9.87 445	11	30	
31	9.82 140	14	9.94 706	25	0.05 293	9.87 434	11	29	
32	9.82 155	14	9.94 732	25	0.05 268	9.87 423	11	28	
33	9.82 169	14	9.94 757	25	0.05 243	9.87 412	11	27	
34	9.82 183	14	9.94 782	25	0.05 217	9.87 401	11	26	
35	9.82 197	14	9.94 808	25	0.05 192	9.87 389	11	25	
36	9.82 212	14	9.94 833	25	0.05 166	9.87 378	11	24	
37	9.82 226	14	9.94 859	25	0.05 141	9.87 367	11	23	
38	9.82 240	14	9.94 884	25	0.05 115	9.87 356	11	22	
39	9.82 254	14	9.94 910	25	0.05 090	9.87 345	11	21	
40	9.82 269	14	9.94 935	25	0.05 064	9.87 333	11	20	
41	9.82 283	14	9.94 961	25	0.05 039	9.87 322	11	19	
42	9.82 297	14	9.94 986	25	0.05 014	9.87 311	11	18	
43	9.82 311	14	9.95 011	25	0.04 988	9.87 300	11	17	
44	9.82 325	14	9.95 037	25	0.04 963	9.87 288	11	16	
45	9.82 339	14	9.95 062	25	0.04 937	9.87 277	11	15	
46	9.82 354	14	9.95 088	25	0.04 912	9.87 266	11	14	
47	9.82 368	14	9.95 113	25	0.04 886	9.87 254	11	13	
48	9.82 382	14	9.95 139	25	0.04 861	9.87 243	11	12	
49	9.82 396	14	9.95 164	25	0.04 836	9.87 232	11	11	
50	9.82 410	14	9.95 189	25	0.04 810	9.87 221	11	10	
51	9.82 424	14	9.95 215	25	0.04 785	9.87 209	11	9	
52	9.82 438	14	9.95 240	25	0.04 759	9.87 198	11	8	
53	9.82 452	14	9.95 266	25	0.04 734	9.87 187	11	7	
54	9.82 467	14	9.95 291	25	0.04 708	9.87 175	11	6	
55	9.82 481	14	9.95 316	25	0.04 683	9.87 164	11	5	
56	9.82 495	14	9.95 342	25	0.04 658	9.87 153	11	4	
57	9.82 509	14	9.95 367	25	0.04 632	9.87 141	11	3	
58	9.82 523	14	9.95 393	25	0.04 607	9.87 130	11	2	
59	9.82 537	14	9.95 418	25	0.04 581	9.87 118	11	1	
60	9.82 551	14	9.95 443	25	0.04 556	9.87 107	11	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

25 25
6 2.5 2.5
7 3.0 2.9
8 3.4 3.3
9 3.8 3.7
10 4.2 4.1
20 8.5 8.3
30 12.7 12.5
40 17.0 16.6
50 21.2 20.8

14 14
6 1.4 1.4
7 1.7 1.6
8 1.9 1.8
9 2.2 2.1
10 2.4 2.5
20 4.8 4.6
30 7.2 7.0
40 9.6 9.3
50 12.1 11.6

11 11
6 1.1 1.1
7 1.3 1.3
8 1.5 1.4
9 1.7 1.6
10 1.9 1.8
20 3.8 3.6
30 5.7 5.5
40 7.6 7.3
50 9.6 9.1

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

42°

137°

	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.82 551	14	9.95 443	25	0.04 556	9.87 107	11	60	
1	9.82 565	14	9.95 469	25	0.04 531	9.87 096	11	59	
2	9.82 579	14	9.95 494	25	0.04 505	9.87 084	11	58	
3	9.82 593	14	9.95 520	25	0.04 480	9.87 073	11	57	
4	9.82 607	14	9.95 545	25	0.04 454	9.87 062	11	56	
5	9.82 621	14	9.95 571	25	0.04 429	9.87 050	11	55	
6	9.82 635	14	9.95 596	25	0.04 404	9.87 039	11	54	
7	9.82 649	14	9.95 621	25	0.04 378	9.87 027	11	53	
8	9.82 663	14	9.95 647	25	0.04 353	9.87 016	11	52	
9	9.82 677	14	9.95 672	25	0.04 327	9.87 004	11	51	
10	9.82 691	14	9.95 697	25	0.04 302	9.86 993	11	50	
11	9.82 705	14	9.95 723	25	0.04 277	9.86 982	11	49	
12	9.82 719	14	9.95 748	25	0.04 251	9.86 970	11	48	
13	9.82 733	13	9.95 774	25	0.04 226	9.86 959	11	47	
14	9.82 746	13	9.95 799	25	0.04 200	9.86 947	11	46	
15	9.82 760	14	9.95 824	25	0.04 175	9.86 936	11	45	
16	9.82 774	14	9.95 850	25	0.04 150	9.86 924	11	44	
17	9.82 788	14	9.95 875	25	0.04 124	9.86 913	11	43	
18	9.82 802	13	9.95 901	25	0.04 099	9.86 901	11	42	
19	9.82 816	13	9.95 926	25	0.04 074	9.86 890	11	41	
20	9.82 830	14	9.95 951	25	0.04 048	9.86 878	11	40	
21	9.82 844	14	9.95 977	25	0.04 023	9.86 867	11	39	
22	9.82 858	13	9.96 002	25	0.03 997	9.86 855	11	38	
23	9.82 871	13	9.96 027	25	0.03 972	9.86 844	11	37	
24	9.82 885	14	9.96 053	25	0.03 947	9.86 832	11	36	
25	9.82 899	13	9.96 078	25	0.03 921	9.86 821	11	35	
26	9.82 913	13	9.96 104	25	0.03 896	9.86 809	11	34	
27	9.82 927	13	9.96 129	25	0.03 871	9.86 798	12	33	
28	9.82 940	14	9.96 154	25	0.03 845	9.86 786	11	32	
29	9.82 954	14	9.96 180	25	0.03 820	9.86 774	11	31	
30	9.82 968	13	9.96 205	25	0.03 795	9.86 763	11	30	
31	9.82 982	13	9.96 230	25	0.03 769	9.86 751	11	29	
32	9.82 996	13	9.96 256	25	0.03 744	9.86 740	11	28	
33	9.83 009	14	9.96 281	25	0.03 718	9.86 728	11	27	
34	9.83 023	13	9.96 306	25	0.03 693	9.86 716	11	26	
35	9.83 037	13	9.96 332	25	0.03 668	9.86 705	11	25	
36	9.83 051	13	9.96 357	25	0.03 642	9.86 693	11	24	
37	9.83 064	13	9.96 383	25	0.03 617	9.86 682	11	23	
38	9.83 078	14	9.96 408	25	0.03 592	9.86 670	12	22	
39	9.83 092	13	9.96 433	25	0.03 566	9.86 658	11	21	
40	9.83 106	14	9.96 459	25	0.03 541	9.86 647	11	20	
41	9.83 119	13	9.96 484	25	0.03 516	9.86 635	12	19	
42	9.83 133	13	9.96 509	25	0.03 490	9.86 623	11	18	
43	9.83 147	13	9.96 535	25	0.03 465	9.86 612	11	17	
44	9.83 160	13	9.96 560	25	0.03 440	9.86 600	11	16	
45	9.83 174	13	9.96 585	25	0.03 414	9.86 588	12	15	
46	9.83 188	14	9.96 611	25	0.03 389	9.86 577	11	14	
47	9.83 201	13	9.96 636	25	0.03 364	9.86 565	12	13	
48	9.83 215	13	9.96 661	25	0.03 338	9.86 553	11	12	
49	9.83 229	14	9.96 687	25	0.03 313	9.86 542	11	11	
50	9.83 242	13	9.96 712	25	0.03 287	9.86 530	12	10	
51	9.83 256	13	9.96 737	25	0.03 262	9.86 518	11	9	
52	9.83 269	13	9.96 763	25	0.03 237	9.86 507	12	8	
53	9.83 283	14	9.96 788	25	0.03 211	9.86 495	11	7	
54	9.83 297	13	9.96 813	25	0.03 186	9.86 483	11	6	
55	9.83 310	13	9.96 839	25	0.03 161	9.86 471	12	5	
56	9.83 324	13	9.96 864	25	0.03 135	9.86 460	11	4	
57	9.83 337	13	9.96 889	25	0.03 110	9.86 448	12	3	
58	9.83 351	14	9.96 915	25	0.03 085	9.86 436	11	2	
59	9.83 365	13	9.96 940	25	0.03 059	9.86 424	12	1	
60	9.83 378	13	9.96 965	25	0.03 034	9.86 412	12	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

6	25	25
7	2.5	2.5
8	3.0	2.9
9	3.4	3.3
10	3.8	3.7
10	4.2	4.1
20	8.5	8.3
30	12.7	12.5
40	17.0	16.6
50	21.2	20.8

6	14	13
7	1.4	1.3
8	1.6	1.6
9	1.8	1.8
10	2.1	2.0
10	2.3	2.2
20	4.6	4.5
30	7.0	6.7
40	9.3	9.0
50	11.6	11.2

6	12	11	11
7	1.2	1.1	1.1
8	1.4	1.3	1.3
9	1.6	1.5	1.4
10	1.8	1.7	1.6
10	2.0	1.9	1.8
20	4.0	3.8	3.6
30	6.0	5.7	5.5
40	8.0	7.6	7.3
50	10.0	9.6	9.1

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.83 378	13	9.96 965	25	0.03 034	9.86 412	11	60	
1	9.83 392	13	9.96 991	25	0.03 009	9.86 401	12	59	
2	9.83 405	13	9.97 016	25	0.02 984	9.86 389	11	58	
3	9.83 419	13	9.97 041	25	0.02 958	9.86 377	12	57	
4	9.83 432	13	9.97 067	25	0.02 933	9.86 365	11	56	
5	9.83 446	13	9.97 092	25	0.02 908	9.86 354	12	55	
6	9.83 459	13	9.97 117	25	0.02 882	9.86 342	12	54	
7	9.83 473	13	9.97 143	25	0.02 857	9.86 330	11	53	
8	9.83 486	13	9.97 168	25	0.02 832	9.86 318	12	52	
9	9.83 500	13	9.97 193	25	0.02 806	9.86 306	12	51	25 25
10	9.83 513	13	9.97 219	25	0.02 781	9.86 294	12	50	6 2.5 2.5
11	9.83 527	13	9.97 244	25	0.02 756	9.86 282	11	49	7 3.0 2.9
12	9.83 540	13	9.97 269	25	0.02 730	9.86 271	12	48	8 3.4 3.3
13	9.83 554	13	9.97 295	25	0.02 705	9.86 259	12	47	9 3.8 3.7
14	9.83 567	13	9.97 320	25	0.02 680	9.86 247	12	46	10 4.2 4.1
15	9.83 580	13	9.97 345	25	0.02 654	9.86 235	11	45	20 8.5 8.3
16	9.83 594	13	9.97 370	25	0.02 629	9.86 223	12	44	30 12.7 12.5
17	9.83 607	13	9.97 396	25	0.02 604	9.86 211	12	43	40 17.0 16.6
18	9.83 621	13	9.97 421	25	0.02 578	9.86 199	12	42	50 21.2 20.8
19	9.83 634	13	9.97 446	25	0.02 553	9.86 187	11	41	
20	9.83 647	13	9.97 472	25	0.02 528	9.86 176	12	40	
21	9.83 661	13	9.97 497	25	0.02 502	9.86 164	12	39	
22	9.83 674	13	9.97 522	25	0.02 477	9.86 152	12	38	
23	9.83 688	13	9.97 548	25	0.02 452	9.86 140	12	37	
24	9.83 701	13	9.97 573	25	0.02 427	9.86 128	12	36	
25	9.83 714	13	9.97 598	25	0.02 401	9.86 116	12	35	
26	9.83 728	13	9.97 624	25	0.02 376	9.86 104	12	34	
27	9.83 741	13	9.97 649	25	0.02 351	9.86 092	12	33	13 13
28	9.83 754	13	9.97 674	25	0.02 325	9.86 080	12	32	6 1.3 1.3
29	9.83 768	13	9.97 699	25	0.02 300	9.86 068	12	31	7 1.6 1.5
30	9.83 781	13	9.97 725	25	0.02 275	9.86 056	12	30	8 1.8 1.7
31	9.83 794	13	9.97 750	25	0.02 249	9.86 044	12	29	9 2.0 1.9
32	9.83 808	13	9.97 775	25	0.02 224	9.86 032	12	28	10 2.2 2.1
33	9.83 821	13	9.97 801	25	0.02 199	9.86 020	12	27	20 4.5 4.3
34	9.83 834	13	9.97 826	25	0.02 174	9.86 008	12	26	30 6.7 6.5
35	9.83 847	13	9.97 851	25	0.02 148	9.85 996	12	25	40 9.0 8.6
36	9.83 861	13	9.97 877	25	0.02 123	9.85 984	12	24	50 11.2 10.8
37	9.83 874	13	9.97 902	25	0.02 098	9.85 972	12	23	
38	9.83 887	13	9.97 927	25	0.02 072	9.85 960	12	22	
39	9.83 900	13	9.97 952	25	0.02 047	9.85 948	12	21	
40	9.83 914	13	9.97 978	25	0.02 022	9.85 936	12	20	
41	9.83 927	13	9.98 003	25	0.01 996	9.85 924	12	19	
42	9.83 940	13	9.98 028	25	0.01 971	9.85 912	12	18	
43	9.83 953	13	9.98 054	25	0.01 946	9.85 900	12	17	
44	9.83 967	13	9.98 079	25	0.01 921	9.85 887	12	16	
45	9.83 980	13	9.98 104	25	0.01 895	9.85 875	12	15	12 12 11
46	9.83 993	13	9.98 129	25	0.01 870	9.85 863	12	14	6 1.2 1.1
47	9.84 006	13	9.98 155	25	0.01 845	9.85 851	12	13	7 1.4 1.3
48	9.84 019	13	9.98 180	25	0.01 819	9.85 839	12	12	8 1.6 1.5
49	9.84 033	13	9.98 205	25	0.01 794	9.85 827	12	11	9 1.9 1.7
50	9.84 046	13	9.98 231	25	0.01 769	9.85 815	12	10	10 2.1 2.0
51	9.84 059	13	9.98 256	25	0.01 744	9.85 803	12	9	20 4.1 4.0
52	9.84 072	13	9.98 281	25	0.01 718	9.85 791	12	8	30 6.2 6.0
53	9.84 085	13	9.98 306	25	0.01 693	9.85 778	12	7	40 8.3 8.0
54	9.84 098	13	9.98 332	25	0.01 668	9.85 766	12	6	50 10.4 10.0
55	9.84 111	13	9.98 357	25	0.01 642	9.85 754	12	5	
56	9.84 124	13	9.98 382	25	0.01 617	9.85 742	12	4	
57	9.84 138	13	9.98 408	25	0.01 592	9.85 730	12	3	
58	9.84 151	13	9.98 433	25	0.01 567	9.85 718	12	2	
59	9.84 164	13	9.98 458	25	0.01 541	9.85 705	12	1	
60	9.84 177	13	9.98 483	25	0.01 516	9.85 693	12	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

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TABLE VII.—LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS.

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	Log. Sin.	d.	Log. Tan.	c. d.	Log. Cot.	Log. Cos.	d.		P. P.
0	9.84 177	13	9.98 483	25	0.01 516	9.85 693	12	60	
1	9.84 190	13	9.98 509	25	0.01 491	9.85 681	12	59	
2	9.84 203	13	9.98 534	25	0.01 465	9.85 669	12	58	
3	9.84 216	13	9.98 559	25	0.01 440	9.85 657	12	57	
4	9.84 229	13	9.98 585	25	0.01 415	9.85 644	12	56	
5	9.84 242	13	9.98 610	25	0.01 390	9.85 632	12	55	
6	9.84 255	13	9.98 635	25	0.01 364	9.85 620	12	54	
7	9.84 268	13	9.98 660	25	0.01 339	9.85 608	12	53	
8	9.84 281	13	9.98 686	25	0.01 314	9.85 595	12	52	
9	9.84 294	13	9.98 711	25	0.01 289	9.85 583	12	51	
10	9.84 307	13	9.98 736	25	0.01 263	9.85 571	12	50	25 25
11	9.84 320	13	9.98 762	25	0.01 238	9.85 559	12	49	6 2.5 2.5
12	9.84 333	13	9.98 787	25	0.01 213	9.85 546	12	48	7 3.0 2.9
13	9.84 346	13	9.98 812	25	0.01 187	9.85 534	12	47	8 3.4 3.3
14	9.84 359	13	9.98 837	25	0.01 162	9.85 522	12	46	9 3.8 3.7
15	9.84 372	13	9.98 863	25	0.01 137	9.85 509	12	45	10 4.2 4.1
16	9.84 385	13	9.98 888	25	0.01 112	9.85 497	12	44	20 8.5 8.3
17	9.84 398	13	9.98 913	25	0.01 086	9.85 485	12	43	30 12.7 12.5
18	9.84 411	13	9.98 938	25	0.01 061	9.85 472	12	42	40 17.0 16.6
19	9.84 424	13	9.98 964	25	0.01 036	9.85 460	12	41	50 21.2 20.8
20	9.84 437	12	9.98 989	25	0.01 010	9.85 448	12	40	
21	9.84 450	13	9.99 014	25	0.00 985	9.85 435	12	39	
22	9.84 463	13	9.99 040	25	0.00 960	9.85 423	12	38	
23	9.84 476	13	9.99 065	25	0.00 935	9.85 411	12	37	
24	9.84 489	13	9.99 090	25	0.00 909	9.85 398	12	36	
25	9.84 502	13	9.99 115	25	0.00 884	9.85 386	12	35	
26	9.84 514	12	9.99 141	25	0.00 859	9.85 374	12	34	
27	9.84 527	13	9.99 166	25	0.00 834	9.85 361	12	33	13 13
28	9.84 540	13	9.99 191	25	0.00 808	9.85 349	12	32	6 1.3 1.3
29	9.84 553	13	9.99 216	25	0.00 783	9.85 336	12	31	7 1.6 1.5
30	9.84 566	12	9.99 242	25	0.00 758	9.85 324	12	30	8 1.8 1.7
31	9.84 579	13	9.99 267	25	0.00 733	9.85 312	12	29	9 2.0 1.6
32	9.84 592	13	9.99 292	25	0.00 707	9.85 299	12	28	10 2.2 2.1
33	9.84 604	12	9.99 318	25	0.00 682	9.85 287	12	27	20 4.5 4.3
34	9.84 617	13	9.99 343	25	0.00 657	9.85 274	12	26	30 6.7 6.5
35	9.84 630	13	9.99 368	25	0.00 631	9.85 262	12	25	40 9.0 8.6
36	9.84 643	12	9.99 393	25	0.00 606	9.85 249	12	24	50 11.2 10.8
37	9.84 656	13	9.99 419	25	0.00 581	9.85 237	12	23	
38	9.84 669	13	9.99 444	25	0.00 556	9.85 224	12	22	
39	9.84 681	12	9.99 469	25	0.00 530	9.85 212	12	21	
40	9.84 694	13	9.99 494	25	0.00 505	9.85 199	12	20	
41	9.84 707	12	9.99 520	25	0.00 480	9.85 187	12	19	
42	9.84 720	13	9.99 545	25	0.00 455	9.85 174	12	18	
43	9.84 732	12	9.99 570	25	0.00 429	9.85 162	12	17	
44	9.84 745	13	9.99 595	25	0.00 404	9.85 149	12	16	
45	9.84 758	12	9.99 621	25	0.00 379	9.85 137	12	15	12 12
46	9.84 771	13	9.99 646	25	0.00 353	9.85 124	12	14	6 1.2 1.2
47	9.84 783	12	9.99 671	25	0.00 328	9.85 112	12	13	7 1.4 1.4
48	9.84 796	13	9.99 697	25	0.00 303	9.85 099	12	12	8 1.6 1.6
49	9.84 809	12	9.99 722	25	0.00 278	9.85 087	12	11	9 1.9 1.8
50	9.84 822	13	9.99 747	25	0.00 252	9.85 074	12	10	10 2.1 2.0
51	9.84 834	12	9.99 772	25	0.00 227	9.85 062	12	9	20 4.1 4.0
52	9.84 847	13	9.99 798	25	0.00 202	9.85 049	12	8	30 6.2 6.0
53	9.84 860	12	9.99 823	25	0.00 177	9.85 037	12	7	40 8.3 8.0
54	9.84 872	13	9.99 848	25	0.00 151	9.85 024	13	6	50 10.4 10.0
55	9.84 885	12	9.99 873	25	0.00 126	9.85 011	12	5	
56	9.84 898	13	9.99 899	25	0.00 101	9.84 999	12	4	
57	9.84 910	12	9.99 924	25	0.00 076	9.84 986	12	3	
58	9.84 923	13	9.99 949	25	0.00 051	9.84 974	12	2	
59	9.84 936	12	9.99 974	25	0.00 025	9.84 961	13	1	
60	9.84 948	12	0.00 000	25	0.00 000	9.84 948	12	0	
	Log. Cos.	d.	Log. Cot.	c. d.	Log. Tan.	Log. Sin.	d.		P. P.

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TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS. 1°

0°

'	Log. Vers.	D	Log. Exsec.	D	Log. Vers.	D	Log. Exsec.	D	'
0	— ∞		— ∞		6.18271	1435	6.18278	1436	0
1	2.62642	60206	2.62642	60206	.19707	1412	.19714	1412	1
2	3.22848	35218	3.22848	35218	.21119	1389	.21126	1390	2
3	3.58066	24987	3.58066	24987	.22509	1368	.22516	1368	3
4	3.83054		3.83054		.23877		.23884		4
5	4.02436	19382	4.02436	19382	6.25223	1346	6.25231	1347	5
6	.18272	15836	.18272	15836	.26549	1326	.26557	1326	6
7	.31662	13389	.31662	13389	.27856	1306	.27864	1306	7
8	.43260	11598	.43260	11598	.29142	1286	.29151	1287	8
9	.53490	10230	.53491	10230	.30410	1268	.30419	1268	9
10	4.62642	9151	4.62642	9151	6.31660	1250	6.31669	1250	10
11	.70920	8278	.70921	8279	.32892	1232	.32901	1232	11
12	.78478	7558	.78478	7557	.34107	1214	.34116	1215	12
13	.85431	6953	.85431	6952	.35305	1198	.35315	1198	13
14	.91868	6437	.91868	6437	.36487	1182	.36497	1182	14
15	4.97860	5992	4.97861	5993	6.37653	1166	6.37663	1166	15
16	5.03466	5605	5.03466	5605	.38803	1150	.38814	1151	16
17	.08732	5266	.08732	5266	.39938	1135	.39949	1135	17
18	.13696	4964	.13697	4964	.41059	1121	.41070	1121	18
19	.18393	4696	.18393	4696	.42165	1106	.42177	1106	19
20	5.22848	4455	5.22849	4456	6.43258	1093	6.43270	1093	20
21	.27086	4238	.27087	4238	.44337	1078	.44349	1078	21
22	.31126	4040	.31127	4040	.45403	1066	.45415	1066	22
23	3.4987	3861	3.4988	3861	.46455	1052	.46468	1053	23
24	.38684	3697	.38685	3697	.47496	1040	.47509	1040	24
25	5.42230	3545	5.42231	3545	6.48524	1028	6.48537	1028	25
26	.45636	3406	.45638	3407	.49539	1016	.49553	1015	26
27	.48915	3278	.48916	3278	.50544	1004	.50557	1004	27
28	.52073	3158	.52075	3159	.51536	992	.51550	993	28
29	.55121	3048	.55123	3048	.52518	981	.52532	982	29
30	5.58066	2944	5.58068	2945	6.53488	970	6.53503	970	30
31	.60914	2848	.60916	2848	.54448	960	.54463	960	31
32	.63672	2757	.63674	2758	.55397	949	.55413	950	32
33	.66344	2672	.66346	2672	.56336	939	.56352	939	33
34	.68937	2593	.68940	2593	.57265	929	.57281	929	34
35	5.71455	2518	5.71457	2517	6.58184	919	6.58201	919	35
36	.73902	2447	.73904	2447	.59093	909	.59110	909	36
37	.76282	2379	.76284	2380	.59993	900	.60011	900	37
38	.78598	2316	.78601	2316	.60884	891	.60902	891	38
39	.80854	2256	.80857	2256	.61766	882	.61784	882	39
40	5.83053	2199	5.83056	2199	6.62639	872	6.62657	873	40
41	.85198	2145	.85201	2145	.63503	864	.63522	864	41
42	.87291	2093	.87295	2093	.64359	855	.64378	856	42
43	.89335	2044	.89338	2043	.65206	847	.65226	848	43
44	.91332	1996	.91335	1997	.66045	839	.66065	839	44
45	5.93284	1952	5.93288	1952	6.66876	831	6.66897	831	45
46	.95193	1909	.95197	1909	.67700	823	.67720	823	46
47	.97061	1868	.97065	1868	.68515	815	.68536	816	47
48	5.98890	1829	5.98894	1829	.69323	808	.69345	808	48
49	6.00680	1790	6.00685	1791	.70124	800	.70145	800	49
50	6.02435	1755	6.02440	1755	6.70917	793	6.70939	794	50
51	.04155	1720	.04160	1720	.71703	786	.71725	786	51
52	.05842	1686	.05847	1687	.72482	779	.72505	779	52
53	.07496	1654	.07501	1654	.73254	772	.73277	772	53
54	.09120	1623	.09125	1623	.74019	765	.74043	765	54
55	6.10714	1594	6.10719	1594	6.74777	758	6.74802	759	55
56	.12279	1565	.12284	1565	.75529	752	.75554	752	56
57	.13816	1537	.13822	1537	.76275	745	.76300	746	57
58	.15327	1511	.15333	1511	.77014	739	.77040	739	58
59	.16811	1484	.16818	1485	.77747	733	.77773	733	59
60	6.18271	1460	6.18278	1460	6.78474	726	6.78500	727	60
'	Log. Vers.	D	Log. Exsec.	D	Log. Vers.	D	Log. Exsec.	D	'

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL

2°

SECANTS.

3°

	Log. Vers.	D	Log. Exsec.	D	Log. Vers.	D	Log. Exsec.	D	
0	6.78474	721	6.78500	721	7.13687	481	7.13746	481	0
1	.79195	714	.79221	715	.14168	478	.14228	479	1
2	.79909	709	.79937	709	.14646	475	.14707	476	2
3	.80618	703	.80646	703	.15122	473	.15183	474	3
4	.81322	697	.81350	698	.15595	470	.15657	471	4
5	6.82019	692	6.82048	692	7.16066	468	7.16129	469	5
6	.82711	686	.82740	687	.16534	466	.16598	466	6
7	.83398	681	.83427	682	.17000	463	.17064	464	7
8	.84079	676	.84109	676	.17463	460	.17528	461	8
9	.84755	670	.84785	671	.17923	458	.17989	459	9
10	6.85425	665	6.85457	666	7.18382	455	7.18448	456	10
11	.86091	660	.86123	660	.18837	453	.18905	454	11
12	.86751	655	.86783	656	.19291	451	.19359	452	12
13	.87407	650	.87439	651	.19742	448	.19811	449	13
14	.88057	646	.88090	646	.20191	446	.20260	447	14
15	6.88703	641	6.88737	641	7.20637	444	7.20707	445	15
16	.89344	636	.89378	636	.21081	442	.21152	442	16
17	.89980	631	.90015	632	.21523	440	.21595	440	17
18	.90612	627	.90647	628	.21963	437	.22035	438	18
19	.91239	622	.91275	623	.22400	435	.22473	436	19
20	6.91862	618	6.91898	618	7.22836	433	7.22909	434	20
21	.92480	613	.92516	614	.23269	431	.23343	431	21
22	.93093	609	.93131	610	.23700	429	.23775	429	22
23	.93703	605	.93741	605	.24129	426	.24204	427	23
24	.94308	601	.94346	601	.24555	424	.24632	425	24
25	6.94909	597	6.94948	597	7.24980	422	7.25057	423	25
26	.95506	592	.95545	593	.25402	420	.25480	421	26
27	.96099	589	.96139	589	.25823	418	.25902	419	27
28	.96688	584	.96728	585	.26241	416	.26321	417	28
29	.97272	581	.97313	581	.26658	414	.26738	415	29
30	6.97853	577	6.97895	577	7.27072	412	7.27153	413	30
31	.98430	573	.98472	574	.27485	410	.27567	411	31
32	.99004	569	.99046	570	.27895	409	.27978	409	32
33	6.99573	535	6.99616	566	.28304	406	.28387	407	33
34	7.00139	562	7.00182	563	.28711	405	.28795	405	34
35	7.00701	558	7.00745	559	7.29116	402	7.29200	404	35
36	.01259	555	.01304	555	.29518	401	.29604	402	36
37	.01814	551	.01860	552	.29919	399	.30006	400	37
38	.02366	548	.02412	548	.30319	397	.30406	398	38
39	.02914	544	.02960	545	.30716	395	.30804	396	39
40	7.03458	541	7.03505	541	7.31112	393	7.31201	394	40
41	.03999	537	.04047	538	.31505	392	.31595	393	41
42	.04537	534	.04585	535	.31897	390	.31988	391	42
43	.05071	531	.05120	531	.32288	388	.32379	389	43
44	.05603	527	.05652	528	.32676	386	.32768	388	44
45	7.06130	525	7.06180	525	7.33063	385	7.33156	385	45
46	.06655	521	.06706	522	.33448	383	.33542	384	46
47	.07177	518	.07228	519	.33831	382	.33926	382	47
48	.07695	515	.07747	516	.34213	380	.34309	380	48
49	.08211	512	.08263	513	.34593	378	.34689	379	49
50	7.08723	509	7.08776	509	7.34971	377	7.35069	377	50
51	.09232	506	.09286	507	.35348	375	.35446	376	51
52	.09739	503	.09793	503	.35723	373	.35822	374	52
53	.10242	500	.10297	501	.36097	371	.36196	373	53
54	.10743	497	.10798	498	.36468	370	.36569	371	54
55	7.11240	495	7.11297	495	7.36839	368	7.36940	369	55
56	.11735	492	.11792	493	.37207	367	.37310	368	56
57	.12227	489	.12285	490	.37574	366	.37678	366	57
58	.12716	486	.12775	487	.37940	364	.38044	365	58
59	.13203	484	.13262	484	.38304	362	.38409	363	59
60	7.13687		7.13748		7.38667		7.38773		60
	Log. Vers.	D	Log. Exsec.	D	Log. Vers.	D	Log. Exsec.	D	

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

4°

5°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.		
0	7.38667	361	7.38773	361	7.58039	289	7.58204	290	0	360	350	340
1	.39028	359	.39134	360	.58328	287	.58494	289	1	6	36.0	35.0
2	.39387	358	.39495	359	.58615	287	.58783	288	2	7	42.0	40.8
3	.39745	356	.39854	357	.58902	286	.59071	287	3	8	48.0	46.6
4	.40102	355	.40211	356	.59188	285	.59358	286	4	9	54.0	51.5
5	7.40457	353	7.40567	354	7.59473	284	7.59645	285	5	10	60.0	58.3
6	.40810	352	.40922	353	.59758	283	.59930	284	6	20	120.0	116.6
7	.41163	350	.41275	352	.60041	282	.60214	283	7	30	180.0	175.0
8	.41513	349	.41627	350	.60323	281	.60498	282	8	40	240.0	233.3
9	.41863	348	.41977	349	.60604	280	.60780	281	9	50	300.0	291.6
10	7.42211	346	7.42326	347	7.60885	279	7.61062	280	10	330	320	310
11	.42557	345	.42673	346	.61164	279	.61342	280	11	6	33.0	32.0
12	.42903	343	.43019	345	.61443	277	.61622	279	12	7	38.5	37.3
13	.43246	342	.43364	343	.61721	277	.61901	278	13	8	44.0	42.6
14	.43589	341	.43708	342	.61998	276	.62179	277	14	9	49.5	48.0
15	7.43930	339	7.44050	340	7.62274	275	7.62456	276	15	10	55.0	53.3
16	.44270	338	.44390	339	.62549	274	.62733	275	16	20	110.0	106.6
17	.44608	337	.44730	338	.62823	273	.63008	274	17	30	165.0	160.0
18	.44946	335	.45068	337	.63096	272	.63282	273	18	40	220.0	213.3
19	.45281	334	.45405	335	.63369	272	.63556	273	19	50	275.0	266.6
20	7.45616	333	7.45740	334	7.63641	270	7.63829	271	20	300	290	280
21	.45949	332	.46075	333	.63911	270	.64101	271	21	6	30.0	29.0
22	.46281	330	.46407	332	.64181	269	.64372	270	22	7	35.0	33.8
23	.46612	329	.46739	330	.64451	268	.64643	269	23	8	40.0	38.6
24	.46941	328	.47070	329	.64719	267	.64912	268	24	9	45.0	43.5
25	7.47270	327	7.47399	328	7.64986	266	7.65181	267	25	10	50.0	48.3
26	.47597	325	.47727	327	.65253	266	.65449	267	26	20	100.0	96.6
27	.47922	324	.48054	325	.65519	265	.65716	266	27	30	150.0	145.0
28	.48247	323	.48379	324	.65784	264	.65982	265	28	40	200.0	193.3
29	.48570	322	.48703	323	.66048	263	.66247	264	29	50	250.0	241.6
30	7.48892	321	7.49026	322	7.66311	263	7.66512	264	30	270	260	250
31	.49213	320	.49348	321	.66574	261	.66776	263	31	6	27.0	26.0
32	.49533	318	.49669	319	.66836	261	.67039	262	32	7	31.5	30.3
33	.49852	317	.49989	318	.67097	260	.67301	261	33	8	36.0	34.6
34	.50169	316	.50307	317	.67357	259	.67562	260	34	9	40.5	39.0
35	7.50485	315	7.50624	316	7.67617	258	7.67823	259	35	10	45.0	43.3
36	.50800	314	.50941	315	.67875	258	.68083	259	36	20	90.0	86.6
37	.51114	313	.51256	313	.68133	257	.68342	258	37	30	135.0	130.0
38	.51427	311	.51569	313	.68390	256	.68601	257	38	40	180.0	173.3
39	.51739	311	.51882	311	.68647	255	.68858	256	39	50	225.0	216.6
40	7.52050	309	7.52194	310	7.68902	255	7.69115	256	40	240	230	220
41	.52359	308	.52504	309	.69157	254	.69371	255	41	6	24.0	23.0
42	.52667	307	.52814	308	.69411	253	.69627	254	42	7	28.0	26.8
43	.52975	306	.53122	307	.69665	252	.69881	253	43	8	32.0	30.6
44	.53281	305	.53429	306	.69917	252	.70135	253	44	9	36.0	34.5
45	7.53586	304	7.53735	305	7.70169	251	7.70388	252	45	10	40.0	38.3
46	.53890	303	.54041	304	.70421	250	.70641	251	46	20	80.0	76.6
47	.54193	302	.54345	303	.70671	249	.70893	250	47	30	120.0	115.0
48	.54495	300	.54648	302	.70921	248	.71144	249	48	40	160.0	153.3
49	.54796	300	.54950	301	.71170	247	.71394	248	49	50	200.0	191.6
50	7.55096	299	7.55251	299	7.71418	247	7.71644	248	50	210	200	190
51	.55395	297	.55550	298	.71666	247	.71892	248	51	6	21.0	20.0
52	.55692	297	.55849	298	.71913	246	.72141	247	52	7	24.5	23.3
53	.55989	295	.56147	296	.72159	245	.72388	246	53	8	28.0	26.6
54	.56285	295	.56444	296	.72404	245	.72635	246	54	9	31.5	30.0
55	7.56580	293	7.56740	295	7.72649	244	7.72881	245	55	10	35.0	33.3
56	.56873	293	.57035	294	.72893	243	.73126	244	56	20	70.0	66.6
57	.57166	292	.57329	292	.73137	242	.73371	243	57	30	105.0	100.0
58	.57458	290	.57621	292	.73379	242	.73615	243	58	40	140.0	133.3
59	.57749	290	.57913	291	.73621	241	.73859	242	59	50	175.0	166.6
60	7.58039	290	7.58204	291	7.73863	241	7.74101	242	60	360	350	340
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.		

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

6°										7°									
Lg. Vers.					Log. Exs.					Lg. Vers.					Log. Exs.				
D					D					D					D				
0	7.73863	241	7.74101	242	7.87238	206	7.87563	208	0	180	9	9	6	18.0	0.9	0.9			
1	.74104	240	.74343	241	.87444	205	.87771	207	1	18.0	0.9	0.9	7	21.0	1.1	1.0			
2	.74344	239	.74585	241	.87650	205	.87978	207	2	21.0	1.1	1.0	8	24.0	1.2	1.2			
3	.74583	239	.74826	240	.87855	204	.88185	206	3	24.0	1.2	1.2	9	27.0	1.4	1.3			
4	.74822	238	.75066	239	.88060	204	.88391	206	4	27.0	1.4	1.3	10	30.0	1.6	1.5			
5	7.75060	237	7.75305	239	7.88264	204	7.88597	205	5	30.0	1.6	1.5	20	60.0	3.1	3.0			
6	.75297	236	.75544	238	.88468	203	.88803	205	6	60.0	3.1	3.0	30	90.0	4.7	4.5			
7	.75534	236	.75782	237	.88672	203	.89008	204	7	90.0	4.7	4.5	40	120.0	6.3	6.0			
8	.75770	235	.76019	237	.88875	202	.89212	204	8	120.0	6.3	6.0	50	150.0	7.9	7.5			
9	.76006	235	.76256	236	.89077	202	.89416	204	9	150.0	7.9	7.5							
10	7.76240	234	7.76492	236	7.89279	202	7.89620	203	10	8	8	7							
11	.76475	233	.76728	235	.89481	201	.89823	203	11	6	0.8	0.8	6	0.8	0.8	0.7			
12	.76703	233	.76963	234	.89682	200	.90025	202	12	7	1.0	0.9	7	1.0	0.9	0.9			
13	.76941	232	.77197	233	.89882	200	.90228	201	13	8	1.1	1.0	8	1.1	1.0	1.0			
14	.77173	232	.77431	233	.90082	199	.90429	201	14	9	1.3	1.2	9	1.3	1.2	1.1			
15	7.77405	231	7.77664	232	7.90282	199	7.90630	201	15	10	1.4	1.3	10	1.4	1.3	1.2			
16	.77636	230	.77897	231	.90481	198	.90831	200	16	20	2.8	2.6	20	2.8	2.6	2.5			
17	.77867	230	.78128	231	.90680	198	.91032	199	17	30	4.2	4.0	30	4.2	4.0	3.7			
18	.78097	229	.78360	230	.90878	197	.91231	199	18	40	5.6	5.3	40	5.6	5.3	5.0			
19	.78326	228	.78590	230	.91076	197	.91431	199	19	50	7.1	6.6	50	7.1	6.6	6.2			
20	7.78554	228	7.78820	229	7.91273	197	7.91630	198	20	7	6	6							
21	.78783	227	.79050	229	.91470	196	.91828	198	21	6	0.7	0.6	6	0.7	0.6	0.6			
22	.79010	227	.79279	228	.91667	196	.92027	198	22	7	0.8	0.7	7	0.8	0.7	0.7			
23	.79237	226	.79507	228	.91863	195	.92224	197	23	8	0.9	0.8	8	0.9	0.8	0.8			
24	.79463	226	.79735	227	.92058	195	.92421	197	24	9	1.0	1.0	9	1.0	1.0	0.9			
25	7.79689	225	7.79962	226	7.92253	195	7.92618	196	25	10	1.1	1.1	10	1.1	1.1	1.0			
26	.79914	224	.80188	226	.92448	194	.92815	196	26	20	2.3	2.1	20	2.3	2.1	2.0			
27	.80138	224	.80414	225	.92642	194	.93010	195	27	30	3.5	3.2	30	3.5	3.2	3.0			
28	.80362	223	.80639	225	.92836	193	.93206	195	28	40	4.6	4.3	40	4.6	4.3	4.0			
29	.80586	223	.80864	224	.93029	193	.93401	195	29	50	5.8	5.4	50	5.8	5.4	5.0			
30	7.80808	222	7.81088	224	7.93222	192	7.93596	194	30	6	0.5	0.5	6	0.5	0.5	0.4			
31	.81031	222	.81312	223	.93415	192	.93790	194	31	7	0.6	0.6	7	0.6	0.6	0.5			
32	.81252	221	.81535	222	.93607	191	.93984	193	32	8	0.7	0.6	8	0.7	0.6	0.5			
33	.81473	220	.81758	222	.93799	191	.94177	193	33	9	0.8	0.7	9	0.8	0.7	0.6			
34	.81694	220	.81980	221	.93990	190	.94370	192	34	10	0.9	0.8	10	0.9	0.8	0.7			
35	7.81914	219	7.82201	221	7.94181	190	7.94562	192	35	20	1.8	1.6	20	1.8	1.6	1.5			
36	.82133	219	.82422	220	.94377	190	.94754	192	36	30	2.7	2.5	30	2.7	2.5	2.2			
37	.82352	218	.82642	219	.94561	189	.94946	191	37	40	3.6	3.3	40	3.6	3.3	3.0			
38	.82570	217	.82862	219	.94751	189	.95137	191	38	50	4.6	4.1	50	4.6	4.1	3.7			
39	.82788	217	.83081	219	.94940	189	.95328	191	39										
40	7.83005	217	7.83300	218	7.95129	188	7.95519	190	40	6	0.3	0.3	6	0.3	0.3	0.2			
41	.83222	216	.83518	217	.95317	187	.95709	189	41	7	0.4	0.3	7	0.4	0.3	0.2			
42	.83438	215	.83735	217	.95505	188	.95898	189	42	8	0.4	0.4	8	0.4	0.4	0.3			
43	.83653	215	.83952	216	.95693	187	.96088	188	43	9	0.5	0.4	9	0.5	0.4	0.4			
44	.83868	214	.84169	216	.95880	186	.96276	188	44	10	0.6	0.5	10	0.6	0.5	0.4			
45	7.84083	214	7.84385	215	7.96066	186	7.96465	188	45	20	1.1	1.0	20	1.1	1.0	0.8			
46	.84297	213	.84600	215	.96253	186	.96653	188	46	30	1.7	1.5	30	1.7	1.5	1.2			
47	.84510	213	.84815	214	.96439	185	.96841	187	47	40	2.3	2.0	40	2.3	2.0	1.6			
48	.84723	212	.85030	213	.96624	185	.97028	187	48	50	2.9	2.5	50	2.9	2.5	2.1			
49	.84935	212	.85243	213	.96809	184	.97215	186	49										
50	7.85147	211	7.85457	213	7.96994	184	7.97401	186	50	6	0.1	0.1	6	0.1	0.1	0.0			
51	.85359	211	.85670	212	.97178	184	.97587	185	51	7	0.2	0.1	7	0.2	0.1	0.0			
52	.85570	210	.85882	211	.97362	183	.97773	185	52	8	0.2	0.1	8	0.2	0.1	0.0			
53	.85780	210	.86094	211	.97546	183	.97958	184	53	9	0.2	0.1	9	0.2	0.1	0.0			
54	.85990	209	.86305	211	.97729	183	.98143	184	54	10	0.2	0.1	10	0.2	0.1	0.1			
55	7.86199	209	7.86516	210	7.97912	182	7.98327	184	55	20	0.5	0.3	20	0.5	0.3	0.1			
56	.86408	208	.86726	210	.98094	182	.98512	183	56	30	0.7	0.5	30	0.7	0.5	0.2			
57	.86616	208	.86936	209	.98276	182	.98695	183	57	40	1.0	0.6	40	1.0	0.6	0.3			
58	.86824	207	.87146	208	.98458	181	.98879	183	58	50	1.2	0.8	50	1.2	0.8	0.4			
59	.87031	206	.87354	208	.98639	181	.99062	182	59										
60	7.87238	206	7.87563	208	7.98820	181	7.99244	182	60										
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.									

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS

8°

9°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.		
0	7.98820	180	7.99244	182	8.09031	160	8.09569	162	0	180	170	160
1	.99000	180	.99427	182	.09192	160	.09732	162	1	6 18.0	17.0	16.0
2	.99180	179	.99609	181	.09352	160	.09894	162	2	7 21.0	19.8	18.0
3	.99360	179	.99790	181	.09512	160	.10056	161	3	8 24.0	22.6	21.0
4	.99539	179	.99971	181	.09671	159	.10217	161	4	9 27.0	25.5	24.0
5	7.99718	178	8.00152	180	8.09830	159	8.10378	161	5	10 30.0	28.3	26.0
6	7.99897	178	.00332	180	.09989	158	.10539	160	6	20 60.0	56.6	53.0
7	8.00075	177	.00512	180	.10148	158	.10700	160	7	30 90.0	85.0	80.0
8	.00253	178	.00692	179	.10306	158	.10860	160	8	40 120.0	113.3	106.0
9	.00431	177	.00871	179	.10464	157	.11020	160	9	50 150.0	141.6	133.0
10	8.00608	176	8.01050	178	8.10622	157	8.11180	160	10	150	140	
11	.00784	176	.01229	178	.10779	157	.11340	159	11	6 15.0	14.0	
12	.00961	176	.01407	178	.10936	157	.11499	159	12	7 17.5	16.3	
13	.01137	176	.01585	177	.11093	156	.11658	158	13	8 20.0	18.6	
14	.01313	175	.01763	177	.11250	156	.11816	158	14	9 22.5	21.0	
15	8.01488	175	8.01940	177	8.11406	156	8.11975	158	15	10 25.0	23.3	
16	.01663	175	.02117	176	.11562	155	.12133	158	16	20 50.0	46.6	
17	.01838	174	.02293	176	.11718	155	.12291	157	17	30 75.0	70.0	
18	.02012	174	.02469	175	.11873	155	.12448	157	18	40 100.0	93.3	
19	.02186	173	.02645	175	.12029	155	.12605	157	19	50 125.0	116.6	
20	8.02359	173	8.02820	175	8.12184	154	8.12762	157	20			
21	.02533	173	.02995	175	.12338	154	.12919	156	21	9 15.0	14.0	
22	.02706	172	.03170	174	.12492	154	.13075	156	22	6 0.9	0.9	0.8
23	.02878	172	.03345	174	.12647	153	.13232	155	23	7 1.1	1.0	1.0
24	.03050	172	.03519	174	.12800	153	.13387	155	24	8 1.2	1.2	1.1
25	8.03222	172	8.03692	173	8.12954	153	8.13543	156	25	9 1.4	1.3	1.3
26	.03394	171	.03866	173	.13107	153	.13698	155	26	10 1.6	1.5	1.4
27	.03565	171	.04039	173	.13260	152	.13854	155	27	20 3.1	3.0	2.8
28	.03736	170	.04212	172	.13413	152	.14008	154	28	30 4.7	4.5	4.3
29	.03906	170	.04384	172	.13565	152	.14163	154	29	40 6.3	6.0	5.6
30	8.04076	170	8.04556	172	8.13717	152	8.14317	154	30	50 7.9	7.5	7.1
31	.04246	169	.04728	171	.13869	151	.14471	154	31			
32	.04416	169	.04899	171	.14021	151	.14625	153	32	8 0.8	0.7	0.7
33	.04585	169	.05070	170	.14172	151	.14778	153	33	7 0.9	0.9	0.8
34	.04754	168	.05241	170	.14323	151	.14932	153	34	8 1.0	1.0	0.9
35	8.04922	168	8.05411	170	8.14474	151	8.15085	153	35	9 1.2	1.1	1.0
36	.05090	168	.05581	170	.14625	150	.15237	152	36	10 1.3	1.2	1.1
37	.05258	167	.05751	169	.14775	150	.15390	152	37	20 2.6	2.5	2.3
38	.05426	167	.05921	169	.14925	149	.15542	152	38	30 4.0	3.7	3.5
39	.05593	167	.06090	169	.15075	149	.15694	152	39	40 5.3	5.0	4.6
40	8.05760	166	8.06259	168	8.15225	149	8.15846	152	40	50 6.6	6.2	5.8
41	.05926	166	.06427	168	.15374	149	.15997	151	41			
42	.06093	166	.06595	168	.15523	149	.16148	151	42	6 0.6	0.6	
43	.06259	165	.06763	167	.15672	148	.16299	150	43	7 0.7	0.7	
44	.06424	165	.06931	167	.15820	148	.16450	150	44	8 0.8	0.8	
45	8.06589	165	8.07098	167	8.15968	148	8.16600	150	45	9 1.0	0.9	
46	.06754	165	.07265	166	.16116	148	.16750	150	46	10 1.1	1.0	
47	.06919	164	.07431	166	.16264	147	.16900	149	47	20 2.1	2.0	
48	.07083	164	.07598	166	.16412	147	.17050	149	48	30 3.2	3.0	
49	.07247	164	.07764	165	.16559	147	.17199	149	49	40 4.3	4.0	
50	8.07411	163	8.07929	165	8.16706	146	8.17349	148	50	50 5.5	4.5	
51	.07575	163	.08095	165	.16852	146	.17497	148	51			
52	.07738	162	.08260	164	.16999	146	.17646	148	52	5 0.5	0.5	
53	.07900	162	.08424	164	.17145	146	.17795	148	53	6 0.6	0.6	
54	.08063	162	.08589	164	.17291	145	.17943	148	54	7 0.7	0.7	
55	8.08225	161	8.08753	163	8.17437	145	8.18091	147	55	8 0.8	0.8	
56	.08387	162	.08917	164	.17582	145	.18238	147	56	9 0.9	0.9	
57	.08549	161	.09081	163	.17728	145	.18386	147	57	10 0.9	0.8	
58	.08710	161	.09244	163	.17873	144	.18533	147	58	20 1.8	1.6	
59	.08871	160	.09407	162	.18017	144	.18680	146	59	30 2.7	2.5	
60	8.09031	160	8.09569	162	8.18162	144	8.18827	146	60	40 3.6	3.3	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.		

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

10°

11°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	8.18162	144	8.18827	146	8.26417	131	8.27223	133	0	
1	.18306	144	.18973	146	.26548	131	.27356	133	1	
2	.18450	144	.19120	146	.26679	131	.27490	133	2	
3	.18594	143	.19266	145	.26810	130	.27623	133	3	
4	.18738	143	.19411	145	.26941	130	.27756	133	4	
5	8.18881	143	8.19557	145	8.27071	130	8.27889	132	5	
6	.19024	142	.19702	145	.27201	130	.28021	132	6	
7	.19167	142	.19847	145	.27331	130	.28153	132	7	
8	.19309	142	.19992	144	.27461	129	.28286	132	8	
9	.19452	142	.20137	144	.27590	129	.28418	132	9	
10	8.19594	142	8.20281	144	8.27719	129	8.28550	131	10	
11	.19736	142	.20425	144	.27849	128	.28681	131	11	
12	.19878	141	.20569	144	.27977	128	.28813	131	12	
13	.20019	141	.20713	143	.28106	128	.28944	131	13	
14	.20160	141	.20857	143	.28235	128	.29075	131	14	
15	8.20301	140	8.21000	143	8.28363	128	8.29206	130	15	
16	.20442	140	.21143	143	.28491	128	.29336	130	16	
17	.20582	140	.21286	142	.28619	128	.29467	130	17	
18	.20723	140	.21428	142	.28747	127	.29597	130	18	
19	.20863	140	.21571	142	.28875	127	.29727	130	19	
20	8.21003	139	8.21713	142	8.29002	127	8.29857	130	20	
21	.21142	139	.21855	141	.29129	127	.29987	129	21	
22	.21282	139	.21996	141	.29256	127	.30117	129	22	
23	.21421	139	.22138	141	.29383	126	.30246	129	23	
24	.21560	138	.22279	141	.29510	126	.30375	129	24	
25	8.21698	138	8.22420	140	8.29636	126	8.30504	129	25	
26	.21837	138	.22561	140	.29763	126	.30633	128	26	
27	.21975	138	.22701	140	.29889	126	.30762	128	27	
28	.22113	137	.22842	140	.30015	125	.30890	128	28	
29	.22251	137	.22982	140	.30140	125	.31019	128	29	
30	8.22389	137	8.23122	140	8.30266	125	8.31147	128	30	
31	.22526	137	.23262	139	.30391	125	.31275	127	31	
32	.22663	136	.23401	139	.30516	125	.31402	127	32	
33	.22800	137	.23540	139	.30642	124	.31530	127	33	
34	.22937	136	.23679	139	.30766	124	.31657	127	34	
35	8.23073	136	8.23818	138	8.30891	124	8.31785	127	35	
36	.23209	136	.23957	138	.31015	124	.31912	127	36	
37	.23346	135	.24095	138	.31140	124	.32039	126	37	
38	.23481	136	.24234	138	.31264	124	.32165	126	38	
39	.23617	135	.24372	137	.31388	123	.32292	126	39	
40	8.23752	135	8.24509	137	8.31511	124	8.32418	126	40	
41	.23888	135	.24647	137	.31635	123	.32544	126	41	
42	.24023	135	.24784	137	.31758	123	.32670	126	42	
43	.24158	134	.24922	137	.31882	123	.32796	125	43	
44	.24292	134	.25059	136	.32005	123	.32922	125	44	
45	8.24426	134	8.25195	136	8.32128	122	8.33047	125	45	
46	.24561	134	.25332	136	.32250	122	.33173	125	46	
47	.24695	133	.25468	136	.32373	122	.33298	125	47	
48	.24828	133	.25604	136	.32495	122	.33423	124	48	
49	.24962	133	.25740	136	.32617	122	.33547	124	49	
50	8.25095	133	8.25876	135	8.32739	122	8.33672	124	50	
51	.25228	133	.26012	135	.32861	121	.33797	124	51	
52	.25361	132	.26147	135	.32983	121	.33921	124	52	
53	.25494	133	.26282	135	.33104	121	.34045	123	53	
54	.25627	132	.26417	134	.33225	121	.34169	123	54	
55	8.25759	132	8.26552	134	8.33347	121	8.34293	124	55	
56	.25891	132	.26686	134	.33468	120	.34417	123	56	
57	.26023	132	.26821	134	.33588	120	.34540	123	57	
58	.26155	131	.26955	134	.33709	120	.34663	123	58	
59	.26286	131	.27089	134	.33829	120	.34786	123	59	
60	8.26417	131	8.27223	134	8.33950	120	8.34909	123	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

130 120

6	13.0	12.0
7	15.1	14.0
8	17.3	16.0
9	19.5	18.0
10	21.6	20.0
20	43.3	40.0
30	65.0	60.0
40	86.6	80.0
50	108.3	100.0

4 3

6	0.4	0.3
7	0.5	0.4
8	0.6	0.5
9	0.7	0.6
10	0.7	0.6
20	1.5	1.1
30	2.2	2.0
40	3.0	2.6
50	3.7	3.2

3 2

6	0.3	0.2
7	0.3	0.3
8	0.4	0.3
9	0.4	0.4
10	0.5	0.4
20	1.0	0.8
30	1.5	1.1
40	2.0	1.6
50	2.5	2.1

2 1

6	0.2	0.1
7	0.2	0.2
8	0.2	0.2
9	0.3	0.2
10	0.3	0.2
20	0.6	0.5
30	1.0	0.7
40	1.3	1.0
50	1.6	1.2

1 0

6	0.1	0.0
7	0.1	0.0
8	0.1	0.0
9	0.1	0.1
10	0.1	0.1
20	0.3	0.1
30	0.5	0.2
40	0.6	0.3
50	0.8	0.4

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

12°

13°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	8.33950	120	8.34909	123	8.40875	110	8.42002	113	0	120 119 118
1	.34070	120	.35032	122	.40985	110	.42116	113	1	6 12.0 11.9 11.8
2	.34190	119	.35155	122	.41096	110	.42229	113	2	7 14.0 13.9 13.7
3	.34309	119	.35277	122	.41206	110	.42343	113	3	8 16.0 15.8 15.7
4	.34429	120	.35399	122	.41317	110	.42456	113	4	9 18.0 17.8 17.7
5	8.34549	119	8.35522	122	8.41427	110	8.42569	113	5	10 20.0 19.8 19.6
6	.34668	119	.35644	121	.41537	110	.42682	113	6	20 40.0 39.6 39.3
7	.34787	119	.35765	122	.41647	109	.42795	113	7	30 60.0 59.5 59.0
8	.34906	119	.35887	121	.41757	110	.42908	112	8	40 80.0 79.3 78.6
9	.35025	119	.36009	121	.41867	110	.43021	112	9	50 100.0 99.1 98.3
10	8.35143	118	8.36130	121	8.41976	109	8.43133	112	10	117 116 115
11	.35262	118	.36251	121	.42086	109	.43246	112	11	6 11.7 11.6 11.5
12	.35380	118	.36372	120	.42195	109	.43358	112	12	7 13.6 13.5 13.4
13	.35498	118	.36493	121	.42304	109	.43470	112	13	8 15.6 15.4 15.3
14	.35616	117	.36614	120	.42413	109	.43582	112	14	9 17.5 17.4 17.2
15	8.35734	118	8.36734	120	8.42522	108	8.43694	111	15	10 19.5 19.3 19.1
16	.35852	117	.36855	120	.42630	109	.43805	111	16	20 39.0 38.6 38.3
17	.35969	117	.36975	120	.42739	108	.43917	111	17	30 58.5 58.0 57.5
18	.36086	117	.37095	120	.42847	108	.44028	111	18	40 78.0 77.3 76.6
19	.36204	117	.37215	120	.42956	108	.44139	111	19	50 97.5 96.6 95.8
20	8.36321	116	8.37335	119	8.43064	108	8.44251	111	20	114 113 112
21	.36437	117	.37454	119	.43172	108	.44362	111	21	6 11.4 11.3 11.2
22	.36554	116	.37574	119	.43280	108	.44473	110	22	7 13.3 13.2 13.0
23	.36671	116	.37693	119	.43388	108	.44583	110	23	8 15.2 15.0 14.9
24	.36787	116	.37812	119	.43495	107	.44694	110	24	9 17.1 16.0 16.8
25	8.36903	116	8.37931	118	8.43603	107	8.44804	110	25	10 19.1 18.8 18.6
26	.37019	116	.38050	119	.43710	107	.44915	110	26	20 38.0 37.6 37.3
27	.37135	115	.38169	118	.43817	107	.45025	110	27	30 57.0 56.5 56.0
28	.37251	115	.38287	118	.43924	107	.45135	109	28	40 76.0 75.3 74.6
29	.37366	115	.38406	118	.44031	106	.45245	110	29	50 95.0 94.1 93.3
30	8.37482	115	8.38524	118	8.44138	107	8.45355	110	30	111 110 109
31	.37597	115	.38642	118	.44245	106	.45465	109	31	6 11.1 11.0 10.9
32	.37712	115	.38760	118	.44351	106	.45574	109	32	7 12.9 12.8 12.7
33	.37827	115	.38878	117	.44458	106	.45684	109	33	8 14.8 14.6 14.5
34	.37942	114	.38995	117	.44564	106	.45793	109	34	9 16.6 16.5 16.3
35	8.38057	114	8.39113	117	8.44670	106	8.45902	109	35	10 18.5 18.3 18.1
36	.38171	114	.39230	117	.44776	105	.46011	109	36	20 37.0 36.6 36.3
37	.38286	114	.39347	117	.44882	106	.46120	108	37	30 55.5 55.0 54.5
38	.38400	114	.39464	117	.44988	105	.46229	109	38	40 74.0 73.3 72.6
39	.38514	114	.39581	116	.45093	105	.46338	108	39	50 92.5 91.6 90.8
40	8.38628	113	8.39698	116	8.45199	105	8.46446	108	40	108 107 106
41	.38741	114	.39814	116	.45304	105	.46555	108	41	6 10.8 10.7 10.6
42	.38855	113	.39931	116	.45409	105	.46663	108	42	7 12.6 12.5 12.3
43	.38969	113	.40047	116	.45514	105	.46771	108	43	8 14.4 14.2 14.1
44	.39082	113	.40162	116	.45619	105	.46879	108	44	9 16.2 16.0 15.9
45	8.39195	113	8.40279	116	8.45724	104	8.46987	107	45	10 18.0 17.8 17.6
46	.39308	113	.40395	115	.45829	105	.47095	108	46	20 36.0 35.6 35.3
47	.39421	113	.40511	115	.45934	104	.47202	107	47	30 54.0 53.5 53.0
48	.39534	112	.40626	115	.46039	104	.47310	107	48	40 72.0 71.3 70.6
49	.39646	112	.40742	115	.46142	104	.47417	107	49	50 90.0 89.1 88.3
50	8.39758	112	8.40857	115	8.46247	104	8.47525	107	50	105 104 103
51	.39871	112	.40972	115	.46351	104	.47632	107	51	6 10.5 10.4 10.0
52	.39983	112	.41087	115	.46455	103	.47739	107	52	7 12.2 12.1 12.0
53	.40095	112	.41202	114	.46558	104	.47846	106	53	8 14.0 13.8 13.0
54	.40207	111	.41317	114	.46662	103	.47953	107	54	9 15.7 15.6 15.1
55	8.40318	111	8.41431	114	8.46766	103	8.48060	106	55	10 17.5 17.3 17.0
56	.40430	111	.41546	114	.46869	103	.48166	106	56	20 35.0 34.6 34.0
57	.40541	111	.41660	114	.46972	103	.48273	106	57	30 52.5 52.0 51.5
58	.40652	111	.41774	114	.47076	103	.48379	106	58	40 70.0 69.3 68.6
59	.40764	111	.41888	114	.47179	103	.48485	106	59	50 87.5 86.6 85.8
60	8.40875	111	8.42002	114	8.47282	103	8.48591	106	60	P. P.
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

14°										15°																			
Lg. Vers. D					Log. Exs. D					Lg. Vers. D					Log. Exs. D					P. P.									
0	8.47282	102	8.48591	106	8.53242	96	8.54748	99	0																				
1	.47384	103	.48697	106	.53338	95	.54847	99	1																				
2	.47487	102	.48803	105	.53434	96	.54946	99	2																				
3	.47590	102	.48909	105	.53530	95	.55045	99	3																				
4	.47692	102	.49014	105	.53625	95	.55144	99	4																				
5	8.47795	102	8.49120	105	8.53721	95	8.55243	99	5																				
6	.47897	102	.49225	105	.53816	95	.55342	99	6																				
7	.47999	102	.49331	105	.53911	95	.55441	98	7																				
8	.48101	102	.49436	105	.54007	95	.55539	98	8																				
9	.48203	102	.49541	105	.54102	95	.55638	98	9																				
10	8.48304	101	8.49646	105	8.54197	95	8.55736	98	10																				
11	.48406	101	.49750	104	.54291	94	.55834	98	11																				
12	.48507	101	.49855	104	.54386	94	.55933	98	12																				
13	.48609	101	.49960	104	.54481	94	.56031	98	13																				
14	.48710	101	.50064	104	.54575	94	.56129	98	14																				
15	8.48811	101	8.50168	104	8.54670	94	8.56226	97	15																				
16	.48912	101	.50273	104	.54764	94	.56324	97	16																				
17	.49013	100	.50377	104	.54858	94	.56422	97	17																				
18	.49114	101	.50481	104	.54952	94	.56519	97	18																				
19	.49215	100	.50585	103	.55046	94	.56617	97	19																				
20	8.49315	100	8.50688	104	8.55140	93	8.56714	97	20																				
21	.49415	100	.50792	103	.55234	94	.56812	97	21																				
22	.49516	100	.50896	103	.55328	93	.56909	97	22																				
23	.49616	100	.50999	103	.55421	93	.57006	97	23																				
24	.49716	100	.51102	103	.55515	93	.57103	97	24																				
25	8.49816	100	8.51205	103	8.55608	93	8.57200	96	25																				
26	.49916	99	.51309	103	.55701	93	.57296	96	26																				
27	.50015	100	.51412	102	.55795	93	.57393	96	27																				
28	.50115	99	.51514	103	.55888	93	.57490	96	28																				
29	.50215	99	.51617	102	.55981	93	.57586	96	29																				
30	8.50314	99	8.51720	102	8.56074	92	8.57682	96	30																				
31	.50413	99	.51822	102	.56166	92	.57779	96	31																				
32	.50512	99	.51925	102	.56259	92	.57875	96	32																				
33	.50611	99	.52027	102	.56352	92	.57971	96	33																				
34	.50710	98	.52129	102	.56444	92	.58067	95	34																				
35	8.50809	99	8.52231	102	8.56536	92	8.58163	95	35																				
36	.50908	98	.52333	101	.56629	92	.58259	95	36																				
37	.51006	98	.52435	102	.56721	92	.58354	95	37																				
38	.51105	98	.52537	101	.56813	92	.58450	95	38																				
39	.51203	98	.52638	101	.56905	92	.58546	95	39																				
40	8.51301	98	8.52740	101	8.56997	92	8.58641	95	40																				
41	.51399	98	.52841	101	.57089	91	.58736	95	41																				
42	.51497	98	.52943	101	.57180	91	.58832	95	42																				
43	.51595	97	.53044	101	.57272	91	.58927	95	43																				
44	.51693	98	.53145	101	.57363	91	.59022	95	44																				
45	8.51791	97	8.53246	101	8.57455	91	8.59117	94	45																				
46	.51888	97	.53347	101	.57546	91	.59211	94	46																				
47	.51986	97	.53448	100	.57637	91	.59306	94	47																				
48	.52083	97	.53548	100	.57728	91	.59401	94	48																				
49	.52180	97	.53649	100	.57819	91	.59495	94	49																				
50	8.52277	97	8.53749	100	8.57910	90	8.59590	94	50																				
51	.52374	97	.53850	100	.58001	91	.59684	94	51																				
52	.52471	96	.53950	100	.58092	90	.59779	94	52																				
53	.52568	97	.54050	100	.58182	90	.59873	94	53																				
54	.52665	96	.54150	100	.58273	90	.59967	94	54																				
55	8.52761	96	8.54250	100	8.58363	90	8.60061	94	55																				
56	.52858	96	.54350	99	.58453	90	.60155	94	56																				
57	.52954	96	.54449	100	.58544	90	.60249	93	57																				
58	.53050	96	.54549	99	.58634	90	.60342	94	58																				
59	.53146	96	.54649	99	.58724	90	.60436	93	59																				
60	8.53242	96	8.54748	99	8.58814	90	8.60530	93	60																				
Lg. Vers. D					Log. Exs. D					Lg. Vers. D					Log. Exs. D					P. P.									
																				</									

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

17°

Lg. Vers.			Log. Exs.			Lg. Vers.			Log. Exs.			P. P.		
D			D			D			D			D		
0	8.58814	90	8.60530	93	8.64043	84	8.65984	88	0			93	92	91
1	.58904	89	.60623	93	.64128	84	.66072	88	1	6	9.3	9.2	9.1	
2	.58993	90	.60716	93	.64212	84	.66160	88	2	7	10.8	10.7	10.6	
3	.59083	89	.60810	93	.64296	84	.66248	88	3	8	12.4	12.2	12.1	
4	.59173	89	.60903	93	.64381	84	.66336	88	4	9	13.9	13.8	13.6	
5	8.59262	89	8.60996	93	8.64465	84	8.66425	88	5	10	15.5	15.3	15.1	
6	.59351	89	.61089	93	.64549	84	.66512	87	6	20	31.0	30.6	30.5	
7	.59441	89	.61182	92	.64633	84	.66600	88	7	30	46.5	46.0	45.5	
8	.59530	89	.61275	93	.64717	84	.66688	88	8	40	62.0	61.3	60.6	
9	.59619	89	.61368	93	.64801	84	.66776	87	9	50	77.5	76.6	75.8	
10	8.59708	89	8.61460	92	8.64884	83	8.66863	87	10			90	89	88
11	.59797	89	.61553	92	.64968	84	.66951	88	11	6	9.0	8.9	8.8	
12	.59886	88	.61645	92	.65052	83	.67039	87	12	7	10.5	10.4	10.2	
13	.59974	88	.61738	92	.65135	83	.67126	87	13	8	12.0	11.8	11.7	
14	.60063	89	.61830	92	.65218	83	.67213	87	14	9	13.5	13.3	13.2	
15	8.60152	88	8.61922	92	8.65302	83	8.67301	87	15	10	15.0	14.8	14.6	
16	.60240	88	.62014	92	.65385	83	.67388	87	16	20	30.0	29.6	29.3	
17	.60328	88	.62106	92	.65468	83	.67475	87	17	30	45.0	44.5	44.0	
18	.60417	88	.62198	92	.65551	83	.67562	87	18	40	60.0	59.3	58.6	
19	.60505	88	.62290	91	.65634	83	.67649	87	19	50	75.0	74.1	73.3	
20	8.60593	88	8.62382	92	8.65717	83	8.67736	86	20			87	86	85
21	.60681	88	.62474	91	.65800	82	.67822	87	21	6	8.7	8.6	8.5	
22	.60769	88	.62565	91	.65883	82	.67909	86	22	7	10.1	10.0	9.9	
23	.60857	87	.62657	91	.65965	83	.67996	86	23	8	11.6	11.4	11.3	
24	.60944	87	.62748	91	.66048	82	.68082	86	24	9	13.0	12.9	12.7	
25	8.61032	87	8.62840	91	8.66131	82	8.68169	86	25	10	14.5	14.3	14.1	
26	.61119	87	.62931	91	.66213	82	.68255	86	26	20	29.0	28.6	28.3	
27	.61207	87	.63022	91	.66295	82	.68341	86	27	30	43.5	43.0	42.5	
28	.61294	87	.63113	91	.66378	82	.68428	86	28	40	58.0	57.3	56.6	
29	.61381	87	.63204	90	.66460	82	.68514	86	29	50	72.5	71.6	70.8	
30	8.61469	87	8.63295	91	8.66542	82	8.68600	86	30			84	83	82
31	.61556	87	.63386	91	.66624	82	.68686	85	31	6	8.4	8.3	8.2	
32	.61643	87	.63477	90	.66706	82	.68772	86	32	7	9.8	9.7	9.5	
33	.61730	86	.63567	90	.66788	82	.68858	86	33	8	11.2	11.0	10.9	
34	.61816	87	.63658	90	.66870	81	.68944	85	34	9	12.6	12.4	12.3	
35	8.61903	86	8.63743	90	8.66951	81	8.69029	86	35	10	14.0	13.8	13.6	
36	.61990	86	.63839	90	.67033	82	.69115	85	36	20	28.0	27.6	27.3	
37	.62076	86	.63929	90	.67115	81	.69201	85	37	30	42.0	41.5	41.0	
38	.62163	86	.64019	90	.67196	81	.69286	85	38	40	56.0	55.3	54.6	
39	.62249	86	.64109	90	.67277	81	.69372	85	39	50	70.0	69.1	68.3	
40	8.62336	86	8.64199	90	8.67359	81	8.69457	85	40			81	80	79
41	.62422	86	.64289	90	.67440	81	.69542	85	41	6	8.1	8.0	7.9	
42	.62508	86	.64379	90	.67521	81	.69627	85	42	7	9.4	9.3	9.2	
43	.62594	86	.64469	89	.67602	81	.69712	85	43	8	10.8	10.6	10.5	
44	.62680	86	.64559	89	.67683	81	.69798	85	44	9	12.1	12.0	11.8	
45	8.62766	86	8.64649	89	8.67764	81	8.69883	84	45	10	13.5	13.3	13.1	
46	.62852	85	.64738	89	.67845	80	.69967	85	46	20	27.0	26.6	26.3	
47	.62937	85	.64828	89	.67926	81	.70052	85	47	30	40.5	40.0	39.5	
48	.63023	85	.64917	89	.68007	80	.70137	84	48	40	54.0	53.3	52.6	
49	.63108	85	.65006	89	.68087	80	.70222	84	49	50	67.5	66.6	65.8	
50	8.63194	85	8.65096	89	8.68168	80	8.70306	84	50			0	0	
51	.63279	85	.65185	89	.68248	80	.70391	84	51	6	0.0	0.0	0.0	
52	.63364	85	.65274	89	.68329	80	.70475	84	52	7	0.0	0.0	0.0	
53	.63449	85	.65363	89	.68409	80	.70560	84	53	8	0.0	0.0	0.0	
54	.63534	85	.65452	89	.68489	80	.70644	84	54	9	0.0	0.0	0.0	
55	8.63619	85	8.65541	88	8.68569	80	8.70728	84	55	10	0.1	0.1	0.1	
56	.63704	85	.65629	88	.68650	80	.70813	84	56	20	0.1	0.1	0.1	
57	.63789	85	.65718	89	.68730	80	.70897	84	57	30	0.2	0.2	0.2	
58	.63874	84	.65807	88	.68810	79	.70981	84	58	40	0.3	0.3	0.3	
59	.63959	84	.65895	88	.68889	80	.71065	84	59	50	0.4	0.4	0.4	
60	8.64043	84	8.65984	88	8.68969	80	8.71149	84	60					
Lg. Vers.			Log. Exs.			Lg. Vers.			Log. Exs.			P. P.		
D			D			D			D			D		

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

18°

19°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	8.68969	79	8.71149	83	8.73625	75	8.76058	79	0	
1	.69049	80	.71232	84	.73700	75	.76137	80	1	
2	.69129	79	.71316	83	.73775	75	.76217	79	2	
3	.69208	79	.71400	84	.73851	75	.76297	79	3	
4	.69288	79	.71484	84	.73926	75	.76376	79	4	84 83 82
5	8.69367	79	8.71567	83	8.74001	75	8.76456	80	5	6 8.4 8.3 8.2
6	.69446	79	.71651	83	.74076	75	.76536	79	6	7 9.8 9.7 9.5
7	.69526	79	.71734	83	.74151	75	.76615	79	7	8 11.2 11.0 10.9
8	.69605	79	.71817	83	.74226	75	.76694	79	8	9 12.6 12.4 12.3
9	.69684	79	.71901	83	.74301	75	.76774	79	9	10 14.0 13.8 13.6
10	8.69763	79	8.71984	83	8.74376	75	8.76853	79	10	20 22.0 21.6 21.3
11	.69842	79	.72067	83	.74451	74	.76932	79	11	30 24.0 23.5 23.0
12	.69921	79	.72150	83	.74526	74	.77011	79	12	40 26.0 25.3 24.6
13	.70000	79	.72233	83	.74600	74	.77090	79	13	50 28.0 27.6 27.3
14	.70079	78	.72316	83	.74675	74	.77169	79	14	60 30.0 29.1 28.3
15	8.70157	78	8.72399	82	8.74749	74	8.77248	79	15	81 80 79
16	.70236	78	.72481	82	.74824	74	.77327	79	16	6 8.1 8.0 7.9
17	.70314	78	.72564	82	.74898	74	.77406	79	17	7 9.4 9.3 9.2
18	.70393	78	.72647	82	.74973	74	.77485	79	18	8 10.8 10.6 10.5
19	.70471	78	.72729	82	.75047	74	.77563	79	19	9 12.1 12.0 11.8
20	8.70550	78	8.72812	82	8.75121	74	8.77642	78	20	10 13.5 13.3 13.1
21	.70628	78	.72894	82	.75195	74	.77720	79	21	20 27.0 26.6 26.3
22	.70706	78	.72977	82	.75269	74	.77799	79	22	30 40.5 40.0 39.5
23	.70784	78	.73059	82	.75343	74	.77877	78	23	40 54.0 53.3 52.6
24	.70862	78	.73141	82	.75417	74	.77956	78	24	50 67.5 66.6 65.3
25	8.70940	78	8.73223	82	8.75491	73	8.78034	78	25	78 77 76
26	.71018	77	.73306	82	.75565	73	.78112	78	26	6 7.8 7.7 7.6
27	.71096	77	.73388	82	.75639	73	.78191	78	27	7 9.1 9.0 8.8
28	.71174	77	.73470	81	.75712	73	.78269	78	28	8 10.4 10.2 10.1
29	.71251	77	.73551	81	.75786	73	.78347	78	29	9 11.7 11.5 11.4
30	8.71329	77	8.73633	82	8.75860	73	8.78425	78	30	10 13.0 12.8 12.6
31	.71406	77	.73715	81	.75933	73	.78503	78	31	20 26.0 25.6 25.3
32	.71484	77	.73797	81	.76006	73	.78581	78	32	30 39.0 38.5 38.0
33	.71561	77	.73878	81	.76080	73	.78659	78	33	40 52.0 51.1 50.6
34	.71639	77	.73960	81	.76153	73	.78736	77	34	50 65.0 64.1 63.3
35	8.71716	77	8.74041	81	8.76226	73	8.78814	78	35	75 74 73
36	.71793	77	.74123	81	.76300	73	.78892	77	36	6 7.5 7.4 7.3
37	.71870	77	.74204	81	.76373	73	.78969	77	37	7 8.7 8.6 8.5
38	.71947	77	.74286	81	.76446	73	.79047	77	38	8 10.0 9.8 9.7
39	.72024	77	.74367	81	.76519	73	.79124	77	39	9 11.2 11.1 10.9
40	8.72101	76	8.74448	81	8.76592	72	8.79202	77	40	10 12.5 12.3 12.1
41	.72178	76	.74529	81	.76664	72	.79279	77	41	20 25.0 24.6 24.3
42	.72255	76	.74610	81	.76737	72	.79357	77	42	30 37.5 37.0 36.5
43	.72331	76	.74691	80	.76810	72	.79434	77	43	40 50.0 49.3 48.6
44	.72408	76	.74772	80	.76883	72	.79511	77	44	50 62.5 61.6 60.8
45	8.72485	76	8.74853	81	8.76955	72	8.79588	77	45	72 71 70
46	.72561	76	.74934	81	.77028	72	.79665	77	46	6 7.2 7.1 7.0
47	.72637	76	.75014	80	.77100	72	.79742	77	47	7 8.4 8.3 8.2
48	.72714	76	.75095	80	.77173	72	.79819	77	48	8 9.6 9.4 9.3
49	.72790	76	.75175	80	.77245	72	.79896	77	49	9 10.8 10.6 10.5
50	8.72866	76	8.75256	80	8.77317	72	8.79973	77	50	10 12.0 11.8 11.6
51	.72942	76	.75336	80	.77390	72	.80050	76	51	20 24.0 23.6 23.3
52	.73018	76	.75417	80	.77462	72	.80126	76	52	30 36.0 35.5 35.0
53	.73094	76	.75497	80	.77534	72	.80203	76	53	40 48.0 47.3 46.6
54	.73170	76	.75577	80	.77606	72	.80280	76	54	50 60.0 59.1 58.3
55	8.73246	76	8.75658	80	8.77678	72	8.80356	76	55	71 70 69
56	.73322	76	.75738	80	.77750	72	.80433	76	56	6 7.1 7.0 6.9
57	.73398	75	.75818	80	.77822	71	.80509	76	57	7 8.4 8.3 8.2
58	.73473	76	.75898	80	.77893	72	.80586	76	58	8 9.6 9.4 9.3
59	.73549	75	.75978	80	.77965	71	.80662	76	59	9 10.8 10.6 10.5
60	8.73625	75	8.76058	80	8.78037	71	8.80738	76	60	10 12.0 11.8 11.6
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

20°

21°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	8.78037	71	8.80738	76	8.82229	68	8.85214	73	0	
1	.78108	71	.80814	76	.82297	68	.85287	73	1	
2	.78180	71	.80891	76	.82366	68	.85360	73	2	
3	.78251	71	.80967	76	.82434	68	.85433	73	3	
4	.78323	71	.81043	76	.82502	68	.85506	73	4	
5	8.78394	71	8.81119	76	8.82569	67	8.85579	73	5	76 75 74
6	.78466	71	.81195	76	.82637	68	.85651	73	6	7.6 7.5 7.4
7	.78537	71	.81271	75	.82705	67	.85724	72	7	8.8 8.7 8.6
8	.78608	71	.81346	76	.82773	68	.85797	72	8	10.1 10.0 9.8
9	.78679	71	.81422	76	.82841	68	.85869	72	9	11.4 11.2 11.1
10	8.78750	71	8.81498	75	8.82908	67	8.85942	72	10	12.6 12.5 12.3
11	.78821	71	.81573	76	.82976	67	.86014	72	11	20 25.3 25.0
12	.78892	71	.81649	75	.83043	67	.86087	72	12	30 38.0 37.5
13	.78963	71	.81725	75	.83111	67	.86159	72	13	40 50.6 50.0
14	.79034	70	.81800	75	.83178	67	.86231	72	14	50 63.3 62.5
15	8.79105	70	8.81876	75	8.83246	67	8.86304	72	15	73 72 71
16	.79175	70	.81951	75	.83313	67	.86376	72	16	6 7.3 7.2
17	.79246	70	.82026	75	.83380	67	.86448	72	17	7 8.5 8.4
18	.79317	70	.82102	75	.83447	67	.86520	72	18	8 9.7 9.6
19	.79387	70	.82177	75	.83515	67	.86592	72	19	9 10.9 10.8
20	8.79458	70	8.82252	75	8.83582	67	8.86664	72	20	10 12.1 12.0
21	.79528	70	.82327	75	.83649	67	.86736	72	21	20 24.3 24.0
22	.79598	70	.82402	75	.83716	67	.86808	72	22	30 36.5 36.0
23	.79669	70	.82477	74	.83783	67	.86880	72	23	40 48.6 48.0
24	.79739	70	.82552	75	.83850	67	.86952	72	24	50 60.8 60.0
25	8.79809	70	8.82627	75	8.83916	66	8.87024	72	25	70 69 68
26	.79879	70	.82702	74	.83983	66	.87095	72	26	6 7.0 6.9
27	.79949	70	.82776	75	.84050	66	.87167	72	27	7 8.1 8.0
28	.80019	70	.82851	75	.84117	66	.87239	71	28	8 9.3 9.2
29	.80089	70	.82926	74	.84183	66	.87310	71	29	9 10.5 10.3
30	8.80159	70	8.83000	74	8.84250	66	8.87382	71	30	10 11.6 11.5
31	.80229	69	.83075	74	.84316	66	.87453	71	31	20 23.3 23.0
32	.80299	69	.83149	74	.84383	66	.87525	71	32	30 35.0 34.5
33	.80369	69	.83224	74	.84449	66	.87596	71	33	40 46.6 46.0
34	.80438	69	.83298	74	.84515	66	.87668	71	34	50 58.3 57.5
35	8.80508	69	8.83373	74	8.84582	66	8.87739	71	35	67 66 65
36	.80577	69	.83447	74	.84648	66	.87810	71	36	6 6.7 6.6
37	.80647	69	.83521	74	.84714	66	.87881	71	37	7 7.8 7.7
38	.80716	69	.83595	74	.84780	66	.87953	71	38	8 8.9 8.8
39	.80786	69	.83670	74	.84846	66	.88024	71	39	9 10.0 9.9
40	8.80855	69	8.83744	74	8.84912	66	8.88095	71	40	10 11.1 11.0
41	.80924	69	.83818	74	.84978	66	.88166	71	41	20 22.3 22.0
42	.80993	69	.83892	74	.85044	66	.88237	71	42	30 33.5 33.0
43	.81063	69	.83966	73	.85110	66	.88308	70	43	40 44.6 44.0
44	.81132	69	.84039	74	.85176	66	.88378	71	44	50 55.8 55.0
45	8.81201	69	8.84113	73	8.85242	65	8.88449	71	45	6 6.7 6.6
46	.81270	69	.84187	73	.85308	65	.88520	70	46	7 7.8 7.7
47	.81339	69	.84261	73	.85373	65	.88591	70	47	8 8.9 8.8
48	.81407	69	.84334	73	.85439	65	.88661	71	48	9 10.0 9.9
49	.81476	69	.84408	73	.85505	65	.88732	71	49	10 11.1 11.0
50	8.81545	69	8.84481	73	8.85570	65	8.88803	70	50	20 22.3 22.0
51	.81614	69	.84555	73	.85626	65	.88873	70	51	30 33.5 33.0
52	.81682	69	.84628	73	.85701	65	.88944	70	52	40 44.6 44.0
53	.81751	69	.84702	73	.85766	65	.89014	70	53	50 55.8 55.0
54	.81819	69	.84775	73	.85832	65	.89085	70	54	6 6.7 6.6
55	8.81888	69	8.84848	73	8.85897	65	8.89155	70	55	7 7.8 7.7
56	.81956	69	.84922	73	.85962	65	.89225	70	56	8 8.9 8.8
57	.82025	69	.84995	73	.86027	65	.89295	70	57	9 10.0 9.9
58	.82093	69	.85068	73	.86092	65	.89366	70	58	10 11.1 11.0
59	.82161	69	.85141	73	.86158	65	.89436	70	59	20 22.3 22.0
60	8.82229	69	8.85214	73	8.86223	65	8.89506	70	60	30 33.5 33.0
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

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TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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												P. P.			
Lg. Vers.		D	Log. Exs.	D	Lg. Vers.		D	Log. Exs.	D						
0	8.93679	59	8.97606	65	8.97170	57	9.01443	62	0						
1	.93738	59	.97671	65	.97227	56	.01505	63	1						
2	.93797	59	.97736	65	.97284	57	.01568	62	2						
3	.93857	59	.97801	64	.97341	57	.01631	63	3						
4	.93916	59	.97865	65	.97398	57	.01694	62	4						
5	8.93975	59	8.97930	65	8.97455	57	9.01756	63	5						
6	.94034	59	.97995	64	.97511	57	.01819	62	6						
7	.94094	59	.98060	65	.97568	56	.01882	62	7						
8	.94153	59	.98125	65	.97625	56	.01944	63	8						
9	.94212	59	.98190	65	.97681	56	.02007	62	9						
10	8.94271	59	8.98254	64	8.97738	57	9.02070	62	10						
11	.94330	59	.98319	64	.97795	56	.02132	62	11						
12	.94389	59	.98383	65	.97851	56	.02195	62	12						
13	.94448	58	.98448	64	.97908	56	.02257	62	13						
14	.94506	58	.98513	64	.97964	56	.02319	62	14						
15	8.94505	59	8.98577	64	8.98020	56	9.02382	62	15						
16	.94624	59	.98642	64	.98077	56	.02444	62	16						
17	.94683	58	.98706	64	.98133	56	.02506	62	17						
18	.94742	58	.98770	64	.98190	56	.02569	62	18						
19	.94800	58	.98835	64	.98246	56	.02631	62	19						
20	8.94859	58	8.98899	64	8.98302	56	9.02693	62	20						
21	.94917	58	.98963	64	.98358	56	.02755	62	21						
22	.94976	58	.99028	64	.98414	56	.02817	62	22						
23	.95034	58	.99092	64	.98470	56	.02880	62	23						
24	.95093	58	.99156	64	.98527	56	.02942	62	24						
25	8.95151	58	8.99220	64	8.98583	56	9.03004	62	25						
26	.95210	58	.99284	64	.98639	56	.03066	62	26						
27	.95268	58	.99348	64	.98695	56	.03128	62	27						
28	.95326	58	.99412	64	.98750	56	.03190	62	28						
29	.95384	58	.99476	64	.98806	56	.03252	62	29						
30	8.95443	58	8.99540	64	8.98862	56	9.03313	62	30						
31	.95501	58	.99604	64	.98918	55	.03375	62	31						
32	.95559	58	.99668	64	.98974	56	.03437	62	32						
33	.95617	58	.99732	63	.99030	55	.03499	62	33						
34	.95675	58	.99796	63	.99085	55	.03561	62	34						
35	8.95733	58	8.99860	64	8.99141	56	9.03622	62	35						
36	.95791	57	.99923	64	.99197	55	.03684	62	36						
37	.95849	57	8.99987	63	.99252	55	.03746	62	37						
38	.95907	58	9.00051	63	.99308	55	.03807	62	38						
39	.95965	58	.00114	63	.99363	55	.03869	62	39						
40	8.96023	57	9.00178	64	8.99419	55	9.03930	62	40						
41	.96080	57	.00242	63	.99474	55	.03992	62	41						
42	.96138	57	.00305	63	.99529	55	.04053	62	42						
43	.96196	58	.00369	63	.99585	55	.04115	62	43						
44	.96253	57	.00432	63	.99640	55	.04176	62	44						
45	8.96311	57	9.00495	63	8.99695	55	9.04238	62	45						
46	.96368	57	.00559	63	.99751	55	.04299	62	46						
47	.96426	57	.00622	63	.99806	55	.04360	62	47						
48	.96483	57	.00686	63	.99861	55	.04421	62	48						
49	.96541	57	.00749	63	.99916	55	.04483	62	49						
50	8.96598	57	9.00812	63	8.99971	55	9.04544	62	50						
51	.96656	57	.00875	63	9.00026	55	.04605	62	51						
52	.96713	57	.00938	63	.00081	55	.04666	62	52						
53	.96770	57	.01002	63	.00136	55	.04727	62	53						
54	.96827	57	.01065	63	.00191	55	.04788	62	54						
55	8.96885	57	9.01128	63	9.00246	55	9.04850	62	55						
56	.96942	57	.01191	63	.00301	55	.04911	62	56						
57	.96999	57	.01254	63	.00356	55	.04972	62	57						
58	.97056	57	.01317	63	.00411	55	.05033	62	58						
59	.97113	57	.01380	63	.00466	55	.05093	62	59						
60	8.97170	57	9.01443	63	9.00520	54	9.05154	62	60						
Lg. Vers.		D	Log. Exs.	D	Lg. Vers.		D	Log. Exs.	D						
												P. P.			

												P. P.			
Lg. Vers.		D	Log. Exs.	D	Lg. Vers.		D	Log. Exs.	D						
0	8.93679	59	8.97606	65	8.97170	57	9.01443	62	0						
1	.93738	59	.97671	65	.97227	56	.01505	63	1						
2	.93797	59	.97736	65	.97284	57	.01568	62	2						
3	.93857	59	.97801	64	.97341	57	.01631	63	3						
4	.93916	59	.97865	65	.97398	57	.01694	62	4						
5	8.93975	59	8.97930	65	8.97455	57	9.01756	63	5						
6	.94034	59	.97995	64	.97511	57	.01819	62	6						
7	.94094	59	.98060	65	.97568	56	.01882	62	7						
8	.94153	59	.98125	65	.97625	56	.01944	63	8						
9	.94212	59	.98190	65	.97681	56	.02007	62	9						
10	8.94271	59	8.98254	64	8.97738	57	9.02070	62	10						
11	.94330	59	.98319	64	.97795	56	.02132	62	11						
12	.94389	59	.98383	65	.97851	56	.02195	62	12						
13	.94448	58	.98448	64	.97908	56	.02257	62	13						
14	.94506	58	.98513	64	.97964	56	.02319	62	14						
15	8.94505	59	8.98577	64	8.98020	56	9.02382	62	15						
16	.94624	59	.98642	64	.98077	56	.02444	62	16						
17	.94683	58	.98706	64	.98133	56	.02506	62	17						
18	.94742	58	.98770	64	.98190	56	.02569	62	18						
19	.94800	58	.98835	64	.98246	56	.02631	62	19						
20	8.94859	58	8.98899	64	8.98302	56	9.02693	62	20						
21	.94917	58	.98963	64	.98358	56	.02755	62	21						
22	.94976	58	.99028	64	.98414	56	.02817	62	22						
23	.95034	58	.99092	64	.98470	56	.02880	62	23						
24	.95093	58	.99156	64	.98527	56	.02942	62	24						
25	8.95151	58	8.99220	64	8.98583	56	9.03004	62	25						
26	.95210	58	.99284	64	.98639	56	.03066	62	26						
27	.95268	58	.99348	64	.98695	56	.03128	62	27						
28	.95326	58	.99412	64	.98750	56	.03190	62	28						
29	.95384	58	.99476	64	.98806	56	.03252	62	29						
30	8.95443	58	8.99540	64	8.98862	56	9.03313	62	30						
31	.95501	58	.99604	64	.98918	55	.03375	62	31						
32	.95559	58	.99668	64	.98974	56	.03437	62	32						
33	.95617	58	.99732	63	.99030	55	.03499	62	33						
34	.95675	58	.99796	63	.99085	55	.03561	62	34						
35	8.95733	58	8.99860	64	8.99141	56	9.03622	62	35						
36	.95791	57	.99923	64	.99197	55	.03684	62	36						
37	.95849	57	8.99987	63	.99252	55	.03746	62	37						
38	.95907	58	9.00051	63	.99308	55	.03807	62	38						
39	.95965	58	.00114	63	.99363	55	.03869	62	39						
40	8.96023	57	9.00178	64	8.99419	55	9.03930	62	40						
41	.96080	57	.00242	63	.99474	55	.03992	62	41						
42	.96138	57	.00305	63	.99529	55	.04053	62	42						
43	.96196	58	.00369	63	.99585	55	.04115	62	43						
44	.96253	57	.00432	63	.99640	55	.04176	62	44						
45	8.96311	57	9.00495	63	8.99695	55	9.04238	62	45						
46	.96368	57	.00559	63	.99751	55	.04299	62	46						
47	.96426	57	.00622	63	.99806	55	.04360	62	47						
48	.96483	57	.00686	63	.99861	55	.04421	62	48						
49	.96541	57	.00749	63	.99916	55	.04483	62	49						

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TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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29°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.06838	50	9.12244	57	9.09823	49	9.15641	56	0	57 57 56
1	.06888	51	.12302	57	.09872	48	.15697	55	1	6 5.7 5.7 5.6
2	.06939	50	.12359	57	.09920	49	.15752	56	2	7 6.7 6.6 6.6
3	.06990	50	.12416	57	.09969	48	.15808	55	3	8 7.6 7.6 7.5
4	.07040	50	.12474	57	.10018	48	.15864	55	4	9 8.6 8.5 8.5
5	9.07091	50	9.12531	57	9.10067	49	9.15920	56	5	10 9.6 9.5 9.4
6	.07141	50	.12588	57	.10115	48	.15975	55	6	20 19.1 19.0 18.8
7	.07192	50	.12645	57	.10164	48	.16031	56	7	30 28.7 28.5 28.2
8	.07242	50	.12703	57	.10213	49	.16087	55	8	40 38.3 38.0 37.6
9	.07293	50	.12760	57	.10261	48	.16142	55	9	50 47.9 47.5 47.1
10	9.07343	50	9.12817	57	9.10310	48	9.16198	56	10	56 55 55
11	.07393	50	.12874	57	.10358	48	.16254	55	11	6 5.6 5.5 5.5
12	.07444	50	.12931	57	.10407	48	.16309	55	12	7 6.5 6.5 6.4
13	.07494	50	.12988	57	.10455	48	.16365	55	13	8 7.4 7.4 7.3
14	.07544	50	.13045	57	.10504	48	.16420	55	14	9 8.4 8.3 8.2
15	9.07594	50	9.13102	57	9.10552	48	9.16476	55	15	10 9.3 9.2 9.1
16	.07644	50	.13159	57	.10601	48	.16531	55	16	20 18.6 18.5 18.3
17	.07695	50	.13216	56	.10649	48	.16587	55	17	30 28.0 27.7 27.5
18	.07745	50	.13273	57	.10697	48	.16642	55	18	40 37.3 37.0 36.6
19	.07795	50	.13330	57	.10746	48	.16698	55	19	50 46.6 46.2 45.8
20	9.07845	50	9.13387	57	9.10794	48	9.16753	55	20	54 54
21	.07895	50	.13444	56	.10842	48	.16808	55	21	6 5.4 5.4 5.4
22	.07945	50	.13500	56	.10890	48	.16864	55	22	7 6.3 6.3 6.3
23	.07995	50	.13557	57	.10939	48	.16919	55	23	8 7.2 7.2 7.2
24	.08045	50	.13614	56	.10987	48	.16974	55	24	9 8.2 8.1 8.1
25	9.08095	50	9.13671	57	9.11035	48	9.17029	55	25	10 9.1 9.0 9.0
26	.08145	50	.13727	56	.11083	48	.17085	55	26	20 18.1 18.0 18.0
27	.08195	49	.13784	57	.11131	48	.17140	55	27	30 27.2 27.0 27.0
28	.08244	49	.13841	56	.11179	48	.17195	55	28	40 36.3 36.0 36.0
29	.08294	50	.13897	56	.11227	48	.17250	55	29	50 45.4 45.0 45.0
30	9.08344	49	9.13954	57	9.11275	48	9.17305	55	30	51 50 50
31	.08394	49	.14011	56	.11323	48	.17361	55	31	6 5.1 5.0 5.0
32	.08443	49	.14067	56	.11371	47	.17416	55	32	7 5.9 5.9 5.8
33	.08493	49	.14124	56	.11419	47	.17471	55	33	8 6.8 6.7 6.6
34	.08543	50	.14180	56	.11467	48	.17526	55	34	9 7.6 7.6 7.5
35	9.08592	49	9.14237	56	9.11515	47	9.17581	55	35	10 8.5 8.4 8.3
36	.08642	49	.14293	56	.11562	48	.17636	55	36	20 17.0 16.8 16.6
37	.08691	49	.14350	56	.11610	48	.17691	55	37	30 25.5 25.2 25.0
38	.08741	49	.14406	56	.11658	47	.17746	55	38	40 34.0 33.6 33.3
39	.08790	49	.14462	56	.11706	47	.17801	55	39	50 42.5 42.1 41.6
40	9.08840	49	9.14519	56	9.11754	47	9.17856	55	40	49 49 48
41	.08889	49	.14575	56	.11801	47	.17910	55	41	6 4.9 4.9 4.8
42	.08939	49	.14631	56	.11849	47	.17965	55	42	7 5.8 5.7 5.6
43	.08988	49	.14688	56	.11897	47	.18020	54	43	8 6.6 6.5 6.4
44	.09037	49	.14744	56	.11944	47	.18075	54	44	9 7.4 7.3 7.3
45	9.09087	49	9.14800	56	9.11992	47	9.18130	55	45	10 8.2 8.1 8.1
46	.09136	49	.14856	56	.12039	47	.18185	54	46	20 16.5 16.3 16.1
47	.09185	49	.14913	56	.12087	47	.18239	55	47	30 24.7 24.5 24.2
48	.09234	49	.14969	56	.12134	47	.18294	54	48	40 33.0 32.6 32.3
49	.09284	49	.15025	56	.12182	47	.18349	54	49	50 41.2 40.8 40.4
50	9.09333	49	9.15081	56	9.12229	47	9.18403	55	50	48 47 47
51	.09382	49	.15137	56	.12277	47	.18458	54	51	6 4.8 4.7 4.7
52	.09431	49	.15193	56	.12324	47	.18513	54	52	7 5.6 5.5 5.5
53	.09480	49	.15249	56	.12371	47	.18567	54	53	8 6.4 6.3 6.2
54	.09529	49	.15305	56	.12419	47	.18622	54	54	9 7.2 7.1 7.0
55	9.09578	49	9.15361	56	9.12466	47	9.18676	54	55	10 8.0 7.9 7.8
56	.09627	49	.15417	56	.12513	47	.18731	55	56	20 16.0 15.8 15.6
57	.09676	48	.15473	56	.12560	47	.18786	54	57	30 24.0 23.7 23.5
58	.09725	49	.15529	55	.12608	47	.18840	54	58	40 32.0 31.6 31.3
59	.09774	49	.15585	56	.12655	47	.18894	54	59	50 40.0 39.6 39.1
60	9.09823	49	9.15641	56	9.12702	47	9.18949	54	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

30°

31°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.12702	47	9.18949	54	9.15483	45	9.22176	53	0	
1	.12749	47	.19003	54	.15528	45	.22229	53	1	
2	.12796	47	.19058	54	.15574	45	.22282	53	2	
3	.12843	47	.19112	54	.15619	45	.22335	53	3	
4	.12890	47	.19167	54	.15665	45	.22388	53	4	
5	9.12937	47	9.19221	54	9.15710	45	9.22441	53	5	
6	.12984	47	.19275	54	.15755	45	.22494	53	6	
7	.13031	47	.19329	54	.15801	45	.22547	53	7	
8	.13078	47	.19384	54	.15846	45	.22600	53	8	
9	.13125	47	.19438	54	.15891	45	.22653	53	9	
10	9.13172	46	9.19492	54	9.15937	45	9.22706	53	10	
11	.13219	46	.19546	54	.15982	45	.22759	53	11	
12	.13266	46	.19601	54	.16027	45	.22812	53	12	
13	.13313	46	.19655	54	.16073	45	.22865	53	13	
14	.13359	46	.19709	54	.16118	45	.22918	52	14	
15	9.13406	46	9.19763	54	9.16163	45	9.22971	53	15	
16	.13453	46	.19817	54	.16208	45	.23024	52	16	
17	.13500	46	.19871	54	.16253	45	.23076	52	17	
18	.13546	46	.19925	54	.16298	45	.23129	52	18	
19	.13593	46	.19979	54	.16343	45	.23182	52	19	
20	9.13639	46	9.20033	54	9.16388	45	9.23235	52	20	
21	.13686	46	.20087	54	.16434	45	.23287	52	21	
22	.13733	46	.20141	54	.16479	45	.23340	52	22	
23	.13779	46	.20195	54	.16523	44	.23393	52	23	
24	.13826	46	.20249	53	.16568	45	.23446	52	24	
25	9.13872	46	9.20303	54	9.16613	45	9.23498	52	25	
26	.13919	46	.20357	54	.16658	45	.23551	52	26	
27	.13965	46	.20411	54	.16703	45	.23603	52	27	
28	.14011	46	.20465	53	.16748	44	.23656	52	28	
29	.14058	46	.20518	54	.16793	45	.23709	52	29	
30	9.14104	46	9.20572	53	9.16838	44	9.23761	52	30	
31	.14151	46	.20626	54	.16882	45	.23814	52	31	
32	.14197	46	.20680	54	.16927	44	.23866	52	32	
33	.14243	46	.20733	54	.16972	45	.23919	52	33	
34	.14289	46	.20787	53	.17017	44	.23971	52	34	
35	9.14336	46	9.20841	53	9.17061	44	9.24024	52	35	
36	.14382	46	.20894	54	.17106	45	.24076	52	36	
37	.14428	46	.20948	53	.17151	44	.24128	52	37	
38	.14474	46	.21002	53	.17195	44	.24181	52	38	
39	.14520	46	.21055	53	.17240	44	.24233	52	39	
40	9.14566	46	9.21109	53	9.17284	44	9.24285	52	40	
41	.14612	46	.21162	53	.17329	44	.24338	52	41	
42	.14658	46	.21216	53	.17373	44	.24390	52	42	
43	.14704	46	.21269	53	.17418	44	.24442	52	43	
44	.14750	46	.21323	53	.17462	44	.24495	52	44	
45	9.14796	46	9.21376	53	9.17507	44	9.24547	52	45	
46	.14842	46	.21430	53	.17551	44	.24599	52	46	
47	.14888	46	.21483	53	.17596	44	.24651	52	47	
48	.14934	45	.21537	53	.17640	44	.24704	52	48	
49	.14980	45	.21590	53	.17684	44	.24756	52	49	
50	9.15026	45	9.21643	53	9.17728	44	9.24808	52	50	
51	.15071	46	.21697	53	.17773	44	.24860	52	51	
52	.15117	46	.21750	53	.17817	44	.24912	52	52	
53	.15163	45	.21803	53	.17861	44	.24964	52	53	
54	.15209	45	.21857	53	.17906	44	.25016	52	54	
55	9.15254	45	9.21910	53	9.17950	44	9.25068	52	55	
56	.15300	45	.21963	53	.17994	44	.25120	52	56	
57	.15346	45	.22016	53	.18038	44	.25172	52	57	
58	.15391	45	.22070	53	.18082	44	.25224	52	58	
59	.15437	46	.22123	53	.18126	44	.25276	52	59	
60	9.15483	46	9.22176	53	9.18170	44	9.25328	52	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

	53	54	53
6	5.4	5.4	5.3
7	6.3	6.3	6.2
8	7.2	7.2	7.1
9	8.2	8.1	8.0
10	9.1	9.0	8.9
20	18.1	18.0	17.8
30	27.2	27.0	26.7
40	36.3	36.0	35.6
50	45.4	45.0	44.6

	53	52	52
6	5.3	5.2	5.2
7	6.2	6.1	6.0
8	7.0	7.0	6.9
9	7.9	7.9	7.8
10	8.8	8.7	8.6
20	17.6	17.5	17.3
30	26.5	26.2	26.0
40	35.3	35.0	34.6
50	44.1	43.7	43.3

	47	47	46
6	4.7	4.7	4.6
7	5.5	5.5	5.4
8	6.3	6.2	6.2
9	7.1	7.0	7.0
10	7.9	7.8	7.7
20	15.8	15.6	15.5
30	23.7	23.5	23.2
40	31.6	31.3	31.0
50	39.6	39.1	38.7

	46	45	45
6	4.6	4.5	4.5
7	5.3	5.3	5.2
8	6.1	6.0	6.0
9	6.9	6.8	6.7
10	7.6	7.6	7.5
20	15.3	15.1	15.0
30	23.0	22.7	22.5
40	30.6	30.3	30.0
50	38.3	37.9	37.5

	44	44	
6	4.4	4.4	
7	5.2	5.1	
8	5.9	5.8	
9	6.7	6.6	
10	7.4	7.3	
20	14.8	14.6	
30	22.2	22.0	
40	29.6	29.3	
50	37.1	36.6	

P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

32°

33°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.18170	44	9.25328	52	9.20771	42	9.28412	51	0	
1	.18214	44	.25380	52	.20814	42	.28463	51	1	
2	.18258	44	.25432	52	.20856	42	.28514	50	2	
3	.18302	44	.25484	51	.20899	43	.28564	51	3	
4	.18346	44	.25536	52	.20942	42	.28615	51	4	
5	9.18390	44	9.25588	52	9.20984	42	9.28665	50	5	
6	.18434	44	.25640	52	.21027	42	.28717	50	6	
7	.18478	44	.25692	51	.21069	42	.28768	51	7	
8	.18522	43	.25743	52	.21112	42	.28818	50	8	
9	.18566	44	.25795	51	.21154	42	.28869	51	9	
10	9.18610	44	9.25847	52	9.21196	42	9.28920	50	10	
11	.18654	43	.25899	51	.21239	42	.28970	50	11	
12	.18697	44	.25950	52	.21281	42	.29021	51	12	
13	.18741	43	.26002	51	.21324	42	.29072	51	13	
14	.18785	43	.26054	51	.21366	42	.29122	50	14	
15	9.18829	44	9.26105	52	9.21408	42	9.29173	51	15	
16	.18872	43	.26157	51	.21451	42	.29223	50	16	
17	.18916	43	.26209	51	.21493	42	.29274	50	17	
18	.18959	44	.26260	51	.21535	42	.29324	51	18	
19	.19003	43	.26312	52	.21577	42	.29375	50	19	
20	9.19047	43	9.26364	51	9.21620	42	9.29426	50	20	
21	.19090	43	.26415	51	.21662	42	.29476	50	21	
22	.19134	43	.26467	51	.21704	42	.29527	50	22	
23	.19177	43	.26518	51	.21746	42	.29577	50	23	
24	.19221	43	.26570	51	.21788	42	.29627	50	24	
25	9.19264	43	9.26621	51	9.21830	42	9.29678	50	25	
26	.19308	43	.26673	51	.21872	42	.29728	50	26	
27	.19351	43	.26724	51	.21914	42	.29779	50	27	
28	.19395	43	.26776	51	.21956	42	.29829	50	28	
29	.19438	43	.26827	51	.21998	42	.29879	50	29	
30	9.19481	43	9.26878	51	9.22040	42	9.29930	50	30	
31	.19525	43	.26930	51	.22082	42	.29980	50	31	
32	.19568	43	.26981	51	.22124	42	.30030	50	32	
33	.19611	43	.27032	51	.22166	42	.30081	50	33	
34	.19654	43	.27084	51	.22208	42	.30131	50	34	
35	9.19698	43	9.27135	51	9.22250	42	9.30181	50	35	
36	.19741	43	.27186	51	.22292	42	.30231	50	36	
37	.19784	43	.27238	51	.22334	42	.30282	50	37	
38	.19827	43	.27289	51	.22376	42	.30332	50	38	
39	.19870	43	.27340	51	.22417	42	.30382	50	39	
40	9.19914	43	9.27391	51	9.22459	42	9.30432	50	40	
41	.19957	43	.27443	51	.22501	42	.30482	50	41	
42	.20000	43	.27494	51	.22543	42	.30533	50	42	
43	.20043	43	.27545	51	.22584	42	.30583	50	43	
44	.20086	43	.27596	51	.22626	42	.30633	50	44	
45	9.20129	43	9.27647	51	9.22668	42	9.30683	50	45	
46	.20172	43	.27698	51	.22709	42	.30733	50	46	
47	.20215	43	.27749	51	.22751	42	.30783	50	47	
48	.20258	43	.27800	51	.22792	42	.30833	50	48	
49	.20301	42	.27852	51	.22834	42	.30883	50	49	
50	9.20343	43	9.27903	51	9.22876	42	9.30933	50	50	
51	.20386	43	.27954	51	.22917	42	.30983	50	51	
52	.20429	43	.28005	51	.22959	42	.31033	50	52	
53	.20472	42	.28056	51	.23000	42	.31083	50	53	
54	.20515	43	.28107	50	.23042	42	.31133	50	54	
55	9.20558	42	9.28157	51	9.23083	42	9.31183	50	55	
56	.20600	43	.28208	51	.23124	42	.31233	50	56	
57	.20643	43	.28259	51	.23166	42	.31283	50	57	
58	.20686	42	.28310	51	.23207	42	.31333	50	58	
59	.20728	43	.28361	50	.23248	42	.31383	49	59	
60	9.20771	43	9.28412	50	9.23290	42	9.31432	49	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

P. P.

35°

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TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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37°

		Lg. Vers.	D	Log. Exs.	D			Lg. Vers.	D	Log. Exs.	D				
														P. P.	
0	9	.28099	39	9.37303	48	9	30398	37	9.40163	47	0	1	48	48	
1		.28138	38	.37352	48		.30436	38	.40210	47	1	6	4.8	4.8	
2		.28177	39	.37400	48		.30474	37	.40258	47	2	7	5.6	5.6	
3		.28816	39	.37448	48		.30511	37	.40305	47	3	8	6.4	6.4	
4		.28255	38	.37496	48		.30549	38	.40352	47	4	9	7.3	7.2	
5	9	.28293	39	9.37544	48	9	.30587	37	9.40399	47	5	10	8.1	8.0	
6		.28332	38	.37592	48		.30624	37	.40447	47	6	20	16.1	16.0	
7		.28371	39	.37640	47		.30662	38	.40494	47	7	30	24.2	24.0	
8		.28410	38	.37687	48		.30700	37	.40541	47	8	40	32.3	32.0	
9		.28448	39	.37735	48		.30737	37	.40588	47	9	50	40.4	40.0	
10	9	.28487	38	9.37783	48	9	.30775	37	9.40635	47	10		47	47	
11		.28526	38	.37831	48		.30812	37	.40682	47	11	6	4.7	4.7	
12		.28564	39	.37879	48		.30850	37	.40730	47	12	7	5.5	5.5	
13		.28603	38	.37927	47		.30887	37	.40777	47	13	8	6.3	6.2	
14		.28642	38	.37975	48		.30925	37	.40824	47	14	9	7.1	7.0	
15	9	.28680	38	9.38023	48	9	.30962	37	9.40871	47	15	10	7.9	7.8	
16		.28719	38	.38071	48		.31000	37	.40918	47	16	20	15.8	15.6	
17		.28757	38	.38119	47		.31037	37	.40965	47	17	30	23.7	23.5	
18		.28796	39	.38166	48		.31075	37	.41012	47	18	40	31.6	31.3	
19		.28835	38	.38214	47		.31112	37	.41059	47	19	50	39.6	39.1	
20	9	.28873	38	9.38262	48	9	.31150	37	9.41106	47	20		46	46	
21		.28912	38	.38310	47		.31187	37	.41153	47	21	6	4.6	4.6	
22		.28950	38	.38357	48		.31224	37	.41200	47	22	7	5.4	5.4	
23		.28988	38	.38405	47		.31262	37	.41247	47	23	8	6.2	6.2	
24		.29027	38	.38453	48		.31299	37	.41294	47	24	9	7.0	7.0	
25	9	.29065	38	9.38501	47	9	.31336	37	9.41341	47	25	10	7.7	7.7	
26		.29104	38	.38548	48		.31374	37	.41388	47	26	20	15.5	15.5	
27		.29142	38	.38596	47		.31411	37	.41435	47	27	30	23.2	23.2	
28		.29180	38	.38644	48		.31448	37	.41482	47	28	40	31.0	31.0	
29		.29219	38	.38692	47		.31485	37	.41529	47	29	50	38.7	38.7	
30	9	.29257	38	9.38739	47	9	.31523	37	9.41576	47	30		39	38	
31		.29295	38	.38787	47		.31560	37	.41623	47	31	6	3.9	3.8	
32		.29334	38	.38834	48		.31597	37	.41670	47	32	7	4.5	4.5	
33		.29372	38	.38882	47		.31634	37	.41717	46	33	8	5.2	5.1	
34		.29410	38	.38930	47		.31671	37	.41763	47	34	9	5.8	5.8	
35	9	.29448	38	9.38977	47	9	.31708	37	9.41810	47	35	10	6.5	6.4	
36		.29487	38	.39025	47		.31746	37	.41857	47	36	20	13.0	12.8	
37		.29525	38	.39072	48		.31783	37	.41904	46	37	30	10.5	19.2	
38		.29563	38	.39120	47		.31820	37	.41951	47	38	40	26.0	25.6	
39		.29601	38	.39168	47		.31857	37	.41998	46	39	50	32.5	32.1	
40	9	.29639	38	9.39215	48	9	.31894	37	9.42044	47	40		38	37	
41		.29677	38	.39263	47		.31931	37	.42091	46	41	6	3.8	3.7	
42		.29715	38	.39310	47		.31968	37	.42138	47	42	7	4.4	4.4	
43		.29754	38	.39358	47		.32005	37	.42185	46	43	8	5.0	5.0	
44		.29792	38	.39405	47		.32042	37	.42231	47	44	9	5.7	5.6	
45	9	.29830	38	9.39453	47	9	.32079	37	9.42278	46	45	10	6.3	6.2	
46		.29868	38	.39500	47		.32116	37	.42325	47	46	20	12.6	12.5	
47		.29906	38	.39548	47		.32153	37	.42372	46	47	30	19.0	18.7	
48		.29944	38	.39595	47		.32190	37	.42418	47	48	40	25.3	25.0	
49		.29982	38	.39642	47		.32227	37	.42465	46	49	50	31.6	31.2	
50	9	.30020	37	9.39690	47	9	.32263	36	9.42512	46	50		37	36	
51		.30057	38	.39737	47		.32300	37	.42558	46	51	6	3.7	3.6	
52		.30095	38	.39785	47		.32337	37	.42605	47	52	7	4.3	4.2	
53		.30133	38	.39832	47		.32374	36	.42652	46	53	8	4.9	4.8	
54		.30171	38	.39879	47		.32411	37	.42698	46	54	9	5.5	5.5	
55	9	.30209	38	9.39927	47	9	.32447	36	9.42745	46	55	10	6.1	6.1	
56		.30247	37	.39974	47		.32484	36	.42792	46	56	20	12.3	12.1	
57		.30285	37	.40021	47		.32521	37	.42838	46	57	30	18.5	18.2	
58		.30322	38	.40069	47		.32558	36	.42885	46	58	40	24.6	24.3	
59		.30360	38	.40116	47		.32594	37	.42931	46	59	50	30.8	30.4	
60	9	.30398	38	9.40163	47	9	.32631	37	9.42978	46	60				
		Lg. Vers.	D	Log. Exs.	D			Lg. Vers.	D	Log. Exs.	D			P. P.	

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

38°										39°									

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.36913	34	9.48488	45	9.38968	34	9.51190	45	0	
1	.36948	34	.48533	45	.39002	33	.51235	44	1	
2	.36982	35	.48578	45	.39035	34	.51270	45	2	
3	.37017	34	.48624	45	.39069	33	.51324	44	3	
4	.37052	34	.48669	45	.39103	33	.51369	44	4	
5	9.37086	34	9.48714	45	9.39137	34	9.51414	45	5	
6	.37121	35	.48759	45	.39170	33	.51458	44	6	
7	.37156	34	.48805	45	.39204	33	.51503	45	7	
8	.37190	34	.48850	45	.39238	34	.51548	44	8	
9	.37225	34	.48895	45	.39271	33	.51592	44	9	
10	9.37259	34	9.48940	45	9.39305	33	9.51637	45	10	
11	.37294	34	.48986	45	.39339	34	.51682	44	11	
12	.37328	34	.49031	45	.39372	33	.51726	45	12	
13	.37363	34	.49076	45	.39406	33	.51771	44	13	
14	.37397	34	.49121	45	.39439	33	.51816	44	14	
15	9.37432	34	9.49166	45	9.39473	33	9.51860	44	15	
16	.37466	34	.49211	45	.39507	34	.51905	45	16	
17	.37501	34	.49257	45	.39540	33	.51950	44	17	
18	.37535	34	.49302	45	.39574	33	.51994	44	18	
19	.37570	34	.49347	45	.39607	33	.52039	44	19	
20	9.37604	34	9.49392	45	9.39641	33	9.52084	45	20	
21	.37639	34	.49437	45	.39674	33	.52128	44	21	
22	.37673	34	.49482	45	.39708	33	.52173	44	22	
23	.37707	34	.49527	45	.39741	33	.52217	44	23	
24	.37742	34	.49572	45	.39774	33	.52262	44	24	
25	9.37776	34	9.49618	45	9.39808	33	9.52306	45	25	
26	.37810	34	.49663	45	.39841	33	.52351	44	26	
27	.37845	34	.49708	45	.39875	33	.52396	44	27	
28	.37879	34	.49753	45	.39908	33	.52440	44	28	
29	.37913	34	.49798	45	.39941	33	.52485	44	29	
30	9.37947	34	9.49843	45	9.39975	33	9.52529	44	30	
31	.37982	34	.49888	45	.40008	33	.52574	44	31	
32	.38016	34	.49933	45	.40041	33	.52618	44	32	
33	.38050	34	.49978	45	.40075	33	.52663	44	33	
34	.38084	34	.50023	45	.40108	33	.52707	44	34	
35	9.38118	34	9.50068	45	9.40141	33	9.52752	44	35	
36	.38153	34	.50113	45	.40175	33	.52796	44	36	
37	.38187	34	.50158	45	.40208	33	.52841	44	37	
38	.38221	34	.50203	45	.40241	33	.52885	44	38	
39	.38255	34	.50248	45	.40274	33	.52930	44	39	
40	9.38289	34	9.50293	45	9.40307	33	9.52974	44	40	
41	.38323	34	.50338	45	.40341	33	.53018	44	41	
42	.38357	34	.50383	45	.40374	33	.53063	44	42	
43	.38391	34	.50427	45	.40407	33	.53107	44	43	
44	.38425	34	.50472	45	.40440	33	.53152	44	44	
45	9.38459	34	9.50517	45	9.40473	33	9.53196	44	45	
46	.38493	34	.50562	45	.40506	33	.53240	44	46	
47	.38527	34	.50607	44	.40540	33	.53285	44	47	
48	.38561	34	.50652	45	.40573	33	.53329	44	48	
49	.38595	34	.50697	45	.40606	33	.53374	44	49	
50	9.38629	34	9.50742	45	9.40639	33	9.53418	44	50	
51	.38663	34	.50787	44	.40672	33	.53462	44	51	
52	.38697	33	.50831	45	.40705	33	.53507	44	52	
53	.38731	34	.50876	45	.40738	33	.53551	44	53	
54	.38765	34	.50921	44	.40771	33	.53595	44	54	
55	9.38799	34	9.50966	44	9.40804	33	9.53640	44	55	
56	.38833	33	.51011	44	.40837	33	.53684	44	56	
57	.38866	34	.51055	45	.40870	33	.53728	44	57	
58	.38900	33	.51100	45	.40903	33	.53773	44	58	
59	.38934	34	.51145	44	.40936	33	.53817	44	59	
60	9.38968	34	9.51190	44	9.40969	33	9.53861	44	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

45	45
6	4.5
7	5.3
8	6.0
9	6.8
10	7.6
20	15.1
30	22.7
40	30.3
50	37.9

44	44
6	4.4
7	5.2
8	5.9
9	6.7
10	7.4
20	14.8
30	22.2
40	29.6
50	37.1

35	34
6	3.5
7	4.1
8	4.6
9	5.2
10	5.8
20	11.6
30	17.5
40	23.3
50	29.1

34	33
6	3.4
7	3.9
8	4.5
9	5.1
10	5.6
20	11.3
30	17.0
40	22.6
50	28.3

33	
6	3.3
7	3.8
8	4.4
9	4.9
10	5.5
20	11.0
30	16.5
40	22.0
50	27.5

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

42°

43°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.40969	32	9.53861	44	9.42918	32	9.56505	43	0	
1	.41001	33	.53906	44	.42950	32	.56549	44	1	
2	.41034	33	.53950	44	.42982	32	.56593	44	2	
3	.41067	33	.53994	44	.43014	32	.56637	44	3	
4	.41100	32	.54038	44	.43046	32	.56680	43	4	
5	9.41133	32	9.54083	44	9.43078	32	9.56724	44	5	
6	.41166	33	.54127	44	.43110	32	.56768	44	6	
7	.41199	33	.54171	44	.43142	31	.56812	43	7	
8	.41231	33	.54215	44	.43174	32	.56856	44	8	
9	.41264	32	.54259	44	.43206	32	.56899	43	9	
10	9.41297	32	9.54304	44	9.43238	32	9.56943	44	10	
11	.41330	33	.54348	44	.43270	32	.56987	44	11	
12	.41362	33	.54392	44	.43302	32	.57031	43	12	
13	.41395	32	.54436	44	.43334	32	.57075	44	13	
14	.41428	32	.54480	44	.43365	31	.57118	43	14	
15	9.41461	33	9.54525	44	9.43397	32	9.57162	44	15	
16	.41493	33	.54569	44	.43429	32	.57206	44	16	
17	.41526	32	.54613	44	.43461	31	.57250	43	17	
18	.41559	32	.54657	44	.43493	32	.57293	44	18	
19	.41591	32	.54701	44	.43525	32	.57337	43	19	
20	9.41624	32	9.54745	44	9.43557	31	9.57381	43	20	
21	.41657	32	.54790	44	.43588	31	.57424	43	21	
22	.41689	32	.54834	44	.43620	32	.57468	44	22	
23	.41722	32	.54878	44	.43652	31	.57512	43	23	
24	.41754	32	.54922	44	.43684	32	.57556	44	24	
25	9.41787	32	9.54966	44	9.43715	31	9.57599	43	25	
26	.41819	32	.55010	44	.43747	32	.57643	43	26	
27	.41852	32	.55054	44	.43779	31	.57687	43	27	
28	.41885	32	.55098	44	.43810	32	.57730	43	28	
29	.41917	32	.55142	44	.43842	31	.57774	43	29	
30	9.41950	32	9.55186	44	9.43874	31	9.57818	44	30	
31	.41982	32	.55230	44	.43906	32	.57861	43	31	
32	.42014	32	.55275	44	.43937	31	.57905	43	32	
33	.42047	32	.55319	44	.43969	31	.57949	44	33	
34	.42079	32	.55363	44	.44000	31	.57992	43	34	
35	9.42112	32	9.55407	44	9.44032	32	9.58036	43	35	
36	.42144	32	.55451	44	.44064	31	.58079	43	36	
37	.42177	32	.55495	44	.44095	31	.58123	44	37	
38	.42209	32	.55539	44	.44127	31	.58167	43	38	
39	.42241	32	.55583	44	.44158	31	.58210	43	39	
40	9.42274	32	9.55627	44	9.44190	31	9.58254	43	40	
41	.42306	32	.55671	44	.44221	31	.58297	44	41	
42	.42338	32	.55715	44	.44253	31	.58341	43	42	
43	.42371	32	.55759	44	.44284	31	.58385	43	43	
44	.42403	32	.55803	44	.44316	31	.58428	43	44	
45	9.42435	32	9.55847	44	9.44347	31	9.58472	43	45	
46	.42467	32	.55890	44	.44379	31	.58515	43	46	
47	.42500	32	.55934	44	.44410	31	.58559	43	47	
48	.42532	32	.55978	44	.44442	31	.58602	43	48	
49	.42564	32	.56022	44	.44473	31	.58646	43	49	
50	9.42596	32	9.56066	44	9.44504	31	9.58689	43	50	
51	.42629	32	.56110	44	.44536	31	.58733	43	51	
52	.42661	32	.56154	43	.44567	31	.58776	43	52	
53	.42693	32	.56198	44	.44599	31	.58820	44	53	
54	.42725	32	.56242	44	.44630	31	.58864	43	54	
55	9.42757	32	9.56286	44	9.44661	31	9.58907	43	55	
56	.42789	32	.56330	44	.44693	31	.58951	43	56	
57	.42822	32	.56374	43	.44724	31	.58994	43	57	
58	.42854	32	.56417	44	.44755	31	.59037	43	58	
59	.42886	32	.56461	43	.44787	31	.59081	43	59	
60	9.42918	32	9.56505	43	9.44818	31	9.59124	43	60	
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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								P. P.	
Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		
0	9.44818	31	9.59124	43	9.46671	30	9.61722	43	0
1	.44849	31	.59168	43	.46701	30	.61765	43	1
2	.44880	31	.59211	43	.46732	30	.61808	43	2
3	.44912	31	.59255	43	.46762	30	.61852	43	3
4	.44943	31	.59298	43	.46793	30	.61895	43	4
5	9.44974	31	9.59342	43	9.46823	30	9.61938	43	5
6	.45005	31	.59385	43	.46853	30	.61981	43	6
7	.45036	31	.59429	43	.46884	30	.62024	43	7
8	.45068	31	.59472	43	.46914	30	.62067	43	8
9	.45099	31	.59515	43	.46945	30	.62110	43	9
10	9.45130	31	9.59559	43	9.46975	30	9.62153	43	10
11	.45161	31	.59602	43	.47005	30	.62196	43	11
12	.45192	31	.59646	43	.47036	30	.62239	43	12
13	.45223	31	.59689	43	.47066	30	.62282	43	13
14	.45254	31	.59732	43	.47096	30	.62326	43	14
15	9.45285	31	9.59776	43	9.47127	30	9.62369	43	15
16	.45316	31	.59819	43	.47157	30	.62412	43	16
17	.45348	31	.59863	43	.47187	30	.62455	43	17
18	.45379	31	.59906	43	.47218	30	.62498	43	18
19	.45410	31	.59949	43	.47248	30	.62541	43	19
20	9.45441	31	9.59993	43	9.47278	30	9.62584	43	20
21	.45472	31	.60036	43	.47308	30	.62627	43	21
22	.45503	31	.60079	43	.47339	30	.62670	43	22
23	.45534	31	.60123	43	.47369	30	.62713	43	23
24	.45565	31	.60166	43	.47399	30	.62756	43	24
25	9.45595	31	9.60209	43	9.47429	30	9.62799	43	25
26	.45626	31	.60253	43	.47459	30	.62842	43	26
27	.45657	31	.60296	43	.47490	30	.62885	43	27
28	.45688	31	.60339	43	.47520	30	.62928	43	28
29	.45719	31	.60383	43	.47550	30	.62971	43	29
30	9.45750	31	9.60426	43	9.47580	30	9.63014	43	30
31	.45781	31	.60469	43	.47610	30	.63057	43	31
32	.45812	31	.60512	43	.47640	30	.63100	43	32
33	.45843	31	.60556	43	.47670	30	.63143	43	33
34	.45873	31	.60599	43	.47700	30	.63186	43	34
35	9.45904	31	9.60642	43	9.47731	30	9.63229	43	35
36	.45935	31	.60685	43	.47761	30	.63272	43	36
37	.45966	31	.60729	43	.47791	30	.63315	43	37
38	.45997	31	.60772	43	.47821	30	.63358	43	38
39	.46027	31	.60815	43	.47851	30	.63401	43	39
40	9.46058	31	9.60858	43	9.47881	30	9.63443	43	40
41	.46089	31	.60902	43	.47911	30	.63486	43	41
42	.46120	31	.60945	43	.47941	30	.63529	43	42
43	.46150	31	.60988	43	.47971	30	.63572	43	43
44	.46181	31	.61031	43	.48001	30	.63615	43	44
45	9.46212	31	9.61075	43	9.48031	30	9.63658	43	45
46	.46242	31	.61118	43	.48061	29	.63701	43	46
47	.46273	31	.61161	43	.48090	30	.63744	43	47
48	.46304	31	.61204	43	.48120	30	.63787	43	48
49	.46334	31	.61247	43	.48150	30	.63830	43	49
50	9.46365	31	9.61291	43	9.48180	30	9.63873	43	50
51	.46396	31	.61334	43	.48210	29	.63915	43	51
52	.46426	31	.61377	43	.48240	30	.63958	43	52
53	.46457	31	.61420	43	.48270	30	.64001	43	53
54	.46487	31	.61463	43	.48300	30	.64044	43	54
55	9.46518	31	9.61506	43	9.48329	29	9.64087	43	55
56	.46549	31	.61550	43	.48359	30	.64130	43	56
57	.46579	31	.61593	43	.48389	30	.64173	43	57
58	.46610	31	.61636	43	.48419	30	.64216	43	58
59	.46640	31	.61679	43	.48449	30	.64258	43	59
60	9.46671	31	9.61722	43	9.48478	29	9.64301	43	60
Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		
								P. P.	

43			43		
6	4.3	4.3			
7	5.1	5.0			
8	5.8	5.7			
9	6.5	6.4			
10	7.2	7.1			
20	14.5	14.3			
30	21.7	21.5			
40	29.0	28.6			
50	36.2	35.8			
42					
6	4.2	4.2			
7	4.9	4.9			
8	5.6	5.6			
9	6.4	6.4			
10	7.1	7.1			
20	14.1	14.1			
30	21.2	21.2			
40	28.3	28.3			
50	35.4	35.4			
31			31		
6	3.1	3.1			
7	3.7	3.6			
8	4.2	4.1			
9	4.7	4.6			
10	5.2	5.1			
20	10.5	10.3			
30	15.7	15.5			
40	21.0	20.6			
50	26.2	25.8			
30			30		
6	3.0	3.0			
7	3.5	3.5			
8	4.0	4.0			
9	4.6	4.5			
10	5.1	5.0			
20	10.1	10.0			
30	15.2	15.0			
40	20.3	20.0			
50	25.4	25.0			
29					
6	2.9	2.9			
7	3.4	3.4			
8	3.9	3.9			
9	4.4	4.4			
10	4.9	4.9			
20	9.8	9.8			
30	14.7	14.7			
40	19.6	19.6			
50	24.6	24.6			

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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47°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.48478	30	9.64301	43	9.50243	29	9.66864	42	0	
1	.48508	29	.64344	42	.50272	29	.66907	43	1	
2	.48538	30	.64387	43	.50301	29	.66950	42	2	
3	.48568	29	.64430	43	.50330	29	.66992	42	3	
4	.48597	29	.64473	43	.50359	29	.67035	42	4	43 42
5	9.48627	30	9.64515	42	9.50388	29	9.67077	42	5	6 4.3 4.2
6	.48657	29	.64558	43	.50417	29	.67120	42	6	7 5.0 4.9
7	.48686	30	.64601	43	.50446	29	.67162	42	7	8 5.7 5.6
8	.48716	29	.64644	42	.50475	29	.67205	43	8	9 6.4 6.4
9	.48746	29	.64687	43	.50504	29	.67248	42	9	10 7.1 7.1
10	9.48775	30	9.64729	42	9.50533	29	9.67290	42	10	20 14.3 14.1
11	.48805	29	.64772	43	.50562	29	.67333	42	11	30 21.5 21.2
12	.48835	30	.64815	43	.50591	29	.67375	42	12	40 28.6 28.3
13	.48866	29	.64858	43	.50619	29	.67418	42	13	50 35.8 35.4
14	.48894	29	.64901	42	.50648	29	.67460	42	14	
15	9.48923	30	9.64943	43	9.50677	29	9.67503	42	15	42
16	.48953	29	.64986	42	.50706	29	.67546	42	16	6 4.2
17	.48983	30	.65029	43	.50735	29	.67588	42	17	7 4.9
18	.49012	29	.65072	42	.50764	29	.67631	42	18	8 5.6
19	.49042	29	.65114	42	.50793	29	.67673	42	19	9 6.3
20	9.49071	30	9.65157	43	9.50821	29	9.67716	42	20	10 7.0
21	.49101	29	.65200	43	.50850	29	.67758	42	21	20 14.0
22	.49130	30	.65243	42	.50879	29	.67801	42	22	30 21.0
23	.49160	29	.65285	42	.50908	29	.67843	42	23	40 28.0
24	.49189	29	.65328	42	.50937	29	.67886	42	24	50 35.0
25	9.49219	30	9.65371	43	9.50965	29	9.67928	42	25	
26	.49248	29	.65414	42	.50994	29	.67971	42	26	30 29
27	.49278	30	.65456	42	.51023	29	.68013	42	27	6 3.0 2.9
28	.49307	29	.65499	42	.51052	29	.68056	42	28	7 3.5 3.4
29	.49336	29	.65542	43	.51080	29	.68098	42	29	8 4.0 3.9
30	9.49365	30	9.65585	42	9.51109	29	9.68141	42	30	9 4.5 4.4
31	.49395	29	.65627	42	.51138	29	.68183	42	31	10 5.0 4.9
32	.49425	30	.65670	42	.51167	29	.68226	42	32	20 10.0 9.8
33	.49454	29	.65713	42	.51195	29	.68268	42	33	30 15.0 14.7
34	.49483	29	.65755	42	.51224	29	.68311	42	34	40 20.0 19.6
35	9.49513	30	9.65798	42	9.51253	29	9.68353	42	35	50 25.0 24.6
36	.49542	29	.65841	43	.51281	29	.68396	42	36	
37	.49571	30	.65884	42	.51310	29	.68438	42	37	29 28
38	.49601	29	.65926	42	.51338	29	.68481	42	38	6 2.9 2.8
39	.49630	29	.65969	43	.51367	29	.68523	42	39	7 3.4 3.3
40	9.49659	30	9.66012	42	9.51396	29	9.68566	42	40	8 3.8 3.8
41	.49689	29	.66054	42	.51424	29	.68608	42	41	9 4.3 4.3
42	.49718	30	.66097	42	.51453	29	.68651	42	42	10 4.8 4.7
43	.49747	29	.66140	42	.51481	29	.68693	42	43	20 9.6 9.5
44	.49776	29	.66182	42	.51510	29	.68735	42	44	30 14.5 14.2
45	9.49806	30	9.66225	43	9.51539	29	9.68778	42	45	40 19.3 19.0
46	.49835	29	.66268	42	.51567	29	.68820	42	46	50 24.1 23.7
47	.49864	30	.66310	42	.51596	29	.68863	42	47	
48	.49893	29	.66353	43	.51624	29	.68905	42	48	
49	.49922	29	.66396	42	.51653	29	.68948	42	49	
50	9.49952	30	9.66438	42	9.51681	29	9.68990	42	50	28
51	.49981	29	.66481	42	.51710	29	.69033	42	51	6 2.8
52	.50010	30	.66523	43	.51738	29	.69075	42	52	7 3.2 3.7
53	.50039	29	.66566	42	.51767	29	.69117	42	53	8 3.7
54	.50068	29	.66609	42	.51795	29	.69160	42	54	9 4.2
55	9.50097	30	9.66651	42	9.51823	29	9.69202	42	55	10 4.6
56	.50126	29	.66694	43	.51852	29	.69245	42	56	20 9.3
57	.50155	30	.66737	42	.51880	29	.69287	42	57	30 14.0
58	.50185	29	.66779	42	.51909	29	.69330	42	58	40 18.6
59	.50214	29	.66822	42	.51937	29	.69372	42	59	50 23.3
60	9.50243	30	9.66864	42	9.51965	29	9.69414	42	60	
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

48°

49°

'	48°		49°		'	P. P.	
	Lg. Vers.	D	Log. Exs.	D			
0	9.51965	28	9.69414	42	0		
1	.51994	28	.69457	42	1		
2	.52022	28	.69499	42	2		
3	.52050	28	.69542	42	3		
4	.52079	28	.69584	42	4		
5	9.52107	28	9.69626	42	5		
6	.52135	28	.69669	42	6		
7	.52164	28	.69711	42	7		
8	.52192	28	.69753	42	8		
9	.52220	28	.69796	42	9		
10	9.52249	28	9.69838	42	10	42	42
11	.52277	28	.69881	42	11	6 4.2	4.2
12	.52305	28	.69923	42	12	7 4.9	4.9
13	.52333	28	.69965	42	13	8 5.6	5.6
14	.52362	28	.70008	42	14	9 6.4	6.3
15	9.52390	28	9.70050	42	15	10 7.1	7.0
16	.52418	28	.70092	42	16	20 14.1	14.0
17	.52446	28	.70135	42	17	30 21.2	21.0
18	.52474	28	.70177	42	18	40 28.3	28.0
19	.52503	28	.70220	42	19	50 35.4	35.0
20	9.52531	28	9.70262	42	20		
21	.52559	28	.70304	42	21		
22	.52587	28	.70347	42	22		
23	.52615	28	.70389	42	23		
24	.52643	28	.70431	42	24		
25	9.52671	28	9.70474	42	25		
26	.52699	28	.70516	42	26	28	28
27	.52727	28	.70558	42	27	6 2.8	2.8
28	.52756	28	.70601	42	28	7 3.3	3.2
29	.52784	28	.70643	42	29	8 3.8	3.7
30	9.52812	28	9.70685	42	30	9 4.3	4.2
31	.52840	28	.70728	42	31	10 4.7	4.6
32	.52868	28	.70770	42	32	20 9.5	9.3
33	.52896	28	.70812	42	33	30 14.2	14.0
34	.52924	28	.70854	42	34	40 19.0	18.6
35	9.52952	28	9.70897	42	35	50 23.7	23.3
36	.52980	28	.70939	42	36		
37	.53008	28	.70981	42	37		
38	.53036	28	.71024	42	38		
39	.53064	28	.71066	42	39		
40	9.53092	28	9.71108	42	40		
41	.53120	27	.71151	42	41		
42	.53147	27	.71193	42	42		
43	.53175	28	.71235	42	43	27	27
44	.53203	28	.71278	42	44	6 2.7	2.7
45	9.53231	27	9.71320	42	45	7 3.2	3.1
46	.53259	28	.71362	42	46	8 3.6	3.6
47	.53287	28	.71404	42	47	9 4.1	4.0
48	.53315	28	.71447	42	48	10 4.6	4.5
49	.53343	28	.71489	42	49	20 9.1	9.0
50	9.53370	27	9.71531	42	50	30 13.7	13.5
51	.53398	28	.71573	42	51	40 18.3	18.0
52	.53426	27	.71616	42	52	50 22.9	22.5
53	.53454	28	.71658	42	53		
54	.53482	28	.71700	42	54		
55	9.53509	27	9.71743	42	55		
56	.53537	28	.71785	42	56		
57	.53565	27	.71827	42	57		
58	.53593	28	.71869	42	58		
59	.53620	27	.71912	42	59		
60	9.53648	28	9.71954	42	60		
'	48°		49°		'	P. P.	
	Lg. Vers.	D	Log. Exs.	D			

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

50°

51°

		Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D						
0	9.55292	27	9.74486	42	9.56900	26	9.77012	42	0						
1	.55319	27	.74528	42	.56926	26	.77055	42	1						
2	.55347	27	.74570	42	.56953	26	.77097	42	2						
3	.55374	27	.74612	42	.56979	26	.77139	42	3						
4	.55401	27	.74654	42	.57005	26	.77181	42	4						
5	9.55428	27	9.74696	42	9.57032	26	9.77223	42	5						
6	.55455	27	.74739	42	.57058	26	.77265	42	6						
7	.55482	27	.74781	42	.57085	26	.77307	42	7						
8	.55509	27	.74823	42	.57111	26	.77349	42	8						
9	.55536	27	.74865	42	.57138	26	.77391	42	9						
10	9.55563	27	9.74907	42	9.57164	26	9.77433	42	10						
11	.55590	27	.74949	42	.57190	26	.77475	42	11						
12	.55617	27	.74991	42	.57217	26	.77517	42	12						
13	.55644	27	.75033	42	.57243	26	.77560	42	13						
14	.55671	27	.75076	42	.57269	26	.77602	42	14						
15	9.55698	27	9.75118	42	9.57296	26	9.77644	42	15						
16	.55725	26	.75160	42	.57322	26	.77686	42	16						
17	.55751	26	.75202	42	.57348	26	.77728	42	17						
18	.55778	27	.75244	42	.57375	26	.77770	42	18						
19	.55805	27	.75286	42	.57401	26	.77812	42	19						
20	9.55832	27	9.75328	42	9.57427	26	9.77854	42	20						
21	.55859	27	.75370	42	.57454	26	.77896	42	21						
22	.55886	26	.75413	42	.57480	26	.77938	42	22						
23	.55913	27	.75455	42	.57506	26	.77980	42	23						
24	.55940	26	.75497	42	.57532	26	.78022	42	24						
25	9.55966	27	9.75539	42	9.57559	26	9.78064	42	25						
26	.55993	27	.75581	42	.57585	26	.78107	42	26						
27	.56020	26	.75623	42	.57611	26	.78149	42	27						
28	.56047	27	.75665	42	.57637	26	.78191	42	28						
29	.56074	27	.75707	42	.57664	26	.78233	42	29						
30	9.56101	26	9.75750	42	9.57690	26	9.78275	42	30						
31	.56127	27	.75792	42	.57716	26	.78317	42	31						
32	.56154	26	.75834	42	.57742	26	.78359	42	32						
33	.56181	27	.75876	42	.57768	26	.78401	42	33						
34	.56208	26	.75918	42	.57794	26	.78443	42	34						
35	9.56234	27	9.75960	42	9.57821	26	9.78485	42	35						
36	.56261	26	.76002	42	.57847	26	.78527	42	36						
37	.56288	27	.76044	42	.57873	26	.78569	42	37						
38	.56315	26	.76086	42	.57899	26	.78611	42	38						
39	.56341	26	.76128	42	.57925	26	.78653	42	39						
40	9.56368	27	9.76171	42	9.57951	26	9.78696	42	40						
41	.56395	26	.76213	42	.57977	26	.78738	42	41						
42	.56421	26	.76255	42	.58003	26	.78780	42	42						
43	.56448	27	.76297	42	.58029	26	.78822	42	43						
44	.56475	26	.76339	42	.58055	26	.78864	42	44						
45	9.56501	26	9.76381	42	9.58082	26	9.78906	42	45						
46	.56528	26	.76423	42	.58108	26	.78948	42	46						
47	.56554	27	.76465	42	.58134	26	.78990	42	47						
48	.56581	26	.76507	42	.58160	26	.79032	42	48						
49	.56608	26	.76549	42	.58186	26	.79074	42	49						
50	9.56634	26	9.76592	42	9.58212	26	9.79116	42	50						
51	.56661	26	.76634	42	.58238	26	.79158	42	51						
52	.56687	26	.76676	42	.58264	26	.79200	42	52						
53	.56714	27	.76718	42	.58290	26	.79242	42	53						
54	.56741	26	.76760	42	.58316	26	.79285	42	54						
55	9.56767	26	9.76802	42	9.58342	26	9.79327	42	55						
56	.56794	26	.76844	42	.58367	26	.79369	42	56						
57	.56820	26	.76886	42	.58393	26	.79411	42	57						
58	.56847	26	.76928	42	.58419	26	.79453	42	58						
59	.56873	26	.76970	42	.58445	26	.79495	42	59						
60	9.56900	26	9.77012	42	9.58471	26	9.79537	42	60						
		Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D						

P. P.

P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

52°

53°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.58471	26	9.79537	42	9.60008	25	9.82062	42	0.	
1	.58497	25	.79579	42	.60034	25	.82104	42	1	
2	.58523	25	.79621	42	.60059	25	.82146	42	2	
3	.58549	26	.79663	42	.60084	25	.82188	42	3	
4	.58575	26	.79705	42	.60110	25	.82230	42	4	
5	9.58601	26	9.79747	42	9.60135	25	9.82272	42	5	
6	.58626	25	.79789	42	.60160	25	.82315	42	6	
7	.58652	26	.79831	42	.60185	25	.82357	42	7	
8	.58678	26	.79874	42	.60211	25	.82399	42	8	
9	.58704	25	.79916	42	.60236	25	.82441	42	9	
10	9.58730	26	9.79958	42	9.60261	25	9.82483	42	10	
11	.58755	25	.80000	42	.60286	25	.82525	42	11	
12	.58781	26	.80042	42	.60312	25	.82567	42	12	
13	.58807	26	.80084	42	.60337	25	.82609	42	13	
14	.58833	25	.80126	42	.60362	25	.82651	42	14	
15	9.58859	26	9.80168	42	9.60387	25	9.82694	42	15	
16	.58884	26	.80210	42	.60412	25	.82736	42	16	
17	.58910	25	.80252	42	.60438	25	.82778	42	17	
18	.58936	26	.80294	42	.60463	25	.82820	42	18	
19	.58962	25	.80336	42	.60488	25	.82862	42	19	
20	9.58987	25	9.80378	42	9.60513	25	9.82904	42	20	
21	.59013	26	.80420	42	.60538	25	.82946	42	21	
22	.59039	25	.80463	42	.60563	25	.82988	42	22	
23	.59064	26	.80505	42	.60589	25	.83031	42	23	
24	.59090	25	.80547	42	.60614	25	.83073	42	24	
25	9.59116	25	9.80589	42	9.60639	25	9.83115	42	25	
26	.59141	25	.80631	42	.60664	25	.83157	42	26	
27	.59167	26	.80673	42	.60689	25	.83199	42	27	
28	.59193	25	.80715	42	.60714	25	.83241	42	28	
29	.59218	25	.80757	42	.60739	25	.83283	42	29	
30	9.59244	25	9.80799	42	9.60764	25	9.83325	42	30	
31	.59270	26	.80841	42	.60789	25	.83368	42	31	
32	.59295	25	.80883	42	.60814	25	.83410	42	32	
33	.59321	25	.80925	42	.60839	25	.83452	42	33	
34	.59346	25	.80968	42	.60864	25	.83494	42	34	
35	9.59372	25	9.81010	42	9.60889	25	9.83536	42	35	
36	.59397	25	.81052	42	.60914	25	.83578	42	36	
37	.59423	26	.81094	42	.60939	25	.83620	42	37	
38	.59449	25	.81136	42	.60964	25	.83663	42	38	
39	.59474	25	.81178	42	.60989	25	.83705	42	39	
40	9.59500	25	9.81220	42	9.61014	25	9.83747	42	40	
41	.59525	25	.81262	42	.61039	25	.83789	42	41	
42	.59551	25	.81304	42	.61064	25	.83831	42	42	
43	.59576	25	.81346	42	.61089	25	.83873	42	43	
44	.59602	25	.81388	42	.61114	25	.83916	42	44	
45	9.59627	25	9.81430	42	9.61139	25	9.83958	42	45	
46	.59653	25	.81473	42	.61164	25	.84000	42	46	
47	.59678	25	.81515	42	.61189	24	.84042	42	47	
48	.59704	25	.81557	42	.61214	25	.84084	42	48	
49	.59729	25	.81599	42	.61239	25	.84126	42	49	
50	9.59754	25	9.81641	42	9.61264	25	9.84168	42	50	
51	.59780	25	.81683	42	.61289	24	.84211	42	51	
52	.59805	25	.81725	42	.61313	25	.84253	42	52	
53	.59831	25	.81767	42	.61338	25	.84295	42	53	
54	.59856	25	.81809	42	.61363	25	.84337	42	54	
55	9.59881	25	9.81851	42	9.61388	25	9.84379	42	55	
56	.59907	25	.81894	42	.61413	25	.84422	42	56	
57	.59932	25	.81936	42	.61438	24	.84464	42	57	
58	.59958	25	.81978	42	.61462	25	.84506	42	58	
59	.59983	25	.82020	42	.61487	25	.84548	42	59	
60	9.60008	25	9.82062	42	9.61512	25	9.84590	42	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

42 42
6 4.2 4.2
7 4.9 4.9
8 5.6 5.6
9 6.4 6.3
10 7.1 7.0
20 14.1 14.0
30 21.2 21.0
40 28.3 28.0
50 35.4 35.0

26 25
6 2.6 2.5
7 3.0 3.0
8 3.4 3.4
9 3.9 3.1
10 4.3 4.2
20 8.6 8.5
30 13.0 12.7
40 17.3 17.0
50 21.6 21.2

25 24
6 2.5 2.4
7 2.9 2.8
8 3.3 3.2
9 3.7 3.7
10 4.1 4.1
20 8.3 8.1
30 12.5 12.2
40 16.6 16.3
50 20.8 20.4

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

54°

55°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.61512	24	9.84590	42	9.62984	24	9.87125	42	0	
1	.61537	25	.84632	42	.63008	24	.87167	42	1	
2	.61562	24	.84675	42	.63032	24	.87209	42	2	
3	.61586	24	.84717	42	.63057	24	.87252	42	3	
4	.61611	25	.84759	42	.63081	24	.87294	42	4	
5	9.61636	24	9.84801	42	9.63105	24	9.87336	42	5	
6	.61661	24	.84843	42	.63129	24	.87379	42	6	
7	.61685	24	.84886	42	.63154	24	.87421	42	7	
8	.61710	25	.84928	42	.63178	24	.87463	42	8	
9	.61735	24	.84970	42	.63202	24	.87506	42	9	
10	9.61760	25	9.85012	42	9.63226	24	9.87548	42	10	
11	.61784	24	.85054	42	.63250	24	.87590	42	11	
12	.61809	24	.85097	42	.63274	24	.87633	42	12	
13	.61834	24	.85139	42	.63299	24	.87675	42	13	
14	.61858	24	.85181	42	.63323	24	.87717	42	14	
15	9.61883	25	9.85223	42	9.63347	24	9.87760	42	15	
16	.61908	24	.85265	42	.63371	24	.87802	42	16	
17	.61932	24	.85308	42	.63395	24	.87844	42	17	
18	.61957	24	.85350	42	.63419	24	.87887	42	18	
19	.61982	25	.85392	42	.63443	24	.87929	42	19	
20	9.62006	24	9.85434	42	9.63468	24	9.87971	42	20	
21	.62031	24	.85476	42	.63492	24	.88014	42	21	
22	.62055	24	.85519	42	.63516	24	.88056	42	22	
23	.62080	24	.85561	42	.63540	24	.88099	42	23	
24	.62105	24	.85603	42	.63564	24	.88141	42	24	
25	9.62129	24	9.85645	42	9.63588	24	9.88183	42	25	
26	.62154	24	.85688	42	.63612	24	.88226	42	26	
27	.62178	24	.85730	42	.63636	24	.88268	42	27	
28	.62203	24	.85772	42	.63660	24	.88310	42	28	
29	.62227	24	.85814	42	.63684	24	.88353	42	29	
30	9.62252	24	9.85857	42	9.63708	24	9.88395	42	30	
31	.62276	24	.85899	42	.63732	24	.88438	42	31	
32	.62301	24	.85941	42	.63756	24	.88480	42	32	
33	.62325	24	.85983	42	.63780	24	.88522	42	33	
34	.62350	24	.86026	42	.63804	24	.88565	42	34	
35	9.62374	24	9.86068	42	9.63828	24	9.88607	42	35	
36	.62399	24	.86110	42	.63852	24	.88650	42	36	
37	.62423	24	.86152	42	.63876	24	.88692	42	37	
38	.62448	24	.86195	42	.63900	24	.88734	42	38	
39	.62472	24	.86237	42	.63924	24	.88777	42	39	
40	9.62497	24	9.86279	42	9.63948	24	9.88819	42	40	
41	.62521	24	.86321	42	.63972	24	.88862	42	41	
42	.62546	24	.86364	42	.63996	24	.88904	42	42	
43	.62570	24	.86406	42	.64019	24	.88947	42	43	
44	.62594	24	.86448	42	.64043	24	.88989	42	44	
45	9.62619	24	9.86490	42	9.64067	24	9.89031	42	45	
46	.62643	24	.86533	42	.64091	24	.89074	42	46	
47	.62668	24	.86575	42	.64115	24	.89116	42	47	
48	.62692	24	.86617	42	.64139	24	.89159	42	48	
49	.62716	24	.86659	42	.64163	24	.89201	42	49	
50	9.62741	24	9.86702	42	9.64187	24	9.89244	42	50	
51	.62765	24	.86744	42	.64210	24	.89286	42	51	
52	.62789	24	.86786	42	.64234	24	.89329	42	52	
53	.62814	24	.86829	42	.64258	24	.89371	42	53	
54	.62838	24	.86871	42	.64282	24	.89414	42	54	
55	9.62862	24	9.86913	42	9.64306	24	9.89456	42	55	
56	.62887	24	.86956	42	.64330	24	.89499	42	56	
57	.62911	24	.86998	42	.64353	24	.89541	42	57	
58	.62935	24	.87040	42	.64377	24	.89583	42	58	
59	.62960	24	.87082	42	.64401	24	.89626	42	59	
60	9.62984	24	9.87125	42	9.64425	24	9.89668	42	60	
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.

42 42
6 4.2 4.2
7 4.9 4.9
8 5.6 5.6
9 6.4 6.3
10 7.1 7.0
20 14.1 14.0
30 21.2 21.0
40 28.3 28.0
50 35.4 35.0

25 24
6 2.5 2.4
7 2.9 2.8
8 3.3 3.2
9 3.7 3.7
10 4.1 4.1
20 8.3 8.1
30 12.5 12.2
40 16.6 16.3
50 20.8 20.4

24 23
6 2.4 2.3
7 2.8 2.7
8 3.2 3.1
9 3.6 3.5
10 4.0 3.9
20 8.0 7.8
30 12.0 11.7
40 16.0 15.6
50 20.0 19.6

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

56°

57°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.	
0	9.64425	23	9.89668	42	9.65835	23	9.92224	43	0		
1	.64443	24	.89711	42	.65859	23	.92267	42	1		
2	.64472	24	.89753	42	.65882	23	.92310	42	2		
3	.64496	23	.89796	42	.65905	23	.92353	43	3		
4	.64520	23	.89838	42	.65928	23	.92395	43	4		
5	9.64543	23	9.89881	42	9.65952	23	9.92438	43	5		
6	.64567	23	.89923	42	.65975	23	.92481	42	6		
7	.64591	23	.89966	42	.65998	23	.92524	43	7		
8	.64614	23	.90008	42	.66021	23	.92566	43	8		
9	.64638	24	.90051	42	.66044	23	.92609	42	9		
10	9.64662	23	9.90094	42	9.66068	23	9.92652	43	10		
11	.64685	23	.90136	42	.66091	23	.92695	42	11		
12	.64709	23	.90179	42	.66114	23	.92737	42	12		
13	.64733	23	.90221	42	.66137	23	.92780	42	13		
14	.64756	23	.90264	42	.66160	23	.92823	42	14		
15	9.64780	24	9.90306	42	9.66183	23	9.92866	43	15		
16	.64804	23	.90349	42	.66207	23	.92909	42	16		
17	.64827	23	.90391	42	.66230	23	.92951	43	17		
18	.64851	24	.90434	42	.66253	23	.92994	42	18		
19	.64875	23	.90476	42	.66276	23	.93037	43	19		
20	9.64898	23	9.90519	42	9.66299	23	9.93080	43	20		
21	.64922	23	.90561	42	.66322	23	.93123	42	21		
22	.64945	23	.90604	42	.66345	23	.93165	43	22		
23	.64969	23	.90647	42	.66368	23	.93208	43	23		
24	.64992	23	.90689	42	.66391	23	.93251	42	24		
25	9.65016	24	9.90732	42	9.66415	23	9.93294	43	25		
26	.65040	23	.90774	42	.66438	23	.93337	43	26		
27	.65063	23	.90817	42	.66461	23	.93380	42	27		
28	.65087	23	.90860	42	.66484	23	.93422	43	28		
29	.65110	23	.90902	42	.66507	23	.93465	43	29		
30	9.65134	23	9.90945	42	9.66530	23	9.93508	42	30		
31	.65157	23	.90987	42	.66553	23	.93551	43	31		
32	.65181	23	.91030	42	.66576	23	.93594	43	32		
33	.65204	23	.91073	42	.66599	23	.93637	43	33		
34	.65228	23	.91115	42	.66622	23	.93680	42	34		
35	9.65251	23	9.91158	42	9.66645	23	9.93722	43	35		
36	.65275	23	.91200	42	.66668	23	.93765	43	36		
37	.65298	23	.91243	42	.66691	23	.93808	43	37		
38	.65321	23	.91286	42	.66714	23	.93851	42	38		
39	.65345	23	.91328	42	.66737	23	.93894	43	39		
40	9.65368	23	9.91371	42	9.66760	23	9.93937	43	40		
41	.65392	23	.91414	42	.66783	23	.93980	43	41		
42	.65415	23	.91456	42	.66805	23	.94023	43	42		
43	.65439	23	.91499	42	.66828	23	.94066	43	43		
44	.65462	23	.91541	42	.66851	23	.94109	42	44		
45	9.65485	23	9.91584	42	9.66874	23	9.94151	43	45		
46	.65509	23	.91627	42	.66897	23	.94194	43	46		
47	.65532	23	.91669	42	.66920	22	.94237	43	47		
48	.65556	23	.91712	42	.66943	23	.94280	43	48		
49	.65579	23	.91755	42	.66966	23	.94323	43	49		
50	9.65602	23	9.91797	42	9.66989	23	9.94366	43	50		
51	.65626	23	.91840	42	.67012	22	.94409	43	51		
52	.65649	23	.91883	42	.67034	23	.94452	43	52		
53	.65672	23	.91926	42	.67057	23	.94495	43	53		
54	.65696	23	.91968	42	.67080	22	.94538	43	54		
55	9.65719	23	9.92011	42	9.67103	23	9.94581	43	55		
56	.65742	23	.92054	42	.67126	23	.94624	43	56		
57	.65765	23	.92096	42	.67149	22	.94667	43	57		
58	.65789	23	.92139	42	.67171	23	.94710	43	58		
59	.65812	23	.92182	42	.67194	22	.94753	43	59		
60	9.65835	23	9.92224	42	9.67217	22	9.94796	43	60		
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.	

43 42
6 4.3 4.2
7 5.0 4.9
8 5.7 5.6
9 6.4 6.4
10 7.1 7.1
20 14.3 14.1
30 21.5 21.2
40 28.6 28.3
50 35.8 35.4

24 23
6 2.4 2.3
7 2.8 2.7
8 3.2 3.1
9 3.6 3.5
10 4.0 3.9
20 8.0 7.8
30 12.0 11.7
40 16.0 15.6
50 20.0 19.6

23 22
6 2.3 2.2
7 2.7 2.6
8 3.0 3.0
9 3.4 3.4
10 3.8 3.7
20 7.6 7.5
30 11.5 11.2
40 15.3 15.0
50 19.1 18.7

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

58°

59°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.67217	23	9.94796	43	9.68571	22	9.97387	43	0	
1	.67240	23	.94839	43	.68593	22	.97430	43	1	
2	.67263	22	.94882	43	.68615	22	.97473	43	2	
3	.67285	23	.94925	43	.68637	22	.97517	43	3	
4	.67308	22	.94968	43	.68660	22	.97560	43	4	
5	9.67331	23	9.95011	43	9.68682	22	9.97603	43	5	44 43
6	.67354	23	.95054	43	.68704	22	.97647	43	6	6 4.4 4.3
7	.67376	23	.95097	43	.68727	22	.97690	43	7	7 5.1 5.1
8	.67399	22	.95140	43	.68749	22	.97734	43	8	8 5.8 5.8
9	.67422	22	.95183	43	.68771	22	.97777	43	9	9 6.6 6.5
10	9.67445	23	9.95226	43	9.68793	22	9.97820	43	10	10 7.3 7.2
11	.67467	22	.95269	43	.68816	22	.97864	43	11	10 14.6 14.5
12	.67490	23	.95313	43	.68838	22	.97907	43	12	30 22.0 21.7
13	.67513	23	.95356	43	.68860	22	.97951	43	13	40 29.3 29.0
14	.67535	22	.95399	43	.68882	22	.97994	43	14	50 36.6 36.2
15	9.67558	23	9.95442	43	9.68905	22	9.98038	43	15	
16	.67581	22	.95485	43	.68927	22	.98081	43	16	
17	.67603	23	.95528	43	.68949	22	.98125	43	17	
18	.67626	23	.95571	43	.68971	22	.98168	43	18	
19	.67649	22	.95614	43	.68993	22	.98211	43	19	43
20	9.67671	23	9.95657	43	9.69016	22	9.98255	43	20	6 4.3
21	.67694	22	.95700	43	.69038	22	.98298	43	21	7 5.0
22	.67717	23	.95744	43	.69060	22	.98342	43	22	8 5.7
23	.67739	22	.95787	43	.69082	22	.98385	43	23	9 6.4
24	.67762	22	.95830	43	.69104	22	.98429	43	24	10 7.1
25	9.67784	23	9.95873	43	9.69126	22	9.98472	43	25	20 14.3
26	.67807	23	.95916	43	.69149	22	.98516	43	26	30 21.5
27	.67830	22	.95959	43	.69171	22	.98559	44	27	40 28.6
28	.67852	22	.96002	43	.69193	22	.98603	43	28	50 35.8
29	.67875	22	.96046	43	.69215	22	.98647	43	29	
30	9.67897	23	9.96089	43	9.69237	22	9.98690	43	30	
31	.67920	22	.96132	43	.69259	22	.98734	43	31	
32	.67942	23	.96175	43	.69281	22	.98777	43	32	
33	.67965	22	.96218	43	.69303	22	.98821	43	33	23 22
34	.67987	22	.96261	43	.69325	22	.98864	43	34	6 2.3 2.2
35	9.68010	23	9.96305	43	9.69347	22	9.98908	43	35	7 2.7 2.6
36	.68032	22	.96348	43	.69369	22	.98952	43	36	8 3.0 3.0
37	.68055	22	.96391	43	.69392	22	.98995	43	37	9 3.4 3.4
38	.68077	22	.96434	43	.69414	22	.99039	43	38	10 3.8 3.7
39	.68100	22	.96478	43	.69436	22	.99082	43	39	20 7.6 7.5
40	9.68122	23	9.96521	43	9.69458	22	9.99126	43	40	30 11.5 11.2
41	.68145	22	.96564	43	.69480	22	.99170	43	41	40 15.3 15.0
42	.68167	22	.96607	43	.69502	22	.99213	43	42	50 19.1 18.7
43	.68190	22	.96650	43	.69524	22	.99257	43	43	
44	.68212	22	.96694	43	.69546	22	.99300	43	44	
45	9.68235	23	9.96737	43	9.69568	22	9.99344	44	45	
46	.68257	22	.96780	43	.69590	22	.99388	43	46	
47	.68280	22	.96824	43	.69612	22	.99431	44	47	22 21
48	.68302	22	.96867	43	.69634	22	.99475	43	48	6 2.2 2.1
49	.68324	22	.96910	43	.69656	22	.99519	43	49	7 2.5 2.5
50	9.68347	23	9.96953	43	9.69678	22	9.99562	43	50	8 2.9 2.8
51	.68369	22	.96997	43	.69700	21	.99606	43	51	9 3.3 3.2
52	.68392	22	.97040	43	.69721	22	.99650	44	52	10 3.6 3.6
53	.68414	22	.97083	43	.69743	22	.99694	43	53	20 7.3 7.1
54	.68436	22	.97127	43	.69765	22	.99737	43	54	30 11.0 10.7
55	9.68459	23	9.97170	43	9.69787	22	9.99781	44	55	40 14.6 14.3
56	.68481	22	.97213	43	.69809	22	.99825	43	56	50 18.3 17.9
57	.68503	22	.97257	43	.69831	21	.99868	44	57	
58	.68526	22	.97300	43	.69853	22	.99912	43	58	
59	.68548	22	.97343	43	.69875	22	.99956	43	59	
60	9.68571	23	9.97387	43	9.69897	22	10.00000	44	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

60°

61°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.69897	22	10.00000	44	9.71197	21	10.02639	44	0	
1	.69919	21	.00044	43	.71218	21	.02684	44	1	
2	.69940	22	.00087	44	.71239	21	.02728	44	2	
3	.69962	22	.00131	43	.71261	21	.02772	44	3	
4	.69984	22	.00175	43	.71282	21	.02816	44	4	
5	9.70006	22	10.00219	44	9.71304	21	10.02861	44	5	45 44
6	.70028	22	.00262	44	.71325	21	.02905	44	6	4.5 4.4
7	.70050	22	.00306	44	.71346	21	.02949	44	7	5.2 5.2
8	.70072	22	.00350	43	.71368	21	.02994	44	8	6.0 5.9
9	.70093	22	.00394	44	.71389	21	.03038	44	9	6.7 6.7
10	9.70115	21	10.00438	44	9.71411	21	10.03082	44	10	7.5 7.4
11	.70137	22	.00482	43	.71432	21	.03127	44	11	15.0 14.8
12	.70159	22	.00525	44	.71453	21	.03171	44	12	22.5 22.2
13	.70181	21	.00569	44	.71475	21	.03215	44	13	30.0 29.6
14	.70202	22	.00613	44	.71496	21	.03260	44	14	40 37.5
15	9.70224	22	10.00657	44	9.71517	21	10.03304	44	15	37.1
16	.70246	21	.00701	43	.71539	21	.03348	44	16	
17	.70268	22	.00745	44	.71560	21	.03393	44	17	
18	.70289	21	.00789	44	.71581	21	.03437	44	18	
19	.70311	21	.00833	43	.71603	21	.03481	44	19	
20	9.70333	22	10.00876	44	9.71624	21	10.03526	44	20	44 43
21	.70355	22	.00920	44	.71645	21	.03570	44	21	4.4 4.3
22	.70376	22	.00964	44	.71667	21	.03615	44	22	5.1 5.1
23	.70398	21	.01008	44	.71688	21	.03659	44	23	5.8 5.8
24	.70420	21	.01052	44	.71709	21	.03704	44	24	6.6 6.5
25	9.70441	22	10.01096	44	9.71730	21	10.03748	44	25	7.3 7.2
26	.70463	21	.01140	44	.71752	21	.03793	44	26	14.6 14.5
27	.70485	22	.01184	44	.71773	21	.03837	44	27	22.0 21.7
28	.70507	21	.01228	44	.71794	21	.03881	44	28	30 29.0
29	.70528	21	.01272	44	.71815	21	.03926	44	29	40 36.6
30	9.70550	22	10.01316	44	9.71837	21	10.03970	44	30	36.2
31	.70572	21	.01360	44	.71858	21	.04015	44	31	
32	.70593	21	.01404	44	.71879	21	.04059	44	32	
33	.70615	21	.01448	44	.71900	21	.04104	44	33	22 21
34	.70636	22	.01492	44	.71922	21	.04149	44	34	2.2 2.1
35	9.70658	21	10.01536	44	9.71943	21	10.04193	44	35	2.5 2.5
36	.70680	21	.01580	44	.71964	21	.04238	44	36	2.9 2.8
37	.70701	21	.01624	44	.71985	21	.04282	44	37	3.3 3.2
38	.70723	22	.01668	44	.72006	21	.04327	44	38	3.6 3.6
39	.70745	21	.01712	44	.72028	21	.04371	44	39	7.3 7.1
40	9.70766	21	10.01756	44	9.72049	21	10.04416	44	40	11.0 10.7
41	.70788	21	.01800	44	.72070	21	.04461	44	41	14.6 14.3
42	.70809	21	.01844	44	.72091	21	.04505	44	42	18.3 17.9
43	.70831	21	.01889	44	.72112	21	.04550	44	43	
44	.70852	22	.01933	44	.72133	21	.04594	44	44	
45	9.70874	21	10.01977	44	9.72154	21	10.04639	44	45	
46	.70896	21	.02021	44	.72176	21	.04684	44	46	
47	.70917	21	.02065	44	.72197	21	.04728	44	47	21
48	.70939	21	.02109	44	.72218	21	.04773	44	48	6 2.1
49	.70960	21	.02153	44	.72239	21	.04818	44	49	7 2.4
50	9.70982	21	10.02197	44	9.72260	21	10.04862	44	50	2.8 2.8
51	.71003	21	.02242	44	.72281	21	.04907	44	51	3 3.1
52	.71025	21	.02286	44	.72302	21	.04952	44	52	10 3.5
53	.71046	21	.02330	44	.72323	21	.04996	44	53	20 7.0
54	.71068	21	.02374	44	.72344	21	.05041	44	54	30 10.5
55	9.71089	21	10.02418	44	9.72365	21	10.05086	44	55	40 14.0
56	.71111	21	.02463	44	.72386	21	.05131	44	56	50 17.5
57	.71132	21	.02507	44	.72408	21	.05175	44	57	
58	.71154	21	.02551	44	.72429	21	.05220	44	58	
59	.71175	21	.02595	44	.72450	21	.05265	44	59	
60	9.71197	21	10.02639	44	9.72471	21	10.05310	44	60	
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.

62°

63°

												P. P.			
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'						
0	9.72471		10.05310		9.73720		10.08015		0						
1	.72492	21	.05354	44	.73740	20	.08061	45	1						
2	.72513	21	.05399	45	.73761	21	.08106	45	2						
3	.72534	21	.05444	45	.73782	20	.08151	45	3						
4	.72555	21	.05489	44	.73802	20	.08197	45	4					46	46
5	9.72576	21	10.05534	45	9.73823	20	10.08242	45	5					4.6	4.6
6	.72597	21	.05579	44	.73843	20	.08288	45	6					5.4	5.3
7	.72618	21	.05623	44	.73864	20	.08333	45	7					6.2	6.1
8	.72639	21	.05668	45	.73884	21	.08379	45	8					7.0	6.9
9	.72660	21	.05713	45	.73905	20	.08424	45	9					7.7	7.6
10	9.72681	21	10.05758	44	9.73926	20	10.08470	45	10					15.5	15.3
11	.72701	20	.05803	44	.73946	20	.08515	45	11					23.2	23.0
12	.72722	21	.05848	45	.73967	20	.08561	45	12					30.2	30.6
13	.72743	21	.05893	45	.73987	20	.08606	45	13					31.0	30.6
14	.72764	21	.05938	45	.74008	20	.08652	45	14					38.7	38.3
15	9.72785	21	10.05983	45	9.74028	20	10.08697	45	15					4.5	4.5
16	.72806	21	.06028	44	.74049	20	.08743	46	16					4.5	4.5
17	.72827	20	.06072	45	.74069	20	.08789	45	17					5.3	5.2
18	.72848	21	.06117	45	.74090	20	.08834	45	18					6.0	6.0
19	.72869	21	.06162	45	.74110	20	.08880	45	19					6.8	6.7
20	9.72890	21	10.06207	45	9.74131	20	10.08926	46	20					7.6	7.5
21	.72911	20	.06252	45	.74151	20	.08971	45	21					15.1	15.0
22	.72931	21	.06297	45	.74172	20	.09017	45	22					22.7	22.5
23	.72952	21	.06342	45	.74192	20	.09062	46	23					30.3	30.0
24	.72973	21	.06387	45	.74213	20	.09108	45	24					37.9	37.5
25	9.72994	20	10.06432	45	9.74233	20	10.09154	46	25						
26	.73015	21	.06477	45	.74254	20	.09200	45	26						
27	.73036	21	.06522	45	.74274	20	.09245	45	27					4.4	4.4
28	.73057	20	.06568	45	.74294	20	.09291	46	28					5.2	5.2
29	.73077	20	.06613	45	.74315	20	.09337	45	29					5.9	5.9
30	9.73098	21	10.06658	45	9.74335	20	10.09382	46	30					6.7	6.7
31	.73119	20	.06703	45	.74356	20	.09428	46	31					7.4	7.4
32	.73140	21	.06748	45	.74376	20	.09474	46	32					14.8	14.8
33	.73161	21	.06793	45	.74396	20	.09520	46	33					22.2	22.2
34	.73181	20	.06838	45	.74417	20	.09566	46	34					29.6	29.6
35	9.73202	21	10.06883	45	9.74437	20	10.09611	45	35					37.1	37.1
36	.73223	20	.06928	45	.74458	20	.09657	46	36						
37	.73244	21	.06974	45	.74478	20	.09703	45	37						
38	.73265	20	.07019	45	.74498	20	.09749	46	38						
39	.73285	20	.07064	45	.74519	20	.09795	46	39						
40	9.73306	20	10.07109	45	9.74539	20	10.09841	45	40						
41	.73327	21	.07154	45	.74559	20	.09886	46	41						
42	.73348	21	.07200	45	.74580	20	.09932	46	42						
43	.73368	20	.07245	45	.74600	20	.09978	46	43						
44	.73389	20	.07290	45	.74620	20	.10024	46	44						
45	9.73410	20	10.07335	45	9.74641	20	10.10070	46	45						
46	.73430	21	.07380	45	.74661	20	.10116	46	46						
47	.73451	21	.07426	45	.74681	20	.10162	46	47						
48	.73472	20	.07471	45	.74702	20	.10208	46	48						
49	.73493	21	.07516	45	.74722	20	.10254	46	49						
50	9.73513	20	10.07562	45	9.74742	20	10.10300	46	50						
51	.73534	21	.07607	45	.74762	20	.10346	46	51						
52	.73555	20	.07652	45	.74783	20	.10392	46	52						
53	.73575	20	.07697	45	.74803	20	.10438	46	53						
54	.73596	20	.07743	45	.74823	20	.10484	46	54						
55	9.73617	21	10.07788	45	9.74844	20	10.10530	46	55						
56	.73637	20	.07834	45	.74864	20	.10576	46	56						
57	.73658	21	.07879	45	.74884	20	.10622	46	57						
58	.73679	21	.07924	45	.74904	20	.10668	46	58						
59	.73699	20	.07970	45	.74924	20	.10714	46	59						
60	9.73720	20	10.08015	45	9.74945	20	10.10760	46	60						
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.					

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECAN

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65°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.74945	20	10.10760	46	9.76146	19	10.13551	47	0	
1	.74965	20	.10807	46	.76166	20	.13598	47	1	
2	.74985	20	.10853	46	.76186	20	.13645	47	2	
3	.75005	20	.10899	46	.76206	19	.13692	47	3	
4	.75026	20	.10945	46	.76225	20	.13739	47	4	
5	9.75046	20	10.10991	46	9.76245	19	10.13785	47	5	
6	.75066	20	.11037	46	.76265	20	.13833	47	6	48 47
7	.75086	20	.11084	46	.76285	19	.13880	47	7	6 4.8 4.7
8	.75106	20	.11130	46	.76304	20	.13927	47	8	7 5.6 5.5
9	.75126	20	.11176	46	.76324	19	.13974	47	9	8 6.4 6.3
10	9.75147	20	10.11222	46	9.76344	20	10.14021	47	10	9 7.2 7.1
11	.75167	20	.11269	46	.76364	19	.14068	47	11	10 8.0 7.9
12	.75187	20	.11315	46	.76384	20	.14115	47	12	20 16.0 15.8
13	.75207	20	.11361	46	.76403	19	.14162	47	13	30 24.0 23.7
14	.75227	20	.11407	46	.76423	20	.14210	47	14	40 32.0 31.6
15	9.75247	20	10.11454	46	9.76443	19	10.14257	47	15	50 40.0 39.6
16	.75267	20	.11500	46	.76463	20	.14304	47	16	
17	.75287	20	.11546	46	.76482	19	.14351	47	17	
18	.75308	20	.11593	46	.76502	20	.14398	47	18	
19	.75328	20	.11639	46	.76522	19	.14445	47	19	
20	9.75348	20	10.11685	46	9.76541	20	10.14493	47	20	
21	.75368	20	.11732	46	.76561	19	.14540	47	21	
22	.75388	20	.11778	46	.76581	20	.14587	47	22	
23	.75408	20	.11825	46	.76600	19	.14634	47	23	
24	.75428	20	.11871	46	.76620	20	.14682	47	24	
25	9.75448	20	10.11917	46	9.76640	19	10.14729	47	25	
26	.75468	20	.11964	46	.76659	20	.14776	47	26	
27	.75488	20	.12010	46	.76679	19	.14823	47	27	
28	.75508	20	.12057	46	.76699	20	.14871	47	28	
29	.75528	20	.12103	46	.76718	19	.14918	47	29	
30	9.75548	20	10.12150	46	9.76738	20	10.14965	47	30	
31	.75568	20	.12196	46	.76758	19	.15013	47	31	
32	.75588	20	.12243	46	.76777	20	.15060	47	32	
33	.75608	20	.12289	46	.76797	19	.15108	47	33	
34	.75628	20	.12336	46	.76817	20	.15155	47	34	
35	9.75648	20	10.12383	47	9.76836	19	10.15202	47	35	
36	.75668	20	.12429	46	.76856	20	.15250	47	36	
37	.75688	20	.12476	46	.76875	19	.15297	47	37	
38	.75708	20	.12522	46	.76895	20	.15345	47	38	
39	.75728	20	.12569	46	.76915	19	.15392	47	39	
40	9.75748	20	10.12616	47	9.76934	20	10.15440	47	40	
41	.75768	20	.12662	46	.76954	19	.15487	47	41	
42	.75788	20	.12709	47	.76973	20	.15535	47	42	
43	.75808	19	.12756	46	.76993	19	.15582	47	43	
44	.75828	20	.12802	46	.77012	20	.15630	47	44	
45	9.75848	20	10.12849	47	9.77032	19	10.15678	48	45	
46	.75868	20	.12896	46	.77052	20	.15725	47	46	
47	.75888	20	.12942	46	.77071	19	.15773	47	47	
48	.75908	20	.12989	46	.77091	20	.15820	47	48	
49	.75928	19	.13036	47	.77110	19	.15868	48	49	
50	9.75947	20	10.13083	47	9.77130	20	10.15916	47	50	
51	.75967	20	.13130	46	.77149	19	.15963	48	51	
52	.75987	20	.13176	47	.77169	20	.16011	47	52	
53	.76007	19	.13223	46	.77188	19	.16059	47	53	
54	.76027	20	.13270	47	.77208	20	.16106	47	54	
55	9.76047	20	10.13317	47	9.77227	19	10.16154	48	55	
56	.76067	20	.13364	47	.77247	20	.16202	47	56	
57	.76087	19	.13411	46	.77266	19	.16250	48	57	
58	.76106	20	.13457	47	.77286	20	.16298	47	58	
59	.76126	20	.13504	47	.77305	19	.16345	48	59	
60	9.76146	20	10.13551	47	9.77325	20	10.16393	48	60	
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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												P. P.			
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D							
0	9.77325	19	10.16393	48	9.78481	19	10.19293	49	0						
1	.77344	19	.16441	47	.78500	19	.19342	49	1						
2	.77363	19	.16489	48	.78519	19	.19391	48	2						
3	.77383	19	.16537	48	.78538	19	.19439	49	3						
4	.77402	19	.16585	48	.78557	19	.19488	49	4						
5	9.77422	19	10.16633	48	9.78576	19	10.19537	49	5						
6	.77441	19	.16680	47	.78595	19	.19586	49	6						
7	.77461	19	.16728	48	.78614	19	.19635	49	7						
8	.77480	19	.16776	48	.78633	19	.19684	49	8						
9	.77499	19	.16824	48	.78652	19	.19733	49	9						
10	9.77519	19	10.16872	48	9.78671	19	10.19782	49	10						
11	.77538	19	.16920	48	.78690	19	.19831	49	11						
12	.77557	19	.16968	48	.78709	19	.19880	49	12						
13	.77577	19	.17016	48	.78728	19	.19929	49	13						
14	.77596	19	.17064	48	.78747	19	.19979	49	14						
15	9.77616	19	10.17112	48	9.78766	19	10.20028	49	15						
16	.77635	19	.17160	48	.78785	19	.20077	49	16						
17	.77654	19	.17209	48	.78804	19	.20126	49	17						
18	.77674	19	.17257	48	.78823	19	.20175	49	18						
19	.77693	19	.17305	48	.78842	19	.20224	49	19						
20	9.77712	19	10.17353	48	9.78861	19	10.20273	49	20						
21	.77732	19	.17401	48	.78880	19	.20323	49	21						
22	.77751	19	.17449	48	.78899	19	.20372	49	22						
23	.77770	19	.17498	48	.78918	19	.20421	49	23						
24	.77790	19	.17546	48	.78937	18	.20470	49	24						
25	9.77809	19	10.17594	48	9.78956	19	10.20520	49	25						
26	.77828	19	.17642	48	.78975	19	.20569	49	26						
27	.77847	19	.17690	48	.78994	19	.20618	49	27						
28	.77867	19	.17739	48	.79013	19	.20668	49	28						
29	.77886	19	.17787	48	.79032	19	.20717	49	29						
30	9.77905	19	10.17835	48	9.79051	18	10.20767	49	30						
31	.77925	19	.17884	48	.79069	19	.20816	49	31						
32	.77944	19	.17932	48	.79088	19	.20865	49	32						
33	.77963	19	.17980	48	.79107	19	.20915	49	33						
34	.77982	19	.18029	48	.79126	19	.20964	49	34						
35	9.78002	19	10.18077	48	9.79145	18	10.21014	49	35						
36	.78021	19	.18126	48	.79164	18	.21063	49	36						
37	.78040	19	.18174	48	.79183	19	.21113	49	37						
38	.78059	19	.18222	48	.79202	18	.21162	50	38						
39	.78078	19	.18271	48	.79220	18	.21212	49	39						
40	9.78098	19	10.18319	48	9.79239	19	10.21262	49	40						
41	.78117	19	.18368	48	.79258	19	.21311	49	41						
42	.78136	19	.18416	49	.79277	19	.21361	49	42						
43	.78155	19	.18465	48	.79296	19	.21410	50	43						
44	.78174	19	.18514	48	.79315	19	.21460	49	44						
45	9.78194	19	10.18562	48	9.79333	18	10.21510	49	45						
46	.78213	19	.18611	48	.79352	19	.21560	49	46						
47	.78232	19	.18659	48	.79371	18	.21609	49	47						
48	.78251	19	.18708	49	.79390	18	.21659	49	48						
49	.78270	19	.18757	48	.79409	19	.21709	49	49						
50	9.78289	19	10.18805	48	9.79427	18	10.21759	49	50						
51	.78309	19	.18854	49	.79446	19	.21808	49	51						
52	.78328	19	.18903	48	.79465	18	.21858	50	52						
53	.78347	19	.18951	49	.79484	19	.21908	49	53						
54	.78366	19	.19000	48	.79503	19	.21958	49	54						
55	9.78385	19	10.19049	48	9.79521	18	10.22008	50	55						
56	.78404	19	.19098	48	.79540	19	.22058	50	56						
57	.78423	19	.19146	49	.79559	19	.22108	50	57						
58	.78442	19	.19195	49	.79578	18	.22158	50	58						
59	.78462	19	.19244	48	.79596	19	.22208	50	59						
60	9.78481	19	10.19293	48	9.79615	19	10.22258	50	60						
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D							

P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS

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'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.79615	18	10.22258	50	9.80728	18	10.25295	51	0	53 52
1	.79634	19	.22308	50	.80747	18	.25347	51	1	6 5.3 5.2
2	.79653	18	.22358	50	.80765	18	.25398	51	2	7 6.2 6.1
3	.79671	18	.22408	50	.80783	18	.25449	51	3	8 7.0 7.0
4	.79690	18	.22458	50	.80802	18	.25501	51	4	9 7.9 7.9
5	9.79709	19	10.22508	50	9.80820	18	10.25552	51	5	10 8.8 8.7
6	.79727	18	.22558	50	.80839	18	.25604	51	6	20 17.6 17.5
7	.79746	18	.22608	50	.80857	18	.25655	51	7	30 26.5 26.2
8	.79765	18	.22658	50	.80875	18	.25707	51	8	40 35.3 35.0
9	.79783	18	.22708	50	.80894	18	.25758	51	9	50 44.1 43.7
10	9.79802	18	10.22759	50	9.80912	18	10.25810	51	10	52 51
11	.79821	18	.22809	50	.80930	18	.25861	51	11	6 5.2 5.1
12	.79839	18	.22859	50	.80949	18	.25913	51	12	7 6.0 6.0
13	.79858	18	.22909	50	.80967	18	.25964	51	13	8 6.9 6.8
14	.79877	18	.22960	50	.80985	18	.26016	51	14	9 7.8 7.7
15	9.79895	18	10.23010	50	9.81003	18	10.26067	52	15	10 8.6 8.6
16	.79914	18	.23060	50	.81022	18	.26119	51	16	20 17.3 17.1
17	.79933	18	.23110	50	.81040	18	.26171	51	17	30 26.0 25.7
18	.79951	18	.23161	50	.81058	18	.26222	52	18	40 34.6 34.3
19	.79970	18	.23211	50	.81077	18	.26274	52	19	50 43.3 42.9
20	9.79988	18	10.23262	50	9.81095	18	10.26326	51	20	51 50
21	.80007	19	.23312	50	.81113	18	.26378	52	21	6 5.1 5.0
22	.80026	18	.23362	50	.81131	18	.26429	51	22	7 5.9 5.9
23	.80044	18	.23413	50	.81150	18	.26481	52	23	8 6.8 6.7
24	.80063	18	.23463	50	.81168	18	.26533	52	24	9 7.6 7.6
25	9.80081	18	10.23514	50	9.81186	18	10.26585	52	25	10 8.5 8.4
26	.80100	19	.23564	50	.81204	18	.26637	51	26	20 17.0 16.8
27	.80119	18	.23615	50	.81223	18	.26689	52	27	30 25.5 25.2
28	.80137	18	.23666	50	.81241	18	.26741	52	28	40 34.0 33.6
29	.80156	18	.23716	50	.81259	18	.26793	52	29	50 42.5 42.1
30	9.80174	18	10.23767	50	9.81277	18	10.26845	52	30	50
31	.80193	18	.23817	51	.81295	18	.26897	52	31	6 5.0
32	.80211	18	.23868	50	.81314	18	.26949	52	32	7 5.8
33	.80230	18	.23919	50	.81332	18	.27001	52	33	8 6.6
34	.80248	18	.23969	50	.81350	18	.27053	52	34	9 7.5
35	9.80267	18	10.24020	50	9.81368	18	10.27105	52	35	10 8.3
36	.80286	18	.24071	51	.81386	18	.27157	52	36	20 16.6
37	.80304	18	.24122	50	.81405	18	.27209	52	37	30 25.0
38	.80323	18	.24172	51	.81423	18	.27261	52	38	40 33.3
39	.80341	18	.24223	51	.81441	18	.27314	52	39	50 41.6
40	9.80360	18	10.24274	50	9.81459	18	10.27366	52	40	19 18
41	.80378	18	.24325	51	.81477	18	.27418	52	41	6 1.9 1.8
42	.80397	18	.24376	51	.81495	18	.27470	52	42	7 2.2 2.1
43	.80415	18	.24427	51	.81513	18	.27523	52	43	8 2.5 2.4
44	.80434	18	.24478	51	.81532	18	.27575	52	44	9 2.8 2.8
45	9.80452	18	10.24529	51	9.81550	18	10.22627	52	45	10 3.1 3.1
46	.80470	18	.24580	51	.81568	18	.27680	52	46	20 6.3 6.1
47	.80489	18	.24631	51	.81586	18	.27732	52	47	30 9.5 9.2
48	.80507	18	.24682	51	.81604	18	.27785	52	48	40 12.6 12.3
49	.80526	18	.24733	51	.81622	18	.27837	52	49	50 15.8 15.4
50	9.80544	18	10.24784	51	9.81640	18	10.27890	52	50	18
51	.80563	18	.24835	51	.81658	18	.27942	52	51	6 1.8
52	.80587	18	.24886	51	.81676	18	.27995	52	52	7 2.1
53	.80600	18	.24937	51	.81695	18	.28047	52	53	8 2.4
54	.80618	18	.24988	51	.81713	18	.28100	52	54	9 2.7
55	9.80636	18	10.25039	51	9.81731	18	10.28152	53	55	10 3.0
56	.80655	18	.25090	51	.81749	18	.28205	52	56	20 6.0
57	.80673	18	.25142	51	.81767	18	.28258	52	57	30 9.0
58	.80692	18	.25193	51	.81785	18	.28310	53	58	40 12.0
59	.80710	18	.25244	51	.81803	18	.28363	52	59	50 15.0
60	9.80728	18	10.25295	51	9.81821	18	10.28416	52	60	P. P.
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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71°

70°				71°				P. P.			
Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D				
0	9.81821	18	10.28416	53	9.82894	17	10.31629	54	0	56	56
1	.81839	18	.28469	52	.82911	17	.31684	54	1	6	5.6
2	.81857	18	.28521	53	.82929	18	.31738	54	2	7	6.6
3	.81875	18	.28574	53	.82947	17	.31793	54	3	8	7.5
4	.81893	18	.28627	53	.82964	17	.31847	54	4	9	8.5
5	9.81911	18	10.28680	52	9.82982	18	10.31902	54	5	10	9.4
6	.81929	18	.28733	53	.83000	17	.31956	54	6	20	18.8
7	.81947	18	.28786	53	.83017	17	.32011	54	7	30	28.2
8	.81965	18	.28839	53	.83035	18	.32066	55	8	40	37.6
9	.81983	18	.28892	53	.83053	17	.32120	54	9	50	47.1
10	9.82001	18	10.28945	53	9.83070	18	10.32175	55	10	55	55
11	.82019	18	.28998	53	.83088	17	.32230	54	11	6	5.5
12	.82037	18	.29051	53	.83106	17	.32284	54	12	7	6.5
13	.82055	18	.29104	53	.83123	17	.32339	54	13	8	7.4
14	.82073	18	.29157	53	.83141	17	.32394	54	14	9	8.3
15	9.82091	17	10.29210	53	9.83159	18	10.32449	55	15	10	9.2
16	.82109	18	.29263	53	.83176	17	.32504	55	16	20	18.5
17	.82127	18	.29316	53	.83194	17	.32558	54	17	30	27.7
18	.82145	18	.29370	53	.83211	18	.32613	55	18	40	37.0
19	.82163	18	.29423	53	.83229	17	.32668	55	19	50	46.2
20	9.82181	18	10.29476	53	9.83247	17	10.32723	55	20	54	54
21	.82199	18	.29529	53	.83264	17	.32778	55	21	6	5.4
22	.82217	18	.29583	53	.83282	17	.32833	55	22	7	6.3
23	.82235	17	.29636	53	.83299	18	.32888	55	23	8	7.2
24	.82252	18	.29689	53	.83317	17	.32944	55	24	9	8.1
25	9.82270	18	10.29743	53	9.83335	17	10.32999	55	25	10	9.1
26	.82288	18	.29796	53	.83352	17	.33054	55	26	20	18.1
27	.82306	18	.29850	53	.83370	17	.33109	55	27	30	27.2
28	.82324	17	.29903	53	.83387	17	.33164	55	28	40	36.3
29	.82342	18	.29957	53	.83405	17	.33220	55	29	50	45.4
30	9.82360	18	10.30010	53	9.83422	17	10.33275	55	30	53	53
31	.82378	18	.30064	53	.83440	18	.33330	55	31	6	5.3
32	.82396	18	.30117	53	.83458	17	.33385	55	32	7	6.2
33	.82413	17	.30171	54	.83475	17	.33441	55	33	8	7.1
34	.82431	18	.30225	53	.83493	17	.33496	55	34	9	8.0
35	9.82449	17	10.30278	54	9.83510	17	10.33552	55	35	10	8.9
36	.82467	18	.30332	53	.83528	17	.33607	55	36	20	17.8
37	.82485	18	.30386	54	.83545	17	.33663	55	37	30	26.7
38	.82503	17	.30440	54	.83563	17	.33718	55	38	40	35.6
39	.82520	18	.30493	53	.83580	17	.33774	55	39	50	44.6
40	9.82538	18	10.30547	54	9.83598	17	10.33829	55	40	52	52
41	.82556	18	.30601	53	.83615	17	.33885	55	41	6	5.2
42	.82574	17	.30655	54	.83633	17	.33941	55	42	7	6.1
43	.82592	17	.30709	54	.83650	17	.33996	55	43	8	7.0
44	.82609	18	.30763	54	.83668	17	.34052	55	44	9	7.9
45	9.82627	17	10.30817	54	9.83685	17	10.34108	56	45	10	8.7
46	.82645	18	.30871	54	.83703	17	.34164	56	46	20	17.5
47	.82663	18	.30925	54	.83720	17	.34220	55	47	30	26.2
48	.82681	17	.30979	54	.83737	17	.34275	56	48	40	35.0
49	.82698	18	.31033	54	.83755	17	.34331	56	49	50	43.7
50	9.82716	17	10.31087	54	9.83772	17	10.34387	56	50	18	17
51	.82734	18	.31141	54	.83790	17	.34443	56	51	6	1.7
52	.82752	17	.31195	54	.83807	17	.34499	56	52	7	2.1
53	.82769	17	.31249	54	.83825	17	.34555	56	53	8	2.4
54	.82787	17	.31303	54	.83842	17	.34611	56	54	9	2.7
55	9.82805	18	10.31358	54	9.83859	17	10.34667	56	55	10	3.0
56	.82823	17	.31412	54	.83877	17	.34723	56	56	20	6.0
57	.82840	17	.31466	54	.83894	17	.34780	56	57	30	9.0
58	.82858	18	.31521	54	.83912	17	.34836	56	58	40	12.0
59	.82876	18	.31575	54	.83929	17	.34892	56	59	50	15.0
60	9.82894	18	10.31629	54	9.83946	17	10.34948	56	60	15.0	14.6
Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D				P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS

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		Lg. Vers.	D	Log. Exs.	D			Lg. Vers.	D	Log. Exs.	D			P. P.	
0	9.83946	17	10.34948	56	9.84980	17	10.38387	58	0			0		61	60
1	.83964	17	.35005	56	.84997	17	.38445	58	1	6	6.1	6.0		6	6.0
2	.83981	17	.35061	56	.85014	17	.38504	58	2	7	7.1	7.0		7	7.0
3	.83999	17	.35117	56	.85031	17	.38562	58	3	8	8.1	8.0		8	8.0
4	.84016	17	.35174	56	.85049	17	.38621	58	4	9	9.1	9.1		9	9.1
5	9.84033	17	10.35230	56	9.85066	17	10.38679	58	5	10	10.1	10.1		10	10.1
6	.84051	17	.35286	56	.85083	17	.38738	58	6	20	20.3	20.1		20	20.1
7	.84068	17	.35343	56	.85100	17	.38796	58	7	30	30.5	30.2		30	30.2
8	.84085	17	.35399	57	.85117	17	.38855	58	8	40	40.6	40.3		40	40.3
9	.84103	17	.35456	57	.85134	17	.38914	58	9	50	50.8	50.4		50	50.4
10	9.84120	17	10.35513	56	9.85151	17	10.38973	59	10					60	59
11	.84137	17	.35569	56	.85168	17	.39031	58	11	6	6.0	5.9		6	5.9
12	.84155	17	.35626	56	.85185	17	.39090	59	12	7	7.0	6.9		7	6.9
13	.84172	17	.35683	57	.85202	17	.39149	59	13	8	8.0	7.9		8	7.9
14	.84189	17	.35739	56	.85219	17	.39208	58	14	9	9.0	8.9		9	8.9
15	9.84207	17	10.35796	57	9.85236	17	10.39267	59	15	10	10.0	9.9		10	9.9
16	.84224	17	.35853	57	.85253	17	.39326	59	16	20	20.0	19.8		20	19.8
17	.84241	17	.35910	57	.85270	17	.39385	59	17	30	30.0	29.7		30	29.7
18	.84259	17	.35967	57	.85287	17	.39444	59	18	40	40.0	39.6		40	39.6
19	.84276	17	.36023	56	.85304	17	.39503	59	19	50	50.0	49.6		50	49.6
20	9.84293	17	10.36080	57	9.85321	17	10.39562	59	20					59	58
21	.84310	17	.36137	57	.85338	17	.39621	59	21	6	5.9	5.8		6	5.8
22	.84328	17	.36194	57	.85355	17	.39681	59	22	7	6.9	6.8		7	6.8
23	.84345	17	.36251	57	.85372	17	.39740	59	23	8	7.8	7.8		8	7.8
24	.84362	17	.36308	57	.85389	16	.39799	59	24	9	8.8	8.8		9	8.8
25	9.84380	17	10.36366	57	9.85405	17	10.39859	59	25	10	9.8	9.7		10	9.7
26	.84397	17	.36423	57	.85422	17	.39918	59	26	20	19.6	19.5		20	19.5
27	.84414	17	.36480	57	.85439	17	.39977	59	27	30	29.5	29.2		30	29.2
28	.84431	17	.36537	57	.85456	17	.40037	59	28	40	39.3	39.0		40	39.0
29	.84449	17	.36594	57	.85473	17	.40096	59	29	50	49.1	48.7		50	48.7
30	9.84466	17	10.36652	57	9.85490	17	10.40156	59	30					58	57
31	.84483	17	.36709	57	.85507	16	.40216	59	31	6	5.8	5.7		6	5.7
32	.84500	17	.36766	57	.85524	17	.40275	59	32	7	6.7	6.7		7	6.7
33	.84517	17	.36824	57	.85541	17	.40335	59	33	8	7.7	7.6		8	7.6
34	.84535	17	.36881	57	.85558	17	.40395	60	34	9	8.7	8.6		9	8.6
35	9.84552	17	10.36938	57	9.85575	17	10.40454	59	35	10	9.6	9.6		10	9.6
36	.84569	17	.36996	58	.85592	16	.40514	59	36	20	19.3	19.1		20	19.1
37	.84586	17	.37054	57	.85608	17	.40574	59	37	30	29.0	28.7		30	28.7
38	.84603	17	.37111	57	.85625	17	.40634	60	38	40	38.6	38.3		40	38.3
39	.84620	17	.37169	57	.85642	17	.40694	60	39	50	48.3	47.9		50	47.9
40	9.84638	17	10.37226	57	9.85659	16	10.40754	60	40					57	56
41	.84655	17	.37284	58	.85676	17	.40814	60	41	6	5.7	5.6		6	5.6
42	.84672	17	.37342	57	.85693	17	.40874	60	42	7	6.6	6.6		7	6.6
43	.84689	17	.37399	57	.85710	16	.40934	60	43	8	7.6	7.5		8	7.5
44	.84706	17	.37457	58	.85726	17	.40994	60	44	9	8.5	8.5		9	8.5
45	9.84724	17	10.37515	57	9.85743	17	10.41054	60	45	10	9.5	9.4		10	9.4
46	.84741	17	.37573	58	.85760	16	.41114	60	46	20	19.0	18.8		20	18.8
47	.84758	17	.37631	58	.85777	17	.41174	60	47	30	28.5	28.2		30	28.2
48	.84775	17	.37689	58	.85794	17	.41235	60	48	40	38.0	37.6		40	37.6
49	.84792	17	.37747	58	.85811	16	.41295	60	49	50	47.5	47.1		50	47.1
50	9.84809	17	10.37805	58	9.85827	17	10.41355	60	50					17	16
51	.84826	17	.37863	58	.85844	17	.41416	60	51	6	1.7	1.7		6	1.6
52	.84844	17	.37921	58	.85861	16	.41476	60	52	7	2.0	2.0		7	1.9
53	.84861	17	.37979	58	.85878	17	.41537	60	53	8	2.3	2.2		8	2.2
54	.84878	17	.38037	58	.85895	17	.41597	60	54	9	2.6	2.5		9	2.5
55	9.84895	17	10.38095	58	9.85911	16	10.41658	61	55	10	2.9	2.8		10	2.7
56	.84912	17	.38153	58	.85928	17	.41719	60	56	20	5.8	5.6		20	5.5
57	.84929	17	.38212	58	.85945	16	.41779	60	57	30	8.7	8.5		30	8.2
58	.84946	17	.38270	58	.85962	17	.41840	61	58	40	11.6	11.3		40	11.0
59	.84963	17	.38328	58	.85979	16	.41901	61	59	50	14.6	14.1		50	13.7
60	9.84980	17	10.38387	58	9.85995	16	10.41962	61	60						
		Lg. Vers.	D	Log. Exs.	D			Lg. Vers.	D	Log. Exs.	D			P. P.	

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

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	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.			
0	9.85995	17	10.41962	60	9.86992	16	10.45693	63	0	67	66	66	
1	.86012	16	.42022	61	.87009	16	.45756	63	1	6	6.7	6.6	6.6
2	.86029	17	.42083	61	.87025	16	.45820	64	2	7	7.8	7.7	7.7
3	.86046	16	.42144	61	.87042	16	.45884	63	3	8	8.9	8.8	8.8
4	.86062	17	.42205	61	.87058	16	.45947	64	4	9	10.0	10.0	9.9
5	9.86079	17	10.42266	61	9.87074	16	10.46011	64	5	10	11.1	11.1	11.0
6	.86096	16	.42327	61	.87091	16	.46075	64	6	20	22.3	22.1	22.0
7	.86113	17	.42388	61	.87107	16	.46139	64	7	30	33.5	33.3	33.0
8	.86129	16	.42450	61	.87124	16	.46203	64	8	40	44.6	44.3	44.0
9	.86146	17	.42511	61	.87140	16	.46267	64	9	50	55.8	55.4	55.0
10	9.86163	16	10.42572	61	9.87157	16	10.46331	64	10	65	65	64	
11	.86179	17	.42633	61	.87173	16	.46395	64	11	6	6.5	6.5	6.4
12	.86196	16	.42695	61	.87189	16	.46460	64	12	7	7.6	7.6	7.5
13	.86213	17	.42756	61	.87206	16	.46524	64	13	8	8.7	8.6	8.6
14	.86230	16	.42817	61	.87222	16	.46588	64	14	9	9.8	9.7	9.7
15	9.86246	16	10.42879	61	9.87239	16	10.46652	64	15	10	10.9	10.8	10.7
16	.86263	17	.42940	61	.87255	16	.46717	64	16	20	21.8	21.6	21.5
17	.86280	16	.43002	61	.87271	16	.46781	64	17	30	32.7	32.5	32.2
18	.86296	17	.43063	62	.87288	16	.46846	64	18	40	43.6	43.3	43.0
19	.86313	16	.43125	61	.87304	16	.46910	64	19	50	54.6	54.1	53.7
20	9.86330	16	10.43187	62	9.87320	16	10.46975	65	20	64	63	63	
21	.86346	17	.43249	61	.87337	16	.47040	64	21	6	6.4	6.3	6.3
22	.86363	16	.43310	62	.87353	16	.47104	64	22	7	7.4	7.4	7.3
23	.86380	17	.43372	61	.87370	16	.47169	64	23	8	8.5	8.4	8.4
24	.86396	16	.43434	62	.87386	16	.47234	65	24	9	9.6	9.5	9.4
25	9.86413	17	10.43496	62	9.87402	16	10.47299	65	25	10	10.6	10.6	10.5
26	.86430	16	.43558	62	.87419	16	.47364	65	26	20	21.3	21.1	21.0
27	.86446	17	.43620	62	.87435	16	.47429	65	27	30	32.0	31.7	31.5
28	.86463	16	.43682	62	.87451	16	.47494	65	28	40	42.6	42.3	42.0
29	.86479	17	.43744	62	.87468	16	.47559	65	29	50	53.3	52.9	52.5
30	9.86496	16	10.43806	62	9.87484	16	10.47624	65	30	62	62	61	
31	.86513	17	.43868	62	.87500	16	.47689	65	31	6	6.2	6.2	6.1
32	.86529	16	.43931	62	.87516	16	.47754	65	32	7	7.3	7.2	7.2
33	.86546	17	.43993	62	.87533	16	.47820	65	33	8	8.3	8.2	8.2
34	.86562	16	.44055	62	.87549	16	.47885	65	34	9	9.4	9.3	9.2
35	9.86579	17	10.44118	62	9.87565	16	10.47950	65	35	10	10.4	10.3	10.2
36	.86596	16	.44180	62	.87582	16	.48016	65	36	20	20.8	20.6	20.5
37	.86612	17	.44242	62	.87598	16	.48081	65	37	30	31.2	31.0	30.7
38	.86629	16	.44305	63	.87614	16	.48147	66	38	40	41.6	41.3	41.0
39	.86645	17	.44368	62	.87631	16	.48213	65	39	50	52.1	51.6	51.2
40	9.86662	16	10.44430	62	9.87647	16	10.48278	65	40	61	60		
41	.86678	17	.44493	63	.87663	16	.48344	66	41	6	6.1	6.0	
42	.86695	16	.44556	62	.87679	16	.48410	66	42	7	7.1	7.0	
43	.86712	17	.44618	63	.87696	16	.48476	66	43	8	8.1	8.0	
44	.86728	16	.44681	62	.87712	16	.48542	65	44	9	9.1	9.1	
45	9.86745	16	10.44744	63	9.87728	16	10.48607	66	45	10	10.1	10.1	
46	.86761	17	.44807	63	.87744	16	.48674	66	46	20	20.3	20.1	
47	.86778	16	.44870	63	.87761	16	.48740	66	47	30	30.5	30.2	
48	.86794	17	.44933	63	.87777	16	.48806	66	48	40	40.6	40.3	
49	.86811	16	.44996	63	.87793	16	.48872	66	49	50	50.8	50.4	
50	9.86827	16	10.45059	63	9.87809	16	10.48938	66	50	17	16	16	
51	.86844	17	.45122	63	.87825	16	.49004	66	51	6	1.7	1.6	1.6
52	.86860	16	.45185	63	.87842	16	.49071	66	52	7	2.0	1.9	1.8
53	.86877	17	.45248	63	.87858	16	.49137	66	53	8	2.2	2.2	2.1
54	.86893	16	.45312	63	.87874	16	.49204	66	54	9	2.5	2.5	2.4
55	9.86910	16	10.45375	63	9.87890	16	10.49270	66	55	10	2.8	2.7	2.6
56	.86926	17	.45439	63	.87906	16	.49337	66	56	20	5.6	5.5	5.3
57	.86943	16	.45502	63	.87923	16	.49403	67	57	30	8.5	8.2	8.0
58	.86959	17	.45565	63	.87939	16	.49470	66	58	40	11.3	11.0	10.6
59	.86976	16	.45629	64	.87955	16	.49537	67	59	50	14.1	13.7	13.3
60	9.86992	16	10.45693	64	9.87971	16	10.49604	67	60	P. P.			
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D					

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS

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'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.87971		10.49604		9.88933		10.53724		0	
1	.87987	16	.49670	66	.88949	16	.53794	70	1	
2	.88003	16	.49737	67	.88964	15	.53865	71	2	
3	.88020	16	.49804	67	.88980	16	.53936	70	3	
4	.88036	16	.49871	67	.88996	16	.54007	71	4	
5	9.88052	16	10.49939	67	9.89012	16	10.54078	71	5	75 74 73
6	.88068	16	.50006	67	.89028	15	.54149	71	6	6 7.5 7.4 7.3
7	.88084	16	.50073	67	.89044	16	.54220	71	7	7 8.7 8.6 8.5
8	.88100	16	.50140	67	.89060	16	.54291	71	8	8 10.0 9.8 9.7
9	.88116	16	.50208	67	.89075	15	.54362	71	9	9 11.2 11.1 10.9
10	9.88133	16	10.50275	67	9.89091	16	10.54433	71	10	10 12.5 12.3 12.1
11	.88149	16	.50342	67	.89107	15	.54505	71	11	20 25.0 24.6 24.3
12	.88165	16	.50410	67	.89123	15	.54576	71	12	30 37.5 37.0 36.5
13	.88181	16	.50477	68	.89139	16	.54647	72	13	40 50.0 49.3 48.6
14	.88197	16	.50545	67	.89155	15	.54719	71	14	50 62.5 61.6 60.8
15	9.88213	16	10.50613	68	9.89170	16	10.54791	71	15	
16	.88229	16	.50681	67	.89186	15	.54862	72	16	6 7.2 7.1 7.0
17	.88245	16	.50748	67	.89202	15	.54934	71	17	7 8.4 8.3 8.2
18	.88261	16	.50816	68	.89218	16	.55006	72	18	8 9.6 9.4 9.4
19	.88277	16	.50884	68	.89234	15	.55078	72	19	9 10.8 10.6 10.6
20	9.88294	16	10.50952	68	9.89249	16	10.55150	72	20	10 12.0 11.8 11.7
21	.88310	16	.51020	68	.89265	15	.55222	72	21	20 24.0 23.6 23.3
22	.88326	16	.51088	68	.89281	15	.55294	72	22	30 36.0 35.5 35.2
23	.88342	16	.51157	68	.89297	15	.55366	72	23	40 48.0 47.3 47.0
24	.88358	16	.51225	68	.89312	15	.55438	72	24	50 60.0 59.1 58.7
25	9.88374	16	10.51293	68	9.89328	16	10.55511	72	25	
26	.88390	16	.51361	68	.89344	15	.55583	72	26	6 6.9 6.8 6.7
27	.88406	16	.51430	68	.89360	16	.55655	73	27	7 8.0 7.9 7.8
28	.88422	16	.51498	68	.89376	15	.55728	72	28	8 9.2 9.0 8.9
29	.88438	16	.51567	69	.89391	15	.55801	72	29	9 10.3 10.2 10.0
30	9.88454	16	10.51636	69	9.89407	16	10.55873	73	30	10 11.5 11.3 11.1
31	.88470	16	.51704	68	.89423	15	.55946	72	31	20 23.0 22.6 22.3
32	.88486	16	.51773	69	.89438	16	.56019	73	32	30 34.5 34.0 33.5
33	.88502	16	.51842	69	.89454	15	.56092	73	33	40 46.0 45.3 44.6
34	.88518	16	.51911	69	.89470	15	.56165	73	34	50 57.5 56.6 55.8
35	9.88534	16	10.51980	69	9.89486	16	10.56238	73	35	
36	.88550	16	.52049	69	.89501	15	.56311	73	36	
37	.88566	16	.52118	69	.89517	15	.56384	73	37	
38	.88582	16	.52187	69	.89533	15	.56457	73	38	
39	.88598	16	.52256	69	.89548	15	.56531	73	39	
40	9.88614	16	10.52325	69	9.89564	16	10.56604	73	40	66 0
41	.88630	16	.52394	69	.89580	15	.56678	73	41	6 6.6 0.0 0.0
42	.88646	15	.52464	69	.89596	15	.56751	74	42	7 7.7 0.0 0.0
43	.88662	16	.52533	69	.89611	15	.56825	73	43	8 8.8 0.0 0.0
44	.88678	16	.52603	69	.89627	15	.56899	74	44	9 9.9 0.1 0.1
45	9.88694	16	10.52672	69	9.89643	16	10.56973	74	45	10 11.0 0.1 0.1
46	.88710	16	.52742	70	.89658	15	.57047	73	46	20 22.0 0.1 0.1
47	.88726	16	.52812	69	.89674	16	.57120	74	47	30 33.0 0.2 0.2
48	.88742	16	.52881	69	.89690	15	.57195	74	48	40 44.0 0.3 0.3
49	.88758	16	.52951	70	.89705	15	.57269	74	49	50 55.0 0.4 0.4
50	9.88774	16	10.53021	70	9.89721	15	10.57343	74	50	
51	.88790	15	.53091	70	.89737	16	.57417	74	51	6 1.6 1.6 1.5
52	.88805	16	.53161	70	.89752	15	.57491	74	52	7 1.9 1.8 1.8
53	.88821	16	.53231	70	.89768	15	.57566	74	53	8 2.2 2.1 2.0
54	.88837	16	.53301	70	.89783	15	.57640	74	54	9 2.5 2.4 2.3
55	9.88853	16	10.53372	70	9.89799	16	10.57715	75	55	10 2.7 2.6 2.6
56	.88869	15	.53442	70	.89815	15	.57790	74	56	20 5.5 5.3 5.1
57	.88885	15	.53512	70	.89830	15	.57864	74	57	30 8.2 8.0 7.7
58	.88901	16	.53583	70	.89846	16	.57939	75	58	40 11.0 10.6 10.3
59	.88917	16	.53653	70	.89862	15	.58014	75	59	50 13.7 13.3 12.9
60	9.88933	16	10.53724	70	9.89877	15	10.58089	75	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

78°

79°

P. P.									
86 85 84									
0	6	8.6	8.5	8.4					
1	7	10.0	9.9	9.8					
2	8	11.4	11.3	11.2					
3	9	12.9	12.7	12.6					
4	10	14.3	14.1	14.0					
5	20	28.6	28.3	28.0					
6	30	43.0	42.5	42.0					
7	40	57.3	56.6	56.0					
8	50	71.6	70.8	70.0					
9									
10	6	8.3	8.2	8.1					
11	7	9.7	9.5	9.4					
12	8	11.0	10.9	10.8					
13	9	12.4	12.3	12.1					
14	10	13.8	13.6	13.5					
15	20	27.6	27.3	27.0					
16	30	41.5	41.0	40.5					
17	40	55.3	54.6	54.0					
18	50	69.1	68.3	67.5					
19									
20	6	8.0	7.9	7.8					
21	7	9.3	9.2	9.1					
22	8	10.6	10.5	10.4					
23	9	12.0	11.8	11.7					
24	10	13.3	13.1	13.0					
25	20	26.6	26.3	26.0					
26	30	40.0	39.5	39.0					
27	40	53.3	52.6	52.0					
28	50	66.6	65.8	65.0					
29									
30	6	7.7	7.6	7.5					
31	7	9.0	8.8	8.7					
32	8	10.2	10.1	10.0					
33	9	11.5	11.4	11.2					
34	10	12.8	12.6	12.5					
35	20	25.6	25.3	25.0					
36	30	38.5	38.0	37.5					
37	40	51.1	50.6	50.0					
38	50	64.1	63.3	62.5					
39									
40	0	0.0	0.0	0.0					
41	1	0.0	0.0	0.0					
42	2	0.0	0.0	0.0					
43	3	0.0	0.0	0.0					
44	4	0.0	0.0	0.0					
45	5	0.0	0.0	0.0					
46	6	0.0	0.0	0.0					
47	7	0.0	0.0	0.0					
48	8	0.0	0.0	0.0					
49	9	0.0	0.0	0.0					
50	10	0.0	0.0	0.0					
51	16	1.5	1.5	1.5					
52	7	1.8	1.8	1.7					
53	8	2.1	2.0	2.0					
54	9	2.4	2.3	2.2					
55	10	2.6	2.6	2.5					
56	20	5.3	5.1	5.0					
57	30	8.0	7.7	7.5					
58	40	10.6	10.3	10.0					
59	50	13.3	12.9	12.5					
60									

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS

80°

81°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.91716	15	10.67749	86	9.92612	14	10.73178	95	0	90 80
1	.91731	15	.67836	87	.92626	15	.73273	94	1	6 9.0 8.0
2	.91746	15	.67923	87	.92641	14	.73368	95	2	7 10.5 9.3
3	.91761	15	.68010	87	.92656	15	.73463	95	3	8 12.0 10.6
4	.91776	15	.68097	87	.92671	15	.73558	95	4	9 13.5 12.0
5	9.91791	15	10.68184	87	9.92686	15	10.73653	95	5	10 15.0 13.3
6	.91807	15	.68272	87	.92700	14	.73748	95	6	20 30.0 26.6
7	.91822	15	.68359	87	.92715	15	.73844	95	7	30 45.0 40.0
8	.91837	15	.68447	87	.92730	14	.73940	96	8	40 60.0 53.3
9	.91852	15	.68534	87	.92745	15	.74035	95	9	50 75.0 66.6
10	9.91867	15	10.68622	88	9.92759	14	10.74131	96	10	9 8
11	.91882	15	.68710	88	.92774	15	.74227	96	11	6 0.9 0.8
12	.91897	15	.68798	88	.92789	14	.74324	96	12	7 1.0 0.9
13	.91912	15	.68886	88	.92804	15	.74420	96	13	8 1.2 1.0
14	.91927	15	.68975	88	.92818	14	.74517	96	14	9 1.3 1.2
15	9.91942	15	10.69063	88	9.92833	15	10.74613	96	15	10 1.5 1.3
16	.91957	15	.69152	88	.92848	14	.74710	97	16	20 3.0 2.6
17	.91972	15	.69240	89	.92862	15	.74807	97	17	30 4.5 4.0
18	.91987	15	.69329	89	.92877	14	.74905	97	18	40 6.0 5.3
19	.92002	15	.69418	89	.92892	15	.75002	97	19	50 7.5 6.6
20	9.92016	14	10.69507	89	9.92907	14	10.75099	97	20	7 6
21	.92031	15	.69596	89	.92921	15	.75197	98	21	6 0.7 0.6
22	.92046	15	.69686	89	.92936	14	.75295	97	22	7 0.8 0.7
23	.92061	15	.69775	89	.92951	15	.75393	98	23	8 0.0 0.8
24	.92076	15	.69865	89	.92965	14	.75491	98	24	9 1.0 0.9
25	9.92091	15	10.69955	90	9.92980	15	10.75589	98	25	10 1.1 1.0
26	.92106	15	.70044	90	.92995	14	.75688	98	26	20 2.3 2.0
27	.92121	15	.70134	90	.93009	15	.75786	98	27	30 3.5 3.0
28	.92136	14	.70224	90	.93024	14	.75885	99	28	40 4.6 4.0
29	.92151	15	.70315	90	.93039	15	.75984	99	29	50 5.8 5.0
30	9.92166	15	10.70405	90	9.93053	14	10.76083	99	30	5 4
31	.92181	15	.70495	91	.93068	15	.76182	99	31	6 0.5 0.4
32	.92196	15	.70586	91	.93083	14	.76282	99	32	7 0.6 0.4
33	.92211	15	.70677	91	.93097	15	.76382	100	33	8 0.6 0.5
34	.92226	15	.70768	91	.93112	14	.76481	99	34	9 0.7 0.6
35	9.92240	14	10.70859	91	9.93127	15	10.76581	100	35	10 0.8 0.6
36	.92255	15	.70950	91	.93141	14	.76681	100	36	20 1.6 1.3
37	.92270	15	.71041	91	.93156	15	.76782	100	37	30 2.5 2.0
38	.92285	15	.71133	91	.93171	14	.76882	100	38	40 3.3 2.6
39	.92300	14	.71224	91	.93185	15	.76983	100	39	50 4.1 3.3
40	9.92315	15	10.71316	92	9.93200	14	10.77083	100	40	15 15
41	.92330	15	.71408	92	.93214	15	.77184	101	41	6 1.5 1.5
42	.92345	15	.71500	92	.93229	14	.77286	101	42	7 1.8 1.7
43	.92360	14	.71592	92	.93244	15	.77387	101	43	8 2.0 2.0
44	.92374	15	.71684	92	.93258	14	.77488	101	44	9 2.3 2.2
45	9.92389	15	10.71776	92	9.93273	15	10.77590	102	45	10 2.6 2.5
46	.92404	14	.71869	92	.93287	14	.77692	102	46	20 5.1 5.0
47	.92419	15	.71961	93	.93302	15	.77794	102	47	30 7.7 7.5
48	.92434	15	.72054	92	.93317	14	.77896	102	48	40 10.3 10.0
49	.92449	14	.72147	92	.93331	15	.77998	102	49	50 12.9 12.5
50	9.92463	15	10.72240	93	9.93346	14	10.78101	102	50	17
51	.92478	15	.72333	93	.93360	15	.78203	103	51	6 1.4
52	.92493	14	.72427	93	.93375	14	.78306	103	52	7 1.7
53	.92508	15	.72520	93	.93389	15	.78409	103	53	8 1.9
54	.92523	15	.72614	93	.93404	14	.78513	103	54	9 2.2
55	9.92538	14	10.72707	94	9.93419	15	10.78616	104	55	10 2.4
56	.92552	15	.72801	94	.93433	14	.78720	103	56	20 4.8
57	.92567	15	.72895	94	.93448	15	.78823	104	57	30 7.2
58	.92582	14	.72990	94	.93462	14	.78927	104	58	40 9.6
59	.92597	15	.73084	94	.93477	15	.79031	104	59	50 12.1
60	9.92612	15	10.73178	94	9.93491	14	10.79136	104	60	P. P.
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

82°

83°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.93491	14	10.79136	104	9.94356	14	10.85766	117	0	
1	9.93506	14	.79240	105	9.94370	14	.85884	117	1	
2	9.93520	14	.79345	104	9.94384	14	.86001	117	2	
3	9.93535	14	.79450	105	9.94398	14	.86119	118	3	
4	9.93549	14	.79555	105	9.94413	14	.86237	118	4	
5	9.93564	14	10.79660	105	9.94427	14	10.86355	118	5	6 13.0 12.0
6	9.93578	14	.79766	105	9.94441	14	.86474	118	6	7 15.1 14.0
7	9.93593	14	.79871	105	9.94456	14	.86592	118	7	8 17.3 16.0
8	9.93607	14	.79977	106	9.94470	14	.86711	119	8	9 19.5 18.0
9	9.93622	14	.80083	106	9.94484	14	.86831	119	9	10 21.6 20.0
10	9.93636	14	10.80189	106	9.94498	14	10.86950	119	10	20 43.3 40.0
11	9.93651	14	.80296	106	9.94512	14	.87070	120	11	30 65.0 60.0
12	9.93665	14	.80402	107	9.94527	14	.87190	120	12	40 86.6 80.0
13	9.93680	14	.80509	107	9.94541	14	.87310	120	13	50 108.3 100.0
14	9.93694	14	.80616	107	9.94555	14	.87431	120	14	
15	9.93709	14	10.80723	107	9.94569	14	10.87552	121	15	
16	9.93723	14	.80831	107	9.94584	14	.87673	121	16	6 11.0 10.0
17	9.93738	14	.80938	108	9.94598	14	.87794	121	17	7 12.8 11.6
18	9.93752	14	.81046	108	9.94612	14	.87916	121	18	8 14.6 13.3
19	9.93767	14	.81154	108	9.94626	14	.88038	122	19	9 16.5 15.0
20	9.93781	14	10.81262	108	9.94640	14	10.88160	122	20	10 18.3 16.6
21	9.93796	14	.81371	108	9.94655	14	.88282	122	21	20 36.6 33.3
22	9.93810	14	.81479	109	9.94669	14	.88405	123	22	30 55.0 50.0
23	9.93824	14	.81588	109	9.94683	14	.88528	123	23	40 73.3 66.6
24	9.93839	14	.81697	109	9.94697	14	.88651	123	24	50 91.6 83.3
25	9.93853	14	10.81806	109	9.94711	14	10.88775	124	25	
26	9.93868	14	.81916	109	9.94726	14	.88898	124	26	3 0.3 0.2
27	9.93882	14	.82025	110	9.94740	14	.89022	124	27	6 0.3 0.2
28	9.93897	14	.82135	110	9.94754	14	.89147	124	28	7 0.3 0.2
29	9.93911	14	.82245	110	9.94768	14	.89271	124	29	8 0.4 0.2
30	9.93925	14	10.82356	110	9.94782	14	10.89396	125	30	9 0.4 0.3
31	9.93940	14	.82466	110	9.94796	14	.89521	125	31	10 0.5 0.3
32	9.93954	14	.82577	111	9.94810	14	.89647	125	32	20 1.0 0.6
33	9.93969	14	.82688	111	9.94825	14	.89773	126	33	30 1.5 1.0
34	9.93983	14	.82799	111	9.94839	14	.89899	126	34	40 2.0 1.3
35	9.93997	14	10.82910	111	9.94853	14	10.90025	126	35	50 2.5 1.6
36	9.94012	14	.83022	111	9.94867	14	.90152	127	36	
37	9.94026	14	.83133	112	9.94881	14	.90279	127	37	
38	9.94041	14	.83245	112	9.94895	14	.90406	127	38	1 0.1 0.0
39	9.94055	14	.83358	112	9.94909	14	.90533	127	39	6 0.1 0.0
40	9.94069	14	10.83470	112	9.94923	14	10.90661	128	40	7 0.1 0.0
41	9.94084	14	.83583	112	9.94938	14	.90789	128	41	8 0.1 0.0
42	9.94098	14	.83695	113	9.94952	14	.90917	128	42	9 0.1 0.1
43	9.94112	14	.83809	113	9.94966	14	.91046	129	43	10 0.1 0.1
44	9.94127	14	.83922	113	9.94980	14	.91175	129	44	20 0.3 0.1
45	9.94141	14	10.84035	113	9.94994	14	10.91304	129	45	30 0.5 0.2
46	9.94155	14	.84149	114	9.95008	14	.91434	130	46	40 0.6 0.3
47	9.94170	14	.84263	114	9.95022	14	.91564	130	47	50 0.8 0.4
48	9.94184	14	.84377	114	9.95036	14	.91694	130	48	
49	9.94198	14	.84492	115	9.95050	14	.91825	130	49	
50	9.94213	14	10.84607	115	9.95064	14	10.91956	131	50	6 1.4 1.4
51	9.94227	14	.84721	115	9.95078	14	.92087	131	51	7 1.7 1.6
52	9.94241	14	.84837	115	9.95093	14	.92218	131	52	8 1.9 1.8
53	9.94256	14	.84952	116	9.95107	14	.92350	131	53	9 2.2 2.1
54	9.94270	14	.85068	116	9.95121	14	.92482	132	54	10 2.4 2.3
55	9.94284	14	10.85183	116	9.95135	14	10.92614	132	55	20 4.8 4.6
56	9.94299	14	.85299	116	9.95149	14	.92747	133	56	30 7.2 7.0
57	9.94313	14	.85416	116	9.95163	14	.92880	133	57	40 9.6 9.3
58	9.94327	14	.85532	117	9.95177	14	.93014	133	58	50 12.1 11.6
59	9.94341	14	.85649	117	9.95191	14	.93147	133	59	
60	9.94356	14	10.85766	117	9.95205	14	10.93281	134	60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

84°

85°

'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	P. P.
0	9.95205	14	10.93281	134	9.96039	14	11.02010	158	0	190 180
1	.95219	14	.93416	135	.96053	13	.02168	159	1	6 19.0 18.0
2	.95233	14	.93551	135	.96067	13	.02327	159	2	7 22.1 21.0
3	.95247	14	.93686	135	.96081	14	.02487	159	3	8 25.3 24.0
4	.95261	14	.93821	135	.96095	14	.02646	159	4	9 28.5 27.0
5	9.95275	14	10.93957	135	9.96108	13	11.02807	160	5	10 31.6 30.0
6	.95289	14	.94093	136	.96122	14	.02968	161	6	20 63.3 60.0
7	.95303	14	.94229	136	.96136	13	.03129	161	7	30 95.0 90.0
8	.95317	14	.94366	137	.96150	14	.03291	161	8	40 126.6 120.0
9	.95331	14	.94503	137	.96163	13	.03453	162	9	50 158.3 150.0
10	9.95345	14	10.94641	137	9.96177	14	11.03616	163	10	170 160
11	.95359	14	.94778	137	.96191	13	.03780	163	11	6 17.0 16.0
12	.95373	13	.94917	138	.96205	14	.03944	164	12	7 19.8 18.6
13	.95387	14	.95055	138	.96218	13	.04108	164	13	8 22.6 21.3
14	.95401	14	.95194	139	.96232	14	.04273	165	14	9 25.5 24.0
15	9.95415	14	10.95333	139	9.96246	13	11.04438	165	15	10 28.3 26.6
16	.95429	14	.95473	139	.96259	13	.04604	166	16	20 56.6 53.3
17	.95443	14	.95613	140	.96273	14	.04771	167	17	30 85.0 80.0
18	.95457	14	.95753	140	.96287	13	.04938	167	18	40 113.3 106.6
19	.95471	14	.95894	141	.96301	14	.05106	167	19	50 141.6 133.3
20	9.95485	14	10.96035	141	9.96314	13	11.05274	168	20	150 140
21	.95499	14	.96176	141	.96328	14	.05443	169	21	6 15.0 14.0
22	.95513	14	.96318	142	.96342	13	.05612	169	22	7 17.5 16.3
23	.95527	13	.96461	142	.96355	13	.05782	169	23	8 20.0 18.6
24	.95540	13	.96603	142	.96369	14	.05952	170	24	9 22.5 21.0
25	9.95554	14	10.96746	143	9.96383	13	11.06123	171	25	10 25.0 23.3
26	.95568	14	.96889	143	.96397	14	.06295	172	26	20 50.0 46.6
27	.95582	14	.97033	144	.96410	13	.06467	172	27	30 75.0 70.0
28	.95596	14	.97177	144	.96424	13	.06640	173	28	40 100.0 93.3
29	.95610	14	.97322	144	.96438	14	.06813	173	29	50 125.0 116.6
30	9.95624	13	10.97467	145	9.96451	13	11.06987	174	30	130 9 8
31	.95638	14	.97612	145	.96465	13	.07161	174	31	6 13.0 10.9 0.8
32	.95652	14	.97758	145	.96479	14	.07336	175	32	7 15.1 11.0 0.9
33	.95666	14	.97904	146	.96492	13	.07512	176	33	8 17.3 11.2 1.0
34	.95680	13	.98050	146	.96506	13	.07688	176	34	9 19.5 11.3 1.2
35	9.95693	14	10.98197	147	9.96519	14	11.07865	177	35	10 21.6 11.5 1.3
36	.95707	14	.98345	147	.96533	13	.08043	177	36	20 43.3 33.0 2.6
37	.95721	14	.98492	148	.96547	13	.08221	178	37	30 65.0 45.4 4.0
38	.95735	14	.98640	149	.96560	14	.08400	179	38	40 86.6 60.5 5.3
39	.95749	13	.98789	149	.96574	13	.08579	179	39	50 108.3 75.6 6.6
40	9.95763	14	10.98938	149	9.96588	13	11.08759	180	40	7 6 5
41	.95777	14	.99087	149	.96601	13	.08940	180	41	6 0.7 0.6 0.5
42	.95791	13	.99237	150	.96615	14	.09121	181	42	7 0.8 0.7 0.6
43	.95804	13	.99387	150	.96629	13	.09303	182	43	8 0.9 0.8 0.6
44	.95818	14	.99538	151	.96642	13	.09486	182	44	9 1.0 0.9 0.7
45	9.95832	14	10.99689	151	9.96656	13	11.09669	183	45	10 1.1 1.0 0.8
46	.95846	13	.99841	152	.96669	13	.09853	184	46	20 2.3 2.0 1.6
47	.95860	14	10.99993	152	.96683	14	.10038	185	47	30 3.5 3.0 2.5
48	.95874	13	11.00145	153	.96697	13	.10223	186	48	40 4.6 4.0 3.3
49	.95888	14	.00298	153	.96710	13	.10409	186	49	50 5.8 5.0 4.1
50	9.95901	14	11.00451	153	9.96724	13	11.10595	187	50	14 14 13
51	.95915	13	.00605	154	.96737	13	.10783	188	51	6 1.4 1.4 1.3
52	.95929	13	.00759	154	.96751	13	.10971	189	52	7 1.7 1.6 1.6
53	.95943	14	.00914	155	.96764	13	.11160	189	53	8 1.9 1.8 1.8
54	.95957	13	.01069	155	.96778	14	.11349	190	54	9 2.2 2.1 2.0
55	9.95970	14	11.01225	155	9.96792	13	11.11539	191	55	10 2.4 2.3 2.2
56	.95984	14	.01381	156	.96805	13	.11730	191	56	20 4.8 4.6 4.5
57	.95998	13	.01537	157	.96819	13	.11922	192	57	30 7.2 7.0 6.7
58	.96012	14	.01694	157	.96832	13	.12114	193	58	40 9.6 9.3 9.0
59	.96026	13	.01852	158	.96846	13	.12307	193	59	50 12.1 11.6 11.2
60	9.96039	13	11.02010	158	9.96859	13	11.12501	193	60	P. P.
'	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D	'	

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

86°

87°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.96859	13	11.12501	195	9.97665	13	11.25785	255	0	250 240
1	.96887	14	.12696	195	.97679	13	.26040	256	1	6 25.0 24.0
2	.96887	14	.12891	196	.97692	13	.26297	257	2	7 29.1 28.0
3	.96900	13	.13087	196	.97705	13	.26554	259	3	8 33.3 32.0
4	.96914	13	.13284	198	.97718	13	.26814	260	4	9 37.5 36.0
5	9.96927	13	11.13482	198	9.97732	13	11.27074	262	5	10 41.6 40.0
6	.96941	13	.13680	199	.97745	13	.27336	263	6	20 83.3 80.0
7	.96954	13	.13879	200	.97758	13	.27599	265	7	30 125.0 120.0
8	.96968	13	.14079	201	.97772	13	.27864	266	8	40 166.6 160.0
9	.96981	13	.14280	202	.97785	13	.28131	267	9	50 208.3 200.0
10	9.96995	13	11.14482	202	9.97798	13	11.28398	269	10	230 220
11	.97008	13	.14684	203	.97811	13	.28668	270	11	6 23.0 22.0
12	.97022	13	.14887	204	.97825	13	.28938	272	12	7 26.8 25.6
13	.97035	13	.15092	205	.97838	13	.29211	274	13	8 30.6 29.3
14	.97049	13	.15297	205	.97851	13	.29485	275	14	9 34.5 33.0
15	9.97062	13	11.15502	206	9.97864	13	11.29760	277	15	10 38.3 36.6
16	.97076	13	.15709	208	.97878	13	.30037	278	16	20 76.6 73.3
17	.97089	13	.15917	208	.97891	13	.30316	280	17	30 115.0 110.0
18	.97103	13	.16125	209	.97904	13	.30596	282	18	40 153.3 146.6
19	.97116	13	.16334	210	.97917	13	.30878	283	19	50 191.6 183.3
20	9.97130	13	11.16544	211	9.97931	13	11.31162	285	20	210 200
21	.97143	13	.16755	212	.97944	13	.31447	287	21	6 21.0 20.0
22	.97157	13	.16967	213	.97957	13	.31734	288	22	7 24.5 23.3
23	.97170	13	.17180	214	.97970	13	.32023	290	23	8 28.0 26.6
24	.97183	13	.17394	214	.97984	13	.32313	292	24	9 31.5 30.0
25	9.97197	13	11.17609	215	9.97997	13	11.32606	294	25	10 35.0 33.3
26	.97210	13	.17824	216	.98010	13	.32900	296	26	20 70.0 66.6
27	.97224	13	.18041	218	.98023	13	.33196	298	27	30 105.0 100.0
28	.97237	13	.18259	218	.98036	13	.33494	299	28	40 140.0 133.3
29	.97251	13	.18477	219	.98050	13	.33793	301	29	50 175.0 166.6
30	9.97264	13	11.18697	220	9.98063	13	11.34095	303	30	190 4 3
31	.97277	13	.18917	221	.98076	13	.34398	305	31	6 19.0 0.4 0.3
32	.97291	13	.19138	222	.98089	13	.34704	307	32	7 22.1 0.4 0.3
33	.97304	13	.19361	223	.98102	13	.35011	309	33	8 25.3 0.5 0.4
34	.97318	13	.19584	224	.98116	13	.35321	311	34	9 28.5 0.6 0.4
35	9.97331	13	11.19809	225	9.98129	13	11.35632	313	35	10 31.6 0.6 0.5
36	.97345	13	.20034	227	.98142	13	.35946	315	36	20 63.3 1.3 1.0
37	.97358	13	.20261	227	.98155	13	.36261	318	37	30 95.0 2.0 1.5
38	.97371	33	.20489	228	.98168	13	.36579	320	38	40 126.6 2.6 2.0
39	.97385	13	.20717	230	.98181	13	.36899	322	39	50 158.3 3.3 2.5
40	9.97398	13	11.20947	230	9.98195	13	11.37221	324	40	2 1 0
41	.97412	13	.21178	232	.98208	13	.37546	326	41	6 0.2 0.1 0.0
42	.97425	13	.21410	233	.98221	13	.37872	328	42	7 0.2 0.1 0.0
43	.97438	13	.21643	234	.98234	13	.38201	331	43	8 0.2 0.1 0.0
44	.97452	13	.21877	235	.98247	13	.38532	333	44	9 0.3 0.1 0.1
45	9.97465	13	11.22112	236	9.98260	13	11.38866	335	45	10 0.3 0.1 0.1
46	.97478	13	.22349	237	.98273	13	.39201	338	46	20 0.6 0.3 0.2
47	.97492	13	.22586	239	.98287	13	.39540	340	47	30 1.0 0.5 0.2
48	.97505	13	.22825	239	.98300	13	.39880	343	48	40 1.3 0.6 0.3
49	.97519	13	.23065	241	.98313	13	.40224	345	49	50 1.6 0.8 0.4
50	9.97532	13	11.23306	242	9.98326	13	11.40569	348	50	14 13 13
51	.97545	13	.23548	243	.98339	13	.40918	351	51	6 1.4 1.3 1.3
52	.97559	13	.23792	245	.98352	13	.41269	353	52	7 1.6 1.6 1.5
53	.97572	13	.24037	246	.98365	13	.41622	356	53	8 1.8 1.8 1.7
54	.97585	13	.24283	247	.98378	13	.41979	359	54	9 2.1 2.0 1.9
55	9.97599	13	11.24530	248	9.98392	13	11.42338	361	55	10 2.3 2.2 2.1
56	.97612	13	.24778	250	.98405	13	.42699	364	56	20 4.6 4.5 4.3
57	.97625	13	.25028	251	.98418	13	.43064	367	57	30 7.0 6.7 6.5
58	.97639	13	.25279	252	.98431	13	.43431	370	58	40 9.3 9.0 8.6
59	.97652	13	.25531	254	.98444	13	.43802	373	59	50 11.6 11.2 10.8
60	9.97665	13	11.25785	254	9.98457	13	11.44175	373	60	P. P.
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		

TABLE VIII.—LOGARITHMIC VERSED SINES AND EXTERNAL SECANTS.

88°

89°

	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.
0	9.98457	13	11.44175	376	9.99235	12	11.75050	742	0	
1	.98470	13	.44551	379	.99248	13	.75792	755	1	
2	.98483	13	.44931	382	.99261	13	.76547	768	2	
3	.98496	13	.45313	386	.99274	13	.77316	781	3	
4	.98509	13	.45699	389	.99287	12	.78097	795	4	
5	9.98522	13	11.46088	392	9.99299	13	11.78892	809	5	
6	.98535	13	.46480	395	.99312	13	.79702	825	6	
7	.98548	13	.46876	399	.99325	12	.80527	840	7	
8	.98562	13	.47275	402	.99338	13	.81367	856	8	
9	.98575	13	.47677	406	.99351	12	.82223	872	9	
10	9.98588	13	11.48083	409	9.99363	13	11.83095	890	10	
11	.98601	13	.48493	413	.99376	13	.83986	908	11	
12	.98614	13	.48906	417	.99389	12	.84894	927	12	
13	.98627	13	.49323	420	.99402	13	.85821	947	13	
14	.98640	13	.49743	425	.99415	13	.86768	967	14	
15	9.98653	13	11.50168	428	9.99428	12	11.87735	989	15	
16	.98666	13	.50597	432	.99440	13	.88724	1009	16	
17	.98679	13	.51029	436	.99453	12	.89735	1031	17	
18	.98692	13	.51466	440	.99466	13	.90769	1059	18	
19	.98705	13	.51906	445	.99479	12	.91829	1085	19	
20	9.98718	13	11.52351	449	9.99491	13	11.92914	1112	20	
21	.98731	13	.52801	454	.99504	13	.94026	1140	21	
22	.98744	13	.53255	458	.99517	12	.95167	1171	22	
23	.98757	13	.53713	463	.99530	13	.96338	1203	23	
24	.98770	13	.54176	467	.99543	12	.97541	1236	24	
25	9.98783	13	11.54643	472	9.99555	13	11.98777	1271	25	
26	.98796	13	.55116	477	.99568	12	12.00048	1309	26	
27	.98809	13	.55593	482	.99581	13	.01358	1349	27	
28	.98822	13	.56076	487	.99594	12	.02707	1391	28	
29	.98835	13	.56563	492	.99606	13	.04098	1436	29	
30	9.98848	13	11.57056	498	9.99619	12	12.05535	1485	30	
31	.98861	13	.57554	504	.99632	13	.07020	1537	31	
32	.98874	13	.58058	509	.99645	12	.08557	1592	32	
33	.98887	13	.58567	515	.99657	13	.10149	1652	33	
34	.98900	13	.59082	520	.99670	12	.11801	1716	34	
35	9.98913	13	11.59602	527	9.99683	13	12.13517	1785	35	
36	.98925	13	.60129	533	.99695	13	.15302	1861	36	
37	.98938	13	.60662	539	.99708	12	.17163	1943	37	
38	.98951	13	.61202	545	.99721	13	.19106	2033	38	
39	.98964	13	.61747	552	.99734	12	.21139	2131	39	
40	9.98977	13	11.62300	559	9.99746	13	12.23271	2240	40	
41	.98990	13	.62859	566	.99759	12	.25511	2361	41	
42	.99003	13	.63425	573	.99772	12	.27872	2495	42	
43	.99016	12	.63998	581	.99784	13	.30367	2645	43	
44	.99029	13	.64579	588	.99797	12	.33013	2815	44	
45	9.99042	13	11.65167	595	9.99810	13	12.35828	3009	45	
46	.99055	13	.65762	604	.99823	12	.38837	3231	46	
47	.99068	13	.66366	611	.99835	12	.42068	3489	47	
48	.99081	12	.66978	620	.99848	13	.45557	3791	48	
49	.99093	13	.67598	628	.99861	12	.49349	4152	49	
50	9.99106	13	11.68227	638	9.99873	12	12.53501	4588	50	
51	.99119	13	.68865	646	.99886	13	.58089	5127	51	
52	.99132	13	.69511	656	.99899	12	.63217	5812	52	
53	.99145	12	.70168	666	.99911	12	.69029	6707	53	
54	.99158	13	.70834	675	.99924	13	.75736	7931	54	
55	9.99171	13	11.71509	686	9.99937	12	12.83667	9704	55	
56	.99184	13	.72196	696	.99949	12	.93371	12506	56	
57	.99197	12	.72892	707	.99962	12	13.05877	17621	57	
58	.99209	13	.73600	719	.99974	13	.23499	30116	58	
59	.99222	13	.74319	730	.99987	12	.53615		59	
60	9.99235	13	11.75050	730	10.00000	12	Infinity		60	
	Lg. Vers.	D	Log. Exs.	D	Lg. Vers.	D	Log. Exs.	D		P. P.

	13	13
6	1.3	1.3
7	1.6	1.5
8	1.8	1.7
9	2.0	1.9
10	2.2	2.1
20	4.5	4.3
30	6.7	6.5
40	9.0	8.6
50	11.2	10.8
	12	
6	1.2	
7	1.4	
8	1.6	
9	1.9	
10	2.1	
20	4.1	
30	6.2	
40	8.3	
50	10.4	

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

0°

1°

	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.00000	One	.00000	Infinite	.01745	.99985	.01746	57.2900	60
1	.00029	One	.00029	3437.75	.01774	.99984	.01775	56.3506	59
2	.00058	One	.00058	1718.87	.01803	.99984	.01804	55.4415	58
3	.00087	One	.00087	1145.92	.01832	.99983	.01833	54.5613	57
4	.00116	One	.00116	859.436	.01862	.99983	.01862	53.7086	56
5	.00145	One	.00145	687.549	.01891	.99982	.01891	52.8821	55
6	.00175	One	.00175	572.957	.01920	.99982	.01920	52.0807	54
7	.00204	One	.00204	491.106	.01949	.99981	.01949	51.3032	53
8	.00233	One	.00233	429.718	.01978	.99980	.01978	50.5485	52
9	.00262	One	.00262	381.971	.02007	.99980	.02007	49.8157	51
10	.00291	One	.00291	343.774	.02036	.99979	.02036	49.1039	50
11	.00320	.99999	.00320	312.521	.02065	.99979	.02066	48.4121	49
12	.00349	.99999	.00349	286.478	.02094	.99978	.02095	47.7395	48
13	.00378	.99999	.00378	264.441	.02123	.99977	.02124	47.0853	47
14	.00407	.99999	.00407	245.552	.02152	.99977	.02153	46.4489	46
15	.00436	.99999	.00436	229.182	.02181	.99976	.02182	45.8294	45
16	.00465	.99999	.00465	214.858	.02211	.99976	.02211	45.2261	44
17	.00495	.99999	.00495	202.219	.02240	.99975	.02240	44.6386	43
18	.00524	.99999	.00524	190.984	.02269	.99974	.02269	44.0661	42
19	.00553	.99998	.00553	180.932	.02298	.99974	.02298	43.5081	41
20	.00582	.99998	.00582	171.885	.02327	.99973	.02328	42.9641	40
21	.00611	.99998	.00611	163.700	.02356	.99972	.02357	42.4335	39
22	.00640	.99998	.00640	156.259	.02385	.99972	.02386	41.9158	38
23	.00669	.99998	.00669	149.465	.02414	.99971	.02415	41.4106	37
24	.00698	.99998	.00698	143.237	.02443	.99970	.02444	40.9174	36
25	.00727	.99997	.00727	137.507	.02472	.99969	.02473	40.4358	35
26	.00756	.99997	.00756	132.219	.02501	.99969	.02502	39.9655	34
27	.00785	.99997	.00785	127.321	.02530	.99968	.02531	39.5059	33
28	.00814	.99997	.00815	122.774	.02560	.99967	.02560	39.0568	32
29	.00844	.99996	.00844	118.540	.02589	.99966	.02589	38.6177	31
30	.00873	.99996	.00873	114.589	.02618	.99966	.02619	38.1885	30
31	.00902	.99996	.00902	110.892	.02647	.99965	.02648	37.7686	29
32	.00931	.99996	.00931	107.426	.02676	.99964	.02677	37.3579	28
33	.00960	.99995	.00960	104.171	.02705	.99963	.02706	36.9560	27
34	.00989	.99995	.00989	101.107	.02734	.99963	.02735	36.5627	26
35	.01018	.99995	.01018	98.2179	.02763	.99962	.02764	36.1776	25
36	.01047	.99995	.01047	95.4895	.02792	.99961	.02793	35.8006	24
37	.01076	.99994	.01076	92.9085	.02821	.99960	.02822	35.4313	23
38	.01105	.99994	.01105	90.4633	.02850	.99959	.02851	35.0695	22
39	.01134	.99994	.01135	88.1436	.02879	.99959	.02881	34.7151	21
40	.01164	.99993	.01164	85.9398	.02908	.99958	.02910	34.3678	20
41	.01193	.99993	.01193	83.8435	.02938	.99957	.02939	34.0273	19
42	.01222	.99993	.01222	81.8470	.02967	.99956	.02968	33.6935	18
43	.01251	.99992	.01251	79.9434	.02996	.99955	.02997	33.3662	17
44	.01280	.99992	.01280	78.1263	.03025	.99954	.03026	33.0452	16
45	.01309	.99991	.01309	76.3900	.03054	.99953	.03055	32.7303	15
46	.01338	.99991	.01338	74.7292	.03083	.99952	.03084	32.4213	14
47	.01367	.99991	.01367	73.1390	.03112	.99952	.03114	32.1181	13
48	.01396	.99990	.01396	71.6151	.03141	.99951	.03143	31.8205	12
49	.01425	.99990	.01425	70.1533	.03170	.99950	.03172	31.5284	11
50	.01454	.99989	.01455	68.7501	.03199	.99949	.03201	31.2416	10
51	.01483	.99989	.01484	67.4019	.03228	.99948	.03230	30.9599	9
52	.01513	.99989	.01513	66.1055	.03257	.99947	.03259	30.6833	8
53	.01542	.99988	.01542	64.8580	.03286	.99946	.03288	30.4116	7
54	.01571	.99988	.01571	63.6567	.03316	.99945	.03317	30.1446	6
55	.01600	.99987	.01600	62.4992	.03345	.99944	.03346	29.8823	5
56	.01629	.99987	.01629	61.3829	.03374	.99943	.03376	29.6245	4
57	.01658	.99986	.01658	60.3058	.03403	.99942	.03405	29.3711	3
58	.01687	.99986	.01687	59.2659	.03432	.99941	.03434	29.1220	2
59	.01716	.99985	.01716	58.2612	.03461	.99940	.03463	28.8771	1
60	.01745	.99985	.01746	57.2900	.03490	.99939	.03492	28.6363	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

2°

3°

	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.03490	.99939	.03492	28.6363	.05234	.99863	.05241	19.0811	60
1	.03519	.99938	.03521	28.3994	.05263	.99861	.05270	18.9755	59
2	.03548	.99937	.03550	28.1664	.05292	.99860	.05299	18.8711	58
3	.03577	.99936	.03579	27.9372	.05321	.99858	.05328	18.7678	57
4	.03606	.99935	.03609	27.7117	.05350	.99857	.05357	18.6656	56
5	.03635	.99934	.03638	27.4899	.05379	.99855	.05387	18.5645	55
6	.03664	.99933	.03667	27.2715	.05408	.99854	.05416	18.4645	54
7	.03693	.99932	.03696	27.0566	.05437	.99852	.05445	18.3655	53
8	.03723	.99931	.03725	26.8450	.05466	.99851	.05474	18.2677	52
9	.03752	.99930	.03754	26.6367	.05495	.99849	.05503	18.1708	51
10	.03781	.99929	.03783	26.4316	.05524	.99847	.05533	18.0750	50
11	.03810	.99927	.03812	26.2296	.05553	.99846	.05562	17.9802	49
12	.03839	.99926	.03842	26.0307	.05582	.99844	.05591	17.8863	48
13	.03868	.99925	.03871	25.8348	.05611	.99842	.05620	17.7934	47
14	.03897	.99924	.03900	25.6418	.05640	.99841	.05649	17.7015	46
15	.03926	.99923	.03929	25.4517	.05669	.99839	.05678	17.6106	45
16	.03955	.99922	.03958	25.2644	.05698	.99838	.05708	17.5205	44
17	.03984	.99921	.03987	25.0798	.05727	.99836	.05737	17.4314	43
18	.04013	.99919	.04016	24.8978	.05756	.99834	.05766	17.3432	42
19	.04042	.99918	.04046	24.7185	.05785	.99833	.05795	17.2558	41
20	.04071	.99917	.04075	24.5418	.05814	.99831	.05824	17.1693	40
21	.04100	.99916	.04104	24.3675	.05844	.99829	.05854	17.0837	39
22	.04129	.99915	.04133	24.1957	.05873	.99827	.05883	16.9990	38
23	.04159	.99913	.04162	24.0263	.05902	.99826	.05912	16.9150	37
24	.04188	.99912	.04191	23.8593	.05931	.99824	.05941	16.8319	36
25	.04217	.99911	.04220	23.6945	.05960	.99822	.05970	16.7496	35
26	.04246	.99910	.04250	23.5321	.05989	.99821	.05999	16.6681	34
27	.04275	.99909	.04279	23.3718	.06018	.99819	.06029	16.5874	33
28	.04304	.99907	.04308	23.2137	.06047	.99817	.06058	16.5075	32
29	.04333	.99906	.04337	23.0577	.06076	.99815	.06087	16.4283	31
30	.04362	.99905	.04366	22.9038	.06105	.99813	.06116	16.3499	30
31	.04391	.99904	.04395	22.7519	.06134	.99812	.06145	16.2722	29
32	.04420	.99902	.04424	22.6020	.06163	.99810	.06175	16.1952	28
33	.04449	.99901	.04454	22.4541	.06192	.99808	.06204	16.1190	27
34	.04478	.99900	.04483	22.3081	.06221	.99806	.06233	16.0435	26
35	.04507	.99898	.04512	22.1640	.06250	.99804	.06262	15.9687	25
36	.04536	.99897	.04541	22.0217	.06279	.99803	.06291	15.8945	24
37	.04565	.99896	.04570	21.8813	.06308	.99801	.06321	15.8211	23
38	.04594	.99894	.04599	21.7426	.06337	.99799	.06350	15.7483	22
39	.04623	.99893	.04628	21.6056	.06366	.99797	.06379	15.6762	21
40	.04653	.99892	.04658	21.4704	.06395	.99795	.06408	15.6048	20
41	.04682	.99890	.04687	21.3369	.06424	.99793	.06437	15.5340	19
42	.04711	.99889	.04716	21.2049	.06453	.99792	.06467	15.4638	18
43	.04740	.99888	.04745	21.0747	.06482	.99790	.06496	15.3943	17
44	.04769	.99886	.04774	20.9460	.06511	.99788	.06525	15.3254	16
45	.04798	.99885	.04803	20.8188	.06540	.99786	.06554	15.2571	15
46	.04827	.99883	.04833	20.6932	.06569	.99784	.06584	15.1893	14
47	.04856	.99882	.04862	20.5691	.06598	.99782	.06613	15.1222	13
48	.04885	.99881	.04891	20.4465	.06627	.99780	.06642	15.0557	12
49	.04914	.99879	.04920	20.3253	.06656	.99778	.06671	14.9898	11
50	.04943	.99878	.04949	20.2056	.06685	.99776	.06700	14.9244	10
51	.04972	.99876	.04978	20.0872	.06714	.99774	.06730	14.8596	9
52	.05001	.99875	.05007	19.9702	.06743	.99772	.06759	14.7954	8
53	.05030	.99873	.05037	19.8546	.06773	.99770	.06788	14.7317	7
54	.05059	.99872	.05066	19.7403	.06802	.99768	.06817	14.6685	6
55	.05088	.99870	.05095	19.6273	.06831	.99766	.06847	14.6059	5
56	.05117	.99869	.05124	19.5156	.06860	.99764	.06876	14.5438	4
57	.05146	.99867	.05153	19.4051	.06889	.99762	.06905	14.4823	3
58	.05175	.99866	.05182	19.2959	.06918	.99760	.06934	14.4212	2
59	.05205	.99864	.05212	19.1879	.06947	.99758	.06963	14.3607	1
60	.05234	.99863	.05241	19.0811	.06976	.99756	.06993	14.3007	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

4°					5°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.06976	.99756	.06993	14.3007	.08716	.99619	.08749	11.4301	60
1	.07005	.99754	.07022	14.2411	.08745	.99617	.08778	11.3919	59
2	.07034	.99752	.07051	14.1821	.08774	.99614	.08807	11.3540	58
3	.07063	.99750	.07080	14.1235	.08803	.99612	.08837	11.3163	57
4	.07092	.99748	.07110	14.0655	.08831	.99609	.08866	11.2789	56
5	.07121	.99746	.07139	14.0079	.08860	.99607	.08895	11.2417	55
6	.07150	.99744	.07168	13.9507	.08889	.99604	.08925	11.2048	54
7	.07179	.99742	.07197	13.8940	.08918	.99602	.08954	11.1681	53
8	.07208	.99740	.07227	13.8378	.08947	.99599	.08983	11.1316	52
9	.07237	.99738	.07256	13.7821	.08976	.99596	.09013	11.0954	51
10	.07266	.99736	.07285	13.7267	.09005	.99594	.09042	11.0594	50
11	.07295	.99734	.07314	13.6719	.09034	.99591	.09071	11.0237	49
12	.07324	.99731	.07344	13.6174	.09063	.99588	.09101	10.9882	48
13	.07353	.99729	.07373	13.5634	.09092	.99586	.09130	10.9529	47
14	.07382	.99727	.07402	13.5098	.09121	.99583	.09159	10.9178	46
15	.07411	.99725	.07431	13.4566	.09150	.99580	.09189	10.8829	45
16	.07440	.99723	.07461	13.4039	.09179	.99578	.09218	10.8483	44
17	.07469	.99721	.07490	13.3515	.09208	.99575	.09247	10.8139	43
18	.07498	.99719	.07519	13.2996	.09237	.99572	.09277	10.7797	42
19	.07527	.99716	.07548	13.2480	.09266	.99570	.09306	10.7457	41
20	.07556	.99714	.07578	13.1969	.09295	.99567	.09335	10.7119	40
21	.07585	.99712	.07607	13.1461	.09324	.99564	.09365	10.6783	39
22	.07614	.99710	.07636	13.0958	.09353	.99562	.09394	10.6450	38
23	.07643	.99708	.07665	13.0458	.09382	.99559	.09423	10.6118	37
24	.07672	.99705	.07695	12.9962	.09411	.99556	.09453	10.5789	36
25	.07701	.99703	.07724	12.9469	.09440	.99553	.09482	10.5462	35
26	.07730	.99701	.07753	12.8981	.09469	.99551	.09511	10.5136	34
27	.07759	.99699	.07782	12.8496	.09498	.99548	.09541	10.4813	33
28	.07788	.99696	.07812	12.8014	.09527	.99545	.09570	10.4491	32
29	.07817	.99694	.07841	12.7536	.09556	.99542	.09600	10.4172	31
30	.07846	.99692	.07870	12.7062	.09585	.99540	.09629	10.3854	30
31	.07875	.99689	.07899	12.6591	.09614	.99537	.09658	10.3538	29
32	.07904	.99687	.07929	12.6124	.09642	.99534	.09688	10.3224	28
33	.07933	.99685	.07958	12.5660	.09671	.99531	.09717	10.2913	27
34	.07962	.99683	.07987	12.5199	.09700	.99528	.09746	10.2602	26
35	.07991	.99680	.08017	12.4742	.09729	.99526	.09776	10.2294	25
36	.08020	.99678	.08046	12.4288	.09758	.99523	.09805	10.1988	24
37	.08049	.99676	.08075	12.3838	.09787	.99520	.09834	10.1683	23
38	.08078	.99673	.08104	12.3390	.09816	.99517	.09864	10.1381	22
39	.08107	.99671	.08134	12.2946	.09845	.99514	.09893	10.1080	21
40	.08136	.99668	.08163	12.2505	.09874	.99511	.09923	10.0780	20
41	.08165	.99666	.08192	12.2067	.09903	.99508	.09952	10.0483	19
42	.08194	.99664	.08221	12.1632	.09932	.99506	.09981	10.0187	18
43	.08223	.99661	.08251	12.1201	.09961	.99503	.10011	9.98931	17
44	.08252	.99659	.08280	12.0772	.09990	.99500	.10040	9.96007	16
45	.08281	.99657	.08309	12.0346	.10019	.99497	.10069	9.93101	15
46	.08310	.99654	.08339	11.9923	.10048	.99494	.10099	9.90211	14
47	.08339	.99652	.08368	11.9504	.10077	.99491	.10128	9.87338	13
48	.08368	.99649	.08397	11.9087	.10106	.99488	.10158	9.84482	12
49	.08397	.99647	.08427	11.8673	.10135	.99485	.10187	9.81641	11
50	.08426	.99644	.08456	11.8262	.10164	.99482	.10216	9.78817	10
51	.08455	.99642	.08485	11.7853	.10192	.99479	.10246	9.76009	9
52	.08484	.99639	.08514	11.7448	.10221	.99476	.10275	9.73217	8
53	.08513	.99637	.08544	11.7045	.10250	.99473	.10305	9.70441	7
54	.08542	.99635	.08573	11.6645	.10279	.99470	.10334	9.67680	6
55	.08571	.99632	.08602	11.6248	.10308	.99467	.10363	9.64935	5
56	.08600	.99630	.08632	11.5853	.10337	.99464	.10393	9.62205	4
57	.08629	.99627	.08661	11.5461	.10366	.99461	.10422	9.59490	3
58	.08658	.99625	.08690	11.5072	.10395	.99458	.10452	9.56791	2
59	.08687	.99622	.08720	11.4685	.10424	.99455	.10481	9.54106	1
60	.08716	.99619	.08749	11.4301	.10453	.99452	.10510	9.51436	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

6°

7°

	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.10453	.99452	.10510	9.51436	.12187	.99255	.12278	8.14435	60
1	.10482	.99449	.10540	9.48781	.12216	.99251	.12308	8.12481	59
2	.10511	.99446	.10569	9.46141	.12245	.99248	.12338	8.10536	58
3	.10540	.99443	.10599	9.43515	.12274	.99244	.12367	8.08600	57
4	.10569	.99440	.10628	9.40904	.12302	.99240	.12397	8.06674	56
5	.10597	.99437	.10657	9.38307	.12331	.99237	.12426	8.04756	55
6	.10626	.99434	.10687	9.35724	.12360	.99233	.12456	8.02848	54
7	.10655	.99431	.10716	9.33155	.12389	.99230	.12485	8.00948	53
8	.10684	.99428	.10746	9.30599	.12418	.99226	.12515	7.99058	52
9	.10713	.99424	.10775	9.28058	.12447	.99222	.12544	7.97176	51
10	.10742	.99421	.10805	9.25530	.12476	.99219	.12574	7.95302	50
11	.10771	.99418	.10834	9.23016	.12504	.99215	.12603	7.93438	49
12	.10800	.99415	.10863	9.20516	.12533	.99211	.12633	7.91582	48
13	.10829	.99412	.10893	9.18028	.12562	.99208	.12662	7.89734	47
14	.10858	.99409	.10922	9.15554	.12591	.99204	.12692	7.87895	46
15	.10887	.99406	.10952	9.13093	.12620	.99200	.12722	7.86064	45
16	.10916	.99402	.10981	9.10646	.12649	.99197	.12751	7.84242	44
17	.10945	.99399	.11011	9.08211	.12678	.99193	.12781	7.82428	43
18	.10973	.99396	.11040	9.05789	.12706	.99189	.12810	7.80622	42
19	.11002	.99393	.11070	9.03379	.12735	.99186	.12840	7.78825	41
20	.11031	.99390	.11099	9.00983	.12764	.99182	.12869	7.77035	40
21	.11060	.99386	.11128	8.98598	.12793	.99178	.12899	7.75254	39
22	.11089	.99383	.11158	8.96227	.12822	.99175	.12929	7.73480	38
23	.11118	.99380	.11187	8.93867	.12851	.99171	.12958	7.71715	37
24	.11147	.99377	.11217	8.91520	.12880	.99167	.12988	7.69957	36
25	.11176	.99374	.11246	8.89185	.12908	.99163	.13017	7.68208	35
26	.11205	.99370	.11276	8.86862	.12937	.99160	.13047	7.66466	34
27	.11234	.99367	.11305	8.84551	.12966	.99156	.13076	7.64732	33
28	.11263	.99364	.11335	8.82252	.12995	.99152	.13106	7.63005	32
29	.11291	.99360	.11364	8.79964	.13024	.99148	.13136	7.61287	31
30	.11320	.99357	.11394	8.77689	.13053	.99144	.13165	7.59575	30
31	.11349	.99354	.11423	8.75425	.13081	.99141	.13195	7.57872	29
32	.11378	.99351	.11452	8.73172	.13110	.99137	.13224	7.56176	28
33	.11407	.99347	.11482	8.70931	.13139	.99133	.13254	7.54487	27
34	.11436	.99344	.11511	8.68701	.13168	.99129	.13284	7.52806	26
35	.11465	.99341	.11541	8.66482	.13197	.99125	.13313	7.51132	25
36	.11494	.99337	.11570	8.64275	.13226	.99122	.13343	7.49465	24
37	.11523	.99334	.11600	8.62078	.13254	.99118	.13372	7.47806	23
38	.11552	.99331	.11629	8.59893	.13283	.99114	.13402	7.46154	22
39	.11580	.99327	.11659	8.57718	.13312	.99110	.13432	7.44509	21
40	.11609	.99324	.11688	8.55555	.13341	.99106	.13461	7.42871	20
41	.11638	.99320	.11718	8.53402	.13370	.99102	.13491	7.41240	19
42	.11667	.99317	.11747	8.51259	.13399	.99098	.13521	7.39616	18
43	.11696	.99314	.11777	8.49128	.13427	.99094	.13550	7.37999	17
44	.11725	.99310	.11806	8.47007	.13456	.99091	.13580	7.36389	16
45	.11754	.99307	.11836	8.44896	.13485	.99087	.13609	7.34786	15
46	.11783	.99303	.11865	8.42795	.13514	.99083	.13639	7.33190	14
47	.11812	.99300	.11895	8.40705	.13543	.99079	.13669	7.31600	13
48	.11840	.99297	.11924	8.38625	.13572	.99075	.13698	7.30018	12
49	.11869	.99293	.11954	8.36555	.13600	.99071	.13728	7.28442	11
50	.11898	.99290	.11983	8.34496	.13629	.99067	.13758	7.26873	10
51	.11927	.99286	.12013	8.32446	.13658	.99063	.13787	7.25310	9
52	.11956	.99283	.12042	8.30406	.13687	.99059	.13817	7.23754	8
53	.11985	.99279	.12072	8.28376	.13716	.99055	.13846	7.22204	7
54	.12014	.99276	.12101	8.26355	.13744	.99051	.13876	7.20661	6
55	.12043	.99272	.12131	8.24345	.13773	.99047	.13906	7.19125	5
56	.12071	.99269	.12160	8.22344	.13802	.99043	.13935	7.17594	4
57	.12100	.99265	.12190	8.20352	.13831	.99039	.13965	7.16071	3
58	.12129	.99262	.12219	8.18370	.13860	.99035	.13995	7.14553	2
59	.12158	.99258	.12249	8.16398	.13889	.99031	.14024	7.13042	1
60	.12187	.99255	.12278	8.14435	.13917	.99027	.14054	7.11537	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

8°					9°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.13917	.99027	.14054	7.11537	.15643	.98769	.15838	6.31375	60
1	.13946	.99023	.14084	7.10038	.15672	.98764	.15868	6.30189	59
2	.13975	.99019	.14113	7.08546	.15701	.98760	.15893	6.29007	58
3	.14004	.99015	.14143	7.07059	.15730	.98755	.15928	6.27829	57
4	.14033	.99011	.14173	7.05579	.15758	.98751	.15958	6.26655	56
5	.14061	.99006	.14202	7.04105	.15787	.98746	.15988	6.25486	55
6	.14090	.99002	.14232	7.02637	.15816	.98741	.16017	6.24321	54
7	.14119	.98998	.14262	7.01174	.15845	.98737	.16047	6.23160	53
8	.14148	.98994	.14291	6.99718	.15873	.98732	.16077	6.22003	52
9	.14177	.98990	.14321	6.98268	.15902	.98728	.16107	6.20851	51
10	.14205	.98986	.14351	6.96823	.15931	.98723	.16137	6.19703	50
11	.14234	.98982	.14381	6.95385	.15959	.98718	.16167	6.18559	49
12	.14263	.98978	.14410	6.93952	.15988	.98714	.16196	6.17419	48
13	.14292	.98973	.14440	6.92525	.16017	.98709	.16226	6.16283	47
14	.14320	.98969	.14470	6.91104	.16046	.98704	.16256	6.15151	46
15	.14349	.98965	.14499	6.89688	.16074	.98700	.16286	6.14023	45
16	.14378	.98961	.14529	6.88278	.16103	.98695	.16316	6.12899	44
17	.14407	.98957	.14559	6.86874	.16132	.98690	.16346	6.11779	43
18	.14436	.98953	.14588	6.85475	.16160	.98686	.16376	6.10664	42
19	.14464	.98948	.14618	6.84082	.16189	.98681	.16405	6.09552	41
20	.14493	.98944	.14648	6.82694	.16218	.98676	.16435	6.08444	40
21	.14522	.98940	.14678	6.81312	.16246	.98671	.16465	6.07340	39
22	.14551	.98936	.14707	6.79936	.16275	.98667	.16495	6.06240	38
23	.14580	.98931	.14737	6.78564	.16304	.98662	.16525	6.05143	37
24	.14608	.98927	.14767	6.77199	.16333	.98657	.16555	6.04051	36
25	.14637	.98923	.14796	6.75838	.16361	.98652	.16585	6.02962	35
26	.14666	.98919	.14826	6.74483	.16390	.98648	.16615	6.01878	34
27	.14695	.98914	.14856	6.73133	.16419	.98643	.16645	6.00797	33
28	.14723	.98910	.14886	6.71789	.16447	.98638	.16674	5.99720	32
29	.14752	.98906	.14915	6.70450	.16476	.98633	.16704	5.98646	31
30	.14781	.98902	.14945	6.69116	.16505	.98629	.16734	5.97576	30
31	.14810	.98897	.14975	6.67787	.16533	.98624	.16764	5.96510	29
32	.14838	.98893	.15005	6.66463	.16562	.98619	.16794	5.95448	28
33	.14867	.98889	.15034	6.65144	.16591	.98614	.16824	5.94390	27
34	.14896	.98884	.15064	6.63831	.16620	.98609	.16854	5.93335	26
35	.14925	.98880	.15094	6.62523	.16648	.98604	.16884	5.92283	25
36	.14954	.98876	.15124	6.61219	.16677	.98600	.16914	5.91236	24
37	.14982	.98871	.15153	6.59921	.16706	.98595	.16944	5.90191	23
38	.15011	.98867	.15183	6.58627	.16734	.98590	.16974	5.89151	22
39	.15040	.98863	.15213	6.57339	.16763	.98585	.17004	5.88114	21
40	.15069	.98858	.15243	6.56055	.16792	.98580	.17033	5.87080	20
41	.15097	.98854	.15272	6.54777	.16820	.98575	.17063	5.86051	19
42	.15126	.98849	.15302	6.53503	.16849	.98570	.17093	5.85024	18
43	.15155	.98845	.15332	6.52234	.16878	.98565	.17123	5.84001	17
44	.15184	.98841	.15362	6.50970	.16906	.98561	.17153	5.82982	16
45	.15212	.98836	.15391	6.49710	.16935	.98556	.17183	5.81966	15
46	.15241	.98832	.15421	6.48456	.16964	.98551	.17213	5.80953	14
47	.15270	.98827	.15451	6.47206	.16992	.98546	.17243	5.79944	13
48	.15299	.98823	.15481	6.45961	.17021	.98541	.17273	5.78938	12
49	.15327	.98818	.15511	6.44720	.17050	.98536	.17303	5.77936	11
50	.15356	.98814	.15540	6.43484	.17078	.98531	.17333	5.76937	10
51	.15385	.98809	.15570	6.42253	.17107	.98526	.17363	5.75941	9
52	.15414	.98805	.15600	6.41026	.17136	.98521	.17393	5.74949	8
53	.15442	.98800	.15630	6.39804	.17164	.98516	.17423	5.73960	7
54	.15471	.98796	.15660	6.38587	.17193	.98511	.17453	5.72974	6
55	.15500	.98791	.15689	6.37374	.17222	.98506	.17483	5.71992	5
56	.15529	.98787	.15719	6.36165	.17250	.98501	.17513	5.71013	4
57	.15557	.98782	.15749	6.34961	.17279	.98496	.17543	5.70037	3
58	.15586	.98778	.15779	6.33761	.17308	.98491	.17573	5.69064	2
59	.15615	.98773	.15809	6.32566	.17336	.98486	.17603	5.68094	1
60	.15643	.98769	.15838	6.31375	.17365	.98481	.17633	5.67128	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

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	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.17365	.98481	.17633	5.67128	.19081	.98163	.19438	5.14455	60
1	.17393	.98476	.17663	5.66165	.19109	.98157	.19468	5.13658	59
2	.17422	.98471	.17693	5.65205	.19138	.98152	.19498	5.12862	58
3	.17451	.98466	.17723	5.64248	.19167	.98146	.19529	5.12069	57
4	.17479	.98461	.17753	5.63295	.19195	.98140	.19559	5.11279	56
5	.17508	.98455	.17783	5.62344	.19224	.98135	.19589	5.10490	55
6	.17537	.98450	.17813	5.61397	.19252	.98129	.19619	5.09704	54
7	.17565	.98445	.17843	5.60452	.19281	.98124	.19649	5.08921	53
8	.17594	.98440	.17873	5.59511	.19309	.98118	.19680	5.08139	52
9	.17623	.98435	.17903	5.58573	.19338	.98112	.19710	5.07360	51
10	.17651	.98430	.17933	5.57638	.19366	.98107	.19740	5.06584	50
11	.17680	.98425	.17963	5.56706	.19395	.98101	.19770	5.05809	49
12	.17708	.98420	.17993	5.55777	.19423	.98096	.19801	5.05037	48
13	.17737	.98414	.18023	5.54851	.19452	.98090	.19831	5.04267	47
14	.17766	.98409	.18053	5.53927	.19481	.98084	.19861	5.03499	46
15	.17794	.98404	.18083	5.53007	.19509	.98079	.19891	5.02734	45
16	.17823	.98399	.18113	5.52090	.19538	.98073	.19921	5.01971	44
17	.17852	.98394	.18143	5.51176	.19566	.98067	.19952	5.01210	43
18	.17880	.98389	.18173	5.50264	.19595	.98061	.19982	5.00451	42
19	.17909	.98383	.18203	5.49356	.19623	.98056	.20012	4.99695	41
20	.17937	.98378	.18233	5.48451	.19652	.98050	.20042	4.98940	40
21	.17966	.98373	.18263	5.47548	.19680	.98044	.20073	4.98188	39
22	.17995	.98368	.18293	5.46648	.19709	.98039	.20103	4.97438	38
23	.18023	.98362	.18323	5.45751	.19737	.98033	.20133	4.96690	37
24	.18052	.98357	.18353	5.44857	.19766	.98027	.20164	4.95945	36
25	.18081	.98352	.18384	5.43966	.19794	.98021	.20194	4.95201	35
26	.18109	.98347	.18414	5.43077	.19823	.98016	.20224	4.94460	34
27	.18138	.98341	.18444	5.42192	.19851	.98010	.20254	4.93721	33
28	.18166	.98336	.18474	5.41309	.19880	.98004	.20285	4.92984	32
29	.18195	.98331	.18504	5.40429	.19908	.97998	.20315	4.92249	31
30	.18224	.98325	.18534	5.39552	.19937	.97992	.20345	4.91516	30
31	.18252	.98320	.18564	5.38677	.19965	.97987	.20376	4.90785	29
32	.18281	.98315	.18594	5.37805	.19994	.97981	.20406	4.90056	28
33	.18309	.98310	.18624	5.36936	.20022	.97975	.20436	4.89330	27
34	.18338	.98304	.18654	5.36070	.20051	.97969	.20466	4.88605	26
35	.18367	.98299	.18684	5.35206	.20079	.97963	.20497	4.87882	25
36	.18395	.98294	.18714	5.34345	.20108	.97958	.20527	4.87162	24
37	.18424	.98288	.18745	5.33487	.20136	.97952	.20557	4.86444	23
38	.18452	.98283	.18775	5.32631	.20165	.97946	.20588	4.85727	22
39	.18481	.98277	.18805	5.31778	.20193	.97940	.20618	4.85013	21
40	.18509	.98272	.18835	5.30928	.20222	.97934	.20648	4.84300	20
41	.18538	.98267	.18865	5.30080	.20250	.97928	.20679	4.83590	19
42	.18567	.98261	.18895	5.29235	.20279	.97922	.20709	4.82882	18
43	.18595	.98256	.18925	5.28393	.20307	.97916	.20739	4.82175	17
44	.18624	.98250	.18955	5.27553	.20336	.97910	.20770	4.81471	16
45	.18652	.98245	.18986	5.26715	.20364	.97905	.20800	4.80769	15
46	.18681	.98240	.19016	5.25880	.20393	.97899	.20830	4.80068	14
47	.18710	.98234	.19046	5.25048	.20421	.97893	.20861	4.79370	13
48	.18738	.98229	.19076	5.24218	.20450	.97887	.20891	4.78673	12
49	.18767	.98223	.19106	5.23391	.20478	.97881	.20921	4.77978	11
50	.18795	.98218	.19136	5.22566	.20507	.97875	.20952	4.77286	10
51	.18824	.98212	.19166	5.21744	.20535	.97869	.20982	4.76595	9
52	.18852	.98207	.19197	5.20925	.20563	.97863	.21013	4.75906	8
53	.18881	.98201	.19227	5.20107	.20592	.97857	.21043	4.75219	7
54	.18910	.98196	.19257	5.19293	.20620	.97851	.21073	4.74534	6
55	.18938	.98190	.19287	5.18480	.20649	.97845	.21104	4.73851	5
56	.18967	.98185	.19317	5.17671	.20677	.97839	.21134	4.73170	4
57	.18995	.98179	.19347	5.16863	.20706	.97833	.21164	4.72490	3
58	.19024	.98174	.19378	5.16058	.20734	.97827	.21195	4.71813	2
59	.19052	.98168	.19408	5.15256	.20763	.97821	.21225	4.71137	1
60	.19081	.98163	.19438	5.14455	.20791	.97815	.21256	4.70463	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

12°					13°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.20791	.97815	.21256	4.70463	.22495	.97437	.23087	4.33148	60
1	.20820	.97809	.21286	4.69791	.22523	.97430	.23117	4.32573	59
2	.20848	.97803	.21316	4.69121	.22552	.97424	.23148	4.32001	58
3	.20877	.97797	.21347	4.68452	.22580	.97417	.23179	4.31430	57
4	.20905	.97791	.21377	4.67786	.22608	.97411	.23209	4.30860	56
5	.20933	.97784	.21408	4.67121	.22637	.97404	.23240	4.30291	55
6	.20962	.97778	.21438	4.66458	.22665	.97398	.23271	4.29724	54
7	.20990	.97772	.21469	4.65797	.22693	.97391	.23301	4.29159	53
8	.21019	.97766	.21499	4.65138	.22722	.97384	.23332	4.28595	52
9	.21047	.97760	.21529	4.64480	.22750	.97378	.23363	4.28032	51
10	.21076	.97754	.21560	4.63825	.22778	.97371	.23393	4.27471	50
11	.21104	.97748	.21590	4.63171	.22807	.97365	.23424	4.26911	49
12	.21132	.97742	.21621	4.62518	.22835	.97358	.23455	4.26352	48
13	.21161	.97735	.21651	4.61868	.22863	.97351	.23485	4.25795	47
14	.21189	.97729	.21682	4.61219	.22892	.97345	.23516	4.25239	46
15	.21218	.97723	.21712	4.60572	.22920	.97338	.23547	4.24685	45
16	.21246	.97717	.21743	4.59927	.22948	.97331	.23578	4.24132	44
17	.21275	.97711	.21773	4.59283	.22977	.97325	.23608	4.23580	43
18	.21303	.97705	.21804	4.58641	.23005	.97318	.23639	4.23030	42
19	.21331	.97698	.21834	4.58001	.23033	.97311	.23670	4.22481	41
20	.21360	.97692	.21864	4.57363	.23062	.97304	.23700	4.21933	40
21	.21388	.97686	.21895	4.56726	.23090	.97298	.23731	4.21387	39
22	.21417	.97680	.21925	4.56091	.23118	.97291	.23762	4.20842	38
23	.21445	.97673	.21956	4.55458	.23146	.97284	.23793	4.20298	37
24	.21474	.97667	.21986	4.54826	.23175	.97278	.23823	4.19756	36
25	.21502	.97661	.22017	4.54196	.23203	.97271	.23854	4.19215	35
26	.21530	.97655	.22047	4.53568	.23231	.97264	.23885	4.18675	34
27	.21559	.97648	.22078	4.52941	.23260	.97257	.23916	4.18137	33
28	.21587	.97642	.22108	4.52316	.23288	.97251	.23946	4.17600	32
29	.21616	.97636	.22139	4.51693	.23316	.97244	.23977	4.17064	31
30	.21644	.97630	.22169	4.51071	.23345	.97237	.24008	4.16530	30
31	.21672	.97623	.22200	4.50451	.23373	.97230	.24039	4.15997	29
32	.21701	.97617	.22231	4.49832	.23401	.97223	.24069	4.15465	28
33	.21729	.97611	.22261	4.49215	.23429	.97217	.24100	4.14934	27
34	.21758	.97604	.22292	4.48600	.23458	.97210	.24131	4.14405	26
35	.21786	.97598	.22322	4.47986	.23486	.97203	.24162	4.13877	25
36	.21814	.97592	.22353	4.47374	.23514	.97196	.24193	4.13350	24
37	.21843	.97585	.22383	4.46764	.23542	.97189	.24223	4.12825	23
38	.21871	.97579	.22414	4.46155	.23571	.97182	.24254	4.12301	22
39	.21899	.97573	.22444	4.45548	.23599	.97176	.24285	4.11778	21
40	.21928	.97566	.22475	4.44942	.23627	.97169	.24316	4.11256	20
41	.21956	.97560	.22505	4.44338	.23656	.97162	.24347	4.10736	19
42	.21985	.97553	.22536	4.43735	.23684	.97155	.24377	4.10216	18
43	.22013	.97547	.22567	4.43134	.23712	.97148	.24408	4.09699	17
44	.22041	.97541	.22597	4.42534	.23740	.97141	.24439	4.09182	16
45	.22070	.97534	.22628	4.41936	.23769	.97134	.24470	4.08666	15
46	.22098	.97528	.22658	4.41340	.23797	.97127	.24501	4.08152	14
47	.22126	.97521	.22689	4.40745	.23825	.97120	.24532	4.07639	13
48	.22155	.97515	.22719	4.40152	.23853	.97113	.24562	4.07127	12
49	.22183	.97508	.22750	4.39560	.23882	.97106	.24593	4.06616	11
50	.22212	.97502	.22781	4.38969	.23910	.97100	.24624	4.06107	10
51	.22240	.97496	.22811	4.38381	.23938	.97093	.24655	4.05599	9
52	.22268	.97489	.22842	4.37793	.23966	.97086	.24686	4.05092	8
53	.22297	.97483	.22872	4.37207	.23995	.97079	.24717	4.04586	7
54	.22325	.97476	.22903	4.36623	.24023	.97072	.24747	4.04081	6
55	.22353	.97470	.22934	4.36040	.24051	.97065	.24778	4.03578	5
56	.22382	.97463	.22964	4.35459	.24079	.97058	.24809	4.03076	4
57	.22410	.97457	.22995	4.34879	.24108	.97051	.24840	4.02574	3
58	.22438	.97450	.23026	4.34300	.24136	.97044	.24871	4.02074	2
59	.22467	.97444	.23056	4.33723	.24164	.97037	.24902	4.01576	1
60	.22495	.97437	.23087	4.33148	.24192	.97030	.24933	4.01078	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

14°					15°				
	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.24192	.97030	.24933	4.01078	.25882	.96593	.26795	3.73205	60
1	.24220	.97023	.24964	4.00582	.25910	.96585	.26826	3.72771	59
2	.24249	.97015	.24995	4.00086	.25938	.96578	.26857	3.72338	58
3	.24277	.97008	.25026	3.99592	.25966	.96570	.26888	3.71907	57
4	.24305	.97001	.25056	3.99099	.25994	.96562	.26920	3.71476	56
5	.24333	.96994	.25087	3.98607	.26022	.96555	.26951	3.71046	55
6	.24362	.96987	.25118	3.98117	.26050	.96547	.26982	3.70616	54
7	.24390	.96980	.25149	3.97627	.26079	.96540	.27013	3.70188	53
8	.24418	.96973	.25180	3.97139	.26107	.96532	.27044	3.69761	52
9	.24446	.96966	.25211	3.96651	.26135	.96524	.27076	3.69335	51
10	.24474	.96959	.25242	3.96165	.26163	.96517	.27107	3.68909	50
11	.24503	.96952	.25273	3.95680	.26191	.96509	.27138	3.68485	49
12	.24531	.96945	.25304	3.95196	.26219	.96502	.27169	3.68061	48
13	.24559	.96937	.25335	3.94713	.26247	.96494	.27201	3.67638	47
14	.24587	.96930	.25366	3.94232	.26275	.96486	.27232	3.67217	46
15	.24615	.96923	.25397	3.93751	.26303	.96479	.27263	3.66796	45
16	.24644	.96916	.25428	3.93271	.26331	.96471	.27294	3.66376	44
17	.24672	.96909	.25459	3.92793	.26359	.96463	.27326	3.65957	43
18	.24700	.96902	.25490	3.92316	.26387	.96456	.27357	3.65538	42
19	.24728	.96894	.25521	3.91839	.26415	.96448	.27388	3.65121	41
20	.24756	.96887	.25552	3.91364	.26443	.96440	.27419	3.64705	40
21	.24784	.96880	.25583	3.90890	.26471	.96433	.27451	3.64289	39
22	.24813	.96873	.25614	3.90417	.26500	.96425	.27482	3.63874	38
23	.24841	.96866	.25645	3.89945	.26528	.96417	.27513	3.63461	37
24	.24869	.96858	.25676	3.89474	.26556	.96410	.27545	3.63048	36
25	.24897	.96851	.25707	3.89004	.26584	.96402	.27576	3.62636	35
26	.24925	.96844	.25738	3.88536	.26612	.96394	.27607	3.62224	34
27	.24954	.96837	.25769	3.88068	.26640	.96386	.27638	3.61814	33
28	.24982	.96829	.25800	3.87601	.26668	.96379	.27670	3.61405	32
29	.25010	.96822	.25831	3.87136	.26696	.96371	.27701	3.60996	31
30	.25038	.96815	.25862	3.86671	.26724	.96363	.27732	3.60588	30
31	.25066	.96807	.25893	3.86208	.26752	.96355	.27764	3.60181	29
32	.25094	.96800	.25924	3.85745	.26780	.96347	.27795	3.59775	28
33	.25122	.96793	.25955	3.85284	.26808	.96340	.27826	3.59370	27
34	.25151	.96786	.25986	3.84824	.26836	.96332	.27858	3.58966	26
35	.25179	.96778	.26017	3.84364	.26864	.96324	.27889	3.58562	25
36	.25207	.96771	.26048	3.83906	.26892	.96316	.27921	3.58160	24
37	.25235	.96764	.26079	3.83449	.26920	.96308	.27952	3.57758	23
38	.25263	.96756	.26110	3.82992	.26948	.96301	.27983	3.57357	22
39	.25291	.96749	.26141	3.82537	.26976	.96293	.28015	3.56957	21
40	.25320	.96742	.26172	3.82083	.27004	.96285	.28046	3.56557	20
41	.25348	.96734	.26203	3.81630	.27032	.96277	.28077	3.56159	19
42	.25376	.96727	.26235	3.81177	.27060	.96269	.28109	3.55761	18
43	.25404	.96719	.26266	3.80726	.27088	.96261	.28140	3.55364	17
44	.25432	.96712	.26297	3.80276	.27116	.96253	.28172	3.54968	16
45	.25460	.96705	.26328	3.79827	.27144	.96246	.28203	3.54573	15
46	.25488	.96697	.26359	3.79378	.27172	.96238	.28234	3.54179	14
47	.25516	.96690	.26390	3.78931	.27200	.96230	.28266	3.53785	13
48	.25545	.96682	.26421	3.78485	.27228	.96222	.28297	3.53393	12
49	.25573	.96675	.26452	3.78040	.27256	.96214	.28329	3.53001	11
50	.25601	.96667	.26483	3.77595	.27284	.96206	.28360	3.52609	10
51	.25629	.96660	.26515	3.77152	.27312	.96198	.28391	3.52219	9
52	.25657	.96653	.26546	3.76709	.27340	.96190	.28423	3.51829	8
53	.25685	.96645	.26577	3.76268	.27368	.96182	.28454	3.51441	7
54	.25713	.96638	.26608	3.75828	.27396	.96174	.28486	3.51053	6
55	.25741	.96630	.26639	3.75388	.27424	.96166	.28517	3.50666	5
56	.25769	.96623	.26670	3.74950	.27452	.96158	.28549	3.50279	4
57	.25798	.96615	.26701	3.74512	.27480	.96150	.28580	3.49894	3
58	.25826	.96608	.26733	3.74075	.27508	.96142	.28612	3.49509	2
59	.25854	.96600	.26764	3.73640	.27536	.96134	.28643	3.49125	1
60	.25882	.96593	.26795	3.73205	.27564	.96126	.28675	3.48741	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

16°					17°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.27564	.96126	.28675	3.48741	.29237	.95630	.30573	3.27085	60
1	.27592	.96118	.28706	3.48359	.29265	.95622	.30605	3.26745	59
2	.27620	.96110	.28738	3.47977	.29293	.95613	.30637	3.26406	58
3	.27648	.96102	.28769	3.47596	.29321	.95605	.30669	3.26067	57
4	.27676	.96094	.28800	3.47216	.29348	.95596	.30700	3.25729	56
5	.27704	.96086	.28832	3.46837	.29376	.95588	.30732	3.25392	55
6	.27731	.96078	.28864	3.46458	.29404	.95579	.30764	3.25055	54
7	.27759	.96070	.28895	3.46080	.29432	.95571	.30796	3.24719	53
8	.27787	.96062	.28927	3.45703	.29460	.95562	.30828	3.24383	52
9	.27815	.96054	.28958	3.45327	.29487	.95554	.30860	3.24049	51
10	.27843	.96046	.28990	3.44951	.29515	.95545	.30891	3.23714	50
11	.27871	.96037	.29021	3.44576	.29543	.95536	.30923	3.23381	49
12	.27899	.96029	.29053	3.44202	.29571	.95528	.30955	3.23048	48
13	.27927	.96021	.29084	3.43829	.29599	.95519	.30987	3.22715	47
14	.27955	.96013	.29116	3.43456	.29626	.95511	.31019	3.22384	46
15	.27983	.96005	.29147	3.43084	.29654	.95502	.31051	3.22053	45
16	.28011	.95997	.29179	3.42713	.29682	.95493	.31083	3.21722	44
17	.28039	.95989	.29210	3.42343	.29710	.95485	.31115	3.21392	43
18	.28067	.95981	.29242	3.41973	.29737	.95476	.31147	3.21063	42
19	.28095	.95972	.29274	3.41604	.29765	.95467	.31178	3.20734	41
20	.28123	.95964	.29305	3.41236	.29793	.95459	.31210	3.20406	40
21	.28150	.95956	.29337	3.40869	.29821	.95450	.31242	3.20079	39
22	.28178	.95948	.29368	3.40502	.29849	.95441	.31274	3.19752	38
23	.28206	.95940	.29400	3.40136	.29876	.95433	.31306	3.19426	37
24	.28234	.95931	.29432	3.39771	.29904	.95424	.31338	3.19100	36
25	.28262	.95923	.29463	3.39406	.29932	.95415	.31370	3.18775	35
26	.28290	.95915	.29495	3.39042	.29960	.95407	.31402	3.18451	34
27	.28318	.95907	.29526	3.38679	.29987	.95398	.31434	3.18127	33
28	.28346	.95898	.29558	3.38317	.30015	.95389	.31466	3.17804	32
29	.28374	.95890	.29590	3.37955	.30043	.95380	.31498	3.17481	31
30	.28402	.95882	.29621	3.37594	.30071	.95372	.31530	3.17159	30
31	.28429	.95874	.29653	3.37234	.30098	.95363	.31562	3.16838	29
32	.28457	.95865	.29685	3.36875	.30126	.95354	.31594	3.16517	28
33	.28485	.95857	.29716	3.36516	.30154	.95345	.31626	3.16197	27
34	.28513	.95849	.29748	3.36158	.30182	.95337	.31658	3.15877	26
35	.28541	.95841	.29780	3.35800	.30209	.95328	.31690	3.15558	25
36	.28569	.95832	.29811	3.35443	.30237	.95319	.31722	3.15240	24
37	.28597	.95824	.29843	3.35087	.30265	.95310	.31754	3.14922	23
38	.28625	.95816	.29875	3.34732	.30292	.95301	.31786	3.14605	22
39	.28652	.95807	.29906	3.34377	.30320	.95293	.31818	3.14288	21
40	.28680	.95799	.29938	3.34023	.30348	.95284	.31850	3.13972	20
41	.28708	.95791	.29970	3.33670	.30376	.95275	.31882	3.13656	19
42	.28736	.95782	.30001	3.33317	.30403	.95266	.31914	3.13341	18
43	.28764	.95774	.30033	3.32965	.30431	.95257	.31946	3.13027	17
44	.28792	.95766	.30065	3.32614	.30459	.95248	.31978	3.12713	16
45	.28820	.95757	.30097	3.32264	.30486	.95240	.32010	3.12400	15
46	.28847	.95749	.30128	3.31914	.30514	.95231	.32042	3.12087	14
47	.28875	.95740	.30160	3.31565	.30542	.95222	.32074	3.11775	13
48	.28903	.95732	.30192	3.31216	.30570	.95213	.32106	3.11464	12
49	.28931	.95724	.30224	3.30868	.30597	.95204	.32139	3.11153	11
50	.28959	.95715	.30255	3.30521	.30625	.95195	.32171	3.10842	10
51	.28987	.95707	.30287	3.30174	.30653	.95186	.32203	3.10532	9
52	.29015	.95698	.30319	3.29829	.30680	.95177	.32235	3.10223	8
53	.29042	.95690	.30351	3.29483	.30708	.95168	.32267	3.09914	7
54	.29070	.95681	.30382	3.29139	.30736	.95159	.32299	3.09606	6
55	.29098	.95673	.30414	3.28795	.30763	.95150	.32331	3.09298	5
56	.29126	.95664	.30446	3.28452	.30791	.95142	.32363	3.08991	4
57	.29154	.95656	.30478	3.28109	.30819	.95133	.32396	3.08685	3
58	.29182	.95647	.30509	3.27767	.30846	.95124	.32428	3.08379	2
59	.29209	.95639	.30541	3.27426	.30874	.95115	.32460	3.08073	1
60	.29237	.95630	.30573	3.27085	.30902	.95106	.32492	3.07768	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—STANDARD SINES, COSINES, AND TANGENTS.

18°					19°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.30902	.95106	.32492	3.07768	.32557	.94552	.34433	2.90421	60
1	.30929	.95097	.32524	3.07464	.32584	.94542	.34465	2.90147	59
2	.30957	.95088	.32556	3.07160	.32612	.94533	.34498	2.89873	58
3	.30985	.95079	.32588	3.06857	.32639	.94523	.34530	2.89600	57
4	.31012	.95070	.32621	3.06554	.32667	.94514	.34563	2.89327	56
5	.31040	.95061	.32653	3.06252	.32694	.94504	.34596	2.89055	55
6	.31068	.95052	.32685	3.05950	.32722	.94495	.34628	2.88783	54
7	.31095	.95043	.32717	3.05648	.32749	.94485	.34661	2.88511	53
8	.31123	.95033	.32749	3.05349	.32777	.94476	.34693	2.88240	52
9	.31151	.95024	.32782	3.05049	.32804	.94466	.34726	2.87970	51
10	.31178	.95015	.32814	3.04749	.32832	.94457	.34758	2.87700	50
11	.31206	.95006	.32846	3.04450	.32859	.94447	.34791	2.87430	49
12	.31233	.94997	.32878	3.04152	.32887	.94438	.34824	2.87161	48
13	.31261	.94988	.32911	3.03854	.32914	.94428	.34856	2.86892	47
14	.31289	.94979	.32943	3.03556	.32942	.94418	.34889	2.86624	46
15	.31316	.94970	.32975	3.03260	.32969	.94409	.34922	2.86356	45
16	.31344	.94961	.33007	3.02963	.32997	.94399	.34954	2.86089	44
17	.31372	.94952	.33040	3.02667	.33024	.94390	.34987	2.85822	43
18	.31399	.94943	.33072	3.02372	.33051	.94380	.35020	2.85555	42
19	.31427	.94933	.33104	3.02077	.33079	.94370	.35052	2.85289	41
20	.31454	.94924	.33136	3.01783	.33106	.94361	.35085	2.85023	40
21	.31482	.94915	.33169	3.01489	.33134	.94351	.35118	2.84758	39
22	.31510	.94906	.33201	3.01196	.33161	.94342	.35150	2.84494	38
23	.31537	.94897	.33233	3.00903	.33189	.94332	.35183	2.84229	37
24	.31565	.94888	.33266	3.00611	.33216	.94322	.35216	2.83965	36
25	.31593	.94878	.33298	3.00319	.33244	.94313	.35248	2.83702	35
26	.31620	.94869	.33330	3.00028	.33271	.94303	.35281	2.83439	34
27	.31648	.94860	.33363	2.99738	.33298	.94293	.35314	2.83176	33
28	.31675	.94851	.33395	2.99447	.33326	.94284	.35346	2.82914	32
29	.31703	.94842	.33427	2.99158	.33353	.94274	.35379	2.82653	31
30	.31730	.94832	.33460	2.98868	.33381	.94264	.35412	2.82391	30
31	.31758	.94823	.33492	2.98580	.33408	.94254	.35445	2.82130	29
32	.31786	.94814	.33524	2.98292	.33436	.94245	.35477	2.81870	28
33	.31813	.94805	.33557	2.98004	.33463	.94235	.35510	2.81610	27
34	.31841	.94795	.33589	2.97717	.33490	.94225	.35543	2.81350	26
35	.31868	.94786	.33621	2.97430	.33518	.94215	.35576	2.81091	25
36	.31896	.94777	.33654	2.97144	.33545	.94206	.35608	2.80833	24
37	.31923	.94768	.33686	2.96858	.33573	.94196	.35641	2.80574	23
38	.31951	.94758	.33718	2.96573	.33600	.94186	.35674	2.80316	22
39	.31979	.94749	.33751	2.96288	.33627	.94176	.35707	2.80059	21
40	.32006	.94740	.33783	2.96004	.33655	.94167	.35740	2.79802	20
41	.32034	.94730	.33816	2.95721	.33682	.94157	.35772	2.79545	19
42	.32061	.94721	.33848	2.95437	.33710	.94147	.35805	2.79289	18
43	.32089	.94712	.33881	2.95155	.33737	.94137	.35838	2.79033	17
44	.32116	.94702	.33913	2.94872	.33764	.94127	.35871	2.78778	16
45	.32144	.94693	.33945	2.94591	.33792	.94118	.35904	2.78523	15
46	.32171	.94684	.33978	2.94309	.33819	.94108	.35937	2.78269	14
47	.32199	.94674	.34010	2.94028	.33846	.94098	.35969	2.78014	13
48	.32227	.94665	.34043	2.93748	.33874	.94088	.36002	2.77761	12
49	.32254	.94656	.34075	2.93468	.33901	.94078	.36035	2.77507	11
50	.32282	.94646	.34108	2.93189	.33929	.94068	.36068	2.77254	10
51	.32309	.94637	.34140	2.92910	.33956	.94058	.36101	2.77002	9
52	.32337	.94627	.34173	2.92632	.33983	.94049	.36134	2.76750	8
53	.32364	.94618	.34205	2.92354	.34011	.94039	.36167	2.76498	7
54	.32392	.94609	.34238	2.92076	.34038	.94029	.36199	2.76247	6
55	.32419	.94599	.34270	2.91799	.34065	.94019	.36232	2.75996	5
56	.32447	.94590	.34303	2.91523	.34093	.94009	.36265	2.75746	4
57	.32474	.94580	.34335	2.91246	.34120	.93999	.36298	2.75496	3
58	.32502	.94571	.34368	2.90971	.34147	.93989	.36331	2.75246	2
59	.32529	.94561	.34400	2.90696	.34175	.93979	.36364	2.74997	1
60	.32557	.94552	.34433	2.90421	.34202	.93969	.36397	2.74748	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

20°					21°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.34202	.93969	.36397	2.74748	.35837	.93358	.38386	2.60509	60
1	.34229	.93959	.36430	2.74499	.35864	.93348	.38420	2.60283	59
2	.34257	.93949	.36463	2.74251	.35891	.93337	.38453	2.60057	58
3	.34284	.93939	.36496	2.74004	.35918	.93327	.38487	2.59831	57
4	.34311	.93929	.36529	2.73756	.35945	.93316	.38520	2.59606	56
5	.34339	.93919	.36562	2.73509	.35973	.93306	.38553	2.59381	55
6	.34366	.93909	.36595	2.73263	.36000	.93295	.38587	2.59156	54
7	.34393	.93899	.36628	2.73017	.36027	.93285	.38620	2.58932	53
8	.34421	.93889	.36661	2.72771	.36054	.93274	.38654	2.58708	52
9	.34448	.93879	.36694	2.72526	.36081	.93264	.38687	2.58484	51
10	.34475	.93869	.36727	2.72281	.36108	.93253	.38721	2.58261	50
11	.34503	.93859	.36760	2.72036	.36135	.93243	.38754	2.58038	49
12	.34530	.93849	.36793	2.71792	.36162	.93232	.38787	2.57815	48
13	.34557	.93839	.36826	2.71548	.36190	.93222	.38821	2.57593	47
14	.34584	.93829	.36859	2.71305	.36217	.93211	.38854	2.57371	46
15	.34612	.93819	.36892	2.71062	.36244	.93201	.38888	2.57150	45
16	.34639	.93809	.36925	2.70819	.36271	.93190	.38921	2.56928	44
17	.34666	.93799	.36958	2.70577	.36298	.93180	.38955	2.56707	43
18	.34694	.93789	.36991	2.70335	.36325	.93169	.38988	2.56487	42
19	.34721	.93779	.37024	2.70094	.36352	.93159	.39022	2.56266	41
20	.34748	.93769	.37057	2.69853	.36379	.93148	.39055	2.56046	40
21	.34775	.93759	.37090	2.69612	.36406	.93137	.39089	2.55827	39
22	.34803	.93748	.37123	2.69371	.36434	.93127	.39122	2.55608	38
23	.34830	.93738	.37157	2.69131	.36461	.93116	.39156	2.55389	37
24	.34857	.93728	.37190	2.68892	.36488	.93106	.39190	2.55170	36
25	.34884	.93718	.37223	2.68653	.36515	.93095	.39223	2.54952	35
26	.34912	.93708	.37256	2.68414	.36542	.93084	.39257	2.54734	34
27	.34939	.93698	.37289	2.68175	.36569	.93074	.39290	2.54516	33
28	.34966	.93688	.37322	2.67937	.36596	.93063	.39324	2.54299	32
29	.34993	.93677	.37355	2.67700	.36623	.93052	.39357	2.54082	31
30	.35021	.93667	.37388	2.67462	.36650	.93042	.39391	2.53865	30
31	.35048	.93657	.37422	2.67225	.36677	.93031	.39425	2.53648	29
32	.35075	.93647	.37455	2.66989	.36704	.93020	.39458	2.53432	28
33	.35102	.93637	.37488	2.66752	.36731	.93010	.39492	2.53217	27
34	.35130	.93626	.37521	2.66516	.36758	.92999	.39526	2.53001	26
35	.35157	.93616	.37554	2.66281	.36785	.92988	.39559	2.52786	25
36	.35184	.93606	.37588	2.66046	.36812	.92978	.39593	2.52571	24
37	.35211	.93596	.37621	2.65811	.36839	.92967	.39626	2.52357	23
38	.35239	.93585	.37654	2.65576	.36867	.92956	.39660	2.52142	22
39	.35266	.93575	.37687	2.65342	.36894	.92945	.39694	2.51929	21
40	.35293	.93565	.37720	2.65109	.36921	.92935	.39727	2.51715	20
41	.35320	.93555	.37754	2.64875	.36948	.92924	.39761	2.51502	19
42	.35347	.93544	.37787	2.64642	.36975	.92913	.39795	2.51289	18
43	.35375	.93534	.37820	2.64410	.37002	.92902	.39829	2.51076	17
44	.35402	.93524	.37853	2.64177	.37029	.92892	.39862	2.50864	16
45	.35429	.93514	.37887	2.63945	.37056	.92881	.39896	2.50652	15
46	.35456	.93503	.37920	2.63714	.37083	.92870	.39930	2.50440	14
47	.35484	.93493	.37953	2.63483	.37110	.92859	.39963	2.50229	13
48	.35511	.93483	.37986	2.63252	.37137	.92849	.39997	2.50018	12
49	.35538	.93472	.38020	2.63021	.37164	.92838	.40031	2.49807	11
50	.35565	.93462	.38053	2.62791	.37191	.92827	.40065	2.49597	10
51	.35592	.93452	.38086	2.62561	.37218	.92816	.40098	2.49386	9
52	.35619	.93441	.38120	2.62332	.37245	.92805	.40132	2.49177	8
53	.35647	.93431	.38153	2.62103	.37272	.92794	.40166	2.48967	7
54	.35674	.93420	.38186	2.61874	.37299	.92784	.40200	2.48758	6
55	.35701	.93410	.38220	2.61646	.37326	.92773	.40234	2.48549	5
56	.35728	.93400	.38253	2.61418	.37353	.92762	.40267	2.48340	4
57	.35755	.93389	.38286	2.61190	.37380	.92751	.40301	2.48132	3
58	.35782	.93379	.38320	2.60963	.37407	.92740	.40335	2.47924	2
59	.35810	.93368	.38353	2.60736	.37434	.92729	.40369	2.47716	1
60	.35837	.93358	.38386	2.60509	.37461	.92718	.40403	2.47509	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

22°					23°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.37461	.92718	.40403	2.47509	.39073	.92050	.42447	2.35585	60
1	.37488	.92707	.40436	2.47302	.39100	.92039	.42482	2.35395	59
2	.37515	.92697	.40470	2.47095	.39127	.92028	.42516	2.35205	58
3	.37542	.92686	.40504	2.46888	.39153	.92016	.42551	2.35015	57
4	.37569	.92675	.40538	2.46682	.39180	.92005	.42585	2.34825	56
5	.37595	.92664	.40572	2.46476	.39207	.91994	.42619	2.34636	55
6	.37622	.92653	.40606	2.46270	.39234	.91982	.42654	2.34447	54
7	.37649	.92642	.40640	2.46065	.39260	.91971	.42688	2.34258	53
8	.37676	.92631	.40674	2.45860	.39287	.91959	.42722	2.34069	52
9	.37703	.92620	.40707	2.45655	.39314	.91948	.42757	2.33881	51
10	.37730	.92609	.40741	2.45451	.39341	.91936	.42791	2.33693	50
11	.37757	.92598	.40775	2.45246	.39367	.91925	.42826	2.33505	49
12	.37784	.92587	.40809	2.45043	.39394	.91914	.42860	2.33317	48
13	.37811	.92576	.40843	2.44839	.39421	.91902	.42894	2.33130	47
14	.37838	.92565	.40877	2.44636	.39448	.91891	.42929	2.32943	46
15	.37865	.92554	.40911	2.44433	.39474	.91879	.42963	2.32756	45
16	.37892	.92543	.40945	2.44230	.39501	.91868	.42998	2.32570	44
17	.37919	.92532	.40979	2.44027	.39528	.91856	.43032	2.32383	43
18	.37946	.92521	.41013	2.43825	.39555	.91845	.43067	2.32197	42
19	.37973	.92510	.41047	2.43623	.39581	.91833	.43101	2.32012	41
20	.37999	.92499	.41081	2.43422	.39608	.91822	.43136	2.31826	40
21	.38026	.92488	.41115	2.43220	.39635	.91810	.43170	2.31641	39
22	.38053	.92477	.41149	2.43019	.39661	.91799	.43205	2.31456	38
23	.38080	.92466	.41183	2.42819	.39688	.91787	.43239	2.31271	37
24	.38107	.92455	.41217	2.42618	.39715	.91775	.43274	2.31086	36
25	.38134	.92444	.41251	2.42418	.39741	.91764	.43308	2.30902	35
26	.38161	.92432	.41285	2.42218	.39768	.91752	.43343	2.30718	34
27	.38188	.92421	.41319	2.42019	.39795	.91741	.43378	2.30534	33
28	.38215	.92410	.41353	2.41819	.39822	.91729	.43412	2.30351	32
29	.38241	.92399	.41387	2.41620	.39848	.91718	.43447	2.30167	31
30	.38268	.92388	.41421	2.41421	.39875	.91706	.43481	2.29984	30
31	.38295	.92377	.41455	2.41223	.39902	.91694	.43516	2.29801	29
32	.38322	.92366	.41490	2.41025	.39928	.91683	.43550	2.29619	28
33	.38349	.92355	.41524	2.40827	.39955	.91671	.43585	2.29437	27
34	.38376	.92343	.41558	2.40629	.39982	.91660	.43620	2.29254	26
35	.38403	.92332	.41592	2.40432	.40008	.91648	.43654	2.29073	25
36	.38430	.92321	.41626	2.40235	.40035	.91636	.43689	2.28891	24
37	.38456	.92310	.41660	2.40038	.40062	.91625	.43724	2.28710	23
38	.38483	.92299	.41694	2.39841	.40088	.91613	.43758	2.28528	22
39	.38510	.92287	.41728	2.39645	.40115	.91601	.43793	2.28348	21
40	.38537	.92276	.41763	2.39449	.40141	.91590	.43828	2.28167	20
41	.38564	.92265	.41797	2.39253	.40168	.91578	.43862	2.27987	19
42	.38591	.92254	.41831	2.39058	.40195	.91566	.43897	2.27806	18
43	.38617	.92243	.41865	2.38863	.40221	.91555	.43932	2.27626	17
44	.38644	.92231	.41899	2.38668	.40248	.91543	.43966	2.27447	16
45	.38671	.92220	.41933	2.38473	.40275	.91531	.44001	2.27267	15
46	.38698	.92209	.41968	2.38279	.40301	.91519	.44036	2.27088	14
47	.38725	.92198	.42002	2.38084	.40328	.91508	.44071	2.26909	13
48	.38752	.92186	.42036	2.37891	.40355	.91496	.44105	2.26730	12
49	.38778	.92175	.42070	2.37697	.40381	.91484	.44140	2.26552	11
50	.38805	.92164	.42105	2.37504	.40408	.91472	.44175	2.26374	10
51	.38832	.92152	.42139	2.37311	.40434	.91461	.44210	2.26196	9
52	.38859	.92141	.42173	2.37118	.40461	.91449	.44244	2.26018	8
53	.38886	.92130	.42207	2.36925	.40488	.91437	.44279	2.25840	7
54	.38912	.92119	.42242	2.36733	.40514	.91425	.44314	2.25663	6
55	.38939	.92107	.42276	2.36541	.40541	.91414	.44349	2.25486	5
56	.38966	.92096	.42310	2.36349	.40567	.91402	.44384	2.25309	4
57	.38993	.92085	.42345	2.36158	.40594	.91390	.44418	2.25132	3
58	.39020	.92073	.42379	2.35967	.40621	.91378	.44453	2.24956	2
59	.39046	.92062	.42413	2.35776	.40647	.91366	.44488	2.24780	1
60	.39073	.92050	.42447	2.35585	.40674	.91355	.44523	2.24604	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

24°

25°

'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.40674	.91355	.44523	2.24604	.42262	.90631	.46631	2.14451	60
1	.40700	.91343	.44558	2.24428	.42288	.90618	.46666	2.14288	59
2	.40727	.91331	.44593	2.24252	.42315	.90606	.46702	2.14125	58
3	.40753	.91319	.44627	2.24077	.42341	.90594	.46737	2.13963	57
4	.40780	.91307	.44662	2.23902	.42367	.90582	.46772	2.13801	56
5	.40806	.91295	.44697	2.23727	.42394	.90569	.46808	2.13639	55
6	.40833	.91283	.44732	2.23553	.42420	.90557	.46843	2.13477	54
7	.40860	.91272	.44767	2.23378	.42446	.90545	.46879	2.13316	53
8	.40886	.91260	.44802	2.23204	.42473	.90532	.46914	2.13154	52
9	.40913	.91248	.44837	2.23030	.42499	.90520	.46950	2.12993	51
10	.40939	.91236	.44872	2.22857	.42525	.90507	.46985	2.12832	50
11	.40966	.91224	.44907	2.22683	.42552	.90495	.47021	2.12671	49
12	.40992	.91212	.44942	2.22510	.42578	.90483	.47056	2.12511	48
13	.41019	.91200	.44977	2.22337	.42604	.90470	.47092	2.12350	47
14	.41045	.91188	.45012	2.22164	.42631	.90458	.47128	2.12190	46
15	.41072	.91176	.45047	2.21992	.42657	.90446	.47163	2.12030	45
16	.41098	.91164	.45082	2.21819	.42683	.90433	.47199	2.11871	44
17	.41125	.91152	.45117	2.21647	.42709	.90421	.47234	2.11711	43
18	.41151	.91140	.45152	2.21475	.42736	.90408	.47270	2.11552	42
19	.41178	.91128	.45187	2.21304	.42762	.90396	.47305	2.11392	41
20	.41204	.91116	.45222	2.21132	.42788	.90383	.47341	2.11233	40
21	.41231	.91104	.45257	2.20961	.42815	.90371	.47377	2.11075	39
22	.41257	.91092	.45292	2.20790	.42841	.90358	.47412	2.10916	38
23	.41284	.91080	.45327	2.20619	.42867	.90346	.47448	2.10758	37
24	.41310	.91068	.45362	2.20449	.42894	.90334	.47483	2.10600	36
25	.41337	.91056	.45397	2.20278	.42920	.90321	.47519	2.10442	35
26	.41363	.91044	.45432	2.20108	.42946	.90309	.47555	2.10284	34
27	.41390	.91032	.45467	2.19938	.42972	.90296	.47590	2.10126	33
28	.41416	.91020	.45502	2.19769	.42999	.90284	.47626	2.09969	32
29	.41443	.91008	.45538	2.19599	.43025	.90271	.47662	2.09811	31
30	.41469	.90996	.45573	2.19430	.43051	.90259	.47698	2.09654	30
31	.41496	.90984	.45608	2.19261	.43077	.90246	.47733	2.09498	29
32	.41522	.90972	.45643	2.19092	.43104	.90233	.47769	2.09341	28
33	.41549	.90960	.45678	2.18923	.43130	.90221	.47805	2.09184	27
34	.41575	.90948	.45713	2.18755	.43156	.90208	.47840	2.09028	26
35	.41602	.90936	.45748	2.18587	.43182	.90196	.47876	2.08872	25
36	.41628	.90924	.45784	2.18419	.43209	.90183	.47912	2.08716	24
37	.41655	.90911	.45819	2.18251	.43235	.90171	.47948	2.08560	23
38	.41681	.90899	.45854	2.18084	.43261	.90158	.47984	2.08405	22
39	.41707	.90887	.45889	2.17916	.43287	.90146	.48019	2.08250	21
40	.41734	.90875	.45924	2.17749	.43313	.90133	.48055	2.08094	20
41	.41760	.90863	.45960	2.17582	.43340	.90120	.48091	2.07939	19
42	.41787	.90851	.45995	2.17416	.43366	.90108	.48127	2.07785	18
43	.41813	.90839	.46030	2.17249	.43392	.90095	.48163	2.07630	17
44	.41840	.90826	.46065	2.17083	.43418	.90082	.48198	2.07476	16
45	.41866	.90814	.46101	2.16917	.43445	.90070	.48234	2.07321	15
46	.41892	.90802	.46136	2.16751	.43471	.90057	.48270	2.07167	14
47	.41919	.90790	.46171	2.16585	.43497	.90045	.48306	2.07014	13
48	.41945	.90778	.46206	2.16420	.43523	.90032	.48342	2.06860	12
49	.41972	.90766	.46242	2.16255	.43549	.90019	.48378	2.06706	11
50	.41998	.90753	.46277	2.16090	.43575	.90007	.48414	2.06553	10
51	.42024	.90741	.46312	2.15925	.43602	.89994	.48450	2.06400	9
52	.42051	.90729	.46348	2.15760	.43628	.89981	.48486	2.06247	8
53	.42077	.90717	.46383	2.15596	.43654	.89968	.48521	2.06094	7
54	.42104	.90704	.46418	2.15432	.43680	.89956	.48557	2.05942	6
55	.42130	.90692	.46454	2.15268	.43706	.89943	.48593	2.05790	5
56	.42156	.90680	.46489	2.15104	.43733	.89930	.48629	2.05637	4
57	.42183	.90668	.46525	2.14940	.43759	.89918	.48665	2.05485	3
58	.42209	.90655	.46560	2.14777	.43785	.89905	.48701	2.05333	2
59	.42235	.90643	.46595	2.14614	.43811	.89892	.48737	2.05182	1
60	.42262	.90631	.46631	2.14451	.43837	.89879	.48773	2.05030	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

65°

707

64°

26°

27°

Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
.43837	.89879	.48773	2.05030	.45399	.89101	.50953	1.96261	60
.43863	.89867	.48809	2.04879	.45425	.89087	.50989	1.96120	59
.43889	.89854	.48845	2.04728	.45451	.89074	.51026	1.95979	58
.43916	.89841	.48881	2.04577	.45477	.89061	.51063	1.95838	57
.43942	.89828	.48917	2.04426	.45503	.89048	.51099	1.95698	56
.43968	.89816	.48953	2.04276	.45529	.89035	.51136	1.95557	55
.43994	.89803	.48989	2.04125	.45554	.89021	.51173	1.95417	54
.44020	.89790	.49026	2.03975	.45580	.89008	.51209	1.95277	53
.44046	.89777	.49062	2.03825	.45606	.88995	.51246	1.95137	52
.44072	.89764	.49098	2.03675	.45632	.88981	.51283	1.94997	51
.44098	.89752	.49134	2.03526	.45658	.88968	.51319	1.94858	50
.44124	.89739	.49170	2.03376	.45684	.88955	.51356	1.94718	49
.44151	.89726	.49206	2.03227	.45710	.88942	.51393	1.94579	48
.44177	.89713	.49242	2.03078	.45736	.88928	.51430	1.94440	47
.44203	.89700	.49278	2.02929	.45762	.88915	.51467	1.94301	46
.44229	.89687	.49315	2.02780	.45787	.88902	.51503	1.94162	45
.44255	.89674	.49351	2.02631	.45813	.88888	.51540	1.94023	44
.44281	.89662	.49387	2.02483	.45839	.88875	.51577	1.93885	43
.44307	.89649	.49423	2.02335	.45865	.88862	.51614	1.93746	42
.44333	.89636	.49459	2.02187	.45891	.88848	.51651	1.93608	41
.44359	.89623	.49495	2.02039	.45917	.88835	.51688	1.93470	40
.44385	.89610	.49532	2.01891	.45942	.88822	.51724	1.93332	39
.44411	.89597	.49568	2.01743	.45968	.88808	.51761	1.93195	38
.44437	.89584	.49604	2.01596	.45994	.88795	.51798	1.93057	37
.44464	.89571	.49640	2.01449	.46020	.88782	.51835	1.92920	36
.44490	.89558	.49677	2.01302	.46046	.88768	.51872	1.92782	35
.44516	.89545	.49713	2.01155	.46072	.88755	.51909	1.92645	34
.44542	.89532	.49749	2.01008	.46097	.88741	.51946	1.92508	33
.44568	.89519	.49786	2.00862	.46123	.88728	.51983	1.92371	32
.44594	.89506	.49822	2.00715	.46149	.88715	.52020	1.92235	31
.44620	.89493	.49858	2.00569	.46175	.88701	.52057	1.92098	30
.44646	.89480	.49894	2.00423	.46201	.88688	.52094	1.91962	29
.44672	.89467	.49931	2.00277	.46226	.88674	.52131	1.91826	28
.44698	.89454	.49967	2.00131	.46252	.88661	.52168	1.91690	27
.44724	.89441	.50004	1.99986	.46278	.88647	.52205	1.91554	26
.44750	.89428	.50040	1.99841	.46304	.88634	.52242	1.91418	25
.44776	.89415	.50076	1.99695	.46330	.88620	.52279	1.91282	24
.44802	.89402	.50113	1.99550	.46355	.88607	.52316	1.91147	23
.44828	.89389	.50149	1.99406	.46381	.88593	.52353	1.91012	22
.44854	.89376	.50185	1.99261	.46407	.88580	.52390	1.90876	21
.44880	.89363	.50222	1.99116	.46433	.88566	.52427	1.90741	20
.44906	.89350	.50258	1.98972	.46458	.88553	.52464	1.90607	19
.44932	.89337	.50295	1.98828	.46484	.88539	.52501	1.90472	18
.44958	.89324	.50331	1.98684	.46510	.88526	.52538	1.90337	17
.44984	.89311	.50368	1.98540	.46536	.88512	.52575	1.90203	16
.45010	.89298	.50404	1.98396	.46561	.88499	.52613	1.90069	15
.45036	.89285	.50441	1.98253	.46587	.88485	.52650	1.89935	14
.45062	.89272	.50477	1.98110	.46613	.88472	.52687	1.89801	13
.45088	.89259	.50514	1.97966	.46639	.88458	.52724	1.89667	12
.45114	.89245	.50550	1.97823	.46664	.88445	.52761	1.89533	11
.45140	.89232	.50587	1.97681	.46690	.88431	.52798	1.89400	10
.45166	.89219	.50623	1.97538	.46716	.88417	.52836	1.89266	9
.45192	.89206	.50660	1.97395	.46742	.88404	.52873	1.89133	8
.45218	.89193	.50696	1.97253	.46767	.88390	.52910	1.89000	7
.45243	.89180	.50733	1.97111	.46793	.88377	.52947	1.88867	6
.45269	.89167	.50769	1.96969	.46819	.88363	.52985	1.88734	5
.45295	.89153	.50806	1.96827	.46844	.88349	.53022	1.88602	4
.45321	.89140	.50843	1.96685	.46870	.88336	.53059	1.88469	3
.45347	.89127	.50879	1.96544	.46896	.88322	.53096	1.88337	2
.45373	.89114	.50916	1.96402	.46921	.88308	.53134	1.88205	1
.45399	.89101	.50953	1.96261	.46947	.88295	.53171	1.88073	0
Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

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28°

29°

	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.46947	.88295	.53171	1.88073	.48481	.87462	.55431	1.80405	60
1	.46973	.88281	.53208	1.87941	.48506	.87448	.55469	1.80281	59
2	.46999	.88267	.53246	1.87809	.48532	.87434	.55507	1.80158	58
3	.47024	.88254	.53283	1.87677	.48557	.87420	.55545	1.80034	57
4	.47050	.88240	.53320	1.87546	.48583	.87406	.55583	1.79911	56
5	.47076	.88226	.53358	1.87415	.48608	.87391	.55621	1.79788	55
6	.47101	.88213	.53395	1.87283	.48634	.87377	.55659	1.79665	54
7	.47127	.88199	.53432	1.87152	.48659	.87363	.55697	1.79542	53
8	.47153	.88185	.53470	1.87021	.48684	.87349	.55736	1.79419	52
9	.47178	.88172	.53507	1.86891	.48710	.87335	.55774	1.79296	51
10	.47204	.88158	.53545	1.86760	.48735	.87321	.55812	1.79174	50
11	.47229	.88144	.53582	1.86630	.48761	.87306	.55850	1.79051	49
12	.47255	.88130	.53620	1.86499	.48786	.87292	.55888	1.78929	48
13	.47281	.88117	.53657	1.86369	.48811	.87278	.55926	1.78807	47
14	.47306	.88103	.53694	1.86239	.48837	.87264	.55964	1.78685	46
15	.47332	.88089	.53732	1.86109	.48862	.87250	.56003	1.78563	45
16	.47358	.88075	.53769	1.85979	.48888	.87235	.56041	1.78441	44
17	.47383	.88062	.53807	1.85850	.48913	.87221	.56079	1.78319	43
18	.47409	.88048	.53844	1.85720	.48938	.87207	.56117	1.78198	42
19	.47434	.88034	.53882	1.85591	.48964	.87193	.56156	1.78077	41
20	.47460	.88020	.53920	1.85462	.48989	.87178	.56194	1.77955	40
21	.47486	.88006	.53957	1.85333	.49014	.87164	.56232	1.77834	39
22	.47511	.87993	.53995	1.85204	.49040	.87150	.56270	1.77713	38
23	.47537	.87979	.54032	1.85075	.49065	.87136	.56309	1.77592	37
24	.47562	.87965	.54070	1.84946	.49090	.87121	.56347	1.77471	36
25	.47588	.87951	.54107	1.84818	.49116	.87107	.56385	1.77351	35
26	.47614	.87937	.54145	1.84689	.49141	.87093	.56424	1.77230	34
27	.47639	.87923	.54183	1.84561	.49166	.87079	.56462	1.77110	33
28	.47665	.87909	.54220	1.84433	.49192	.87064	.56501	1.76990	32
29	.47690	.87896	.54258	1.84305	.49217	.87050	.56539	1.76869	31
30	.47716	.87882	.54296	1.84177	.49242	.87036	.56577	1.76749	30
31	.47741	.87868	.54333	1.84049	.49268	.87021	.56616	1.76629	29
32	.47767	.87854	.54371	1.83922	.49293	.87007	.56654	1.76510	28
33	.47793	.87840	.54409	1.83794	.49318	.86993	.56693	1.76390	27
34	.47818	.87826	.54446	1.83667	.49344	.86978	.56731	1.76271	26
35	.47844	.87812	.54484	1.83540	.49369	.86964	.56769	1.76151	25
36	.47869	.87798	.54522	1.83413	.49394	.86949	.56808	1.76032	24
37	.47895	.87784	.54560	1.83286	.49419	.86935	.56846	1.75913	23
38	.47920	.87770	.54597	1.83159	.49445	.86921	.56885	1.75794	22
39	.47946	.87756	.54635	1.83033	.49470	.86906	.56923	1.75675	21
40	.47971	.87743	.54673	1.82906	.49495	.86892	.56962	1.75556	20
41	.47997	.87729	.54711	1.82780	.49521	.86878	.57000	1.75437	19
42	.48022	.87715	.54748	1.82654	.49546	.86863	.57039	1.75319	18
43	.48048	.87701	.54786	1.82528	.49571	.86849	.57078	1.75200	17
44	.48073	.87687	.54824	1.82402	.49596	.86834	.57116	1.75082	16
45	.48099	.87673	.54862	1.82276	.49622	.86820	.57155	1.74964	15
46	.48124	.87659	.54900	1.82150	.49647	.86805	.57193	1.74846	14
47	.48150	.87645	.54938	1.82025	.49672	.86791	.57232	1.74728	13
48	.48175	.87631	.54975	1.81899	.49697	.86777	.57271	1.74610	12
49	.48201	.87617	.55013	1.81774	.49723	.86762	.57309	1.74492	11
50	.48226	.87603	.55051	1.81649	.49748	.86748	.57348	1.74375	10
51	.48252	.87589	.55089	1.81524	.49773	.86733	.57386	1.74257	9
52	.48277	.87575	.55127	1.81399	.49798	.86719	.57425	1.74140	8
53	.48303	.87561	.55165	1.81274	.49824	.86704	.57464	1.74022	7
54	.48328	.87546	.55203	1.81150	.49849	.86690	.57503	1.73905	6
55	.48354	.87532	.55241	1.81025	.49874	.86675	.57541	1.73788	5
56	.48379	.87518	.55279	1.80901	.49899	.86661	.57580	1.73671	4
57	.48405	.87504	.55317	1.80777	.49924	.86646	.57619	1.73555	3
58	.48430	.87490	.55355	1.80653	.49950	.86632	.57657	1.73438	2
59	.48456	.87476	.55393	1.80529	.49975	.86617	.57696	1.73321	1
60	.48481	.87462	.55431	1.80405	.50000	.86603	.57735	1.73205	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

30°

31°

	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.50000	.86603	.57735	1.73205	.51504	.85717	.60086	1.66428	60
1	.50025	.86588	.57774	1.73089	.51529	.85702	.60126	1.66318	59
2	.50050	.86573	.57813	1.72973	.51554	.85687	.60165	1.66209	58
3	.50076	.86559	.57851	1.72857	.51579	.85672	.60205	1.66099	57
4	.50101	.86544	.57890	1.72741	.51604	.85657	.60245	1.65990	56
5	.50126	.86530	.57929	1.72625	.51628	.85642	.60284	1.65881	55
6	.50151	.86515	.57968	1.72509	.51653	.85627	.60324	1.65772	54
7	.50176	.86501	.58007	1.72393	.51678	.85612	.60364	1.65663	53
8	.50201	.86486	.58046	1.72278	.51703	.85597	.60403	1.65554	52
9	.50227	.86471	.58085	1.72163	.51728	.85582	.60443	1.65445	51
10	.50252	.86457	.58124	1.72047	.51753	.85567	.60483	1.65337	50
11	.50277	.86442	.58162	1.71932	.51778	.85551	.60522	1.65228	49
12	.50302	.86427	.58201	1.71817	.51803	.85536	.60562	1.65120	48
13	.50327	.86413	.58240	1.71702	.51828	.85521	.60602	1.65011	47
14	.50352	.86398	.58279	1.71588	.51852	.85506	.60642	1.64903	46
15	.50377	.86384	.58318	1.71473	.51877	.85491	.60681	1.64795	45
16	.50403	.86369	.58357	1.71358	.51902	.85476	.60721	1.64687	44
17	.50428	.86354	.58396	1.71244	.51927	.85461	.60761	1.64579	43
18	.50453	.86340	.58435	1.71129	.51952	.85446	.60801	1.64471	42
19	.50478	.86325	.58474	1.71015	.51977	.85431	.60841	1.64363	41
20	.50503	.86310	.58513	1.70901	.52002	.85416	.60881	1.64256	40
21	.50528	.86295	.58552	1.70787	.52026	.85401	.60921	1.64148	39
22	.50553	.86281	.58591	1.70673	.52051	.85385	.60960	1.64041	38
23	.50578	.86266	.58631	1.70560	.52076	.85370	.61000	1.63934	37
24	.50603	.86251	.58670	1.70446	.52101	.85355	.61040	1.63826	36
25	.50628	.86237	.58709	1.70332	.52126	.85340	.61080	1.63719	35
26	.50654	.86222	.58748	1.70219	.52151	.85325	.61120	1.63612	34
27	.50679	.86207	.58787	1.70106	.52175	.85310	.61160	1.63505	33
28	.50704	.86192	.58826	1.69992	.52200	.85294	.61200	1.63398	32
29	.50729	.86178	.58865	1.69879	.52225	.85279	.61240	1.63292	31
30	.50754	.86163	.58905	1.69766	.52250	.85264	.61280	1.63185	30
31	.50779	.86148	.58944	1.69653	.52275	.85249	.61320	1.63079	29
32	.50804	.86133	.58983	1.69541	.52299	.85234	.61360	1.62972	28
33	.50829	.86119	.59022	1.69428	.52324	.85218	.61400	1.62866	27
34	.50854	.86104	.59061	1.69316	.52349	.85203	.61440	1.62760	26
35	.50879	.86089	.59101	1.69203	.52374	.85188	.61480	1.62654	25
36	.50904	.86074	.59140	1.69091	.52399	.85173	.61520	1.62548	24
37	.50929	.86059	.59179	1.68979	.52423	.85157	.61561	1.62442	23
38	.50954	.86045	.59218	1.68866	.52448	.85142	.61601	1.62336	22
39	.50979	.86030	.59258	1.68754	.52473	.85127	.61641	1.62230	21
40	.51004	.86015	.59297	1.68643	.52498	.85112	.61681	1.62125	20
41	.51029	.86000	.59336	1.68531	.52522	.85096	.61721	1.62019	19
42	.51054	.85985	.59376	1.68419	.52547	.85081	.61761	1.61914	18
43	.51079	.85970	.59415	1.68308	.52572	.85066	.61801	1.61808	17
44	.51104	.85956	.59454	1.68196	.52597	.85051	.61842	1.61703	16
45	.51129	.85941	.59494	1.68085	.52621	.85035	.61882	1.61598	15
46	.51154	.85926	.59533	1.67974	.52646	.85020	.61922	1.61493	14
47	.51179	.85911	.59573	1.67863	.52671	.85005	.61962	1.61388	13
48	.51204	.85896	.59612	1.67752	.52696	.84989	.62003	1.61283	12
49	.51229	.85881	.59651	1.67641	.52720	.84974	.62043	1.61179	11
50	.51254	.85866	.59691	1.67530	.52745	.84959	.62083	1.61074	10
51	.51279	.85851	.59730	1.67419	.52770	.84943	.62124	1.60970	9
52	.51304	.85836	.59770	1.67309	.52794	.84928	.62164	1.60865	8
53	.51329	.85821	.59809	1.67198	.52819	.84913	.62204	1.60761	7
54	.51354	.85806	.59849	1.67088	.52844	.84897	.62245	1.60657	6
55	.51379	.85792	.59888	1.66978	.52869	.84882	.62285	1.60553	5
56	.51404	.85777	.59928	1.66867	.52893	.84866	.62325	1.60449	4
57	.51429	.85762	.59967	1.66757	.52918	.84851	.62366	1.60345	3
58	.51454	.85747	.60007	1.66647	.52943	.84836	.62406	1.60241	2
59	.51479	.85732	.60046	1.66538	.52967	.84820	.62446	1.60137	1
60	.51504	.85717	.60086	1.66428	.52992	.84805	.62487	1.60033	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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58°

32°

33°

	Sin.	Cos	Tan.	Cot	Sin.	Cos.	Tan.	Cot.	
0	.52992	.84805	.62487	1.60033	.54464	.83867	.64941	1.53986	60
1	.53017	.84789	.62527	1.59930	.54488	.83851	.64982	1.53888	59
2	.53041	.84774	.62568	1.59826	.54513	.83835	.65024	1.53791	58
3	.53066	.84759	.62608	1.59723	.54537	.83819	.65065	1.53693	57
4	.53091	.84743	.62649	1.59620	.54561	.83804	.65106	1.53595	56
5	.53115	.84728	.62689	1.59517	.54585	.83788	.65148	1.53497	55
6	.53140	.84712	.62730	1.59414	.54610	.83772	.65189	1.53400	54
7	.53164	.84697	.62770	1.59311	.54635	.83756	.65231	1.53302	53
8	.53189	.84681	.62811	1.59208	.54659	.83740	.65272	1.53205	52
9	.53214	.84666	.62852	1.59105	.54683	.83724	.65314	1.53107	51
10	.53238	.84650	.62892	1.59002	.54708	.83708	.65355	1.53010	50
11	.53263	.84635	.62933	1.58900	.54732	.83692	.65397	1.52913	49
12	.53288	.84619	.62973	1.58797	.54756	.83676	.65438	1.52816	48
13	.53312	.84604	.63014	1.58695	.54781	.83660	.65480	1.52719	47
14	.53337	.84588	.63055	1.58593	.54805	.83645	.65521	1.52622	46
15	.53361	.84573	.63095	1.58490	.54829	.83629	.65563	1.52525	45
16	.53386	.84557	.63136	1.58388	.54854	.83613	.65604	1.52429	44
17	.53411	.84542	.63177	1.58286	.54878	.83597	.65646	1.52332	43
18	.53435	.84526	.63217	1.58184	.54902	.83581	.65688	1.52235	42
19	.53460	.84511	.63258	1.58083	.54927	.83565	.65729	1.52139	41
20	.53484	.84495	.63299	1.57981	.54951	.83549	.65771	1.52043	40
21	.53509	.84480	.63340	1.57879	.54975	.83533	.65813	1.51946	39
22	.53534	.84464	.63380	1.57778	.54999	.83517	.65854	1.51850	38
23	.53558	.84448	.63421	1.57676	.55024	.83501	.65896	1.51754	37
24	.53583	.84433	.63462	1.57575	.55048	.83485	.65938	1.51658	36
25	.53607	.84417	.63503	1.57474	.55072	.83469	.65980	1.51562	35
26	.53632	.84402	.63544	1.57372	.55097	.83453	.66021	1.51466	34
27	.53656	.84386	.63584	1.57271	.55121	.83437	.66063	1.51370	33
28	.53681	.84370	.63625	1.57170	.55145	.83421	.66105	1.51275	32
29	.53705	.84355	.63666	1.57069	.55169	.83405	.66147	1.51179	31
30	.53730	.84339	.63707	1.56969	.55194	.83389	.66189	1.51084	30
31	.53754	.84324	.63748	1.56868	.55218	.83373	.66230	1.50988	29
32	.53779	.84308	.63789	1.56767	.55242	.83356	.66272	1.50893	28
33	.53804	.84292	.63830	1.56667	.55266	.83340	.66314	1.50797	27
34	.53828	.84277	.63871	1.56566	.55291	.83324	.66356	1.50702	26
35	.53853	.84261	.63912	1.56466	.55315	.83308	.66398	1.50607	25
36	.53877	.84245	.63953	1.56366	.55339	.83292	.66440	1.50512	24
37	.53902	.84230	.63994	1.56265	.55363	.83276	.66482	1.50417	23
38	.53926	.84214	.64035	1.56165	.55388	.83260	.66524	1.50322	22
39	.53951	.84198	.64076	1.56065	.55412	.83244	.66566	1.50228	21
40	.53975	.84182	.64117	1.55966	.55436	.83228	.66608	1.50133	20
41	.54000	.84167	.64158	1.55866	.55460	.83212	.66650	1.50038	19
42	.54024	.84151	.64199	1.55766	.55484	.83195	.66692	1.49944	18
43	.54049	.84135	.64240	1.55666	.55509	.83179	.66734	1.49849	17
44	.54073	.84120	.64281	1.55567	.55533	.83163	.66776	1.49755	16
45	.54097	.84104	.64322	1.55467	.55557	.83147	.66818	1.49661	15
46	.54122	.84088	.64363	1.55368	.55581	.83131	.66860	1.49566	14
47	.54146	.84072	.64404	1.55269	.55605	.83115	.66902	1.49472	13
48	.54171	.84057	.64446	1.55170	.55630	.83098	.66944	1.49378	12
49	.54195	.84041	.64487	1.55071	.55654	.83082	.66986	1.49284	11
50	.54220	.84025	.64528	1.54972	.55678	.83066	.67028	1.49190	10
51	.54244	.84009	.64569	1.54873	.55702	.83050	.67071	1.49097	9
52	.54269	.83994	.64610	1.54774	.55726	.83034	.67113	1.49003	8
53	.54293	.83978	.64652	1.54675	.55750	.83017	.67155	1.48909	7
54	.54317	.83962	.64693	1.54576	.55775	.83001	.67197	1.48816	6
55	.54342	.83946	.64734	1.54478	.55799	.82985	.67239	1.48722	5
56	.54366	.83930	.64775	1.54379	.55823	.82969	.67282	1.48629	4
57	.54391	.83915	.64817	1.54281	.55847	.82953	.67324	1.48536	3
58	.54415	.83899	.64858	1.54183	.55871	.82936	.67366	1.48442	2
59	.54440	.83883	.64899	1.54085	.55895	.82920	.67409	1.48349	1
60	.54464	.83867	.64941	1.53986	.55919	.82904	.67451	1.48256	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

	34°				35°				
	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.55919	.82904	.67451	1.48256	.57358	.81915	.70021	1.42815	60
1	.55943	.82887	.67493	1.48163	.57381	.81899	.70064	1.42726	59
2	.55968	.82871	.67536	1.48070	.57405	.81882	.70107	1.42638	58
3	.55992	.82855	.67578	1.47977	.57429	.81865	.70151	1.42550	57
4	.56016	.82839	.67620	1.47885	.57453	.81848	.70194	1.42462	56
5	.56040	.82822	.67663	1.47792	.57477	.81832	.70238	1.42374	55
6	.56064	.82806	.67705	1.47699	.57501	.81815	.70281	1.42286	54
7	.56088	.82790	.67748	1.47607	.57524	.81798	.70325	1.42198	53
8	.56112	.82773	.67790	1.47514	.57548	.81782	.70368	1.42110	52
9	.56136	.82757	.67832	1.47422	.57572	.81765	.70412	1.42022	51
10	.56160	.82741	.67875	1.47330	.57596	.81748	.70455	1.41934	50
11	.56184	.82724	.67917	1.47238	.57619	.81731	.70499	1.41847	49
12	.56208	.82708	.67960	1.47146	.57643	.81714	.70542	1.41759	48
13	.56232	.82692	.68002	1.47053	.57667	.81698	.70586	1.41672	47
14	.56256	.82675	.68045	1.46962	.57691	.81681	.70629	1.41584	46
15	.56280	.82659	.68088	1.46870	.57715	.81664	.70673	1.41497	45
16	.56305	.82643	.68130	1.46778	.57738	.81647	.70717	1.41409	44
17	.56329	.82626	.68173	1.46686	.57762	.81631	.70760	1.41322	43
18	.56353	.82610	.68215	1.46595	.57786	.81614	.70804	1.41235	42
19	.56377	.82593	.68258	1.46503	.57810	.81597	.70848	1.41148	41
20	.56401	.82577	.68301	1.46411	.57833	.81580	.70891	1.41061	40
21	.56425	.82561	.68343	1.46320	.57857	.81563	.70935	1.40974	39
22	.56449	.82544	.68386	1.46229	.57881	.81546	.70979	1.40887	38
23	.56473	.82528	.68429	1.46137	.57904	.81530	.71023	1.40800	37
24	.56497	.82511	.68471	1.46046	.57928	.81513	.71066	1.40714	36
25	.56521	.82495	.68514	1.45955	.57952	.81496	.71110	1.40627	35
26	.56545	.82478	.68557	1.45864	.57976	.81479	.71154	1.40540	34
27	.56569	.82462	.68600	1.45773	.57999	.81462	.71198	1.40454	33
28	.56593	.82446	.68642	1.45682	.58023	.81445	.71242	1.40367	32
29	.56617	.82429	.68685	1.45592	.58047	.81428	.71285	1.40281	31
30	.56641	.82413	.68728	1.45501	.58070	.81412	.71329	1.40195	30
31	.56665	.82396	.68771	1.45410	.58094	.81395	.71373	1.40109	29
32	.56689	.82380	.68814	1.45320	.58118	.81378	.71417	1.40022	28
33	.56713	.82363	.68857	1.45229	.58141	.81361	.71461	1.39936	27
34	.56736	.82347	.68900	1.45139	.58165	.81344	.71505	1.39850	26
35	.56760	.82330	.68942	1.45049	.58189	.81327	.71549	1.39764	25
36	.56784	.82314	.68985	1.44958	.58212	.81310	.71593	1.39679	24
37	.56803	.82297	.69028	1.44868	.58236	.81293	.71637	1.39593	23
38	.56832	.82281	.69071	1.44778	.58260	.81276	.71681	1.39507	22
39	.56856	.82264	.69114	1.44688	.58283	.81259	.71725	1.39421	21
40	.56880	.82248	.69157	1.44598	.58307	.81242	.71769	1.39336	20
41	.56904	.82231	.69200	1.44508	.58330	.81225	.71813	1.39250	19
42	.56928	.82214	.69243	1.44418	.58354	.81208	.71857	1.39165	18
43	.56952	.82198	.69286	1.44329	.58378	.81191	.71901	1.39079	17
44	.56976	.82181	.69329	1.44239	.58401	.81174	.71946	1.38994	16
45	.57000	.82165	.69372	1.44149	.58425	.81157	.71990	1.38909	15
46	.57024	.82148	.69416	1.44060	.58449	.81140	.72034	1.38824	14
47	.57047	.82132	.69459	1.43970	.58472	.81123	.72078	1.38738	13
48	.57071	.82115	.69502	1.43881	.58496	.81106	.72122	1.38653	12
49	.57095	.82098	.69545	1.43792	.58519	.81089	.72167	1.38568	11
50	.57119	.82082	.69588	1.43703	.58543	.81072	.72211	1.38484	10
51	.57143	.82065	.69631	1.43614	.58567	.81055	.72255	1.38399	9
52	.57167	.82048	.69675	1.43525	.58590	.81038	.72299	1.38314	8
53	.57191	.82032	.69718	1.43436	.58614	.81021	.72344	1.38229	7
54	.57215	.82015	.69761	1.43347	.58637	.81004	.72388	1.38145	6
55	.57238	.81999	.69804	1.43258	.58661	.80987	.72432	1.38060	5
56	.57262	.81982	.69847	1.43169	.58684	.80970	.72477	1.37976	4
57	.57286	.81965	.69891	1.43080	.58708	.80953	.72521	1.37891	3
58	.57310	.81949	.69934	1.42992	.58731	.80936	.72565	1.37807	2
59	.57334	.81932	.69977	1.42903	.58755	.80919	.72610	1.37722	1
60	.57358	.81915	.70021	1.42815	.58779	.80902	.72654	1.37638	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

TABLE IX.—SINUS, COSINES, TANGENTS, AND COTANGENTS.

36°					37°				
	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.58779	.80902	.72654	1.37638	.60182	.79864	.75355	1.32704	60
1	.58802	.80885	.72699	1.37554	.60205	.79846	.75401	1.32624	59
2	.58826	.80867	.72743	1.37470	.60228	.79829	.75447	1.32544	58
3	.58849	.80850	.72788	1.37386	.60251	.79811	.75492	1.32464	57
4	.58873	.80833	.72832	1.37302	.60274	.79793	.75538	1.32384	56
5	.58896	.80816	.72877	1.37218	.60298	.79776	.75584	1.32304	55
6	.58920	.80799	.72921	1.37134	.60321	.79758	.75629	1.32224	54
7	.58943	.80782	.72966	1.37050	.60344	.79741	.75675	1.32144	53
8	.58967	.80765	.73010	1.36967	.60367	.79723	.75721	1.32064	52
9	.58990	.80748	.73055	1.36883	.60390	.79706	.75767	1.31984	51
10	.59014	.80730	.73100	1.36800	.60414	.79688	.75812	1.31904	50
11	.59037	.80713	.73144	1.36716	.60437	.79671	.75858	1.31825	49
12	.59061	.80696	.73189	1.36633	.60460	.79653	.75904	1.31745	48
13	.59084	.80679	.73234	1.36549	.60483	.79635	.75950	1.31666	47
14	.59108	.80662	.73278	1.36466	.60506	.79618	.75996	1.31586	46
15	.59131	.80644	.73323	1.36383	.60529	.79600	.76042	1.31507	45
16	.59154	.80627	.73368	1.36300	.60553	.79583	.76088	1.31427	44
17	.59178	.80610	.73413	1.36217	.60576	.79565	.76134	1.31348	43
18	.59201	.80593	.73457	1.36134	.60599	.79547	.76180	1.31269	42
19	.59225	.80576	.73502	1.36051	.60622	.79530	.76226	1.31190	41
20	.59248	.80558	.73547	1.35968	.60645	.79512	.76272	1.31110	40
21	.59272	.80541	.73592	1.35885	.60668	.79494	.76318	1.31031	39
22	.59295	.80524	.73637	1.35802	.60691	.79477	.76364	1.30952	38
23	.59318	.80507	.73681	1.35719	.60714	.79459	.76410	1.30873	37
24	.59342	.80489	.73726	1.35637	.60738	.79441	.76456	1.30795	36
25	.59365	.80472	.73771	1.35554	.60761	.79424	.76502	1.30716	35
26	.59389	.80455	.73816	1.35472	.60784	.79406	.76548	1.30637	34
27	.59412	.80438	.73861	1.35389	.60807	.79388	.76594	1.30558	33
28	.59436	.80420	.73906	1.35307	.60830	.79371	.76640	1.30480	32
29	.59459	.80403	.73951	1.35224	.60853	.79353	.76686	1.30401	31
30	.59482	.80386	.73996	1.35142	.60876	.79335	.76733	1.30323	30
31	.59506	.80368	.74041	1.35060	.60899	.79318	.76779	1.30244	29
32	.59529	.80351	.74086	1.34978	.60922	.79300	.76825	1.30166	28
33	.59552	.80334	.74131	1.34896	.60945	.79282	.76871	1.30087	27
34	.59576	.80316	.74176	1.34814	.60968	.79264	.76918	1.30009	26
35	.59599	.80299	.74221	1.34732	.60991	.79247	.76964	1.29931	25
36	.59622	.80282	.74267	1.34650	.61015	.79229	.77010	1.29853	24
37	.59646	.80264	.74312	1.34568	.61038	.79211	.77057	1.29775	23
38	.59669	.80247	.74357	1.34487	.61061	.79193	.77103	1.29696	22
39	.59693	.80230	.74402	1.34405	.61084	.79176	.77149	1.29618	21
40	.59716	.80212	.74447	1.34323	.61107	.79158	.77196	1.29541	20
41	.59739	.80195	.74492	1.34242	.61130	.79140	.77242	1.29463	19
42	.59763	.80178	.74538	1.34160	.61153	.79122	.77289	1.29385	18
43	.59786	.80160	.74583	1.34079	.61176	.79105	.77335	1.29307	17
44	.59809	.80143	.74628	1.33998	.61199	.79087	.77382	1.29229	16
45	.59832	.80125	.74674	1.33916	.61222	.79069	.77428	1.29152	15
46	.59856	.80108	.74719	1.33835	.61245	.79051	.77475	1.29074	14
47	.59879	.80091	.74764	1.33754	.61268	.79033	.77521	1.28997	13
48	.59902	.80073	.74810	1.33673	.61291	.79016	.77568	1.28919	12
49	.59926	.80056	.74855	1.33592	.61314	.78998	.77615	1.28842	11
50	.59949	.80038	.74900	1.33511	.61337	.78980	.77661	1.28764	10
51	.59972	.80021	.74946	1.33430	.61360	.78962	.77708	1.28687	9
52	.59995	.80003	.74991	1.33349	.61383	.78944	.77754	1.28610	8
53	.60019	.79986	.75037	1.33268	.61406	.78926	.77801	1.28533	7
54	.60042	.79968	.75082	1.33187	.61429	.78908	.77848	1.28456	6
55	.60065	.79951	.75128	1.33107	.61451	.78891	.77895	1.28379	5
56	.60089	.79934	.75173	1.33026	.61474	.78873	.77941	1.28302	4
57	.60112	.79916	.75219	1.32946	.61497	.78855	.77988	1.28225	3
58	.60135	.79899	.75264	1.32865	.61520	.78837	.78035	1.28148	2
59	.60158	.79881	.75310	1.32785	.61543	.78819	.78082	1.28071	1
60	.60182	.79864	.75355	1.32704	.61566	.78801	.78129	1.27994	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

38°					39°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.61566	.78801	.78129	1.27994	.62932	.77715	.80978	1.23490	60
1	.61589	.78783	.78175	1.27917	.62955	.77696	.81027	1.23416	59
2	.61612	.78765	.78222	1.27841	.62977	.77678	.81075	1.23343	58
3	.61635	.78747	.78269	1.27764	.63000	.77660	.81123	1.23270	57
4	.61658	.78729	.78316	1.27688	.63022	.77641	.81171	1.23196	56
5	.61681	.78711	.78363	1.27611	.63045	.77623	.81220	1.23123	55
6	.61704	.78694	.78410	1.27535	.63068	.77605	.81268	1.23050	54
7	.61726	.78676	.78457	1.27458	.63090	.77586	.81316	1.22977	53
8	.61749	.78658	.78504	1.27382	.63113	.77568	.81364	1.22904	52
9	.61772	.78640	.78551	1.27306	.63135	.77550	.81413	1.22831	51
10	.61795	.78622	.78598	1.27230	.63158	.77531	.81461	1.22758	50
11	.61818	.78604	.78645	1.27153	.63180	.77513	.81510	1.22685	49
12	.61841	.78586	.78692	1.27077	.63203	.77494	.81558	1.22612	48
13	.61864	.78568	.78739	1.27001	.63225	.77476	.81606	1.22539	47
14	.61887	.78550	.78786	1.26925	.63248	.77458	.81655	1.22467	46
15	.61909	.78532	.78834	1.26849	.63271	.77439	.81703	1.22394	45
16	.61932	.78514	.78881	1.26774	.63293	.77421	.81752	1.22321	44
17	.61955	.78496	.78928	1.26698	.63316	.77402	.81800	1.22249	43
18	.61978	.78478	.78975	1.26622	.63338	.77384	.81849	1.22176	42
19	.62001	.78460	.79022	1.26546	.63361	.77366	.81898	1.22104	41
20	.62024	.78442	.79070	1.26471	.63383	.77347	.81946	1.22031	40
21	.62046	.78424	.79117	1.26395	.63406	.77329	.81995	1.21959	39
22	.62069	.78405	.79164	1.26319	.63428	.77310	.82044	1.21886	38
23	.62092	.78387	.79212	1.26244	.63451	.77292	.82092	1.21814	37
24	.62115	.78369	.79259	1.26169	.63473	.77273	.82141	1.21742	36
25	.62138	.78351	.79306	1.26093	.63496	.77255	.82190	1.21670	35
26	.62160	.78333	.79354	1.26018	.63518	.77236	.82238	1.21598	34
27	.62183	.78315	.79401	1.25943	.63540	.77218	.82287	1.21526	33
28	.62206	.78297	.79449	1.25867	.63563	.77199	.82336	1.21454	32
29	.62229	.78279	.79496	1.25792	.63585	.77181	.82385	1.21382	31
30	.62251	.78261	.79544	1.25717	.63608	.77162	.82434	1.21310	30
31	.62274	.78243	.79591	1.25642	.63630	.77144	.82483	1.21238	29
32	.62297	.78225	.79639	1.25567	.63653	.77125	.82531	1.21166	28
33	.62320	.78206	.79686	1.25492	.63675	.77107	.82580	1.21094	27
34	.62342	.78188	.79734	1.25417	.63698	.77088	.82629	1.21023	26
35	.62365	.78170	.79781	1.25343	.63720	.77070	.82678	1.20951	25
36	.62388	.78152	.79829	1.25268	.63742	.77051	.82727	1.20879	24
37	.62411	.78134	.79877	1.25193	.63765	.77033	.82776	1.20808	23
38	.62433	.78116	.79924	1.25118	.63787	.77014	.82825	1.20736	22
39	.62456	.78098	.79972	1.25044	.63810	.76996	.82874	1.20665	21
40	.62479	.78079	.80020	1.24969	.63832	.76977	.82923	1.20593	20
41	.62502	.78061	.80067	1.24895	.63854	.76959	.82972	1.20522	19
42	.62524	.78043	.80115	1.24820	.63877	.76940	.83022	1.20451	18
43	.62547	.78025	.80163	1.24746	.63899	.76921	.83071	1.20379	17
44	.62570	.78007	.80211	1.24672	.63922	.76903	.83120	1.20308	16
45	.62592	.77988	.80258	1.24597	.63944	.76884	.83169	1.20237	15
46	.62615	.77970	.80306	1.24523	.63966	.76866	.83218	1.20166	14
47	.62638	.77952	.80354	1.24449	.63989	.76847	.83268	1.20095	13
48	.62660	.77934	.80402	1.24375	.64011	.76828	.83317	1.20024	12
49	.62683	.77916	.80450	1.24301	.64033	.76810	.83366	1.19953	11
50	.62706	.77897	.80498	1.24227	.64056	.76791	.83415	1.19882	10
51	.62728	.77879	.80546	1.24153	.64078	.76772	.83465	1.19811	9
52	.62751	.77861	.80594	1.24079	.64100	.76754	.83514	1.19740	8
53	.62774	.77843	.80642	1.24005	.64123	.76735	.83564	1.19669	7
54	.62796	.77824	.80690	1.23931	.64145	.76717	.83613	1.19599	6
55	.62819	.77806	.80738	1.23858	.64167	.76698	.83662	1.19528	5
56	.62842	.77788	.80786	1.23784	.64190	.76679	.83712	1.19457	4
57	.62864	.77769	.80834	1.23710	.64212	.76661	.83761	1.19387	3
58	.62887	.77751	.80882	1.23637	.64234	.76642	.83811	1.19316	2
59	.62909	.77733	.80930	1.23563	.64256	.76623	.83860	1.19246	1
60	.62932	.77715	.80978	1.23490	.64279	.76604	.83910	1.19175	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

40°					41°				
'	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	'
0	.64279	.76604	.83910	1.19175	.65606	.75471	.86929	1.15037	60
1	.64301	.76586	.83960	1.19105	.65628	.75452	.86980	1.14969	59
2	.64323	.76567	.84009	1.19035	.65650	.75433	.87031	1.14902	58
3	.64346	.76548	.84059	1.18964	.65672	.75414	.87082	1.14834	57
4	.64368	.76530	.84108	1.18894	.65694	.75395	.87133	1.14767	56
5	.64390	.76511	.84158	1.18824	.65716	.75375	.87184	1.14699	55
6	.64412	.76492	.84208	1.18754	.65738	.75356	.87236	1.14632	54
7	.64435	.76473	.84258	1.18684	.65759	.75337	.87287	1.14565	53
8	.64457	.76455	.84307	1.18614	.65781	.75318	.87338	1.14498	52
9	.64479	.76436	.84357	1.18544	.65803	.75299	.87389	1.14430	51
10	.64501	.76417	.84407	1.18474	.65825	.75280	.87441	1.14363	50
11	.64524	.76398	.84457	1.18404	.65847	.75261	.87492	1.14296	49
12	.64546	.76380	.84507	1.18334	.65869	.75241	.87543	1.14229	48
13	.64568	.76361	.84556	1.18264	.65891	.75222	.87595	1.14162	47
14	.64590	.76342	.84606	1.18194	.65913	.75203	.87646	1.14095	46
15	.64612	.76323	.84656	1.18125	.65935	.75184	.87698	1.14028	45
16	.64635	.76304	.84706	1.18055	.65956	.75165	.87749	1.13961	44
17	.64657	.76286	.84756	1.17986	.65978	.75146	.87801	1.13894	43
18	.64679	.76267	.84806	1.17916	.66000	.75126	.87852	1.13828	42
19	.64701	.76248	.84856	1.17846	.66022	.75107	.87904	1.13761	41
20	.64723	.76229	.84906	1.17777	.66044	.75088	.87955	1.13694	40
21	.64746	.76210	.84956	1.17708	.66066	.75069	.88007	1.13627	39
22	.64768	.76192	.85006	1.17638	.66088	.75050	.88059	1.13561	38
23	.64790	.76173	.85057	1.17569	.66109	.75030	.88110	1.13494	37
24	.64812	.76154	.85107	1.17500	.66131	.75011	.88162	1.13428	36
25	.64834	.76135	.85157	1.17430	.66053	.74992	.88204	1.13361	35
26	.64856	.76116	.85207	1.17361	.66175	.74973	.88265	1.13295	34
27	.64878	.76097	.85257	1.17292	.66197	.74953	.88317	1.13228	33
28	.64901	.76078	.85308	1.17223	.66218	.74934	.88369	1.13162	32
29	.64923	.76059	.85358	1.17154	.66240	.74915	.88421	1.13096	31
30	.64945	.76041	.85408	1.17085	.66262	.74896	.88473	1.13029	30
31	.64967	.76022	.85458	1.17016	.66284	.74876	.88524	1.12963	29
32	.64989	.76003	.85509	1.16947	.66306	.74857	.88576	1.12897	28
33	.65011	.75984	.85559	1.16878	.66327	.74838	.88628	1.12831	27
34	.65033	.75965	.85609	1.16809	.66349	.74818	.88680	1.12765	26
35	.65055	.75946	.85660	1.16741	.66371	.74799	.88732	1.12699	25
36	.65077	.75927	.85710	1.16672	.66393	.74780	.88784	1.12633	24
37	.65100	.75908	.85761	1.16603	.66414	.74760	.88836	1.12567	23
38	.65122	.75889	.85811	1.16535	.66436	.74741	.88888	1.12501	22
39	.65144	.75870	.85862	1.16466	.66458	.74722	.88940	1.12435	21
40	.65166	.75851	.85912	1.16398	.66480	.74703	.88992	1.12369	20
41	.65188	.75832	.85963	1.16329	.66501	.74683	.89045	1.12303	19
42	.65210	.75813	.86014	1.16261	.66523	.74664	.89097	1.12238	18
43	.65232	.75794	.86064	1.16192	.66545	.74644	.89149	1.12172	17
44	.65254	.75775	.86115	1.16124	.66566	.74625	.89201	1.12106	16
45	.65276	.75756	.86166	1.16056	.66588	.74606	.89253	1.12041	15
46	.65298	.75738	.86216	1.15987	.66610	.74586	.89306	1.11975	14
47	.65320	.75719	.86267	1.15919	.66632	.74567	.89358	1.11909	13
48	.65342	.75700	.86318	1.15851	.66653	.74548	.89410	1.11844	12
49	.65364	.75680	.86368	1.15783	.66675	.74528	.89463	1.11778	11
50	.65386	.75661	.86419	1.15715	.66697	.74509	.89515	1.11713	10
51	.65408	.75642	.86470	1.15647	.66718	.74489	.89567	1.11648	9
52	.65430	.75623	.86521	1.15579	.66740	.74470	.89620	1.11582	8
53	.65452	.75604	.86572	1.15511	.66762	.74451	.89672	1.11517	7
54	.65474	.75585	.86623	1.15443	.66783	.74431	.89725	1.11452	6
55	.65496	.75566	.86674	1.15375	.66805	.74412	.89777	1.11387	5
56	.65518	.75547	.86725	1.15308	.66827	.74392	.89830	1.11321	4
57	.65540	.75528	.86776	1.15240	.66848	.74373	.89883	1.11256	3
58	.65562	.75509	.86827	1.15172	.66870	.74353	.89935	1.11191	2
59	.65584	.75490	.86878	1.15104	.66891	.74334	.89988	1.11126	1
60	.65606	.75471	.86929	1.15037	.66913	.74314	.90040	1.11061	0
'	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	'

TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS.

42°

43°

	Sin.	Cos.	Tan.	Cot.	Sin.	Cos.	Tan.	Cot.	
0	.66913	.74314	.90040	1.11061	.68200	.73135	.93252	1.07237	60
1	.66935	.74295	.90093	1.10996	.68221	.73116	.93306	1.07174	59
2	.66956	.74276	.90146	1.10931	.68242	.73096	.93360	1.07112	58
3	.66978	.74256	.90199	1.10867	.68264	.73076	.93415	1.07049	57
4	.66999	.74237	.90251	1.10802	.68285	.73056	.93469	1.06987	56
5	.67021	.74217	.90304	1.10737	.68306	.73036	.93524	1.06925	55
6	.67043	.74198	.90357	1.10672	.68327	.73016	.93578	1.06862	54
7	.67064	.74178	.90410	1.10607	.68349	.72996	.93633	1.06800	53
8	.67086	.74159	.90463	1.10543	.68370	.72976	.93688	1.06738	52
9	.67107	.74139	.90516	1.10478	.68391	.72957	.93742	1.06676	51
10	.67129	.74120	.90569	1.10414	.68412	.72937	.93797	1.06613	50
11	.67151	.74100	.90621	1.10349	.68434	.72917	.93852	1.06551	49
12	.67172	.74080	.90674	1.10285	.68455	.72897	.93906	1.06489	48
13	.67194	.74061	.90727	1.10220	.68476	.72877	.93961	1.06427	47
14	.67215	.74041	.90781	1.10156	.68497	.72857	.94016	1.06365	46
15	.67237	.74022	.90834	1.10091	.68518	.72837	.94071	1.06303	45
16	.67258	.74002	.90887	1.10027	.68539	.72817	.94125	1.06241	44
17	.67280	.73983	.90940	1.09963	.68561	.72797	.94180	1.06179	43
18	.67301	.73963	.90993	1.09899	.68582	.72777	.94235	1.06117	42
19	.67323	.73944	.91046	1.09834	.68603	.72757	.94290	1.06056	41
20	.67344	.73924	.91099	1.09770	.68624	.72737	.94345	1.05994	40
21	.67366	.73904	.91153	1.09706	.68645	.72717	.94400	1.05932	39
22	.67387	.73885	.91206	1.09642	.68666	.72697	.94455	1.05870	38
23	.67409	.73865	.91259	1.09578	.68688	.72677	.94510	1.05809	37
24	.67430	.73846	.91313	1.09514	.68709	.72657	.94565	1.05747	36
25	.67452	.73826	.91365	1.09450	.68730	.72637	.94620	1.05685	35
26	.67473	.73806	.91419	1.09386	.68751	.72617	.94676	1.05624	34
27	.67495	.73787	.91473	1.09322	.68772	.72597	.94731	1.05562	33
28	.67516	.73767	.91526	1.09258	.68793	.72577	.94786	1.05501	32
29	.67538	.73747	.91580	1.09195	.68814	.72557	.94841	1.05439	31
30	.67559	.73728	.91633	1.09131	.68835	.72537	.94896	1.05378	30
31	.67580	.73708	.91687	1.09067	.68857	.72517	.94952	1.05317	29
32	.67602	.73688	.91740	1.09003	.68878	.72497	.95007	1.05255	28
33	.67623	.73669	.91794	1.08940	.68899	.72477	.95062	1.05194	27
34	.67645	.73649	.91847	1.08876	.68920	.72457	.95118	1.05133	26
35	.67666	.73629	.91901	1.08813	.68941	.72437	.95173	1.05072	25
36	.67688	.73610	.91955	1.08749	.68962	.72417	.95229	1.05010	24
37	.67709	.73590	.92008	1.08686	.68983	.72397	.95284	1.04949	23
38	.67730	.73570	.92062	1.08622	.69004	.72377	.95340	1.04888	22
39	.67752	.73551	.92116	1.08559	.69025	.72357	.95395	1.04827	21
40	.67773	.73531	.92170	1.08496	.69046	.72337	.95451	1.04766	20
41	.67795	.73511	.92224	1.08432	.69067	.72317	.95506	1.04705	19
42	.67816	.73491	.92277	1.08369	.69088	.72297	.95562	1.04644	18
43	.67837	.73472	.92331	1.08306	.69109	.72277	.95618	1.04583	17
44	.67859	.73452	.92385	1.08243	.69130	.72257	.95673	1.04522	16
45	.67880	.73432	.92439	1.08179	.69151	.72236	.95729	1.04461	15
46	.67901	.73413	.92493	1.08116	.69172	.72216	.95785	1.04401	14
47	.67923	.73393	.92547	1.08053	.69193	.72196	.95841	1.04340	13
48	.67944	.73373	.92601	1.07990	.69214	.72176	.95897	1.04279	12
49	.67965	.73353	.92655	1.07927	.69235	.72156	.95952	1.04218	11
50	.67987	.73333	.92709	1.07864	.69256	.72136	.96008	1.04158	10
51	.68008	.73314	.92763	1.07801	.69277	.72116	.96064	1.04097	9
52	.68029	.73294	.92817	1.07738	.69298	.72095	.96120	1.04036	8
53	.68051	.73274	.92872	1.07676	.69319	.72075	.96176	1.03976	7
54	.68072	.73254	.92926	1.07613	.69340	.72055	.96232	1.03915	6
55	.68093	.73234	.92980	1.07550	.69361	.72035	.96288	1.03855	5
56	.68115	.73215	.93034	1.07487	.69382	.72015	.96344	1.03794	4
57	.68136	.73195	.93088	1.07425	.69403	.71995	.96400	1.03734	3
58	.68157	.73175	.93143	1.07362	.69424	.71974	.96457	1.03674	2
59	.68179	.73155	.93197	1.07299	.69445	.71954	.96513	1.03613	1
60	.68200	.73135	.93252	1.07237	.69466	.71934	.96569	1.03553	0
	Cos.	Sin.	Cot.	Tan.	Cos.	Sin.	Cot.	Tan.	

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TABLE IX.—NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS

44°

44°

'	Sin.	Cos.	Tan.	Cot.	'	'	Sin.	Cos.	Tan.	Cot.	'
0	.69466	.71934	.96569	1.03553	60	30	.70091	.71325	.98270	1.01761	30
1	.69487	.71914	.96625	1.03493	59	31	.70112	.71305	.98327	1.01702	29
2	.69508	.71894	.96681	1.03433	58	32	.70132	.71284	.98384	1.01642	28
3	.69529	.71873	.96738	1.03372	57	33	.70153	.71264	.98441	1.01583	27
4	.69549	.71853	.96794	1.03312	56	34	.70174	.71243	.98499	1.01524	26
5	.69570	.71833	.96850	1.03252	55	35	.70195	.71223	.98556	1.01465	25
6	.69591	.71813	.96907	1.03192	54	36	.70215	.71203	.98613	1.01406	24
7	.69612	.71792	.96963	1.03132	53	37	.70236	.71182	.98671	1.01347	23
8	.69633	.71772	.97020	1.03072	52	38	.70257	.71162	.98728	1.01288	22
9	.69654	.71752	.97076	1.03012	51	39	.70277	.71141	.98786	1.01229	21
10	.69675	.71732	.97133	1.02952	50	40	.70298	.71121	.98843	1.01170	20
11	.69696	.71711	.97189	1.02892	49	41	.70319	.71100	.98901	1.01112	19
12	.69717	.71691	.97246	1.02832	48	42	.70339	.71080	.98958	1.01053	18
13	.69737	.71671	.97302	1.02772	47	43	.70360	.71059	.99016	1.00994	17
14	.69758	.71650	.97359	1.02713	46	44	.70381	.71039	.99073	1.00935	16
15	.69779	.71630	.97416	1.02653	45	45	.70401	.71019	.99131	1.00876	15
16	.69800	.71610	.97472	1.02593	44	46	.70422	.70998	.99189	1.00818	14
17	.69821	.71590	.97529	1.02533	43	47	.70443	.70978	.99247	1.00759	13
18	.69842	.71569	.97586	1.02474	42	48	.70463	.70957	.99304	1.00701	12
19	.69862	.71549	.97643	1.02414	41	49	.70484	.70937	.99362	1.00642	11
20	.69883	.71529	.97700	1.02355	40	50	.70505	.70916	.99420	1.00583	10
21	.69904	.71508	.97756	1.02295	39	51	.70525	.70896	.99478	1.00525	9
22	.69925	.71488	.97813	1.02236	38	52	.70546	.70875	.99536	1.00467	8
23	.69946	.71468	.97870	1.02176	37	53	.70567	.70855	.99594	1.00408	7
24	.69966	.71447	.97927	1.02117	36	54	.70587	.70834	.99652	1.00350	6
25	.69987	.71427	.97984	1.02057	35	55	.70608	.70813	.99710	1.00291	5
26	.70008	.71407	.98041	1.01998	34	56	.70628	.70793	.99768	1.00233	4
27	.70029	.71386	.98098	1.01939	33	57	.70649	.70772	.99826	1.00175	3
28	.70049	.71366	.98155	1.01879	32	58	.70670	.70752	.99884	1.00116	2
29	.70070	.71345	.98213	1.01820	31	59	.70690	.70731	.99942	1.00058	1
30	.70091	.71325	.98270	1.01761	30	60	.70711	.70711	1.00000	1.00000	0
	Cos.	Sin.	Cot.	Tan.			Cos.	Sin.	Cot.	Tan.	

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TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

0°		1°		2°		3°			
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.00000	.00000	.00015	.00015	.00061	.00061	.00137	.00137	0
1	.00000	.00000	.00016	.00016	.00062	.00062	.00139	.00139	1
2	.00000	.00000	.00016	.00016	.00063	.00063	.00140	.00140	2
3	.00000	.00000	.00017	.00017	.00064	.00064	.00142	.00142	3
4	.00000	.00000	.00017	.00017	.00065	.00065	.00143	.00143	4
5	.00000	.00000	.00018	.00018	.00066	.00066	.00145	.00145	5
6	.00000	.00000	.00018	.00018	.00067	.00067	.00146	.00147	6
7	.00000	.00000	.00019	.00019	.00068	.00068	.00148	.00148	7
8	.00000	.00000	.00020	.00020	.00069	.00069	.00150	.00150	8
9	.00000	.00000	.00020	.00020	.00070	.00070	.00151	.00151	9
10	.00000	.00000	.00021	.00021	.00071	.00072	.00153	.00153	10
11	.00001	.00001	.00021	.00021	.00073	.00073	.00154	.00155	11
12	.00001	.00001	.00022	.00022	.00074	.00074	.00156	.00156	12
13	.00001	.00001	.00023	.00023	.00075	.00075	.00158	.00158	13
14	.00001	.00001	.00023	.00023	.00076	.00076	.00159	.00159	14
15	.00001	.00001	.00024	.00024	.00077	.00077	.00161	.00161	15
16	.00001	.00001	.00024	.00024	.00078	.00078	.00162	.00163	16
17	.00001	.00001	.00025	.00025	.00079	.00079	.00164	.00164	17
18	.00001	.00001	.00026	.00026	.00081	.00081	.00166	.00166	18
19	.00002	.00002	.00026	.00026	.00082	.00082	.00168	.00168	19
20	.00002	.00002	.00027	.00027	.00083	.00083	.00169	.00169	20
21	.00002	.00002	.00028	.00028	.00084	.00084	.00171	.00171	21
22	.00002	.00002	.00028	.00028	.00085	.00085	.00173	.00173	22
23	.00002	.00002	.00029	.00029	.00087	.00087	.00174	.00175	23
24	.00002	.00002	.00030	.00030	.00088	.00088	.00176	.00176	24
25	.00003	.00003	.00031	.00031	.00089	.00089	.00177	.00178	25
26	.00003	.00003	.00031	.00031	.00090	.00090	.00179	.00180	26
27	.00003	.00003	.00032	.00032	.00091	.00091	.00181	.00182	27
28	.00003	.00003	.00033	.00033	.00093	.00093	.00183	.00183	28
29	.00004	.00004	.00034	.00034	.00094	.00094	.00185	.00185	29
30	.00004	.00004	.00034	.00034	.00095	.00095	.00187	.00187	30
31	.00004	.00004	.00035	.00035	.00096	.00097	.00188	.00189	31
32	.00004	.00004	.00036	.00036	.00098	.00098	.00190	.00190	32
33	.00005	.00005	.00037	.00037	.00099	.00099	.00192	.00192	33
34	.00005	.00005	.00037	.00037	.00100	.00100	.00194	.00194	34
35	.00005	.00005	.00038	.00038	.00102	.00102	.00196	.00196	35
36	.00005	.00005	.00039	.00039	.00103	.00103	.00197	.00198	36
37	.00006	.00006	.00040	.00040	.00104	.00104	.00199	.00200	37
38	.00006	.00006	.00041	.00041	.00106	.00106	.00201	.00201	38
39	.00006	.00006	.00041	.00041	.00107	.00107	.00203	.00203	39
40	.00007	.00007	.00042	.00042	.00108	.00108	.00205	.00205	40
41	.00007	.00007	.00043	.00043	.00110	.00110	.00207	.00207	41
42	.00007	.00007	.00044	.00044	.00111	.00111	.00208	.00209	42
43	.00008	.00008	.00045	.00045	.00112	.00113	.00210	.00211	43
44	.00008	.00008	.00046	.00046	.00114	.00114	.00212	.00213	44
45	.00009	.00009	.00047	.00047	.00115	.00115	.00214	.00215	45
46	.00009	.00009	.00048	.00048	.00117	.00117	.00216	.00216	46
47	.00009	.00009	.00048	.00048	.00118	.00118	.00218	.00218	47
47	.00010	.00010	.00049	.00049	.00119	.00120	.00220	.00220	48
49	.00010	.00010	.00050	.00050	.00121	.00121	.00222	.00222	49
50	.00011	.00011	.00051	.00051	.00122	.00122	.00224	.00224	50
51	.00011	.00011	.00052	.00052	.00124	.00124	.00226	.00226	51
52	.00011	.00011	.00053	.00053	.00125	.00125	.00228	.00228	52
53	.00012	.00012	.00054	.00054	.00127	.00127	.00230	.00230	53
54	.00012	.00012	.00055	.00055	.00128	.00128	.00232	.00232	54
55	.00013	.00013	.00056	.00056	.00130	.00130	.00234	.00234	55
56	.00013	.00013	.00057	.00057	.00131	.00131	.00236	.00236	56
57	.00014	.00014	.00058	.00058	.00133	.00133	.00238	.00238	57
58	.00014	.00014	.00059	.00059	.00134	.00134	.00240	.00240	58
59	.00015	.00015	.00060	.00060	.00136	.00136	.00242	.00242	59
60	.00015	.00015	.00061	.00061	.00137	.00137	.00244	.00244	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	4°		5°		6°		7°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.00244	.00244	.00381	.00382	.00548	.00551	.00745	.00751	0
1	.00246	.00246	.00383	.00385	.00551	.00554	.00749	.00755	1
2	.00248	.00248	.00386	.00387	.00554	.00557	.00752	.00758	2
3	.00250	.00250	.00388	.00390	.00557	.00560	.00756	.00762	3
4	.00252	.00252	.00391	.00392	.00560	.00563	.00760	.00765	4
5	.00254	.00254	.00393	.00395	.00563	.00566	.00763	.00769	5
6	.00256	.00257	.00396	.00397	.00566	.00569	.00767	.00773	6
7	.00258	.00259	.00398	.00400	.00569	.00573	.00770	.00776	7
8	.00260	.00261	.00401	.00403	.00572	.00576	.00774	.00780	8
9	.00262	.00263	.00404	.00405	.00576	.00579	.00778	.00784	9
10	.00264	.00265	.00406	.00408	.00579	.00582	.00781	.00787	10
11	.00266	.00267	.00409	.00411	.00582	.00585	.00785	.00791	11
12	.00269	.00269	.00412	.00413	.00585	.00588	.00789	.00795	12
13	.00271	.00271	.00414	.00416	.00588	.00592	.00792	.00799	13
14	.00273	.00274	.00417	.00419	.00591	.00595	.00796	.00802	14
15	.00275	.00276	.00420	.00421	.00594	.00598	.00800	.00806	15
16	.00277	.00278	.00422	.00424	.00598	.00601	.00803	.00810	16
17	.00279	.00280	.00425	.00427	.00601	.00604	.00807	.00813	17
18	.00281	.00282	.00428	.00429	.00604	.00608	.00811	.00817	18
19	.00284	.00284	.00430	.00432	.00607	.00611	.00814	.00821	19
20	.00286	.00287	.00433	.00435	.00610	.00614	.00818	.00825	20
21	.00288	.00289	.00436	.00438	.00614	.00617	.00822	.00828	21
22	.00290	.00291	.00438	.00440	.00617	.00621	.00825	.00832	22
23	.00293	.00293	.00441	.00443	.00620	.00624	.00829	.00836	23
24	.00295	.00296	.00444	.00446	.00623	.00627	.00833	.00840	24
25	.00297	.00298	.00447	.00449	.00626	.00630	.00837	.00844	25
26	.00299	.00300	.00449	.00451	.00630	.00634	.00840	.00848	26
27	.00301	.00302	.00452	.00454	.00633	.00637	.00844	.00851	27
28	.00304	.00305	.00455	.00457	.00636	.00640	.00848	.00855	28
29	.00306	.00307	.00458	.00460	.00640	.00644	.00852	.00859	29
30	.00308	.00309	.00460	.00463	.00643	.00647	.00856	.00863	30
31	.00311	.00312	.00463	.00465	.00646	.00650	.00859	.00867	31
32	.00313	.00314	.00466	.00468	.00649	.00654	.00863	.00871	32
33	.00315	.00316	.00469	.00471	.00653	.00657	.00867	.00875	33
34	.00317	.00318	.00472	.00474	.00656	.00660	.00871	.00878	34
35	.00320	.00321	.00474	.00477	.00659	.00664	.00875	.00882	35
36	.00322	.00323	.00477	.00480	.00663	.00667	.00878	.00886	36
37	.00324	.00326	.00480	.00482	.00666	.00671	.00882	.00890	37
38	.00327	.00328	.00483	.00485	.00669	.00674	.00886	.00894	38
39	.00329	.00330	.00486	.00488	.00673	.00677	.00890	.00898	39
40	.00332	.00333	.00489	.00491	.00676	.00681	.00894	.00902	40
41	.00334	.00335	.00492	.00494	.00680	.00684	.00898	.00906	41
42	.00336	.00337	.00494	.00497	.00683	.00688	.00902	.00910	42
43	.00339	.00340	.00497	.00500	.00686	.00691	.00906	.00914	43
44	.00341	.00342	.00500	.00503	.00690	.00695	.00909	.00918	44
45	.00343	.00345	.00503	.00506	.00693	.00698	.00913	.00922	45
46	.00346	.00347	.00506	.00509	.00697	.00701	.00917	.00926	46
47	.00348	.00350	.00509	.00512	.00700	.00705	.00921	.00930	47
48	.00351	.00352	.00512	.00515	.00703	.00708	.00925	.00934	48
49	.00353	.00354	.00515	.00518	.00707	.00712	.00929	.00938	49
50	.00356	.00357	.00518	.00521	.00710	.00715	.00933	.00942	50
51	.00358	.00359	.00521	.00524	.00714	.00719	.00937	.00946	51
52	.00361	.00362	.00524	.00527	.00717	.00722	.00941	.00950	52
53	.00363	.00364	.00527	.00530	.00721	.00726	.00945	.00954	53
54	.00365	.00367	.00530	.00533	.00724	.00730	.00949	.00958	54
55	.00368	.00369	.00533	.00536	.00728	.00733	.00953	.00962	55
56	.00370	.00372	.00536	.00539	.00731	.00737	.00957	.00966	56
57	.00373	.00374	.00539	.00542	.00735	.00740	.00961	.00970	57
58	.00375	.00377	.00542	.00545	.00738	.00744	.00965	.00975	58
59	.00378	.00379	.00545	.00548	.00742	.00747	.00969	.00979	59
60	.00381	.00382	.00548	.00551	.00745	.00751	.00973	.00983	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	8°	9°	10°	11°	
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.
0	.00973	.00983	.01231	.01247	.01519
1	.00977	.00987	.01236	.01251	.01524
2	.00981	.00991	.01240	.01256	.01529
3	.00985	.00995	.01245	.01261	.01534
4	.00989	.00999	.01249	.01265	.01540
5	.00994	.01004	.01254	.01270	.01545
6	.00998	.01008	.01259	.01275	.01550
7	.01002	.01012	.01263	.01279	.01555
8	.01006	.01016	.01268	.01284	.01560
9	.01010	.01020	.01272	.01289	.01565
10	.01014	.01024	.01277	.01294	.01570
11	.01018	.01029	.01282	.01298	.01575
12	.01022	.01033	.01286	.01303	.01580
13	.01027	.01037	.01291	.01308	.01586
14	.01031	.01041	.01296	.01313	.01591
15	.01035	.01046	.01300	.01318	.01596
16	.01039	.01050	.01305	.01322	.01601
17	.01043	.01054	.01310	.01327	.01606
18	.01047	.01059	.01314	.01332	.01612
19	.01052	.01063	.01319	.01337	.01617
20	.01056	.01067	.01324	.01342	.01622
21	.01060	.01071	.01329	.01346	.01627
22	.01064	.01076	.01333	.01351	.01632
23	.01069	.01080	.01338	.01356	.01638
24	.01073	.01084	.01343	.01361	.01643
25	.01077	.01089	.01348	.01366	.01648
26	.01081	.01093	.01352	.01371	.01653
27	.01086	.01097	.01357	.01376	.01659
28	.01090	.01102	.01362	.01381	.01664
29	.01094	.01106	.01367	.01386	.01669
30	.01098	.01111	.01371	.01391	.01675
31	.01103	.01115	.01376	.01395	.01680
32	.01107	.01119	.01381	.01400	.01685
33	.01111	.01124	.01386	.01405	.01690
34	.01116	.01128	.01391	.01410	.01696
35	.01120	.01133	.01396	.01415	.01701
36	.01124	.01137	.01400	.01420	.01706
37	.01129	.01142	.01405	.01425	.01712
38	.01133	.01146	.01410	.01430	.01717
39	.01137	.01151	.01415	.01435	.01723
40	.01142	.01155	.01420	.01440	.01728
41	.01146	.01160	.01425	.01445	.01733
42	.01151	.01164	.01430	.01450	.01739
43	.01155	.01169	.01435	.01455	.01744
44	.01159	.01173	.01439	.01461	.01750
45	.01164	.01178	.01444	.01466	.01755
46	.01168	.01182	.01449	.01471	.01760
47	.01173	.01187	.01454	.01476	.01766
48	.01177	.01191	.01459	.01481	.01771
49	.01182	.01196	.01464	.01486	.01777
50	.01186	.01200	.01469	.01491	.01782
51	.01191	.01205	.01474	.01496	.01788
52	.01195	.01209	.01479	.01501	.01793
53	.01200	.01214	.01484	.01506	.01799
54	.01204	.01219	.01489	.01512	.01804
55	.01209	.01223	.01494	.01517	.01810
56	.01213	.01228	.01499	.01522	.01815
57	.01218	.01233	.01504	.01527	.01821
58	.01222	.01237	.01509	.01532	.01826
59	.01227	.01242	.01514	.01537	.01832
60	.01231	.01247	.01519	.01543	.01837
					.01543
					.01548
					.01553
					.01558
					.01564
					.01569
					.01574
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					.01769
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					.02228
					.02234

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

12°			13°		14°		15°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.02185	.02234	.02563	.02630	.02970	.03061	.03407	.03528	0
1	.02191	.02240	.02570	.02637	.02977	.03069	.03415	.03536	1
2	.02197	.02247	.02576	.02644	.02985	.03076	.03422	.03544	2
3	.02203	.02253	.02583	.02651	.02992	.03084	.03430	.03552	3
4	.02210	.02259	.02589	.02658	.02999	.03091	.03438	.03560	4
5	.02216	.02266	.02596	.02665	.03006	.03099	.03445	.03568	5
6	.02222	.02272	.02602	.02672	.03013	.03106	.03453	.03576	6
7	.02228	.02279	.02609	.02679	.03020	.03114	.03460	.03584	7
8	.02234	.02285	.02616	.02686	.03027	.03121	.03468	.03592	8
9	.02240	.02291	.02622	.02693	.03034	.03129	.03476	.03601	9
10	.02246	.02298	.02629	.02700	.03041	.03137	.03483	.03609	10
11	.02252	.02304	.02635	.02707	.03048	.03144	.03491	.03617	11
12	.02258	.02311	.02642	.02714	.03055	.03152	.03498	.03625	12
13	.02265	.02317	.02649	.02721	.03063	.03159	.03506	.03633	13
14	.02271	.02323	.02655	.02728	.03070	.03167	.03514	.03642	14
15	.02277	.02330	.02662	.02735	.03077	.03175	.03521	.03650	15
16	.02283	.02336	.02669	.02742	.03084	.03182	.03529	.03658	16
17	.02289	.02343	.02675	.02749	.03091	.03190	.03537	.03666	17
18	.02295	.02349	.02682	.02756	.03098	.03198	.03544	.03674	18
19	.02302	.02356	.02689	.02763	.03106	.03205	.03552	.03683	19
20	.02308	.02362	.02696	.02770	.03113	.03213	.03560	.03691	20
21	.02314	.02369	.02702	.02777	.03120	.03221	.03567	.03699	21
22	.02320	.02375	.02709	.02784	.03127	.03228	.03575	.03708	22
23	.02327	.02382	.02716	.02791	.03134	.03236	.03583	.03716	23
24	.02333	.02388	.02722	.02799	.03142	.03244	.03590	.03724	24
25	.02339	.02395	.02729	.02806	.03149	.03251	.03598	.03732	25
26	.02345	.02402	.02736	.02813	.03156	.03259	.03606	.03741	26
27	.02352	.02408	.02743	.02820	.03163	.03267	.03614	.03749	27
28	.02358	.02415	.02749	.02827	.03171	.03275	.03621	.03758	28
29	.02364	.02421	.02756	.02834	.03178	.03282	.03629	.03766	29
30	.02370	.02428	.02763	.02842	.03185	.03290	.03637	.03774	30
31	.02377	.02435	.02770	.02849	.03193	.03298	.03645	.03783	31
32	.02383	.02441	.02777	.02856	.03200	.03306	.03653	.03791	32
33	.02389	.02448	.02783	.02863	.03207	.03313	.03660	.03799	33
34	.02396	.02454	.02790	.02870	.03214	.03321	.03668	.03808	34
35	.02402	.02461	.02797	.02878	.03222	.03329	.03676	.03816	35
36	.02408	.02468	.02804	.02885	.03229	.03337	.03684	.03825	36
37	.02415	.02474	.02811	.02892	.03236	.03345	.03692	.03833	37
38	.02421	.02481	.02818	.02899	.03244	.03353	.03699	.03842	38
39	.02427	.02488	.02824	.02907	.03251	.03360	.03707	.03850	39
40	.02434	.02494	.02831	.02914	.03258	.03368	.03715	.03858	40
41	.02440	.02501	.02838	.02921	.03266	.03376	.03723	.03867	41
42	.02447	.02508	.02845	.02928	.03273	.03384	.03731	.03875	42
43	.02453	.02515	.02852	.02936	.03281	.03392	.03739	.03884	43
44	.02459	.02521	.02859	.02943	.03288	.03400	.03747	.03892	44
45	.02466	.02528	.02866	.02950	.03295	.03408	.03754	.03901	45
46	.02472	.02535	.02873	.02958	.03303	.03416	.03762	.03909	46
47	.02479	.02542	.02880	.02965	.03310	.03424	.03770	.03918	47
48	.02485	.02548	.02887	.02972	.03318	.03432	.03778	.03927	48
49	.02492	.02555	.02894	.02980	.03325	.03439	.03786	.03935	49
50	.02498	.02562	.02900	.02987	.03333	.03447	.03794	.03944	50
51	.02504	.02569	.02907	.02994	.03340	.03455	.03802	.03952	51
52	.02511	.02576	.02914	.03002	.03347	.03463	.03810	.03961	52
53	.02517	.02582	.02921	.03009	.03355	.03471	.03818	.03969	53
54	.02524	.02589	.02928	.03017	.03362	.03479	.03826	.03978	54
55	.02530	.02596	.02935	.03024	.03370	.03487	.03834	.03987	55
56	.02537	.02603	.02942	.03032	.03377	.03495	.03842	.03995	56
57	.02543	.02610	.02949	.03039	.03385	.03503	.03850	.04004	57
58	.02550	.02617	.02956	.03046	.03392	.03512	.03858	.04013	58
59	.02556	.02624	.02963	.03054	.03400	.03520	.03866	.04021	59
60	.02563	.02630	.02970	.03061	.03407	.03528	.03874	.04030	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	16°		17°		18°		19°		
	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.03874	.04030	.04370	.04569	.04894	.05146	.05448	.05762	0
1	.03882	.04039	.04378	.04578	.04903	.05156	.05458	.05773	1
2	.03890	.04047	.04387	.04588	.04912	.05166	.05467	.05783	2
3	.03898	.04056	.04395	.04597	.04921	.05176	.05477	.05794	3
4	.03906	.04065	.04404	.04606	.04930	.05186	.05486	.05805	4
5	.03914	.04073	.04412	.04616	.04939	.05196	.05496	.05815	5
6	.03922	.04082	.04421	.04625	.04948	.05206	.05505	.05826	6
7	.03930	.04091	.04429	.04635	.04957	.05216	.05515	.05836	7
8	.03938	.04100	.04438	.04644	.04967	.05226	.05524	.05847	8
9	.03946	.04108	.04446	.04653	.04976	.05236	.05534	.05858	9
10	.03954	.04117	.04455	.04663	.04985	.05246	.05543	.05869	10
11	.03963	.04126	.04464	.04672	.04994	.05256	.05553	.05879	11
12	.03971	.04135	.04472	.04682	.05003	.05266	.05562	.05890	12
13	.03979	.04144	.04481	.04691	.05012	.05276	.05572	.05901	13
14	.03987	.04152	.04489	.04700	.05021	.05286	.05582	.05911	14
15	.03995	.04161	.04498	.04710	.05030	.05297	.05591	.05922	15
16	.04003	.04170	.04507	.04719	.05039	.05307	.05601	.05933	16
17	.04011	.04179	.04515	.04729	.05048	.05317	.05610	.05944	17
18	.04019	.04188	.04524	.04738	.05057	.05327	.05620	.05955	18
19	.04028	.04197	.04533	.04748	.05067	.05337	.05630	.05965	19
20	.04036	.04206	.04541	.04757	.05076	.05347	.05639	.05976	20
21	.04044	.04214	.04550	.04767	.05085	.05357	.05649	.05987	21
22	.04052	.04223	.04559	.04776	.05094	.05367	.05658	.05998	22
23	.04060	.04232	.04567	.04786	.05103	.05378	.05668	.06009	23
24	.04069	.04241	.04576	.04795	.05112	.05388	.05678	.06020	24
25	.04077	.04250	.04585	.04805	.05122	.05398	.05687	.06030	25
26	.04085	.04259	.04593	.04815	.05131	.05408	.05697	.06041	26
27	.04093	.04268	.04602	.04824	.05140	.05418	.05707	.06052	27
28	.04102	.04277	.04611	.04834	.05149	.05429	.05716	.06063	28
29	.04110	.04286	.04620	.04843	.05158	.05439	.05726	.06074	29
30	.04118	.04295	.04628	.04853	.05168	.05449	.05736	.06085	30
31	.04126	.04304	.04637	.04863	.05177	.05460	.05746	.06096	31
32	.04135	.04313	.04646	.04872	.05186	.05470	.05755	.06107	32
33	.04143	.04322	.04655	.04882	.05195	.05480	.05765	.06118	33
34	.04151	.04331	.04663	.04891	.05205	.05490	.05775	.06129	34
35	.04159	.04340	.04672	.04901	.05214	.05501	.05785	.06140	35
36	.04168	.04349	.04681	.04911	.05223	.05511	.05794	.06151	36
37	.04176	.04358	.04690	.04920	.05232	.05521	.05804	.06162	37
38	.04184	.04367	.04699	.04930	.05242	.05532	.05814	.06173	38
39	.04193	.04376	.04707	.04940	.05251	.05542	.05824	.06184	39
40	.04201	.04385	.04716	.04950	.05260	.05552	.05833	.06195	40
41	.04209	.04394	.04725	.04959	.05270	.05563	.05843	.06206	41
42	.04218	.04403	.04734	.04969	.05279	.05573	.05853	.06217	42
43	.04226	.04413	.04743	.04979	.05288	.05584	.05863	.06228	43
44	.04234	.04422	.04752	.04989	.05298	.05594	.05873	.06239	44
45	.04243	.04431	.04760	.04998	.05307	.05604	.05882	.06250	45
46	.04251	.04440	.04769	.05008	.05316	.05615	.05892	.06261	46
47	.04260	.04449	.04778	.05018	.05326	.05625	.05902	.06272	47
48	.04268	.04458	.04787	.05028	.05335	.05636	.05912	.06283	48
49	.04276	.04468	.04796	.05038	.05344	.05646	.05922	.06295	49
50	.04285	.04477	.04805	.05047	.05354	.05657	.05932	.06306	50
51	.04293	.04486	.04814	.05057	.05363	.05667	.05942	.06317	51
52	.04302	.04495	.04823	.05067	.05373	.05678	.05951	.06328	52
53	.04310	.04504	.04832	.05077	.05382	.05688	.05961	.06339	53
54	.04319	.04514	.04841	.05087	.05391	.05699	.05971	.06350	54
55	.04327	.04523	.04850	.05097	.05401	.05709	.05981	.06362	55
56	.04336	.04532	.04858	.05107	.05410	.05720	.05991	.06373	56
57	.04344	.04541	.04867	.05116	.05420	.05730	.06001	.06384	57
58	.04353	.04551	.04876	.05126	.05429	.05741	.06011	.06395	58
59	.04361	.04560	.04885	.05136	.05439	.05751	.06021	.06407	59
60	.04370	.04569	.04894	.05146	.05448	.05762	.06031	.06418	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	20°		21°		22°		23°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.06031	.06418	.06642	.07115	.07282	.07853	.07950	.08636	0
1	.06041	.06429	.06652	.07126	.07293	.07866	.07961	.08649	1
2	.06051	.06440	.06663	.07138	.07303	.07879	.07972	.08663	2
3	.06061	.06452	.06673	.07150	.07314	.07892	.07984	.08676	3
4	.06071	.06463	.06684	.07162	.07325	.07904	.07995	.08690	4
5	.06081	.06474	.06694	.07174	.07336	.07917	.08006	.08703	5
6	.06091	.06486	.06705	.07186	.07347	.07930	.08018	.08717	6
7	.06101	.06497	.06715	.07199	.07358	.07943	.08029	.08730	7
8	.06111	.06508	.06726	.07211	.07369	.07955	.08041	.08744	8
9	.06121	.06520	.06736	.07223	.07380	.07968	.08052	.08757	9
10	.06131	.06531	.06747	.07235	.07391	.07981	.08064	.08771	10
11	.06141	.06542	.06757	.07247	.07402	.07994	.08075	.08784	11
12	.06151	.06554	.06768	.07259	.07413	.08006	.08086	.08798	12
13	.06161	.06565	.06778	.07271	.07424	.08019	.08098	.08811	13
14	.06171	.06577	.06789	.07283	.07435	.08032	.08109	.08825	14
15	.06181	.06588	.06799	.07295	.07446	.08045	.08121	.08839	15
16	.06191	.06600	.06810	.07307	.07457	.08058	.08132	.08852	16
17	.06201	.06611	.06820	.07320	.07468	.08071	.08144	.08866	17
18	.06211	.06622	.06831	.07332	.07479	.08084	.08155	.08880	18
19	.06221	.06634	.06841	.07344	.07490	.08097	.08167	.08893	19
20	.06231	.06645	.06852	.07356	.07501	.08109	.08178	.08907	20
21	.06241	.06657	.06863	.07368	.07512	.08122	.08190	.08921	21
22	.06252	.06668	.06873	.07380	.07523	.08135	.08201	.08934	22
23	.06262	.06680	.06884	.07393	.07534	.08148	.08213	.08948	23
24	.06272	.06691	.06894	.07405	.07545	.08161	.08225	.08962	24
25	.06282	.06703	.06905	.07417	.07556	.08174	.08236	.08975	25
26	.06292	.06715	.06916	.07429	.07568	.08087	.08248	.08989	26
27	.06302	.06726	.06926	.07442	.07579	.08200	.08259	.09003	27
28	.06312	.06738	.06937	.07454	.07590	.08213	.08271	.09017	28
29	.06323	.06749	.06948	.07466	.07601	.08226	.08282	.09030	29
30	.06333	.06761	.06958	.07479	.07612	.08239	.08294	.09044	30
31	.06343	.06773	.06969	.07491	.07623	.08252	.08306	.09058	31
32	.06353	.06784	.06980	.07503	.07634	.08265	.08317	.09072	32
33	.06363	.06796	.06990	.07516	.07645	.08278	.08329	.09086	33
34	.06374	.06807	.07001	.07528	.07657	.08291	.08340	.09099	34
35	.06384	.06819	.07012	.07540	.07668	.08305	.08352	.09113	35
36	.06394	.06831	.07022	.07553	.07679	.08318	.08364	.09127	36
37	.06404	.06843	.07033	.07565	.07690	.08331	.08375	.09141	37
38	.06415	.06854	.07044	.07578	.07701	.08344	.08387	.09155	38
39	.06425	.06866	.07055	.07590	.07713	.08357	.08399	.09169	39
40	.06435	.06878	.07065	.07602	.07724	.08370	.08410	.09183	40
41	.06445	.06889	.07076	.07615	.07735	.08383	.08422	.09197	41
42	.06456	.06901	.07087	.07627	.07746	.08397	.08434	.09211	42
43	.06466	.06913	.07098	.07640	.07757	.08410	.08445	.09224	43
44	.06476	.06925	.07108	.07652	.07769	.08423	.08457	.09238	44
45	.06486	.06936	.07119	.07665	.07780	.08436	.08469	.09252	45
46	.06497	.06948	.07130	.07677	.07791	.08449	.08481	.09266	46
47	.06507	.06960	.07141	.07690	.07802	.08463	.08492	.09280	47
48	.06517	.06972	.07151	.07702	.07814	.08476	.08504	.09294	48
49	.06528	.06984	.07162	.07715	.07825	.08489	.08516	.09308	49
50	.06538	.06995	.07173	.07727	.07836	.08503	.08528	.09323	50
51	.06548	.07007	.07184	.07740	.07848	.08516	.08539	.09337	51
52	.06559	.07019	.07195	.07752	.07859	.08529	.08551	.09351	52
53	.06569	.07031	.07206	.07765	.07870	.08542	.08563	.09365	53
54	.06580	.07043	.07216	.07778	.07881	.08556	.08575	.09379	54
55	.06590	.07055	.07227	.07790	.07893	.08569	.08586	.09393	55
56	.06600	.07067	.07238	.07803	.07904	.08582	.08598	.09407	56
57	.06611	.07079	.07249	.07816	.07915	.08596	.08610	.09421	57
58	.06621	.07091	.07260	.07828	.07927	.08609	.08622	.09435	58
59	.06632	.07103	.07271	.07841	.07938	.08623	.08634	.09449	59
60	.06642	.07115	.07282	.07853	.07950	.08636	.08645	.09464	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	24°		25°		26°		27°		
	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.08645	.09464	.09369	.10338	.10121	.11260	.10899	.12233	0
1	.08657	.09478	.09382	.10353	.10133	.11276	.10913	.12245	1
2	.08669	.09492	.09394	.10368	.10146	.11292	.10926	.12266	2
3	.08681	.09506	.09406	.10383	.10159	.11308	.10939	.12283	3
4	.08693	.09520	.09418	.10398	.10172	.11323	.10952	.12299	4
5	.08705	.09535	.09431	.10413	.10184	.11339	.10965	.12316	5
6	.08717	.09549	.09443	.10428	.10197	.11355	.10979	.12333	6
7	.08728	.09563	.09455	.10443	.10210	.11371	.10992	.12349	7
8	.08740	.09577	.09468	.10458	.10223	.11387	.11005	.12366	8
9	.08752	.09592	.09480	.10473	.10236	.11403	.11019	.12383	9
10	.08764	.09606	.09493	.10488	.10248	.11419	.11032	.12400	10
11	.08776	.09620	.09505	.10503	.10261	.11435	.11045	.12416	11
12	.08788	.09635	.09517	.10518	.10274	.11451	.11058	.12433	12
13	.08800	.09649	.09530	.10533	.10287	.11467	.11072	.12450	13
14	.08812	.09663	.09542	.10549	.10300	.11483	.11085	.12467	14
15	.08824	.09678	.09554	.10564	.10313	.11499	.11098	.12484	15
16	.08836	.09692	.09567	.10579	.10326	.11515	.11112	.12501	16
17	.08848	.09707	.09579	.10594	.10338	.11531	.11125	.12518	17
18	.08860	.09721	.09592	.10609	.10351	.11547	.11138	.12534	18
19	.08872	.09735	.09604	.10625	.10364	.11563	.11152	.12551	19
20	.08884	.09750	.09617	.10640	.10377	.11579	.11165	.12568	20
21	.08896	.09764	.09629	.10655	.10390	.11595	.11178	.12585	21
22	.08908	.09779	.09642	.10670	.10403	.11611	.11192	.12602	22
23	.08920	.09793	.09654	.10686	.10416	.11627	.11205	.12619	23
24	.08932	.09808	.09666	.10701	.10429	.11643	.11218	.12636	24
25	.08944	.09822	.09679	.10716	.10442	.11659	.11232	.12653	25
26	.08956	.09837	.09691	.10731	.10455	.11675	.11245	.12670	26
27	.08968	.09851	.09704	.10747	.10468	.11691	.11259	.12687	27
28	.08980	.09866	.09716	.10762	.10481	.11708	.11272	.12704	28
29	.08992	.09880	.09729	.10777	.10494	.11724	.11285	.12721	29
30	.09004	.09895	.09741	.10793	.10507	.11740	.11299	.12738	30
31	.09016	.09909	.09754	.10808	.10520	.11756	.11312	.12755	31
32	.09028	.09924	.09767	.10824	.10533	.11772	.11326	.12772	32
33	.09040	.09939	.09779	.10839	.10546	.11789	.11339	.12789	33
34	.09052	.09953	.09792	.10854	.10559	.11805	.11353	.12807	34
35	.09064	.09968	.09804	.10870	.10572	.11821	.11366	.12824	35
36	.09076	.09982	.09817	.10885	.10585	.11838	.11380	.12841	36
37	.09089	.09997	.09829	.10901	.10598	.11854	.11393	.12858	37
38	.09101	.10012	.09842	.10916	.10611	.11870	.11407	.12875	38
39	.09113	.10026	.09854	.10932	.10624	.11886	.11420	.12892	39
40	.09125	.10041	.09867	.10947	.10637	.11903	.11434	.12910	40
41	.09137	.10055	.09880	.10963	.10650	.11919	.11447	.12927	41
42	.09149	.10071	.09892	.10978	.10663	.11936	.11461	.12944	42
43	.09161	.10085	.09905	.10994	.10676	.11952	.11474	.12961	43
44	.09174	.10100	.09918	.11009	.10689	.11968	.11488	.12979	44
45	.09186	.10115	.09930	.11025	.10702	.11985	.11501	.12996	45
46	.09198	.10130	.09943	.11041	.10715	.12001	.11515	.13013	46
47	.09210	.10144	.09955	.11056	.10728	.12018	.11528	.13031	47
48	.09222	.10159	.09968	.11072	.10741	.12034	.11542	.13048	48
49	.09234	.10174	.09981	.11087	.10755	.12051	.11555	.13065	49
50	.09247	.10189	.09993	.11103	.10768	.12067	.11569	.13083	50
51	.09259	.10204	.10006	.11119	.10781	.12084	.11583	.13100	51
52	.09271	.10218	.10019	.11134	.10794	.12100	.11596	.13117	52
53	.09283	.10233	.10032	.11150	.10807	.12117	.11610	.13135	53
54	.09296	.10248	.10044	.11166	.10820	.12133	.11623	.13152	54
55	.09308	.10263	.10057	.11181	.10833	.12150	.11637	.13170	55
56	.09320	.10278	.10070	.11197	.10847	.12166	.11651	.13187	56
57	.09332	.10293	.10082	.11213	.10860	.12183	.11664	.13205	57
58	.09345	.10308	.10095	.11229	.10873	.12199	.11678	.13222	58
59	.09357	.10323	.10108	.11244	.10886	.12216	.11692	.13240	59
60	.09369	.10338	.10121	.11260	.10899	.12233	.11705	.13257	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

28°		29°		30°		31°			
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.11705	.13257	.12538	.14335	.13397	.15470	.14283	.16663	0
1	.11719	.13275	.12552	.14354	.13412	.15489	.14298	.16684	1
2	.11733	.13292	.12566	.14372	.13427	.15509	.14313	.16704	2
3	.11746	.13310	.12580	.14391	.13441	.15528	.14328	.16725	3
4	.11760	.13327	.12595	.14409	.13456	.15548	.14343	.16745	4
5	.11774	.13345	.12609	.14428	.13470	.15567	.14358	.16766	5
6	.11787	.13362	.12623	.14446	.13485	.15587	.14373	.16786	6
7	.11801	.13380	.12637	.14465	.13499	.15606	.14388	.16806	7
8	.11815	.13398	.12651	.14483	.13514	.15626	.14403	.16827	8
9	.11828	.13415	.12665	.14502	.13529	.15645	.14418	.16848	9
10	.11842	.13433	.12679	.14521	.13543	.15665	.14433	.16868	10
11	.11856	.13451	.12694	.14539	.13558	.15684	.14449	.16889	11
12	.11870	.13468	.12708	.14558	.13573	.15704	.14464	.16909	12
13	.11883	.13486	.12722	.14576	.13587	.15724	.14479	.16930	13
14	.11897	.13504	.12736	.14595	.13602	.15743	.14494	.16950	14
15	.11911	.13521	.12750	.14614	.13616	.15763	.14509	.16971	15
16	.11925	.13539	.12765	.14632	.13631	.15782	.14524	.16992	16
17	.11938	.13557	.12779	.14651	.13646	.15802	.14539	.17012	17
18	.11952	.13575	.12793	.14670	.13660	.15822	.14554	.17033	18
19	.11966	.13593	.12807	.14689	.13675	.15841	.14569	.17054	19
20	.11980	.13610	.12822	.14707	.13690	.15861	.14584	.17075	20
21	.11994	.13628	.12836	.14726	.13705	.15881	.14599	.17095	21
22	.12007	.13646	.12850	.14745	.13719	.15901	.14615	.17116	22
23	.12021	.13664	.12864	.14764	.13734	.15920	.14630	.17137	23
24	.12035	.13682	.12879	.14782	.13749	.15940	.14645	.17158	24
25	.12049	.13700	.12893	.14801	.13763	.15960	.14660	.17178	25
26	.12063	.13718	.12907	.14820	.13778	.15980	.14675	.17199	26
27	.12077	.13735	.12921	.14839	.13793	.16000	.14690	.17220	27
28	.12091	.13753	.12936	.14858	.13808	.16019	.14706	.17241	28
29	.12104	.13771	.12950	.14877	.13822	.16039	.14721	.17262	29
30	.12118	.13789	.12964	.14896	.13837	.16059	.14736	.17283	30
31	.12132	.13807	.12979	.14914	.13852	.16079	.14751	.17304	31
32	.12146	.13825	.12993	.14933	.13867	.16099	.14766	.17325	32
33	.12160	.13843	.13007	.14952	.13881	.16119	.14782	.17346	33
34	.12174	.13861	.13022	.14971	.13896	.16139	.14797	.17367	34
35	.12188	.13879	.13036	.14990	.13911	.16159	.14812	.17388	35
36	.12202	.13897	.13051	.15009	.13926	.16179	.14827	.17409	36
37	.12216	.13916	.13065	.15028	.13941	.16199	.14843	.17430	37
38	.12230	.13934	.13079	.15047	.13955	.16219	.14858	.17451	38
39	.12244	.13952	.13094	.15066	.13970	.16239	.14873	.17472	39
40	.12257	.13970	.13108	.15085	.13985	.16259	.14888	.17493	40
41	.12271	.13988	.13122	.15105	.14000	.16279	.14904	.17514	41
42	.12285	.14006	.13137	.15124	.14015	.16299	.14919	.17535	42
43	.12299	.14024	.13151	.15143	.14030	.16319	.14934	.17556	43
44	.12313	.14042	.13166	.15162	.14044	.16339	.14949	.17577	44
45	.12327	.14061	.13180	.15181	.14059	.16359	.14965	.17598	45
46	.12341	.14079	.13195	.15200	.14074	.16380	.14980	.17620	46
47	.12355	.14097	.13209	.15219	.14089	.16400	.14995	.17641	47
48	.12369	.14115	.13223	.15239	.14104	.16420	.15011	.17662	48
49	.12383	.14134	.13238	.15258	.14119	.16440	.15026	.17683	49
50	.12397	.14152	.13252	.15277	.14134	.16460	.15041	.17704	50
51	.12411	.14170	.13267	.15296	.14149	.16481	.15057	.17726	51
52	.12425	.14188	.13281	.15315	.14164	.16501	.15072	.17747	52
53	.12439	.14207	.13296	.15335	.14179	.16521	.15087	.17768	53
54	.12454	.14225	.13310	.15354	.14194	.16541	.15103	.17790	54
55	.12468	.14243	.13325	.15373	.14208	.16562	.15118	.17811	55
56	.12482	.14262	.13339	.15393	.14223	.16582	.15134	.17832	56
57	.12496	.14280	.13354	.15412	.14238	.16602	.15149	.17854	57
58	.12510	.14299	.13368	.15431	.14253	.16623	.15164	.17875	58
59	.12524	.14317	.13383	.15451	.14268	.16643	.15180	.17896	59
60	.12538	.14335	.13397	.15470	.14283	.16663	.15195	.17918	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	32°		33°		34°		35°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.15195	.17918	.16133	.19236	.17096	.20622	.18085	.22077	0
1	.15211	.17939	.16149	.19259	.17113	.20645	.18101	.22102	1
2	.15226	.17961	.16165	.19281	.17129	.20669	.18118	.22127	2
3	.15241	.17982	.16181	.19304	.17145	.20693	.18135	.22152	3
4	.15257	.18004	.16196	.19327	.17161	.20717	.18152	.22177	4
5	.15272	.18025	.16212	.19349	.17178	.20740	.18168	.22202	5
6	.15288	.18047	.16228	.19372	.17194	.20764	.18185	.22227	6
7	.15303	.18068	.16244	.19394	.17210	.20788	.18202	.22252	7
8	.15319	.18090	.16260	.19417	.17227	.20812	.18218	.22277	8
9	.15334	.18111	.16276	.19440	.17243	.20836	.18235	.22302	9
10	.15350	.18133	.16292	.19463	.17259	.20859	.18252	.22327	10
11	.15365	.18155	.16308	.19485	.17276	.20883	.18269	.22352	11
12	.15381	.18176	.16324	.19508	.17292	.20907	.18286	.22377	12
13	.15396	.18198	.16340	.19531	.17308	.20931	.18302	.22402	13
14	.15412	.18220	.16355	.19554	.17325	.20955	.18319	.22428	14
15	.15427	.18241	.16371	.19576	.17341	.20979	.18336	.22453	15
16	.15443	.18263	.16387	.19599	.17357	.21003	.18353	.22478	16
17	.15458	.18285	.16403	.19622	.17374	.21027	.18369	.22503	17
18	.15474	.18307	.16419	.19645	.17390	.21051	.18386	.22528	18
19	.15489	.18328	.16435	.19668	.17407	.21075	.18403	.22554	19
20	.15505	.18350	.16451	.19691	.17423	.21099	.18420	.22579	20
21	.15520	.18372	.16467	.19713	.17439	.21123	.18437	.22604	21
22	.15536	.18394	.16483	.19736	.17456	.21147	.18454	.22629	22
23	.15552	.18416	.16499	.19759	.17472	.21171	.18470	.22655	23
24	.15567	.18437	.16515	.19782	.17489	.21195	.18487	.22680	24
25	.15583	.18459	.16531	.19805	.17505	.21220	.18504	.22706	25
26	.15598	.18481	.16547	.19828	.17522	.21244	.18521	.22731	26
27	.15614	.18503	.16563	.19851	.17538	.21268	.18538	.22756	27
28	.15630	.18525	.16579	.19874	.17554	.21292	.18555	.22782	28
29	.15645	.18547	.16595	.19897	.17571	.21316	.18572	.22807	29
30	.15661	.18569	.16611	.19920	.17587	.21341	.18588	.22833	30
31	.15676	.18591	.16627	.19944	.17604	.21365	.18605	.22858	31
32	.15692	.18613	.16644	.19967	.17620	.21389	.18622	.22884	32
33	.15708	.18635	.16660	.19990	.17637	.21414	.18639	.22909	33
34	.15723	.18657	.16676	.20013	.17653	.21438	.18656	.22935	34
35	.15739	.18679	.16692	.20036	.17670	.21462	.18673	.22960	35
36	.15755	.18701	.16708	.20059	.17686	.21487	.18690	.22986	36
37	.15770	.18723	.16724	.20083	.17703	.21511	.18707	.23012	37
38	.15786	.18745	.16740	.20106	.17719	.21535	.18724	.23037	38
39	.15802	.18767	.16756	.20129	.17736	.21560	.18741	.23063	39
40	.15818	.18790	.16772	.20152	.17752	.21584	.18758	.23089	40
41	.15833	.18812	.16788	.20176	.17769	.21609	.18775	.23114	41
42	.15849	.18834	.16805	.20199	.17786	.21633	.18792	.23140	42
43	.15865	.18856	.16821	.20222	.17802	.21658	.18809	.23166	43
44	.15880	.18878	.16837	.20246	.17819	.21682	.18826	.23192	44
45	.15896	.18901	.16853	.20269	.17835	.21707	.18843	.23217	45
46	.15912	.18923	.16869	.20292	.17852	.21731	.18860	.23243	46
47	.15928	.18945	.16885	.20316	.17868	.21756	.18877	.23269	47
48	.15943	.18967	.16902	.20339	.17885	.21781	.18894	.23295	48
49	.15959	.18990	.16918	.20363	.17902	.21805	.18911	.23321	49
50	.15975	.19012	.16934	.20386	.17918	.21830	.18928	.23347	50
51	.15991	.19034	.16950	.20410	.17935	.21855	.18945	.23373	51
52	.16006	.19057	.16966	.20433	.17952	.21879	.18962	.23399	52
53	.16022	.19079	.16983	.20457	.17968	.21904	.18979	.23424	53
54	.16038	.19102	.16999	.20480	.17985	.21929	.18996	.23450	54
55	.16054	.19124	.17015	.20504	.18001	.21953	.19013	.23476	55
56	.16070	.19146	.17031	.20527	.18018	.21978	.19030	.23502	56
57	.16085	.19169	.17047	.20551	.18035	.22003	.19047	.23529	57
58	.16101	.19191	.17064	.20575	.18051	.22028	.19064	.23555	58
59	.16117	.19214	.17080	.20598	.18068	.22053	.19081	.23581	59
60	.16133	.19236	.17096	.20622	.18085	.22077	.19098	.23607	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS

	36°	37°	38°	39°	
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.
0	.19098	.23607	.20136	.25214	.21199
1	.19115	.23633	.20154	.25241	.21217
2	.19133	.23659	.20171	.25269	.21235
3	.19150	.23685	.20189	.25296	.21253
4	.19167	.23711	.20207	.25324	.21271
5	.19184	.23738	.20224	.25351	.21289
6	.19201	.23764	.20242	.25379	.21307
7	.19218	.23790	.20259	.25406	.21324
8	.19235	.23816	.20277	.25434	.21342
9	.19252	.23843	.20294	.25462	.21360
10	.19270	.23869	.20312	.25489	.21378
11	.19287	.23895	.20329	.25517	.21396
12	.19304	.23922	.20347	.25545	.21414
13	.19321	.23948	.20365	.25572	.21432
14	.19338	.23975	.20382	.25600	.21450
15	.19356	.24001	.20400	.25628	.21468
16	.19373	.24028	.20417	.25656	.21486
17	.19390	.24054	.20435	.25683	.21504
18	.19407	.24081	.20453	.25711	.21522
19	.19424	.24107	.20470	.25739	.21540
20	.19442	.24134	.20488	.25767	.21558
21	.19459	.24160	.20506	.25795	.21576
22	.19476	.24187	.20523	.25823	.21595
23	.19493	.24213	.20541	.25851	.21613
24	.19511	.24240	.20559	.25879	.21631
25	.19528	.24267	.20576	.25907	.21649
26	.19545	.24293	.20594	.25935	.21667
27	.19562	.24320	.20612	.25963	.21685
28	.19580	.24347	.20629	.25991	.21703
29	.19597	.24373	.20647	.26019	.21721
30	.19614	.24400	.20665	.26047	.21739
31	.19632	.24427	.20682	.26075	.21757
32	.19649	.24454	.20700	.26104	.21775
33	.19666	.24481	.20718	.26132	.21794
34	.19684	.24508	.20736	.26160	.21812
35	.19701	.24534	.20753	.26188	.21830
36	.19718	.24561	.20771	.26216	.21848
37	.19736	.24588	.20789	.26245	.21866
38	.19753	.24615	.20807	.26273	.21884
39	.19770	.24642	.20824	.26301	.21902
40	.19788	.24669	.20842	.26330	.21921
41	.19805	.24696	.20860	.26358	.21939
42	.19822	.24723	.20878	.26387	.21957
43	.19840	.24750	.20895	.26415	.21975
44	.19857	.24777	.20913	.26443	.21993
45	.19875	.24804	.20931	.26472	.22012
46	.19892	.24832	.20949	.26500	.22030
47	.19909	.24859	.20967	.26529	.22048
48	.19927	.24886	.20985	.26557	.22066
49	.19944	.24913	.21002	.26586	.22084
50	.19962	.24940	.21020	.26615	.22103
51	.19979	.24967	.21038	.26643	.22121
52	.19997	.24995	.21056	.26672	.22139
53	.20014	.25022	.21074	.26701	.22157
54	.20032	.25049	.21092	.26729	.22176
55	.20049	.25077	.21109	.26758	.22194
56	.20066	.25104	.21127	.26787	.22212
57	.20084	.25131	.21145	.26815	.22231
58	.20101	.25159	.21163	.26844	.22249
59	.20119	.25186	.21181	.26873	.22267
60	.20136	.25214	.21199	.26902	.22285
					.22876
					.22304
					.22322
					.22340
					.22359
					.22377
					.22395
					.22414
					.22432
					.22450
					.22469
					.22487
					.22506
					.22524
					.22542
					.22561
					.22579
					.22598
					.22616
					.22634
					.22653
					.22671
					.22690
					.22708
					.22727
					.22745
					.22764
					.22782
					.22801
					.22819
					.22838
					.22856
					.22875
					.22893
					.22912
					.22930
					.22949
					.22967
					.22986
					.23004
					.23023
					.23041
					.23060
					.23079
					.23097
					.23116
					.23134
					.23153
					.23172
					.23190
					.23209
					.23228
					.23246
					.23265
					.23283
					.23302
					.23321
					.23339
					.23358
					.23377
					.23396
					.23413
					.23445
					.23477
					.23509
					.23541

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	40°		41°		42°		43°		
	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.23396	.30541	.24529	.32501	.25686	.34563	.26865	.36733	0
1	.23414	.30573	.24548	.32535	.25705	.34599	.26884	.36770	1
2	.23433	.30605	.24567	.32568	.25724	.34634	.26904	.36807	2
3	.23452	.30636	.24586	.32602	.25744	.34669	.26924	.36844	3
4	.23470	.30668	.24605	.32636	.25763	.34704	.26944	.36881	4
5	.23489	.30700	.24625	.32669	.25783	.34740	.26964	.36919	5
6	.23508	.30732	.24644	.32703	.25802	.34775	.26984	.36956	6
7	.23527	.30764	.24663	.32737	.25822	.34811	.27004	.36993	7
8	.23545	.30796	.24682	.32770	.25841	.34846	.27024	.37030	8
9	.23564	.30829	.24701	.32804	.25861	.34882	.27043	.37068	9
10	.23583	.30861	.24720	.32838	.25880	.34917	.27063	.37105	10
11	.23602	.30893	.24739	.32872	.25900	.34953	.27083	.37143	11
12	.23620	.30925	.24759	.32905	.25920	.34988	.27103	.37180	12
13	.23639	.30957	.24778	.32939	.25939	.35024	.27123	.37218	13
14	.23658	.30989	.24797	.32973	.25959	.35060	.27143	.37255	14
15	.23677	.31022	.24816	.33007	.25978	.35095	.27163	.37293	15
16	.23696	.31054	.24835	.33041	.25998	.35131	.27183	.37330	16
17	.23714	.31086	.24854	.33075	.26017	.35167	.27203	.37368	17
18	.23733	.31119	.24874	.33109	.26037	.35203	.27223	.37406	18
19	.23752	.31151	.24893	.33143	.26056	.35238	.27243	.37443	19
20	.23771	.31183	.24912	.33177	.26076	.35274	.27263	.37481	20
21	.23790	.31216	.24931	.33211	.26096	.35310	.27283	.37519	21
22	.23808	.31248	.24950	.33245	.26115	.35346	.27303	.37556	22
23	.23827	.31281	.24970	.33279	.26135	.35382	.27323	.37594	23
24	.23846	.31313	.24989	.33314	.26154	.35418	.27343	.37632	24
25	.23865	.31346	.25008	.33348	.26174	.35454	.27363	.37670	25
26	.23884	.31378	.25027	.33382	.26194	.35490	.27383	.37708	26
27	.23903	.31411	.25047	.33416	.26213	.35526	.27403	.37746	27
28	.23922	.31443	.25066	.33451	.26233	.35562	.27423	.37784	28
29	.23941	.31476	.25085	.33485	.26253	.35598	.27443	.37822	29
30	.23959	.31509	.25104	.33519	.26272	.35634	.27463	.37860	30
31	.23978	.31541	.25124	.33554	.26292	.35670	.27483	.37898	31
32	.23997	.31574	.25143	.33588	.26312	.35707	.27503	.37936	32
33	.24016	.31607	.25162	.33622	.26331	.35743	.27523	.37974	33
34	.24035	.31640	.25182	.33657	.26351	.35779	.27543	.38012	34
35	.24054	.31672	.25201	.33691	.26371	.35815	.27563	.38051	35
36	.24073	.31705	.25220	.33726	.26390	.35852	.27583	.38089	36
37	.24092	.31738	.25240	.33760	.26410	.35888	.27603	.38127	37
38	.24111	.31771	.25259	.33795	.26430	.35924	.27623	.38165	38
39	.24130	.31804	.25278	.33830	.26449	.35961	.27643	.38204	39
40	.24149	.31837	.25297	.33864	.26469	.35997	.27663	.38242	40
41	.24168	.31870	.25317	.33899	.26489	.36034	.27683	.38280	41
42	.24187	.31903	.25336	.33934	.26509	.36070	.27703	.38319	42
43	.24206	.31936	.25356	.33968	.26528	.36107	.27723	.38357	43
44	.24225	.31969	.25375	.34003	.26548	.36143	.27743	.38396	44
45	.24244	.32002	.25394	.34038	.26568	.36180	.27764	.38434	45
46	.24262	.32035	.25414	.34073	.26588	.36217	.27784	.38473	46
47	.24281	.32068	.25433	.34108	.26607	.36253	.27804	.38512	47
48	.24300	.32101	.25452	.34142	.26627	.36290	.27824	.38550	48
49	.24320	.32134	.25472	.34177	.26647	.36327	.27844	.38589	49
50	.24339	.32168	.25491	.34212	.26667	.36363	.27864	.38628	50
51	.24358	.32201	.25511	.34247	.26686	.36400	.27884	.38666	51
52	.24377	.32234	.25530	.34282	.26706	.36437	.27905	.38705	52
53	.24396	.32267	.25549	.34317	.26726	.36474	.27925	.38744	53
54	.24415	.32301	.25569	.34352	.26746	.36511	.27945	.38783	54
55	.24434	.32334	.25588	.34387	.26766	.36548	.27965	.38822	55
56	.24453	.32368	.25608	.34423	.26785	.36585	.27985	.38860	56
57	.24472	.32401	.25627	.34458	.26805	.36622	.28005	.38899	57
58	.24491	.32434	.25647	.34493	.26825	.36659	.28026	.38938	58
59	.24510	.32468	.25666	.34528	.26845	.36696	.28046	.38977	59
60	.24529	.32501	.25686	.34563	.26865	.36733	.28066	.39016	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

44°

45°

46°

47°

	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.28066	.39016	.29289	.41421	.30534	.43956	.31800	.46628	0
1	.28086	.39055	.29310	.41463	.30555	.43999	.31821	.46674	1
2	.28106	.39095	.29330	.41504	.30576	.44042	.31843	.46719	2
3	.28127	.39134	.29351	.41545	.30597	.44086	.31864	.46765	3
4	.28147	.39173	.29372	.41586	.30618	.44129	.31885	.46811	4
5	.28167	.39212	.29392	.41627	.30639	.44173	.31907	.46857	5
6	.28187	.39251	.29413	.41669	.30660	.44217	.31928	.46903	6
7	.28208	.39291	.29433	.41710	.30681	.44260	.31949	.46949	7
8	.28228	.39330	.29454	.41752	.30702	.44304	.31971	.46995	8
9	.28248	.39369	.29475	.41793	.30723	.44347	.31992	.47041	9
10	.28268	.39409	.29495	.41835	.30744	.44391	.32013	.47087	10
11	.28289	.39448	.29516	.41876	.30765	.44435	.32035	.47134	11
12	.28309	.39487	.29537	.41918	.30786	.44479	.32056	.47180	12
13	.28329	.39527	.29557	.41959	.30807	.44523	.32077	.47226	13
14	.28350	.39566	.29578	.42001	.30828	.44567	.32099	.47272	14
15	.28370	.39606	.29599	.42042	.30849	.44610	.32120	.47319	15
16	.28390	.39646	.29619	.42084	.30870	.44654	.32141	.47365	16
17	.28410	.39685	.29640	.42126	.30891	.44698	.32163	.47411	17
18	.28431	.39725	.29661	.42168	.30912	.44742	.32184	.47458	18
19	.28451	.39764	.29681	.42210	.30933	.44787	.32205	.47504	19
20	.28471	.39804	.29702	.42251	.30954	.44831	.32227	.47551	20
21	.28492	.39844	.29723	.42293	.30975	.44875	.32248	.47598	21
22	.28512	.39884	.29743	.42335	.30996	.44919	.32270	.47644	22
23	.28532	.39924	.29764	.42377	.31017	.44963	.32291	.47691	23
24	.28553	.39963	.29785	.42419	.31038	.45007	.32312	.47738	24
25	.28573	.40003	.29805	.42461	.31059	.45052	.32334	.47784	25
26	.28593	.40043	.29826	.42503	.31080	.45096	.32355	.47831	26
27	.28614	.40083	.29847	.42545	.31101	.45141	.32377	.47878	27
28	.28634	.40123	.29868	.42587	.31122	.45185	.32398	.47925	28
29	.28655	.40163	.29888	.42630	.31143	.45229	.32420	.47972	29
30	.28675	.40203	.29909	.42672	.31165	.45274	.32441	.48019	30
31	.28695	.40243	.29930	.42714	.31186	.45319	.32462	.48066	31
32	.28716	.40283	.29951	.42756	.31207	.45363	.32484	.48113	32
33	.28736	.40324	.29971	.42799	.31228	.45408	.32505	.48160	33
34	.28757	.40364	.29992	.42841	.31249	.45452	.32527	.48207	34
35	.28777	.40404	.30013	.42883	.31270	.45497	.32548	.48254	35
36	.28797	.40444	.30034	.42926	.31291	.45542	.32570	.48301	36
37	.28818	.40485	.30054	.42968	.31312	.45587	.32591	.48349	37
38	.28838	.40525	.30075	.43011	.31334	.45631	.32613	.48396	38
39	.28859	.40565	.30096	.43053	.31355	.45676	.32634	.48443	39
40	.28879	.40606	.30117	.43096	.31376	.45721	.32656	.48491	40
41	.28900	.40646	.30138	.43139	.31397	.45766	.32677	.48538	41
42	.28920	.40687	.30158	.43181	.31418	.45811	.32699	.48586	42
43	.28941	.40727	.30179	.43224	.31439	.45856	.32720	.48633	43
44	.28961	.40768	.30200	.43267	.31461	.45901	.32742	.48681	44
45	.28981	.40808	.30221	.43310	.31482	.45946	.32763	.48728	45
46	.29002	.40849	.30242	.43352	.31503	.45992	.32785	.48776	46
47	.29022	.40890	.30263	.43395	.31524	.46037	.32806	.48824	47
48	.29043	.40930	.30283	.43438	.31545	.46082	.32828	.48871	48
49	.29063	.40971	.30304	.43481	.31567	.46127	.32849	.48919	49
50	.29084	.41012	.30325	.43524	.31588	.46173	.32871	.48967	50
51	.29104	.41053	.30346	.43567	.31609	.46218	.32893	.49015	51
52	.29125	.41093	.30367	.43610	.31630	.46263	.32914	.49063	52
53	.29145	.41134	.30388	.43653	.31651	.46309	.32936	.49111	53
54	.29166	.41175	.30409	.43696	.31673	.46354	.32957	.49159	54
55	.29187	.41216	.30430	.43739	.31694	.46400	.32979	.49207	55
56	.29207	.41257	.30451	.43783	.31715	.46445	.33001	.49255	56
57	.29228	.41298	.30471	.43826	.31736	.46491	.33022	.49303	57
58	.29248	.41339	.30492	.43869	.31758	.46537	.33044	.49351	58
59	.29269	.41380	.30513	.43912	.31779	.46582	.33065	.49399	59
60	.29289	.41421	.30534	.43956	.31800	.46628	.33087	.49448	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS

	48°		49°		50°		51°		
	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.33087	.49448	.34394	.52425	.35721	.55572	.37068	.58902	0
1	.33109	.49496	.34416	.52476	.35744	.55626	.37091	.58959	1
2	.33130	.49544	.34438	.52527	.35766	.55680	.37113	.59016	2
3	.33152	.49593	.34460	.52579	.35788	.55734	.37136	.59073	3
4	.33173	.49641	.34482	.52630	.35810	.55789	.37158	.59130	4
5	.33195	.49690	.34504	.52681	.35833	.55843	.37181	.59188	5
6	.33217	.49738	.34526	.52732	.35855	.55897	.37204	.59245	6
7	.33238	.49787	.34548	.52784	.35877	.55951	.37226	.59302	7
8	.33260	.49835	.34570	.52835	.35900	.56005	.37249	.59360	8
9	.33282	.49884	.34592	.52886	.35922	.56060	.37272	.59418	9
10	.33303	.49933	.34614	.52938	.35944	.56114	.37294	.59475	10
11	.33325	.49981	.34636	.52989	.35967	.56169	.37317	.59533	11
12	.33347	.50030	.34658	.53041	.35989	.56223	.37340	.59590	12
13	.33368	.50079	.34680	.53092	.36011	.56278	.37362	.59648	13
14	.33390	.50128	.34702	.53144	.36034	.56332	.37385	.59706	14
15	.33412	.50177	.34724	.53196	.36056	.56387	.37408	.59764	15
16	.33434	.50226	.34746	.53247	.36078	.56442	.37430	.59822	16
17	.33455	.50275	.34768	.53299	.36101	.56497	.37453	.59880	17
18	.33477	.50324	.34790	.53351	.36123	.56551	.37476	.59938	18
19	.33499	.50373	.34812	.53403	.36146	.56606	.37498	.59996	19
20	.33520	.50422	.34834	.53455	.36168	.56661	.37521	.60054	20
21	.33542	.50471	.34856	.53507	.36190	.56716	.37544	.60112	21
22	.33564	.50521	.34878	.53559	.36213	.56771	.37567	.60171	22
23	.33586	.50570	.34900	.53611	.36235	.56826	.37589	.60229	23
24	.33607	.50619	.34923	.53663	.36258	.56881	.37612	.60287	24
25	.33629	.50669	.34945	.53715	.36280	.56937	.37635	.60346	25
26	.33651	.50718	.34967	.53768	.36302	.56992	.37658	.60404	26
27	.33673	.50767	.34989	.53820	.36325	.57047	.37680	.60463	27
28	.33694	.50817	.35011	.53872	.36347	.57103	.37703	.60521	28
29	.33716	.50866	.35033	.53924	.36370	.57158	.37726	.60580	29
30	.33738	.50916	.35055	.53977	.36392	.57213	.37749	.60639	30
31	.33760	.50966	.35077	.54029	.36415	.57269	.37771	.60698	31
32	.33782	.51015	.35099	.54082	.36437	.57324	.37794	.60756	32
33	.33803	.51065	.35122	.54134	.36460	.57380	.37817	.60815	33
34	.33825	.51115	.35144	.54187	.36482	.57436	.37840	.60874	34
35	.33847	.51165	.35166	.54240	.36504	.57491	.37862	.60933	35
36	.33869	.51215	.35188	.54292	.36527	.57547	.37885	.60992	36
37	.33891	.51265	.35210	.54345	.36549	.57603	.37908	.61051	37
38	.33912	.51314	.35232	.54398	.36572	.57659	.37931	.61111	38
39	.33934	.51364	.35254	.54451	.36594	.57715	.37954	.61170	39
40	.33956	.51415	.35277	.54504	.36617	.57771	.37976	.61229	40
41	.33978	.51465	.35299	.54557	.36639	.57827	.37999	.61288	41
42	.34000	.51515	.35321	.54610	.36662	.57883	.38022	.61348	42
43	.34022	.51565	.35343	.54663	.36684	.57939	.38045	.61407	43
44	.34044	.51615	.35365	.54716	.36707	.57995	.38068	.61467	44
45	.34065	.51665	.35388	.54769	.36729	.58051	.38091	.61526	45
46	.34087	.51716	.35410	.54822	.36752	.58108	.38113	.61586	46
47	.34109	.51766	.35432	.54876	.36775	.58164	.38136	.61646	47
48	.34131	.51817	.35454	.54929	.36797	.58221	.38159	.61705	48
49	.34153	.51867	.35476	.54982	.36820	.58277	.38182	.61765	49
50	.34175	.51918	.35499	.55036	.36842	.58333	.38205	.61825	50
51	.34197	.51968	.35521	.55089	.36865	.58390	.38228	.61885	51
52	.34219	.52019	.35543	.55143	.36887	.58447	.38251	.61945	52
53	.34241	.52069	.35565	.55196	.36910	.58503	.38274	.62005	53
54	.34262	.52120	.35588	.55250	.36932	.58560	.38296	.62065	54
55	.34284	.52171	.35610	.55303	.36955	.58617	.38319	.62125	55
56	.34306	.52222	.35632	.55357	.36978	.58674	.38342	.62185	56
57	.34328	.52273	.35654	.55411	.37000	.58731	.38365	.62246	57
58	.34350	.52323	.35677	.55465	.37023	.58788	.38388	.62306	58
59	.34372	.52374	.35699	.55518	.37045	.58845	.38411	.62366	59
60	.34394	.52425	.35721	.55572	.37068	.58902	.38434	.62427	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	52°		53°		54°		55°		
	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.38434	.62427	.39819	.66164	.41221	.70130	.42642	.74345	0
1	.38457	.62487	.39842	.66228	.41245	.70198	.42668	.74417	1
2	.38480	.62548	.39865	.66292	.41269	.70267	.42690	.74490	2
3	.38503	.62609	.39888	.66357	.41292	.70335	.42714	.74562	3
4	.38526	.62669	.39911	.66421	.41316	.70403	.42738	.74635	4
5	.38549	.62730	.39935	.66486	.41339	.70472	.42762	.74708	5
6	.38571	.62791	.39958	.66550	.41363	.70540	.42785	.74781	6
7	.38594	.62852	.39981	.66615	.41386	.70609	.42809	.74854	7
8	.38617	.62913	.40005	.66679	.41410	.70677	.42833	.74927	8
9	.38640	.62974	.40028	.66744	.41433	.70746	.42857	.75000	9
10	.38663	.63035	.40051	.66809	.41457	.70815	.42881	.75073	10
11	.38686	.63096	.40074	.66873	.41481	.70884	.42905	.75146	11
12	.38709	.63157	.40098	.66938	.41504	.70953	.42929	.75219	12
13	.38732	.63218	.40121	.67003	.41528	.71022	.42953	.75293	13
14	.38755	.63279	.40144	.67068	.41551	.71091	.42976	.75366	14
15	.38778	.63341	.40168	.67133	.41575	.71160	.43000	.75440	15
16	.38801	.63402	.40191	.67199	.41599	.71229	.43024	.75513	16
17	.38824	.63464	.40214	.67264	.41622	.71298	.43048	.75587	17
18	.38847	.63525	.40237	.67329	.41646	.71368	.43072	.75661	18
19	.38870	.63587	.40261	.67394	.41670	.71437	.43096	.75734	19
20	.38893	.63648	.40284	.67460	.41693	.71506	.43120	.75808	20
21	.38916	.63710	.40307	.67525	.41717	.71576	.43144	.75882	21
22	.38939	.63772	.40331	.67591	.41740	.71646	.43168	.75956	22
23	.38962	.63834	.40354	.67656	.41764	.71715	.43192	.76031	23
24	.38985	.63895	.40378	.67722	.41788	.71785	.43216	.76105	24
25	.39009	.63957	.40401	.67788	.41811	.71855	.43240	.76179	25
26	.39032	.64019	.40424	.67853	.41835	.71925	.43264	.76253	26
27	.39055	.64081	.40448	.67919	.41859	.71995	.43287	.76328	27
28	.39078	.64144	.40471	.67985	.41882	.72065	.43311	.76402	28
29	.39101	.64206	.40494	.68051	.41906	.72135	.43335	.76477	29
30	.39124	.64268	.40518	.68117	.41930	.72205	.43359	.76552	30
31	.39147	.64330	.40541	.68183	.41953	.72275	.43383	.76626	31
32	.39170	.64393	.40565	.68250	.41977	.72346	.43407	.76701	32
33	.39193	.64455	.40588	.68316	.42001	.72416	.43431	.76776	33
34	.39216	.64518	.40611	.68382	.42024	.72487	.43455	.76851	34
35	.39239	.64580	.40635	.68449	.42048	.72557	.43479	.76926	35
36	.39262	.64643	.40658	.68515	.42072	.72628	.43503	.77001	36
37	.39286	.64705	.40682	.68582	.42096	.72698	.43527	.77077	37
38	.39309	.64768	.40705	.68648	.42119	.72769	.43551	.77152	38
39	.39332	.64831	.40728	.68715	.42143	.72840	.43575	.77227	39
40	.39355	.64894	.40752	.68782	.42167	.72911	.43599	.77303	40
41	.39378	.64957	.40775	.68848	.42191	.72982	.43623	.77378	41
42	.39401	.65020	.40799	.68915	.42214	.73053	.43647	.77454	42
43	.39424	.65083	.40822	.68982	.42238	.73124	.43671	.77530	43
44	.39447	.65146	.40846	.69049	.42262	.73195	.43695	.77606	44
45	.39471	.65209	.40869	.69116	.42285	.73267	.43720	.77681	45
46	.39494	.65272	.40893	.69183	.42309	.73338	.43744	.77757	46
47	.39517	.65336	.40916	.69250	.42333	.73409	.43768	.77833	47
48	.39540	.65399	.40939	.69318	.42357	.73481	.43792	.77910	48
49	.39563	.65462	.40963	.69385	.42381	.73552	.43816	.77986	49
50	.39586	.65526	.40986	.69452	.42404	.73624	.43840	.78062	50
51	.39610	.65589	.41010	.69520	.42428	.73696	.43864	.78138	51
52	.39633	.65653	.41033	.69587	.42452	.73768	.43888	.78215	52
53	.39656	.65717	.41057	.69655	.42476	.73840	.43912	.78291	53
54	.39679	.65780	.41080	.69723	.42499	.73911	.43936	.78368	54
55	.39702	.65844	.41104	.69790	.42523	.73983	.43960	.78445	55
56	.39726	.65908	.41127	.69858	.42547	.74056	.43984	.78521	56
57	.39749	.65972	.41151	.69926	.42571	.74128	.44008	.78598	57
58	.39772	.66036	.41174	.69994	.42595	.74200	.44032	.78675	58
59	.39795	.66100	.41198	.70062	.42619	.74272	.44057	.78752	59
60	.39819	.66164	.41221	.70130	.42642	.74345	.44081	.78829	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	56°		57°		58°		59°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.44081	.78829	.45536	.83608	.47008	.88708	.48496	.94160	0
1	.44105	.78906	.45560	.83690	.47033	.88796	.48521	.94254	1
2	.44129	.78984	.45585	.83773	.47057	.88884	.48546	.94349	2
3	.44153	.79061	.45609	.83855	.47082	.88972	.48571	.94443	3
4	.44177	.79138	.45634	.83938	.47107	.89060	.48596	.94537	4
5	.44201	.79216	.45658	.84020	.47131	.89148	.48621	.94632	5
6	.44225	.79293	.45683	.84103	.47156	.89237	.48646	.94726	6
7	.44250	.79371	.45707	.84186	.47181	.89325	.48671	.94821	7
8	.44274	.79449	.45731	.84269	.47206	.89414	.48696	.94916	8
9	.44298	.79527	.45756	.84352	.47230	.89503	.48721	.95011	9
10	.44322	.79604	.45780	.84435	.47255	.89591	.48746	.95106	10
11	.44346	.79682	.45805	.84518	.47280	.89680	.48771	.95201	11
12	.44370	.79761	.45829	.84601	.47304	.89769	.48796	.95296	12
13	.44395	.79839	.45854	.84685	.47329	.89858	.48821	.95392	13
14	.44419	.79917	.45878	.84768	.47354	.89948	.48846	.95487	14
15	.44443	.79995	.45903	.84852	.47379	.90037	.48871	.95583	15
16	.44467	.80074	.45927	.84935	.47403	.90126	.48896	.95678	16
17	.44491	.80152	.45951	.85019	.47428	.90216	.48921	.95774	17
18	.44516	.80231	.45976	.85103	.47453	.90305	.48946	.95870	18
19	.44540	.80309	.46000	.85187	.47478	.90395	.48971	.95966	19
20	.44564	.80388	.46025	.85271	.47502	.90485	.48996	.96062	20
21	.44588	.80467	.46049	.85355	.47527	.90575	.49021	.96158	21
22	.44612	.80546	.46074	.85439	.47552	.90665	.49046	.96255	22
23	.44637	.80625	.46098	.85523	.47577	.90755	.49071	.96351	23
24	.44661	.80704	.46123	.85608	.47601	.90845	.49096	.96448	24
25	.44685	.80783	.46147	.85692	.47626	.90935	.49121	.96544	25
26	.44709	.80862	.46172	.85777	.47651	.91026	.49146	.96641	26
27	.44734	.80942	.46196	.85861	.47676	.91116	.49171	.96738	27
28	.44758	.81021	.46221	.85946	.47701	.91207	.49196	.96835	28
29	.44782	.81101	.46246	.86031	.47725	.91297	.49221	.96932	29
30	.44806	.81180	.46270	.86116	.47750	.91388	.49246	.97029	30
31	.44831	.81260	.46295	.86201	.47775	.91479	.49271	.97127	31
32	.44855	.81340	.46319	.86286	.47800	.91570	.49296	.97224	32
33	.44879	.81419	.46344	.86371	.47825	.91661	.49321	.97322	33
34	.44903	.81499	.46368	.86457	.47849	.91752	.49346	.97420	34
35	.44928	.81579	.46393	.86542	.47874	.91844	.49372	.97517	35
36	.44952	.81659	.46417	.86627	.47899	.91935	.49397	.97615	36
37	.44976	.81740	.46442	.86713	.47924	.92027	.49422	.97713	37
38	.45001	.81820	.46466	.86799	.47949	.92118	.49447	.97811	38
39	.45025	.81900	.46491	.86885	.47974	.92210	.49472	.97910	39
40	.45049	.81981	.46516	.86970	.47998	.92302	.49497	.98008	40
41	.45073	.82061	.46540	.87056	.48023	.92394	.49522	.98107	41
42	.45098	.82142	.46565	.87142	.48048	.92486	.49547	.98205	42
43	.45122	.82222	.46589	.87229	.48073	.92578	.49572	.98304	43
44	.45146	.82303	.46614	.87315	.48098	.92670	.49597	.98403	44
45	.45171	.82384	.46639	.87401	.48123	.92762	.49623	.98502	45
46	.45195	.82465	.46663	.87488	.48148	.92855	.49648	.98601	46
47	.45219	.82546	.46688	.87574	.48172	.92947	.49673	.98700	47
48	.45244	.82627	.46712	.87661	.48197	.93040	.49698	.98799	48
49	.45268	.82709	.46737	.87748	.48222	.93133	.49723	.98899	49
50	.45292	.82790	.46762	.87834	.48247	.93226	.49748	.98998	50
51	.45317	.82871	.46786	.87921	.48272	.93319	.49773	.99098	51
52	.45341	.82953	.46811	.88008	.48297	.93412	.49799	.99198	52
53	.45365	.83034	.46836	.88095	.48322	.93505	.49824	.99298	53
54	.45390	.83116	.46860	.88183	.48347	.93598	.49849	.99398	54
55	.45414	.83198	.46885	.88270	.48372	.93692	.49874	.99498	55
56	.45439	.83280	.46909	.88357	.48396	.93785	.49899	.99598	56
57	.45463	.83362	.46934	.88445	.48421	.93879	.49924	.99698	57
58	.45487	.83444	.46959	.88532	.48446	.93973	.49950	.99799	58
59	.45512	.83526	.46983	.88620	.48471	.94066	.49975	.99899	59
60	.45536	.83608	.47008	.88708	.48496	.94160	.50000	1.00000	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	60°		61°		62°		63°	
	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.
0	.50000	1.00000	.51519	1.06267	.53053	1.13005	.54601	1.20269
1	.50025	1.00101	.51544	1.06375	.53079	1.13122	.54627	1.20395
2	.50050	1.00202	.51570	1.06483	.53104	1.13239	.54653	1.20521
3	.50076	1.00303	.51595	1.06592	.53130	1.13356	.54679	1.20647
4	.50101	1.00404	.51621	1.06701	.53156	1.13473	.54705	1.20773
5	.50126	1.00505	.51646	1.06809	.53181	1.13590	.54731	1.20900
6	.50151	1.00607	.51672	1.06918	.53207	1.13707	.54757	1.21026
7	.50176	1.00708	.51697	1.07027	.53233	1.13825	.54782	1.21153
8	.50202	1.00810	.51723	1.07137	.53258	1.13942	.54808	1.21280
9	.50227	1.00912	.51748	1.07246	.53284	1.14060	.54834	1.21407
10	.50252	1.01014	.51774	1.07356	.53310	1.14178	.54860	1.21535
11	.50277	1.01116	.51799	1.07465	.53336	1.14296	.54886	1.21662
12	.50303	1.01218	.51825	1.07575	.53361	1.14414	.54912	1.21790
13	.50328	1.01320	.51850	1.07685	.53387	1.14533	.54938	1.21918
14	.50353	1.01422	.51876	1.07795	.53413	1.14651	.54964	1.22045
15	.50378	1.01525	.51901	1.07905	.53439	1.14770	.54990	1.22174
16	.50404	1.01628	.51927	1.08015	.53464	1.14889	.55016	1.22302
17	.50429	1.01730	.51952	1.08126	.53490	1.15008	.55042	1.22430
18	.50454	1.01833	.51978	1.08236	.53516	1.15127	.55068	1.22559
19	.50479	1.01936	.52003	1.08347	.53542	1.15246	.55094	1.22688
20	.50505	1.02039	.52029	1.08458	.53567	1.15366	.55120	1.22817
21	.50530	1.02143	.52054	1.08569	.53593	1.15485	.55146	1.22946
22	.50555	1.02246	.52080	1.08680	.53619	1.15605	.55172	1.23075
23	.50581	1.02349	.52105	1.08791	.53645	1.15725	.55198	1.23205
24	.50606	1.02453	.52131	1.08903	.53670	1.15845	.55224	1.23334
25	.50631	1.02557	.52156	1.09014	.53696	1.15965	.55250	1.23464
26	.50656	1.02661	.52182	1.09126	.53722	1.16085	.55276	1.23594
27	.50682	1.02765	.52207	1.09238	.53748	1.16206	.55302	1.23724
28	.50707	1.02869	.52233	1.09350	.53774	1.16326	.55328	1.23855
29	.50732	1.02973	.52259	1.09462	.53799	1.16447	.55354	1.23985
30	.50758	1.03077	.52284	1.09574	.53825	1.16568	.55380	1.24116
31	.50783	1.03182	.52310	1.09686	.53851	1.16689	.55406	1.24247
32	.50808	1.03286	.52335	1.09799	.53877	1.16810	.55432	1.24378
33	.50834	1.03391	.52361	1.09911	.53903	1.16932	.55458	1.24509
34	.50859	1.03496	.52386	1.10024	.53928	1.17053	.55484	1.24640
35	.50884	1.03601	.52412	1.10137	.53954	1.17175	.55510	1.24772
36	.50910	1.03706	.52438	1.10250	.53980	1.17297	.55536	1.24903
37	.50935	1.03811	.52463	1.10363	.54006	1.17419	.55563	1.25035
38	.50960	1.03916	.52489	1.10477	.54032	1.17541	.55589	1.25167
39	.50986	1.04022	.52514	1.10590	.54058	1.17663	.55615	1.25300
40	.51011	1.04128	.52540	1.10704	.54083	1.17786	.55641	1.25432
41	.51036	1.04233	.52566	1.10817	.54109	1.17909	.55667	1.25565
42	.51062	1.04339	.52591	1.10931	.54135	1.18031	.55693	1.25697
43	.51087	1.04445	.52617	1.11045	.54161	1.18154	.55719	1.25830
44	.51113	1.04551	.52642	1.11159	.54187	1.18277	.55745	1.25963
45	.51138	1.04658	.52668	1.11274	.54213	1.18401	.55771	1.26097
46	.51163	1.04764	.52694	1.11388	.54238	1.18524	.55797	1.26230
47	.51189	1.04870	.52719	1.11503	.54264	1.18648	.55823	1.26364
48	.51214	1.04977	.52745	1.11617	.54290	1.18772	.55849	1.26498
49	.51239	1.05084	.52771	1.11732	.54316	1.18895	.55876	1.26632
50	.51265	1.05191	.52796	1.11847	.54342	1.19019	.55902	1.26766
51	.51290	1.05298	.52822	1.11963	.54368	1.19144	.55928	1.26900
52	.51316	1.05405	.52848	1.12078	.54394	1.19268	.55954	1.27035
53	.51341	1.05512	.52873	1.12193	.54420	1.19393	.55980	1.27169
54	.51366	1.05619	.52899	1.12309	.54446	1.19517	.56006	1.27304
55	.51392	1.05727	.52924	1.12425	.54471	1.19642	.56032	1.27439
56	.51417	1.05835	.52950	1.12540	.54497	1.19767	.56058	1.27574
57	.51443	1.05942	.52976	1.12657	.54523	1.19892	.56084	1.27710
58	.51468	1.06050	.53001	1.12773	.54549	1.20018	.56111	1.27845
59	.51494	1.06158	.53027	1.12889	.54575	1.20143	.56137	1.27981
60	.51519	1.06267	.53053	1.13005	.54601	1.20269	.56163	1.28117

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	64°		65°		66°		67°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.56163	1.28117	.57738	1.36620	.59326	1.45859	.60927	1.55930	0
1	.56189	1.28253	.57765	1.36768	.59353	1.46020	.60954	1.56106	1
2	.56215	1.28390	.57791	1.36916	.59379	1.46181	.60980	1.56282	2
3	.56241	1.28526	.57817	1.37064	.59406	1.46342	.61007	1.56458	3
4	.56267	1.28663	.57844	1.37212	.59433	1.46504	.61034	1.56634	4
5	.56294	1.28800	.57870	1.37361	.59459	1.46665	.61061	1.56811	5
6	.56320	1.28937	.57896	1.37509	.59486	1.46827	.61088	1.56988	6
7	.56346	1.29074	.57923	1.37658	.59512	1.46989	.61114	1.57165	7
8	.56372	1.29211	.57949	1.37808	.59539	1.47152	.61141	1.57342	8
9	.56398	1.29349	.57976	1.37957	.59566	1.47314	.61168	1.57520	9
10	.56425	1.29487	.58002	1.38107	.59592	1.47477	.61195	1.57698	10
11	.56451	1.29625	.58028	1.38256	.59619	1.47640	.61222	1.57876	11
12	.56477	1.29763	.58055	1.38406	.59645	1.47804	.61248	1.58054	12
13	.56503	1.29901	.58081	1.38556	.59672	1.47967	.61275	1.58233	13
14	.56529	1.30040	.58108	1.38707	.59699	1.48131	.61302	1.58412	14
15	.56555	1.30179	.58134	1.38857	.59725	1.48295	.61329	1.58591	15
16	.56582	1.30318	.58160	1.39008	.59752	1.48459	.61356	1.58771	16
17	.56608	1.30457	.58187	1.39159	.59779	1.48624	.61383	1.58950	17
18	.56634	1.30596	.58213	1.39311	.59805	1.48789	.61409	1.59130	18
19	.56660	1.30735	.58240	1.39462	.59832	1.48954	.61436	1.59311	19
20	.56687	1.30875	.58266	1.39614	.59859	1.49119	.61463	1.59491	20
21	.56713	1.31015	.58293	1.39766	.59885	1.49284	.61490	1.59672	21
22	.56739	1.31155	.58319	1.39918	.59912	1.49450	.61517	1.59853	22
23	.56765	1.31295	.58345	1.40070	.59938	1.49616	.61544	1.60035	23
24	.56791	1.31438	.58372	1.40222	.59965	1.49782	.61570	1.60217	24
25	.56818	1.31576	.58398	1.40375	.59992	1.49948	.61597	1.60399	25
26	.56844	1.31717	.58425	1.40528	.60018	1.50115	.61624	1.60581	26
27	.56870	1.31858	.58451	1.40681	.60045	1.50282	.61651	1.60763	27
28	.56896	1.31999	.58478	1.40835	.60072	1.50449	.61678	1.60946	28
29	.56923	1.32140	.58504	1.40988	.60098	1.50617	.61705	1.61129	29
30	.56949	1.32282	.58531	1.41142	.60125	1.50784	.61732	1.61313	30
31	.56975	1.32424	.58557	1.41296	.60152	1.50952	.61759	1.61496	31
32	.57001	1.32566	.58584	1.41450	.60178	1.51120	.61785	1.61680	32
33	.57028	1.32708	.58610	1.41605	.60205	1.51289	.61812	1.61864	33
34	.57054	1.32850	.58637	1.41760	.60232	1.51457	.61839	1.62049	34
35	.57080	1.32993	.58663	1.41914	.60259	1.51626	.61866	1.62234	35
36	.57106	1.33135	.58690	1.42070	.60285	1.51795	.61893	1.62419	36
37	.57133	1.33278	.58716	1.42225	.60312	1.51965	.61920	1.62604	37
38	.57159	1.33422	.58743	1.42380	.60339	1.52134	.61947	1.62790	38
39	.57185	1.33565	.58769	1.42536	.60365	1.52304	.61974	1.62976	39
40	.57212	1.33708	.58796	1.42692	.60392	1.52474	.62001	1.63162	40
41	.57238	1.33852	.58822	1.42848	.60419	1.52645	.62027	1.63348	41
42	.57264	1.33996	.58849	1.43005	.60445	1.52815	.62054	1.63535	42
43	.57291	1.34140	.58875	1.43162	.60472	1.52986	.62081	1.63722	43
44	.57317	1.34284	.58902	1.43318	.60499	1.53157	.62108	1.63909	44
45	.57343	1.34429	.58928	1.43476	.60526	1.53329	.62135	1.64097	45
46	.57369	1.34573	.58955	1.43633	.60552	1.53500	.62162	1.64285	46
47	.57396	1.34718	.58981	1.43790	.60579	1.53672	.62189	1.64473	47
48	.57422	1.34863	.59008	1.43948	.60606	1.53845	.62216	1.64662	48
49	.57448	1.35009	.59034	1.44106	.60633	1.54017	.62243	1.64851	49
50	.57475	1.35154	.59061	1.44264	.60659	1.54190	.62270	1.65040	50
51	.57501	1.35300	.59087	1.44423	.60686	1.54363	.62297	1.65229	51
52	.57527	1.35446	.59114	1.44582	.60713	1.54536	.62324	1.65419	52
53	.57554	1.35592	.59140	1.44741	.60740	1.54709	.62351	1.65609	53
54	.57580	1.35738	.59167	1.44900	.60766	1.54883	.62378	1.65799	54
55	.57606	1.35885	.59194	1.45059	.60793	1.55057	.62405	1.65989	55
56	.57633	1.36031	.59220	1.45219	.60820	1.55231	.62431	1.66180	56
57	.57659	1.36178	.59247	1.45378	.60847	1.55405	.62458	1.66371	57
58	.57685	1.36325	.59273	1.45539	.60873	1.55580	.62485	1.66563	58
59	.57712	1.36473	.59300	1.45699	.60900	1.55755	.62512	1.66755	59
60	.57738	1.36620	.59326	1.45859	.60927	1.55930	.62539	1.66947	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	68°		69°		70°		71°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.62539	1.66947	.64163	1.79043	.65798	1.92380	.67443	2.07155	0
1	.62566	1.67139	.64190	1.79254	.65825	1.92614	.67471	2.07415	1
2	.62593	1.67332	.64218	1.79466	.65853	1.92849	.67498	2.07675	2
3	.62620	1.67525	.64245	1.79679	.65880	1.93083	.67526	2.07936	3
4	.62647	1.67718	.64272	1.79891	.65907	1.93318	.67553	2.08197	4
5	.62674	1.67911	.64299	1.80104	.65935	1.93554	.67581	2.08459	5
6	.62701	1.68105	.64326	1.80318	.65962	1.93790	.67608	2.08721	6
7	.62728	1.68299	.64353	1.80531	.65989	1.94026	.67636	2.08983	7
8	.62755	1.68494	.64381	1.80746	.66017	1.94263	.67663	2.09246	8
9	.62782	1.68689	.64408	1.80960	.66044	1.94500	.67691	2.09510	9
10	.62809	1.68884	.64435	1.81175	.66071	1.94737	.67718	2.09774	10
11	.62836	1.69079	.64462	1.81390	.66099	1.94975	.67746	2.10038	11
12	.62863	1.69275	.64489	1.81605	.66126	1.95213	.67773	2.10303	12
13	.62890	1.69471	.64517	1.81821	.66154	1.95452	.67801	2.10568	13
14	.62917	1.69667	.64544	1.82037	.66181	1.95691	.67829	2.10834	14
15	.62944	1.69864	.64571	1.82254	.66208	1.95931	.67856	2.11101	15
16	.62971	1.70061	.64598	1.82471	.66236	1.96171	.67884	2.11367	16
17	.62998	1.70258	.64625	1.82688	.66263	1.96411	.67911	2.11635	17
18	.63025	1.70455	.64653	1.82906	.66290	1.96652	.67939	2.11903	18
19	.63052	1.70653	.64680	1.83124	.66318	1.96893	.67966	2.12171	19
20	.63079	1.70851	.64707	1.83342	.66345	1.97135	.67994	2.12440	20
21	.63106	1.71050	.64734	1.83561	.66373	1.97377	.68021	2.12709	21
22	.63133	1.71249	.64761	1.83780	.66400	1.97619	.68049	2.12979	22
23	.63161	1.71448	.64789	1.83999	.66427	1.97862	.68077	2.13249	23
24	.63188	1.71647	.64816	1.84219	.66455	1.98106	.68104	2.13520	24
25	.63215	1.71847	.64843	1.84439	.66482	1.98349	.68132	2.13791	25
26	.63242	1.72047	.64870	1.84659	.66510	1.98594	.68159	2.14063	26
27	.63269	1.72247	.64898	1.84880	.66537	1.98838	.68187	2.14335	27
28	.63296	1.72448	.64925	1.85102	.66564	1.99083	.68214	2.14608	28
29	.63323	1.72649	.64952	1.85323	.66592	1.99329	.68242	2.14881	29
30	.63350	1.72850	.64979	1.85545	.66619	1.99574	.68270	2.15155	30
31	.63377	1.73052	.65007	1.85767	.66647	1.99821	.68297	2.15429	31
32	.63404	1.73254	.65034	1.85990	.66674	2.00067	.68325	2.15704	32
33	.63431	1.73456	.65061	1.86213	.66702	2.00315	.68352	2.15979	33
34	.63458	1.73659	.65088	1.86437	.66729	2.00562	.68380	2.16255	34
35	.63485	1.73862	.65116	1.86661	.66756	2.00810	.68408	2.16531	35
36	.63512	1.74065	.65143	1.86885	.66784	2.01059	.68435	2.16808	36
37	.63539	1.74269	.65170	1.87109	.66811	2.01308	.68463	2.17085	37
38	.63566	1.74473	.65197	1.87334	.66839	2.01557	.68490	2.17363	38
39	.63594	1.74677	.65225	1.87560	.66866	2.01807	.68518	2.17641	39
40	.63621	1.74881	.65252	1.87785	.66894	2.02057	.68546	2.17920	40
41	.63648	1.75086	.65279	1.88011	.66921	2.02308	.68573	2.18199	41
42	.63675	1.75292	.65306	1.88238	.66949	2.02559	.68601	2.18479	42
43	.63702	1.75497	.65334	1.88465	.66976	2.02810	.68628	2.18759	43
44	.63729	1.75703	.65361	1.88692	.67003	2.03062	.68656	2.19040	44
45	.63756	1.75909	.65388	1.88920	.67031	2.03315	.68684	2.19322	45
46	.63783	1.76116	.65416	1.89148	.67058	2.03568	.68711	2.19604	46
47	.63810	1.76323	.65443	1.89376	.67086	2.03821	.68739	2.19886	47
48	.63838	1.76530	.65470	1.89605	.67113	2.04075	.68767	2.20169	48
49	.63865	1.76737	.65497	1.89834	.67141	2.04329	.68794	2.20453	49
50	.63892	1.76945	.65525	1.90063	.67168	2.04584	.68822	2.20737	50
51	.63919	1.77154	.65552	1.90293	.67196	2.04839	.68849	2.21021	51
52	.63946	1.77362	.65579	1.90524	.67223	2.05094	.68877	2.21306	52
53	.63973	1.77571	.65607	1.90754	.67251	2.05350	.68905	2.21592	53
54	.64000	1.77780	.65634	1.90986	.67278	2.05607	.68932	2.21878	54
55	.64027	1.77990	.65661	1.91217	.67306	2.05864	.68960	2.22165	55
56	.64055	1.78200	.65689	1.91449	.67333	2.06121	.68988	2.22452	56
57	.64082	1.78410	.65716	1.91681	.67361	2.06379	.69015	2.22740	57
58	.64109	1.78621	.65743	1.91914	.67388	2.06637	.69043	2.23028	58
59	.64136	1.78832	.65771	1.92147	.67416	2.06896	.69071	2.23317	59
60	.64163	1.79043	.65798	1.92380	.67443	2.07155	.69098	2.23607	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	72°		73°		74°		75°		
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	'
0	.69098	2.23607	.70763	2.42030	.72436	2.62796	.74118	2.86370	0
1	.69126	2.23897	.70791	2.42356	.72464	2.63164	.74146	2.86790	1
2	.69154	2.24187	.70818	2.42683	.72492	2.63533	.74174	2.87211	2
3	.69181	2.24478	.70846	2.43010	.72520	2.63903	.74202	2.87633	3
4	.69209	2.24770	.70874	2.43337	.72548	2.64274	.74231	2.88056	4
5	.69237	2.25062	.70902	2.43666	.72576	2.64645	.74259	2.88479	5
6	.69264	2.25355	.70930	2.43995	.72604	2.65018	.74287	2.88904	6
7	.69292	2.25648	.70958	2.44324	.72632	2.65391	.74315	2.89330	7
8	.69320	2.25942	.70985	2.44655	.72660	2.65765	.74343	2.89756	8
9	.69347	2.26237	.71013	2.44986	.72688	2.66140	.74371	2.90184	9
10	.69375	2.26531	.71041	2.45317	.72716	2.66515	.74399	2.90613	10
11	.69403	2.26827	.71069	2.45650	.72744	2.66892	.74427	2.91042	11
12	.69430	2.27123	.71097	2.45983	.72772	2.67269	.74455	2.91473	12
13	.69458	2.27420	.71125	2.46316	.72800	2.67647	.74484	2.91904	13
14	.69486	2.27717	.71153	2.46651	.72828	2.68025	.74512	2.92337	14
15	.69514	2.28015	.71180	2.46986	.72856	2.68405	.74540	2.92770	15
16	.69541	2.28313	.71208	2.47321	.72884	2.68785	.74568	2.93204	16
17	.69569	2.28612	.71236	2.47658	.72912	2.69167	.74596	2.93640	17
18	.69597	2.28912	.71264	2.47995	.72940	2.69549	.74624	2.94076	18
19	.69624	2.29212	.71292	2.48333	.72968	2.69931	.74652	2.94514	19
20	.69652	2.29512	.71320	2.48671	.72996	2.70315	.74680	2.94952	20
21	.69680	2.29814	.71348	2.49010	.73024	2.70700	.74709	2.95392	21
22	.69708	2.30115	.71375	2.49350	.73052	2.71085	.74737	2.95832	22
23	.69735	2.30418	.71403	2.49691	.73080	2.71471	.74765	2.96274	23
24	.69763	2.30721	.71431	2.50032	.73108	2.71858	.74793	2.96716	24
25	.69791	2.31024	.71459	2.50374	.73136	2.72246	.74821	2.97160	25
26	.69818	2.31328	.71487	2.50716	.73164	2.72635	.74849	2.97604	26
27	.69846	2.31633	.71515	2.51060	.73192	2.73024	.74878	2.98050	27
28	.69874	2.31939	.71543	2.51404	.73220	2.73414	.74906	2.98497	28
29	.69902	2.32244	.71571	2.51748	.73248	2.73806	.74934	2.98944	29
30	.69929	2.32551	.71598	2.52094	.73276	2.74198	.74962	2.99393	30
31	.69957	2.32858	.71626	2.52440	.73304	2.74591	.74990	2.99843	31
32	.69985	2.33166	.71654	2.52787	.73332	2.74984	.75018	3.00293	32
33	.70013	2.33474	.71682	2.53134	.73360	2.75379	.75047	3.00745	33
34	.70040	2.33783	.71710	2.53482	.73388	2.75775	.75075	3.01198	34
35	.70068	2.34092	.71738	2.53831	.73416	2.76171	.75103	3.01652	35
36	.70096	2.34403	.71766	2.54181	.73444	2.76568	.75131	3.02107	36
37	.70124	2.34713	.71794	2.54531	.73472	2.76966	.75159	3.02563	37
38	.70151	2.35025	.71822	2.54883	.73500	2.77365	.75187	3.03020	38
39	.70179	2.35336	.71850	2.55235	.73529	2.77765	.75216	3.03479	39
40	.70207	2.35649	.71877	2.55587	.73557	2.78166	.75244	3.03938	40
41	.70235	2.35962	.71905	2.55940	.73585	2.78568	.75272	3.04398	41
42	.70263	2.36276	.71933	2.56294	.73613	2.78970	.75300	3.04860	42
43	.70290	2.36590	.71961	2.56649	.73641	2.79374	.75328	3.05322	43
44	.70318	2.36905	.71989	2.57005	.73669	2.79778	.75356	3.05786	44
45	.70346	2.37221	.72017	2.57361	.73697	2.80183	.75385	3.06251	45
46	.70374	2.37537	.72045	2.57718	.73725	2.80589	.75413	3.06717	46
47	.70401	2.37854	.72073	2.58076	.73753	2.80996	.75441	3.07184	47
48	.70429	2.38171	.72101	2.58434	.73781	2.81404	.75469	3.07652	48
49	.70457	2.38489	.72129	2.58794	.73809	2.81813	.75497	3.08121	49
50	.70485	2.38808	.72157	2.59154	.73837	2.82223	.75526	3.08591	50
51	.70513	2.39128	.72185	2.59514	.73865	2.82633	.75554	3.09063	51
52	.70540	2.39448	.72213	2.59876	.73893	2.83045	.75582	3.09535	52
53	.70568	2.39768	.72241	2.60238	.73921	2.83457	.75610	3.10009	53
54	.70596	2.40089	.72269	2.60601	.73950	2.83871	.75639	3.10484	54
55	.70624	2.40411	.72296	2.60965	.73978	2.84285	.75667	3.10960	55
56	.70652	2.40734	.72324	2.61330	.74006	2.84700	.75695	3.11437	56
57	.70679	2.41057	.72352	2.61695	.74034	2.85116	.75723	3.11915	57
58	.70707	2.41381	.72380	2.62061	.74062	2.85533	.75751	3.12394	58
59	.70735	2.41705	.72408	2.62428	.74090	2.85951	.75780	3.12875	59
60	.70763	2.42030	.72436	2.62796	.74118	2.86370	.75808	3.13357	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	76°		77°		78°		79°		
	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.75808	3.13357	.77505	3.44541	.79209	3.80973	.80919	4.24084	0
1	.75836	3.13839	.77533	3.45102	.79237	3.81633	.80948	4.24870	1
2	.75864	3.14323	.77562	3.45664	.79266	3.82294	.80976	4.25658	2
3	.75892	3.14809	.77590	3.46228	.79294	3.82956	.81005	4.26448	3
4	.75921	3.15295	.77618	3.46793	.79323	3.83621	.81033	4.27241	4
5	.75949	3.15782	.77647	3.47360	.79351	3.84288	.81062	4.28036	5
6	.75977	3.16271	.77675	3.47928	.79380	3.84956	.81090	4.28833	6
7	.76005	3.16761	.77703	3.48498	.79408	3.85627	.81119	4.29634	7
8	.76034	3.17252	.77732	3.49069	.79437	3.86299	.81148	4.30436	8
9	.76062	3.17744	.77760	3.49642	.79465	3.86973	.81176	4.31241	9
10	.76090	3.18238	.77788	3.50216	.79493	3.87649	.81205	4.32049	10
11	.76118	3.18733	.77817	3.50791	.79522	3.88327	.81233	4.32859	11
12	.76147	3.19228	.77845	3.51368	.79550	3.89007	.81262	4.33671	12
13	.76175	3.19725	.77874	3.51947	.79579	3.89689	.81290	4.34486	13
14	.76203	3.20224	.77902	3.52527	.79607	3.90373	.81319	4.35304	14
15	.76231	3.20723	.77930	3.53109	.79636	3.91058	.81348	4.36124	15
16	.76260	3.21224	.77959	3.53692	.79664	3.91746	.81376	4.36947	16
17	.76288	3.21726	.77987	3.54277	.79693	3.92436	.81405	4.37772	17
18	.76316	3.22229	.78015	3.54863	.79721	3.93128	.81433	4.38600	18
19	.76344	3.22734	.78044	3.55451	.79750	3.93821	.81462	4.39430	19
20	.76373	3.23239	.78072	3.56041	.79778	3.94517	.81491	4.40263	20
21	.76401	3.23746	.78101	3.56632	.79807	3.95215	.81519	4.41099	21
22	.76429	3.24255	.78129	3.57224	.79835	3.95914	.81548	4.41937	22
23	.76458	3.24764	.78157	3.57819	.79864	3.96616	.81576	4.42778	23
24	.76486	3.25275	.78186	3.58414	.79892	3.97320	.81605	4.43622	24
25	.76514	3.25787	.78214	3.59012	.79921	3.98025	.81633	4.44468	25
26	.76542	3.26300	.78242	3.59611	.79949	3.98733	.81662	4.45317	26
27	.76571	3.26814	.78271	3.60211	.79978	3.99443	.81691	4.46169	27
28	.76599	3.27330	.78299	3.60813	.80006	4.00155	.81719	4.47023	28
29	.76627	3.27847	.78328	3.61417	.80035	4.00869	.81748	4.47881	29
30	.76655	3.28366	.78356	3.62023	.80063	4.01585	.81776	4.48740	30
31	.76684	3.28885	.78384	3.62630	.80092	4.02303	.81805	4.49603	31
32	.76712	3.29406	.78413	3.63238	.80120	4.03024	.81834	4.50468	32
33	.76740	3.29929	.78441	3.63849	.80149	4.03746	.81862	4.51337	33
34	.76769	3.30452	.78470	3.64461	.80177	4.04471	.81891	4.52208	34
35	.76797	3.30977	.78498	3.65074	.80206	4.05197	.81919	4.53081	35
36	.76825	3.31503	.78526	3.65690	.80234	4.05926	.81948	4.53958	36
37	.76854	3.32031	.78555	3.66307	.80263	4.06657	.81977	4.54837	37
38	.76882	3.32560	.78583	3.66925	.80291	4.07390	.82005	4.55720	38
39	.76910	3.33090	.78612	3.67545	.80320	4.08125	.82034	4.56605	39
40	.76938	3.33622	.78640	3.68167	.80348	4.08863	.82063	4.57493	40
41	.76967	3.34154	.78669	3.68791	.80377	4.09602	.82091	4.58383	41
42	.76995	3.34689	.78697	3.69417	.80405	4.10344	.82120	4.59277	42
43	.77023	3.35224	.78725	3.70044	.80434	4.11088	.82148	4.60174	43
44	.77052	3.35761	.78754	3.70673	.80462	4.11835	.82177	4.61073	44
45	.77080	3.36299	.78782	3.71303	.80491	4.12583	.82206	4.61976	45
46	.77108	3.36839	.78811	3.71935	.80520	4.13334	.82234	4.62881	46
47	.77137	3.37380	.78839	3.72569	.80548	4.14087	.82263	4.63790	47
48	.77165	3.37923	.78868	3.73205	.80577	4.14842	.82292	4.64701	48
49	.77193	3.38466	.78896	3.73843	.80605	4.15599	.82320	4.65616	49
50	.77222	3.39012	.78924	3.74482	.80634	4.16359	.82349	4.66533	50
51	.77250	3.39558	.78953	3.75123	.80662	4.17121	.82377	4.67454	51
52	.77278	3.40106	.78981	3.75766	.80691	4.17886	.82406	4.68377	52
53	.77307	3.40656	.79010	3.76411	.80719	4.18652	.82435	4.69304	53
54	.77335	3.41206	.79038	3.77057	.80748	4.19421	.82463	4.70234	54
55	.77363	3.41759	.79067	3.77705	.80776	4.20193	.82492	4.71166	55
56	.77392	3.42312	.79095	3.78355	.80805	4.20966	.82521	4.72102	56
57	.77420	3.42867	.79123	3.79007	.80833	4.21742	.82549	4.73041	57
58	.77448	3.43424	.79152	3.79661	.80862	4.22521	.82578	4.73983	58
59	.77477	3.43982	.79180	3.80316	.80891	4.23301	.82607	4.74929	59
60	.77505	3.44541	.79209	3.80973	.80919	4.24084	.82635	4.75877	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

	80°	81°	82°	83°	
'	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.
0	.82635	4.75877	.84357	5.39245	.86083
1	.82664	4.76829	.84385	5.40422	.86112
2	.82692	4.77784	.84414	5.41602	.86140
3	.82721	4.78742	.84443	5.42787	.86169
4	.82750	4.79703	.84471	5.43977	.86198
5	.82778	4.80667	.84500	5.45171	.86227
6	.82807	4.81635	.84529	5.46369	.86256
7	.82836	4.82606	.84558	5.47572	.86284
8	.82864	4.83581	.84586	5.48779	.86313
9	.82893	4.84558	.84615	5.49991	.86342
10	.82922	4.85539	.84644	5.51208	.86371
11	.82950	4.86524	.84673	5.52429	.86400
12	.82979	4.87511	.84701	5.53655	.86428
13	.83008	4.88502	.84730	5.54886	.86457
14	.83036	4.89497	.84759	5.56121	.86486
15	.83065	4.90495	.84788	5.57361	.86515
16	.83094	4.91496	.84816	5.58606	.86544
17	.83122	4.92501	.84845	5.59855	.86573
18	.83151	4.93509	.84874	5.61110	.86601
19	.83180	4.94521	.84903	5.62369	.86630
20	.83208	4.95536	.84931	5.63633	.86659
21	.83237	4.96555	.84960	5.64902	.86688
22	.83266	4.97577	.84989	5.66176	.86717
23	.83294	4.98603	.85018	5.67454	.86746
24	.83323	4.99633	.85046	5.68738	.86774
25	.83352	5.00666	.85075	5.70027	.86803
26	.83380	5.01703	.85104	5.71321	.86832
27	.83409	5.02743	.85133	5.72620	.86861
28	.83438	5.03787	.85162	5.73924	.86890
29	.83467	5.04834	.85190	5.75233	.86919
30	.83495	5.05886	.85219	5.76547	.86947
31	.83524	5.06941	.85248	5.77866	.86976
32	.83553	5.08000	.85277	5.79191	.87005
33	.83581	5.09062	.85305	5.80521	.87034
34	.83610	5.10129	.85334	5.81856	.87063
35	.83639	5.11199	.85363	5.83196	.87092
36	.83667	5.12273	.85392	5.84542	.87120
37	.83696	5.13350	.85420	5.85893	.87149
38	.83725	5.14432	.85449	5.87250	.87178
39	.83754	5.15517	.85478	5.88612	.87207
40	.83782	5.16607	.85507	5.89979	.87236
41	.83811	5.17700	.85536	5.91352	.87265
42	.83840	5.18797	.85564	5.92731	.87294
43	.83868	5.19898	.85593	5.94115	.87322
44	.83897	5.21004	.85622	5.95505	.87351
45	.83926	5.22113	.85651	5.96900	.87380
46	.83954	5.23226	.85680	5.98301	.87409
47	.83983	5.24343	.85708	5.99708	.87438
48	.84012	5.25464	.85737	6.01120	.87467
49	.84041	5.26590	.85766	6.02538	.87496
50	.84069	5.27719	.85795	6.03962	.87524
51	.84098	5.28853	.85823	6.05392	.87553
52	.84127	5.29991	.85852	6.06828	.87582
53	.84155	5.31133	.85881	6.08269	.87611
54	.84184	5.32279	.85910	6.09717	.87640
55	.84213	5.33429	.85939	6.11171	.87669
56	.84242	5.34584	.85967	6.12630	.87698
57	.84270	5.35743	.85996	6.14096	.87726
58	.84299	5.36906	.86025	6.15568	.87755
59	.84328	5.38073	.86054	6.17046	.87784
60	.84357	5.39245	.86083	6.18530	.87813
					7.20551
					.87813
					7.20551
					.87842
					7.22500
					.87871
					7.24457
					.87900
					7.26425
					.87929
					7.28402
					.87957
					7.30388
					.87986
					7.32384
					.88015
					7.34390
					.88044
					7.36405
					.88073
					7.38431
					.88102
					7.40466
					.88131
					7.42511
					.88160
					7.44566
					.88188
					7.46632
					.88217
					7.48707
					.88246
					7.50793
					.88275
					7.52889
					.88304
					7.54996
					.88333
					7.57113
					.88362
					7.59241
					.88391
					7.61379
					.88420
					7.63528
					.88449
					7.65688
					.88477
					7.67859
					.88506
					7.70041
					.88535
					7.72234
					.88564
					7.74438
					.88593
					7.76653
					.88622
					7.78880
					.88651
					7.81118
					.88680
					7.83367
					.88709
					7.85628
					.88737
					7.87901
					.88766
					7.90186
					.88795
					7.92482
					.88824
					7.94791
					.88853
					7.97111
					.88882
					7.99444
					.88911
					8.01788
					.88940
					8.04146
					.88969
					8.06515
					.88998
					8.08897
					.89027
					8.11292
					.89055
					8.13699
					.89084
					8.16120
					.89113
					8.18553
					.89142
					8.20999
					.89171
					8.23459
					.89200
					8.25931
					.89229
					8.28417
					.89258
					8.30917
					.89287
					8.33430
					.89316
					8.35957
					.89345
					8.38497
					.89374
					8.41052
					.89403
					8.43620
					.89431
					8.46203
					.89460
					8.48800
					.89489
					8.51411
					.89518
					8.54037
					.89547
					8.56677

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

84°

85°

86°

	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.89547	8.56677	.91284	10.47371	.93024	13.33559	0
1	.89576	8.59332	.91313	10.51199	.93053	13.39547	1
2	.89605	8.62002	.91342	10.55052	.93082	13.45586	2
3	.89634	8.64687	.91371	10.58932	.93111	13.51676	3
4	.89663	8.67387	.91400	10.62837	.93140	13.57817	4
5	.89692	8.70103	.91429	10.66769	.93169	13.64011	5
6	.89721	8.72833	.91458	10.70728	.93198	13.70258	6
7	.89750	8.75579	.91487	10.74714	.93227	13.76558	7
8	.89779	8.78341	.91516	10.78727	.93257	13.82913	8
9	.89808	8.81119	.91545	10.82768	.93286	13.89323	9
10	.89836	8.83912	.91574	10.86837	.93315	13.95788	10
11	.89865	8.86722	.91603	10.90934	.93344	14.02310	11
12	.89894	8.89547	.91632	10.95060	.93373	14.08890	12
13	.89923	8.92389	.91661	10.99214	.93402	14.15527	13
14	.89952	8.95248	.91690	11.03397	.93431	14.22223	14
15	.89981	8.98123	.91719	11.07610	.93460	14.28979	15
16	.90010	9.01015	.91748	11.11852	.93489	14.35795	16
17	.90039	9.03923	.91777	11.16125	.93518	14.42672	17
18	.90068	9.06849	.91806	11.20427	.93547	14.49611	18
19	.90097	9.09792	.91835	11.24761	.93576	14.56614	19
20	.90126	9.12752	.91864	11.29125	.93605	14.63679	20
21	.90155	9.15730	.91893	11.33521	.93634	14.70810	21
22	.90184	9.18725	.91922	11.37948	.93663	14.78005	22
23	.90213	9.21739	.91951	11.42408	.93692	14.85268	23
24	.90242	9.24770	.91980	11.46900	.93721	14.92597	24
25	.90271	9.27819	.92009	11.51424	.93750	14.99995	25
26	.90300	9.30887	.92038	11.55982	.93779	15.07462	26
27	.90329	9.33973	.92067	11.60572	.93808	15.14999	27
28	.90358	9.37077	.92096	11.65197	.93837	15.22607	28
29	.90386	9.40201	.92125	11.69856	.93866	15.30287	29
30	.90415	9.43343	.92154	11.74550	.93895	15.38041	30
31	.90444	9.46505	.92183	11.79278	.93924	15.45869	31
32	.90473	9.49685	.92212	11.84042	.93953	15.53772	32
33	.90502	9.52886	.92241	11.88841	.93982	15.61751	33
34	.90531	9.56106	.92270	11.93677	.94011	15.69808	34
35	.90560	9.59348	.92299	11.98549	.94040	15.77944	35
36	.90589	9.62605	.92328	12.03458	.94069	15.86159	36
37	.90618	9.65885	.92357	12.08404	.94098	15.94456	37
38	.90647	9.69186	.92386	12.13388	.94127	16.02835	38
39	.90676	9.72507	.92415	12.18411	.94156	16.11297	39
40	.90705	9.75849	.92444	12.23472	.94186	16.19843	40
41	.90734	9.79212	.92473	12.28572	.94215	16.28476	41
42	.90763	9.82596	.92502	12.33712	.94244	16.37196	42
43	.90792	9.86001	.92531	12.38891	.94273	16.46005	43
44	.90821	9.89428	.92560	12.44112	.94302	16.54903	44
45	.90850	9.92877	.92589	12.49373	.94331	16.63893	45
46	.90879	9.96348	.92618	12.54676	.94360	16.72975	46
47	.90908	9.99841	.92647	12.60021	.94389	16.82152	47
48	.90937	10.03356	.92676	12.65408	.94418	16.91424	48
49	.90966	10.06894	.92705	12.70838	.94447	17.00794	49
50	.90995	10.10455	.92734	12.76312	.94476	17.10262	50
51	.91024	10.14039	.92763	12.81829	.94505	17.19830	51
52	.91053	10.17646	.92792	12.87391	.94534	17.29501	52
53	.91082	10.21277	.92821	12.92999	.94563	17.39274	53
54	.91111	10.24932	.92850	12.98651	.94592	17.49153	54
55	.91140	10.28610	.92879	13.04350	.94621	17.59139	55
56	.91169	10.32313	.92908	13.10096	.94650	17.69233	56
57	.91197	10.36040	.92937	13.15889	.94679	17.79438	57
58	.91226	10.39792	.92966	13.21730	.94708	17.89755	58
59	.91255	10.43569	.92995	13.27620	.94737	18.00185	59
60	.91284	10.47371	.93024	13.33559	.94766	18.10732	60

TABLE X.—NATURAL VERSED SINES AND EXTERNAL SECANTS.

87°

88°

89°

	Vers.	Ex. sec.	Vers.	Ex. sec.	Vers.	Ex. sec.	
0	.94766	18.10732	.96510	27.65371	.98255	53.29869	0
1	.94795	18.21397	.96539	27.89440	.98284	57.26976	1
2	.94825	18.32182	.96568	28.13917	.98313	58.27431	2
3	.94854	18.43088	.96597	28.38812	.98342	59.31411	3
4	.94883	18.54119	.96626	28.64137	.98371	60.39105	4
5	.94912	18.65275	.96655	28.89903	.98400	61.50715	5
6	.94941	18.76560	.96684	29.16120	.98429	62.66460	6
7	.94970	18.87976	.96714	29.42802	.98458	63.86572	7
8	.94999	18.99524	.96743	29.69960	.98487	65.11304	8
9	.95028	19.11208	.96772	29.97607	.98517	66.40927	9
10	.95057	19.23028	.96801	30.25758	.98546	67.75736	10
11	.95086	19.34989	.96830	30.54425	.98575	69.16047	11
12	.95115	19.47093	.96859	30.83623	.98604	70.62285	12
13	.95144	19.59341	.96888	31.13366	.98633	72.14583	13
14	.95173	19.71737	.96917	31.43671	.98662	73.73586	14
15	.95202	19.84283	.96946	31.74554	.98691	75.39655	15
16	.95231	19.96982	.96975	32.06030	.98720	77.13274	16
17	.95260	20.09838	.97004	32.38118	.98749	78.94968	17
18	.95289	20.22852	.97033	32.70835	.98778	80.85315	18
19	.95318	20.36027	.97062	33.04199	.98807	82.84947	19
20	.95347	20.49368	.97092	33.38232	.98836	84.94561	20
21	.95377	20.62876	.97121	33.72952	.98866	87.14924	21
22	.95406	20.76555	.97150	34.08380	.98895	89.46886	22
23	.95435	20.90409	.97179	34.44539	.98924	91.91387	23
24	.95464	21.04440	.97208	34.81452	.98953	94.49471	24
25	.95493	21.18653	.97237	35.19141	.98982	97.22303	25
26	.95522	21.33050	.97266	35.57633	.99011	100.1119	26
27	.95551	21.47635	.97295	35.96953	.99040	103.1757	27
28	.95580	21.62413	.97324	36.37127	.99069	106.4311	28
29	.95609	21.77386	.97353	36.78185	.99098	109.8966	29
30	.95638	21.92559	.97382	37.20155	.99127	113.5930	30
31	.95667	22.07935	.97411	37.63068	.99156	117.5444	31
32	.95696	22.23520	.97440	38.06957	.99186	121.7780	32
33	.95725	22.39316	.97470	38.51855	.99215	126.3253	33
34	.95754	22.55328	.97499	38.97797	.99244	131.2223	34
35	.95783	22.71563	.97528	39.44820	.99273	136.5111	35
36	.95812	22.88022	.97557	39.92963	.99302	142.2406	36
37	.95842	23.04712	.97586	40.42266	.99331	148.4684	37
38	.95871	23.21637	.97615	40.92772	.99360	155.2623	38
39	.95900	23.38802	.97644	41.44525	.99389	162.7033	39
40	.95929	23.56212	.97673	41.97571	.99418	170.8883	40
41	.95958	23.73873	.97702	42.51961	.99447	179.9350	41
42	.95987	23.91790	.97731	43.07746	.99476	189.9868	42
43	.96016	24.09969	.97760	43.64980	.99505	201.2212	43
44	.96045	24.28414	.97789	44.23720	.99535	213.8600	44
45	.96074	24.47134	.97819	44.84026	.99564	228.1839	45
46	.96103	24.66132	.97848	45.45963	.99593	244.5540	46
47	.96132	24.85417	.97877	46.09596	.99622	263.4427	47
48	.96161	25.04994	.97906	46.74997	.99651	285.4795	48
49	.96190	25.24869	.97935	47.42241	.99680	311.5230	49
50	.96219	25.45051	.97964	48.11406	.99709	342.7752	50
51	.96248	25.65546	.97993	48.82576	.99738	380.9723	51
52	.96277	25.86360	.98022	49.55840	.99767	428.7187	52
53	.96307	26.07503	.98051	50.31290	.99796	490.1070	53
54	.96336	26.28981	.98080	51.09027	.99825	571.9581	54
55	.96365	26.50804	.98109	51.89156	.99855	686.5496	55
56	.96394	26.72978	.98138	52.71790	.99884	858.4369	56
57	.96423	26.95513	.98168	53.57046	.99913	1144.916	57
58	.96452	27.18417	.98197	54.45053	.99942	1717.874	58
59	.96481	27.41700	.98226	55.35946	.99971	3436.747	59
60	.96510	27.65371	.98255	56.29869	1.00000	Infinite	60

TABLE XI.—REDUCTION OF BAROMETER READING TO 32° F.

Temp. ° Fahr.	Inches.										
	26.0	26.5	27.0	27.5	28.0	28.5	29.0	29.5	30.0	30.5	31.0
45	-.039	-.039	-.040	-.041	-.042	-.042	-.043	-.044	-.045	-.045	-.046
46	.041	.042	.043	.043	.044	.045	.046	.046	.047	.048	.049
47	.043	.044	.045	.046	.047	.048	.048	.049	.050	.051	.052
48	.046	.047	.047	.048	.049	.050	.051	.052	.053	.053	.054
49	.048	.049	.050	.051	.052	.052	.054	.054	.055	.056	.057
50	.050	.051	.052	.053	.054	.055	.056	.057	.058	.059	.060
51	.053	.054	.055	.056	.057	.058	.059	.060	.061	.062	.063
52	.055	.056	.057	.058	.059	.060	.061	.062	.064	.065	.066
53	.057	.058	.060	.061	.062	.063	.064	.065	.066	.067	.068
54	.060	.061	.062	.063	.064	.065	.067	.068	.069	.070	.071
55	.062	.063	.064	.065	.066	.068	.069	.070	.071	.073	.074
56	.064	.065	.067	.068	.069	.070	.072	.073	.074	.075	.077
57	.067	.068	.069	.070	.072	.073	.075	.076	.077	.078	.080
58	.069	.070	.071	.073	.074	.076	.077	.078	.080	.081	.082
59	.072	.073	.074	.075	.077	.078	.080	.081	.083	.084	.085
60	.074	.076	.077	.078	.079	.081	.082	.084	.085	.086	.088
61	.076	.077	.079	.080	.082	.083	.085	.086	.088	.089	.091
62	.079	.080	.082	.083	.085	.086	.088	.089	.091	.092	.094
63	.081	.082	.084	.085	.087	.088	.090	.091	.093	.095	.096
64	.083	.085	.086	.088	.090	.091	.093	.094	.096	.097	.099
65	.086	.087	.089	.090	.092	.093	.095	.097	.099	.100	.102
66	.088	.089	.091	.093	.095	.096	.098	.099	.101	.103	.105
67	.090	.092	.094	.095	.097	.099	.101	.102	.104	.106	.108
68	.093	.094	.096	.098	.100	.101	.103	.105	.107	.108	.110
69	.095	.097	.099	.100	.102	.104	.106	.107	.110	.111	.113
70	.097	.099	.101	.103	.105	.106	.109	.110	.112	.114	.116
71	.100	.101	.103	.105	.107	.109	.111	.113	.115	.117	.119
72	.102	.104	.106	.108	.110	.112	.114	.116	.118	.120	.122
73	.104	.106	.108	.110	.112	.114	.116	.118	.120	.122	.124
74	.107	.109	.111	.113	.115	.117	.119	.121	.123	.125	.127
75	.109	.111	.113	.115	.117	.119	.122	.124	.126	.128	.130
76	.111	.113	.116	.118	.120	.122	.124	.126	.128	.130	.133
77	.114	.116	.118	.120	.122	.124	.127	.129	.131	.133	.136
78	.116	.118	.120	.122	.125	.127	.129	.131	.134	.136	.138
79	.118	.120	.123	.125	.127	.129	.132	.134	.137	.139	.141
80	.121	.123	.125	.127	.130	.132	.135	.137	.139	.141	.144
81	.123	.125	.128	.130	.132	.134	.137	.139	.142	.144	.147
82	.125	.128	.130	.132	.135	.137	.140	.142	.145	.147	.149
83	.128	.130	.133	.135	.138	.140	.142	.145	.147	.149	.152
84	.130	.132	.135	.138	.140	.142	.145	.147	.150	.152	.155
85	.132	.134	.137	.140	.143	.145	.148	.150	.153	.155	.158
86	.135	.137	.140	.142	.145	.148	.150	.153	.155	.158	.161
87	.137	.139	.142	.144	.148	.150	.153	.155	.158	.161	.163
88	.139	.142	.145	.147	.150	.152	.155	.158	.161	.163	.166
89	.142	.144	.147	.150	.153	.155	.158	.161	.164	.166	.169
90	.144	.147	.150	.153	.155	.158	.161	.164	.166	.169	.172
91	-.146	-.149	-.152	-.155	-.158	-.160	-.163	-.166	-.169	-.172	-.175

TABLE XII.—BAROMETRIC ELEVATIONS.*

<i>B</i>	<i>A</i>	Diff. for .01.	<i>B</i>	<i>A</i>	Diff. for .01.	<i>B</i>	<i>A</i>	Diff. for .01.
Inches.	Feet.	Feet.	Inches.	Feet.	Feet.	Inches.	Feet.	Feet.
20.0	11.047	-13.6	23.7	6.423	-11.5	27.4	2.470	-9.9
20.1	10.911	13.5	23.8	6.308	11.4	27.5	2.371	9.9
20.2	10.776	13.4	23.9	6.194	11.4	27.6	2.272	9.9
20.3	10.642	13.4	24.0	6.080	11.3	27.7	2.173	9.8
20.4	10.508	13.3	24.1	5.967	11.3	27.8	2.075	9.8
20.5	10.375	13.3	24.2	5.854	11.3	27.9	1.977	9.7
20.6	10.242	13.2	24.3	5.741	11.2	28.0	1.880	9.7
20.7	10.110	13.1	24.4	5.629	11.1	28.1	1.783	9.7
20.8	9.979	13.1	24.5	5.518	11.1	28.2	1.686	9.7
20.9	9.848	13.0	24.6	5.407	11.1	28.3	1.589	9.6
21.0	9.718	12.9	24.7	5.296	11.0	28.4	1.493	9.6
21.1	9.589	12.9	24.8	5.186	10.9	28.5	1.397	9.5
21.2	9.460	12.8	24.9	5.077	10.9	28.6	1.302	9.5
21.3	9.332	12.8	25.0	4.968	10.9	28.7	1.207	9.5
21.4	9.204	12.7	25.1	4.859	10.8	28.8	1.112	9.4
21.5	9.077	12.6	25.2	4.751	10.8	28.9	1.018	9.4
21.6	8.951	12.6	25.3	4.643	10.8	29.0	.924	9.4
21.7	8.825	12.5	25.4	4.535	10.7	29.1	.830	9.4
21.8	8.700	12.5	25.5	4.428	10.7	29.2	.736	9.3
21.9	8.575	12.4	25.6	4.321	10.6	29.3	.643	9.3
22.0	8.451	12.4	25.7	4.215	10.6	29.4	.550	9.2
22.1	8.327	12.3	25.8	4.109	10.5	29.5	.458	9.2
22.2	8.204	12.2	25.9	4.004	10.5	29.6	.366	9.2
22.3	8.082	12.2	26.0	3.899	10.4	29.7	.274	9.2
22.4	7.960	12.2	26.1	3.794	10.4	29.8	.182	9.1
22.5	7.838	12.1	26.2	3.690	10.4	29.9	.091	9.1
22.6	7.717	12.0	26.3	3.586	10.3	30.0	.0	9.1
22.7	7.597	12.0	26.4	3.483	10.3	30.1	-.91	9.0
22.8	7.477	11.9	26.5	3.380	10.3	30.2	.181	9.0
22.9	7.358	11.9	26.6	3.277	10.2	30.3	.271	9.0
23.0	7.239	11.8	26.7	3.175	10.2	30.4	.361	9.0
23.1	7.121	11.7	26.8	3.073	10.1	30.5	.451	8.9
23.2	7.004	11.7	26.9	2.972	10.1	30.6	.540	8.9
23.3	6.887	11.6	27.0	2.871	10.1	30.7	.629	8.8
23.4	6.770	11.6	27.1	2.770	10.0	30.8	.717	8.8
23.5	6.654	11.6	27.2	2.670	10.0	30.9	.805	8.8
23.6	6.538	-11.5	27.3	2.570	-10.0	31.0	-.893	-8.8
23.7	6.423		27.4	2.470				

* Compiled from Report of U. S. C. & G. Survey for 1881, App. 10 Table XI.

TABLE XIII.—COEFFICIENTS FOR CORRECTIONS FOR TEMPERATURE AND HUMIDITY.*

<i>t+t'</i>	<i>C</i>	Diff. for 1°.	<i>t+t'</i>	<i>C</i>	Diff. for 1°.	<i>t+t'</i>	<i>C</i>	Diff. for 1°.
0°	-.1024	10.9	60°	-.0380	10.7	120°	+.0262	10.6
10	.0915	10.9	70	.0273	10.7	130	.0368	10.4
20	.0806	10.8	80	.0166	10.8	140	.0472	10.3
30	.0698	10.6	90	-.0058	10.7	150	.0575	10.2
40	.0592	10.6	100	+.0049	10.7	160	.0677	10.2
50	.0486	10.6	110	.0156	10.6	170	.0779	10.0
60	-.0380		120	+.0262		180	+.0879	

* Compiled from Report of U. S. C. & G. Survey for 1881, App. 10, Tables I, IV.

1	$\sin a = \frac{1}{\operatorname{cosec} a} = \frac{\tan a}{\sqrt{1+\tan^2 a}} = \sqrt{\frac{1-\cos 2a}{2}} = \frac{1}{\sqrt{1+\cot^2 a}}$ $= \cos a \tan a = \sqrt{1-\cos^2 a} = 2 \sin \frac{1}{2}a \cos \frac{1}{2}a$ $= \frac{1+\cos a}{\cot \frac{1}{2}a} = \frac{2 \tan \frac{1}{2}a}{1+\tan^2 \frac{1}{2}a} = \operatorname{vers} a \cot \frac{1}{2}a.$
2	$\cos a = \frac{1}{\sec a} = \frac{\cot a}{\sqrt{1+\cot^2 a}} = \frac{1}{\sqrt{1+\tan^2 a}}$ $= 1 - \operatorname{vers} a = \sin a \cot a = \sqrt{1-\sin^2 a} = 2 \cos^2 \frac{1}{2}a - 1$ $= \sin a \cot \frac{1}{2}a - 1 = \cos^2 \frac{1}{2}a - \sin^2 \frac{1}{2}a = 1 - 2 \sin^2 \frac{1}{2}a.$
3	$\tan a = \frac{1}{\cot a} = \frac{\sin a}{\cos a} = \frac{\sec a}{\operatorname{cosec} a} = \frac{1}{\sqrt{\operatorname{cosec}^2 a - 1}}$ $= \operatorname{vers} 2a \operatorname{cosec} 2a = \cot a - 2 \cot 2a = \sin a \sec a$ $= \frac{\sin 2a}{1+\cos 2a} = \operatorname{exsec} a \cot \frac{1}{2}a = \operatorname{exsec} 2a \cot 2a.$
4	$\cot a = \frac{1}{\tan a} = \frac{\cos a}{\sin a} = \frac{\sin 2a}{1-\cos 2a} = \frac{1+\cos 2a}{\sin 2a}$ $= \sqrt{\operatorname{cosec}^2 a - 1} = \cot \frac{1}{2}a - \operatorname{cosec} a.$
5	$\operatorname{vers} a = 1 - \cos a = \sin a \tan \frac{1}{2}a = 2 \sin^2 \frac{1}{2}a = \cos a \operatorname{exsec} a.$
6	$\operatorname{exsec} a = \sec a - 1 = \tan a \tan \frac{1}{2}a = \operatorname{vers} a \sec a.$
7	$\sin \frac{1}{2}a = \sqrt{\frac{\operatorname{vers} a}{2}} = \frac{\sin a}{2 \cos \frac{1}{2}a} = \frac{\operatorname{vers} a \cos \frac{1}{2}a}{\sin a}.$
8	$\cos \frac{1}{2}a = \sqrt{\frac{1+\cos a}{2}} = \frac{\sin a}{2 \sin \frac{1}{2}a} = \frac{\sin a \sin \frac{1}{2}a}{\operatorname{vers} a}.$
9	$\tan \frac{1}{2}a = \operatorname{vers} a \operatorname{cosec} a = \operatorname{cosec} a - \cot a = \frac{\tan a}{1+\sec a}.$
10	$\cot \frac{1}{2}a = \frac{1+\cos a}{\sin a} = \operatorname{cosec} a + \cot a = \frac{\tan a}{\operatorname{exsec} a} = \frac{1}{\operatorname{cosec} a - \cot a}.$
11	$\operatorname{vers} \frac{1}{2}a = 1 - \sqrt{\frac{1}{2}(1+\cos a)}.$
12	$\operatorname{exsec} \frac{1}{2}a = \frac{1}{\sqrt{\frac{1}{2}(1+\cos a)}} - 1.$

- 13 $\sin 2a = 2 \sin a \cos a = \frac{2 \tan a}{1 + \tan^2 a}.$
- 14 $\cos 2a = \cos^2 a - \sin^2 a = 1 - 2 \sin^2 a = 2 \cos^2 a - 1$
 $= \frac{1 - \tan^2 a}{1 + \tan^2 a}.$
- 15 $\tan 2a = \frac{2 \tan a}{1 - \tan^2 a}.$
- 16 $\cot 2a = \frac{1}{2} \cot a - \frac{1}{2} \tan a = \frac{\cot^2 a - 1}{2 \cot a} = \frac{1 - \tan^2 a}{2 \tan a}.$
- 17 $\text{vers } 2a = 2 \sin^2 a = 1 - \cos 2a = 2 \sin a \cos a \tan a.$
- 18 $\text{exsec } 2a = \frac{\tan 2a}{\cot a} = \frac{2 \tan^2 a}{1 - \tan^2 a} = \frac{2 \sin^2 a}{1 - 2 \sin^2 a}.$
- 19 $\sin (a \pm b) = \sin a \cos b \pm \cos a \sin b.$
- 20 $\cos (a \pm b) = \cos a \cos b \mp \sin a \sin b.$
- 21 $\sin a + \sin b = 2 \sin \frac{1}{2}(a+b) \cos \frac{1}{2}(a-b).$
- 22 $\sin a - \sin b = 2 \sin \frac{1}{2}(a-b) \cos \frac{1}{2}(a+b).$
- 23 $\cos a + \cos b = 2 \cos \frac{1}{2}(a+b) \cos \frac{1}{2}(a-b).$
- 24 $\cos a - \cos b = -2 \sin \frac{1}{2}(a+b) \sin \frac{1}{2}(a-b).$

Call the sides of any triangle A, B, C , and the opposite angles a, b , and c . Call $s = \frac{1}{2}(A+B+C)$.

- 25 $\tan \frac{1}{2}(a-b) = \frac{A-B}{A+B} \tan \frac{1}{2}(a+b) = \frac{A-B}{A+B} \cot \frac{1}{2}c.$
- 26 $C = (A+B) \frac{\cos \frac{1}{2}(a+b)}{\cos \frac{1}{2}(a-b)} = (A-B) \frac{\sin \frac{1}{2}(a+b)}{\sin \frac{1}{2}(a-b)}.$
- 27 $\sin \frac{1}{2}a = \sqrt{\frac{(s-B)(s-C)}{BC}}.$
- 28 $\cos \frac{1}{2}a = \sqrt{\frac{s(s-A)}{BC}}.$
- 29 $\text{vers } a = \frac{2(s-B)(s-C)}{BC}.$
- 30 $\text{Area} = \sqrt{s(s-A)(s-B)(s-C)} = A^2 \frac{\sin b \sin c}{2 \sin a}.$

TABLE XXXI.—USEFUL FORMULÆ AND CONSTANTS.

	Logarithm.	
Circumference of a circle (radius = r) = $2\pi r$.		
Area of a circle = πr^2 .		
Area of sector (length of arc = l) = $\frac{1}{2}lr$.		
" " " (angle of arc = α°) = $\frac{\alpha}{360}\pi r^2$.		
Area of segment (chord = c , mid. ord. = m) = $\frac{2}{3}cm$ (approx.).		
Area of a circle to radius 1	} = π =	3.1415927
Circumference of a circle to diameter 1		
Surface of a sphere to diameter 1		
Volume of a sphere to radius 1 = $4\pi \div 3$ =		4.1887902
Arc equal to radius expressed in {	degrees =	57.2957795
	minutes =	3437.7467708
	seconds =	206264.8062471
Length of arc of 1° , radius unity.....		0.01745329
Sine of one second = 0.0000048481.....		4.6855749
Cubic inches in United States standard gallon = 231.....		2.3636120
Weight of one cubic foot of water at maximum density (therm. 39°.8 F., barom. 30'').....		62.379
Weight of one cubic foot of water at ordinary temperature (therm. 62° F.).....		62.321
Acceleration due to gravity at latitude of New York in feet per square second.....		32.15945
Feet in one metre.....		3.280869
Metres in one foot.....		0.304797

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
1	1	1	1.0000000	1.0000000	1.000000000
2	4	8	1.4142136	1.2599210	.500000000
3	9	27	1.7320508	1.4422496	.333333333
4	16	64	2.0000000	1.5874011	.250000000
5	25	125	2.2360680	1.7099759	.200000000
6	36	216	2.4494897	1.8171206	.166666667
7	49	343	2.6457513	1.9129312	.142857143
8	64	512	2.8284271	2.0000000	.125000000
9	81	729	3.0000000	2.0800837	.111111111
10	100	1000	3.1622777	2.1544347	.100000000
11	121	1331	3.3166248	2.239801	.090909091
12	144	1728	3.4641016	2.2894286	.083333333
13	169	2197	3.6055513	2.3513347	.076923077
14	196	2744	3.7416574	2.4101422	.071428571
15	225	3375	3.8729833	2.4662121	.066666667
16	256	4096	4.0000000	2.5198421	.062500000
17	289	4913	4.1231056	2.5712816	.058823529
18	324	5832	4.2426407	2.6207414	.055555556
19	361	6859	4.3588989	2.6684016	.052631579
20	400	8000	4.4721360	2.7144177	.050000000
21	441	9261	4.5825757	2.7589243	.047619048
22	484	10648	4.6904158	2.8020393	.045454545
23	529	12167	4.7958315	2.8438670	.043478261
24	576	13824	4.8989795	2.8844991	.041666667
25	625	15625	5.0000000	2.9240177	.040000000
26	676	17576	5.0990195	2.9624960	.038461538
27	729	19683	5.1961524	3.0000000	.037037037
28	784	21952	5.2915026	3.0365889	.035714286
29	841	24389	5.3851648	3.0723168	.034482759
30	900	27000	5.4772256	3.1072325	.033333333
31	961	29791	5.5677644	3.1413806	.032258065
32	1024	32768	5.6568542	3.1748021	.031250000
33	1089	35937	5.7445626	3.2075343	.030303030
34	1156	39304	5.8309519	3.2396118	.029411765
35	1225	42875	5.9160798	3.2710663	.028571429
36	1296	46656	6.0000000	3.3019272	.027777778
37	1369	50653	6.0827625	3.3322218	.027027027
38	1444	54872	6.1644140	3.3619754	.026315789
39	1521	59319	6.2449980	3.3912114	.025641026
40	1600	64000	6.3245553	3.4199519	.025000000
41	1681	68921	6.4031242	3.4482172	.024390244
42	1764	74088	6.4807407	3.4760266	.023809524
43	1849	79507	6.5574385	3.5033981	.023255814
44	1936	85184	6.6332496	3.5303483	.022727273
45	2025	91125	6.7082039	3.5568933	.022222222
46	2116	97336	6.7823300	3.5830479	.021739130
47	2209	103823	6.8556546	3.6088261	.021276600
48	2304	110592	6.9282032	3.6342411	.020833333
49	2401	117649	7.0000000	3.6593057	.020408163
50	2500	125000	7.0710678	3.6840314	.020000000
51	2601	132651	7.1414284	3.7084298	.019607843
52	2704	140608	7.2111026	3.7325111	.019230769
53	2809	148877	7.2801099	3.7562858	.018867925
54	2916	157464	7.3484692	3.7797631	.018518519
55	3025	166375	7.4161985	3.8029525	.018181818
56	3136	175616	7.4833148	3.8258624	.017857143
57	3249	185193	7.5498344	3.8485011	.017543860
58	3364	195112	7.6157731	3.8708766	.017241379
59	3481	205379	7.6811457	3.8929965	.016949153
60	3600	216000	7.7459667	3.9148676	.016666667

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
61	3721	226981	7.8102497	3.9364972	.016393443
62	3844	238328	7.8740079	3.9578915	.016129032
63	3969	250047	7.9372539	3.9790571	.015873016
64	4096	262144	8.0000000	4.0000000	.015625000
65	4225	274625	8.0622577	4.0207256	.015384615
66	4356	287496	8.1240384	4.0412401	.015151515
67	4489	300763	8.1853528	4.0615480	.014925373
68	4624	314432	8.2462113	4.0816551	.014705882
69	4761	328509	8.3066239	4.1015661	.014492754
70	4900	343000	8.3666003	4.1212853	.014285714
71	5041	357911	8.4261498	4.1408178	.014084507
72	5184	373248	8.4852814	4.1601676	.013888889
73	5329	389017	8.5440037	4.1793390	.013698630
74	5476	405224	8.6023253	4.1983364	.013513514
75	5625	421875	8.6602540	4.2171633	.013333333
76	5776	438976	8.7177979	4.2358236	.013157895
77	5929	456533	8.7749644	4.2543210	.012987013
78	6084	474552	8.8317609	4.2726586	.012820513
79	6241	493039	8.8881944	4.2908404	.012658228
80	6400	512000	8.9442719	4.3088695	.012500000
81	6561	531441	9.0000000	4.3267487	.012345679
82	6724	551368	9.0553851	4.3444815	.012195122
83	6889	571787	9.1104336	4.3620707	.012048193
84	7056	592704	9.1651514	4.3795191	.011904762
85	7225	614125	9.2195445	4.3968296	.011764706
86	7396	636056	9.2736185	4.4140049	.011627907
87	7569	658503	9.3273791	4.4310476	.011494253
88	7744	681472	9.3808315	4.4479602	.011363636
89	7921	704969	9.4339811	4.4647451	.011235955
90	8100	729000	9.4868330	4.4814047	.011111111
91	8281	753571	9.5393920	4.4979414	.010989011
92	8464	778688	9.5916630	4.5143574	.010869565
93	8649	804357	9.6436508	4.5306549	.010752688
94	8836	830584	9.6953597	4.5468359	.010638298
95	9025	857375	9.7467943	4.5629026	.010526316
96	9216	884736	9.7979590	4.5788570	.010416667
97	9409	912673	9.8488578	4.5947009	.010309278
98	9604	941192	9.8994949	4.6104363	.010204082
99	9801	970299	9.9498744	4.6260650	.010101010
100	10000	1000000	10.0000000	4.6415888	.010000000
101	10201	1030301	10.0498756	4.6570095	.009900990
102	10404	1061208	10.0995049	4.6723287	.009803922
103	10609	1092727	10.1488916	4.6875482	.009708738
104	10816	1124864	10.1980390	4.7026694	.009615385
105	11025	1157625	10.2469508	4.7176940	.009523810
106	11236	1191016	10.2956301	4.7326235	.009433962
107	11449	1225043	10.3440804	4.7474594	.009345794
108	11664	1259712	10.3923048	4.7622032	.009259259
109	11881	1295029	10.4403065	4.7768562	.009174312
110	12100	1331000	10.4880885	4.7914199	.009090909
111	12321	1367631	10.5356538	4.8058955	.009009009
112	12544	1404928	10.5830052	4.8202845	.008928571
113	12769	1442897	10.6301458	4.8345881	.008849558
114	12996	1481544	10.6770783	4.8488076	.008771930
115	13225	1520875	10.7238053	4.8629442	.008695652
116	13456	1560896	10.7703296	4.8769990	.008620690
117	13689	1601613	10.8166538	4.8909732	.008547009
118	13924	1643032	10.8627805	4.9048681	.008474576
119	14161	1685159	10.9087121	4.9186847	.008403361
120	14400	1728000	10.9544512	4.9324242	.008333333

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
121	14641	1771561	11.0000000	4.9460874	.008264463
122	14884	1815848	11.0453610	4.9596757	.008196721
123	15129	1860867	11.0905365	4.9731898	.008130081
124	15376	1906624	11.1355287	4.9866310	.008064516
125	15625	1953125	11.1803399	5.0000000	.008000000
126	15876	2000376	11.2249722	5.0132979	.007936508
127	16129	2048383	11.2694277	5.0265257	.007874016
128	16384	2097152	11.3137085	5.0396842	.007812500
129	16641	2146689	11.3578167	5.0527743	.007751938
130	16900	2197000	11.4017543	5.0657970	.007692308
131	17161	2248091	11.4455231	5.0787531	.007633588
132	17424	2299968	11.4891253	5.0916434	.007575758
133	17689	2352637	11.5325626	5.1044687	.007518797
134	17956	2406104	11.5758369	5.1172299	.007462687
135	18225	2460375	11.6189500	5.1299278	.007407407
136	18496	2515456	11.6619038	5.1425632	.007352941
137	18769	2571353	11.7046999	5.1551367	.007299270
138	19044	2628072	11.7473401	5.1676493	.007246377
139	19321	2685619	11.7898261	5.1801015	.007194245
140	19600	2744000	11.8321596	5.1924941	.007142857
141	19881	2803221	11.8743421	5.2048279	.007092199
142	20164	2863288	11.9163753	5.2171034	.007042254
143	20449	2924207	11.9582607	5.2293215	.006993007
144	20736	2985984	12.0000000	5.2414828	.006944444
145	21025	3048625	12.0415946	5.2535879	.006896552
146	21316	3112136	12.0830460	5.2656374	.006849315
147	21609	3176523	12.1243557	5.2776321	.006802721
148	21904	3241792	12.1655251	5.2895725	.006756757
149	22201	3307949	12.2065556	5.3014592	.006711409
150	22500	3375000	12.2474487	5.3132928	.006666667
151	22801	3442951	12.2882057	5.3250740	.006622517
152	23104	3511808	12.3288280	5.3368033	.006578947
153	23409	3581577	12.3693169	5.3484812	.006535948
154	23716	3652264	12.4096736	5.3601084	.006493506
155	24025	3723875	12.4498996	5.3716854	.006451613
156	24336	3796416	12.4899960	5.3832126	.006410256
157	24649	3869893	12.5299641	5.3946907	.006369427
158	24964	3944312	12.5698051	5.4061202	.006329114
159	25281	4019679	12.6095202	5.4175015	.006289308
160	25600	4096000	12.6491106	5.4288352	.006250000
161	25921	4173281	12.6885775	5.4401218	.006211180
162	26244	4251528	12.7279221	5.4513618	.006172840
163	26569	4330747	12.7671453	5.4625556	.006134969
164	26896	4410944	12.8062485	5.4737037	.006097561
165	27225	4492125	12.8452326	5.4848066	.006060606
166	27556	4574296	12.8840987	5.4958647	.006024096
167	27889	4657463	12.9228480	5.5068784	.005988024
168	28224	4741632	12.9614814	5.5178484	.005952381
169	28561	4826809	13.0000000	5.5287748	.005917160
170	28900	4913000	13.0384048	5.5396583	.005882353
171	29241	5000211	13.0766968	5.5504991	.005847953
172	29584	5088448	13.1148770	5.5612978	.005813953
173	29929	5177717	13.1529464	5.5720546	.005780347
174	30276	5268024	13.1909060	5.5827702	.005747126
175	30625	5359375	13.2287566	5.5934447	.005714286
176	30976	5451776	13.2664992	5.6040787	.005681818
177	31329	5545233	13.3041347	5.6146724	.005649718
178	31684	5639752	13.3416641	5.6252263	.005617978
179	32041	5735339	13.3790882	5.6357408	.005586592
180	32400	5832000	13.4164079	5.6462162	.005555556

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
181	32761	5929741	13.4536240	5.6566528	.005524862
182	33124	6028568	13.4907376	5.6670511	.005494505
183	33489	6128487	13.5277493	5.6774114	.005464481
184	33856	6229504	13.5646600	5.6877340	.005434783
185	34225	6331625	13.6014705	5.6980192	.005405405
186	34596	6434856	13.6381817	5.7082675	.005376344
187	34969	6539203	13.6747943	5.7184791	.005347594
188	35344	6644672	13.7113092	5.7286543	.005319149
189	35721	6751269	13.7477271	5.7387936	.005291005
190	36100	6859000	13.7840488	5.7488971	.005263158
191	36481	6967871	13.8202750	5.7589652	.005235602
192	36864	7077888	13.8564065	5.7689982	.005208333
193	37249	7189057	13.8924440	5.7789966	.005181347
194	37636	7301384	13.9283883	5.7889604	.005154639
195	38025	7414875	13.9642400	5.7988900	.005128205
196	38416	7529536	14.0000000	5.8087857	.005102041
197	38809	7645373	14.0356688	5.8186479	.005076142
198	39204	7762392	14.0712473	5.8284767	.005050505
199	39601	7880599	14.1067360	5.8382725	.005025126
200	40000	8000000	14.1421356	5.8480355	.005000000
201	40401	8120601	14.1774469	5.8577660	.004975124
202	40804	8242408	14.2126704	5.8674643	.004950495
203	41209	8365427	14.2478068	5.8771307	.004926108
204	41616	8489664	14.2828569	5.8867653	.004901961
205	42025	8615125	14.3178211	5.8963685	.004878049
206	42436	8741816	14.3527001	5.9059406	.004854369
207	42849	8869743	14.3874946	5.9154817	.004830918
208	43264	8998912	14.4222051	5.9249921	.004807692
209	43681	9129329	14.4568323	5.9344721	.004784689
210	44100	9261000	14.4913767	5.9439220	.004761905
211	44521	9393931	14.5258390	5.9533418	.004739336
212	44944	9528128	14.5602198	5.9627320	.004716981
213	45369	9663597	14.5945195	5.9720926	.004694836
214	45796	9800344	14.6287388	5.9814240	.004672897
215	46225	9938375	14.6628783	5.9907264	.004651163
216	46656	10077696	14.6969385	6.0000000	.004629630
217	47089	10218313	14.7309199	6.0092450	.004608295
218	47524	10360232	14.7648231	6.0184617	.004587156
219	47961	10503459	14.7986486	6.0276502	.004566210
220	48400	10648000	14.8323970	6.0368107	.004545455
221	48841	10793861	14.8660687	6.0459435	.004524887
222	49284	10941048	14.8996644	6.0550489	.004504505
223	49729	11089567	14.9331845	6.0641270	.004484305
224	50176	11239424	14.9666295	6.0731779	.004464286
225	50625	11390625	15.0000000	6.0822020	.004444444
226	51076	11543176	15.0332964	6.0911994	.004424779
227	51529	11697083	15.0665192	6.1001702	.004405286
228	51984	11852352	15.0996689	6.1091147	.004385965
229	52441	12008989	15.1327460	6.1180332	.004366812
230	52900	12167000	15.1657509	6.1269257	.004347826
231	53361	12326391	15.1986842	6.1357924	.004329004
232	53824	12487168	15.2315462	6.1446337	.004310345
233	54289	12649337	15.2643375	6.1534495	.004291845
234	54756	12812904	15.2970585	6.1622401	.004273504
235	55225	12977875	15.3297097	6.1710058	.004255319
236	55696	13144256	15.3622915	6.1797466	.004237288
237	56169	13312053	15.3948043	6.1884628	.004219409
238	56644	13481272	15.4272486	6.1971544	.004201681
239	57121	13651919	15.4596248	6.2058218	.004184100
240	57600	13824000	15.4919334	6.2144650	.004166667

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
241	58081	13997521	15.5241747	6.2230843	.004149378
242	58564	14172488	15.5563492	6.2316797	.004132231
243	59049	14348907	15.5884573	6.2402515	.004115226
244	59536	14526784	15.6204994	6.2487998	.004098361
245	60025	14706125	15.6524758	6.2573248	.004081633
246	60516	14886936	15.6843871	6.2658266	.004065041
247	61009	15069223	15.7162336	6.2743054	.004048583
248	61504	15252992	15.7480157	6.2827613	.004032258
249	62001	15438249	15.7797338	6.2911946	.004016064
250	62500	15625000	15.8113883	6.2996053	.004000000
251	63001	15813251	15.8429795	6.3079935	.003984064
252	63504	16003008	15.8745079	6.3163596	.003968254
253	64009	16194277	15.9059737	6.3247035	.003952569
254	64516	16387064	15.9373775	6.3330256	.003937008
255	65025	16581375	15.9687194	6.3413257	.003921569
256	65536	16777216	16.0000000	6.3496042	.003906250
257	66049	16974593	16.0312195	6.3578611	.003891051
258	66564	17173512	16.0623784	6.3660968	.003875969
259	67081	17373979	16.0934769	6.3743111	.003861004
260	67600	17576000	16.1245155	6.3825043	.003846154
261	68121	17779581	16.1554944	6.3906765	.003831418
262	68644	17984728	16.1864141	6.3988279	.003816794
263	69169	18191447	16.2172747	6.4069585	.003802281
264	69696	18399744	16.2480768	6.4150687	.003787879
265	70225	18609625	16.2788206	6.4231583	.003773585
266	70756	18821096	16.3095064	6.4312276	.003759398
267	71289	19034163	16.3401346	6.4392767	.003745318
268	71824	19248832	16.3707055	6.4473057	.003731343
269	72361	19465109	16.4012195	6.4553148	.003717472
270	72900	19683000	16.4316767	6.4633041	.003703704
271	73441	19902511	16.4620776	6.4712736	.003690037
272	73984	20123648	16.4924225	6.4792235	.003676471
273	74529	20346417	16.5227116	6.4871541	.003663004
274	75076	20570824	16.5529454	6.4950653	.003649635
275	75625	20796875	16.5831240	6.5029572	.003636364
276	76176	21024576	16.6132477	6.5108300	.003623188
277	76729	21253933	16.6433170	6.5186839	.003610108
278	77284	21484952	16.6733320	6.5265189	.003597122
279	77841	21717639	16.7032931	6.5343351	.003584229
280	78400	21952000	16.7332005	6.5421326	.003571429
281	78961	22188041	16.7630546	6.5499116	.003558719
282	79524	22425768	16.7928556	6.5576722	.003546099
283	80089	22665187	16.8226038	6.5654144	.003533569
284	80656	22906304	16.8522995	6.5731385	.003521127
285	81225	23149125	16.8819430	6.5808443	.003508772
286	81796	23393656	16.9115345	6.5885323	.003496503
287	82369	23639903	16.9410743	6.5962023	.003484321
288	82944	23887872	16.9705627	6.6038545	.003472222
289	83521	24137569	17.0000000	6.6114890	.003460208
290	84100	24389000	17.0293864	6.6191060	.003448276
291	84681	24642171	17.0587221	6.6267054	.003436426
292	85264	24897088	17.0880075	6.6342874	.003424658
293	85849	25153757	17.1172428	6.6418522	.003412969
294	86436	25412184	17.1464282	6.6493998	.003401361
295	87025	25672375	17.1755640	6.6569302	.003389831
296	87616	25934336	17.2046505	6.6644437	.003378378
297	88209	26198073	17.2336879	6.6719403	.003367003
298	88804	26463592	17.2626765	6.6794200	.003355705
299	89401	26730899	17.2916165	6.6868831	.003344482
300	90000	27000000	17.3205081	6.6943295	.003333333

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
301	90601	27270901	17.3493516	6.7017593	.003322259
302	91204	27543608	17.3781472	6.7091729	.003311258
303	91809	27818127	17.4068952	6.7165700	.003300330
304	92416	28094464	17.4355958	6.7239508	.003289474
305	93025	28372625	17.4642492	6.7313155	.003278689
306	93636	28652616	17.4928557	6.7386641	.003267974
307	94249	28934443	17.5214155	6.7459967	.003257329
308	94864	29218112	17.5499288	6.7533134	.003246753
309	95481	29503629	17.5783958	6.7606143	.003236246
310	96100	29791000	17.6068169	6.7678995	.003225806
311	96721	30080231	17.6351921	6.7751690	.003215434
312	97344	30371328	17.6635217	6.7824229	.003205128
313	97969	30664297	17.6918060	6.7896613	.003194888
314	98596	30959144	17.7200451	6.7968844	.003184713
315	99225	31255875	17.7482393	6.8040921	.003174603
316	99856	31554496	17.7763888	6.8112847	.003164557
317	100489	31855013	17.8044938	6.8184620	.003154574
318	101124	32157432	17.8325545	6.8256242	.003144654
319	101761	32461759	17.8605711	6.8327714	.003134796
320	102400	32768000	17.8885438	6.8399037	.003125000
321	103041	33076161	17.9164729	6.8470213	.003115265
322	103684	33386248	17.9443584	6.8541240	.003105590
323	104329	33698267	17.9722008	6.8612120	.003095975
324	104976	34012224	18.0000000	6.8682855	.003086420
325	105625	34328125	18.0277564	6.8753443	.003076923
326	106276	34645976	18.0554701	6.8823888	.003067485
327	106929	34965783	18.0831413	6.8894188	.003058104
328	107584	35287552	18.1107703	6.8964345	.003048780
329	108241	35611289	18.1383571	6.9034359	.003039514
330	108900	35937000	18.1659021	6.9104232	.003030303
331	109561	36264691	18.1934054	6.9173964	.003021148
332	110224	36594368	18.2208672	6.9243556	.003012048
333	110889	36926037	18.2482876	6.9313008	.003003003
334	111556	37259704	18.2756669	6.9382321	.002994012
335	112225	37595375	18.3030052	6.9451496	.002985075
336	112896	37933056	18.3303028	6.9520533	.002976190
337	113569	38272753	18.3575598	6.9589434	.002967359
338	114244	38614472	18.3847763	6.9658198	.002958580
339	114921	38958219	18.4119526	6.9726826	.002949853
340	115600	39304000	18.4390889	6.9795321	.002941176
341	116281	39651821	18.4661853	6.9863681	.002932551
342	116964	40001688	18.4932420	6.9931906	.002923977
343	117649	40353607	18.5202592	7.0000000	.002915452
344	118336	40707534	18.5472370	7.0067962	.002906977
345	119025	41063625	18.5741756	7.0135791	.002898551
346	119716	41421736	18.6010752	7.0203490	.002890173
347	120409	41781923	18.6279360	7.0271058	.002881844
348	121104	42144192	18.6547581	7.0338497	.002873563
349	121801	42508549	18.6815417	7.0405806	.002865330
350	122500	42875000	18.7082869	7.0472987	.002857143
351	123201	43243551	18.7349940	7.0540041	.002849003
352	123904	43614208	18.7616630	7.0606967	.002840909
353	124609	43986977	18.7882942	7.0673767	.002832861
354	125316	44361864	18.8148877	7.0740440	.002824859
355	126025	44738875	18.8414437	7.0806988	.002816901
356	126736	45118016	18.8679623	7.0873411	.002808989
357	127449	45499293	18.8944436	7.0939709	.002801120
358	128164	45882712	18.9208879	7.1005885	.002793296
359	128881	46268279	18.9472953	7.1071937	.002785515
360	129600	46656000	18.9736660	7.1137866	.002777778

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
361	130321	47045881	19.0000000	7.1203674	.002770083
362	131044	47437928	19.0262976	7.1269360	.002762431
363	131769	47832147	19.0525589	7.1334925	.002754821
364	132496	48228544	19.0787840	7.1400370	.002747253
365	133225	48627125	19.1049732	7.1465695	.002739728
366	133956	49027896	19.1311265	7.1530901	.002732240
367	134689	49430863	19.1572441	7.1595988	.002724796
368	135424	49836032	19.1833261	7.1660957	.002717391
369	136161	50243409	19.2093727	7.1725809	.002710027
370	136900	50653000	19.2353841	7.1790544	.002702703
371	137641	51064811	19.2613603	7.1855162	.002695418
372	138384	51478848	19.2873015	7.1919663	.002688172
373	139129	51895117	19.3132079	7.1984050	.002680965
374	139876	52313624	19.3390796	7.2048322	.002673797
375	140625	52734375	19.3649167	7.2112479	.002666667
376	141376	53157376	19.3907194	7.2176522	.002659574
377	142129	53582633	19.4164878	7.2240450	.002652520
378	142884	54010152	19.4422221	7.2304268	.002645503
379	143641	54439939	19.4679223	7.2367972	.002638522
380	144400	54872000	19.4935887	7.2431565	.002631579
381	145161	55306341	19.5192213	7.2495045	.002624672
382	145924	55742968	19.5448203	7.2558415	.002617801
383	146689	56181887	19.5703858	7.2621675	.002610966
384	147456	56623104	19.5959179	7.2684824	.002604167
385	148225	57066825	19.6214169	7.2747864	.002597403
386	148996	57512456	19.6468827	7.2810794	.002590674
387	149769	57960603	19.6723156	7.2873617	.002583979
388	150544	58411072	19.6977156	7.2936330	.002577320
389	151321	58863869	19.7230829	7.2998936	.002570694
390	152100	59319000	19.7484177	7.3061436	.002564103
391	152881	59776471	19.7737199	7.3123828	.002557545
392	153664	60236288	19.7989899	7.3186114	.002551020
393	154449	60698457	19.8242276	7.3248295	.002544529
394	155236	61162984	19.8494332	7.3310369	.002538071
395	156025	61629875	19.8746069	7.3372339	.002531646
396	156816	62099136	19.8997487	7.3434205	.002525253
397	157609	62570773	19.9248588	7.3495966	.002518892
398	158404	63044792	19.9499373	7.3557624	.002512563
399	159201	63521199	19.9749844	7.3619178	.002506266
400	160000	64000000	20.0000000	7.3680630	.002500000
401	160801	64481201	20.0249844	7.3741979	.002493766
402	161604	64964808	20.0499377	7.3803227	.002487562
403	162409	65450827	20.0748599	7.3864373	.002481390
404	163216	65939264	20.0997512	7.3925418	.002475248
405	164025	66430125	20.1246118	7.3986363	.002469136
406	164836	66923416	20.1494417	7.4047206	.002463054
407	165649	67419143	20.1742410	7.4107950	.002457002
408	166464	67917312	20.1990099	7.4168595	.002450980
409	167281	68417929	20.2237484	7.4229142	.002444988
410	168100	68921000	20.2484567	7.4289589	.002439024
411	168921	69426531	20.2731349	7.4349938	.002433090
412	169744	69934528	20.2977831	7.4410189	.002427184
413	170569	70444997	20.3224014	7.4470342	.002421308
414	171396	70957944	20.3469899	7.4530399	.002415459
415	172225	71473375	20.3715488	7.4590359	.002409639
416	173056	71991296	20.3960781	7.4650223	.002403846
417	173889	72511713	20.4205779	7.4709991	.002398082
418	174724	73034632	20.4450483	7.4769664	.002392344
419	175561	73560059	20.4694895	7.4829242	.002386635
420	176400	74088000	20.4939015	7.4888724	.002380952

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
421	177241	74618461	20.5182845	7.4948113	.002375297
422	178084	75151448	20.5426386	7.5007406	.002369668
423	178929	75689697	20.5669638	7.5066607	.002364066
424	179776	76225024	20.5912603	7.5125715	.002358491
425	180625	76765625	20.6155281	7.5184730	.002352941
426	181476	77308776	20.6397674	7.5243652	.002347418
427	182329	77854483	20.6639783	7.5302482	.002341920
428	183184	78402752	20.6881609	7.5361221	.002336449
429	184041	78953589	20.7123152	7.5419867	.002331002
430	184900	79507000	20.7364414	7.5478423	.002325581
431	185761	80062991	20.7605395	7.5536888	.002320186
432	186624	80621568	20.7846097	7.5595263	.002314815
433	187489	81182737	20.8086520	7.5653548	.002309469
434	188356	81746504	20.8326667	7.5711743	.002304147
435	189225	82312875	20.8566536	7.5769849	.002298851
436	190096	82881856	20.8806130	7.5827865	.002293578
437	190969	83453453	20.9045450	7.5885793	.002288330
438	191844	84027672	20.9284495	7.5943633	.002283105
439	192721	84604519	20.9523268	7.6001385	.002277904
440	193600	85184000	20.9761770	7.6059049	.002272727
441	194481	85766121	21.0000000	7.6116626	.002267574
442	195364	86350888	21.0237960	7.6174116	.002262443
443	196249	86938307	21.0475652	7.6231519	.002257336
444	197136	87528384	21.0713075	7.6288837	.002252252
445	198025	88121125	21.0950231	7.6346067	.002247191
446	198916	88716536	21.1187121	7.6403213	.002242152
447	199809	89314623	21.1423745	7.6460272	.002237136
448	200704	89915392	21.1660105	7.6517247	.002232143
449	201601	90518849	21.1896201	7.6574138	.002227171
450	202500	91125000	21.2132034	7.6630943	.002222222
451	203401	91733851	21.2367606	7.6687665	.002217295
452	204304	92345408	21.2602916	7.6744303	.002212389
453	205209	92959677	21.2837967	7.6800857	.002207508
454	206116	93576664	21.3072758	7.6857328	.002202643
455	207025	94196375	21.3307290	7.6913717	.002197802
456	207936	94818816	21.3541565	7.6970023	.002192982
457	208849	95443993	21.3775583	7.7026246	.002188184
458	209764	96071912	21.4009346	7.7082388	.002183406
459	210681	96702579	21.4242483	7.7138448	.002178649
460	211600	97336000	21.4476106	7.7194426	.002173913
461	212521	97972181	21.4709106	7.7250325	.002169197
462	213444	98611128	21.4941853	7.7306141	.002164502
463	214369	99252847	21.5174348	7.7361877	.002159827
464	215296	99897344	21.5406592	7.7417532	.002155172
465	216225	100544625	21.5638587	7.7473109	.002150538
466	217156	101194696	21.5870331	7.7528606	.002145923
467	218089	101847563	21.6101828	7.7584023	.002141328
468	219024	102503232	21.6333077	7.7639361	.002136752
469	219961	103161709	21.6564078	7.7694620	.002132196
470	220900	103823000	21.6794834	7.7749801	.002127660
471	221841	104487111	21.7025344	7.7804904	.002123142
472	222784	105154048	21.7255610	7.7859928	.002118644
473	223729	105823817	21.7485632	7.7914875	.002114165
474	224676	106496424	21.7715411	7.7969745	.002109705
475	225625	107171875	21.7944947	7.8024538	.002105263
476	226576	107850176	21.8174242	7.8079254	.002100840
477	227529	108531333	21.8403297	7.8133892	.002096436
478	228484	109215352	21.8632111	7.8188456	.002092050
479	229441	109902239	21.8860686	7.8242942	.002087683
480	230400	110592000	21.9089023	7.8297353	.002083333

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
481	231361	111284641	21.9317122	7.8351688	.002079002
482	232324	111980168	21.9544984	7.8405949	.002074689
483	233289	112678587	21.9772610	7.8460134	.002070393
484	234256	113379904	22.0000000	7.8514244	.002066116
485	235225	114084125	22.0227155	7.8568281	.002061856
486	236196	114791256	22.0454077	7.8622242	.002057613
487	237169	115501303	22.0680765	7.8676130	.002053388
488	238144	116214272	22.0907220	7.8729944	.002049180
489	239121	116930169	22.1133444	7.8783684	.002044990
490	240100	117649000	22.1359436	7.8837352	.002040816
491	241081	118370771	22.1585198	7.8890946	.002036660
492	242064	119095488	22.1810730	7.8944468	.002032520
493	243049	119823157	22.2036033	7.8997917	.002028398
494	244036	120553784	22.2261108	7.9051294	.002024291
495	245025	121287375	22.2485955	7.9104599	.002020202
496	246016	122023936	22.2710575	7.9157832	.002016129
497	247009	122763473	22.2934968	7.9210994	.002012072
498	248004	123505992	22.3159136	7.9264085	.002008032
499	249001	124251499	22.3383079	7.9317104	.002004008
500	250000	125000000	22.3606798	7.9370053	.002000000
501	251001	125751501	22.3830293	7.9422931	.001996008
502	252004	126506008	22.4053565	7.9475739	.001992032
503	253009	127263527	22.4276615	7.9528477	.001988072
504	254016	128024064	22.4499443	7.9581144	.001984127
505	255025	128787625	22.4722051	7.9633743	.001980198
506	256036	129554216	22.4944438	7.9686271	.001976285
507	257049	130323843	22.5166605	7.9738731	.001972387
508	258064	131096512	22.5388553	7.9791122	.001968504
509	259081	131872229	22.5610283	7.9843444	.001964637
510	260100	132651000	22.5831796	7.9895697	.001960784
511	261121	133432831	22.6053091	7.9947883	.001956947
512	262144	134217728	22.6274170	8.0000000	.001953125
513	263169	135005697	22.6495033	8.0052049	.001949318
514	264196	135796744	22.6715681	8.0104032	.001945525
515	265225	136590875	22.6936114	8.0155946	.001941748
516	266256	137388096	22.7156334	8.0207794	.001937984
517	267289	138188413	22.7376340	8.0259574	.001934236
518	268324	138991832	22.7596134	8.0311287	.001930502
519	269361	139798359	22.7815715	8.0362935	.001926782
520	270400	140608000	22.8035085	8.0414515	.001923077
521	271441	141420761	22.8254244	8.0466030	.001919386
522	272484	142236648	22.8473193	8.0517479	.001915709
523	273529	143055667	22.8691933	8.0568862	.001912046
524	274576	143877824	22.8910463	8.0620180	.001908397
525	275625	144703125	22.9128785	8.0671432	.001904762
526	276676	145531576	22.9346899	8.0722620	.001901141
527	277729	146363183	22.9564808	8.0773743	.001897533
528	278784	147197952	22.9782506	8.0824800	.001893939
529	279841	148035889	23.0000000	8.0875794	.001890359
530	280900	148877000	23.0217289	8.0926723	.001886792
531	281961	149721291	23.0434372	8.0977589	.001883239
532	283024	150568768	23.0651252	8.1028390	.001879699
533	284089	151419437	23.0867928	8.1079128	.001876173
534	285156	152273304	23.1084400	8.1129803	.001872659
535	286225	153130375	23.1300670	8.1180414	.001869159
536	287296	153990656	23.1516738	8.1230962	.001865672
537	288369	154854153	23.1732605	8.1281447	.001862197
538	289444	155720872	23.1948270	8.1331870	.001858736
539	290521	156590819	23.2163735	8.1382230	.001855288
540	291600	157464000	23.2379001	8.1432529	.001851852

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
541	292681	158340421	23.2594067	8.1482765	.001848429
542	293764	159220088	23.2808935	8.1532939	.001845018
543	294849	160103007	23.3023604	8.1583051	.001841621
544	295936	160989184	23.3238076	8.1633102	.001838235
545	297025	161878625	23.3452351	8.1683092	.001834862
546	298116	162771336	23.3666429	8.1733020	.001831502
547	299209	163667323	23.3880311	8.1782888	.001828154
548	300304	164566592	23.4093998	8.1832695	.001824818
549	301401	165469149	23.4307490	8.1882441	.001821494
550	302500	166375000	23.4520788	8.1932127	.001818182
551	303601	167284151	23.4733892	8.1981753	.001814882
552	304704	168196608	23.4946802	8.2031319	.001811594
553	305809	169112377	23.5159520	8.2080825	.001808318
554	306916	170031464	23.5372046	8.2130271	.001805054
555	308025	170953875	23.5584380	8.2179657	.001801802
556	309136	171879616	23.5796522	8.2228985	.001798561
557	310249	172808693	23.6008474	8.2278254	.001795332
558	311364	173741112	23.6220236	8.2327463	.001792115
559	312481	174676879	23.6431808	8.2376614	.001788909
560	313600	175616000	23.6643191	8.2425706	.001785714
561	314721	176558481	23.6854386	8.2474740	.001782531
562	315844	177504328	23.7065392	8.2523715	.001779359
563	316969	178453547	23.7276210	8.2572633	.001776199
564	318096	179406144	23.7486842	8.2621492	.001773050
565	319225	180362125	23.7697286	8.2670294	.001769912
566	320356	181321496	23.7907545	8.2719039	.001766784
567	321489	182284263	23.8117618	8.2767726	.001763668
568	322624	183250432	23.8327506	8.2816355	.001760563
569	323761	184220009	23.8537209	8.2864928	.001757469
570	324900	185193000	23.8746728	8.2913444	.001754386
571	326041	186169411	23.8956063	8.2961903	.001751313
572	327184	187149248	23.9165215	8.3010304	.001748252
573	328329	188132517	23.9374184	8.3058651	.001745201
574	329476	189119224	23.9582971	8.3106941	.001742160
575	330625	190109375	23.9791576	8.3155175	.001739130
576	331776	191102976	24.0000000	8.3203353	.001736111
577	332929	192100033	24.0208243	8.3251475	.001733102
578	334084	193100552	24.0416306	8.3299542	.001730104
579	335241	194104539	24.0624188	8.3347553	.001727116
580	336400	195112000	24.0831891	8.3395509	.001724138
581	337561	196122941	24.1039416	8.3443410	.001721170
582	338724	197137368	24.1246762	8.3491256	.001718213
583	339889	198155287	24.1453929	8.3539047	.001715266
584	341056	199176704	24.1660919	8.3586784	.001712329
585	342225	200201625	24.1867732	8.3634466	.001709402
586	343396	201230056	24.2074369	8.3682095	.001706485
587	344569	202262003	24.2280829	8.3729668	.001703578
588	345744	203297472	24.2487113	8.3777188	.001700680
589	346921	204336469	24.2693222	8.3824653	.001697793
590	348100	205379000	24.2899156	8.3872065	.001694915
591	349281	206425071	24.3104916	8.3919423	.001692047
592	350464	207474688	24.3310501	8.3966729	.001689189
593	351649	208527857	24.3515913	8.4013981	.001686341
594	352836	209584584	24.3721152	8.4061180	.001683502
595	354025	210644875	24.3926218	8.4108326	.001680672
596	355216	211708736	24.4131112	8.4155419	.001677852
597	356409	212776173	24.4335834	8.4202460	.001675042
598	357604	213847192	24.4540385	8.4249448	.001672241
599	358801	214921799	24.4744765	8.4296383	.001669449
600	360000	216000000	24.4948974	8.4343267	.001666667

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
601	361201	217081801	24.5153013	8.4390098	.001663894
602	362404	218167208	24.5356883	8.4436877	.001661130
603	363609	219256227	24.5560583	8.4483605	.001658375
604	364816	220348864	24.5764115	8.4530281	.001655629
605	366025	221445125	24.5967478	8.4576906	.001652893
606	367236	222545016	24.6170673	8.4623479	.001650165
607	368449	223648543	24.6373700	8.4670001	.001647446
608	369664	224755712	24.6576560	8.4716471	.001644737
609	370881	225866529	24.6779254	8.4762892	.001642036
610	372100	226981000	24.6981781	8.4809261	.001639344
611	373321	228099131	24.7184142	8.4855579	.001636661
612	374544	229220928	24.7386338	8.4901848	.001633987
613	375769	230346397	24.7588368	8.4948065	.001631321
614	376996	231475544	24.7790234	8.4994233	.001628664
615	378225	232608375	24.7991935	8.5040350	.001626016
616	379456	233744896	24.8193473	8.5086417	.001623377
617	380689	234885113	24.8394847	8.5132435	.001620746
618	381924	236029032	24.8596058	8.5178403	.001618123
619	383161	237176659	24.8797106	8.5224321	.001615509
620	384400	238328000	24.8997992	8.5270189	.001612903
621	385641	239483061	24.9198716	8.5316009	.001610306
622	386884	240641848	24.9399278	8.5361780	.001607717
623	388129	241804367	24.9599679	8.5407501	.001605136
624	389376	242970624	24.9799920	8.5453173	.001602564
625	390625	244140625	25.0000000	8.5498797	.001600000
626	391876	245314376	25.0199920	8.5544372	.001597444
627	393129	246491883	25.0399681	8.5589899	.001594896
628	394384	247673152	25.0599282	8.5635377	.001592357
629	395641	248858189	25.0798724	8.5680807	.001589825
630	396900	250047000	25.0998008	8.5726189	.001587302
631	398161	251239591	25.1197134	8.5771523	.001584786
632	399424	252435968	25.1396102	8.5816809	.001582278
633	400689	253636137	25.1594913	8.5862047	.001579779
634	401956	254840104	25.1793566	8.5907238	.001577287
635	403225	256047875	25.1992063	8.5952380	.001574803
636	404496	257259456	25.2190404	8.5997476	.001572327
637	405769	258474853	25.2388589	8.6042525	.001569859
638	407044	259694072	25.2586619	8.6087526	.001567398
639	408321	260917119	25.2784493	8.6132480	.001564945
640	409600	262144000	25.2982213	8.6177388	.001562500
641	410881	263374721	25.3179778	8.6222248	.001560062
642	412164	264609288	25.3377189	8.6267063	.001557632
643	413449	265847707	25.3574447	8.6311830	.001555210
644	414736	267089984	25.3771551	8.6356551	.001552795
645	416025	268336125	25.3968502	8.6401226	.001550388
646	417316	269586136	25.4165301	8.6445855	.001547988
647	418609	270840023	25.4361947	8.6490437	.001545595
648	419904	272097792	25.4558441	8.6534974	.001543210
649	421201	273359449	25.4754784	8.6579465	.001540832
650	422500	274625000	25.4950976	8.6623911	.001538462
651	423801	275894451	25.5147016	8.6668310	.001536098
652	425104	277167808	25.5342907	8.6712665	.001533742
653	426409	278445507	25.5538647	8.6756974	.001531394
654	427716	279726264	25.5734237	8.6801237	.001529052
655	429025	281011375	25.5929678	8.6845456	.001526718
656	430336	282300416	25.6124969	8.6889630	.001524390
657	431649	283593393	25.6320112	8.6933759	.001522070
658	432964	284890312	25.6515107	8.6977843	.001519757
659	434281	286191179	25.6709953	8.7021882	.001517451
660	435600	287496000	25.6904652	8.7065877	.001515152

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
661	436921	288804781	25.7099203	8.7109827	.001512859
662	438244	290117528	25.7293607	8.7153734	.001510574
663	439569	291434247	25.7487864	8.7197596	.001508296
664	440896	292754944	25.7681975	8.7241414	.001506024
665	442225	294079625	25.7875939	8.7285187	.001503759
666	443556	295408296	25.8069758	8.7328918	.001501502
667	444889	296740963	25.8263431	8.7372604	.001499250
668	446224	298077632	25.8456960	8.7416246	.001497006
669	447561	299418309	25.8650343	8.7459846	.001494768
670	448900	300763000	25.8843582	8.7503401	.001492537
671	450241	302111711	25.9036677	8.7546913	.001490313
672	451584	303464448	25.9229628	8.7590383	.001488095
673	452929	304821217	25.9422435	8.7633809	.001485884
674	454276	306182024	25.9615100	8.7677192	.001483680
675	455625	307546875	25.9807621	8.7720532	.001481481
676	456976	308915776	26.0000000	8.7763830	.001479290
677	458329	310288733	26.0192237	8.7807084	.001477105
678	459684	311665752	26.0384331	8.7850296	.001474926
679	461041	313046839	26.0576284	8.7893466	.001472754
680	462400	314432000	26.0768096	8.7936593	.001470588
681	463761	315821241	26.0959767	8.7979679	.001468429
682	465124	317214568	26.1151297	8.8022721	.001466276
683	466489	318611987	26.1342687	8.8065722	.001464129
684	467856	320013504	26.1533937	8.8108681	.001461988
685	469225	321419125	26.1725047	8.8151598	.001459854
686	470596	322828856	26.1916017	8.8194474	.001457726
687	471969	324242703	26.2106848	8.8237307	.001455604
688	473344	325660672	26.2297511	8.8280099	.001453488
689	474721	327082769	26.2488095	8.8322850	.001451379
690	476100	328509000	26.2678511	8.8365559	.001449275
691	477481	329939371	26.286889	8.8408227	.001447178
692	478864	331373888	26.305829	8.8450854	.001445087
693	480249	332812557	26.3248932	8.8493440	.001443001
694	481636	334255384	26.3438797	8.8535985	.001440922
695	483025	335702375	26.3628527	8.8578489	.001438849
696	484416	337153536	26.3818119	8.8620952	.001436782
697	485809	338608873	26.4007576	8.8663375	.001434720
698	487204	340068392	26.4196896	8.8705757	.001432665
699	488601	341532099	26.4386081	8.8748099	.001430615
700	490000	343000000	26.4575131	8.8790400	.001428571
701	491401	344472101	26.4764046	8.8832661	.001426534
702	492804	345948408	26.4952826	8.8874882	.001424501
703	494209	347428927	26.5141472	8.8917063	.001422475
704	495616	348913664	26.5329983	8.8959204	.001420455
705	497025	350402625	26.5518361	8.9001304	.001418440
706	498436	351895816	26.5706605	8.9043366	.001416431
707	499849	353393243	26.5894716	8.9085387	.001414427
708	501264	354894912	26.6082694	8.9127369	.001412429
709	502681	356400829	26.6270539	8.9169311	.001410437
710	504100	357911000	26.6458252	8.9211214	.001408451
711	505521	359425431	26.6645833	8.9253078	.001406470
712	506944	360944128	26.6833281	8.9294902	.001404494
713	508369	362467097	26.7020598	8.9336687	.001402525
714	509796	363994344	26.7207784	8.9378433	.001400560
715	511225	365525875	26.7394839	8.9420140	.001398501
716	512656	367061696	26.7581763	8.9461809	.001396648
717	514089	368601813	26.7768557	8.9503438	.001394700
718	515524	370146232	26.7955220	8.9545029	.001392758
719	516961	371694959	26.8141754	8.9586581	.001390821
720	518400	373248000	26.8328157	8.9628095	.001388889

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
721	519841	374805361	26.8514432	8.9669570	.001386963
722	521284	376367048	26.8700577	8.9711007	.001385042
723	522729	377933067	26.8886593	8.9752406	.001383126
724	524176	379503424	26.9072481	8.9793766	.001381215
725	525625	381078125	26.9258240	8.9835089	.001379310
726	527076	382657176	26.9443872	8.9876373	.001377410
727	528529	384240583	26.9629375	8.9917620	.001375516
728	529984	385828352	26.9814751	8.9958829	.001373626
729	531441	387420489	27.0000000	9.0000000	.001371742
730	532900	389017000	27.0185122	9.0041134	.001369863
731	534361	390617891	27.0370117	9.0082229	.001367989
732	535824	392223168	27.0554985	9.0123288	.001366120
733	537289	393832837	27.0739727	9.0164309	.001364256
734	538756	395446904	27.0924344	9.0205293	.001362398
735	540225	397065375	27.1108834	9.0246239	.001360544
736	541696	398688256	27.1293199	9.0287149	.001358696
737	543169	400315553	27.1477439	9.0328021	.001356852
738	544644	401947272	27.1661554	9.0368857	.001355014
739	546121	403583419	27.1845544	9.0409655	.001353180
740	547600	405224000	27.2029410	9.0450419	.001351351
741	549081	406869021	27.2213152	9.0491142	.001349528
742	550564	408518488	27.2396769	9.0531831	.001347709
743	552049	410172407	27.2580263	9.0572482	.001345895
744	553536	411830784	27.2763634	9.0613098	.001344086
745	555025	413493625	27.2946881	9.0653677	.001342282
746	556516	415160936	27.3130006	9.0694220	.001340483
747	558009	416832723	27.3313007	9.0734726	.001338688
748	559504	418508992	27.3495887	9.0775197	.001336898
749	561001	420189749	27.3678644	9.0815631	.001335113
750	562500	421875000	27.3861279	9.0856030	.001333333
751	564001	423564751	27.4043792	9.0896392	.001331558
752	565504	425259008	27.4226184	9.0936719	.001329787
753	567009	426957777	27.4408455	9.0977010	.001328021
754	568516	428661064	27.4590604	9.1017265	.001326260
755	570025	430368875	27.4772633	9.1057485	.001324503
756	571536	432081216	27.4954542	9.1097669	.001322751
757	573049	433798093	27.5136330	9.1137818	.001321004
758	574564	435519512	27.5317998	9.1177931	.001319261
759	576081	437245479	27.5499546	9.1218010	.001317523
760	577600	438976000	27.5680975	9.1258053	.001315789
761	579121	440711081	27.5862284	9.1298061	.001314060
762	580644	442450728	27.6043475	9.1338034	.001312336
763	582169	444194947	27.6224546	9.1377971	.001310616
764	583696	445943744	27.6405499	9.1417874	.001308901
765	585225	447697125	27.6586334	9.1457742	.001307190
766	586756	449455096	27.6767050	9.1497576	.001305483
767	588289	451217663	27.6947648	9.1537375	.001303781
768	589824	452984832	27.7128129	9.1577139	.001302083
769	591361	454756609	27.7308492	9.1616869	.001300390
770	592900	456533000	27.7488739	9.1656565	.001298701
771	594441	458314011	27.7668868	9.1696225	.001297017
772	595984	460099648	27.7848880	9.1735852	.001295337
773	597529	461889917	27.8028775	9.1775445	.001293661
774	599076	463684824	27.8208555	9.1815003	.001291990
775	600625	465484375	27.8388218	9.1854527	.001290323
776	602176	467288576	27.8567766	9.1894018	.001288660
777	603729	469097433	27.8747197	9.1933474	.001287001
778	605284	470910952	27.8926514	9.1972894	.001285347
779	606841	472729139	27.9105715	9.2012286	.001283697
780	608400	474552000	27.9284801	9.2051641	.001282051

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
781	609961	476379541	27.9463772	9.2090962	.001280410
782	611524	478211768	27.9642629	9.2130250	.001278772
783	613089	480048687	27.9821372	9.2169505	.001277139
784	614656	481890304	28.0000000	9.2208726	.001275510
785	616225	483736625	28.0178515	9.2247914	.001273885
786	617796	485587656	28.0356915	9.2287068	.001272265
787	619369	487443403	28.0535203	9.2326189	.001270648
788	620944	489303872	28.0713377	9.2365277	.001269036
789	622521	491169069	28.0891438	9.2404333	.001267427
790	624100	493039000	28.1069386	9.2443355	.001265823
791	625681	494913671	28.1247222	9.2482344	.001264223
792	627264	496793088	28.1424946	9.2521300	.001262626
793	628849	498677257	28.1602557	9.2560224	.001261034
794	630436	500566184	28.1780056	9.2599114	.001259446
795	632025	502459875	28.1957444	9.2637973	.001257862
796	633616	504358336	28.2134720	9.2676798	.001256281
797	635209	506261573	28.2311884	9.2715592	.001254705
798	636804	508169592	28.2488938	9.2754352	.001253133
799	638401	510082399	28.2665881	9.2793081	.001251564
800	640000	512000000	28.2842712	9.2831777	.001250000
801	641601	513922401	28.3019434	9.2870440	.001248439
802	643204	515849608	28.3196045	9.2909072	.001246883
803	644809	517781627	28.3372546	9.2947671	.001245330
804	646416	519718464	28.3548938	9.2986239	.001243781
805	648025	521660125	28.3725219	9.3024775	.001242236
806	649636	523606616	28.3901391	9.3063278	.001240695
807	651249	525557943	28.4077454	9.3101750	.001239157
808	652864	527514112	28.4253408	9.3140190	.001237624
809	654481	529475129	28.4429253	9.3178599	.001236094
810	656100	531441000	28.4604989	9.3216975	.001234568
811	657721	533411731	28.4780617	9.3255320	.001233046
812	659344	535387328	28.4956137	9.3293634	.001231527
813	660969	537367797	28.5131549	9.3331916	.001230012
814	662596	539353144	28.5306852	9.3370167	.001228501
815	664225	541343375	28.5482048	9.3408386	.001226994
816	665856	543338496	28.5657137	9.3446575	.001225490
817	667489	545338513	28.5832119	9.3484731	.001223990
818	669124	547343432	28.6006993	9.3522857	.001222494
819	670761	549353259	28.6181760	9.3560952	.001221001
820	672400	551368000	28.6356421	9.3599016	.001219512
821	674041	553387661	28.6530976	9.3637049	.001218027
822	675684	555412248	28.6705424	9.3675051	.001216545
823	677329	557441767	28.6879766	9.3713022	.001215067
824	678976	559476224	28.7054002	9.3750963	.001213592
825	680625	561515625	28.7228132	9.3788873	.001212121
826	682276	563559976	28.7402157	9.3826752	.001210654
827	683929	565609283	28.7576077	9.3864600	.001209190
828	685584	567663552	28.7749891	9.3902419	.001207729
829	687241	569722789	28.7923601	9.3940206	.001206273
830	688900	571787000	28.8097206	9.3977964	.001204819
831	690561	573856191	28.8270706	9.4015691	.001203369
832	692224	575930368	28.8444102	9.4053387	.001201923
833	693889	578009537	28.8617394	9.4091054	.001200480
834	695556	580093704	28.8790582	9.4128690	.001199041
835	697225	582182875	28.8963666	9.4166297	.001197605
836	698896	584277056	28.9136646	9.4203873	.001196172
837	700569	586376253	28.9309523	9.4241420	.001194743
838	702244	588480472	28.9482297	9.4278936	.001193317
839	703921	590589719	28.9654967	9.4316423	.001191895
840	705600	592704000	28.9827535	9.4353880	.001190476

TABLE XXXII.—SQUARES, CUBES, SQUARE ROOTS,

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
841	707281	594823321	29.0000000	9.4391307	.001189061
842	708964	596947688	29.0172363	9.4428704	.001187648
843	710649	599077107	29.0344623	9.4466072	.001186240
844	712336	601211584	29.0516781	9.4503410	.001184834
845	714025	603351125	29.0688837	9.4540719	.001183432
846	715716	605495736	29.0860791	9.4577999	.001182033
847	717409	607645423	29.1032644	9.4615249	.001180638
848	719104	609800192	29.1204396	9.4652470	.001179245
849	720801	611960049	29.1376046	9.4689661	.001177856
850	722500	614125000	29.1547595	9.4726824	.001176471
851	724201	616295051	29.1719043	9.4763957	.001175088
852	725904	618470208	29.1890390	9.4801061	.001173709
853	727609	620650477	29.2061637	9.4838136	.001172333
854	729316	622835864	29.2232784	9.4875182	.001170960
855	731025	625026375	29.2403830	9.4912200	.001169591
856	732736	627222016	29.2574777	9.4949188	.001168224
857	734449	629422793	29.2745623	9.4986147	.001166861
858	736164	631628712	29.2916370	9.5023078	.001165501
859	737881	633839779	29.3087018	9.5059980	.001164144
860	739600	636056000	29.3257566	9.5096854	.001162791
861	741321	638277381	29.3428015	9.5133699	.001161440
862	743044	640503928	29.3598365	9.5170515	.001160093
863	744769	642735647	29.3768616	9.5207303	.001158749
864	746496	644972544	29.3938769	9.5244063	.001157407
865	748225	647214625	29.4108823	9.5280794	.001156069
866	749956	649461896	29.4278779	9.5317497	.001154734
867	751689	651714363	9.4448637	9.5354172	.001153403
868	753424	653972032	29.4618397	9.5390818	.001152074
869	755161	656234909	29.4788059	9.5427437	.001150748
870	756900	658503000	29.4957624	9.5464027	.001149425
871	758641	660776311	29.5127091	9.5500589	.001148106
872	760384	663054848	29.5296461	9.5537123	.001146789
873	762129	665338617	29.5465734	9.5573630	.001145475
874	763876	667627624	29.5634910	9.5610108	.001144165
875	765625	669921875	29.5803989	9.5646559	.001142857
876	767376	672221376	29.5972972	9.5682982	.001141553
877	769129	674526133	29.6141858	9.5719377	.001140251
878	770884	676836152	29.6310648	9.5755745	.001138952
879	772641	679151439	29.6479342	9.5792085	.001137656
880	774400	681472000	29.6647939	9.5828397	.001136364
881	776161	683797841	29.6816442	9.5864682	.001135074
882	777924	686128968	29.6984848	9.5900939	.001133787
883	779689	688465387	29.7153159	9.5937169	.001132503
884	781456	690807104	29.7321375	9.5973373	.001131222
885	783225	693154125	29.7489496	9.6009548	.001129944
886	784996	695506456	29.7657521	9.6045696	.001128668
887	786769	697864103	29.7825452	9.6081817	.001127396
888	788544	700227072	29.7993289	9.6117911	.001126126
889	790321	702595369	29.8161030	9.6153977	.001124859
890	792100	704969000	29.8328678	9.6190017	.001123596
891	793881	707347971	29.8496231	9.6226030	.001122334
892	795664	709732288	29.8663690	9.6262016	.001121076
893	797449	712121957	29.8831056	9.6297975	.001119821
894	799236	714516984	29.8998328	9.6333907	.001118568
895	801025	716917375	29.9165506	9.6369812	.001117318
896	802816	719323136	29.9332591	9.6405690	.001116071
897	804609	721734273	29.9499583	9.6441542	.001114827
898	806404	724150792	29.9666481	9.6477367	.001113586
899	808201	726572699	29.9833287	9.6513166	.001112347
900	810000	729000000	30.0000000	9.6548938	.001111111

CUBE ROOTS, AND RECIPROCAL.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
901	811801	731432701	30.0166620	9.6584684	.001109878
902	813604	733870808	30.0333148	9.6620403	.001108647
903	815409	736314327	30.0499584	9.6656096	.001107420
904	817216	738763264	30.0665928	9.6691762	.001106195
905	819025	741217625	30.0832179	9.6727403	.001104972
906	820836	743677416	30.0998339	9.6763017	.001103753
907	822649	746142643	30.1164407	9.6798604	.001102536
908	824464	748613312	30.1330383	9.6834166	.001101322
909	826281	751089429	30.1496269	9.6869701	.001100110
910	828100	753571000	30.1662063	9.6 05211	.001098901
911	829921	756058031	30.1827765	9.6940694	.001097695
912	831744	758550528	30.1993377	9.6976151	.001096491
913	833569	761048497	30.2158899	9.7011583	.001095290
914	835396	763551944	30.2324329	9.7046989	.001094092
915	837225	766060875	30.2489669	9.7082369	.001092896
916	839056	768575296	30.2654919	9.7117723	.001091703
917	840889	771095213	30.2820079	9.7153051	.001090513
918	842724	773620632	30.2985148	9.7188354	.001089325
919	844561	776151559	30.3150128	9.7223631	.001088139
920	846400	778688000	30.3315018	9.7258883	.001086957
921	848241	781229961	30.3479818	9.7294109	.001085776
922	850084	783777448	30.3644529	9.7329309	.001084599
923	851929	786330467	30.3809151	9.7364484	.001083423
924	853776	788889024	30.3973683	9.7399634	.001082251
925	855625	791453125	30.4138127	9.7434758	.001081081
926	857476	794022776	30.4302481	9.7469857	.001079914
927	859329	796597983	30.4466747	9.7504930	.001078749
928	861184	799178752	30.4630924	9.7539979	.001077586
929	863041	801765089	30.4795013	9.7575002	.001076426
930	864900	804357000	30.4959014	9.7610001	.001075269
931	866761	806954491	30.5122926	9.7644974	.001074114
932	868624	809557568	30.5286750	9.7679922	.001072961
933	870489	812166237	30.5450487	9.7714845	.001071811
934	872356	814780504	30.5614136	9.7749743	.001070664
935	874225	817400375	30.5777697	9.7784616	.001069519
936	876096	820025856	30.5941171	9.7819466	.001068376
937	877969	822656953	30.6104557	9.7854288	.001067236
938	879844	825293672	30.6267857	9.7889087	.001066098
939	881721	827936019	30.6431069	9.7923861	.001064963
940	883600	830584000	30.6594194	9.7958611	.001063830
941	885481	833237621	30.6757233	9.7993336	.001062699
942	887364	835896888	30.6920185	9.8028036	.001061571
943	889249	838561807	30.7083051	9.8062711	.001060445
944	891136	841232384	30.7245830	9.8097362	.001059322
945	893025	843908625	30.7408523	9.8131989	.001058201
946	894916	846590536	30.7571130	9.8166591	.001057082
947	896809	849278123	30.7733651	9.8201169	.001055966
948	898704	851971392	30.7896086	9.8235723	.001054852
949	900601	854670349	30.8058436	9.8270252	.001053741
950	902500	857375000	30.8220700	9.8304757	.001052632
951	904401	860085351	30.8382879	9.8339238	.001051525
952	906304	862801408	30.8544972	9.8373695	.001050420
953	908209	865523177	30.8706981	9.8408127	.001049318
954	910116	868250664	30.8868904	9.8442536	.001048218
955	912025	870983875	30.9030743	9.8476920	.001047120
956	913936	8737722816	30.9192497	9.8511280	.001046025
957	915849	876467493	30.9354166	9.8545617	.001044932
958	917764	879217912	30.9515751	9.8579929	.001043841
959	919681	881974079	30.9677251	9.8614218	.001042753
960	921600	884736000	30.9838668	9.8648483	.001041667

TABLE XXXII.—SQUARES, CUBES, ETC.

No.	Squares.	Cubes.	Square Roots.	Cube Roots.	Reciprocals.
961	923521	887503681	31.0000000	9.8682724	.001040583
962	925444	890277128	31.0161248	9.8716941	.001039501
963	927369	893056347	31.0322413	9.8751135	.001038422
964	929296	895841344	31.0483494	9.8785305	.001037344
965	931225	898632125	31.0644491	9.8819451	.001036269
966	933156	901428696	31.0805405	9.8853574	.001035197
967	935089	904231063	31.0966236	9.8887673	.001034126
968	937024	907039323	31.1126984	9.8921749	.001033058
969	938961	909853209	31.1287648	9.8955801	.001031992
970	940900	912673000	31.1448230	9.8989830	.001030928
971	942841	915498611	31.1608729	9.9023835	.001029866
972	944784	918330048	31.1769145	9.9057817	.001028807
973	946729	921167317	31.1929479	9.9091776	.001027749
974	948676	924010424	31.2089731	9.9125712	.001026694
975	950625	926859375	31.2249900	9.9159624	.001025641
976	952576	929714176	31.2409987	9.9193513	.001024590
977	954529	932574833	31.2569992	9.9227379	.001023541
978	956484	935441352	31.2729915	9.9261222	.001022495
979	958441	938313739	31.2889757	9.9295042	.001021450
980	960400	941192000	31.3049517	9.9328839	.001020408
981	962361	944076141	31.3209195	9.9362613	.001019368
982	964324	946966168	31.3368792	9.9396363	.001018330
983	966289	949862087	31.3528308	9.9430092	.001017294
984	968256	952763904	31.3687743	9.9463797	.001016260
985	970225	955671625	31.3847097	9.9497479	.001015228
986	972196	958585256	31.4006369	9.9531138	.001014199
987	974169	961504803	31.4165561	9.9564775	.001013171
988	976144	964430272	31.4324673	9.9598389	.001012146
989	978121	967361669	31.4483704	9.9631981	.001011122
990	980100	970299000	31.4642654	9.9665549	.001010101
991	982081	973242271	31.4801525	9.9699095	.001009082
992	984064	976191488	31.4960315	9.9732619	.001008065
993	986049	979146657	31.5119025	9.9766120	.001007049
994	988036	982107784	31.5277655	9.9799599	.001006036
995	990025	985074875	31.5436206	9.9833055	.001005025
996	992016	988047936	31.5594677	9.9866488	.001004016
997	994009	991026973	31.5753068	9.9899900	.001003009
998	996004	994011992	31.5911380	9.9933289	.001002004
999	998001	997002999	31.6069613	9.9966656	.001001001
1000	1000000	1000000000	31.6227766	10.0000000	.001000000
1001	1002001	1003003001	31.6385840	10.0033322	.0009990010
1002	1004004	1006012008	31.6543836	10.0066622	.0009980040
1003	1006009	1009027027	31.6701752	10.0099989	.0009970090
1004	1008016	1012048064	31.6859590	10.0133155	.0009960159
1005	1010025	1015075125	31.7017349	10.0166389	.0009950249
1006	1012036	1018108216	31.7175030	10.0199601	.0009940358
1007	1014049	1021147343	31.7332633	10.0232791	.0009930487
1008	1016064	1024192512	31.7490157	10.0265958	.0009920635
1009	1018081	1027243729	31.7647603	10.0299104	.0009910803
1010	1020100	1030301000	31.7804972	10.0332228	.0009900990
1011	1022121	1033364331	31.7962262	10.0365330	.0009891197
1012	1024144	1036433728	31.8119474	10.0398410	.0009881423
1013	1026169	1039509197	31.8276609	10.0431469	.0009871668
1014	1028196	1042590744	31.8433666	10.0464506	.0009861933
1015	1030225	1045678375	31.8590646	10.0497521	.0009852217
1016	1032256	1048772096	31.8747549	10.0530514	.0009842520
1017	1034289	1051871913	31.8904374	10.0563485	.0009832842
1018	1036324	1054977832	31.9061123	10.0596435	.0009823183
1019	1038361	1058089859	31.9217794	10.0629364	.0009813543
1020	1040400	1061208000	31.9374388	10.0662271	.0009803922

**TABLE XXXIII.—CUBIC YARDS PER 100 FEET OF LEVEL
SECTIONS. SLOPE 1:1.**

Depth, <i>d</i>	Base 12 feet.	Base 14 feet.	Base 16 feet.	Base 18 feet.	Base 20 feet.	Base 28 feet.	Base 30 feet.	Base 32 feet.
1	48	56	63	70	78	107	115	122
2	104	119	133	148	163	222	237	252
3	167	189	211	233	256	344	367	389
4	237	267	296	326	356	474	504	533
5	315	352	389	426	463	611	648	685
6	400	444	489	533	578	756	800	844
7	493	544	596	648	700	907	959	1011
8	593	652	711	770	830	1067	1126	1185
9	700	767	833	900	967	1233	1300	1367
10	815	889	963	1037	1111	1407	1481	1556
11	937	1019	1100	1181	1263	1589	1670	1752
12	1067	1156	1244	1333	1422	1778	1867	1956
13	1204	1300	1396	1493	1589	1974	2070	2167
14	1348	1452	1556	1659	1763	2178	2281	2385
15	1500	1611	1722	1833	1944	2389	2500	2611
16	1659	1778	1896	2015	2133	2607	2726	2844
17	1826	1952	2078	2204	2330	2833	2959	3085
18	2000	2133	2267	2400	2533	3067	3200	3333
19	2181	2322	2463	2604	2744	3307	3448	3589
20	2370	2519	2667	2815	2963	3556	3704	3852
21	2567	2722	2878	3033	3189	3811	3967	4122
22	2770	2933	3096	3259	3422	4074	4237	4400
23	2981	3152	3322	3493	3663	4344	4515	4685
24	3200	3378	3556	3733	3911	4622	4800	4978
25	3426	3611	3796	3981	4167	4907	5093	5278
26	3659	3852	4044	4237	4430	5200	5393	5585
27	3900	4100	4300	4500	4700	5500	5700	5900
28	4148	4356	4563	4770	4978	5807	6015	6222
29	4404	4619	4833	5048	5263	6122	6337	6552
30	4667	4889	5111	5333	5556	6444	6667	6889
31	4937	5167	5396	5626	5856	6774	7004	7233
32	5215	5452	5689	5926	6163	7111	7348	7585
33	5500	5744	5989	6233	6478	7456	7700	7944
34	5793	6044	6296	6548	6800	7807	8059	8311
35	6093	6352	6611	6870	7130	8167	8426	8685
36	6400	6667	6933	7200	7467	8533	8800	9067
37	6715	6989	7263	7537	7811	8907	9181	9456
38	7037	7319	7600	7881	8163	9289	9570	9852
39	7367	7656	7944	8233	8522	9678	9967	10256
40	7704	8000	8296	8593	8889	10074	10370	10667
41	8048	8352	8656	8959	9263	10478	10781	11085
42	8400	8711	9022	9333	9644	10889	11200	11511
43	8759	9078	9396	9715	10033	11307	11626	11944
44	9126	9452	9778	10104	10430	11733	12059	12385
45	9500	9833	10167	10500	10833	12167	12500	12833
46	9881	10222	10563	10904	11244	12607	12948	13289
47	10270	10619	10967	11315	11663	13056	13404	13752
48	10667	11022	11378	11733	12089	13511	13867	14222
49	11070	11433	11796	12159	12522	13974	14337	14700
50	11481	11852	12222	12593	12963	14444	14815	15185
51	11900	12278	12656	13033	13411	14922	15300	15678
52	12326	12711	13096	13481	13867	15407	15793	16178
53	12759	13152	13544	13937	14330	15900	16293	16685
54	13200	13600	14000	14400	14800	16400	16800	17200
55	13648	14056	14463	14870	15278	16907	17315	17722
56	14104	14519	14933	15348	15763	17422	17837	18252
57	14567	14989	15411	15833	16256	17944	18367	18789
58	15037	15467	15896	16326	16756	18474	18904	19333
59	15515	15952	16389	16826	17263	19011	19448	19885
60	16000	16444	16889	17333	17778	19556	20000	20444

**TABLE XXXIII.—CUBIC YARDS PER 100 FEET OF LEVEL
SECTIONS. SLOPE 1.5:1.**

Depth d	Base 12 feet.	Base 14 feet.	Base 16 feet.	Base 18 feet.	Base 20 feet.	Base 28 feet.	Base 30 feet.	Base 32 feet.
1	50	57	65	72	80	109	117	124
2	111	126	141	156	170	230	244	259
3	183	206	228	250	272	361	383	406
4	267	296	326	356	385	504	533	563
5	361	398	435	472	509	657	694	731
6	467	511	556	600	644	822	867	911
7	583	635	687	739	791	998	1050	1102
8	711	770	830	889	948	1185	1244	1304
9	850	917	983	1050	1117	1383	1450	1517
10	1000	1074	1148	1222	1296	1593	1667	1741
11	1161	1243	1324	1406	1487	1813	1894	1976
12	1333	1422	1511	1600	1689	2044	2133	2222
13	1517	1613	1709	1806	1902	2287	2383	2480
14	1711	1815	1919	2022	2126	2541	2644	2748
15	1917	2028	2139	2250	2361	2806	2917	3028
16	2133	2252	2370	2489	2607	3081	3200	3319
17	2361	2487	2613	2739	2865	3369	3494	3620
18	2600	2733	2867	3000	3133	3667	3800	3933
19	2850	2991	3131	3272	3413	3976	4117	4257
20	3111	3259	3407	3556	3704	4296	4444	4593
21	3383	3539	3694	3850	4006	4628	4783	4939
22	3667	3830	3993	4156	4319	4970	5133	5296
23	3961	4131	4302	4472	4642	5324	5494	5665
24	4267	4444	4622	4800	4978	5689	5867	6044
25	4583	4769	4954	5139	5324	6065	6250	6435
26	4911	5104	5296	5489	5681	6452	6644	6837
27	5250	5450	5650	5850	6050	6850	7050	7250
28	5600	5807	6015	6222	6430	7259	7467	7674
29	5961	6176	6391	6606	6820	7680	7894	8109
30	6333	6556	6778	7000	7222	8111	8333	8556
31	6717	6946	7176	7406	7635	8554	8783	9013
32	7111	7348	7585	7822	8059	9007	9244	9481
33	7517	7761	8006	8250	8494	9472	9717	9961
34	7933	8185	8437	8689	8941	9948	10200	10452
35	8361	8620	8880	9139	9398	10435	10694	10954
36	8800	9067	9333	9600	9867	10933	11200	11467
37	9250	9524	9798	10072	10346	11443	11717	11991
38	9711	9993	10274	10556	10837	11963	12244	12526
39	10183	10472	10761	11050	11339	12494	12783	13072
40	10667	10963	11259	11556	11852	13037	13333	13630
41	11161	11465	11769	12072	12376	13591	13894	14198
42	11667	11978	12289	12600	12911	14156	14467	14778
43	12183	12502	12820	13139	13457	14731	15050	15369
44	12711	13037	13363	13689	14015	15319	15644	15970
45	13250	13583	13917	14250	14583	15917	16250	16583
46	13800	14141	14481	14822	15163	16526	16867	17207
47	14361	14709	15057	15406	15754	17146	17494	17843
48	14933	15289	15644	16000	16356	17778	18133	18489
49	15517	15880	16243	16606	16969	18420	18783	19146
50	16111	16481	16852	17222	17593	19074	19444	19815
51	16717	17094	17472	17850	18228	19739	20117	20494
52	17333	17719	18104	18489	18874	20415	20800	21185
53	17961	18354	18746	19139	19531	21102	21494	21887
54	18600	19000	19400	19800	20200	21800	22200	22600
55	19250	19657	20065	20472	20880	22509	22917	23324
56	19911	20326	20741	21156	21570	23230	23644	24059
57	20583	21006	21428	21850	22272	23961	24383	24805
58	21267	21696	22126	22556	22985	24704	25133	25563
59	21961	22398	22835	23272	23709	25457	25894	26331
60	22667	23111	23556	24000	24444	26222	26667	27111

**TABLE XXXIII.—CORRECTIVE PERCENTAGE FACTORS FOR
TABLES OF LEVEL SECTIONS.**

To be applied when cross-sections are *not* level. See § 95.

Side slope = 1.5:1 or $\beta = 33^\circ 41'$.

Trans-verse surface slope.		$b=12$ feet and $d=$			$b=20$ feet and $d=$			$b=30$ feet and $d=$		
α°	Per-cent	10 feet.	20 feet.	50 feet.	10 feet.	20 feet.	50 feet.	10 feet.	20 feet.	50 feet.
5	9	% 1.9	% 1.8	% 1.8	% 2.1	% 1.8	% 1.8	% 2.3	% 2.0	% 1.8
10	18	8.2	7.7	7.5	9.0	8.0	7.6	10.0	8.4	7.7
15	27	21	20	19	23	21	20	26	22	20
20	36	46	44	43	51	45	44	57	48	44
30	57	327	324	317	358	336	321	400	354	326

Side slope = 1:1 or $\beta = 45^\circ$.

Trans-verse surface slope.		$b=12$ feet and $d=$			$b=20$ feet and $d=$			$b=30$ feet and $d=$		
α°	Per-cent	10 feet.	20 feet.	50 feet.	10 feet.	20 feet.	50 feet.	10 feet.	20 feet.	50 feet.
5	9	% 0.9	% 0.8	% 0.8	% 1.0	% 0.9	% 0.8	% 1.2	% 0.9	% 0.8
10	18	3.7	3.4	3.2	4.3	3.6	3.3	5.0	4.0	3.4
15	27	9.0	8.2	7.8	10.3	8.7	8.0	12.1	9.5	8.2
20	36	18	16	15	20	17	16	24	19	16
30	57	58	53	50	67	56	51	78	61	53

TABLE XXXIV.—ANNUAL CHARGE AGAINST A TIE, BASED ON THE ORIGINAL COST AND ASSUMED LIFE OF THE TIE; INTEREST COMPOUNDED AT 5%. (See § 217.)

Original cost of tie in cents.		Life of tie in years.																	
		3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20
20	7.34	5.64	4.62	3.94	3.46	3.09	2.81	2.59	2.41	2.26	2.13	2.02	1.93	1.85	1.77	1.71	1.65	1.60	
25	9.18	7.05	5.77	4.92	4.32	3.87	3.52	3.24	3.01	2.82	2.66	2.53	2.41	2.31	2.22	2.14	2.07	2.01	
30	11.02	8.46	6.93	5.91	5.18	4.64	4.22	3.89	3.61	3.38	3.19	3.03	2.89	2.77	2.66	2.57	2.48	2.41	
35	12.85	9.87	8.08	6.90	6.05	5.42	4.92	4.53	4.21	3.95	3.73	3.54	3.37	3.23	3.10	2.99	2.90	2.81	
40	14.69	11.28	9.24	7.88	6.91	6.19	5.63	5.18	4.81	4.51	4.26	4.04	3.85	3.79	3.55	3.42	3.31	3.21	
45	16.52	12.69	10.39	8.87	7.78	6.96	6.33	5.83	5.42	5.08	4.79	4.55	4.34	4.15	3.99	3.85	3.72	3.61	
50	18.36	14.10	11.55	9.85	8.64	7.74	7.03	6.48	6.02	5.64	5.32	5.05	4.82	4.61	4.43	4.28	4.14	4.01	
55	20.20	15.51	12.70	10.84	9.51	8.51	7.74	7.12	6.62	6.21	5.86	5.56	5.30	5.07	4.88	4.71	4.55	4.41	
60	22.03	16.92	13.86	11.82	10.37	9.28	8.44	7.77	7.22	6.77	6.39	6.06	5.78	5.54	5.32	5.13	4.96	4.81	
65	23.87	18.33	15.01	12.81	11.23	10.06	9.14	8.42	7.83	7.33	6.92	6.57	6.26	6.00	5.77	5.56	5.38	5.22	
70	25.70	19.74	16.17	13.79	12.10	10.83	9.85	9.07	8.43	7.90	7.45	7.07	6.74	6.46	6.21	5.99	5.79	5.62	
75	27.54	21.15	17.32	14.78	12.96	11.60	10.55	9.72	9.03	8.46	7.98	7.58	7.22	6.92	6.65	6.42	6.20	6.02	
80	29.38	22.56	18.48	15.76	13.83	12.38	11.25	10.36	9.63	9.03	8.52	8.08	7.71	7.38	7.10	6.84	6.62	6.42	
85	31.21	23.97	19.63	16.75	14.69	13.15	11.96	11.01	10.23	9.59	9.05	8.59	8.19	7.84	7.54	7.27	7.03	6.82	
90	33.05	25.38	20.79	17.73	15.55	13.92	12.66	11.66	10.84	10.15	9.58	9.09	8.67	8.30	7.98	7.70	7.45	7.22	
95	34.88	26.79	21.94	18.71	16.42	14.70	13.37	12.30	11.44	10.72	10.12	9.60	9.15	8.76	8.42	8.12	7.86	7.62	
100	36.72	28.20	23.10	19.70	17.28	15.47	14.07	12.95	12.04	11.28	10.65	10.10	9.63	9.23	8.87	8.55	8.27	8.02	
For each 5 cents, add }		1.836	1.410	1.155	.985	.864	.774	.703	.648	.602	.564	.532	.505	.482	.461	.443	.428	.414	.401

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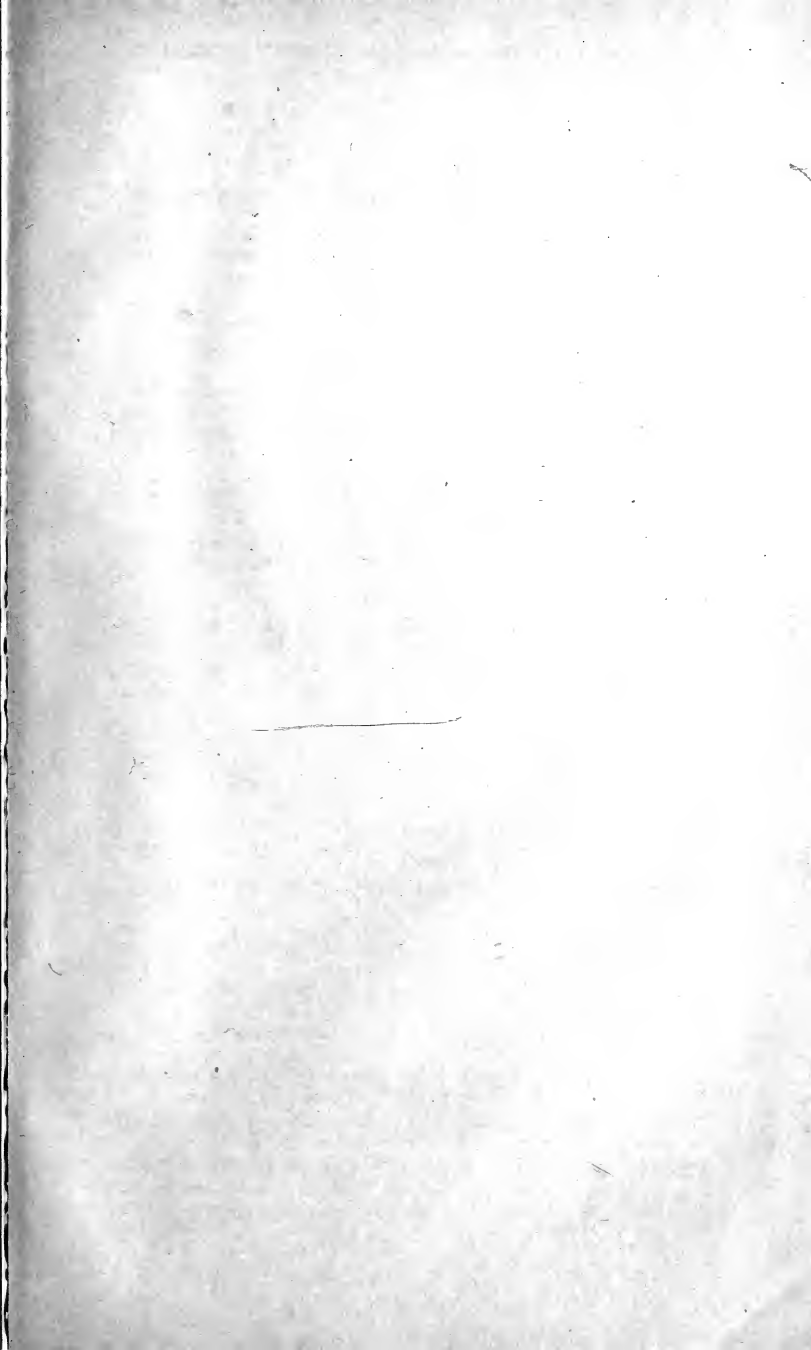
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